

# GEORGIA INSTITUTE OF TECHNOLOGY

OFFICE OF RESEARCH ADMINISTRATION

Date: 21 January 1970

## RESEARCH PROJECT INITIATION

Project Title: **Study of Design Parameters of Space Base and Space Shuttle Heat Rejection Systems**

Project No.: **B-1114**

Project Director: **Dr. William Z. Black and Dr. Wolfgang Wulff**

Sponsor: **NASA Manned Spacecraft Center, Houston, Texas 77058**

Agreement Period: From 21 January 1970 until 20 January 1972

Type Agreement: **Contract No. NAS 9-10415**

Amount: **\$99,969**

### Technical Monitor

**Mr. W. E. Simon  
Mail Code EP5  
NASA Manned Spacecraft Center  
Power Generation Branch  
Houston, Texas 77058**

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**GEORGIA INSTITUTE OF TECHNOLOGY**  
**OFFICE OF RESEARCH ADMINISTRATION**  
**RESEARCH PROJECT TERMINATION**

Date: September 29, 1972

Project Title **Study of Design Parameters of Space Base & Space Shuttle Heat Rejection Systems**

Project No: **E-25-609 (Old B-1114)**

Principal Investigator: **Dr. W. Z. Black & Dr. Wolfgang Wulff**

Sponsor: **NASA - Manned Spacecraft Center; Houston, Texas**

Effective Termination Date: August 31, 1972

Clearance of Accounting Charges: by August 31, 1972

Grant/Contract Closeout Actions Remaining: **Final Invoice & Closing Documents**  
**Final Report of Inventions**

Assigned to: School of Mechanical Engineering

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GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

B-1114

SCHOOL OF  
MECHANICAL ENGINEERING

February 5, 1970

Mr. William Simon  
Power and Propulsion Division EPS  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 1  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base and  
Shuttle Heat Rejection Systems".

I. General

During the first week of January, 1970, eight graduate students were interviewed and two PhD. candidates accepted as Research Assistants, in accordance with the contract. The Students are Mr. S. L. Yao and Mr. S. M. Morcos. They began work on this program officially on January 19, 1970.

The general strategy was mapped out for the system simulation. Firstly, it was decided to treat the non-linear system simulation as an initial value problem. This approach affords the uniform treatment of both the steady-state problem (Space Base) as well as the transient problem (Shuttle) and lends itself to the application of standard integration techniques. No computational instabilities or convergence problems need to be anticipated.

Secondly, it is planned to subdivide the numerical solution in elementary subtasks and to code these in subprograms. Such a structure allows great flexibility for later modifications in that only specific units need to be replaced, and it also greatly facilitates debugging of the program.

Thirdly, it was decided to develop concurrently a simplified analysis of the entire system. This analysis should serve to identify quickly relational trends between the system response and the governing parameters. It should also help to determine relative significance between overlapping transfer mechanisms.

II. Accomplishments

Six dependent variables describe the system: three fluid properties (pressure, temperature, velocity) and three temperatures (fin, tube, meteoroid protection), all as functions of time and space. The six pertinent differential equations were developed in a form suitable for computer solution (Runge-Kutta or Predictor Corrector Methods). The formulation is non-dimensional.

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Page 2  
Mr. W.Simon  
February 3, 1970

Computer Codes are available for the integration of simultaneous differential equations, for Simpson integration and for differentiation. A code for the solution of simultaneous algebraic equations was written and is being debugged. It will be used for solving radiosity equations.

"Direct exchange areas" for the radiative heat exchange between tube and fin were formulated and are being prepared for numerical evaluation.

The simplified system analysis was developed. The problem reduced to the solution of a single transcendental equation for the fin root temperature. A single system parameter describes the system operating conditions. The formulation of the simplified analysis seems to permit direct optimization.

A search of pertinent literature has been started. Three NASA publications and one reference manual written under a NASA contract have been collected and reviewed. The collecting of further references will be continued in future months.

A review has been initiated of appropriate relations for the Nusselt Number for laminar and turbulent flow of gases and liquids as well as liquid metals. The selected expressions will be programmed later in subroutines.

### III. Future Plans

Both Dr. Wulff and Dr. Black will visit Houston sometime during the first week in March for the first of the bi-monthly progress meetings. At that time they will present their second letter report. It was anticipated that this letter report was to be delivered to Mr. William Simon during his visit to Atlanta on February 6, but his visit was canceled.

During the next month, it is anticipated that the following work will be initiated:

A study will be made to determine an appropriate expression for meteoroid protection thickness. Expressions taken from NASA publication SP-78 and TRW Users Manual ER-6792 are already under consideration.

Computer programming will be continued for numerical calculations of the fin temperature distribution and system simulation. An interpolation subroutine will be written to provide fin, and tube fluid properties from tabulated values. Direct exchange areas in terms of the geometry of the tubes and fins will be programmed.



Page 3  
Mr. W. Simon  
February 5, 1970

Respectfully Submitted

W. Wulff  
Co-Investigator

W. Z. Black  
Co-Investigator

Approved:

S. P. Kezios, Director  
School of Mechanical Engineering

WZB:lb

GEORGIA INSTITUTE OF TECHNOLOGY

ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

March 3, 1970

Mr. William Simon  
Power and Propulsion Division EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 2  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of  
Space Base and Shuttle Heat Rejection  
Systems".

I. Accomplishments

Five subroutines have been programmed and debugged. These programs are now being tested with simulated input conditions to check their correctness. These five subroutines are particularly suited for use in the heat rejection study. They are:

1. Numerical evaluation of the first derivative.
2. Numerical evaluation of the second derivative. These two subroutines will be used for the evaluation of the discretized differential equations which simulate the performance of the heat rejection system.
3. Matrix Inversion - This subroutine will be used to determine the radiosity of each fin and tube element.
4. Interpolation routine - This subroutine will be used to evaluate coolant fluid and fin properties from tabulated values.
5. Integration by Simpson's Rule - This subroutine will be used to determine the net heat flux leaving the surface of the radiator by the radiation mode.

Equations for the direct exchange areas between tube and fin and two fin elements have been written, non-dimensionalized and checked against the results of a simplified geometry. These equations have been coded as a subroutine.

The simplified system analysis was continued. The simplified analysis is based on the one-dimensional fin and it is being used to seek an optimum number of tubes and an optimum tube length. The preliminary results indicate no optimum tube length while there is a optimum number of tubes.

The steady-state momentum, energy and continuity equations have been

Page 2

Mr. William Simon

March 3, 1970

written for the coolant fluid so that the initial conditions for radiator operation can be determined.

Expressions for meteroid protection thickness have been reviewed. An expression for protection thickness of radiator tubes has shown that a significant portion of the radiator weight can be attributed to the protection layer. A rather large range of thicknesses can be calculated depending upon the degree of conservatism of the designer.

## II. Future Plans

The simplified analysis will be continued in search of other radiator properties and geometries which will contribute to an optimum design.

The expressions for the direct exchange areas will be debugged.

Properties of several fin materials and coolant fluids will be tabulated for future use in the governing differential equations.

The expression for meteroid protection thickness will be programmed and debugged. A search of more recent literature will be undertaken for the purpose of finding reliable values for experimental constants which are used in the expression for protection thickness.

Respectfully Submitted,

W. Wulff  
Co-Investigator

W. Z. Black  
Co-Investigator

S.P. Kezios, Director  
School of Mechanical Engineering

3-4-70

WZB:lb

*File*

GEORGIA INSTITUTE OF TECHNOLOGY

ATLANTA, GEORGIA 30332

April 1, 1970

SCHOOL OF  
MECHANICAL ENGINEERING



Mr. William Simon  
Power and Propulsion Division EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058

Subject: Letter Report No. 3 NASA Contract NAS9-10415 "A Study of the Design Parameters of Space Base and Shuttle, Heat Rejection Systems."

I. Accomplishments

The five subroutines written last month and described in Letter Report No. 2 have been checked out successfully.

A complete system of heat rejection materials has been selected so that their thermophysical properties may be tabulated and programmed as subroutines. The system consists of:

1. Tube and fin material: copper or aluminum
2. Meteoroid protection material: Beryluim
3. Coolant fluid: liquid - Dow Corning 200  
gas - Nitrogen  
liquid metal - NaK

At the present time the properties are being accumulated for these materials. Power polynomials and other suitable functions are being fitted to represent the property data where this is feasible. Other property data will be numerically interpolated (Aitken algorithm) by a subroutine written and successfully tested for this purpose.

A program has been written for the solution of the coolant fluid energy, continuity and momentum equations. This program will provide the initial conditions for the governing differential equations of transient heat transfer.

A subroutine for the meteoroid thickness has been written and is presently being debugged.

The subroutine for the exchange areas between the fin and tube and two adjacent fin elements has been written and debugged. A special case of an infinitely long fin is being used to check the accuracy and validity of this program.

A short explanation of the contents and the theory behind the simplified analysis is included in this report. Although a simplified analysis was not a part of the contract, we are happy to provide details concerning its operation.

Page 2

Mr. William Simon

April 1, 1970

## II. Future Plans

In the coming month the governing differential equations will be programmed to provide time derivatives of the fin temperature, tube temperature, meteoroid protection temperature and coolant fluid properties.

The control program will be written and tested. The purpose of the control program will be to provide output and to terminate the integration process.

A subprogram will be written that will calculate the radiosity from each fin and tube element. This program will be used to evaluate the net radiant energy leaving each radiator element.

Respectfully submitted,

W. Wulff      //  
Co-Investigator

W. Z. Black  
Co-Investigator

Approved:

S. P. Kezios, Director  
School of Mechanical Engineering

WW/jt

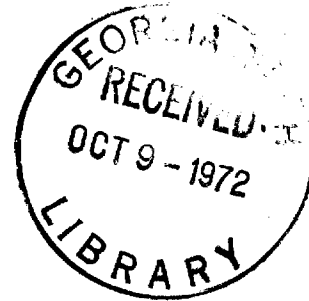
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GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

May 1, 1970

Mr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 4  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space  
Base and Shuttle Heat Rejection Systems"

I. Accomplishments

The subprogram for all six of the governing differential equations has been coded. The program includes the appropriate boundary conditions.

The subprogram which calculates the meteoroid thickness has been completed and successfully tested.

Power polynomials have been fitted to a total number of twelve properties as functions of the system temperature (and pressure or density in the case of coolant fluid properties). The curves for the following properties have been evaluated and punched on cards:

1. Coolant fluid properties for 1CS Dow Corning 200  
Thermal conductivity as a function of temperature  
Dynamic viscosity as a function of temperature  
Specific heat at zero pressure as a function of temperature.

The reference used for these properties is ASME Research Publication, Pressure-Viscosity Report Vol. II 1953

2. Fin properties for aluminum and copper and meteoroid protection properties for beryllium  
Thermal Conductivity  $k$  as a function of temperature  
Specific heat  $c$  as a function of temperature  
Modulus of elasticity as a function of temperature.

The reference used for  $k$  and  $c$  is Thermophysical Properties Research Center, Purdue University, Vol. and Vol. 2, Part II, Y. S. Touloukian, Editor.

The reference used for the modulus of elasticity data is TRW Property Data for Space Radiator-Condenser Design, Prepared under contract NAS 9-4884 April 1966.

The following units have been selected for several properties. They will be used consistently throughout the program.

dynamic viscosity	lbf hr/ft <sup>2</sup>
velocity	ft/hr
thermal conductivity	Btu/hr ft R
specific heat	Btu/slug R
density	slug/ft <sup>3</sup>

A subprogram for the evaluation of the radiative heat transfer between fin and tube as well as between fin, tube and other radiatively active surfaces has been formulated and coded. This program utilizes two previously written subprograms, the program for the radiative exchange areas and the matrix inversion program.

## II. Future Plans

The subprogram to evaluate the net radiant energy leaving each fin and meteoroid protection layer element will be continued.

The subprograms for all six governing equations will be checked and tested under simulated conditions.

An equation of state for Dow Corning 200 will be derived and programmed. The equation will be developed from the data of the isothermal compressibility and bulk modulus of the fluid.

A main program will be written which will call all property subroutines and check each property for strategic choices of pressure and temperature. The property value returned by the subroutine will be compared with values taken from references.

Respectfully submitted,

W. Wulff, Co-Investigator

W. Z. Black, Co-Investigator

Approved:

S. P. Kazios, Director  
School of Mechanical Engineering

GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

June 1, 1970

Mr. William Simon  
Power Generation Branch, EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 5  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space  
Base and Shuttle Heat Rejection Systems"

The major accomplishments of this project to date were outlined in a meeting held at MSC on May 27, 1970.

During the meeting it was decided to:

1. Use the 026 key punch exclusively.
2. Develop two main programs for the Calcomp plotting, one for the MSC and one for the Georgia Tech installation.
3. Consider Z93 as the surface coating material. Optical properties are available from Wright Patterson AFB reports, publications of Atomic International and of NASA Lewis.
4. Utilize existing computer programs for the treatment of the external radiant heat flux, Midwest Research Programs, and for the convective heating during ascent and re-entry, MSC Aerodynamics Section. Data transfer through cards or magnetic tape is acceptable.

Accomplishments

Work is continuing on programs for the six governing differential equations, for the fluid properties, for radiant flux and for the equation of state of the coolant fluid.

The programs for the thermal conductivity, specific heat, modulus of elasticity and the temperature variation of thermal conductivity for the fin, tube and protection material have been completed. The programs for the isobaric thermal expansion coefficient, isothermal compressibility, dynamic viscosity, thermal conductivity and specific heat of the coolant fluid have been completed.



Page 2  
Mr. Simon  
June 1, 1970

Future Plans

The development of the program to calculate radiant flux leaving each fin and protection element will be continued.

The two parts of the subprogram for the six differential equations will be combined after each part has been thoroughly checked.

Property subroutines for the surface coatings will be initiated. The properties sought will be hemispherical infrared emittance and hemispherical solar absorbtance.

The preparation for using NASA programs for external heat flux on the shuttle radiator system during ascent and re-entry and incident radiant flux on the space base radiator will be started.

Respectfully submitted,

W. Wulff, ~~Co~~-Investigator

W. Z. Black, Co-Investigator

Approved:

S. P. Kezios, Director  
School of Mechanical Engineering

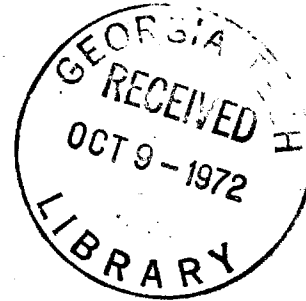
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SCHOOL OF  
MECHANICAL ENGINEERING

June 29, 1970

Mr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 6  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base and  
Shuttle Heat Rejection Systems"

Accomplishments

The derivative subprograms which evaluate the time derivatives of the system parameters have been completed and are being checked out.

Significant portions of the Main program and the Control subprograms have been written. Further progress has been made in the coding and testing of the Radiant Flux subprogram. The functions of these routines have been explained in our Letter Reports No. 3 and 4.

Work is continuing on the property subprograms. During the past month two property subprograms for Dow Corning 200 were coded and checked. They are: thermal conductivity as a function of temperature and dynamic viscosity as a function of fluid density and temperature. A third program for the specific heat at constant pressure has been coded but not checked. Reference material for the optical properties of surface coatings has been collected.

A literature search is being carried out in the course of developing and aerodynamic heating model for the ascent and descent phases of the shuttle simulation.

Future Plans

Work will be continued on the Radiant Flux and the property subprograms.

The Control, Main and Derivative subprograms will be combined and checked.

Further effort will be spent on the development of a suitable model for aerodynamic heating.

Page 2  
Mr. Simon  
June 29, 1970

Respectfully submitted,

W. Wulff, Co-Investigator

W. Z. Black, Co-Investigator

Approved:

S. P. Kezios, Director  
School of Mechanical Engineering

WZB:lb

GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

July 29, 1970

Mr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



SUBJECT: Letter Report No. 7  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems".

Accomplishments

All programs previously punched with the 029 keypunch have been converted to the 026 keypunch.

The subprogram for Dow Corning 200 properties have been coded and successfully checked. This subprogram consists of the following eight properties all as a function of density and temperature: dynamic viscosity, thermal conductivity, pressure, isothermal compressibility, isobaric expansion coefficient, specific heat at constant pressure, enthalpy and

$$\frac{1}{K} \left( \frac{dK}{dT} \right)$$

Work on the RADIANT FLUX subprogram has continued.

The CONTROL, MAIN, INITIALIZATION, and DERIVATIVE subprograms have been combined. Preliminary test runs involving the Runge-Kutta Integration have been executed. Further diagnostic runs are necessary. The output of transient variables in tabular form has been coded and successfully tested.

A preliminary aerodynamic heating model has been selected. Upper surface heat transfer coefficients reported by F. L. Guard and H. D. Schultz in ASME paper 70-HT/SpT-16 entitled "Space Shuttle Aerodynamic Heating Considerations" will be used. Atmospheric properties to be used in this portion of the program are presently being coded. Ascent and reentry convective heating loads provided by this model will be compared with the ones given by the SMD analysis before the final model is selected.

Future Plans

Work will continue on the aerodynamic heating and radiant flux subprograms. Properties for the coolant fluid helium will be coded. The

Page 2  
Mr. Simon  
July 29, 1970

skeleton form of the final program will be checked out further. Subroutines will be added to the final program only after the final program has been thoroughly checked.

Respectfully Submitted,

W. Wulff, Co-Investigator

W. Z. Black, Co-Investigator

Approved:

S. P. Kezios, Director  
School of Mechanical Engineering

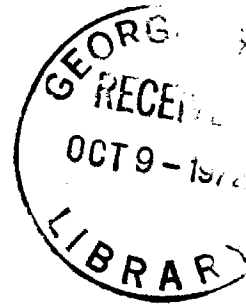
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GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

August 31, 1970

Mr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 8  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space  
Base and Shuttle Heat Rejection Systems

Accomplishments

The aerodynamic heating subprogram CONVEC has been coded. Both laminar and turbulent flow Nusselt numbers are calculated within this subprogram. A separate subprogram ATMOS which calculates atmospheric properties to be used in the CONVEC subprogram has been coded and checked. The properties used in ATMOS were taken from the 1962 Standard Atmosphere. CONVEC provides a Nusselt number for each fin and meteoroid protection nodes for both ascent and reentry of the shuttle vehicle.

Work on the RADIANT FLUX subprogram has continued. This subprogram now accounts for partial overlap of the fin by the meteoroid protection layer. Incident flux values for sun, earth and earth albedo given by the Midwest Research Program are presently being prepared for use with RADIANT FLUX program.

Further progress has been made on the integration of the CONTROL MAIN, INITIALIZATION and DERIVATIVE subprograms. Computations made within the DERIVATIVE subprogram have been checked, but further refinement of the main integration is needed. Parts of the METEOROID PROTECTION subprogram have been incorporated into the integrated program.

Programs for the equation of state, isothermal compressibility and isobaric expansion coefficient for Helium have been programmed. The equation of state program has been checked.

Future Plans

Work will continue on the aerodynamic heating and radiant flux subprogram.

Page 2  
Mr. William Simon  
August 31, 1970

The remaining properties for Helium will be programmed.

Further testing and checking of the assembled program will be carried out. Other subprograms will be incorporated into the assembled program as they are completed and tested.

Respectfully submitted,

W. Wulff  
Co-Investigator

W. Z. Black  
Co-Investigator

Approved:

S. P. Kezios, Director  
School of Mechanical Engineering

WZB:1b

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GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

October 30, 1970

Mr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 10  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

The check out of the PROPERTY subprogram for the coolant fluid Helium has been completed.

Work has begun on coding the optical properties for Z 93 which is to be used as a thermal control surface coating. In spite of an intensive literature search, no data on the temperature dependence of solar absorptance and infrared emittance could be found so far. We would appreciate any data provided by MSC while we shall continue to search for possible sources.

Properties of the atmosphere, required for the calculation of aerodynamic heating and coded in a subroutine called ATMOS, have been extended from an elevation of 300,000 feet to 500,000 feet. The properties for the extended range were taken from the latest issue (1966) of the U. S. Standard Atmosphere. Atmospheric properties returned by ATMOS have been checked for altitudes less than 300,000 feet as reported in Letter Report No. 8. Properties for altitudes greater than 300,000 feet are presently being checked.

The calculation of shuttle velocity and altitude as a function of time have been incorporated into subprograms called VELSH and ATLSH respectively. Presently velocity and altitude data supplied by MSC are being used as typical flight profiles. These data are read into the program in array form in order to accommodate any arbitrary flight profile.

Work has continued on the RADIANT FLUX subprogram.

The subprogram DERIV has been incorporated into the integrated program. There remains the phasing in of the subroutines for subprograms which supply the radiant flux (QRAD) and the convective flux (CONVEC).

Page 2  
Mr. Simon  
October 30, 1970

Further modification and checkout of the integrated program has been carried out. A typical output of the integrated program is included with this Letter Report.

Future Plans

Work will continue on the RADIANT FLUX and CONVEC subprograms. Coding of the optical properties of Z 93 will be continued. Once these properties have been coded and thoroughly checked, coding of the properties of the coolant fluid NaK will be initiated. Further modification of the integrated program will be made as completed subprograms are added to it.

Respectfully Submitted,

\_\_\_\_\_  
W. Wulff  
Co-Investigator

\_\_\_\_\_  
W. Z. Black  
Co-Investigator

Approved:

WZB:lb

\_\_\_\_\_  
S. P. Kezios, Director  
School of Mechanical Engineering

GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

December 2, 1970

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 11  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

All properties for the coolant fluid NaK have been coded and successfully checked. All computed property values deviate no more than 1.2% from the referenced values for temperatures ranging between 200F and 1400F.

Work has continued on the radiant flux subroutine (QRAD). The previously coded interpolation routine (YINT) serves to evaluate, at any time step, the incident radiant flux values from the results of MRI program.

Work has continued on the convective heating subroutine (CONVEC). A separate subroutine (NUS) called within CONVEC has been written to evaluate the Nusselt number for both free and forced, laminar and turbulent flow regimes. The check out of the subroutine NUS is in its final stages. When this subroutine has been completely checked, the subroutine CONVEC will be mated with the integrated program.

The major derivative subprogram DERIVM for the main integration is completed and checked. A lumped-parameter computing feature has been included for the computation of the conduction through the tube wall and the meteoroid protection layer. This computing feature reduces the computing time considerably in the case of low Biot numbers.

Future Plans

The two subroutines QRAD and CONVEC will be added to the integrated program. Further checking of the integrated program will then be necessary.

Page 2

Dr. Simon

December 2, 1970

Work will continue on the coding of the thermophysical properties of the surface coating Z93. The emittance and absorptance values for Z93 will consist of single values at infrared and visible wavelengths and they will be temperature independent. A further literature search carried out in the past month has not produced the temperature dependence of emittance or absorptance.

A large percentage of the effort during the month of December will go towards the writing of the final report for Phase I.

Respectfully submitted,

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W. Wolff  
Co-Investigator

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W. Z. Black  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

WZB:lb

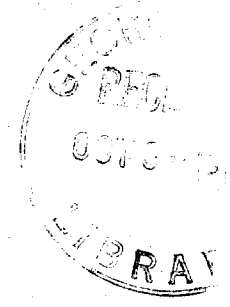
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GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

December 30, 1970

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



SUBJECT: Letter Report No. 12  
NASA Contract NAS 9-10415  
"A Study of Design Parameters of Space Base  
and Shuttle Heat Rejection Systems".

Accomplishments

The set of subprograms design to compute the aerodynamic heating during both the ascent and reentry has been completed and prepared for inclusion in the main deck assembly.

The set of subroutines for the computation of the radiant heat flux from the fin and flow channel has been modified to allow the mapping of different grids for the computation of the fin temperature distribution (fine grid) and of the radiant heat flux distribution (coarse grid). This set of subroutines is also completed and prepared for inclusion into the main deck assembly.

Several checks for internal consistency within the main integration have been carried out, based on local and global conservation of energy. The results are also being compared with the simplified analysis developed earlier in the program.

The coding of the optical properties of the surface coding Z93 is completed.

The major portions of the final report for Phase I have been drafted.

Future Plans

The month of January will be devoted to the completion of the final report, to the execution of several test runs and to a literature survey of appropriate optimization techniques. The simplified analysis will be further developed for the purpose of supporting the optimization.

Page 2  
Dr. William Simon  
December 30, 1970

Respectfully submitted,

\_\_\_\_\_  
W. Z. Black  
Co-Investigator

\_\_\_\_\_  
W. Wulff  
Co-Investigator

Approved:

\_\_\_\_\_  
S. P. Kezios, Director  
School of Mechanical Engineering

WWS:lb

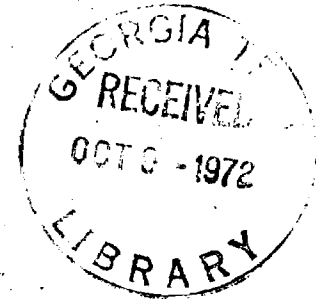
GEORGIA INSTITUTE OF TECHNOLOGY

ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

February 4, 1971

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 13  
NASA Construct NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

A draft of the final report for Phase I has been completed. A complete rough draft copy has been typed and typing of the final copy has started.

A final check of the aerodynamic heating program as well as other internal checks on the main deck assembly have been completed.

All nondimensional groups used to describe the performance of the radiator system have been assembled. Selected groups will be used as descriptive parameters during the optimization phase of the study.

Future Plans

A small amount of work will be needed for the completion of the final report for Phase I. Work will also continue on the selection of proper optimization techniques to be used during Phase II.

Respectfully submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

WW/WZB:jkt

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GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

April 5, 1971

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 15  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

The major accomplishments of the project during Phases I, II and III were summarized in a year end report presented at MSC on March 23, 1971. In addition to the year end report, separate meetings were held with Mr. John Orsag concerning the MRI program, Mr. Robert Dotts and Mrs. Dott Lee concerning the aerodynamic heating model and Mr. B. G. Cour-Palais concerning the meteoroid protection model.

During the meeting with Mr. Orsag, it was decided that the MRI program will be run for expected flight profiles and irradiation data will be stored on magnetic tape. These data will be entered into the assembled program as a tape input. In this regard the irradiation data supplied by the contractor has been successfully transferred to a Georgia Tech tape, called and printed out.

Ascent and reentry orbiter profiles supplied by Mrs. Lee of the Structures and Mechanics Division on March 23 have been used to generate a more recent convective heating rate for the orbiter vehicle. Comparisons are presently being made between these heating rates and those predicted by the SMD's unit sphere method.

The meeting with Mr. Cour-Palais produced general agreement between MSC's most recent meteoroid protection model and the model presented within the Phase report; however a modified model based on the Brinell hardness of the meteoroid protection material was suggested and the ranges of several empirical constants were modified.

The subprogram for the enthalpy of helium (HFL), which was not delivered at the time of the year end report, has been run and tested. A copy of this subprogram is included with this Letter Report.

Page 2  
Dr. Simon  
April 5, 1971

The improved simplified analysis reported in last months Letter Report has been coded and tested successfully.

Future Plans

The assembled version of the program will be modified in an attempt to reduce the computer run time. The subprograms which evaluate the convective and radiative heat fluxes from the radiator system will be tested separately and modified in an attempt to significantly reduce the number of times these subroutines are called, and also the time required for computation within these subprograms.

Respectfully Submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

WW:WZB/lb

GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

May 6, 1971

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 16  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

The major accomplishments during the month of April were summarized in a meeting held at Georgia Tech with Dr. Simon on April 26 and 27.

The results of the simplified analysis indicated that close tube-spacing and consequently significant radiative interactions between adjacent tubes and between tube and fin are associated with optimum radiator configurations. The simplified analysis was therefore modified to account for radiation from the flow channels and for the partial blocking of the radiant heat flux from channels and fins. This modification has been programmed and successfully tested.

The large-scale computer program optimization has been continued. Simplifications are being incorporated in the most time-consuming sub-programs.

The shape factor subroutine is being modified to include the tube-to-tube and the tube-to-far fin interactions; the view factor for the latter interaction has been coded and is being checked.

The aerodynamic heating model has been run for the cold wall case using the reentry and ascent profiles supplied by the Structures and Mechanics Division. A comparison of these results and those of SMD's unit sphere method are shown on the curves attached to this Letter Report.

The aerodynamic heating programs have been streamlined and simplified in order to minimize computer run time. The calculation required for a heating profile have been reduced by 50% with a resulting difference between the results of the old and new analysis being no greater than 0.75%.

Work has begun on the task of placing the entire program on FASTRAN. The programs necessary for entering the program on the drum, subdividing it into working units, and combining the units into several frequently used

Page 2  
May 6, 1971  
Dr. William Simon

versions have been coded and punched. These FASTRAN decks will be included with the other programs that are to be submitted to the contractor at the time of the year end report.

Future Plans

Work will continue on efforts to further reduce the computer run time particularly in the programs for the calculations of the radiant flux.

The program modification for the shape factor will be checked and integrated into the main program.

The work required to place the entire program on FASTRAN will continue.

The effort to find radiator geometries which lead towards an optimum heat loss per unit weight by using the simplified analysis will be continued.

Respectfully Submitted,

W. Z. Black  
Co-Investigator

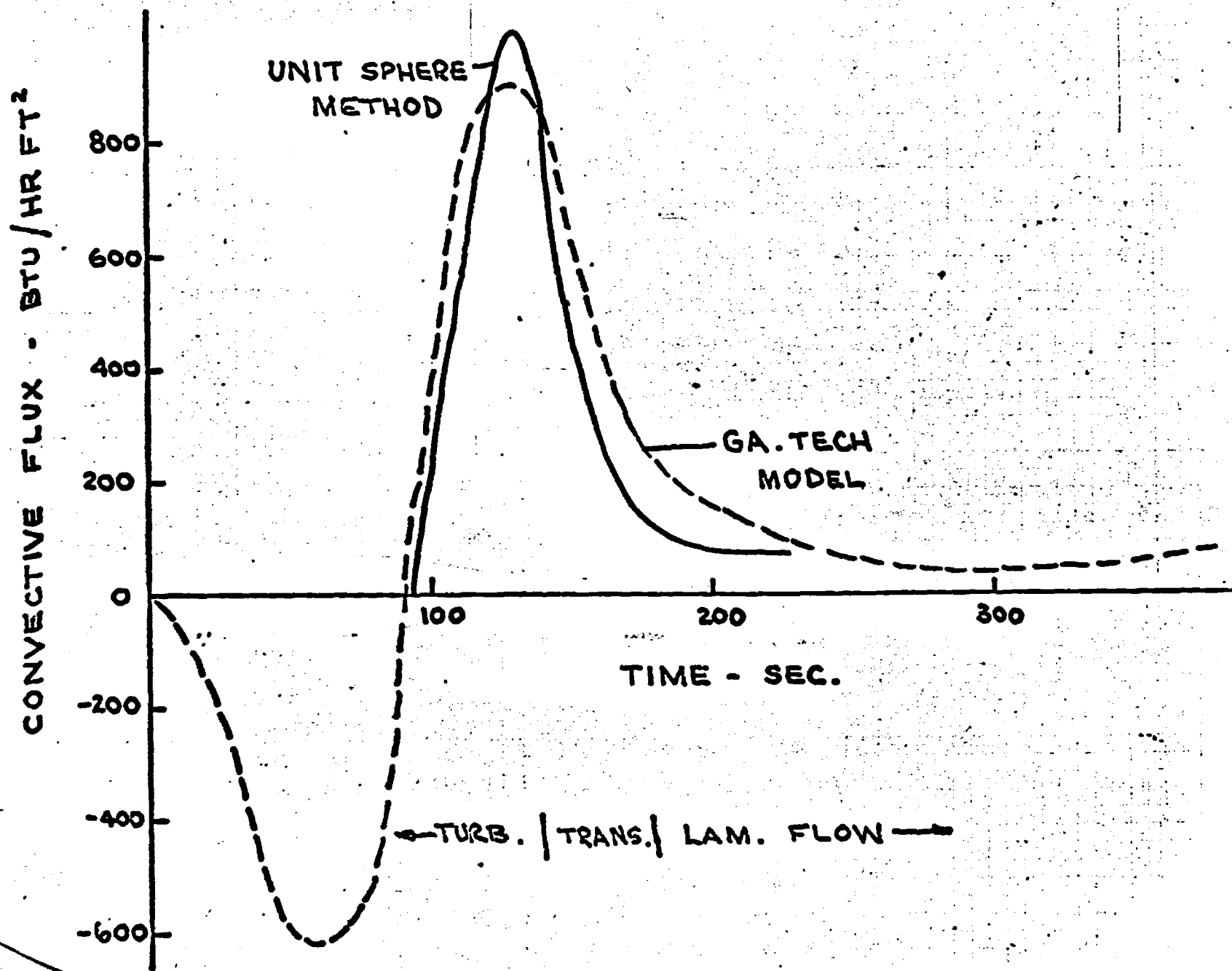
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Co-Investigator

Approved:

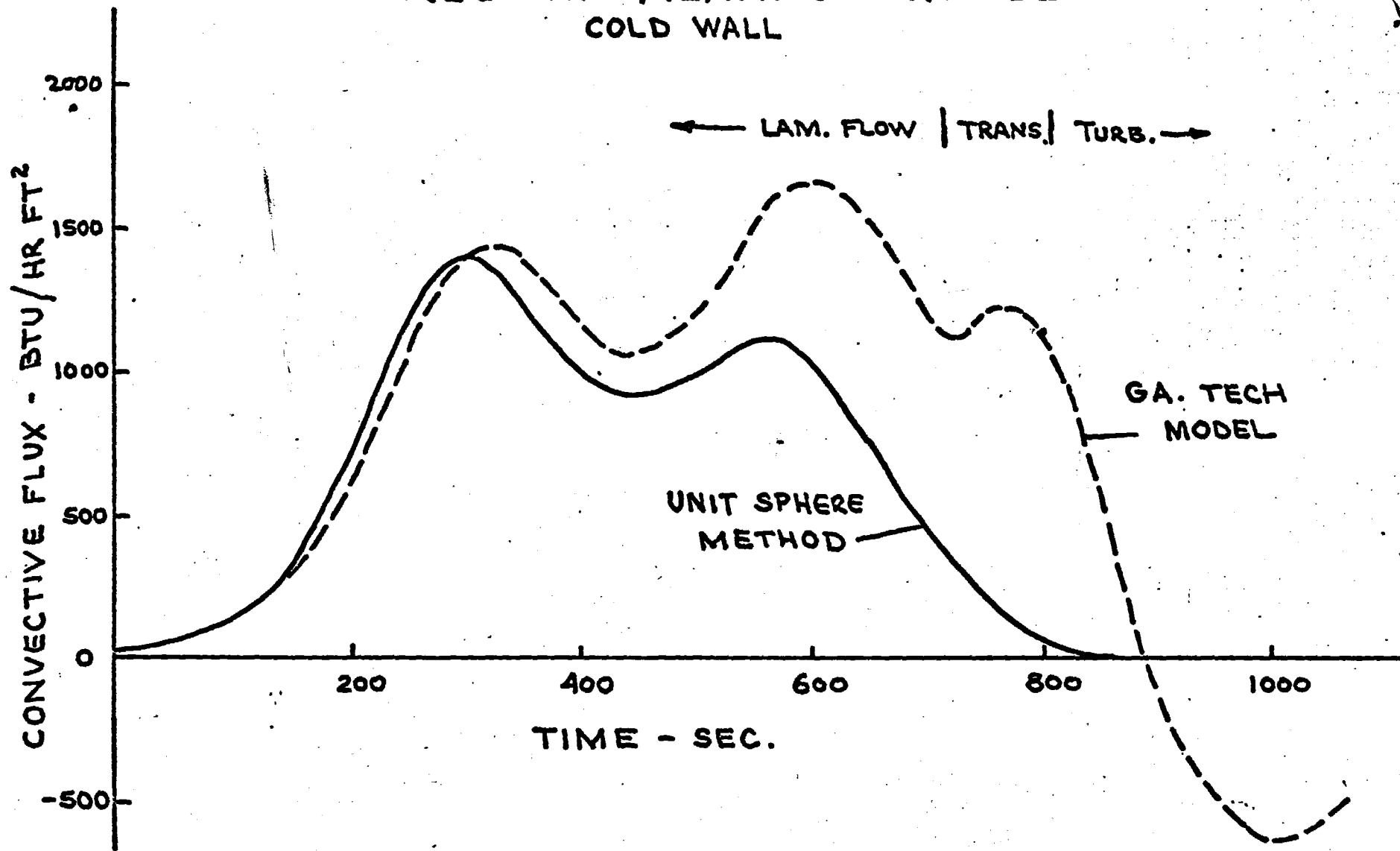
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S. P. Kezios, Director  
School of Mechanical Engineering

# ASCENT HEATING PROFILE COLD WALL



# REENTRY HEATING PROFILE COLD WALL

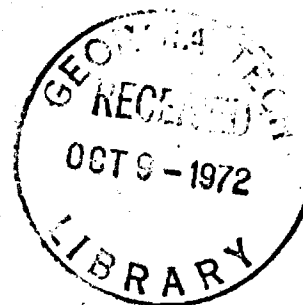


GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

June 7, 1971

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 17  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

Expressions have been written for the thermodynamic properties of the two 3-M Company coolant fluids FC-75 and FC-43.

Expressions for the shape factor between two elements on adjacent fins and for elements on adjacent tubes have been written and coded. The expression for the former has been checked.

The subprograms used to calculate the net radiant flux from the radiator system have been studied for possible ways to decrease the computation time. The shape factor subroutine execution time has been reduced by a factor of five.

The streamlining of the convective flux subroutines has been completed. A final reduction in the computation time of approximately 10% has been realized, so that the present computation time is about 40% of the original run time before simplification. The total time required to compute the instantaneous convective flux is now approximately 100 milliseconds.

As outlined in the Annual Report, the program has been separated into a permanent deck, property decks and a data deck all of which have been loaded on FASTRAN drums. Checks are presently being made between the results produced by the FASTRAN decks and results previously produced from card images.

Future Plans

Work required to reduce the run time of the radiation subroutines will continue.

The property programs for the coolants FC-75 and FC-43 will be coded and checked. The shape factor program for tube to tube exchange

Page 2  
Dr. W. Simon  
June 7, 1971

will be checked. The work on the FASTRAN decks will be completed.

A study will be started to find the location of the adiabatic plane which separates two tubes with unequal fluid temperatures and two neighboring fins with unequal incident radiant flux. If the location of such a plane is known, then the original program may be modified to allow more flexibility, because it may then be used to study cases where two adjacent coolant channels are not fed by a common manifold.


The effort to find radiator geometries which lead towards optimum heat loss will be continued.

Respectfully submitted,

  
W. Z. Black  
Co-Investigator

  
W. Wulff  
Co-Investigator

Approved:

  
S. P. Kezios, Director  
School of Mechanical Engineering

WW/WZB/lb

CC: Mr. Martial Davoust



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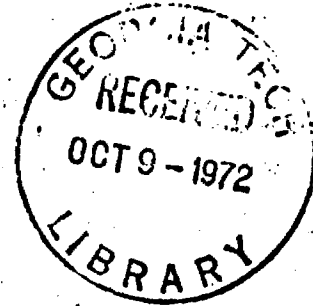
GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

*B 1114 / Black*

July 7, 1971

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 18  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

The thermodynamic properties for the two coolant fluids FC 43 and FC 75 have been coded and checked. The resulting property subroutines have been loaded on FASTRAN drums.

The work on the FASTRAN decks has been completed with all existing program elements loaded on FASTRAN drums. Several preliminary checks have been made between the output produced from the card images and the FASTRAN decks. More extensive runs and comparisons involving realistic input parameters suggested by Dr. Simon will be made in the coming months.

The effort to reduce the computation time required in the radiant flux subprograms has been completed.

The cause of unrealistic oscillations in the fin radiosity has been traced to the evaluation of the shape factor between tube and fin elements closest to the tube. Since the shape factor between closest pairs of elements is typically several orders of magnitude larger than any other shape factor, the integration necessary for the determination of the radiant flux resulted in sizeable truncation errors. The current shape factor analysis replaces the local value with a mean value and as a result the oscillation in radiant flux has disappeared. The differential view factor integrated along the tube, including the mean value result, checks within a fraction of one percent of the shape factor for a fin element with respect to all of the tube. Expressions for all shape factors have been completely checked. The shape factors evaluated in the subroutine SHAPEF are now:

1. Tube to adjacent fin.
2. Tube to far fin.
3. Tube to tube.
4. Fin to adjacent fin.

Page 2

Dr. Simon

July 7, 1971

The simplified analysis has been extended to account for the blockage or radiant energy by the tubes. As a result the analysis now requires four parameters rather than three for the specification of the system geometry.

#### Future Plans

The analysis started last month for the location of the adiabatic plane separating two flow channels with unequal fluid temperatures will be continued. Work to date has concentrated on two cases. The first case involves a single flow channel which forms a "U" shape and encloses a fin element. The second case is that of a fin element which separates two channels which have known, but different, inlet temperatures and flow rates.

Work will continue on the simplified analysis and on the determination of operating conditions for a series of anticipated input parameters. The simplified analysis will be used to select areas of operating conditions which appear to lead towards optimum operation.

Respectfully submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved: \_\_\_\_\_

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S. P. Kezios, Director  
School of Mechanical Engineering

WW/WZB:lb

CC: Mr. Martial Davoust

GEORGIA INSTITUTE OF TECHNOLOGY

ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

August 3, 1971

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058

Subject: Letter Report No. 19  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."



Accomplishments

Modifications to the assembled program are being made so that output can be obtained when the coolant fluid is assumed to be incompressible. The original program encounters division by zero whenever the isothermal compressibility is set equal to zero. This situation occurred for the first time when the incompressible fluids FC43 and FC75 were used.

Global energy balances are being made on radiator system as a check on the output data of the assembled program.

Several sections of the final report have been written. These sections include: thermodynamic properties of the coolant fluids FC43 and FC75 and the development of the shape factor between two adjacent tubes, between two elements on adjacent fins and between the tube and an element on the far fin.

The values for incident solar flux and earth albedo have been added to the computer printout when the radiator system is in orbit.

As reported in Letter Report No. 18 a separate program has been written for the determination of the location of the adiabatic plane separating two flow channels which are connected to two different manifolds. This program, called ADIABH, has been coded and is presently being checked.

Work has continued on the modifications to the Simplified Analysis and on the optimization programs. Six additional, ordinary, first-order differential equations were developed and coded and are being checked to produce the rates of change of coolant exit temperature with respect to three non-dimensional fin system parameters. These rates indicate which parameters affect the system performance most effectively. These rates must also be zero at an optimum.

Page 2  
Dr. W. Simon  
August 4, 1971

Future Plans

In the next several months the major portion of the effort will be concentrated on the Simplified Analysis and the optimization techniques. Production runs will be made from the remote terminal.

Respectfully submitted,

\_\_\_\_\_  
W. Z. Black  
Co-Investigator

\_\_\_\_\_  
W. Wulff  
Co-Investigator

Approved:

\_\_\_\_\_  
S. P. Kezios, Director  
School of Mechanical Engineering

WW/WZB:1b

GEORGIA INSTITUTE OF TECHNOLOGY

ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

September 9, 1971



Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058

Subject: Letter Report No. 20  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

The program ADIABH reported in Letter Report No. 18 has been completed and checked.

Further checks of the main program output have been made. Global energy balances on the fin system have uncovered several program errors. Corrections have been made on the computer cards and on the programs stored on FASTRAN drums.

Work has begun on the selection of reasonable input variables so that production runs may be made. These runs will be made from the remote terminal once the checks of the global energy balances produce consistent results.

The development of the optimization has been continued and the influence coefficients of the radiator performance with respect to the geometric system parameters are coded and checked out. Work has begun on the Newton-Raphson algorithm used to seek the geometric parameters of optimum performance.

Future Plans

Work on the simplified analysis and the optimization techniques will continue.

The program will be modified to accept tape input data from the MRI program. Presently incident solar and infrared flux values are read in as

Page 2  
September 9, 1971  
Mr. Simon

program input. These read statements will be removed and the program modified to read this data from taped output of the MRI program.

Respectfully submitted,

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W. Z. Black,  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

WW/WZB:1b

CC: Mr. Martial Davoust

GEORGIA INSTITUTE OF TECHNOLOGY

ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

October 5, 1971

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 21  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

Checks of a global energy balance applied to the fin system have uncovered several program errors which have been corrected.

Rough drafts of two sections of the final report have been written. These sections include the description of the program ADIABH which serves to compute the location of the adiabatic plane between adjacent tubes with non-symmetric flow conditions and the formulation of additional shape factors which were not included in the final report submitted at the end of the first year.

The subroutine QINCID has been modified so that incident radiant flux data may be entered into the program from cards or from tapes. The tape input option will allow irradiation data to be transferred from the MRI program. The format of the tape data was assumed to be identical to that of the sample tape given to Georgia Tech earlier this year with one exception. A group number must be supplied to each data set so that the proper data set may be selected from several groups that may be stored on the tape.

At the time of modifying QINCID, the number of flux values that must be specified for each side of the tube has been increased from three to six. It is felt that this increase will lead to a more representative value of the incident flux on the tube.

A print-out copy of the simplified analysis program and a description of the program operation are being prepared and they will be given to Dr. Simon in a meeting to be held in Houston within the next month.

Page 2  
October 5, 1971  
Dr. Simon

Optimization on the basis of the simplified analysis has been continued.

Personnel changes took place on Sept. 20, 1971 as follows:  
Mr. W. W. Carr, a Ph.D. candidate and Mr. J. R. Huntley, a M.S. student, were added to the project. Mr. S. L. Yao is returning to Taiwan and is no longer employed on the contract.

#### Future Plans

Work will continue on the simplified analysis and the optimization program.

The program ADIABH will be modified to accept incident flux data from tape.

Fastran decks will be assembled and will be given to Dr. Simon in this month's meeting if they are completed by that time.

Production runs will be started when the remaining internal program checks have been completed.

Respectfully submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

WW/WZB:jp



E-25-609

GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

November 3, 1971

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 22  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

A progress meeting was held in Houston on October 20, 1971. Present were T. Redding, W. Dusenbery, W. Chandler, W. Simon, W. Black and W. Wulff. A brief summary of the progress made to date and the current status of the contract were given. The contents of the new simplified analysis were discussed and a copy of the program was given to Dr. Simon. Later a test run of the simplified analysis was made.

During the meeting the adaptability of the program to physical cases that deviate from the basic geometry of the original analysis were discussed. Priorities were established on work to be completed in the future. These priorities were:

1. Provide for incident fluxes to be read from a MRI tape to be supplied by the contractor.
2. Integrate the separate program ADIABH into the main program so that individual runs are not necessary.
3. Expand the use of the program ADIABAH to cover a "U" shaped tube and more than two adjacent, asymmetrically loaded tubes.
4. Modify the main program to accept fluid inlet temperatures and mass flow rates that vary with time.

The possibility of a no cost extension to the contract was discussed. The co-investigators were asked to determine the number of man-months of funding that will be left in the contract budget if the present funding level is continued. The co-investigators were also asked to determine how far they would be able to proceed through the priority list before the present contract expired.

Page 2  
Dr. Simon  
November 3, 1971

The co-investigators intend to submit a letter in the near future requesting a no cost extension of the contract. There will be funds remaining in the budget to support the current effort for 60 additional days plus 30 days for the writing of a final report beyond the expiration date of the present contract. The letter will also include the goals to be achieved during the extension period. The co-investigators also intend to complete the first item on the list of priorities above before the contract expires. The remaining three items will form the basis of the work to be completed during the extension period.

#### Accomplishments

Work has already begun on the insertion of the MRI flux data into the main program. The Georgia Tech program has been modified to accept MRI input from magnetic tape as was reported in Letter Report No. 21. Changes in the program logic are being made so that shadowing of the tube by the fin will be accounted for in the Georgia Tech program. The program assumes that the flux output for the n-sided polygon specified by the MRI program has the fin orientated so that it is attached to the polygon between the surfaces labeled as 1 and n. Further modifications of the Georgia Tech program will produce consecutive runs for a curved radiator panel if the user supplies the angles between the normal to the fin elements.

The separate program ADIABH now accepts flux data from the MRI program. Whenever two adjacent fin elements are not in one plane, the program assumes that the fin radiates only from the convex side of the fin and the concave side does not radiate.

Another section of the final report has been written in rough draft form. The section describes the FASTRAND program and it also gives several examples of FASTRAND programs with the resulting output.

The internal program checks that were first reported in Letter Report No. 19 are being continued. Separate energy balances on the tube, fluid and fin have been completed. Additional runs have been made so that more detailed output may be obtained and the terms of energy balances checked more thoroughly. As a result of these energy balances, the various energy terms determined by a desk calculator check within a few percent of the computer results under both transient and steady state cases.


The program output has been modified to include two additional terms: the energy stored in the radiator panel and the energy conducted into the fin from both the inlet and exit manifold.


Page 3  
Dr. Simon  
November 3, 1971

Future Plans


Future work will concentrate on the optimization program, on the effort to integrate the MRI program and on the check of the internal energy balances.

Respectfully submitted,

  
\_\_\_\_\_  
W. Z. Black  
Co-Investigator

  
\_\_\_\_\_  
W. Wulff  
Co-Investigator

Approved:

  
\_\_\_\_\_  
S. P. Kezios, Director  
School of Mechanical Engineering

WW:WZB/lb

CC: Mr. Martial Davoust

E-25-609

GEORGIA INSTITUTE OF TECHNOLOGY  
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SCHOOL OF  
MECHANICAL ENGINEERING

December 15, 1971



Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058

Subject: Letter Report No. 23  
NASA Contract NAS 9-10415  
"A study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

The internal program checks first reported in Letter Report No. 19 have been continued. A detailed check of the terms in the energy equation for the coolant fluid has indicated that the truncation error in the equation for the time rate of temperature change for the first tube element is too large to represent the local axial temperature drop in the fluid. This equation has been modified and it is presently being checked.

The optimization program based on the simplified analysis has been continued. Sixteen of the eighteen derivatives of the outlet fluid and fin base temperatures with respect to the design parameters  $U$ ,  $V$ ,  $N_c$  have been programed and completely checked.

The MRI program has been integrated into the main program. Test cases have been run which read incident flux data from an MRI tape.

Future Plans

The final two derivatives needed to complete the optimization technique will be programed and checked.

Dr. William Simon  
Letter Report No. 23  
December 15, 1971  
Page 2

The program to extend the application of ADIABH to cases of more than two parallel tubes or counterflow through two or more "U" shaped tubes will be initiated.

Respectfully Submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

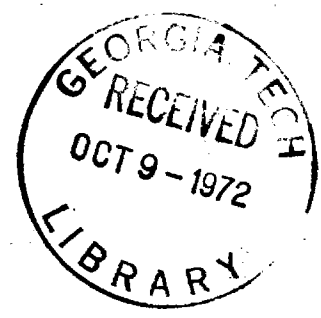
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GEORGIA INSTITUTE OF TECHNOLOGY  
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SCHOOL OF  
MECHANICAL ENGINEERING

January 6, 1972



Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058

Subject: Letter Report No. 24  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space  
Base and Shuttle Heat Rejection Systems."

Accomplishments

The expansion of the subprogram ADIABH listed as item #2 in the letter to Dr. Simon dated Nov. 12, 1971 which itemizes the goals of the no-cost extension of this contract has been started. The work completed to date involves the formulation of equations which will extend the use of ADIABH beyond the present two parallel tube cases to one involving an arbitrary number of parallel tubes which are either straight or U-shaped.

The subprogram QINCID which averages the radiant flux over the flow channels and acts as a transfer point between the MRI and main program has been modified so that incident fluxes may be calculated for a radiator panel that does not lie in one plane. The curved panel is assumed to be composed of a series of continuous plane segments. The user is expected to supply angles between the normals to the adjacent plane segments as program input. Test runs have been made of these modifications and the results have been completely checked.

The modification to the coolant fluid energy equation for the first tube element reported in Letter Report No. 23 has been made. This modification has resulted in some improvement in the global energy balance for the entire radiator system, but a difference between the steady state energy leaving the system by radiation and the energy lost by the coolant fluid of approximately 20% still remains. Modifications to the energy equations will be continued to obtain internal consistency in the energy balance.

All but two derivative programs needed for the optimization have been completed and verified.

Dr. William Simon  
Letter Report No. 24  
January 6, 1972  
Page 2

#### Future Plans

The above mentioned modifications to the subprogram ADIABH will be programmed and checked.

The work on the optimization program and on the program changes necessary to achieve consistency checks of the global energy balances will be continued.

Respectfully Submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

WZB:jp

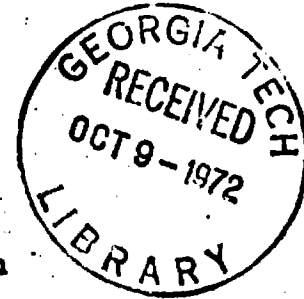
cc:Mr. Martial Davoust

E-25-609

GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

February 8, 1972



Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058

Subject: Letter Report No. 25  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems."

Accomplishments

Internal program checks of the global energy balance have indicated large truncation errors in the temperature field calculation at the tube-manifold intersection, as a result of fluid inlet temperature changes. The program was modified to treat the entire fluid flow field quasi-statically rather than only the velocity and pressure fields. This allows for better fluid and tube wall temperature coupling resulting in a more gradual decrease in the coolant fluid temperature near the inlet to the tube. The modifications are justified because the fluid residence time is small compared to the equilibration period of the radiator system.

The final two derivatives of the coolant fluid temperature that are used in the optimization program have been coded and checked. A Newton-Raphson technique has been coded and is being verified to iterate toward system parameters that result in maximum heat rejection for the constraint of constant system area.

The programming of ADIABH for cases of more than two parallel tubes has been completed and checked. The current version of the program is limited to ten adjacent tubes although this restriction may be easily relaxed. Any combination of coolant flow rates, inlet fluid temperatures and sink temperatures which lead to a condition for which there is no adiabatic interface is identified with diagnostic print-out. This situation does not produce a fin system which adheres to the boundary conditions of the present program.



Dr. William Simon  
February 8, 1972  
Page 2

### Future Plans

A progress report meeting has been tentatively scheduled for March 1, 1972 in Houston. Both Drs. Wulff and Black will attend.

The checks of the global energy balances will be continued.

The optimization program will be expanded to include the identification of system parameters which yield maximum heat loss for a given system volume.

The program ADIABH will be expanded to include the case of "U" shaped tubes. Once this change has been made the program will be inserted into the main program unit.

• Respectfully Submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

E-25-609

GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

March 3, 1972

Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058



Subject: Letter Report No. 26  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base  
and Shuttle Heat Rejection Systems"

Accomplishments

Global Energy Balance checks have been continued. It was found that the treatment of the coolant fluid flow as a quasi-steady process did not alter the results, and that there were two coding errors in the calculation of the volumetric heat capacity and of the check itself. The over-all balance does not yet check completely. Checking is continued.

The radiator system optimization program was executed under the constraint of constant total projected area. The results indicate that the optimum system has short thick fin panels and long coolant tubes; the optimum is not realistic. A meaningful optimum process should involve additional constraints. Optimization under constant projected area constraint should be performed for fixed, pre-selected fin panel thickness.

Optimization under constant thickness, constant area and under constant system weight has begun, coding of the program modifications are in progress.

The program which calculates the location of the adiabatic plane separating two adjacent flow channels for the case of U shaped tubes has been checked.

Future Plans

Optimization of the fin system will be continued under two different sets of conditions:

- (1) Minimum weight
- (2) Minimum area with given panel thickness

Dr. William Simon  
March 3, 1972  
Page 2

The program ADIABH and supporting subroutines which call the MRI program for input of sink temperatures will be inserted into the main program unit.

Respectfully Submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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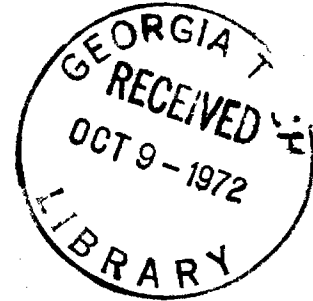
S. P. Kezios, Director  
School of Mechanical Engineering

WZB:jel

GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

SCHOOL OF  
MECHANICAL ENGINEERING

April 14, 1972



Dr. William Simon  
Power Generation Branch EP5  
National Aeronautics and Space Administration  
Manned Spacecraft Center  
Houston, Texas 77058

Subject: Letter Report No. 27  
NASA Contract NAS 9-10415  
"A Study of the Design Parameters of Space Base and  
Shuttle Heat Rejection Systems."

Accomplishments

Program changes discussed at the meeting MSC on Monday, March 6, 1972 were carried out as follows:

Input units for mass have been changed to lbm. All other input units are consistent with those listed in Letter Report No. 4. Also the units for input parameters used for the determination of the meteoroid protection thickness will remain in the metric system, since a large majority of the available experimental values are tabulated in the metric system.

The program ADIABH has been checked as a separate program unit. This unit is currently being integrated into the large program.

The original plans for supplying a CALCOMP routine that would plot system results have been dropped due to an incompatibility between the plotting systems of Georgia Tech and NASA. The results to be plotted for purpose of the final report will be completed by hand.

The check of the global energy balances been continued. Several test runs have been completed using various grid sizes. Results from these runs indicate that fluid and fin temperature are rather insensitive to grid size while the unbalance in energy terms is more sensitive to a change in grid size. For example using a typical fin system, a 5 x 5 fin grid size has an unbalance in the energy terms (energy storage, conduction from manifold, enthalpy drop of coolant fluid, net radiant flux) of less than 7%. For the identical system the selection of a 9x9 grid reduces this unbalance to about 1%. The percentage change in the outlet fluid temperature for the same change in grid size, however, is only 0.2%. The changes in the temperature for the interior fin nodes are insignificant.

It is felt that the unbalance can be attributed to the truncation errors involved in attempting to fit a parabola through the interior fin nodes.

April 13, 1972

This error is particularly significant at the manifold-fin interface where the slope of the temperature profile and the radiant flux is quite steep.

Typing of the rough draft of the final report has been started with approximately 100 pages of text completed. Descriptions of the simplified analysis, the program ADIABH, additional property subroutines and the control deck for mass storage runs are currently being written.

The program logic necessary for accepting variable inlet fluid temperatures and variable inlet mass flow rates has been inserted in the main program. A test case for variable inlet properties is currently being used to check this addition to the program.

The most recent test runs of the large program unit have resulted in a ratio of real time to computation time of approximately 10 to 1.

#### Future Plans

Work will continue on writing and typing of the final report.

The program ADIABH will be integrated into the main program and checked.

Respectfully submitted,

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W. Z. Black  
Co-Investigator

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W. Wulff  
Co-Investigator

Approved:

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S. P. Kezios, Director  
School of Mechanical Engineering

E-252609  
Interim report

GEORGIA INSTITUTE OF TECHNOLOGY  
SCHOOL OF MECHANICAL ENGINEERING  
ATLANTA, GEORGIA

STUDY OF DESIGN PARAMETERS OF SPACE BASE  
AND SPACE SHUTTLE HEAT REJECTION SYSTEMS

INITIAL PHASES - I, II and III

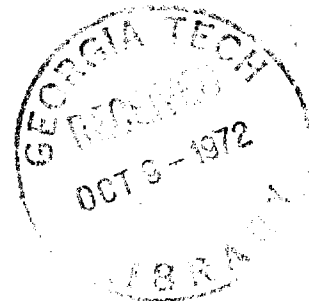
Contract No. NAS 9-10415

by

William Z. Black and Wolfgang Wulff

Sponsored by the  
Power Generation Branch  
Manned Spacecraft Center  
NATIONAL AERONAUTICS AND SPACE ADMINISTRATION  
Houston, Texas

February 1971



STUDY OF DESIGN PARAMETERS OF SPACE BASE  
AND SPACE SHUTTLE HEAT REJECTION SYSTEMS  
INITIAL PHASES - I, II and III

by

William Z. Black and Wolfgang Wulff

School of Mechanical Engineering  
Georgia Institute of Technology  
Atlanta, Georgia 30332

February 1971

Sponsored by the  
Power Generation Branch  
Manned Spacecraft Center  
NATIONAL AERONAUTICS AND SPACE ADMINISTRATION  
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Director, School of Mechanical Engineering

## FOREWORD

This report covers the first year phase of a two year research project carried out by the School of Mechanical Engineering at the Georgia Institute of Technology, Atlanta, Georgia for the NASA Manned Spacecraft Center, Houston, Texas, under Contract No. NAS 9-10415. The project title is "Study of Design Parameters of Space Base and Space Shuttle Heat Rejection Systems". The work is monitored by Dr. W. E. Simon of the Power Generation Branch of NASA MSC, Houston, Texas, and was carried out by Dr. W. Z. Black and Dr. W. Wulff as Co-Investigators and Mr. S. M. Morcos and Mr. S. L. Yao, both graduate students, all of the School of Mechanical Engineering under the direction of Dr. S. P. Kezios.

The work carried out by Dr. W. Z. Black is reflected in Part I, Sections A. d and e, and in Part II Chapters 2, 7, 8 and 13. Dr. Black is responsible for the coding of programs No. 14 through 20 and for the supervision of coding programs No. 37 through 46 which was carried out by Mr. Morcos. Dr. Wulff is responsible for the general computer program structure as described in Part I, the analyses in Part II, Chapters 1 and 3 through 6 and 9 through 12, as well as the numerical analysis in Part III, Chapters 14 through 19. Dr. Wulff coded programs No. 1 through 7, 23 through 28 and supervised Mr. Yao in coding programs No. 8 through 13 and No. 29 and supervised Mr. Morcos in coding programs No. 30 through 36 and No. 47 and 48. Dr. Wulff is also responsible for the Simplified Analysis.



The first year program is aimed at the numerical simulation of the Space Radiator System while the second year of the program will be devoted to the system optimization.

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## SUMMARY

A transient heat transfer analysis was carried out on a space radiator heat rejection system exposed to an arbitrarily prescribed combination of aerodynamic heating, solar, albedo and planetary irradiation. The radiator system consists of equally spaced parallel coolant flow channels, all in one plane and connected by plane fin panels of trapezoidal cross-section, symmetric with respect to two normal planes, one passing through the tube axis, the other through the center between adjacent tubes. Investigated was one typical tube-fin element and the result extended over the entire system, on the basis of the above symmetry.

The analysis permits to consider both gaseous and liquid coolant fluids, including liquid metals. The flow channels are covered with a meteoroid protection layer whose thickness is predicted. The entire radiator system is covered with the same passive thermal control coating with optically diffuse but wavelength and temperature dependent optical properties.

The major results of the analysis are the prediction of both transient and steady-state, two-dimensional temperature profiles, the local and total heat rejection rates, and the coolant flow pressure drop in the flow channel, and of the total system weight and the protection layer thickness.

A computer program consisting of 48 program units was coded to execute the numerical solution of the system of differential equations occurring in the analysis and to predict principal design parameters. The modular program structure permits readily later modifications.

A simplified analysis was carried out to aid the detailed analysis as well as the optimizations planned for the second program phase. This analysis led to a small computer program. Regarding the heat rejection rates, its results are, for two test cases carried out so far, within 4% in agreement with the results of the detailed analysis.

The analysis developed so far will be an integral part of the radiator system optimization planned for the second year program phase.

This report is written in three parts. The user of the computer code needs to concern himself only with Part I, titled Operation Manual.

The analysis and the governing equations are contained in Part II, titled Analysis. The numerical techniques are covered in Part III. Details which the reader intending to expand the program may need are placed into the appendices where one can also find the simplified analysis.

## I. OPERATIONAL MANUAL

This chapter familiarizes the user of the computer code developed during the first year program phase, with the deck composition, the data input formats, the running mode options and the output formats.

The program consists of 48 program units and the Data Set. The source language is FORTRAN V. The program was executed, via temporary files, through batch processing on the UNIVAC 1108 computer operating under EXEC 8, Version 26.70.36:C4, at the Georgia Tech RECC Data Processing Center.

Executive Control Statements are not discussed as they may be freely chosen by the user in accordance with the Executive Control Language. All statements are in 026 key punch code.

Changes in both the program structure, the running options and the execution modes are expected to arise during the second year program phase. The program as presently written serves to simulate the radiator system performance. Even though successive runs with system parameter variations are feasible at present, there is no provision for systematic system optimizations.



### A. The Deck Assembly

The program structure is dictated by the calling sequence of the Runge-Kutta-Simpson integration procedure (see Section III-15). The Source Deck consists of two parts. The first part, referred to as the Permanent Part, requires no attention from the user unless he plans major program modifications, for instance to accommodate geometrical configurations other than the one considered here. The Permanent Part contains the first 29 program units.

The second part, called the Selective Part, is assembled from case to case by the user in accordance with his choice of the

- (i) coolant fluid
- (ii) flow channel material
- (iii) meteoroid protection layer material
- (iv) fin panel material
- (v) thermal control coating

There are always 19 program units in the Selective Part.

Following the Source Deck is the Data Set which will be discussed in Chapter I-B and consists of four groups one for each of the following input specifications:

- (i) ascent and reentry profiles
- (ii) solar, albedo and planetary irradiation
- (iii) system specifications
- (iv) program options.

Most of the user's activities will be reflected in his manipulation of the Data Set, especially of the data in groups (iii) and (iv) above.

The Permanent Part contains 29 program units grouped in seven sets from

(a) through (g) as follows:

a) the principal integration program set with

1. MAIN, the main program unit which

- |          |  |
|----------|--|
| provides | α) input and output  |
|          | β) unit conversion   |
|          | γ) computation of design parameters                                  |
|          | δ) preparations for principal integration,                           |
| calls    | α) RKS, DERIVM, CNTLM, for principal integration                     |
|          | β) FLSTRT to establish the initial flow field                        |
|          | γ) FUNCTION subprograms for fluid and structural material properties |
|          | δ) QINCID, SHAPEF (see group c)                                      |
|          | ε) mathematical procedure subprograms,                               |

2. SUBROUTINE DERIVM (Y,DY,TIME) which is called from RKS,

- |          |  |
|----------|--|
| receives | α) the current system state variables Y                              |
|          | β) the current time TIME,  |
| provides | α) all derivatives of the variables Y with respect to time,          |
| calls    | α) CONVEC, QRAD  |
|          | β) FLSTRT to establish the flow field                                |
|          | γ) FUNCTION subprograms for fluid and structural material properties |
|          | δ) mathematical procedure subprograms,                               |

3. SUBROUTINE CNTLM (Y,DY,DX,X,NTRY,IFVD) which

is called from RKS,

receives  $\alpha$ ) all system state variables Y

$\beta$ ) their derivatives DY

$\gamma$ ) current time X, step size DX,

provides  $\alpha$ ) output during integration

$\beta$ ) integration step size control

$\gamma$ ) integration termination criteria,

calls  $\alpha$ ) HFL, fluid enthalpy subprogram

$\beta$ ) mathematical procedure subprograms,

- b) the secondary integration program set, necessary to establish the dynamic fluid flow field, with

4. SUBROUTINE FLSTRT (RE,PR,DELTA) which

is called from MAIN, DERIVM

receives  $\alpha$ ) RE, the mean Reynolds number

$\beta$ ) PR, the mean Prandtl number

$\gamma$ ) DELTA, the diameter-to-length ratio for the tube,

provides  $\alpha$ ) fluid flow (initial and current) conditions

$\beta$ ) table heading for initial conditions,

calls  $\alpha$ ) TRNSPT to compute friction factor and convective film coefficient (in non-dimensional form)

$\beta$ ) RKSF, DERIVL, CNTLN

$\gamma$ ) fluid flow property subprograms

$\delta$ ) mathematical procedure subprograms

5. SUBROUTINE DERIVL (Y,DY,X) which

is called from           RKSF,

receives            a) local fluid flow variables Y

β) position along flow channel axis, X,

provides                    α) spatial derivatives DY of Y,

6. SUBROUTINE CNTLN (Y,DY,DX,X,NTRY,IFVD) which

is called from           RKSF,

receives  $\alpha$ ) the flow variables  $Y$

β) their spatial derivatives DY

$\gamma$ ) position  $X$ , interval  $DX$  along channel axis,

provides            α) output when initial conditions are established

β) termination criteria for RKSF integration,

7. SUBROUTINE TRNSPT (RE,PRL,DELTA,FR,FNU) which computes non-dimensional friction factor and convective film coefficient, Eqs. 3.7 through 9 and Eqs. 3.16 through 18, and

is called from FLSTRT,

receives                      α) Reynolds number RE

$\beta$ ) Prandtl number PRL

$\gamma$ ) diameter-to-length ratio DELTA.

provides  $\alpha) FR = 4f/\delta$ , Eq. 3.32
$$\beta) \text{ FNU} = 4 N_{\text{Nu}} / (\delta N_{\text{Re}} N_{\text{Pr}}), \text{ Eq. 3.37}$$

- c) The program set used to calculate the incident net radiant flux, with,

8. SUBROUTINE QINCID which

is called from MAIN,

- provides
- $\alpha$ ) data transfer from the SRI Incident Radiation Computer program
  - $\beta$ ) averaging of radiant flux over circumference of tube cross-section,

9. SUBROUTINE SHAPEF which

is called from MAIN,

provides exchange function SS as defined by Eq. 6.15, see Appendix D.,

10. SUBROUTINE TRMATX which

is called from QRAD,

provides the evaluation of the transfer matrix  $M_{ij}$ , defined by Eq. 6.7,

11. SUBROUTINE EXITAV which

is called from QRAD,

provides the evaluation of the exitation vector  $P_j$  in Eq. 6.18,

12. SUBROUTINE ABSORB which

is called from QRAD,

- receives
- $\alpha$ ) total hemispherical emittance  $\epsilon$ , see Eq. 6.5
  - $\beta$ ) temperature distributions on the fin and tube surfaces
  - $\gamma$ ) exchange function SS
  - $\delta$ ) functions  $x_{ij}$  and  $x_{ijk}$  defined by Eqs. 6.10 and 6.11,

provides                    absorptance matrix  $\alpha_{ij}$  as defined by Eq. 6.9

calls                       $\alpha$ ) SHAPEF    for SS

$\beta$ ) EMIT        for (T)

$\gamma$ ) AVGEMT    for  $x_{ij}$  and  $x_{ijk}$

$\delta$ ) DEFINT    for integration,

13. SUBROUTINE QRAD which

is called from            DERIVM ,

receives                   $\alpha$ ) transfer matrix and exitation vector  
                              $M_{ij}$  and  $P_j$  , respectively,

provides                   $\alpha$ ) inversion of transfer matrix

$\beta$ ) solution of radiosity equations

$\gamma$ ) grid mapping,

calls                       $\alpha$ ) TRMATX

$\beta$ ) EXITAV

$\gamma$ ) YINT, INTERP,

e) the program set used to calculate the aerodynamic heating during ascent and reentry, consisting of

14. FUNCTION ALTSH (TIME, IOPTN, ALTA, ALTR, TA, TR, NA, NR) which

interpolates user-supplied altitude profiles for a given instant, and

is called from            CONVEC,

receives                    α) current real time, TIME, in seconds,  
                               measured from start of transition phase,  
                               i.e. ascent: TIME = 0, ALTSH = ALTA (1)  
                               reentry: TIME = 0, ALTSH = ALTR (1)

                              β) ordered pairs of time and altitude  
                               (TA,ALTA) and (TR,ALTR) for ascent and  
                               reentry, respectively

                              γ) NA and NR, the number of ordered pairs  
                               and elements in TA,ALTA and TR,ALTR,  
                               respectively,

                              δ) IØPTN = 0 for ascent  
                                       1 for reentry,

provides                    data interpolation,

calls                        YINT for interpolation,

15. FUNCTION VELSH (TIME, IØPTN, VELA, VELR, TA, TR, NA, NR) which  
       interpolates user-supplied velocity profiles for any time  
       during shuttle service, and

is called from            CØNVEC,

receives                    the same data as program unit No. 14  
                               except that ALTA and ALTR are replaced by  
                               the velocities VELA and VELR for ascent  
                               and reentry, respectively,

provides                    data interpolation,

16. SUBROUTINE ATMØS (ELEV, TPE, CP, TATM, CATM) which computes the  
       atmospheric temperature and the speed of sound as functions of  
       altitude and

is called from            CØNVEC,

receives                    α) current shuttle elevation, ELEV, in feet

                              β) ordered pairs of atmospheric temperature  
                               (TPE) and specific heat at constant prpressure  
                               (CP),

provides                   atmospheric temperature, TATM in degrees  
R and speed of sound, CATM, in ft/sec,

calls                      YINT for interpolation, PØLY for evaluation  
of polynomials

17. SUBROUTINE REFP (ELEV, TPE, CP, REFTP, REFPR, REFVIS, REFRHØ, REFK,

REFCP, REFGAM) which computes the atmospheric air properties  
as a function of elevation and of high speed reference  
temperature and

is called from           CØNVEC,

receives                α) current orbiter elevation, ELEV, in feet

                         β) ordered pairs of atmospheric temperature  
                          TPE in degrees R and specific heat at  
                          constant pressure CP in Btu/(lbm R)

                         γ) current high speed reference temperature  
                          REFTP, in degrees R,

provides                atmospheric Prandtl number REFPR  
                          (dimensionless), dynamic viscosity REFVIS<sub>3</sub>  
                          in lbm/(ft sec), density REFRHØ in lbm/ft<sup>3</sup>,  
                          thermal conductivity REFK in Btu/(sec ft R),  
                          specific heat at constant pressure REFCP  
                          in Btu/(lbm/R) and the ratio of specific  
                          heats REFGAM (dimensionless).

calls                    YINT for interpolation

                          PØLY for polynomial expansion,

18. SUBROUTINE NUS (MACHNØ, TATM, CATM, ITN, PRATM, VISATM, RHØATM, STAGX,

VERTX, NSRAD, NUS1) which provides a single Nusselt number for the cal-  
culation of the aerodynamic heating of the orbiter radiator  
system at the current time and

is called from           CØNVEC



receives

- α) current orbiter Mach number MACH $\emptyset$   
(dimensionless)
- β) current atmospheric temperature TATM, in  
degrees R
- γ) current atmospheric velocity of sound,  
CATM, in ft/sec
- ε) temperature of the coolant fluid at the  
inlet plane TIN, in degrees R
- ζ) atmospheric Prandtl number PRATM,  
evaluated at the high speed reference  
temperature (dimensionless)
- η) atmospheric density RH $\emptyset$ ATM, evaluated  
at the high speed reference temperature  
in lbm/ft
- θ) distance from the stagnation point on the  
shuttle vehicle to the midpoint of the  
radiator panel measured along a streamline,  
STAGX, in feet
- ι) overall dimension of the radiator panel  
measure in a direction parallel to the  
acceleration of gravity, VERTX, in feet
- κ) integer value indicating number of non-  
adiabatic sides of the radiator panel,  
NSRAD i.e. NSRAD = 1 for single non-  
adiabatic surface or NSRAD = 2 for both  
sides of radiator being non-adiabatic,

provides

mean Nusselt number NUS1, for meteoroid layer and fin at current orbiter elevation during either ascent or reentry (dimensionless),

calls

YINT for interpolation,

19. SUBROUTINE CONVEC (TIME) which computes normalized aerodynamic heating of fin and meteoroid protection layer for both ascent and reentry phases the of orbiter operation and is called from DRIVM,

- receives             $\alpha$ ) current real time TIME, in seconds,  
                         measured from start of transition phase;  
                         i.e. TIME must be measured with respect  
                         to same reference point as user supplied  
                         time arrays in FUNCTION ALTSH and VELSH,
- provides            normalized convective flux from fin sur-  
                         face CONFN (I,J), and from meteoroid protection  
                         surface CONMP(I),
- calls                 $\alpha$ ) ALTSH for altitude of orbiter
- $\beta$ ) VELSH for velocity of orbiter
- $\gamma$ ) ATMOS for atmospheric properties
- $\delta$ ) REFP for atmospheric properties evaluated  
                         at high speed reference temperature
- $\epsilon$ ) YINT for interpolation
- $\zeta$ ) NUS for Nusselt number,

f) the program to compute the thickness of the meteoroid protection layer

20. FUNCTION TK (GAMMA,A,BETA,DENSM,THETA,PHI,AN,ALPHA,VELM,PØ, TAU,DENSTY,W,TNN,AMAN,TIN,RØUT) which computes the thickness for given environmental conditions, tube and manifold areas, experimental constants and protection layer properties and is called from        MAIN,

- receives             $\alpha$ ) nondimensional experimental constant,  
                         GAMMA, see Eq. 8.9
- $\beta$ ) nondimensional experimental constant A,  
                         see Eq. 8.9
- $\gamma$ ) nondimensional constant which relates  
                         meteoroid flux to mass, BETA
- $\delta$ ) density of the meteoroid particle, DENSM,  
                         in gm/cm

- ε) nondimensional experimental constant  
THETA, see Eq. 8.9
- ζ) nondimensional experimental constant  
PHI, see Eq. 8.9
- η) nondimensional constant AN, used to describe penetration depth as a function of angle of incidence of meteoroid particle
- θ) velocity of meteoroid particle relative to radiator panel, VELM, in ft/sec
- ι) probability of no damage caused by impact of meteoroid, P $\phi$
- κ) time TAU, the radiator panel is exposed to meteoroid environment, in days
- λ) density of protection material DENST, in lbm/ft
- μ) axial length of single flow channel exposed to meteoroid environment, W, in inches
- ν) integer number of flow channels, TNN,
- ξ) area of the manifold that is exposed, AMAN, in ft
- φ) temperature of coolant fluid at inlet plane of flow channel, TIN, in degrees R
- π) outside radius R $\phi$ UT of the unprotected flow channel, in inches,

provides meteoroid protection thickness TK, in inches,

calls ELAS for modulus of elasticity of protection material,

- g) the program set consisting of all the mathematical procedures discussed in detail in Chapter III:

21. SUBROUTINE RKSF (see Sect. III.15) which  
     is called from       FLSTRT, for dynamic fluid flow field,  
     calls                DERIVL,CNTLN,
22. SUBROUTINE RKS (see Sect. III.15) which  
     is called from       MAIN, for principal integration,  
     calls                DERIVM,CNTLM
23. FUNCTION POLY (X,A,M) (see Sect. III.16),
24. FUNCTION YINT (X,Y,M,N,P) (see Sect. III.17),
25. SUBROUTINE DDX (Y,DY,DX,N)
26. SUBROUTINE D2DX2 (Y,D2Y,DX,N)    } (see Sect. III.18)
27. FUNCTION DEFINT (Y,DX,N)
28. SUBROUTINE FINT (Y,YO,DX,N,F)    } (see Sect. III.19),
29. SUBROUTINE INTERP (NX1,MZ1,XX1,ZZ1,YY1,NX2,MZ2,XX2,ZZ2,YY2) maps a  
     two-dimensional function YY1 from one grid (XX1,ZZ1) onto  
     another grid (XX2,ZZ2) and  
     is called from       QRAD,  
     receives            α) the number of nodal points (NX1,MZ1) of  
                           the original grid  
                           β) the function YY1  
                           γ) the number of nodal points (NX2,MZ2) of  
                           new grid,  
     provides            the function YY2 at the new grid,  
     calls               YINT,  
     is restricted to    NX1,MZ1,NX2,MZ2 ≤ 10

The Selective Part contains five groups of thermophysical properties, with the total of 19 program units. Each group constitutes a property package, one each for

- (i) the coolant fluid (7 program units)
- (ii) the flow channel material (3 program units)
- (iii) the meteoroid protection layer (4 program units)
- (iv) the fin material ( 3 program units)
- (v) the thermal control coating (2 program units)

The choice of a particular material combination must be reflected in the corresponding data specification of the Input Data Set, (see Sect. c of Chap. B).

a) The coolant property subprogram set consists of

- 30. FUNCTION RHOF (P,T) which computes the fluid density  $RHOF$  in slug/ft<sup>3</sup> as a function of pressure  $P$  in lb /ft<sup>2</sup>, and absolute temperature  $T$  in R,
- 31. FUNCTION BETA ( $RHOF$ ,T) which computes the isobaric expansion coefficient  $\beta$  , Sect. II.C.11, in 1/R as a function of density  $RHOF$  in slug/ft<sup>3</sup>, and temperature  $T$  in R,
- 32. FUNCTION CAPPA (P,T) which computes the isothermal compressibility, Eq. 11.3, in ft<sup>2</sup>/lbf, as a function of pressure  $P$  in lbf/ft<sup>2</sup>, and temperature  $T$  in R,
- 33. FUNCTION CPF ( $RHOF$ ,T) which computes the specific heat at constant pressure  $CPF$ , in Btu/(slug R), as a function of density  $RHOF$  in slug/ft<sup>3</sup>, and temperature  $T$ , in R,

34. FUNCTION HFL (RHØ,T) which computes the fluid enthalpy HFL, in Btu/slug, as a function of density RHØ in slug/ft<sup>3</sup>, and temperature T, in R,
35. FUNCTION VISC (RHØ,T) which computes the fluid dynamic viscosity VISC, in slug/(ft sec) as a function of density RHØ in slug/ft<sup>3</sup>, and temperature T, in R,
36. FUNCTION THCF (RHØ,T) which computes the fluid thermal conductivity THCF in Btu/(hr ft R) as a function of density RHØ in slug/ft<sup>3</sup>, and temperature T, in R,

For specific details see Sect. II.11 and 12 and Appedix B.

- b) The coolant channel material property subprogram set consists of three elements, 37 through 39:

37. FUNCTION THCTB (T) computes the thermal conductivity THCTB in Btu/(hr ft R), of the tube wall material as a function of the absolute temperature T, in R,
38. FUNCTION DTHCTB (T) computes the relative change of thermal conductivity k with temperature T, namely  $(dk_w/dT)/k_w$  in 1/R, as a function of temperature T, in R,
39. FUNCTION CPTB (T) computes the specific heat at constant pressure CPTB, in Btu/(slug R) as a function of T, in R.

For specific details of property representation see Appendix A.

- c) The meteoroid protection layer material properties are coded in a set of four elements, 40 through 43:

40. <u>FUNCTION</u> <u>THCMP</u> (T)	}	same purpose for protection layer as for channel wall.  (see 36, 37 and 38 respectively)
41. <u>FUNCTION</u> <u>DTHCMP</u> (T)		
42. <u>FUNCTION</u> <u>CPTB</u> (T)		
43. <u>FUNCTION</u> <u>ELAS</u> (T) computes the modulus of elasticity ELAS in $\text{lb/in}^2$ as a function of temperature T, in R.		

- d) The fin material properties required are coded in three FUNCTION subprograms, numbered 44 through 46:

44. FUNCTION THCFN (T) computes the thermal conductivity THCFN, in  $\text{Btu}/(\text{hr ft R})$  for the fin material as a function of absolute temperature T, in R,

45. FUNCTION DTHCFN (T) computes the relative change of thermal conductivity, namely  $(dk_f/dT)/k_f$  in  $1/R$ , as a function of temperature T, in R,

46. FUNCTION CPFN (T) computes the specific heat at constant pressure CPFN, in  $\text{Btu}/(\text{slug R})$  as a function of temperature T, in R.

- e) Optical properties of the thermal control coating are coded in two subprograms, numbered 47 and 48.

47. FUNCTION EMIT (T) computes the total hemispherical emittance in accordance with Eq. 6.5 as a function of the surface temperature T in R, and is called by ABSORB and EXITAV,

48. SUBROUTINE AVGEMIT (T,XX,XXX,N) evaluates the expressions for  $x_{ij}$  and  $x_{ijk}$  given as Eqs. 6.10 and 11 for N interacting surface elements. It is called by ABSORB (see program No. 12).

This completes the Source Deck discussion. Recall that the Source Deck composition implies a particular selection of coolant fluid and structural materials.



### B. The Input Data Set

Input data are divided in four groups and assembled in the order of increasing frequency of changes so that data records most likely to be changed are at the end of the Data Set. This enables the user to keep large portions of the Job Deck on mass storage. The four groups in the Data Set define

- (i) the ascent and reentry profiles
- (ii) the incident thermal radiant flux as a function of time
- (iii) the system specifications, other than material selections
- (iv) the options of program execution.

Item (ii) may be eliminated during the second contract phase provided a contractor-supplied computer program can be arranged to transfer its results internally (after simultaneous execution).

The program may be executed successively after reading new data records beginning either at the first record of group (i), or at the first record of group (ii) or at any variable in groups (iii) and (iv). When data are reread from group (i) then data in groups (ii) or (iii) or (iv) may be skipped; the same holds true for reentering the Data Set from any other group.

None of the data groups contains a fixed number of records. The number of records in groups (i) and (ii) are determined and specified to the computer by the user. Groups (iii) and (iv) contain a minimum of records as specified below; the number of additional records in this group and of additional sets of groups (i) and (ii) depends on the number of additional executions and parameter variations during the same job. In the description of data records (card images) the numbering starts from one in each group.

a) Ascent and Reentry Data

Card No.	FORMAT	Variable
1	2I10	NA,NR the number of ordered pairs of elements in the velocity-time and altitude-time arrays for ascent and reentry profiles, respectively
2	10F8.3	TA, elements of time array (NA values) in seconds, selected for ascent velocity and elevation profiles
INT*(NA/10+0.9) + 1 = N <sub>1</sub>		
N + 1 through N+INT(NR/10+0.9) = N <sub>2</sub>	10F8.3	TR, elements of time array (NR values) in seconds, selected for reentry velocity and elevation profiles
N +1 through N +INT(NA/8+0.875) = N <sub>3</sub>	8E10.3	VELA , elements of velocity array (NA values) in ft/sec, selected for ascent velocity profile of orbiter
N +1 through N +INT(NR/8+0.875) = N <sub>4</sub>	8E10.3	VELR, elements of velocity array (NR values) in ft/sec, selected for reentry velocity profile of orbiter
N + 1 through N +INT(NA/8+0.875) = N <sub>5</sub>	8E10.3	ALTA, elements of altitude array (NA values) in ft, selected for ascent elevation profile of orbiter
N + 1 through N +INT(NR/8+0.875)	8E10.3	ALTR, elements of altitude array (NR values) in ft, selected for reentry elevation profile of orbiter

---

\*

INT(x) = Integer part of x

b) Irradiation Data

This data set is temporary; it follows the last card of the data described in (a) above.

Card No.	FORMAT	Variable
1	213	NTM, NSRAD (i) number of instances at which incident fluxes are to be specified, $NTM < 100$ (ii) NSRAD = 1 one panel is exposed NSRAD = 2 both panel sides are exposed;
2 through NTM+1	10F8.3	on each card: (i) time TM (in seconds) (ii) 3 values of solar (iii) 3 values of albedo (iv) 3 values of planetary } irradiation } Btu/hr ft ) incident at TM and each coming from one of three directions, in this order: (i) -30° (ii) 0°      from the panel normal (iii) +30°      (upper side)
NTM+2 through 2*NTM+1	10F8.3	same data as above, now for lower fin panel side, only if NSRAD = 2

c) System Specifications

The system specifications define the radiator system and follow immediately after the ascent and reentry profile definitions.

The first four cards in the group of the system specification data contain the names of

- (i) the tube material
- (ii) the fin material
- (iii) the coolant fluid
- (iv) the meteoroid protection layer material

in alphanumeric form, beginning with an alphabetic character. Each name is punched on a separate card, beginning in the first column and extending no further than through the first twelve columns. These names serve to identify the printout of results. The user must supply these names in agreement with the composition of the Selective Part described in Chapter I.A. The subsequent data records contain numerical data.

Numerical system specifications are read into the computer in NAMELIST format, one NAMELIST each with name and input data as follows:

- TUBE for flow channel specifications,
- FLØW for coolant fluid flow specifications,
- FIN for fin parameter specifications,
- PRØTLR for the protection layer specifications,
- MANIFD for the manifold input data.

Each input data record under NAMELIST format has

- (i) a blank in its first column,
- (ii) a \$ - sign in its second column,

(iii) the NAMELIST name, starting in the third column, followed by one or more blanks,

(iv) the string of data in free field format as follows:

$$A = b,$$

where (A) is the variable name and (b) is a number; imbedded blanks between (A), (=), (b), ( , ) and a following (A) are permissible,

(v) the comma of the last datum is to be placed by a \$ - sign which delimits the NAMELIST data string.

For more details concerning NAMELIST formatting, the reader may consult standard texts on FORTRAN PROGRAMMING. The NAMELIST data strings must be sequenced in the order in which they are listed here:

(i) NAMELIST/TUBE/ contains 7 variables:

DITBI	internal (hydraulic) tube diameter in inch,
STBI	tube wall thickness, in inch,
XL	tube length, in ft,
RH $\phi$ TB	tube wall density, in slug/ft <sup>2</sup> ,
MZ	number of nodal points along the tube axis (no. of intervals plus one), $\leq 10$ ,
NRTB	number of nodal points in radial direction , in the tube wall, $\leq 5$ ,
NTBS	number of flow channels.

(ii) NAMelist/Flow/ contains 8 variables:

MDOTI	total coolant mass flow rate, in lbm/hr,
TO	entrance (and reference) temperature in degrees R,
PO	entrance (and reference) pressure in lbf/ft .

(iii) NAMelist/FIN/ contains 5 variables:

NX	number of nodal points on the fin, perpendicular to the channel axis, $NX \leq 10$ ,
SRDOTI	fin root thickness, in inch,
HFNI	fin height, from root to tip, in inch,
STIPI	fin tip thickness, in inch,
RHOFN	fin material density, in slug/ft.

(iv) NAMelist/PROTLR/ contains 13 variables:

NRMP	number of radial nodal points in the protection layer, $\leq 5$ ,
RHOMP	protection layer material density, in slug/ft <sup>3</sup> ,
RHOMET	meteoroid density in g/cm
VELM	meteoroid velocity in ft/sec,
TAU	time that vulnerable area is exposed to meteoroid environment, in days,
PROB	probability of no damage caused by meteoroid impacts, dimensionless,
ALPHA	experimental constant that relates meteoroid flux and mass (See Eq. 8.4) in gm/(day ft <sup>2</sup> ),
BTA	experimental constant that relates meteoroid flux and mass (See Eq. 8.4) dimensionless,
GAMMA	empirical constant used to adjust predicted penetration depths to ones observed experimentally (See Eq. 8.1) dimensionless,
PHI	empirical constant (see Eq. 8.1) dimensionless,

THETA	empirical constant (see Eq. 8.1) dimensionless,
ATK	empirical constant used to account for spalling on a target of finite thickness (see Eq. 8.8) dimensionless,
AN	experimental constant that describes penetration depth as a function of angle of incidence (see Eq. 8.6) dimensionless

(v) NAMelist/MANIFD/ contains

AMAN	total mainfold area, projected into the fin plane, in $\text{ft}^2$ .
------	---

Any one of these variables may be changed for successive program executions by entering them in NAMelist/GINPT/ as discussed below.

d) Execution Control Options

The user has the choice of options for the execution of the first as well as successive computer runs. These options are likely to increase in the second contract phase.

Two data records are considered to be execution controls: RUNØPF and GINPT.

(vi) NAMelist/RUNØPT/ contains 8 variables:

MSTØTR	= 1	to compute steady state conditions
	= 2	to simulate transient system performance
DTWRTE		fixed time interval, in hr, between data printout during integration
TEND		termination time, in hr, for transient performance calculation
ALIMIT		absolute error limit per integration step, see Eq. 15.3

RLIMIT		relative error limit per integration step, see Eq. 15.3
TI		initial temperature, in degrees R
LIMWRT		maximum number of data recording during integration toward steady state, exclusive of initial conditions record and steady state record.
NCØNV	= 0	no aerodynamic heating
	= 1	ascent
	= 2	reentry

(vii) NAMelist/GINPT/ contains every variable name listed in items

(i) through (vi). If this record is omitted, the program will terminate. If this NAMELIST is encountered with any or all variables the execution is restarted with

new system specifications and/or  
 new run option controls and  
 old ascent and reentry profiles and  
 old irradiation profiles.

There may be as many NAMELIST/GINPT/ records as computing time permits.



### C. Computer Output

The first output record consists of a list of all system specifications and execution control data as listed in Section c of Chapter I.B, items (i) through (vi), all in NAMELIST format (see Fig. 1).

Next, there are listed the initial coolant fluid flow properties in nondimensional form, together with their respective reference quantities. Table headings and variable identifications are self-explanatory (see Fig. 2).

The initial line conditions are followed by a list of system parameters which can be computed prior to the integration of the governing differential equations. These parameters (see Fig. 2) include material specifications, important dimensions, weights and the total area.

Following the above preliminary results is a series of current system descriptions printed during the integration process (see Fig. 3). These descriptions include the current time, the fin panel temperature distribution, the local heat rejection rates and the total radiant and aerodynamic heat rejection rates. Furthermore, a second table lists the coolant fluid flow properties as functions of axial distance as well as the local cooling rate. All properties are tabulated in nondimensional form; every table is preceeded by a list of necessary reference quantities.

Specified in the second table, below the reference quantities, are the inlet -, the exit - and the total change in stagnation enthalpy fluxes; they are labelled, respectively, HI, HE and DH. Also, for comparison, there is listed the enthalpy flux at reference state ( $T_o, p_o$ ) and reference velocity  $w_o$ .

The last table of current output data constitutes either the system conditions at the specified termination time, TEND in hr, or the steady-state condition, depending on whether MSTØTR = 1 or MSTØTR = 2, respectively.

The results of two full-scale computer results are compared below with the corresponding results obtained by the simplified analysis which is discussed in Appendix D. The first of the results listed below corresponds to the case for which the complete output records are reproduced as Figures 1 through 3.

Case	$\phi$	$N_c$	$\eta$	$N_{Re}$	Heat Rejection Btu/hr		Rel. Error %
					full scale	simplified	
1	0.05763	0.0134	0.9822	158.7	$1.1679 \times 10^5$	$1.159 \times 10^5$	0.7
2	0.05763	0.0544	0.9337	158.7	$1.1689 \times 10^5$	$1.126 \times 10^5$	3.7

TABLE 1 Comparison of Full-Scale and Simplified Analyses

```

6XUT
$TUBE
D1TB1 = .25000000E+00
STB1 = .10000000E+00
XL = .15000000E+02
RH0TB = .52599999E+01
MZ = +5
NKTB1 = +3
NTBS = +12

```

```

$END
$FLOW
MDOY1 = .60000000E+03
TO = .50000000E+03
PO = .25000000E+04

```

```

$END
$FIN
NX = +5
SR0UT1 = .50000000E-01
HFN1 = .40000000E+01
STIP1 = .50000000E-01
RHOFN = .52599999E+01
STA6X = .20000000E+02
VERTX = .10000000E+02
NSRAD = +2

```

```

$END
$PROFLK
NRMP = +3
RHOMP = .35300000E+01
RHOMET = .25000000E+00
VELM = .65600000E+05
TAU = .36500000E+04
PKOB = .98999999E+00
ALPHA = .18000000E-09
BTA = .12130000E+01
GAMMA = .15000000E+01
PHI = .50000000E+00
THETA = .66666700E+00
ATK = .17500000E+01
AN = .10000000E+01

```

```

$END
$MANIFD
AMAN = .43000000E+02

```

```

$END
$KUNOPT
MSTOTR = +2
DTWFE2 = .99999999E-02
TENL = .33333000E-01
ALIMIT = .50000000E-04
RLIMIT = .10000000E-08
T1 = .40000000E+03
LIMRT = +3
NCOFV = +1
IOPIN = +0

```

```

$END

```

Figure 1. Record of System Specifications  
and Execution Control Data

PT.NO.	POSITION Z	PRESSURE P	VELOCITY W	FLUID TEMPERATURE T	WALL TEMPERATURE TWI
1	.000	1.000000	.937067	.800000	.800000
2	.250	.937382	.937095	.800002	.800000
3	.500	.974767	.937122	.800004	.800000
4	.750	.962144	.937149	.800006	.800000
5	1.000	.949525	.937176	.800007	.800000

INLET PRESSURE P0 = 2500.000 LBF/SQ.FT  
 REF. VELOCITY W0 = .73055 FT/SEC  
 REF. TEMPERATURE T00 = 500.000 R

REYNOLDS NO = .15829+03  
 PRANDTL NO = 94.083629  
 DELTA = .001389  
 REL.PRESSURE IS .135116+04

INIT. NUSSLEY NO. NU = .408159+01  
 WALL BIOT NO. BI = .221421-02

# SYSTEM PARAMETERS \*\*\*\*\*

NUMBER OF TUBES , NTBS = 12  
 TUBE LENGTH , XL = 15.000 FT  
 INTERNAL DIAMETER , DITB = .250 IN  
 WALL THICKNESS , STB = .100 IN  
 MATERIAL ALUMINUM  
 MASS (ALL TUBES) , MTB = .7202 SLUG

FIN HEIGHT , HFN = 4.000 IN  
 THICKNESS AT ROOT , SROOT = .050 IN  
 THICKNESS AT TIP , STIP = .050 IN  
 MATERIAL ALUMINUM  
 MASS (ALL FIN) , MFN = 2.4200 SLUG

COOLANT FLUID IS SILICONE OIL  
 MASS (IN ALL TUBES) , MFL = 1.3620 SLUG

PROTECTION LAYER THICKNESS, SMP = .063 IN  
 MASS, MMP = .349 SLUG  
 MATERIAL IS BERYLLIUM

TOTAL MASS (EXCL. MANIFLD.) MTOT = 5.0517 SLUG  
 TOTAL AREA (SINGLE NORMAL PROJECTION) , ATOT = 126.7500 SQ FT

Figure 2. Initial Coolant Flow Conditions  
and System Design Parameters

RELATIVE TIME IS 120.8109

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 500.000 R  
 REF. RADIANT HEAT FLUX  
 PER UNIT AXIAL LENGTH, QREF = .8570+03 BTU/(HR FT)  
 TOT. RADIANT REJECTION: QTOT = .1294+05 BTU/HR

AERODYN. HEATING POWER, QCONV = .0000 BTU/HR

AXIAL DIST. Z	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	1.6029	.9226	.9226	.9226	.9226	.9226
.250	1.0167	.8360	.8267	.8201	.8162	.8147
.500	.9433	.8200	.8112	.8050	.8013	.8001
.750	.8618	.8008	.7929	.7873	.7839	.7829
1.000 (EXIT)	1.0769	.8353	.8353	.8353	.8353	.8353

## FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 500.000 R  
 REFERENCE PRESSURE, P00 = 2500.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .731 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .867533+04 BTU/HR  
 EXIT CURRENTLY EI = -.301338+04 BTU/HR

INLET CURRENTLY HI = .867551+04 BTU/HR  
 TOT. REJECTION DH = .116889+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMP TF	PROTECT. LAYER TEMPERATURES		ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
				TWI	TMP		
.000	1.00000	1.0000	1.0000	.9226	.9226	42.723417	.00000
.250	.98656	.9901	.9660	.8360	.8360	71.745527	.25084
.500	.97325	.9813	.9423	.8200	.8200	67.445761	.54166
.750	.96004	.9733	.9112	.8008	.8008	60.915485	.79972
1.000 (EXIT)	.94692	.9666	.8322	.8353	.8353	31.430184	1.00000

Figure 3. Typical Record of Current System Variables

## II ANALYSIS

The analysis is presented in three major parts. In the first part are developed the governing equations of transient heat transfer within and from the radiator system; these equations are the basis of the numerical simulation. The second part of the analysis is devoted to the computation of design parameters dictated by operational conditions, while in the third part there is covered the development of thermodynamic and transport properties of structural materials, coolant fluids, and the atmosphere.

## NOMENCLATURE

$a$	empirical constant used to account for spalling of protection layer (Eq. 8.8)	
$a$	function of temperature, used to express the isothermal compressibility and the zero-pressure isobaric thermal expansion coefficient, see also $b$ and $c$ (Eq. 11.9)	
$a_z$	axial fluid flow acceleration	ft/sec
$A$	area	ft <sup>2</sup>
$A_n$	coefficients in polynomial for atmospheric pressure at high altitudes (Eq. 13.8)	-
$A_s$	surface area	ft <sup>2</sup>
$A_x$	cross-sectional area perpendicular to $x$ coordinate	ft <sup>2</sup>
$A_z$	cross-sectional area perpendicular to $z$ coordinate	ft <sup>2</sup>
$b$	defined in Eq. 11.9, see also $a$	
$B$	fin geometrical parameter (Eq. 2.14)	-
$B_N$	coefficients in polynomial for atmospheric density at high altitudes (Eq. 13.9)	-
$c$	atmospheric speed of sound	ft/sec
$c$	negative slope of fin sides	-
$c$	defined in Eq. 11.9, see also $a$	
$c$	speed of light	ft/sec
$\bar{c}$	speed of sound in meteoroid protection material	ft/sec
$c_p$	specific heat at constant pressure	Btu/(slug R)
$c_v$	specific heat at constant volume	Btu/(slug R)

$c_v$	zero-pressure specific heat at constant volume	Btu/(slug R)
$d$	diameter of meteoroid particle	in
$d$	tube diameter	ft
$D$	fin geometrical parameter (Eq. 2.15)	-
$E$	emissive power	Btu/(hr ft <sup>2</sup> )
$E$	modulus of elasticity	lbf/in <sup>2</sup>
$E$	relative error	-
$f$	Fanning friction factor	-
$F$	cumulative meteoroid flux (eq. 8.4)	1/(ft <sup>2</sup> day)
$F$	dimensionless parameter, Eqs. 3.31-34	
$g_c$	32.174 ft lbf/(lbf sec <sup>2</sup> )	
$h$	Plank's constant, $h = 6.625 \times 10^{-34}$ W s <sup>2</sup>	
$h$	specific enthalpy	Btu/slug
$H$	geopotential altitude	ft
$H$	fin height, from root to tip	ft
$h_c$	convective film coefficient	Btu/(hr ft <sup>2</sup> R)
$h_i$	convective heat transfer coefficient used in reference enthalpy method	lbf/(hr ft <sup>2</sup> )
$k$	Boltzmann constant, $k = 1.380 \times 10^{-23}$ W s/K	
$k$	thermal conductivity	Btu/(hr ft R)
$L$	temperature gradient in atmosphere (Table 5)	(K/km)
$L$	tube (and fin) length	ft
$M$	Mach number of orbiter	-
$M$	mass	slug
$M$	molecular weight	-
$M_{ij}$	transfer matrix Eq. 6.19	



$M_m$	inverted M	-
M	defined by Eq. 10.22	
n	constant that describes depth of penetration as a function of angle of incidence (Eq. 8.6)	-
n	number of tubes	
n	refractive index	-
N	cumulative number of meteoroid impacts	-
$N_{Bi}$	Biot number	-
$N_{Fo}$	Fourier number	-
$N_{Gr}$	Grashof number	-
$N_{Nc}$	Conductance parameter	-
$N_{Nu}$	Nusselt number	-
$N_{Re}$	Reynolds number	-
$N_{Pr}$	Prandtl number	-
p	absolute pressure	lbf/ft <sup>2</sup>
$P_j$	nondimensional exitation vector, Eq. 6.21	-
$P_j$	exitation vector, Eq. 6.18	
$P_o$	probability of no damage due to meteoroid impact	-
$P_\infty$	depth of penetration of a meteoroid particle into an infinite target	in
$q''$	heat flux	Btu/(hr ft <sup>2</sup> )
$\tilde{q}$	nondimensional heat flux	-
$\bar{Q}_o$	nondimensional inlet power flux	-
$\bar{Q}_{rad}$	defined by Eq. 10.21	-
r, z	polar coordinates, see Fig. 4	ft
r	recovery factor	-

$R^*$	universal gas constant	ft lbf/(lb mole R)
$R$	outer meteoroid protection layer radius	ft
$s$	thickness	ft
$s_f = \frac{s_r}{2}$	fin half thickness at root	ft
$\bar{s}_f$	normalized fin thickness (Eq. 2.13)	-
$\overline{s_i s_j}$	direct exchange area (Eq. 6.1)	ft <sup>2</sup>
$S$	constant in atmospheric viscosity equation (Eq. 3.7)	K
$SS$	direct exchange "area", partially evaluated	ft
$t$	time	hr
$T$	absolute temperature	R
$u$	specific interval energy	Btu/slug
$V$	velocity of orbiter	ft/sec
$\bar{V}$	velocity of meteoroid relative to radiator	ft/sec
$w$	coolant flow velocity	ft/sec
$w_j$	nondimensional radiosity, Eq. 6.21	-
$W_j$	radiosity	Btu/(hr ft <sup>2</sup> )
$x$	distance from orbiter stagnation point to radiator panel	ft
$x, z$	rect. Cart. coordinates, see Fig. 4	ft
$x_{ij}, x_{ijk}$	auxiliary parameters, defined by Eqs. 6.10 and 6.11	-
$y$	overall length of radiator in direction parallel to gravity	ft
$y$	transverse fin coordinate	ft
$z$	axial distance	ft
$Z$	geometric altitude	ft

## Greek Symbols

$\alpha$	experimental meteoroid flux parameter, (Eq. 8.4)	$1/(\text{ft}^2 \text{day gm}^\beta)$
$\alpha$	thermal diffusivity	$\text{ft}^2/\text{hr}$
$\alpha_{ij}$	total hemispherical absorptance	-
$\beta$	constant in atmospheric viscosity equation, (Eq. 3.7)	$\text{kg}/(\text{sec mK}^{1/2})$
$\beta$	experimental meteoroid flux parameter, (Eq. 8.4)	-
$\beta$	isobaric thermal expansion coefficient, see Eq. 3.5	$1/R$
$\gamma$	ratio of specific heats	-
$\delta$	diameter-over-length ratio	-
$\delta_{ij}$	Kronecker delta	-
$\epsilon$	total hemisphere emittance	-
$\zeta$	dimensionless axial distance, Eq. 3.19	-
$\eta$	dimensionless radius, Eq. 3.25	-
$\theta$	dimensionless temperature, Eq. 3.21	-
$\theta$	empirical constant (Eq. 8.1)	-
$\kappa$	isothermal compressibility, Eq. 3.4	$\frac{2}{\text{ft}^2/\text{lbf}}$
$\lambda$	angle between path of meteoroid and normal to protected surface (Eq. 8.6)	deg.
$\lambda$	wave length	ft
$\lambda_1$	parameter, defined by Eq. 4.11	-
$\lambda_2$	parameter, defined by Eq. 4.12	-
$\mu$	dynamic viscosity	$\text{slug}/(\text{ft sec})$
$\nu$	dimensionless density	-
$\nu$	kinematic viscosity	$\text{ft}^2/\text{sec}$
$\xi$	dimensionless x coordinate	-

$\pi$	dimensionless pressure, Eq. 3.22	
$\rho$	density	slug/ft <sup>3</sup>
$\sigma$	Stefan-Boltzmann constant	Btu/(hr ft <sup>2</sup> R <sup>4</sup> )
$\tau$	dimensionless time, Eq. 3.20	-
$\tau$	time radiator is exposed to meteoroid environment	day
$\phi$	angle between surface normal and radiation beam	-
$\phi$	dimensionless property Eqs. 3.27-3.30, Eq. 4.4	-
$\phi, \phi^*$	polar angle in Eq. 6.15	-
$\phi_{Nc}$	modified conduction parameter, Eq. 5.6	
$\phi$	empirical constant (Eq. 8.1)	-
$\phi$	dimensionless parameter, Eq. D. 13	-
$\chi$	quadrature coefficient	-
$\psi$	circumferential fraction, defined by Eq. 4.3	-
$\psi$	residual, see Eqs. 12.1 and 12.2	-
$\omega$	dimensionless velocity Eq. 3.23	-

### Subscripts

aw	adiabatic wall
aero	aerodynamic
b	values at the endpoints of straight line segments of atmospheric temperature profile, (Table 5)
c	coolant fluid
c	critical value
c	enclosure
c <sub>p</sub>	referring to specific heat at constant pressure

e	outer surface of meteoroid protection layer, (environment)
f	referring to friction factor
f	fin
F	Fourier number
i, j	position index
m	meteoroid protection layer
M	Mach number
net,rad	net radiant
p	meteoroid particle
p	referring to pressure
r	fin root
t	fin tip
t	target material
t	tubes
w	channel wall
w	surface or wall condition
z	axial distance
z	thermal equation of state
o	entrance (reference) conditions
o	sea level values
1	upper fin side
2	lower fin side
$\infty$	free stream condition
$\zeta$	partial differentiation with resp. to $\zeta$
$\eta$	partial differentiation with resp. to $\eta$
$\kappa$	referring to compressibility

$\lambda$  monochromatic

$\tau$  partial differentiation with respect to  $\tau$

### Superscripts

\* evaluated at reference temperature  $T^*$

## A. HEAT TRANSFER

### 1. Introduction

The objective of the radiator simulation is to predict both the transient and the steady state heat transfer characteristics under pre-determined operating conditions, stationary in the former case and dynamic in the latter. Both cases were treated as initial value problems, and the principal governing equations are partial differential equations which are linear in the time-derivatives of first order at most.

Under stationary boundary conditions steady state will be reached, regardless of initial conditions\*, as all partial derivatives with respect to time vanish on account of dissipative effects within the system. For the computer simulation of steady state conditions this means that the process of advancing in time can be discontinued as soon as all variables  $y_i$ ,  $i = 1, 2, \dots, N$ , have reached their expected asymptotic values  $(y_i)_\infty$  with sufficient accuracy, that is when for some chosen  $\epsilon_i$

$$\delta_i = |y_i - (y_i)_\infty| \leq \epsilon_i \quad (1.1)$$

has been reached. The expected asymptotic values  $(y_i)_\infty$  can be estimated on the basis of the recognition that for large enough values of the time,  $t > t_M$

$$y_i \rightarrow (y_i)_\infty \pm ae^{-bt} \quad (1.2)$$

with,  $a > 0$ ,  $b > 0$ , to be determined;  $t > t_M$ .

---

\*

Subject to certain continuity requirements which are discussed in Chapter III.

The evaluation  $\delta_i$  during the numerical integration is covered in Chapter III.

The radiator system consists of four components:

- (i) the fin
- (ii) the coolant fluid
- (iii) the coolant flow channel
- (iv) the meteoroid protection layer.

Two coordinate systems were introduced (see Figure 4 ), one for the fin and the other for the flow channel and the meteoroid protection layer.

The rectangular Cartesian system  $(x,z)$  for the fin has its  $z$ -axis parallel to the tube axis, starting at the inlet plane, and its  $x$ -axis passing through the line of profile symmetry, with  $x = 0$  designating the root of the fin. Cylindrical coordinates  $(r,z)$  are used for both the tube and the meteoroid protection layer, with the  $z$ -axis along the tube.

The radiator system is describable in terms of the following six dependent state variables

- (i) for the fin:

the fin temperature  $T(t;x,z)$

- (ii) for the coolant fluid:

the fluid temperature  $T(t;z)$

the fluid pressure  $p(t;z)$

the fluid velocity  $w(t;z)$

- (iii) for the fluid flow channel:

the tube wall temperature  $T(t;r,z)$

- (iv) for the meteoroid protection layer

the protection layer temperature  $T(t;r,z)$



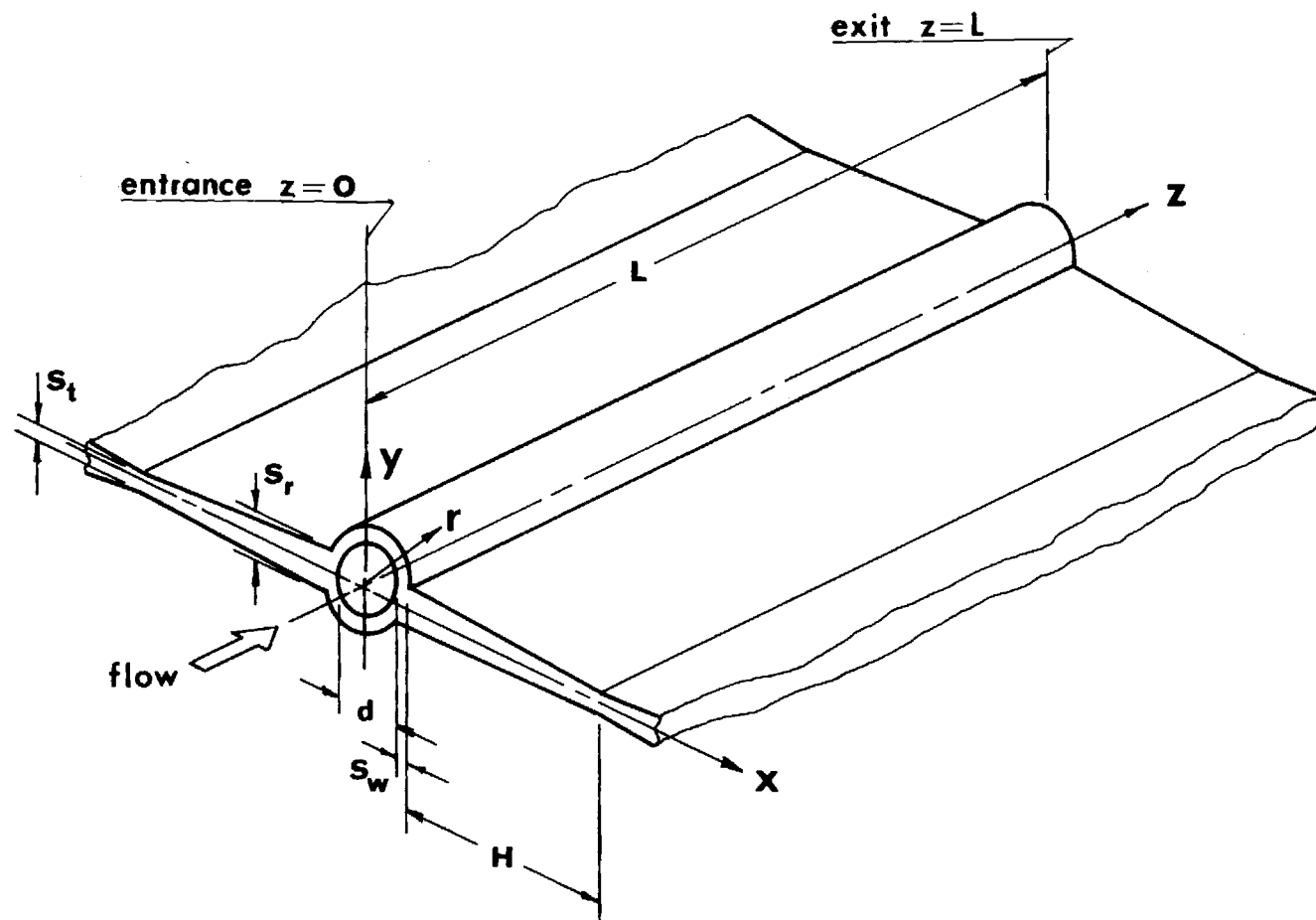


Figure 4. Fin Element and Coordinate System

These six dependent variables must satisfy four equations of energy conservation, one for each component of the system, further the equations of mass conservation and of momentum balance for the fluid. These conservation equations take on the form of partial differential equations subject to initial and boundary conditions. Finally, the energy conservation equation for the fin involves the net radiant and convective power fluxes leaving the fin.

In the following Sections, III A.2 through III A.7 are discussed, in that order, the six principal governing equations associated with the four system components listed above, the subsidiary equations governing the radiative heat exchange and, lastly, the convective heat transfer between coolant fluid and tube wall and between the fin and the atmosphere during ascent and reentry.

## 2. The Fin

The objective of this section is to develop the energy equation for the fin. The derivation is based on the assumptions that the thermal conductivity and specific heat of the fin material are functions of temperature while the fin density is constant. The energy balance for the fin accounts for both radiative and convective fluxes from the fin surfaces. The development of a method to predict the net radiant flux from the fin surfaces can be found in Section II.6. The procedure used for the evaluation of the convective flux from the fin surface is outlined in Section II.7.

The energy balance on a differential volume of the fin can be expressed as

$$\frac{\partial}{\partial x} \left[ k_f A_x \frac{\partial T_f}{\partial x} \right] dx + \frac{\partial}{\partial z} \left[ k_f A_z \frac{\partial T_f}{\partial z} \right] dz + q''_{\text{net,rad}} A_s = \rho_f V c_{pf} \frac{\partial T_f}{\partial t} \quad (2.1)$$

where  $T$  is the fin temperature and the coordinate system is shown in Fig. 4. The areas  $A_x$  and  $A_z$  represent the cross-sectional areas of the fin perpendicular to the heat flow in the  $x$  and  $z$  directions, and  $A_s$  represents the total non-adiabatic surface area of the fin element.

The symbol  $V$  is the volume of the fin element. The properties of the fin material are represented by the symbols  $k_f$ ,  $\rho_f$  and  $c_{pf}$  which stand for the thermal conductivity, density and specific heat, respectively.

The terms  $q''_{\text{net,rad}}$  and  $q''_{\text{aero}}$  appearing in Eq. 2.1 denote the radiative and convective heat gain from the surroundings to the fin surfaces.

The first two terms in Eq. 2.1 constitute the net conduction of energy into the fin element. The third and fourth terms on the left hand side of the equation stand for the net radiation and convection gain, respectively, from the surfaces of the fin element. The term of the right hand side of the equation represents the storage of internal energy within the fin element.

The appropriate areas and the volume to be substituted into Eq. 2.1 are

$$A_x = 2[s_f - cx] dz \quad (2.2)$$

$$A_z = 2[s_f - cx] dx \quad (2.3)$$

$$A_s = dx dz / [c^2 + 1]^{1/2} \quad (2.4)$$

$$V = 2[s_f - cx] dx dz \quad (2.5)$$

where  $s_f$  is the fin half thickness at its root and  $c$  is the negative slope of the fin side surfaces.

Substituting Eqs. 2.2 through 2.5 into Eq. 2.1 and simplifying the resulting equation yields

$$\rho_f c_{pf} \frac{\partial T_f}{\partial t} = k_f \left[ \frac{\partial^2 T_f}{\partial x^2} + \frac{\partial^2 T_f}{\partial z^2} \right] + \frac{dk_f}{dT} \left[ \left( \frac{\partial T_f}{\partial x} \right)^2 + \left( \frac{\partial T_f}{\partial z} \right)^2 \right] \quad (2.6)$$

$$- \left( \frac{ck_f}{t-cx} \right) \frac{\partial T}{\partial x} + \frac{\sqrt{c^2+1}}{2(t-cx)} [q''_{\text{net,rad}} + q''_{\text{aero}}]$$

The Normalization of Eq. 2.6 is achieved by defining nine dimensionless quantities:

Let

$$\xi = x/H \quad (2.7)$$

$$\zeta = z/L \quad (2.8)$$

represent the nondimensional fin coordinate perpendicular and parallel to the tube, respectively. Then

$$\tau = tw_o/L \quad (2.9)$$

is the dimensionless time. The symbol  $w_o$  stands for the velocity of the coolant fluid entering the tube.

The nondimensional fin temperature is defined as

$$\theta_f(\tau, \xi, \zeta) = \frac{T_f(\tau, \xi, \zeta)}{T_o} \quad (2.10)$$

where  $T_o$  is the temperature of the coolant fluid entering the tube.

The dimensionless conduction parameter  $N_{Nc}$  and the Fourier number are defined as

$$N_{Nc} = H\sigma T_o^3/k_f \quad (2.11)$$

$$N_{Fo} = \frac{\alpha_f L}{w_o H^2} = \frac{k_f L}{\rho_f c_{pf} w_o H^2} \quad (2.12)$$

and the nondimensional geometrical quantities are defined as

$$\bar{s}_f = s_f/H \quad (2.13)$$

$$B(\xi) = 2(\bar{s}_f - c\xi) \quad (2.14)$$

$$D = (c^2 + 1)^{1/2} \quad (2.15)$$

where  $c$  is the negative slope of the fin side surfaces. For a non-tapered fin  $c$  is zero.

Both the net radiative and convective flux terms appearing in Eq. 2.6 are normalized by dividing each term by  $\sigma T_o^4$ , or

$$\tilde{q}_{net,rad} = \frac{q''_{net,rad}}{\sigma T_o^4} \quad (2.16)$$

$$\tilde{q}_{aero} = \frac{q''_{aero}}{\sigma T_o^4} \quad (2.17)$$

The energy equation, Eq. 2.6, may now be written in terms of the non-dimensional quantities given in Eqs. 2.7 through 2.17. The resulting nondimensional equation is

$$\begin{aligned} \dot{\theta}_f = N_{Fo} \left\{ (\theta_f)_{\xi\xi} + \left(\frac{H}{L}\right)^2 (\theta_f)_{\zeta\zeta} + \frac{T_o}{k_f} \frac{dk_f}{dt} \left[ (\theta_f)_{\xi}^2 + \left(\frac{H}{L}\right)^2 (\theta_f)_{\zeta}^2 \right] \right. \\ \left. - 2 \frac{c}{B} (\theta_f)_{\xi} + \frac{D}{B} N_{Nc} [\tilde{q}_{\text{net,rad}} + \tilde{q}_{\text{net,conv}}] \right\} \end{aligned} \quad (2.18)$$

The dot superscript appearing in Eq. 2.18 denotes differentiation with respect to nondimensional time and the subscripts  $\xi$  and  $\zeta$  represent partial differentiation with respect to the dimensionless coordinates indicated.

The normalized energy equation, Eq. 2.18, defines the rate of change of the dimensionless fin temperature  $\theta_f$ .

The Boundary Conditions for Eq. 2.18 are taken as follows:

- (i) The fin root is at the temperature of the outside of the tube.
- (ii) The fin tip is insulated.
- (iii) The portion of the fin in contact with the inlet manifold is at the outside temperature of the manifold.
- (iv) The portion of the fin in contact with the exit manifold is at the temperature of the outlet manifold.

Written in mathematical terms these four boundary conditions are

$$\theta_f(\tau, 0, \zeta) = \theta_w(\tau, \xi_o, \zeta) \quad (2.19)$$

$$\frac{\partial \theta_f}{\partial \xi} (\tau, 1, \zeta) = 0 \quad (2.20)$$

$$\theta_f(\tau, \xi, 0) = \theta_w(\tau, \xi_o, 0) \quad (2.21)$$

$$\theta_f(\tau, \xi, 1) = \theta_w(\tau, \xi_o, 1) \quad (2.22)$$

Implied in Eqs. 2.21 and 22 is, firstly, that the fluid temperature in the manifolds be equal to the fluid temperature at the tube inlet (inlet manifold) and at the tube exit (exit manifold) and that it remain unaltered along the manifold and vary only in time; secondly, that the temperature drop through the manifold wall be equal to that through the tube wall. The latter assumption is well justified because the temperature drop is exceedingly small.

It may be noted in connection with the boundary conditions that the radiative interactions action between manifold and fin as well as between manifold and tube are not taken into consideration at this time.

The Initial Condition for Eq. 2.18 may be any arbitrary relation representing a continuous temperature distribution over the fin, including the boundaries. The selection of the initial fin temperature distribution is left to the user.



### 3. The Coolant Fluid

The objective is to develop a unified treatment of all possible coolant fluids, that is gases, dielectric fluids and liquid metals. Three principal governing equations are sought which, together with the necessary thermodynamic and transport properties specified for each fluid of interest, define the fluid temperature  $T_c$ , the fluid pressure  $p$  (and thus the thermodynamic state of the fluid) and the fluid velocity  $w$ , all as functions of time  $t$  and axial distance  $z$ .

The Continuity Equation for one dimensional flow through channels of constant cross-sections reads

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial z} (\rho w) = 0 \quad (3.1)$$

where  $\rho$  represents the fluid density. Replacing the density through the thermal equation of state

$$p = p(\rho, T_c) \quad (3.2)$$

renders the continuity equation in terms of derivatives of the primary variables  $T_c$ ,  $p$  and  $w$ :

$$\kappa \frac{\partial p}{\partial t} - \beta \frac{\partial T_c}{\partial t} = - \frac{\partial w}{\partial z} + w \left( \beta \frac{\partial T_c}{\partial z} - \kappa \frac{\partial p}{\partial z} \right) \quad (3.3)$$

where  $\kappa$  and  $\beta$  stand for the isothermal compressibility and the isobaric expansion coefficient, respectively

$$\kappa = \frac{1}{\rho} \left( \frac{\partial \rho}{\partial p} \right)_{T_c} \quad (3.4)$$

$$\beta = - \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T_c} \right)_p \quad (3.5)$$

### The Momentum Equation

$$\rho \frac{\partial w}{\partial t} + w \frac{\partial w}{\partial z} = - \frac{\partial p}{\partial z} - \frac{4f}{d} \rho \frac{w^2}{2} + \rho a_z \quad (3.6)$$

constitutes the balance between inertia forces on the left-hand side, pressure forces wall friction and external field forces (gravity) in axial direction, on the right-hand side of Eq. 3.6. The Fanning friction factor  $f$  is a function of the Reynolds-number  $N_{Re} = d w / \nu$ , where  $d$  and  $\nu$  stand for the tube diameter and the kinematic viscosity, respectively and subscript  $o$  designates the fluid inlet conditions. The following relations are used to compute the Fanning friction factor:

for  $N_{Re} < 2300$  (laminar flow

$$4f = \frac{64}{N_{Re}} \quad (3.7)$$

for  $2300 \leq N_{Re} \leq 10^6$  (Ref. 1)

$$4f = 0.0054 + 0.396 N_{Re}^{-0.3} \quad (3.8)$$

for  $N_{Re} > 10^6$  (Ref. 2)

$$4f = 0.0032 + 0.221 N_{Re}^{-0.237} \quad (3.9)$$

Equations 3.8 and 9 could be replaced by a single equation (Ref. 3)

$$4f = \left[ 0.86859 \ln \frac{N_{Re}}{1.964 \ln N_{Re}} - 3.8215 \right]^{-2} \quad (3.10)$$

which, however, requires more computational effort.

The Energy Equation. Let  $u$ ,  $h$ ,  $\bar{h}_c$  and  $T_w = T_w(t; r_i, k)$  represent, respectively the internal energy and the enthalpy of the fluid, the convective film coefficient and the tube wall temperature at the fluid-wall interface. For one-dimensional flow through channels of constant cross-section and with heating or cooling from the wall, conservation of energy requires that, with respect to a stationary reference frame

$$\begin{aligned} \frac{\partial}{\partial t} \left[ \rho \left( u + \frac{w^2}{2} \right) \right] + \frac{\partial}{\partial z} \left[ \rho w \left( h + \frac{w^2}{2} + a_z z \right) \right] &= \frac{4}{d} \bar{h}_c (T_w - T_c) \\ + \frac{\partial}{\partial z} \left( k_c \frac{\partial T}{\partial z} \right) \end{aligned} \quad (3.11)$$

The first term constitutes the storage of thermal and kinetic energies; the storage of potential energy, being negligibly small for expected accelerations, is ignored. The second term on the left-hand side stands for the convection of thermal, kinetic and potential energies as well as the power associated with the pressure. The right-hand side contains, firstly, the convective heat transfer from the channel wall to the fluid and, secondly, the axial heat conduction term which will later be shown to be negligibly small for all fluids. The factor  $4/d$  in front of the convective heat transfer term results from the ratio of the channel circumference to the cross-sectional area, evaluated for a circular tube. The symbol  $k_c$  represents the thermal conductivity of the coolant.

Given a caloric equation of state

$$u = u(\rho, T_c) \quad (3.12)$$

or an equivalent expression for the specific heat,  $c_p$ , one can write, with the aid of Eqs. 3.2, 4 and 5

$$dh = c_p dT_c + \frac{1}{\rho} (1 - \beta T_c) dp \quad (3.13)$$

$$du = (c_p - \frac{p\beta}{\rho}) dT_c + \frac{1}{\rho} (\kappa p - \beta T) dp \quad (3.14)$$

and then recast Eq. 3.11 in terms of the derivatives of the principal variables  $T_c$ ,  $p$  and  $w$ :

$$\begin{aligned} \rho c_p \frac{\partial T_c}{\partial t} - \beta T_c \frac{\partial p}{\partial t} + \rho w \frac{\partial w}{\partial t} &= \frac{4}{d} \bar{h}_c (T_w - T_c) \\ &+ \frac{\partial}{\partial z} (k_c \frac{\partial T_c}{\partial z}) - \rho w [c_p \frac{\partial T_c}{\partial z} + \\ &\frac{1}{\rho} (1 - \beta T) \frac{\partial p}{\partial z} + w \frac{\partial w}{\partial z} + a_z] \end{aligned} \quad (3.15)$$

Thus, three governing equations, Eqs. 3.3, 6 and 15, have been established which define the three principal variables  $T_c$ ,  $p$  and  $w$ , provide the initial and boundary conditions are properly specified and the convective film coefficient can be predicted. Since the normalization of these equations, followed by an order-of-magnitude comparison will indicate that the conductive

term in Eq. 3.15 is negligible so as to simplify the boundary conditions, the discussion of the initial and boundary conditions is deferred until after the normalization.

The convective film coefficient is computed from the following relationships between the Nusselt number  $N_{Nu} = \bar{h}_c d/k_c$ , the Reynolds number  $N_{Re}$  and the Prandtl number  $N_{Pr} = \mu c_p/k_c$  where  $\mu$  represents the dynamic viscosity:

For  $N_{Pr} < 0.1$  (liquid metals)

$$N_{Nu} = 6.5 + 0.025 (N_{Re} N_{Pr})^{0.8} \quad (3.16)$$

which produces Nusselt numbers between those appropriate for uniform heat flux (Martinelli) and for uniform wall temperature (Seban and Shimazaki) (Ref. 4).

For  $N_{Pr} > 0.1$  and

for  $N_{Re} < 2300$  (laminar flow, Ref. 5)

$$N_{Nu} = 3.65 + \frac{0.0668 (N_{Re} N_{Pr} \delta)}{1 + 0.045 (N_{Re} N_{Pr} \delta)} \quad (3.17)$$

for  $N_{Re} \geq 2300$  (turbulent flow, Ref. 5)

$$N_{Nu} = 0.116 (N_{Re}^{0.667} - 125) N_{Pr}^{0.333} (1 + \delta) \quad (3.18)$$

where  $\delta = d/L$  stands for the tube diameter-over-length ratio.

The Normalization of Eqs. 3.3, 6 and 15 is carried out for the purpose of scaling and performing an order-of-magnitude comparison. The computational effort is also reduced in the process.

Let

$$\zeta = z/L \quad (3.19)$$

$$\tau = tw_o/L \quad (3.20)$$

represent the nondimensional axial distance and the nondimensional time, respectively, and let the dimensionless state variables be defined as

$$\theta_c(\tau, \zeta) = \frac{T_c(t, z)}{T_o} \quad (3.21)$$

$$\pi(\tau, \zeta) = \frac{p(t, z)}{p_o} \quad (3.22)$$

$$\omega(\tau, \zeta) = \frac{w(t, z)}{w_o} \quad (3.23)$$

and to represent the nondimensional temperature, pressure and velocity of the fluid, that is, the principal dependent fluid flow variables. The subscript "o" designates the constant reference state of the fluid at the tube entrance. Introduce next the nondimensional density

$$v(\tau, \zeta) = \frac{\rho(t, z)}{\rho_o}, \quad (3.24)$$

the nondimensional radial distance from the channel axis

$$\eta = \frac{2r}{d}, \quad (3.25)$$

and the nondimensional tube wall temperature

$$\theta_w(\tau; \eta, \zeta) = \frac{T_w(t; r, z)}{T_o} \quad (3.26)$$

Notice, that all dependent variables lie between zero and unity, except  $v$  and  $\omega$  whose product ( $v\omega$ ) remains essentially equal to unity with neither  $v$  nor  $\omega$  departing far from unity. The reference temperature  $T_o$  is, under normal operating conditions, the highest temperature in the system.

Furthermore, consider the following  $\phi$ -values to vary along the channel axis:

$$\phi_\beta = T_o^\beta \quad (3.27)$$

$$\phi_k = p_o^k \quad (3.28)$$

$$\phi_{cp} = \frac{c_p}{c_p(\rho_o, T_o)} = \frac{c_p}{c_{p,o}} \quad (3.29)$$

$$\phi_k = \frac{k}{k(\rho_o, T_o)} = \frac{k}{k_o} \quad (3.30)$$

and, finally, the following constant F-parameters:

$$F_p = \frac{p_o}{\rho_o w_o^2} \quad (3.31)$$

$$F_f = 4f \frac{L}{d} = \frac{4f}{\delta} \quad (3.32)$$

$$F_z = \frac{P_o}{\rho_o c_{p,o} T_o} \quad (3.33)$$

$$F_M = F_z / F_p \quad (3.34)$$

Let the dot above a variable designate partial differential with respect to the nondimensional time  $\tau$  and the subscript  $\zeta$  partial differentiation with respect to the dimensionless axial coordinate  $\zeta$ . Then the principal conservation equations, Eqs. 3.3, 6 and 15 read, respectively and in non-dimensional form:

$$\phi_\beta [\dot{\theta}_c + \omega(\theta_c)_\zeta] = \omega_\zeta + \phi_\kappa (\pi + \omega\pi_\zeta) \quad (3.35)$$

$$\omega = \omega_\omega \omega_\zeta = -\frac{F_p}{\nu} \pi_\zeta - F_f \frac{\omega^2}{2} \quad (3.36)$$

$$\begin{aligned} \dot{\theta}_c - F_z \frac{\phi_\beta}{\phi_{cp}} \frac{\theta_c}{\nu} + F_M \frac{1}{\phi_{cp}} \omega\omega_\zeta &= \delta \frac{4N_{Nu}}{N_{Re} N_{Pr}} \cdot \frac{1}{\nu\phi_{cp}} (\theta_\omega - \theta_c) + \frac{\delta^2}{4N_{Nu}} \\ &\times [\phi_\kappa (\theta_c)_\zeta]_\zeta - \omega(\theta_c)_\zeta + \\ &F_z \frac{1-\theta_{c\beta}}{\phi_{cp}} \cdot \frac{1}{\nu} \pi_\zeta + F_M \frac{1}{\phi_{cp}} \omega\omega_\zeta \end{aligned} \quad (3.37)$$

These are the three equations which define the three time rates of change of



$\dot{\theta}_c$ ,  $\dot{\pi}$  and  $\dot{\omega}$ . It can be seen from Eq. 3.37 that the axial conduction is always small of an order less than  $\delta^2$  (since  $N_{Nu} > 3$ ) when compared with the convective term, unless the fluid should reach the wall temperature within the very first small fraction of the tube length. Axial conduction is hence ignored as  $\delta^2 \approx 10^{-6}$ , and the order of differentiation of Eq. 3.37 is reduced to one.

$$\dot{\theta}_c - F_z \frac{\phi_\beta}{\phi_{cp}} \frac{\theta_c}{v} \dot{\pi} + F_M \frac{1}{\phi_{cp}} \dot{\omega} = \frac{4N_{Nu}}{\delta N_{Rc} N_{Pr}} \frac{\theta_\omega - \theta_c}{v_{cp}} - \omega(\theta_c)_\zeta + F_z$$

$$\frac{1 - \theta_{c\beta}}{v\phi_{cp}} \pi_\zeta + \frac{F_M}{\phi_{cp}} \omega\omega_\zeta \quad (3.38)$$

The Boundary Conditions to be imposed on Eqs. 3.35, 36 and 38 are chosen, at the channel entrance, to be

- (i) constant mass flow rate  $\dot{m}$ .
- (ii) constant inlet pressure  $p_0$ .
- (iii) continuous transition, of the inlet fluid temperature, from an initial temperature  $\theta = \theta_i$  to the constant operational temperature  $\theta(t,0) = 1$ .

These boundary conditions accomodate the calculation of the steady state conditions as well as of the most likely start-up operation toward stationary operating conditions. Notice that there are no step changes implied in any of the dependent variables which is essential for the numerical integration. Writing these boundary conditions more specifically one gets at  $\zeta = 0$

$$\theta(\tau, 0) = 1 - (1 - \theta_i) e^{-7\tau} \quad (3.39)$$

$$\pi(\tau, 0) = 1 \quad (3.40)$$

$$\omega(\tau, 0) = \frac{1}{v(\pi, \theta)} \quad (3.41)$$

Here, the time constant was chosen arbitrarily so as to have the fluid reach, within 0.1%, its steady inlet temperature at the inlet during the time interval that it takes a fluid particle to pass through the channel. Equation 3.41 is given through the thermal equation of state, Eq. 3.2.

The Initial Conditions appropriate to the system of Eqs. 3.35, 36

and 38 are derived from the requirement that the flow should initially be at steady state with the fluid inlet temperature equal to the uniform channel wall temperature. Setting the time derivatives equal to zero in Eqs. 3.35, 36 and 38 results in a system of three ordinary differential equations which are linear in  $d\pi/d\zeta$ ,  $d\theta_c/d\zeta$  and  $d\omega/d\zeta$  :

$$F_p \frac{d\pi}{d\zeta} + \frac{d\omega}{d\zeta} = -F_f v \frac{\omega^2}{2} \quad (3.42)$$

$$F_z \frac{\phi_\beta}{v} \frac{d\pi}{d\zeta} + \frac{d\theta_c}{d\zeta} + \frac{F_M}{\phi_{cp}} \omega \frac{d\omega}{d\zeta} = \frac{4N_{Nu}}{\delta N_{Rc} N_{Pr}} (\theta_w - \theta_c) \quad (3.43)$$

$$\phi_K \frac{d\pi}{d\zeta} - \phi_\beta \frac{d\theta_c}{d\zeta} + \frac{1}{\omega} \frac{d\omega}{d\zeta} = 0 \quad (3.44)$$

These equations can be solved subject to the boundary conditions

at  $\zeta = 0$

$$\left. \begin{aligned} \pi &= 1 \\ \theta_c &= \theta_i \\ \omega &= \frac{1}{v(\pi, \theta_c)} \end{aligned} \right\} \quad (3.45)$$

provided the function  $\theta_w = \theta_w(0; 1, \zeta)$  prescribing the initial channel wall temperature is specified. For the present analysis  $\theta_w$  was set equal to  $\theta_i$ . Equations 3.42 through 3.45 define the initial flow field.

Quasi-Steady Flow. It may be recognized that the momentum transport takes place at a much smaller time scale than the transport of thermal energy in that the pressure and the velocity fields adjust virtually instantaneously to a change in flow inlet conditions while the response of the temperature field to a change in channel wall temperature takes considerably longer, the reason being that the pressure perturbations propagate along the channel with the speed of sound. Unless one is specifically interested in the motion of sound waves one may consider the dynamics of the flow field, that is the pressure and velocity distributions, as part of the boundary conditions and imposed instantly and adiabatically by the flow inlet conditions.

The fluid temperature remains stationary during the dynamic adjustment and the time rate of change of both pressure and temperature remain small since ordinarily the pressure gradient remains balanced by the wall shear (and by the convective acceleration in the case of a gaseous coolant medium). Consequently, Eqs. 3.42 and 44 may serve to establish the pressure and velocity fields at all times, subject to boundary conditions given by Eqs. 3.40 and 3.41 while the temperature field remains defined by Eqs. 3.35, 36, 38 and the

initial conditions discussed above. Particularly, solving Eqs. 3.35, 36 and 38 for  $\dot{\theta}_c$  gives the differential equation which governs the temperature field:

$$\dot{\theta}_c = \frac{1}{1 - \frac{F_z}{\phi_{cp}} \frac{\phi_\beta^2}{\phi_\kappa} \frac{\theta}{v}} \left\{ \frac{4N_{Nu}}{\delta N_{Re} N_{Pr}} (\theta_w - \theta_c) + F_z \frac{\phi_\beta}{\phi_{cp} \phi_\kappa} \times \left[ \phi_\beta \omega(\theta_c)_\zeta - \frac{\theta_c}{v} \omega_\zeta \right] - \omega[(\theta_c)_\zeta + F_M F_f \frac{1}{\phi_{cp}} \frac{\omega^2}{2}] \right\} \quad (3.46)$$

This completes the discussion of the development of the governing differential equations for the coolant fluid. All equations are solved numerically as discussed in Chapter III. The thermodynamic properties  $c_p$ ,  $\beta$ , and  $\kappa$  are derived from the thermal equation of state, Eq. 3.2 and from the zero-pressure specific heat or other available properties. All thermodynamic functions as well as the transport properties  $\kappa$  and  $\mu$  are considered, in general, as functions of two state variables, that is, of  $\rho$  and  $T$  or of  $p$  and  $T$ , as discussed in Section C of Chapter II.

#### 4. The Flow Channel

The flow channel is treated as a circular tube with inner radius  $r_i = d/2$  and outer radius  $r_o = r_i + s_t$ . The tube wall temperature  $T_w(t; r, z)$  is defined through the familiar equation of energy conservation, written for the case of circular symmetry:

$$\frac{\partial T_w}{\partial t} = \alpha_w \left[ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T_w}{\partial r} \right) + \frac{\partial^2 T_w}{\partial z^2} \right] + \frac{1}{(\rho c)_w} \frac{dk_w}{dT} \left[ \left( \frac{\partial T_w}{\partial r} \right)^2 + \left( \frac{\partial T_w}{\partial z} \right)^2 \right], \quad (4.1)$$

where  $\alpha_w$  represents the thermal diffusivity of the wall material and  $(\rho c)_w$  its volumetric heat capacity. The boundary conditions are at  $r = r_i$

$$\bar{h}_c (T_w - T_c) = k_w \frac{\partial T_w}{\partial r} \quad (4.2)$$

at  $r = r_o$

$$k_w \frac{\partial T_w}{\partial r} - \psi k_f \frac{\partial T_f}{\partial x} - (2\pi - \psi) k_m \frac{\partial T_m}{\partial r} = 0 \quad (4.3)$$

Here

- $\bar{h}_c$  is the convective film coefficient,
- $T_c$  the fluid temperature,
- $k$  the thermal conductivity,
- $\psi$  the portion of outer tube circumference in contact with fins, expressed in radians,

and the subscripts w, f and m designate, respectively, tube wall, fin and meteoroid protection layer. Equations 4.2 and 3 constitute the continuity

of the heat flux at the fluid-wall interface and at the wall-fin and wall-protection layer interfaces. Circumferential temperature variations are ignored.

After introducing

$$\begin{aligned}\eta &= \frac{r}{r_i}, \quad \zeta = \frac{x}{H}; \quad \tau = \frac{t w_o}{L}; \quad \theta_w = \frac{T_w}{T_o}, \\ \phi_c &= \frac{k_c}{k_w}, \quad \phi_f = \frac{k_f}{k_w} \frac{r_i}{H}, \\ \phi_m &= \frac{k_m}{k_w} \quad N_{Bi} = \frac{\bar{h}_c d}{k_w},\end{aligned}\tag{4.4}$$

$$\delta = \frac{d}{L}, \quad \phi_{\kappa w} = \frac{T_o}{k_w} \frac{dk_w}{dT}, \quad \phi_{Fo,w} = \frac{\alpha_w L}{w_o r_i 2}$$

with  $H$  representing the fin height, that is, the distance between the fin root and fin tip, one may recast Eqs. 4.1, 2 and 3 to read

$$\begin{aligned}\dot{\theta}_w &= \phi_{Fo,w} \left\{ \frac{1}{\eta} \frac{\partial}{\partial \eta} \left( \eta \frac{\partial \theta_w}{\partial \eta} \right) + \left( \frac{\delta}{2} \right)^2 \frac{\partial^2 \theta_w}{\partial \zeta^2} \right. \\ &\quad \left. + \phi_{\kappa w} \left[ \left( \frac{\partial \theta_w}{\partial \eta} \right)^2 + \left( \frac{\delta}{2} \right)^2 \left( \frac{\partial \theta_w}{\partial \zeta} \right)^2 \right] \right\}\end{aligned}\tag{4.5}$$

at  $\eta = 1$

$$\frac{1}{2} N_{Bi} (\theta_w - \theta_c) = \frac{\partial \theta_w}{\partial \eta}\tag{4.6}$$

at  $\eta = \eta_0$

$$\frac{\partial \theta_w}{\partial \eta} - \psi \phi_f \frac{\partial \theta_f}{\partial \xi} - (2\pi - \psi) \phi_m \frac{\partial \theta_m}{\partial \eta} = 0 \quad (4.7)$$

The superscript dot represents partial differentiation with respect to nondimensional time  $\tau$ . As before in the treatment of the coolant fluid one recognizes that, with  $\delta^2 \approx 10^{-6}$ , axial conduction remains insignificant. Thus, Eq. 4.5 takes on this final form

$$\dot{\theta}_w = \phi_{Fo,w} \left\{ \frac{1}{\eta} \frac{\partial}{\partial \eta} \left( \eta \frac{\partial \theta_w}{\partial \eta} \right) + \phi_{kw} \left( \frac{\partial \theta_w}{\partial \eta} \right)^2 \right\} \quad (4.8)$$

Equations 4.6, 7 and 8 define the tube wall temperature, provided Eqs. 4.6 and 7 hold at some initial time and the initial temperature  $\theta_w(0; \eta, \zeta)$  is prescribed as a sufficiently smooth function of  $\eta$  and  $\zeta$ .

Low Biot Number. When one compares possible heat fluxes at the fluid-wall interface with the possible radiant fluxes from the outer surface of the meteoroid protection layer covering the tube, one concludes that the maximum fluxes occur at the inner tube wall and that the Biot number  $N_{Bi}$  in Eq. 4.6 is the largest ratio to be expected of external to internal thermal conductances. Thus, if  $N_{Bi}$  is small, say less than 0.05 (Ref. 6), then the temperature variation inside the tube wall is too small for experimental detection and a computation of the detailed temperature distribution on the basis of Eq. 4.8 cannot be justified as the associated computational effort is considerable.

In cases where the equivalent Biot number, representing the total thermal resistance within the channel wall and the protection layer, is

small, the tube, the protection layer and a representative portion of the fin root are combined into a single control volume as depicted in Fig. 5.

The equivalent Biot number  $\bar{N}_{Bi}$  and the chosen limit are

$$\bar{N}_{Bi} = N_{Nu} \frac{k_c}{d} \left( \frac{s_w}{k_w} + \frac{s_m}{k_m} \right) < 0.05 \quad (4.9)$$

where  $s_w$  and  $s_m$  stand for, respectively, the tube wall and the protection layer thicknesses. The combined volumetric heat capacity per unit of axial distance for the control volume consists of three parts, the first one for the tube wall, the second for the protection layer and the third for the first half fin element:

$$\pi d (\rho c s)_w \left\{ \left( 1 + \frac{s_w}{d} \right) + \left( 1 + 2 \frac{s_w}{d} + \frac{s_m}{d} \right) \frac{(\rho c s)_m}{(\rho c s)_w} + \frac{\Delta x}{\pi d} \frac{s_r}{s_w} \times \right. \\ \left. \left[ 1 - \frac{\Delta \xi}{4} \left( 1 - \frac{s_t}{s_r} \right) \right] \frac{(\rho c)_f}{(\rho c)_w} \right\} = \lambda_1 \pi d (\rho c s)_w \quad (4.10)$$

where the subscripts w, m, f, r and t designate, respectively, parameters of the tube wall, the meteoroid protection layer, the fin, the fin root and the fin tip and where

d is the tube diameter,

$\rho$  the density,

s the thickness,

$\Delta x$  the node spacing on the fin, and

$\Delta \xi = \Delta x/H$ , the nondimensional node spacing

Heat enters the control volume, per units of time and axial distance,



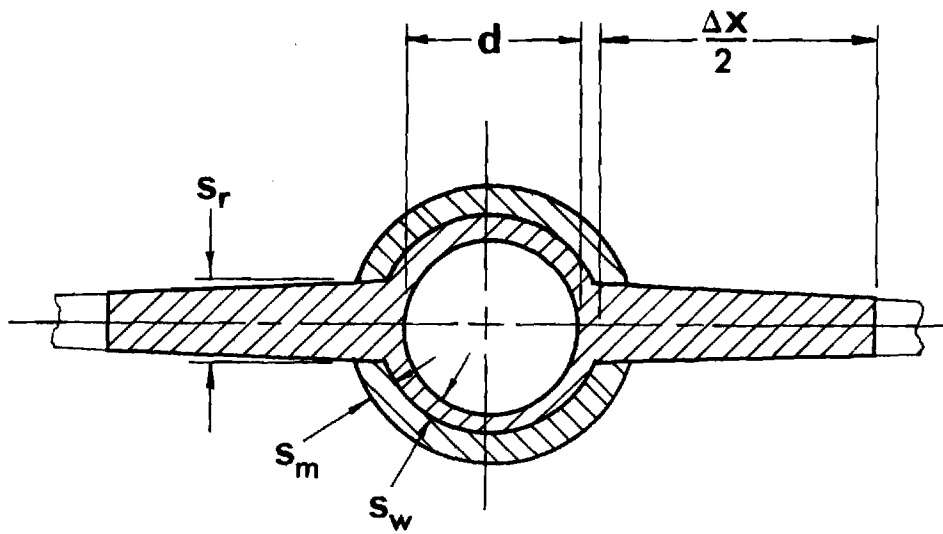


Figure 5. Control Volume for Low-Biot Number Cases

by convection from the fluid

$$\pi d \bar{h}_c (T_w - T_c),$$

and by convection and/or radiation from outside

$$(2\pi - \psi) \left( \frac{d}{2} + s_w + s_m \right) q''_m + \Delta x (q''_1 + q''_2)_f$$

where the subscripts 1 and 2 designate upper and lower fin sides, respectively, and where  $q''$  represents the sum of the convective and of the net radiant fluxes entering the outer surfaces. In the case of  $q''_m$ , the average over upper and lower portions of the outer circumference is to be taken.

Heat leaves the control volume through the fin by conduction, again per units of time and axial distance

$$2 \left( s \frac{\partial T}{\partial x} \right)_f$$

evaluated at  $x = \Delta x/2$ .

Combining the last four expressions into the energy balance leads to this expression

$$\lambda_1 \pi d (\rho c s)_w \frac{\partial T_w}{\partial t} = \pi d \bar{h}_c (T_w - T_c) + \pi d \lambda_2 q''_m + \Delta x (q''_1 + q''_2)_f + 2 \left( s \frac{dT_f}{dx} \right) \Big|_{\frac{\Delta x}{2}} \quad (4.11)$$

where  $\lambda_1$  is defined by Eq. 4.10 and

$$\lambda_2 = (2 - \frac{\psi}{\pi}) \left( \frac{1}{2} + \frac{s_w + s_m}{d} \right) \quad (4.12)$$

Equation 4.11 replaces Eqs. 4.6, 7 and 8 as well as Eqs. 5.3, and 5 governing the temperature distribution in the meteoroid protection layer and, lastly,

Eq. 2.19 which constitutes one of the boundary conditions for the differential equation governing the fin temperature distribution. The only condition under which all the above equations may be replaced by the single equation, Eq. 4.11, is given by Eq. 4.9. Finally, it may be noted that Eq. 4.11 could also be normalized but since no new dimensionless groups result from such normalization it is omitted here.

## 5. The Meteoroid Protection Layer

The differential equation governing the temperature distribution within the meteoroid protection layer which covers the flow channel, is identical to that for the channel wall, Eq. 4.8, except for the two dimensionless parameters, the Fourier coefficient  $\phi_{Fo}$  and the conductivity temperature coefficient  $\phi_k$  which now must be evaluated for the protection layer material:

$$\phi_{Fo,m} = \frac{\alpha_m}{\alpha_w} \phi_{Fo,w} \quad (5.1)$$

$$\phi_{k,m} = \frac{T_o}{k_m} \frac{dk_m}{dT} \quad (5.2)$$

After ignoring the axial conduction for the reasons stated in Section 4 one obtains as the nondimensional energy conservation equation

$$\dot{\theta}_m = \phi_{Fo,m} \left\{ \frac{1}{\eta} \frac{\partial}{\partial \eta} \left( \eta \frac{\partial \theta_m}{\partial \eta} \right) + \phi_{k,m} \left( \frac{\partial \theta_m}{\partial \eta} \right)^2 \right\} \quad (5.3)$$

Two boundary conditions are required, one of which is given by Eq. 4.7 while the other one is dictated by the heat flux continuity at the outer boundary:

at  $r = r_e$

$$k_m \frac{\partial T_m}{\partial r} = q''_m \quad (5.4)$$

where  $q''$  is defined as the net flux entering both by radiation and/or aerodynamic heating. Equation 5.4 reads in nondimensional form at  $\eta = \eta_e$

$$\frac{\partial \theta_m}{\partial \eta} = \phi_{Nc} (\tilde{q}_{net,rad} + \tilde{q}_{aero}) \quad (5.5)$$

with  $\phi_{Nc}$  representing a local conductance parameter

$$\phi_{Nc} = \frac{\sigma d T_o^3}{2k_m} \quad (5.6)$$

and

$$\tilde{q}_{net,rad} = \frac{q''_{net,rad}}{\sigma T_o^4} \quad (5.7)$$

$$\tilde{q}_{aero} = \frac{q''_{aero}}{\sigma T_o^4} \quad (5.8)$$

Here,  $\sigma$  represents the Stefan-Boltzmann constant, and  $q''_{net,rad}$  and  $q''_{aero}$  the net radiant incident heat flux and the convective heat flux, respectively.

If the initial temperature distribution is given and if Eqs. 4.7 and 5.5 hold initially then the protection layer temperature  $\theta$  is completely defined as a function of time and location by Eqs. 5.3, 5 and 4.7, provided the incident radiation and the aerodynamic heating are prescribed as functions of time. The prediction of these incident heat fluxes is discussed in Section 6 and 7 of this Chapter.

In the case of low Biot numbers at the coolant fluid-tube wall interface the meteoroid protection layer is lumped together with the tube wall as discussed in Section 4 of Chapter II.

## 6. Radiation

The objective of this section is to develop a procedure by which to predict the net radiant heat flux incident on fin and tube surfaces which are exposed to any combination so solar, albedo and planetary irradiation. Included into the assessment of radiative transfer is the radiative energy exchange between fin panel and flow channel or its protective coating as well as the effect of structural panels in the vicinity of the fin system; however not included is any possible gas radiation as could conceivably be encountered during reentry.

In seeking the proper mathematical model it is recognized, firstly, that the prevailing thermal radiant energy lies in either the visible (solar irradiation) or the infrared portion of the spectrum, and, secondly, that the fin system is coated, for the purpose of optical optimization, with a dielectric paint. Consequently, spectrally dependent optical properties must be dealt with, but, for the latter reason, the transfer matrix of radiative exchange can be expected to remain temperature insensitive over some range of operational conditions, a fact which is very much appreciated from the computational view point.

For the analysis, the fin surface  $A_f$ , the outer surface on the flow channel  $A_n$ , and the structural surface(s)  $A_n$ , are all considered as parts of an enclosure C which is completed by a set of arbitrarily concave, non-reflecting imaginary surfaces  $A_e$  which connect  $A_m$ ,  $A_f$  and  $A_n$  and along which is specified the emerging net radiant heat flux representing solar, albedo and planetary irradiation. The sum  $A_m + A_f + A_n + A_e$  is the inner surface  $A_c$  of the enclosure

Three steps are necessary for the prediction of the incident net radiant heat flux  $q''_{\text{net,rad}}$ . Firstly, the elemental exchange areas\*

$$\frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} = \frac{\cos \phi_i \cos \phi_j}{\pi r^2} \quad (6.1)$$

need to be computed on the basis of the geometric relation between fin panels and flow channels. Here, the symbol  $r$  designates the distance between two area elements  $dA_i$  and  $dA_j$  which are visible from each other, and  $\phi_i$  and  $\phi_j$  represent the respective angles between  $r$  and the surface normals on  $dA_i$  and  $dA_j$ . The next step is to compute, from its definition, the radiosity or leaving radiant flux density,  $W_j$ :

$$W_j = \int_0^\infty W_{j,\lambda} d\lambda = \int_0^\infty \epsilon_{j,\lambda} E_{j,\lambda} d\lambda + \quad (6.2)$$

$$\int_{A_c} \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} \int_0^\infty (1 - \epsilon_{j,\lambda}) W_{i,\lambda} d\lambda dA_i$$

where  $W_{j,\lambda}$ ,  $E_{j,\lambda}$  and  $\epsilon_{j,\lambda}$  stand for the monochromatic radiosity, the monochromatic black body emissive power

$$E_{j,\lambda} = \frac{2 hc^2 n^2}{\lambda^5} \frac{1}{e^{(hc)/(\lambda T_j)} - 1} \quad (6.3)$$

\*

For terminology and notations consult Ref. 7, Chapter 2.

and the monochromatic hemispherical emittance, respectively; the subscripts  $i$  and  $j$  designate two discrete points on  $A_c$ , and  $\lambda$  represents the wavelength. The Eq. 6.3 constitutes Planck's law of monochromatic emissive power intensity;  $h$ ,  $c$  and  $k$  stand for, respectively, Planck's constant, the speed of light in vacuo and the Boltzmann constant. The third and last step is to calculate, on the basis of local energy balance, the net incident heat flux

$$(q''_{\text{net,rad}})_j = \int_0^\infty q''_{j,\lambda} d\lambda = \int_{A_c} (W_i - W_j) \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} dA_i \quad (6.4)$$

It should be obvious that Eqs. 6.1, 2 and 4 contain all the necessary fundamental principles but their evaluation will introduce a number of simplifications and modifications, each selected for the particular system of interest. Specifically, Eq. 6.4 will have to supplement Eq. 6.2 for portions of  $A_c$  where the heat flux is specified. More importantly however, there is a choice to be made in view of the computational process regarding particularly Eq. 6.2. One may either solve the monochromatic version of Eq. 6.2  $n$  times for the  $n$  significant spectral intervals encountered and thus face the ultimate task of solving  $n \times N$  simultaneous linear algebraic equations when  $N$  discrete points on  $A_c$  need to be considered (possibly at several time steps during the calculation process) and then integrate the resulting total interchange areas over the spectrum (see Ch. 5.6 of Ref. 7).



Or, one may force the non-gray surface analysis into a gray surface analysis by placing the burden of complexity on the evaluation of appropriate optical properties. Both techniques afford any desirable accuracy of allowing for the spectral differences in surface properties, limited only by available calculation time; but the latter technique was chosen because, as a result of this choice, the complexity remains at peripheral parts of the computer code which are more accessible for later modifications toward greater sophistication, also the complexity may turn out, in almost all cases, to reduce partly to simple hand calculations.

After introducing

$$\epsilon_i = \frac{\int_0^\infty \epsilon_{i,\lambda} E_{i,\lambda} d\lambda}{\int_0^\infty E_{i,\lambda} d\lambda} = \frac{1}{E_i} \int_0^\infty \epsilon_{i,\lambda} E_{i,\lambda} d\lambda \quad (6.5)$$

$$\alpha_{ij} = \frac{\int_0^\infty \epsilon_{i,\lambda} W_{j,\lambda} d\lambda}{\int_0^\infty W_{j,\lambda} d\lambda} \quad (6.6)$$

Equation 6.2 simplifies to

$$W_j = \epsilon_j E_j + \int_{A_c} \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} (1 - \alpha_{ij}) W_i dA_i \quad (6.7)$$

which reduces to the gray-surface radiosity equation whenever Eq. 6.6 reduces to  $\alpha_{ij} = \epsilon_i$ . The difficulty now lies in evaluating Eq. 6.6 even though the radiosity  $W_{j,\lambda}$  is as yet unknown.

By successively substituting the right-hand side of Eq. 6.2, in its monochromatic form, for  $W_{i,\lambda}$  on the right-hand side of that equation, one obtains first (Ref. 8)

$$W_{j,\lambda} = \epsilon_{j,\lambda} E_{j,\lambda} + (1 - \epsilon_{j,\lambda}) \left\{ \int_{A_c} \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} \left[ \epsilon_{i,\lambda} E_{i,\lambda} + (1 - \epsilon_{i,\lambda}) \int_{A_c} \frac{\partial^2 \overline{s_i s_m}}{\partial A_i \partial A_m} (\epsilon_{m,\lambda} E_{m,\lambda} + \dots) dA_m \right] dA_i \right\} \quad (6.8)$$

and subsequently  $\alpha_{ij}$  as the quotient of two infinite series. Since the enclosure radiation is dominated by the fin-sun and fin-sky interaction and since Eq. 6.8 contributes significantly only to the fin-flow channel interaction, the infinite series in Eq. 6.8 may be terminated after two terms (two reflections; the resulting error is less than the uncertainty in  $\epsilon_\lambda$ ), and Eq. 6.6 becomes:

$$\alpha_{ij} = \frac{X_{ij} E_j + \int_{A_c} \frac{\partial^2 \overline{s_j s_k}}{\partial A_j \partial A_k} E_k (X_{ik} - X_{ijk}) dA_k}{\epsilon_j E_j + \int_{A_c} \frac{\partial^2 \overline{s_j s_k}}{\partial A_i \partial A_k} E_k (\epsilon_k - X_{jk}) dA_k} \quad (6.9)$$

where

$$X_{ij} = \int_0^\infty E_{j,\lambda} \epsilon_{j,\lambda} \epsilon_{i,\lambda} d\lambda / E_j \quad (6.10)$$

$$X_{ijk} = \int_0^{\infty} \frac{E_{k,\lambda} \epsilon_{k,\lambda} \epsilon_{j,\lambda} \epsilon_{i,\lambda} d\lambda}{E_k} \quad (6.11)$$

In cases where the net radiant flux is specified over portions of  $A$ , the emissive power  $E$  is to be replaced by the net radiant flux  $q''$  in Eqs. 6.9, 10 and 12, which results in one additional term each in the numerator and the denominator of Eq. 6.9.

In summary, the incident net radiant heat flux for the diffuse, non-gray enclosure is calculated on the basis of an approximate gray-surface analysis in accordance with Eq. 6.4, 7 and 9 through 11. The spectral differences of the surfaces are accounted for in Eqs. 9 through 11. The remainder of this section is devoted to the solution of the radiosity equation, Eq. 6.7.

Recalling that  $A_c$  is the sum  $A_m + A_n + A_f + A_e$  of the outer channel surface  $A_m$ , the possibly present, nearby structural surfaces  $A_n$ , the fin surface  $A_f$ , and the remainder of the enclosure  $A_e$ , one recognizes that the integrals over  $A_c$  in Eqs. 6.4, 7 and 9 need to be evaluated twice for each of the four parts, namely once with  $j = 1$  representing the fin area and then with  $j = 2$  for the exposed channel area. Since the incident solar, albedo and planetary radiant flux intensities are uniform over the fin area and averaged over the circumference of the channel area

$$\int_{A_e} \frac{\partial^2 \overline{s_1 s_1}}{\partial A_1 \partial A_1} (1 - \alpha_{i1}) W_1 dA_1 = (1 - \alpha_{e1}) q''_{e1} \quad (6.12)$$

$$\int_{A_e} \frac{\partial^2 \overline{s_i s_2}}{\partial A_i \partial A_2} (1 - \alpha_{i2}) W_i dA_i = (1 - \alpha_{e2}) q''_{e2} \quad (6.13)$$

where  $q''$  designates incident solar, albedo and planetary heat fluxes, appropriately averaged over a chosen area element. Should any structural surfaces obstruct the incident radiant fluxes ( $A_n \neq 0$ ) then the right-hand sides of Eqs. 6.12 and 13 would have to be modified and reduced in the shaded portions of  $A_1$  and  $A_2$ ; and, if there are  $m$  such surfaces,

$$\int_{A_n} \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} (1 - \alpha_{ij}) W_i dA_i = \sum_{k=1}^m \frac{\partial \overline{s_k s_j}}{\partial A_j} (1 - \alpha_{kj}) W_k, \quad (6.14)$$

$$j = 1, 2, \dots, m + 2.$$

Obviously, the radiosity and the temperature are assumed to be uniform over each structural component. No such structural components were considered in the program phase being reported on here, and Eq. 6.14 is taken to be zero.

This leaves only the integrals over  $A_m$  and  $A_f$  to be discussed. Moreover, since Eq. 6.1 is symmetric with respect to its subscripts  $i$  and  $j$ , the elemental exchange area is to be evaluated only once.

Considering first the fin, that is  $j = 1$  and  $i = 2$ , and the fact that over the channel surface the radiosity and the temperature are considered to be functions of axial distance only

$$\begin{aligned}
& \int_{A_2} \frac{\partial^2 \overline{s_1 s_1}}{\partial A_1 \partial A_1} (1 - \alpha_{1i}) W_i dA_i = \\
& \int_{z=0}^{z=L} [1 - \alpha_{1i}(z)] W_i(z) \int_{\phi=0}^{\phi^*} \frac{\partial}{\partial A_1} \left( \frac{\overline{s_1 s_1}}{\Delta A_1} \right) R d\phi dz = \quad (6.15) \\
& \int_{z=0}^{z=L} [1 - \alpha_{1i}(z)] W_i(z) SS(z; x_f, z_f) dz
\end{aligned}$$

where  $L$  designates the tube length;  $R$  and  $\phi$  are the polar coordinates of  $A_2$ , with origin on the tube axis, with  $\phi = 0$  and  $\phi = \phi^*$  representing, respectively, the root of the fin and the contact line between the tube and its tangent plane through the center of  $\Delta A_1$  on the fin. The first step in Eq. 6.15 was obtained by integrating over  $\Delta A_1$  and subsequently applying the mean-value theorem of integral calculus, while the second step simply defines the exchange function for every point  $(x_f, z_f)$  on the fin which was integrated in closed form for the right-circular flow channel. The result is shown in Appendix D.

The exchange function of the tube with respect to the fin is obtained by dividing  $SS$  in Eq. 6.15 by  $(R\phi^*)$ . Thus

$$\begin{aligned}
& \int_{A_1} \frac{\partial^2 \overline{s_2 s_1}}{\partial A_2 \partial A_1} (1 - \alpha_{2i}) W_i dA_i = \\
& \frac{1}{R\phi^*} \int_{z=0}^L \int_{x=0}^H [1 - \alpha(x, x_f, z_f)] W(x_f, z_f) SS(z, x_f, z_f) dx_f dz_f \quad (6.16)
\end{aligned}$$

For the numerical evaluation of the integrals a suitable quadrature such as the trapezoidal rule is chosen so as to render Eq. 6.7 in this form

$$P_j = \sum_i M_{ji} W_i \quad (6.17)$$

$i, j = 1, 2, \dots, N$

which is a system of  $N$  linear algebraic equations for the  $N = (n_x + 1) \times (n_z + 2)$  unknown values of the radioactivity  $W_i$ . Here,  $n_x$  and  $n_z$  are the numbers of subdivisions chosen in the  $x$ - and the  $z$ - directions, respectively. The vector  $P$  on the left-hand side of Eq. 6.17 is called the excitation vector

$$P_j = -\epsilon_j E_j - (1 - \alpha_{ej}) q''_{ej} \quad (6.18)$$

on the right-hand side, the transfer matrix  $M_{ji}$  is given by

$$\delta_{ji} - \chi_i [1 - \alpha_{ji}] SS_{ji} = M_{ji} \quad (6.19)$$

where

$$\delta_{ji} = \begin{matrix} 0 & i \neq j \\ 1 & i = j \end{matrix} \quad \text{for}$$

is the Kronecker delta,  $\chi_i$  is a suitable quadrature coefficient and  $SS_{ji}$  is given either by Eq. 6.15 or by Eq. 6.16 depending on whether  $j$  refers to the fin or to the channel, respectively. There is no matrix multiplication implied in Eq. 6.19, hence the underscores.

In the present program phase, Eq. 6.17 is solved at every time step only when the transfer matrix is sufficiently temperature sensitive, otherwise the transfer matrix is completely inverted only once to yield

the unknown radiosity at any time.

$$W_i = \sum_j (M_{ij})^{-1} P_j \quad (6.20)$$

through a simple matrix multiplication. It may be noted that the most significant temperature dependence of optical properties is contained in the excitation vector  $P_j$ .

All radiant heat fluxes are normalized with respect to  $\sigma T_o^4$  where  $T_o$  designates the reference temperature, that is the fluid inlet temperature. Exchange factors are nondimensional and need not be normalized.

$$w_j = \frac{W_j}{\sigma T_o^4}, \quad \tilde{q}_j = \frac{q_j}{\sigma T_o^4}, \quad p_j = \frac{P_j}{\sigma T_o^4} \quad (6.21)$$

The nondimensional forms of Eqs. 6.4 and 6.20 are used to compute  $\tilde{q}_{\text{net,rad}}$  in Eqs. 2.17, 4.11 and 5.5.

## 7. Aerodynamic Heating

The aerodynamic heating model used to evaluate the convective flux from the radiator surface on the orbiter vehicle is subdivided into three major regimes. The first regime encompasses low speed flow for which the heat transfer coefficients are determined from expressions appearing in standard heat transfer texts for flow over a flat plate. The second regime consists of a model for high speed flow in which the convective heat transfer coefficient is evaluated from an experimental correlation for flow over the upper surface of the shuttle orbiter vehicle (Ref. 9). The third section of the model encompasses a low to high speed transitional flow regime. Within this regime the heat transfer coefficient is an interpolated value that lies between the values obtained in the low and high speed regimes. Calculations for the convective heat flux for all three regimes are based on Eckerts reference enthalpy method (Ref. 10).

Within each of the three regimes the heat transfer coefficient is calculated for cases where the flow is laminar, transitional or fully turbulent. In addition to the evaluation of the heat flux when the flow is forced, the procedure accounts for heat transfer by free convection at times when the shuttle vehicle is stationary or moving with a relatively low velocity.

The program for the evaluation of the aerodynamic heating rate is divided into six sub-tasks each of which is written as a separate subprogram. This procedure allows for changes in the periphery of the program without affecting the program foundation. The basic calculations are carried out in and controlled from the SUBROUTINE CONVEC. Atmospheric temperature and speed of sound are evaluated within the SUBROUTINE ATMOS. Atmospheric



properties evaluated at the reference temperature are calculated within the SUBROUTINE REFP. The orbiter velocity and altitude are evaluated in the FUNCTION subprograms VELSH and ALTSH, respectively. The SUBROUTINE NUS evaluates the Nusselt number for the radiator system.

It should be noted that the analysis does not account for the effects of shock wave interaction or interference heating caused by flow interference between the orbiter, booster or any supporting structure.

The analysis for the determination of the aerodynamic heating first requires the evaluation of a reference temperature which is used for the determination of all air properties. The reference temperature is a function of the Prandtl number and recovery factor of the air, as well as the vehicle Mach number.

The Mach number in turn is a function of the altitude and velocity of the orbiter at any instant time. Altitude and velocity profiles for the orbiter are contained in data arrays supplied by the user (see Part I, Chapter B). If the value of the integer I is used to specify either ascent or reentry phase, then the N paired data points which define the velocity V and elevation Z as a function of time t may be expressed functionally as

$$V_i = V_i(I, t_i) \quad i = 1, 2, \dots, N \quad (7.1)$$

$$Z_i = Z_i(I, t_i) \quad i = 1, 2, \dots, N \quad (7.2)$$

Once the orbiter velocity and altitude are known as a function of time, the vehicle Mach number M may be calculated from the equation

$$M = \frac{V}{c} \quad (7.3)$$

The speed of sound  $c$  is an atmospheric property that is a function only of the altitude.

The reference temperature is a function of the recovery factor  $r$  which for laminar flow is

$$r = \sqrt{N_{Pr}}$$

and for turbulent flow (Ref. 11) is

$$r = N_{Pr}^{(8 + 0.528M^2/(22 + M^2))} \quad (7.4)$$

where the Prandtl number  $N_{Pr}$  is an atmospheric property. To avoid a discontinuity in the value of the recovery factor between the laminar and turbulent flow models, Eq. 7.4 was used as the expression for the recovery factor for both flow models. The resulting error in the reference temperature was found to be approximately 5 R in an extreme case.

All of the properties for the atmosphere used in the evaluation of the heat transfer coefficient are evaluated at the high speed reference temperature. Eckert (Ref. 10) recommends the expression

$$i^* = 0.5 (i_\infty + i_w) + 0.11 r (\gamma - 1) M^2 i_\infty \quad (7.5)$$

for the reference enthalpy  $i^*$  which can be converted to the reference temperature  $T^*$  once the relationship

$$T^* = T(i^*) \quad (7.6)$$

between the atmospheric enthalpy and temperature is known. The subscripts " $\infty$ " and " $w$ " in Eq. 7.5 refer to the enthalpy of the air evaluated at the free stream and surface temperature, respectively, and  $\gamma$  is the ratio of the specific heats for air.

It should be mentioned that when velocities are low ( $M \rightarrow 0$ ) Eqs. 7.5 and 6 yield a reference temperature that approaches the film temperature  $(T_\infty + T_w)/2$ . As a result Eqs. 7.5 and 6 were used to evaluate the reference temperature for all three flow regimes, i.e., low speed, high speed and the transitional regimes.

The convective flux is the product of the convective heat transfer coefficient and the difference between the air enthalpy evaluated at the surface temperature and at the adiabatic wall temperature. The adiabatic wall enthalpy  $i_{aw}$  is related to the free stream enthalpy, recovery factor and vehicle velocity by the relationship

$$i_{aw} = i_\infty + \frac{rV^2}{2g_c} \quad (7.7)$$

The convective heat transfer coefficient  $h_i$  used in the reference enthalpy method may be expressed in terms of the Nusselt number  $N_{Nu}$

$$N_{Nu} = \frac{h_{i,xc}^*}{k^*} \quad (7.8)$$

where  $c_p^*$  and  $k^*$  denote the atmospheric specific heat and thermal conductivity evaluated at the reference temperature  $T^*$ . The symbol  $x$  denotes a characteristic length of the radiator system which for forced convection is the distance from the stagnation point on the shuttle to the center of the radiator panel.

The expressions for the orbiter Nusselt number selected for the low speed and high speed regime and for laminar, transitional and turbulent flow are summarized in Table 2. Values for the Nusselt number for conditions lying between the low and high speed regimes were obtained by interpolation so that the convective heat flux from the shuttle varies

TABLE 2

## FORCED CONVECTION NUSSELT NUMBER FOR ORBITER

	LOW SPEED REGIME $M \leq 0.5$	HIGH SPEED REGIME $M \geq 1.0$
LAMINAR FLOW $Re < 1.0 \times 10^5$	$N_{Nu} = 0.332 N_{Re}^{1/2} N_{Pr}^{1/3}$	$N_{Nu} = 0.375 N_{Re}^{0.514} N_{Pr}$
TRANSITIONAL FLOW $1 \times 10^5 \leq Re \leq 1 \times 10^6$	$N_{Nu} = 6.78 \times 10^{-5} N_{Re}^{1.238} N_{Pr}^{1/3}$	$N_{Nu} = 3.39 \times 10^{-4} N_{Re}^{1.111} N_{Pr}$
TURBULENT FLOW $Re > 1 \times 10^6$	$N_{Nu} = 0.0288 N_{Re}^{0.8} N_{Pr}^{1/3}$	$N_{Nu} = 0.0346 N_{Re}^{0.7746} N_{Pr}$

continually from one regime to the other. The symbols  $N_{Re}$  and  $N_{Nu}$  appearing in the table denote the Reynolds number and Prandtl number, respectively,

where

$$N_{Re} = \frac{\rho^* V_\infty}{\mu^*}$$

$$N_{Pr} = \frac{\mu^* c_p^*}{k^*}$$

The "\*" superscript on each property indicates that the property is evaluated at the temperature  $T^*$ .

For the low speed regime the expressions for the Nusselt number are those for laminar transitional and turbulent flow over a flat plate (Ref.6\*) Nusselt number relationships for high speed flow regime are taken from Ref. 9 where experimental wind tunnel data are presented for a delta space shuttle orbiter. The data are for leeward surface heat transfer at angles of attack between  $10^\circ$  and  $30^\circ$  and Mach numbers of 8 and 16. The Nusselt number is shown to be relatively independent of angle of attack so that the high speed correlation may be applied to both the ascent phase for which the angle of attack is approximately zero and the reentry phase when the angle of attack may approach  $60^\circ$ . The scatter in the data of Ref. 9 from the selected Nusselt relationships is on the order of 100%.

The leeward surface correlation was selected because the aerodynamic heating rates in this region of the shuttle are relatively low when compared to heating rates for the lower body or windward surface. Estimates place the peak reentry temperature of the lower surface stagnation line

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\*

around 2100 F while the ~~peak~~ leeward temperature is estimated to be about 600 F (Ref. 12). Therefore placing the radiator on the upper surface of the orbiter not only will result in a more efficient operation upon reentry, but also will minimize the need for reservicing of the surface coating on the radiator panels.

The aerodynamic heating analysis includes free convection from the radiator surface during pre-launch operation and when the shuttle vehicle is moving with a relatively low velocity. The expression for the free convection Nusselt number is a function of the Grashof Prandtl product where the Grashof number  $N_{Gr}$  is given by

$$N_{Gr} = g\beta \left(\frac{\rho}{\mu}\right)^2 y^3 (T_w - T_\infty)$$

where  $g$  represents the acceleration of gravity,  $\rho$ ,  $\mu$  and  $T_\infty$  are the atmospheric density, dynamic viscosity and temperature, respectively and  $T$  denotes the radiator surface temperature. The symbol  $y$  denotes the overall dimension of the radiator panel in the direction parallel to the acceleration of gravity. Since the atmosphere is assumed to be an ideal gas, the coefficient of thermal expansion  $\beta$  is simply the reciprocal of the average absolute temperature of the air

$$\beta = \frac{1}{T_w + T_\infty}$$

The free convection Nusselt numbers  $N_{Nu_f}$  and their applicable ranges of Grashof Prandtl product used for this analysis are

$$N_{Nu_f} = 1.585 (N_{Gr} N_{Pr})^{0.195} \quad (10^{-1} < N_{Gr} N_{Pr} < 10^9) \quad (7.9a)$$

$$N_{Nu_f} = 0.590 (N_{Gr} N_{Pr})^{0.250} \quad (10^4 \leq N_{Gr} N_{Pr} \leq 10^9) \quad (7.9b)$$

$$N_{Nu_f} = 0.130 (N_{Gr} N_{Pr})^{0.333} \quad (N_{Gr} N_{Pr} > 10^9) \quad (7.9c)$$

The values for the free convection Nusselt number given in Eqs. 7.9 were multiplied by the ratio  $(x/y)$  and then added to the forced convection Nusselt number to obtain a value that accounts for combined free and forced convection in the low speed regime.

Equations 7.1 through 7.8 combined with the appropriate Nusselt number relationship from Table 2 for forced convection and Eqs. 7.9 for free convection are sufficient to determine the convective heat flux into the radiator surface which is given by

$$q''_{aero} = h_i (i_\infty - i_{aw}) \quad (7.10)$$

The convective flux may be normalized by dividing by the heat flux  $\sigma T_o^4$  or

$$q_{aero} = \frac{h_i (i_\infty - i_{aw})}{\sigma T_o^4} \quad (7.11)$$

where  $T_o$  is the fluid temperature at the inlet plane to the flow channel. The normalized convective heat flux given by Eq. 7.11 is used in the energy equation for both the fin (Eq. 2.18) and the meteoroid protection layer (Eq. 5.5).

## B. DESIGN PARAMETERS

Certain design parameters are necessary for the design specification, for the selection of the optimum radiator system and even the system definition as required for the heat transfer analysis. In the following are discussed, in that order, the prediction of the meteoroid protection layer thickness, the system weight and a collection of nondimensional groups which define the radiator system, the operational conditions, and the performance characteristics.



## 8. Meteoroid Protection Thickness

In this section an engineering equation is developed to predict the thickness of a meteoroid protection layer required to cover all radiator surfaces that might be damaged by the impact of a meteoroid. Several assumptions have been made during the derivation. They are:

1. The meteoroid particle is spherical.
2. The meteoroid flux is isotropic.
3. Poissons distribution law describes the probability of an impact of a meteoroid.

It should be mentioned that any equation used to predict meteoroid protection thickness is only as accurate as the experimental data used in that equation. Even though much information has been published in recent years concerning protection theories, there is still considerable question as to the density, velocity and mass distribution of meteoroid particles in outer space. In addition to these uncertainties, two basic models for penetration theory have been proposed within the last decade and there appears to be no close agreement between the two. Experimental verification of either model has been hampered by the fact that particle velocities used in experimental tests have only recently approached the meteoroid velocity range. In short, an extremely reliable theory for the prediction of protection layer thickness does not presently exist.

Structural materials that can be used in this study as a protection layer are copper, aluminum and beryllium. While both copper and aluminum were selected primarily as fin and tube materials due to their superior heat

transfer characteristics, beryllium was chosen for its protection capabilities. The penetration theory predicts a protection layer thickness that decreases as the modulus of elasticity increases. Therefore beryllium becomes an attractive protection material due to its high modulus of elasticity and its relatively low density. In fact studies (Ref. 13 and 14) have shown that beryllium can significantly reduce protection layer weight.

The basic equation (Ref. 15) for the depth of penetration of a meteoroid particle into a target of infinite depth is

$$P_{\infty} = \gamma d \left( \frac{\rho_p}{\rho_t} \right)^{\phi} \left( \frac{\bar{v}}{\bar{c}} \right)^{\theta} \quad (8.1)$$

where

$\gamma$  empirical constant generally accepted to be in the range of 1.5 to 2.5.

$\rho_p$  density of the meteoroid particle

$\rho_t$  density of the target material

$\bar{v}$  velocity of the meteoroid particle

$\bar{c}$  velocity of sound in the target material

$d$  diameter of the meteoroid particle

$\phi$  constant between 1/3 and 2/3

$\theta$  constant between 1/3 and 2/3

The ratio of the meteoroid velocity and the velocity of sound in the target represents a target Mach number. The velocity of sound in the target can be expressed in terms of the modulus of elasticity

$$\bar{c} = \sqrt{E_t g_c / \rho_t} \quad (8.2)$$

where

$E_t$  modulus of elasticity of the target material  
 $g_c$  proportionality constant relating mass units to force units  
 $\rho_t$  target density

If the meteoroid particle is assumed to be spherical, the diameter may be written in terms of its mass

$$d = \left( \frac{6M_p}{\pi \rho_p} \right)^{1/3} \quad (8.3)$$

where

$M_p$  meteoroid mass  
 $\rho_p$  meteoroid density

The probability that an exposed surface will be struck by a meteoroid during a period of time can only be determined after the distribution of meteoroids of a given mass is known. This information is usually given in the form of an equation such as

$$F = \alpha M_p^{-\beta} \quad (8.4)$$

where

$F$  cumulative number of impacts of particles with mass  $M$  or larger per unit area per unit time

$M_p$  mass of the meteoroid particle

The symbols  $\alpha$  and  $\beta$  represent experimentally determined constants. Published (Ref. 16) values of  $\alpha$  and  $\beta$  vary over a considerable range, but they lie within the limiting values,

$$1.3 \times 10^{-15} < \alpha < 2.54 \times 10^{-9} \quad [\text{ft}^2 \text{ day gm}^{-\beta}]^{-1}$$

$$1.11 < \beta < 1.34 \quad [\text{Dimensionless}]$$

The cumulative number of impacts on a surface with a vulnerable area of  $A$  during a mission time of  $\tau$  by a meteoroid of mass  $M_p$  or larger is then

$$N = FA\tau = A\tau \propto M_p^{-\beta}$$

It is generally assumed (Ref. 17) that meteoroids are randomly distributed in outer space and that each collision can be described by Poisson's probability law. From the Poisson distribution function, the probability of zero events occurring  $P_0$  when the average number of events is  $N$  is

$$P_0 = e^{-N}$$

or

$$\ln P_0 = -N$$

Substituting the value for  $N$  gives the probability that no meteoroid of mass  $M$  or larger will impact on the surface of area  $A$  during time  $\tau$  of

$$\ln P_0 = A\tau \propto M_p^{-\beta}$$

or

$$M_p = \frac{\alpha A \tau}{-\ln P_0}^{1/\beta} \quad (8.5)$$

To account for the fact that all meteoroids do not strike the protection layer normally, the meteoroid velocity  $\bar{V}$  may be replaced by a critical velocity  $\bar{V}_c$  where

$$\bar{V}_c = \bar{V}(\cos \lambda)^n \quad (8.6)$$

$\lambda$  angle between the direction of  $\bar{V}$  and the normal to the protection surface

n an experimentally determined constant

If n is selected to be unity the damage to the protection layer caused by an oblique collision is based on the meteoroid's normal component of velocity. A more conservative approach would be to set  $n = 0$  in which case all particles are considered to impact normally.

If the meteoroid flux is assumed to be isotropic the angle dependence may be replaced by

$$(\cos \lambda)^n = \left( \frac{2}{3n\theta\beta + 2} \right)^{1/3\beta} \quad (8.7)$$

Finally account must be taken for the fact that the meteoroid will not impact on an infinite target, but one with a finite thickness. As a result even though the meteoroid may not penetrate the protection layer, spalling may damage the radiator panel. To account for spalling the thickness of meteoroid protection  $t$  used should be larger than the predicted penetration into an infinite target or

$$t = aP_{\infty} \quad (8.8)$$

Accepted values of  $a$  lie between 1.5 and 2.0

Equations 8.1 through 8.7 may now be substituted into Eq. 8.8 to give an expression for the meteoroid protection layer thickness. The result is

$$t = ay \left( \frac{6}{\pi} \right)^{1/3} \left( \frac{\alpha A_T}{-\ln P_0} \right)^{1/3\beta} \rho_P^{-1/3} \left( \frac{\rho_P}{\rho_t} \right)^{\phi} \left( \frac{V}{12(E_t g_c / \rho_t)^{1/2}} \right)^{\theta} \left( \frac{2}{3n\theta\beta + 2} \right)^{1/3\beta} \quad (8.9)$$

where

- $t$  - thickness of protection layer (inches)
- $a$  - experimental constant (dimensionless)
- $\gamma$  - experimental constant (dimensionless)
- $\alpha$  - experimental constant that relates meteoroid flux to mass  
( $\text{gm}/(\text{day ft}^2)$ )
- $\beta$  - experimental constant that relates meteoroid flux to mass  
(dimensionless)
- $A$  - vulnerable area requiring protection ( $\text{ft}^2$ )
- $\tau$  - mission time (days)
- $P_o$  - probability of no damage caused by impact of meteoroid  
(dimensionless)
- $\rho_p$  - density of meteoroid particle ( $\text{gm}/\text{cm}^3$ )
- $\rho_t$  - density of protection layer ( $\text{lb}_m/\text{ft}^3$ )
- $\bar{V}$  - velocity of meteoroid ( $\text{ft}/\text{sec}$ )
- $E_t$  - modulus of elasticity of protection material ( $\text{lb}_f/\text{in}^2$ )
- $g$  -  $32.174 \text{ lb}_m \text{ ft}/\text{lb}_f \text{ sec}^2$
- $\theta$  - experimental constant (dimensionless)
- $\phi$  - experimental constant (dimensionless)
- $n$  - experimental constant that describes penetration depth as a  
function of angle of incident (dimensionless)

#### Selection of Values for Experimental Constants

Values for the experimental constant  $\rho$ ,  $\beta$ ,  $\rho_p$  and  $\bar{V}$  used in Eq. 8.9 were selected from the Manned Spacecraft Center publication for meteoroid

environment criterion (Ref. 18) .

The values for  $\alpha$  and  $\beta$  for meteoroids having a mass between 1 gm  $10^{-6}$  gm used in Eq. 8.4 are

$$\alpha = 1.888 \times 10^{-10} \quad \text{gm}^\beta / (\text{ft}^2 \text{day}) \quad (8.10)$$

$$\beta = 1.213 \quad (8.11)$$

The average meteoroid density is

$$\rho_p = 0.5 \text{ gm/cm}^3 \quad (8.12)$$

and the average meteoroid velocity is

$$\bar{V} = 20 \text{ km/sec.} \quad (8.13)$$

Values chosen for the remaining constants appearing in Eq. 8.9 are summarized in Table 3. Values of these constants are also listed in the table that will yield optimistic (minimum) and pessimistic (maximum) thicknesses for the meteoroid protection layer.

	Recommended Value	Pessimistic Value	Optimistic Value
a	1.75	2.0	1.5
$\gamma$	1.50	2.5	1.5
$\phi$	1/2	1/3	2/3
$\theta$	2/3	2/3	1/3
n	1.0	0	1.0

TABLE 3. Empirical Constants for Meteoroid Protection Layer Thickness

The following is an analysis of the sensitivity that the meteoroid protection layer thickness has to the uncertainty in the values of the five parameters listed in Table 3. This information will enable the user to judge his selection of these constants from within the recommended ranges.

An expression for the error in the meteoroid protection thickness may be obtained by taking the logarithm of Eq. 8.9 followed by differentiating the resulting equation. This process yields the equation

$$\frac{dt}{t} = \frac{da}{a} + \frac{d\gamma}{\gamma} + \phi \left( \frac{\rho_p}{\rho_t} \right) \frac{d\phi}{\phi} + \left[ \theta \ln \left( \frac{v}{c} \right) - \frac{n\theta}{3n\theta\beta+2} \right] \frac{d\theta}{\theta} - \left[ \frac{n\theta}{3n\theta\beta+2} \right] \frac{dn}{n} \quad (8.14)$$

If the symbol  $E_a$  is selected to represent the relative error in the meteoroid protection thickness resulting from a relative uncertainty in the value for the parameter  $a$  then it is evident from Eq. 8.14 that

$$E_a = \frac{dt/t}{da/a} = 1.0$$

when all other parameters are held constant. Similarly the error caused by an uncertainty in the value of  $\gamma$  will be

$$E_\gamma = \frac{dt/t}{d\gamma/\gamma} = 1.0$$

When the value for  $\theta$  is taken to be the recommended value of  $2/3$  and  $\beta$  is set equal to the value fixed by MSC's environmental model (Eq. 8.11), the resulting error in the meteoroid protection layer thickness due to an



uncertainty in the value for  $n$  is

$$E_n = \frac{dt/t}{dn/n} = - \left( \frac{n\theta}{3n\theta\beta+2} \right) = - 0.15$$

The magnitude of the error for the final two parameters  $\phi$  and  $\theta$  are a function of the material selected for the protection layer. In order to give an indication of the range of errors that can be expected for various protection materials, the errors were calculated for the three structural materials that were selected in the program: aluminum, beryllium and copper. If the meteoroid particle density is assumed to be fixed at the value recommended by the MSC environmental model (Eq. 8.12), then an uncertainty in the value of  $\phi$  from the recommended value of  $1/2$  would cause an error in the protection thickness of

$$E_\phi = \frac{dt/t}{d\phi/\phi} = \phi \ln \left( \frac{\rho_p}{\rho_t} \right)$$

which for each of the three structural materials results in the following errors

$E_\phi = - 0.85$	aluminum
$E_\phi = - 0.65$	beryllium
$E_\phi = - 1.44$	copper

The high density and low modulus of elasticity of copper makes its protection characteristics rather undesirable. For this reason the error of 1.44 indicated for copper probably will never be experienced in practice, and this value should be considered as a limiting case.

An uncertainty in the value of  $\theta$  from the recommended value of  $2/3$  would cause an error in the protection layer thickness equal to

$$E_{\theta} = \frac{dt/t}{d\theta/\theta} = \left[ \theta \ln \left( \frac{\bar{V}}{c} \right) - \frac{n\theta}{3n\theta\beta+2} \right]$$

which for each of the three structural materials is

$$\begin{aligned} E_{\theta} &= 0.77 && \text{aluminum} \\ E_{\theta} &= 0.20 && \text{beryllium} \\ E_{\theta} &= 1.01 && \text{copper} \end{aligned}$$

The errors calculated in the analysis are summarized in Table 4

TABLE 4              Ratio of Relative Error in Thickness  
To Relative Uncertainty in Empirical Constants

Protection Material	Aluminum	Beryllium	Copper
Parameter			
a	1.0	1.0	1.0
$\gamma$	1.0	1.0	1.0
$\phi$	- 0.85	- 0.65	- 1.44
$\theta$	0.77	0.20	1.01
n	- 0.15	- 0.15	- 0.15

If the error values for copper are excluded, the protection thickness is most sensitive to variations in the parameters a and  $\gamma$  and least sensitive to variations in the parameter n. Even though the protection thickness is least sensitive to the selection of n, it should be noted that the 100% variation between the optimistic and pessimistic value of n is the largest of all of the parameters. Also it should be noted that the signs on the values in Table 4 indicate that an increase in the parameters a,  $\gamma$  and  $\theta$  result in

an increase in the protection layer thickness, while an increase in the values for  $\phi$  and  $n$  result in a decrease in the protection layer thickness. This fact can be verified by the choice of the values of each of the parameters listed in Table 3. The values labeled as those which will produce a pessimistic value for the protection thickness are maximum values for  $a$ ,  $\gamma$  and  $\theta$  and minimum values for  $\phi$  and  $n$ .

To further evaluate the effect the meteoroid protection thickness has on the performance of the fin system, the temperature of the coolant fluid at the exit plane of the flow channel was evaluated first under the "pessimistic" conditions for the meteoroid protection layer, second for the "recommended" conditions and finally for the "optimistic" conditions. The results of these computer runs are shown below.

Case	Protection Layer Thickness-inches	Normalized Outlet Fluid Temperature $T/T_o$
Pessimistic	0.377	0.8855
Recommended	0.063	0.8922
Optimistic	0.020	0.8932

Even though the thickness of the meteoroid protection layer varies by nearly a factor of 20, the resulting error in the enthalpy drop is only

$$\frac{0.8932 - 0.8855}{1 - 0.8922} \cdot 100\% = 7.15\%$$

## 9. The Mass of the System

The system mass is computed, firstly as a convenience for the user and secondly for the purpose of the planned system optimization. The system mass includes

(i) the mass of the fluid in all tubes

$$M_c = n_t \frac{d^2 \pi}{4} \int_0^L \rho_c dz = n_t \rho_{c,o} \frac{d^2 \pi}{4} L \int_0^1 v d\zeta \quad (9.1)$$

(ii) the mass of the fins

$$M_f = n_t H L \rho_f (s_r + s_t) \quad (9.2)$$

(iii) the mass of all tubes

$$M_w = n_t s_w L \cdot \pi (d + s_w) \rho_w \quad (9.3)$$

and

(iv) the mass of the protection layer

$$M_m = n_t s_m L \rho_m [\pi(d + s_w + s_m) - s_r] \quad (9.4)$$

but it does not include the thermal coating nor the mass of the manifold and the fluid in the manifold. In Eqs. 9.1 through 4 represent

$n_t$	the number of tubes
$d$	the tube diameter
$\rho$	the density
$L$	the tube length
$H$	the fin height, distance between fin root and fin tip
$s$	the thickness

while the subscripts designate  $\rho$  and  $s$  as follows

c	coolant fluid
f	fin
m	meteoroid protection layer
r	fin root
t	fin tip
w	tube wall
o	inlet condition.

The integral in Eq. 9.1 is time-dependent and evaluated at the initial conditions.

## 10. Nondimensional System Parameters

The governing equations in the preceding radiator system analysis are developed in non-dimensional form for the purpose of (i) reducing the number of parameters, (ii) evolving the set of relevant parameters, and (iii) presenting the results in a general form which is applicable to groups of systems rather than an individual system. Although these tasks are planned for the second contract phase a summary of parameters is presented here, first for the detailed analysis discussed in the preceding chapters and then for the simplified analysis presented in Appendix C.

The transient flow field in the coolant channel and the temperature field over the fin can be represented as functions of:

### a) the independent variables

time	$\tau = \frac{tw_o}{L}$	(10.1)
------	-------------------------	--------

axial coordinate	$\zeta = \frac{z}{L}$	(10.2)
------------------	-----------------------	--------

radial coordinate	$\eta = \frac{2r}{d}$	(10.3)
-------------------	-----------------------	--------

transverse coordinate

normal to the channel

axis	$\xi = \frac{x}{H}$
------	---------------------

b) the dependent system variables

$$\text{fin temperature} \quad \theta_f (\xi, \zeta; \tau) = \frac{T_f}{T_o} \quad (10.4)$$

$$\text{channel temperature} \quad \theta_w (\eta, \zeta; \tau) = \frac{T_w}{T_o} \quad (10.5)$$

meteroid protection

$$\text{layer temperature} \quad \theta_m (\eta, \zeta; \tau) = \frac{T_m}{T_o} \quad (10.6)$$

$$\text{coolant fluid temperature} \quad \theta_c (\zeta; \tau) = \frac{T_c}{T_o} \quad (10.7)$$

$$\text{coolant fluid pressure} \quad \pi(\zeta, \tau) = \frac{p}{p_o} \quad (10.8)$$

$$\text{coolant fluid velocity} \quad \omega(\zeta; \tau) = \frac{w}{w_o} \quad (10.9)$$

The solution to the problem will depend on the geometry of the system, the material properties and the definition of operational conditions. There was no attempt made to establish similitude with respect to the material properties because the scaling laws would either be too restrictive to allow for general property variations or too complex (for instance, the concept of corresponding states for gases). Consequently, the  $\phi$  - parameters defined in Eqs. 3.27 through 3.30, in Eq. 4.4 and in Eqs. 5.1, 2 and 6 are omitted from this summary; they constitute temperature and pressure variation of properties. This leaves the following list of parameters, in addition to  $n$ , the number of tubes:

c) the geometric parameters

fin height-to-length ratio  $\bar{H} = \frac{H}{L}$  (10.10)

fin profile slope  $c = \frac{s_r - s_t}{2H}$  (10.11)

fin root thickness  $\bar{s}_r = \frac{s_r}{H}$  (10.12)

tube diameter-to-length ratio  $\delta = \frac{d}{L}$  (10.13)

channel wall thickness  $\bar{s}_w = \frac{2s_w}{d}$  (10.14)

protection layer thickness  $\bar{s}_m = \frac{2s_m}{d}$  (10.15)

d) the operational parameters

coolant flow Reynolds number  $N_{Re} = \frac{dw_o}{\nu_o}$  (10.16)

Prandtl number

(representing coolant selection)  $N_{Pr} = \left( \frac{\mu_c}{k_c} \right)_o$  (10.17)

inlet pressure heat  $F_p = \left( \frac{p}{\rho w} \right)_o$  (10.18)

compressibility  $\bar{Q}_o = \frac{\pi n \rho d^2 c_p}{4 \sigma T_o^3}$  (10.19)

inlet coolant power flux  $F_z = \left( \frac{p}{\rho c_p T} \right)_o$  (10.20)



where  $n$  is the number of tubes

incident radiant heat flux

$$\bar{Q}_{\text{rad}} = \frac{\alpha_o q''}{\epsilon_o \sigma T_o^4} \quad (10.21)$$

meteoroid velocity

$$M_m = \frac{v_m}{c_m} \quad (10.22)$$

protection layer density

(representing selection of

protection layer material)

$$\phi_\rho = \frac{\rho_{mt}}{\rho_m} \quad (10.23)$$

where  $\rho_{mt}$  is the density of the meteoroids.

Similarity for ascent and reentry operations is difficult to establish unless one restricts oneself to similar velocity-altitude profiles which can be represented by the

max. Mach number

$$M_{\text{max}} = (v/c)_{\text{max}} \quad (10.24)$$

and its corresponding (through same altitude)

Reynolds number

$$(N_{\text{Re}})_\infty = \frac{\rho_\infty v L}{\mu_\infty} \quad (10.25)$$

Prandtl number

$$(N_{\text{Pr}})_\infty = \frac{\mu_\infty c_{p\infty}}{k_\infty} \quad (10.26)$$

Grashof number

(before launch)

$$N_{\text{Gr}} = \frac{g L^3 \beta \Delta T}{\nu^2}$$

This completes the list of non-dimensional groups resulting from the detailed analysis. The simplified, steady-state analysis involves only ten parameters which govern the solution. The reader is referred to Appendix D for details. There are two operational parameters,

(i) the characteristic length

$$L_o = \dot{M} c_p / (\epsilon \sigma T_o^3) \quad (10.27)$$

where  $\dot{M} = n \rho_c w_o \frac{d^2 \pi}{4}$  stands for the total mass flux.

(ii)

$$\Phi = \epsilon \sigma T_o^3 \dot{M} c_p / k_c^2 \quad (10.28)$$

and five geometrical parameters, namely  $L^*$ ,  $H^*$ ,  $d^*$  and  $n$ , where all starred quantities are obtained by dividing the unstarred quantities by  $L_o$  of Eq. 10.27. The fluid property is represented by two groups, the Prandtl number and the ratio  $k_f/k_c$  of thermal conductivities.

Finally, the system mass is defined through the above nine parameters and the mass augmentation factor  $F$ , that is the mass of tube, fluid and protection layer divided by the fin panel mass

$$F = \pi \left[ \frac{\rho_w}{\rho_f} \frac{d_m}{d} \frac{s_w}{d} \left( 1 + \frac{\rho_m}{\rho_f} \frac{A_m}{A_w} \right) + \frac{\rho_c}{4\rho_f} \right] \quad (10.29)$$

where  $d_m = d + S_w$

$A$  cross-section, with subscripts as defined before.

### C. Properties

The fundamental principles used to prepare the required thermophysical properties for inclusion into the computer code are exhibited in this chapter while the specific details concerning the materials treated in this program phase are placed into the appendices.

The principles involved are those of macroscopic thermodynamics treated in most elementary texts. The approach of deriving analytic expressions for the required properties is not unique because the starting point is dictated by the availability of experimental data. The result, however, must be of the same form regardless of whether, for instance, the coolant fluid is gaseous or liquid.

The properties of the structural materials are the least problematic ones since they depend on the temperature at most; and the standard polynomial collocation methods are entirely sufficient. Care must be exercised, however, that the collocation imply continuous fourth derivatives for highest integration efficiency or, less desirable, at least continuous representation of the property itself which may exclude piecewise allocation of degrees higher than one.

What is said about the properties of structural materials holds in principle for the description of the atmosphere whose properties depend only on altitude. Even though the optical properties of the thermal coating depend, in general, on wave-length and temperature, the spectral dependence is integrated into the averaged ("gray") properties (see Eqs. 6.5 and 6), and the results are functions of one variable, the temperature. Consequently, there remains but the discussion of the coolant fluid properties which

depend strictly on two state variables.

In macroscopic thermodynamics there are required two equations of state for the description of a substance, namely the thermal equation of state  $f(\rho, p, T) = 0$  which relates any one of density  $\rho$ , pressure  $p$  and temperature  $T$  to the remaining two, and the caloric equation of state, perhaps in the form of  $c_v^0 = f(T)$  where  $c_v^0$  is the zero-pressure specific heat at constant volume. These two equations are sufficient to develop all of the required thermodynamic functions, namely:

- (i) specific heat at constant pressure  $c_p(\rho, T)$
- (ii) isobaric thermal expansion coefficient  $\beta(\rho, T)$
- (iii) isothermal bulk modulus  $\kappa(p, T)$
- (iv) enthalpy  $h(\rho, T)$

These functions are discussed in Section 11.

The transport properties, namely the thermal conductivity  $k(\rho, T)$  and the dynamic viscosity  $\mu(\rho, T)$  are correlated on the basis of residuals as explained in Section 12. The properties of the atmosphere are dealt with in Section 13.

## 11. Thermodynamic Properties

The first step in developing thermodynamic properties is to secure a thermal equation of state

$$f(p, \rho, T) = 0 \quad (11.1)$$

For almost all pure gases and air, this equation can be found in the literature, either in the form suggested by Benedict-Webb-Rubin (virial expansion) or in that suggested by Beattie-Bridgeman. Both equations are explicit in  $p$ ,

$$p = p(\rho, T) \quad (11.2)$$

so that  $\kappa$  is immediately obtained from Eq. 11.2 by implicit differentiation

$$\kappa(\rho, T) = \frac{1}{\rho} \left( \frac{\partial \rho}{\partial p} \right)_T \quad (11.3)$$

which can be evaluated as  $\kappa(p, T)$  after inversion of Eq. (11.2) into

$$\rho = \rho(p, T) \quad (11.4)$$

The inversion of Eq. 11.2 is facilitated by computing  $(\partial p / \partial \rho)_T$  from Eq. 11.2 and subsequently applying the Newton-Raphson method along the specified isotherm with temperature  $T$  in Eq. 11.4.

The isobaric thermal expansion coefficient

$$\beta = - \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p$$

is also obtained by implicit differentiation of Eq. 11.2 while keeping the left-hand side constant.

After having collocated the zero-pressure specific heat at constant volume that is  $c_v^0(T)$  by a power polynomial in  $T$ , one obtains first the specific heat at constant volume

$$c_v(\rho, T) = c_v^0(T) - T \int_0^p \left( \frac{\partial^2 p}{\partial T^2} \right)_\rho \frac{d\rho'}{(\rho')^2} \quad (11.5)$$

and then the specific heat at constant pressure

$$c_p(\rho, T) = c_v(\rho, T) + \frac{T\beta^2}{\rho\kappa} \quad (11.6)$$

The derivative in Eq. 11.5 is obtained from Eq. 11.2; and  $\beta$  and  $\kappa$  are both function of  $\rho$  or  $p$  and  $T$ .

Finally the enthalpy  $h$  is calculated from its definition

$$h(\rho, T) = \frac{p}{\rho} + u(\rho, T) \quad (11.7)$$

where the internal energy  $u$  may be obtained by two successive integrations, the first one along an isotherm (over  $\rho$ ), and the second one along an isochore (over  $T$ ):

$$u = u(\rho_0, T_0) + \int_{T_0}^T \int_{\rho_0}^{\rho} c_v(\rho', T') d\rho' dT' + \int_{\rho_0}^{\rho} \left[ p - \frac{T\beta}{\kappa} \right] \frac{d\rho'}{(\rho')^2} \quad (11.8)$$

Liquids can be treated, in principle, as gases; except that the equation of state, Eq. 11.1, is rarely available. One may find, with little difficulty the zero-pressure isobaric expansion coefficient  $\beta_0 = c(T)$ , and then represent adequately the isothermal bulk modulus by

$$\kappa(p, T) = a(T) + b(T) p \quad (11.9)$$

Under any circumstances, one must satisfy

$$\left( \frac{\partial \beta}{\partial p} \right)_T + \left( \frac{\partial \kappa}{\partial T} \right)_p = 0 \quad (11.10)$$

which yields, from Eq. 11.9

$$\beta(p, T) = -a'p - \frac{b'}{2} p^2 + c \quad (11.11)$$

where primes indicate differentiation with respect to  $T$ , of the polynomials  $a(T)$  and  $b(T)$  in Eq. 11.9. Equations 11.9 and 11 yield for the density

$$\rho(p, T) = \rho(0, T_0) e^{[a(T)p + \frac{1}{2} b(T)p^2 - \int_{T_0}^T c(T') dT']} \quad (11.12)$$

The specific heats and the enthalpy are to be derived as for gases (see Eqs. 11.6 and 7). Other possibilities are to develop  $\kappa$  from the speed of sound and the ratio of specific heats; but the reader is warned not to imply  $\kappa = 0$  or  $c_p = c_v$ , unless there is sufficient evidence to support these assumptions.

## 12. Transport Properties

While the thermal conductivity  $k$  and the dynamic viscosity  $\mu$  of liquids may often times be adequately represented by functions of temperature  $T$  alone (facilitated by polynomial collocation), these same properties for gases depend on density as well. It is recognized that the difference, or residue

$$\psi_1(\rho) = k(\rho, T) - k^*(T) \quad (12.1)$$

between the thermal conductivity  $k(\rho, T)$  and the low-pressure thermal conductivity  $k^*(T)$  depends only on the density. Similarly, for the dynamic viscosity

$$\psi_2(\rho) = \mu(\rho, T) - \mu^*(T) . \quad (12.2)$$

Hence  $k$  and  $\mu$  can be represented by the sum of two polynomials, one in  $\rho$  and the other in  $T$ . The residuals  $\psi_1$  and  $\psi_2$  are published for a number of gases or may be developed from property data (Ref. 19 for  $N_2$  and He)



### 13. Atmospheric Properties

For the prediction of aerodynamic heat fluxes incident on the radiator system during ascent and reentry (see Sect. 7) the evaluation of the following atmospheric properties are required:

Temperature

Pressure

Density

Molecular Weight

Speed of Sound

Viscosity

Thermal Conductivity

Specific Heat at Constant Pressure

Enthalpy

The model for these atmospheric properties is presented in two sections. The first covers altitudes from sea-level to 301,000 feet. Within this range the molecular weight is assumed to be constant and the temperature variation with altitude is a sequence of connected line segments. The second section of the model covers altitudes above 301,000 feet where the molecular weight decreases linearly with altitude. For this altitude range the approximate polynomial expressions for density and pressure suggested in Part 4 of Ref. 20 were used. Errors between the values given by the approximate expression and the 1962 Standard Model are less than 5% over the entire range of altitudes.

Atmospheric air is assumed to be an ideal gas for all altitudes. Therefore compressibility effects at low altitudes are neglected. The error in computed densities resulting from the ideal gas assumption may be

as high as 0.05% for altitudes below 6 miles, but becomes less than 0.01% above 12 miles (Ref. 20). The air is also assumed to be in hydrostatic equilibrium.

All properties except geopotential altitude, specific heat and enthalpy are evaluated from expressions presented in Refs. 20 and 21. The expression for geopotential altitude was taken from Ref. 22, while specific heat and enthalpy data were taken from Ref. 23.

The model developed for the atmospheric properties is considered to be applied to altitudes up to 100 miles and to latitudes between 30 and 60°N. It is anticipated that atmospheric properties are not needed for altitudes exceeding 100 miles, because the convective heat flux from the fin system will be negligible at this altitude and above.

The properties for the earth's atmosphere are known with increased uncertainty as the altitude increases. In fact the 1962 Standard Atmosphere (Ref. 21) consists of four regions as follows:

0 - 20 km	Standard
20 - 32 km	Proposed Standard
32 - 90 km	Tentative
90 - 700 km	Speculative

Any uncertainty in the atmospheric properties will naturally be reflected as an error in the convective heat flux from the shuttle vehicle. Fortunately during the ascent phase of the shuttle operation the convective flux from the radiator system is fairly small compared to the radiative flux by the time the shuttle has approached altitudes for which the atmospheric properties are considered to be "speculative"; on the other hand during re-entry, significant convective fluxes are known to exist at altitudes above 90km. As a result every effort should be made to revise the existing

atmospheric property model at high altitudes as new data become available.

The atmospheric model is based on several primary constants. The sea-level pressure, temperature, molecular weight, density and acceleration of gravity and the universal gas constant were assigned the fixed values of

$$P_o = 2116.22 \text{ lbf/ft}^2$$

$$T_o = 518.67 \text{ R}$$

$$M_o = 28.9644$$

$$\rho_o = 0.07647 \text{ lbm/ft}^3$$

$$g_o = 32.1741 \text{ ft/sec}^2$$

$$R^* = 1545.31 \text{ ft lbf/lb mole R}$$

### 13.1 Properties for Altitudes Less than 301,000 feet.

#### a. Geopotential Altitude - H

The state variables for air are expressed in terms of the single variable, the geopotential altitude

$$H = \int_0^Z \frac{g(z)}{g_0} dz =$$

$$Z - 1.573126 \times 10^{-7} Z^2 + 2.4656553 \times 10^{-14} Z^3$$

$$- 3.8667054 \times 10^{-21} Z^4 + 6.0621354 \times 10^{-28} Z^5$$

$$- 9.5013649 \times 10^{-35} Z^6 \quad (13.1)$$

where Z is the geometric altitude in meters,  $g_0$  is the acceleration of gravity at sea-level and  $g(Z)$  denotes the local acceleration of gravity.

See Ref. 22 for details.

#### b. Temperature - T

The general expression of the temperature as a function of geopotential altitude is

$$T = T_b + L(H - H_b) \quad (13.2)$$

$T_b$  and  $H_b$  are the endpoints of straight-line segments representing  $T(H)$  and are listed, together with  $L(H)$  in the following table.

$H_b$ (km)	L (K/km)	$T_b$ (K)
0		288.15
	- 6.5	
11		216.65
	0.0	
20		216.65
	1.0	
32		228.65
	2.8	
47		270.65
	0.0	
52		270.65
	-2.0	
61		252.65
	-4.0	
79		180.65
	0.0	
90		180.65

TABLE 5. Lapse Rate and Base Temperatures for  
Atmospheric Model

## c. Molecular Weight - M

The molecular weight is constant at a value of 28.9644.

## d. Pressure - P

Within a region where the temperature varies linearly, the ideal equation of state and the hydrostatic yield the following expressions for pressure:

$$\frac{P}{P_b} = \left[ \frac{T_b}{T_b + L(H - H_b)} \right]^{\frac{g_o M_o}{R^* L}} \quad (L \neq 0) \quad (13.3)$$

$$\frac{P}{P_b} = \exp \left[ - \frac{g_o M_o (H - H_b)}{R^* T_b} \right] \quad (L = 0) \quad (13.4)$$

$$\begin{aligned} \text{i.e } 11 &\leq H \leq 20 \text{ km} \\ 47 &\leq H \leq 52 \text{ km} \\ 79 &\leq H \leq 90 \text{ km} \end{aligned}$$

The subscripts "o" denotes a quantity evaluated at sea-level and the subscript "b" denotes a quantity evaluated at the base of one of the straight line segments of the atmospheric model.

c. Density -  $\rho$ 

The density may be calculated from the ideal equation of state once the temperature and pressure have been evaluated.

$$\rho = \frac{MP}{R^* T} \quad (13.5)$$

f. Speed of Sound -  $c$ 

The speed of sound was evaluated from the expression

$$c = \left[ \gamma \frac{R^*T}{M} \right]^{1/2} \quad (13.6)$$

For altitudes less than 301,000 feet the ratio of specific heats is taken to have a fixed value of 1.40

g. Viscosity -  $\mu$ 

The dynamic viscosity was evaluated from the expression

$$\mu = \frac{\beta T^{3/2}}{T + S} \quad (13.7)$$

where

$$\beta = 1.458 \times 10^{-6} \frac{\text{kg}}{\text{sec m(K)}^{1/2}}$$

and

$$S = 110.4 \text{ K}$$

h. Specific Heat at Constant Pressure  $c_p$  and Enthalpy  $i$ 

Values for  $c_p$  and  $i$  between the temperatures of 100 R and 6400 R were taken from the standard Gas Tables (Ref. 23) and placed in the program in tabular form. A value of  $c_p$  and  $i$  at any temperature intermediate to a pair of tabular values was determined by an interpolation routine (see Section 17).

### 13.2 Properties for Altitudes Greater than 301,000 Feet

#### a. Pressure - P

The pressure for altitudes between 301,000 and 528,000 feet is based on the polynomial approximation given in Part IV of Ref. 20. The pressure is written in terms of the sea-level pressure  $P_o$  in the form

$$P = P_o \left\{ \sum_{n=0}^{11} [A_n Z^n]^{-4} \right\} \quad (13.8)$$

where  $Z$  is the geometric altitude and values for  $A_n$  appear in Table 4.1 of Ref. 20.

#### b. Density - $\rho$

The density is written in terms of a similar polynomial

$$\rho = \rho_o \left\{ \sum_{n=0}^{11} [B_n Z^n]^{-4} \right\} \quad (13.9)$$

where values of  $B_n$  appear in Table 4.1 of Ref. 20.

#### c. Molecular Weight - M

The molecular weight is assumed to vary linearly with altitude  $Z$  (see Fig. 1.2.7 in Ref. 21). The resulting expression for  $M$  is

$$M = 28.9644 - 0.030949 (Z - 90)$$

where  $Z$  is in km.

#### d. Temperature - T

The temperature is calculated from the values for pressure, density and molecular weight indicated above from the ideal equation of state

$$T = \frac{PM}{\rho R^*}$$

#### e. Speed of Sound - c



For altitudes greater than 301,000 feet the equation for the speed of sound is the same one as used for the lower altitudes, but the ratio of the specific heats is no longer assumed to be equal to 1.40. The ratio of the specific heats varies with the molecular weight according to the expression

$$\gamma = \frac{c_p}{c_p - R^*/M}$$

The remaining properties are calculated using identical expressions to those outlined in the Section (13.1) for altitudes less than 301,000 feet.

### III. NUMERICAL TECHNIQUES

#### 14. Introduction

The analysis carried out in Chapter II lead, as far as the mathematical problem formulation is concerned, to three initial value problems and one matrix manipulation. The three initial-value problems are to establish

- (i) the initial conditions for the coolant fluid, defined by Eqs. 3.42 through 3.45,
- (ii) the dynamics of the coolant flow, defined by Eqs. 3.40 through 42 and 44,
- (iii) the temperature field throughout the system, defined by Eqs. 2.18 through 2.22 for the fin, Eqs. 3.39 and 46 for the coolant, Eqs. 4.6, 7 and 8 for the channel wall, Eqs. 4.7, 5.3 and 5 for the protection layer, or Eq. 4.11 replacing Eqs. 2.19, 4.6, 7 and 8 and 5.3 and 5 in the case where Eq. 4.9 is satisfied. These equations must be supplemented by the specification of the initial, non-dimensional temperature everywhere in the system.

Each initial-value problem is solved by a fourth-order Runge-Kutta-Simpson integration discussed in Section 15.

The radiosity equation, Eq. 6.17 requires the matrix manipulation, namely either the solution of a system of linear algebraic equations, or a matrix inversion whenever the optical properties of the thermal control coating are considered temperature independent. Either task is accomplished by elementary row operations which transform, in a single process, the augmented coefficient matrix into a row-reduced echelon matrix. The reader

is referred for this transformation to standard texts on linear algebra (Ref. 9)

Additional mathematical operations are programmed as subprograms which may be generally applied and which are discussed in Sections 16 through 21 in this order: an evaluation of polynomials in one variable, an Aitken interpolation, first and second differentiation, definite integration and integration with variable upper integration limit for functions of equally spaced arguments.

### 15. The Runge-Kutta-Simpson Integration

Two types of initial-value problems are to be solved in this program. The first type includes Item (i) and (ii) mentioned in the Introduction, namely the fluid dynamics exclusive of the transient fluid temperature field, and involves ordinary, first-order differential equations, linear in the derivatives with respect to the axial distance  $\zeta$ , that is Eqs. 3.42, 43 and 44. The equations are solved explicitly for these derivatives so as to take on the general form of Eq. 15.1:

$$\frac{dy_i}{dx} = f_i(x, y_1, y_2, \dots, y_n) \quad (15.1)$$

$$y_i(0) = a_i, \quad i = 1, 2, \dots, N \quad (15.2)$$

Equation 15.2 constitutes the appropriate initial conditions. The other type of initial-values problem, mentioned as item (iii) in the Introduction, involves partial differential equations which are linear and of the first order in the time-derivatives; moreover, all equations, Eqs. 2.18, 3.46, 4.8 and 5.3, are explicit in the time derivatives. Having subdivided the radiator system into intervals, equally spaced in each appropriate domain (fluid, wall, fin, etc.), and then written the different equations corresponding to each one of the resulting  $N$  interior nodal points, one may discretize the spatial derivatives occurring on the right-hand sides of the partial differential equations. The result is a set of ordinary differential equations, with time as the independent variable but of a form which is identical to Eq. 15.1. Equation 15.2 is given by the initial

temperature distribution; a uniform temperature was chosen for the first start of the integration ( $a_i = a$ ;  $i = 1, 2, \dots, N$ ), subsequent integrations during optimization runs are expected to start from the previously computed steady-state temperature distributions. The boundary conditions may be satisfied in three different ways. Either, the temperatures are computed directly from the finite-difference equation representing the boundary conditions at the end of every time step, or secondly, the boundary conditions may be included into the system of Eqs. 15.1 after differentiation with respect to time, or lastly, an equation of the form of Eqs. 15.1 may be derived directly from a control volume bounded at one side by the boundary of interest. We have utilized all three possibilities.

Discretization introduces obviously a truncation error; all spatial derivatives are represented consistently with a truncation error proportional to the square of the local spatial interval (see Sect. 18) but higher-order terms may be included anytime by modifying a single program unit each for the first and second derivatives.

The system of Eqs. 15.1 and 15.2 is solved by a fourth-order Runge-Kutta integration, that is if the  $f_i$  in Eqs. 15.1 have continuous fourth-order derivatives, the time-related accuracy of the integration is of order four (Refs. 26 and 27). Under much weaker conditions, namely uniform Lipschitz continuity (Ref. 26), the accuracy is still first-order and stability is secured. It may be noted that the Lipschitz continuity is also the prerequisite for uniqueness of the solution to Eqs. 15.1.

An existing single-precision, floating-point Runge-Kutta-Simpson SUBROUTINE RKS, written by R. Schubert at the Aerospace Corporation was

used. Its fixed-step integration mode was employed for the integration of the fluid flow variables along the channel axis, while the transient temperature field was integrated with variable time steps, chosen automatically so as to keep the "truncation error" per time step below a specified limit. The absolute and relative errors  $A_i$  and  $R_i$  are specified, by the user, for each variable  $y_i$ , and after every Runge-Kutta integration step a Simpson integration is carried out over the same interval and with the intermediate derivatives as used in the former integration. From the difference  $D_i$  between the two integrations is calculated the "truncation error" measure

$$E_m = \max E_i = \left| \frac{D_i}{A_i + R_i |y_i|} \right|, \quad i = 1, \dots, N \quad (15.3)$$

and if  $0.75 < E_m$  then the time step DEL is divided by  $\sqrt[5]{10}$  and the step is repeated, if  $0.075 < E_m \leq 0.75$  then DEL is multiplied by  $\sqrt[5]{10}$  for the subsequent step.

All variables  $y_i$  are set equal to their initial values in the program which calls RKS. During the integration RKS interacts directly with two other subroutines, namely DERIV and CNTRL, whose names are the first elements of the argument list in the call statement. The first subroutine, DERIV, serves to compute all  $N$  derivatives  $dy_i/dx$  in accordance with Eqs. 15.1. The second subroutine, CNTRL, controls the output during integration and the termination of integration. Output of current values of all variables along with important system parameters is provided under two different integration modes: general transient system simulation,

MSTOR = 0 in NAMELIST/RUNOPT/, produces output in arbitrarily chosen, fixed time steps, DTWRT, up to the final time TEND, both specified in NAMELIST/RUNOPT/ and in hours; the second mode serves to compute the steady-state conditions and is invoked by setting MSTOR = 1 and by specifying the number LIMWRT of time intervals DTWRT at which output is desired.

The integration under the second mode (MSTOR = 1) is terminated as soon as the expected truncation error due to program termination is less than five times the specified relative error per time step, RLIMIT, that is  $R_i$  in Eq. 15.3. The largest truncation error associated with the  $j$ -th time step is anticipated on the basis of Eqs. 1.1 and 2 as follows

$$\delta_j = \max_i \delta_{i,j} = \max_i \left\{ \Delta_j \tau \frac{\dot{y}_{i,j}}{\ln \frac{\dot{y}_{i,j} - 1}{\dot{y}_{i,j}}} \right\}, \quad i = 1, 2, \dots, N \quad (15.4)$$

The maximum is taken from all  $N$  modal points,  $\Delta_j \tau$  is the current integration step size with index  $j$ , and  $\dot{y}_i$  stands for the  $dy_i/dx$  in Eqs. 15.1.

The argument list of RKS (and RKSF) is as follows:

- |       |   |  |
|-------|---|--|
| (i)   | DERIV, name of derivative subroutine                    | } declared as EXTERNAL<br>in calling program |
| (ii)  | CNTRL, name of control subroutine                       |  |
| (iii) | Y, array name*, containing the $y_i$ 's in Eqs. 15.1 ** |  |
| (iv)  | DY, array name*, containing the $dy_i/dx$ in Eqs. 15.1  |  |
| (v)   | A, array name*, containing the $A_i$ 's in Eq. 15.3**   |  |
| (vi)  | R, array name*, containing the $R_i$ 's in Eq. 15.3**   |  |
| (vii) | T, the independent variable X in Eqs. 15.1**            |  |

\*

declared in calling program as array with dimension size equal to the number of differential equations.

\*\*

to be specified prior to the calling statement.

- (viii) DEL, the integration step\*\*, DEL  $\neq$  0
- (ix) N , (integer) the number of equations\*\*
- (x) IFVD = 0: variable step size \*\*, see Eq. 15.3  
       = 1: fixed step size equal to DEL
- (xi) IBKP = 0: adjust step size at most once before repeat,\*\*  
       = 1: adjust in accordance to Eq. 15.3
- (xii) NTRY = 1: continue integration\*\*, normal start  
       = 2: return from RKS  
       = 3: repeat last step with new DEL  
       = 4: restart  
       } to be changed in CNTRL
- (xiii) IERR = 0, normal integration  
       = -1 indicates singularity when IFVD = 0  
       = +1 indicates denominator vanishes in Eq. 15.3 at  
           some time during integration.
- (xiv) through (xx) are array names\* with which the user need not to  
       be concerned except YS that contains the y 's in Eq. 15.1 at  
       the previous times step: DELY, PD, SD, YS, YST, DYST, YSIMP.

The SUBROUTINE DERIV communicates with RKS only via its argument list which contains, in this order, Y, DY, and T, as specified above under iii, iv, and vii. Here, the current values of Y and T are supplied to DERIV, and the corresponding values of DY returned by DERIV to RKS.

---

\* declared in calling program as array with dimension size equal to the number of differential equations.

\*\* to be specified prior to the calling statement.



The SUBROUTINE CNTRL communicates with RKS also via its argument list. It contains Y, DY, DEL, T, NTRY, IFVD as specified above under iii, iv, viii, vii, xii and x, respectively. From the array Y are available for output all the results of integration. The time step may be modified to reach a specific time value; and by specifying NTRY one controls the integration process from within CNTRL during the integration. Finally, one may switch from variable to fixed step size during the integration by resetting IFVD.

This completes the discussion of the integration of both ordinary and partial differential equations as they occur in the analysis developed in Chapter II. The discussion is deemed sufficient to enable the user to apply the RKS routine to other problems as well.

## 16. The Evaluation of Polynomials

All polynomials

$$z = a_0 + a_1x + a_2x^2 + \dots + a_Nx^N \quad (16.1)$$

are carried out in a function subprogram based on the simple, efficient recurrence relation

$$\begin{aligned} z_0 &= a_N \\ z_{i+1} &= x \cdot z_i + a_{N-i-1} \\ i &= 0, 1, \dots, N-1 \\ z &= z_N \end{aligned} \quad (16.2)$$

The coefficients  $a$ ,  $i = 0, 1, \dots, N$  must be placed, in the calling program, into an array of dimension  $(N + 1)$ ,  $N$  is an arbitrary positive integer.

The procedure is coded as a function subprogram called `POLY(X,A,M)`, where  $X$  is the argument  $x$  in Eq. 16.1,  $A$  is the array containing  $M = N + 1$  elements starting with  $A(1) = a$ .

### 17. Aitken Interpolation

Experimental data and supporting computer results which are not represented by analytic expressions are interpolated by Aitkens interpolation technique (Ref.27). An  $(n + 1)$ -point Lagrangian interpolation is reduced to a sequence of  $1/2 n (n + 1)$  linear interpolations. The interval spacing is arbitrary; and any number  $M \geq n$  of ordered pairs  $(x_i, y_i)$  can be supplied in the calling program. The  $n$  points of interpolation are spaced equally about the point  $x$  of interpolation. It should be noted, however that unless  $n = N$  or  $n = 2$  the result  $y(x)$  is not continuous in general. Care must also be taken that all nodes  $x_1, x_2, \dots$  are distinct.

The procedure is coded as a function subprogram called YINT(X,Y,M,N,P), where X and Y are the names of arrays that have the same dimension M and contain the ordered pairs  $(x_i, y_i)$ ,  $i = 1, 2, \dots, M$  such that  $x_1 < x_2 < \dots < x_M$ . The number  $n$  of points used for the interpolation is specified as N, and the value of  $x$  at which to interpolate is supplied as P. Note that  $2 \leq N \leq M$  must be satisfied.

## 18. Numerical Differentiation

The first and second derivative of tabulated functions of equally spaced arguments is carried out in SUBROUTINE DDY(Y,DY,DX,N) and in SUBROUTINE D2DX2(Y,D2Y,DX,N), respectively. Each subroutine requires that two arrays be declared in a DIMENSION statement in the calling program, to have at least N elements; one array for the set of ordinates  $Y \rightarrow y_i$  supplied by the calling program, the other array for the return of the results, that is  $DY \rightarrow dy_i/dx$  or  $D2Y \rightarrow d^2y_i/dx^2$ . The argument interval  $\Delta x$  and the number of ordinates  $y_i$  are to be specified as DX and N, respectively. However, in order that terms of order  $\Delta x$  be retained including at the endpoints of the domain, N must be no less than 3 for DDX and 4 for D2DX2. The truncation error is of order  $y'''(\Delta x)^2$  and  $y^{IV}(\Delta x)^2$ , respectively, for DDX and D2DX2.

19. Numerical Integration

The definite integral

$$F = \int_{x_1}^{x_N} y(x_i) dx, \quad 1 \leq i \leq N; \quad N \geq 2$$

and the indefinite integral

$$G_j = \int_{x_1}^{x_j} y(x_i) dx + G(x_1)$$

$$1 \leq i \leq N; \quad 1 < j \leq N; \quad N > 3$$

of a tabulated function  $y_i$  of an equally spaced argument,  $x_i$ ,

$x_1 + \Delta x, x_1 + 2 \Delta x, \dots, x_1 + (N - 1) \Delta x$  is carried out by a modified Simpson integration in the `FUNCTION DEFINT(Y,DX,N)` Subprogram and in the `SUBROUTINE FINT(Y,YO,DX,N,F)`, respectively.

For `DEFINT` the ordinates  $y_i$  are to be placed in the array `Y` whose dimension of no less than  $N$  elements must be declared in the calling program. The argument interval and the number of ordinates are specified as `DX` and `N`, respectively.

For `FINT` there are two array declarations necessary in the calling program, both for at least  $N$  elements; one for the integrand  $Y \rightarrow y_i$  and the other for the integral  $F \rightarrow G_i$ . The integration constant  $G(x_1)$ , the argument interval and the number of ordinates are to be supplied as `YO,DX` and `N`, respectively.

The truncation error of composite Simpson integration is

$$\frac{x_1 - x_N}{180} (\Delta x)^4 y^{(iv)}(\xi) \quad \text{with} \quad x_1 \leq (\xi) \leq x_N.$$

APPENDIX A  
STRUCTURAL MATERIAL PROPERTIES

Appendix A contains thermodynamic and mechanical properties for the three structural materials: copper, aluminum and beryllium. Four properties are evaluated for each material: specific heat at constant pressure, thermal conductivity, modulus of elasticity while  $(1/k)(dk/dT)$  is computed by differentiating the thermal conductivity relationship with respect to temperature.

All property relationships are presented in analytical form obtained by fitting a power polynomial through the data points. The data points listed in the tables are taken from the reference entered before each table. Numerical techniques used for the curve fitting process are explained in Section IV.

The polynomial expression for each property has been compared with the referenced data and within the listed temperature range has been found to deviate by no more than the percentage error indicated.

## I. COPPER

1. Specific Heat

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 1, 1967, pp. 456-7.

Data Points:

T	$c_p$
600 R	0.0920 Btu/(lbm R)
1000	0.0975
2000	0.1112

Polynomial Fit:

Temperature Range: 400 - 2000 R

Equation:

$$c_p = (0.08375 + 1.375 \times 10^{-5} T R^{-1}) \times 32.174 \text{ Btu/ (slug R)} \quad (\text{A.1})$$

Maximum Error: There was no difference between the computed value and the input data within the accuracy of the computer.

2. Thermal Conductivity

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 1, 1967, pp. 458-9.



Data Points:

T	k
600 R	228.369 Btu/(hr ft R)
800	225.708
1000	222.805
1200	219.418
1400	215.306

Polynomial Fit:

Temperature Range: 500 - 1800 R

Equation:

$$k = (228.369 - 2.62067\theta - 0.04033\theta^3) \text{ Btu/(hr ft R)} \quad (\text{A.2})$$

where

$$\theta = \frac{T - 600.0\text{R}}{200.0\text{R}} \quad (\text{A.3})$$

Maximum Error: 0.87%

### 3. Temperature Variation of Thermal Conductivity

Eq. A. 2 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{1}{200} \frac{(-2.62067 - 0.121 \theta^2)}{(228.369 - 2.62067\theta - 0.04033 \theta^3)} \text{ R}^{-1} \quad (\text{A.4})$$

#### 4. Modulus Elasticity:

Reference: "Material Manual," TRW Equipment Laboratories, February

1966, Report No. ER-6756, Contract No. NAS 9-4884, Fig. 50.

Data Points:

T	Y
0 F	$16.55 \times 10^6 \text{ lbf/in}^2$
400	14.35
800	9.65
1200	3.82

Polynomial Fit:

Temperature Range: 500 - 1600 R

Equation:

$$Y = (16.55 - 0.4933\theta - 1.935\theta^2 + 0.2283\theta^3) \times 1.44 \times 10^8 \text{ (lb f/ft}^2\text{)} \quad (\text{A.5})$$

where

$$\theta = \frac{T - 459.67 \text{ R}}{200 \text{ R}} \quad (\text{A.6})$$

Maximum Error: 0.44%

## II. ALUMINUM 7075

1. Specific Heat

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 2-11. 1967, pp. 810-11.

Data Points:

T	$c_p$
400 R	0.182 Btu/(lbm R)
600	0.209
800	0.226
1000	0.244
1200	0.270

Polynomial Fit:

Temperature Range: 300 - 1200 R

Equation:

$$c_p = (0.182 + 0.03616\theta - 0.011417\theta^2 + 0.00233\theta^3 - 0.000083\theta^4) \times 32.174 \text{ Btu/(slug R)} \quad (\text{A.7})$$

where

$$\theta = \frac{T - 400 \text{ R}}{200 \text{ R}} \quad (\text{A.8})$$

Maximum Error: 0.34%

## 2. Thermal Conductivity

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 2-11, pp. 812-13.

Data Points:

T	k
400 R	88.50 Btu/(hr ft R)
600	100.395
800	105.96
1000	104.024
1200	99.18

Polynomial Fit

Temperature Range: 300 - 1200 R

Equation:

$$k = (88.5 + 13.0665\theta + 0.33275\theta^2 - 1.758\theta^3 + 0.25375\theta^4)$$

(A.9)

Btu/(hr ft R)

where

$$\theta = \frac{T - 400 \text{ R}}{200 \text{ R}} \quad (\text{A.10})$$

Maximum Error: 0.96%

### 3. Temperature Variation of Thermal Conductivity:

Equation A.9 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{1}{200} \frac{(13.0665 + 0.66550 - 5.250^2 + 1.0150^3)}{(88.5 + 13.06650 + 0.3327502 - 1.7580^3 + 0.2537504)}$$

(A.11)

### 4. Modulus of Elasticity

Reference: "Material Manual," TRW Equipment Laboratories, February 1966, Report ER-6756, NAS 9-4884, Fig. 50.

Data Points:

T	Y
0 F	$10.71 \times 10^6 \text{ lbf/in}^2$
200	9.90
400	8.50
600	6.15

Polynomial Fit:

Temperature Range: 500 - 1200 R

Equation:

$$Y = (10.71 - 0.630 - 0.1150^2 - 0.060^3) \quad (\text{A.12})$$

$$\times 1.44 \times 10^8 \text{ lbf/ft}^2$$

where

$$\theta = \frac{T - 459.67 R}{200 R} \quad (A.13)$$

Maximum Error: 0.38%

### III. BERYLLIUM (1/2 - 3% Be O)

#### 1. Specific Heat

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 6-II, 1967, pp. 753-4.

Data Points:

T	$c_p$
800 R	0.536 Btu/(lbm R)
1000	0.585
1200	0.622
1400	0.652
1600	0.680

Polynomial Fit:

Temperature Range: 400 - 1700 R

Equation:

$$c_p = (0.536 + 0.05667\theta - 0.0085\theta^2 + 0.00083\theta^3) \quad (\text{A.14})$$

X 32.174 Btu/(slug R)

where

$$\theta = \frac{T - 800 \text{ R}}{200 \text{ R}} \quad (\text{A.15})$$

Maximum Error: 0.88%

## 2. Thermal Conductivity

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol 6-II, 1967, pp. 757-9.

Data Points:

T	k
400 R	108.863 Btu/(hr ft R)
600	98.944
800	89.751
1000	80.80
1200	72.091

Polynomial Fit :

Temperature Range: 400 - 1700 R

Equation:

$$k = (108.863 - 10.5643\theta + 0.82683\theta^2 - 0.20167\theta^3 + 0.020167\theta^4) \text{ Btu/(hr ft R)} \quad (\text{A.16})$$

where

$$\theta = \frac{T - 400 \text{ R}}{200 \text{ R}} \quad (\text{A.17})$$

Maximum Error: 0.90%



### 3. Temperature Variation of Thermal Conductivity

Equation A.15 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{1}{200} \frac{(-10.5643 + 1.65367\theta - 0.60501\theta^2 + 0.080668\theta^3)}{(108.863 - 10.5643\theta + 0.82683\theta^2 - 0.20167\theta^3 + 0.020167\theta^4)} R^{-1} \quad (A.18)$$

### 4. Modulus of Elasticity

Reference: "Material Manual," TRW Equipment Laboratories, February 1966, Report ER-6756, Contract No. NAS 9-4884, Fig. 51.

Data Points:

T	Y
0 F	44.36 x 10 <sup>6</sup> lbf/in <sup>2</sup>
400	40.41
800	33.95
1200	21.80

Polynomial Fit:

Temperature Range: 500 - 1700 R

Equation:

$$Y = (44.36 - 3.755\theta + 0.335\theta^2 - 0.53\theta^3) \times 1.44 \times 10^8 \text{ lb f/ft}^2 \quad (A.19)$$

where

$$\theta = \frac{T - 459.67 \text{ R}}{400 \text{ R}} \quad (\text{A.20})$$

Maximum Error: 0.28%

APPENDIX B  
COOLANT FLUID PROPERTIES

## I. HELIUM

1. Equation of State Explicit in Pressure

Reference: Akin, S. W., Trans. ASME, Vol. 72, p. 751, 1950.

Equation: The National Bureau of Standards has published a Benedict-Webb-Rubin equation for helium; this equation was found to be valid only up to the specified pressure limit of 3000 lbf/in<sup>2</sup>. Preference was therefore given to the following Beattie-Bridgeman equation:

$$p = \rho^2 \left[ RT(1 - a) \left( \frac{1}{\rho} + B_1 \right) - A \right] \quad (\text{B.1})$$

where

$$a = \frac{c}{T^3} \rho$$

$$A = A_1 (1 - a \rho)$$

The values of the constants in Eq. B.1, in MKSA units, are:

$$R = 2.07702 \times 10^3$$

$$A = 1.369595 \times 10^2$$

$$B = 3.5002295 \times 10^{-3}$$

$$C = 1.0000658 \times 10^1$$

$$a = 1.496103 \times 10^{-2}$$

Temperature Range: 160 - 860 R

Pressure Range: 2116 - 360000 lbf/ft<sup>2</sup>

Maximum Error: 0.095%

## 2. Equation of State Explicit in Density

Since the equation of state is needed explicit in density, Eq. B.1 was solved using Newton-Raphson iteration method along an isotherm to give

$$\rho_{i+1} = \rho_i - \frac{P - P(\rho_i)}{(\partial P / \partial \rho)_T} \quad (\text{B.2})$$

Using Eq. B.1, one obtains

$$\left( \frac{\partial P}{\partial \rho} \right)_T = RT + 2(RB_1T - A_1 - \frac{CR}{T^2})\rho + 3(A_1a - \frac{CRB_1}{T^2})\rho^2 \quad (\text{B.3})$$

## 3. Isobaric Thermal Expansion Coefficient

The isobaric thermal expansion coefficient is defined by the equation

$$\beta = -\frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_P \quad (\text{B.4})$$

Since the equation of state (Eq. B.1) is explicit in the pressure, one can write  $\beta$  as:

$$\beta = \frac{1}{\rho} \frac{(\partial P / \partial T)_\rho}{(\partial P / \partial \rho)_T} \quad (\text{B.5})$$

or

$$\beta = \frac{R[\rho + (B_1 + \frac{2C}{T^3})\rho^2 + \frac{2CB_1}{T^3}\rho^3]}{\rho[RT + 2\rho(RB_1T - A_1 - \frac{CR}{T^2}) + 3\rho^2(A_1a - \frac{CRB_1}{T^2})]} \quad (\text{B.6})$$

#### 4. Isothermal Compressibility

The isothermal compressibility is defined by the equation

$$\kappa = \frac{1}{\rho} \left( \frac{\partial \rho}{\partial p} \right)_T \quad (\text{B.7})$$

Making use of Eq. B.3, the isothermal compressibility can be written as

$$\kappa = 1/\rho \left[ RT + 2\rho(RB_1T - A_1 - \frac{CR}{T^2}) + 3\rho^2(A_1a - \frac{CRB_1}{T^2}) \right] \quad (\text{B.8})$$

#### 5. Specific Heat at Constant Pressure:

Equation: Experimental and quantum statistical data for helium show that at zero-pressure, the specific heat at constant volume is independent of temperature

$$c_v^0 = \frac{3}{2} R \quad (\text{B.9})$$

Using Maxwell's equations, the following expression is obtained

$$c_v = c_v^0 - T \int_{\rho_0}^{\rho} \left( \frac{\partial^2 p}{\partial T^2} \right)_{\rho} \frac{d\rho'}{(\rho')^2} \quad (\text{B.10})$$

From the equation of state, (Eq. B.1) the integration is carried out in closed form to give

$$c_v = R \left[ \frac{3}{2} + 6a \left( 1 + \frac{\rho}{2} B_1 \right) \right] \quad (\text{B.11})$$

The relation between specific heat at constant pressure and that at constant volume is given by:

$$c_p = c_v + \frac{T\beta^2}{\rho\kappa} \quad (\text{B.12})$$

Temperature Range: 180 - 900 R

Pressure Range: 2116 - 216000 lbf/ft

Maximum Error: 0.58%

## 6. Enthalpy

The variation of internal energy with both temperature and density is

$$du = c_v dT + \left[ p - T \left( \frac{\partial p}{\partial T} \right)_\rho \right] \frac{d\rho}{\rho^2}$$

carrying out the integration along an isochore and an isotherm, one obtains

$$u = u_o + R \left[ \frac{3}{2} (T - T_o) + 3 \rho c \left( 1 + \frac{B_1}{2} \right) \left( \frac{1}{T_o^2} - \frac{1}{T^2} \right) \right] + \left( \frac{3 R C}{T^2} + A_1 \right)$$

$$(\rho_o - \rho) + \frac{1}{2} \left( \frac{3 C R B_1}{T^2} - A_1 a \right) (\rho_o^2 - \rho^2)$$

where  $u_o = 3.9922 \times 10^4$  j/kg

$T_o = 10.938889$  K

$\rho_o = 4.669193$  kg/m<sup>3</sup>

The enthalpy may then be determined from the equation

$$h = u + \frac{P}{\rho}$$

## 7. Thermal Conductivity

Reference: Akin, S. W., Trans. ASME, Vol. 72, p. 751, 1950

Data Points:

T	k
160 R	0.0404 Btu/(hr ft R)
360	0.0676
560	0.090
760	0.1094

Polynomial Fit:

Temperature Range: 160 - 860 R

Equation:

$$k = (0.0404 + 0.0302 \theta - 0.0033 \theta^2 + 0.0003 \theta^3) \text{ Btu/(hr ft R)} \quad (\text{B.13})$$

where

$$\theta = \frac{T - 160 \text{ R}}{200 \text{ R}}$$

Maximum Error: 0.54%



## 8. Temperature Variation of Thermal Conductivity

Eq. B. 13 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{1}{200} \frac{(0.0302 - 0.0066 \theta + 0.0009 \theta^2)}{(0.0404 + 0.0302 \theta - 0.0033 \theta^2 + 0.0003 \theta^3)} \frac{1}{R} \quad (\text{B.14})$$

## 9. Dynamic Viscosity

Reference: Akin, S. W., Trans. ASME, Vol. 72, p. 751, 1950

Equation: Viscosity correlations are ususally based on the concept of residual viscosity:

$$\psi_1 (\rho) = \mu(\rho, T) - \mu^*(T) \quad (\text{B.15})$$

where

$\psi_1$  is the residual viscosity (function of density alone).

$\mu^*$  is the dynamic viscosity at atmospheric pressure.

For helium, the dynamic viscosity is given by:

$$\mu = \mu^* = 2.58394 \times 10^{-5} T/^{\circ}\text{R})^{0.647} \text{ slug/(ft hr)} \quad (\text{B.16})$$

Temperature Range: 160 - 660 R

Maximum Error: 0.29%

## II SILICON OIL

The following properties are for Dow Corning 200 Silicon Oil (1 Centistoke at 77 F).

### 1. Isothermal Compressibility

Reference: Gunst, S. B., "Density-Pressure Relationships for Two Low-Viscosity Dimethyl Siloxanes," Trans. ASME 72, May 1950, pp. 401-7.

Data Points: Variation of  $\kappa$  with temperature at 0 psig and 500 psig are given below:

t	$\kappa_0$	$\kappa_{500}$
100 F	$12.35 \times 10^{-6} \text{ in}^2/\text{lb f}$	$11.60 \times 10^{-6} \text{ in}^2/\text{lb f}$
150	16.05	14.94
200	20.45	18.82
250	26.25	23.86
300	36.55	31.88

Polynomial Fit:

Temperature Range: 560 - 760 R

Pressure Range: 2116 - 74116  $\text{lb f/ft}^2$

Equation: The variation of  $\kappa_0$  with temperature at 0 psig is given by

$$\kappa_o = (12.35 + 2.9833 \theta + 1.1 \theta^2 - 0.48333 \theta^3 + 0.1 \theta^4) \times 10^{-6} \text{ in}^2/\text{lb f} \quad (\text{B.17})$$

where 
$$\theta = \frac{T - 559.67 \text{ R}}{50 \text{ R}}$$

$\kappa$  is assumed to vary linearly on the above range of pressure,

hence

$$\kappa = a + b p \quad (\text{B.18})$$

where

$$a = \kappa_o$$

$$b = \left( \frac{\partial \kappa}{\partial p} \right)_T \approx \frac{\kappa_{500} - \kappa_o}{500 \text{ psi}}$$

and  $p$  is the pressure in psig

Fitting a power polynomial through  $\left( \frac{\partial \kappa}{\partial p} \right)_T$  with the same values for temperature as indicated in the table results in the equation

$$b = (-1.5 - 0.0133 \theta - 1.18 \theta^2 + 0.57333 \theta^3 - 0.1 \theta^4) \times 10^{-9} \text{ in}^4/\text{lb f}^2 \quad (\text{B.19})$$

Maximum Error: There was no difference between the computed and the input data within the accuracy of the computation.

## 2. Equation of State Explicit in Density

Reference: Gunst, S. B., "Density-Pressure Relationships for Two Low-Viscosity Dimethyl Siloxanes," Trans. ASME, May 1950, pp. 401-7.

Data Points: Values for the variation of density with temperature at 0 psig are given below:

t	$\rho_o$
150 F	0.7767 gm/cm <sup>3</sup>
200	0.7479
250	0.7188
300	0.6900

Polynomial Fit:

Temperature Range: 540 - 760 R

Pressure Range: 2116 - 146116 lbf/ft<sup>2</sup>

Equation:

$$\rho_o = (0.7767 - 0.0288 \theta) \times 1.94 \quad \text{slug/ft}^3 \quad (\text{B.20})$$

where

$$\theta = \frac{T - 609.67 \text{ R}}{50 \text{ R}}$$

The variation of density with pressure is given by

$$dz = \frac{d\rho}{\rho} = -\beta dT + \kappa dp \quad (\text{B.21})$$

Integration along the isotherm  $T_o = 609.67 \text{ R}$  and from

$p' = 0 \text{ psig}$  to  $p' = p$ , using Eq. B.18 results in

$$z(p, T_o) = a(T_o) p + b(T_o) \frac{p^2}{2}$$

Integration along the isobar  $p$ , from  $T' = T_o$ , to  $T' = T$ ,

making use of Eq. B.25, yields

$$z(p, T) - z(p, T_o) = p \int_{T_o}^T a'(T') dT' + \frac{p^2}{2} \int_{T_o}^T b'(T') dT' - \int_{T_o}^T c(T') dT'$$

which after simplification reduces to

$$\rho = \rho_o e^{ap + \frac{1}{2} b p^2} \quad (B.22)$$

Maximum Error: 0.128%

### 3. Equation of State Explicit in Pressure

Since the equation of state is needed explicit in pressure,

Eq. B.22 was rearranged to yield

$$p = \frac{1}{b} \left[ -a + \sqrt{a^2 + 2b \ln \frac{\rho}{\rho_o}} \right] \quad (B.23)$$

### 4. Isobaric Thermal Expansion Coefficient

From Eq. B.20, the zero-pressure isobaric thermal expansion coefficient can be written as:

$$\beta_o = - \frac{1}{\rho_o} \left( \frac{\partial \rho_o}{\partial T} \right)_p$$

or

$$\beta_o = \frac{0.0288}{50(0.7767 - 0.0288 \theta)} \quad \frac{1}{R} \quad (B.24)$$

Making use of Eq. B.21, together with the principle of an exact differential, one may write

$$\left( \frac{\partial \beta}{\partial p} \right)_T = - \left( \frac{\partial \kappa}{\partial T} \right)_p$$

From Eq. B.18, the variation of  $\beta$  with both pressure and temperature is given by

$$\beta = \beta_0 - a' p - \frac{b'^2}{2} p^2 \quad (\text{B.25})$$

where the prime superscript indicates differentiation with respect to temperature. The expressions for  $a$  and  $b$  as a function of temperature are given in Eqs. B.17 and 19, respectively.

### 5. Specific Heat at Constant Pressure

Reference: Dow Corning, Bulletin 05-145, February 1966.

Data Points: The available data for the variation of zero pressure specific heat at constant pressure for 2 centistokes are:

$t$	$c_p^0$	
80 F	0.448	Btu/(lbm F)
160	0.454	
240	0.463	
320	0.476	
400	0.491	

#### Polynomial Fit:

Temperature Range: 540 - 860 R

Pressure Range : 2116 - 74116 lbf/ft<sup>2</sup>

Equation: The above data for 2 centistokes silicon oil were

multiplied by the ratio of  $c_p^0$  for 1 centistoke to 2 centistokes at 77°F, to give the following expression for the zero-pressure specific heat at constant pressure for 1 centistoke silicon oil

$$c_p^o = (0.46 + 0.00471 \theta + 0.00141 \theta^2 + 0.000043 \theta^3) \quad (\text{B.26})$$

x32.174 Btu/(slug R)

where

$$\theta = \frac{T - 539.67 \text{ R}}{80 \text{ R}}$$

The variation of  $c_p$  with pressure is given by

$$c_p = c_p^o - T \int_{p_o}^p \left( \frac{\partial^2}{\partial T^2} \right)_p \left( \frac{1}{\rho} \right) dp' \quad (\text{B.27})$$

Since the exponent in Eq. B.22 is small, the equation for the density, when expanded in a power series, may be truncated after the second term in the expansion. Applying Eq. B.27 to the two-term expression for density as a function of pressure and temperature by integrating along an isotherm  $T$  from  $p' = 0$  psig to  $p' = p$ , results in the expression for

$$c_p = c_p^o - \frac{T I}{\rho_o} \quad (\text{B.28})$$

where

$$I = z_1 p + z_2 p^2 + z_3 p^3 + z_4 p^4 + z_5 p^5$$

and

$$z_1 = 2 \left( \frac{\rho_o'}{\rho_o} \right)^2$$

$$z_2 = \frac{1}{2} \left[ \left( 2 \frac{\rho_o'}{\rho_o} a' - a'' \right) - a z_1 \right]$$

$$z_3 = \frac{1}{3} \left[ a'^2 - \frac{b''}{2} + \frac{\rho_o'}{\rho_o} b' \right] - a \left( 2 \frac{\rho_o'}{\rho_o} a' - a'' \right) + \frac{1}{2} (a^2 - b) z_1$$

$$\begin{aligned}
z_4 &= \frac{1}{4} [a'b' - a (a'^2 - \frac{b''}{2} + \frac{\rho'_0}{\rho_0} a') + \frac{1}{2} (a^2 - b) (2 \frac{\rho'_0}{\rho_0} a' - a'')] \\
&\quad + \frac{1}{2} ab z_1] \\
z_5 &= \frac{1}{10} [\frac{b'^2}{2} - 2a a'b' + (a^2 - b) (a'^2 - \frac{b''}{2} + \frac{\rho'_0 b'}{\rho_0}) \\
&\quad + ab (2 \frac{\rho'_0}{\rho_0} a' - a'') + \frac{1}{2} b^2 z_1]
\end{aligned}$$

where the prime superscript indicates differentiation with temperature.

**Maximum Error:** For zero pressure specific heat at constant pressure the maximum error was 0.065%. For higher pressures no experimental data were available for comparison. However the expression for enthalpy was numerically differentiated with respect to temperature at constant pressure and compared with the computed values of specific heat at constant pressure. The comparison showed no difference within the accuracy of the computation.

## 6. Enthalpy

The variation of enthalpy with both pressure and temperature is given by:

$$dh = c_p^0 dT + \frac{1}{\rho} [1 - T\beta] dp$$

This expression was integrated along the isobar  $p = 0$  psig from  $T' = 539.65$  R to  $T' = T$ , and then along an isotherm  $T$  from  $p' = 0$  psig to  $p' = p$ , to give



$$h = \left\{ 80(0.46 + 0.00471 \theta + 0.00141 \theta^2 + 0.000043 \theta^3) \right.$$

$$+ \frac{1}{\rho_o \cdot 337.37} \left[ -\frac{1}{20} b' T p^5 - \frac{1}{8} (a b' + a' b) T p^4 \right.$$

$$+ \frac{1}{6} (T(b' - a a') - b(1 + T \frac{\rho'_o}{\rho_o}) ) p^3 + \frac{1}{2} (a' T - a (1$$

$$+ T \frac{\rho'_o}{\rho_o}) ) p^2 + (1 + T \frac{\rho'_o}{\rho_o}) p] \times 32.174 \left. \right\} \text{ Btu/slug} \quad (\text{B.29})$$

## 7. Dynamic Viscosity

Reference: Dow Corning, Bulletin 05-153, July 1966.

Data Points:

t	v
0 F	1.98 centistokes
100	0.874
200	0.56
300	0.41

$$\ln \nu = (0.683 - 1.0845 \theta + 0.3065 \theta^2 - 0.04 \theta^3) \quad (\text{B.30})$$

where  $\nu$  is in centistokes

$$\mu = \nu \rho \quad (\text{B.31})$$

Maximum Error: 0.77%

## 8. Thermal Conductivity

Reference: Dow Corning, Bulletin 05-145, February 1966.

Data Points: The available data for the variation of thermal conductivity with temperature for 2 centistokes are:

t		k	
-100	F	0.0674	Btu/(hr ft F)
100		0.0626	
300		0.0578	

Polynomial Fit:

Temperature Range: 360 - 860 R

Equation: The procedure that was used for specific heat at constant pressure was followed to get an expression for the variation of thermal conductivity with temperature for 1 centistoke silicon oil. The resulting expression for the thermal conductivity is given by

$$k = (7.0052 - 2.2105 \times 10^{-3} T \text{ R}) \times 10^{-2} \text{ Btu/(hr ft R)} \quad (\text{B.32})$$

Maximum Error: There was no difference between the computed  
and the input data within the accuracy of the computation.

#### 9. Temperature Variation of Thermal Conductivity

Eq. B.32 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{-2.2105 \times 10^{-3}}{7.0052 - 2.2105 \times 10^{-3} T/^{\circ}\text{R}} \quad 1/R \quad (\text{B.33})$$

## III NAK - 78.6

The following are the physical properties of NaK(78.6 wt% K) extracted from the latest version of the "Liquid Metals Handbook, Sodium and NaK Supplement" (to be published). Some typical properties are:

Melting Point: 92 F

Boiling Point: 1445 F

Surface Tension: 0.00739 lbf/ft at Melting point

1. Equation of State Explicit in Density

Data Points: Values for the variation of density with temperature at zero pressure is given by

t	$\rho_o$
200 F	53.21 lbm/ft <sup>3</sup>
500	50.68
800	48.15
1100	45.62
1400	43.09

Polynomial Fit:

Temperature Range: 200 - 1400 °F

Equation:

$$\rho_o = (58.773064 - 0.008433 T/R)/32.174 \text{ slug/ft}^3 \quad (\text{B.34})$$

Since the isothermal compressibility is assumed to be independent of pressure, Eq. B.22 for silicon reduces to

$$\rho = \rho_0 e^{\kappa p} \quad (\text{B.35})$$

where  $p$  is the gage pressure. Since the exponent is small, the expression for the density, when expanded in a power series, may be truncated after the second term and the variation of density with both pressure and temperature is given by

$$\rho = \rho_0 (1 + \kappa p) \quad (\text{B.36})$$

Maximum Error: At zero pressure, there was no difference between the computed and the input data, within the accuracy of the computation. For higher pressures there were no experimental data available for comparison.

## 2. Equation of State Explicit in Pressure

Since the equation of state is needed explicit in pressure, Eq. B.36 was rearranged to yield

$$p = \frac{1}{\kappa} \left( \frac{\rho}{\rho_0} - 1 \right) \quad (\text{B.37})$$

## 3. Isobaric Thermal Expansion Coefficient

The isobaric thermal expansion coefficient is defined to be

From Eq. B.34, the zero pressure isobaric thermal expansion coefficient is given by:

$$\beta_o = \frac{0.008433}{(58.773064 - 0.008433T)} \frac{1}{R} \quad (B.38)$$

Due to the lack of experimental data, the isobaric thermal expansion coefficient was assumed to be independent of pressure.

#### 4. Isothermal Compressibility

In view of the experimental difficulties associated with the measurement of isothermal compressibility at elevated temperatures, such data are not generally available for liquid metals. However, the well-known relationship between velocity of sound  $c$ , density  $\rho$ , and isentropic compressibility  $\kappa_s$  is

$$\kappa_s = \frac{1}{\rho c^2} \quad (B.39)$$

which makes an alternative approach to the problem possible, if velocities of sound can be measured. Under these circumstances the isothermal compressibility may be obtained from the relation

$$\kappa = \gamma \kappa_s \quad (B.40)$$

where

$$\gamma = \frac{c_p}{c_v}$$

The relation between  $c_p$  and  $c_v$  is given by

$$c_p - c_v = \frac{T\beta^2}{\rho\kappa} \quad (\text{B.41})$$

From Eqs. B.40 and B.41, one gets

$$\kappa = \kappa_s + \frac{T\beta^2}{\rho c_p}$$

or

$$\kappa_o = \left\{ \frac{1}{\rho} \left( \frac{1}{c^2} + \frac{T\beta^2}{c_p} \right) \right\}_{\substack{T_o \\ p_o}} \quad (\text{B.42})$$

Due to the lack of experimental data Eq. B.42 was evaluated at the absolute pressure  $p_o$  of one atmosphere and the temperature of  $T_o = 1260$  R.

at  $T_o = 1260$  R

$$\begin{aligned} \rho_o &= 48.15 \text{ lbm/ft}^3 \\ c_o &= 7544 \text{ ft/sec} \\ \beta_o &= 1.75149 \times 10^{-4} \text{ 1/R} \\ C_{p,o} &= 0.2091 \text{ Btu/(lbm R)} \end{aligned}$$

## 5. Specific Heat at Constant Pressure

Data Points: For zero-pressure specific heat at constant pressure are given by:

t	$c_p^o$
200 F	0.2255 Btu/(lbm F)
500	0.1239
800	0.2093
1100	0.2091
1400	0.2120

Polynomial Fit:

Temperature Range: 200 - 1400 F

Equation:

$$c_p^o = (0.2255 - 0.016292 \theta + 0.00539 \theta^2 - 0.000758 \theta^3 + 0.000054 \theta^4) \times 32.174 \text{ Btu/(slug R)} \quad (\text{B.44})$$

where

$$\theta = \frac{T - 659.67 \text{ R}}{300 \text{ R}}$$

The variation of  $c_p$  with pressure is given by



$$c_p = c_p^o - T \int_{p_o}^p \frac{\partial^2}{\partial T^2} \left( \frac{1}{\rho} \right)_p dp'$$

Integrating along the isotherm  $T$  from  $p = 0$  psig to  $p' = p$ , using Eqs. B.36 and B.38 results in

$$c_p = c_p^o - \frac{2 T \beta_o^2}{\rho_o \kappa} \ln (1 + \kappa p) \quad (B.45)$$

Maximum Error: At zero pressure, the maximum error was 0.075%.

## 6. Enthalpy

The variation of enthalpy with both pressure and temperature is given by

$$dh = c_p dT + \frac{1}{\rho} [1 - T \beta] dp$$

The enthalpy was arbitrarily chosen to be zero near the melting point, or  $T = 469.67^\circ R$ . The above expression was integrated along an isobar  $p = 0$  psig from  $T' = 469.67^\circ R$  to  $T' = T$ , and then along an isotherm  $T$  from  $p' = 0$  psig to  $p' = p$ , to give

$$\begin{aligned} h = & [300 (0.2255 \theta - 0.016292 \frac{\theta^2}{2} + 0.00539 \frac{\theta^3}{3} \\ & - 0.000758 \frac{\theta^4}{4} + 0.000054 \frac{\theta^5}{5}) + \frac{1 - T\beta}{\rho_o \kappa} \ln (1 + \kappa \rho) \quad (B.46) \\ & \times \frac{1}{778.26}] 32.174 \text{ Btu/slug} \end{aligned}$$

where  $\rho$  is in  $\text{lbm/ft}^3$ ,  $\kappa$  is in  $\text{ft}^2/\text{lbf}$  and  $p$  is gage pressure in  $\text{lbf/ft}^2$ .

## 7. Thermal Conductivity

Data Points:

t	k
200 F	13.36 Btu/(hr ft F)
500	14.57
800	15.18
1100	15.03
1400	14.13

Polynomial Fit:

Temperature Range: 200 - 1400 F

Equation:

$$k = (13.36 + 1.414167 \theta - 0.142083 \theta^2 - 0.069167 \theta^3 + 0.007083 \theta^4) \text{ Btu/(hr ft R)} \quad (\text{B.47})$$

where

$$\theta = \frac{T - 659.67 \text{ R}}{300 \text{ R}}$$

Maximum Error: 0.2%

## 8. Temperature Variation of Thermal Conductivity:

Eq. B.47 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{1}{300} \frac{(1.414167 - 0.284166\theta - 0.207501\theta^2 + 0.028332\theta^3)}{(13.36 + 1.414167\theta - 0.142083\theta^2 - 0.069167\theta^3 + 0.007083\theta^4)} \frac{1}{R} \quad (\text{B.48})$$

9. Dynamic Viscosity

Data Points:

t	$\mu$
200 F	1.1316 lbm/(ft hr)
500	0.746
800	0.534
1100	0.411
1400	0.340

Polynomial Fit:

Temperature Range: 200 - 1400 F

Equation:

$$\mu = (1.316 - 0.896667 \theta + 0.419833 \theta^2 - 0.102833 \theta^3 + 0.009667 \theta^4) / 32.174 \quad \text{slug/(ft hr)} \quad (\text{B.49})$$

where

$$\theta = \frac{T - 659.67R}{300R}$$

Maximum Error: 1.2%

## APPENDIX C

Optical Properties

Three optical properties are required in the radiative analysis discussed in Chapter 6, namely the total hemispherical emittance

$$\varepsilon(T) = \frac{1}{E_b(T)} \int_0^{\infty} \varepsilon_{\lambda}(T) E_{b,\lambda}(T) d\lambda \quad (C.1)$$

and the two auxiliary functions (see Eqs. 6.10 and 6.11)

$$XX(T_1, T_2) = \frac{1}{E_b(T_2)} \int_0^{\infty} \varepsilon_{\lambda}(T_1) \varepsilon_{\lambda}(T_2) E_{b,\lambda}(T_2) d\lambda \quad (C.2)$$

$$XXX(T_1, T_2, T_3) = \frac{1}{E_b(T_3)} \int_0^{\infty} \varepsilon_{\lambda}(T_1) \varepsilon_{\lambda}(T_2) \varepsilon_{\lambda}(T_3) E_{b,\lambda}(T_3) d\lambda \quad (C.3)$$

In view of the temperature independence of the spectral emittance for dielectrics, the two functions XX and XXX are functions of a single temperature, the temperature of the surface element represented by the last subscript on the left-hand sides of Eqs. 6.10 and 6.11:

$$XX(T) = \frac{1}{E_b(T)} \int_0^{\infty} \varepsilon_{\lambda}^2 E_{b,\lambda}(T) d\lambda \quad (C.4)$$

$$XXX(T) = \frac{1}{E_b(T)} \int_0^{\infty} \varepsilon_{\lambda}^3 E_{b,\lambda}(T) d\lambda \quad (C.5)$$

These functions are evaluated for the zinc oxide/potassium silicate coating Z-93 on the basis of spectral reflectance data measured by IITRI and published in the NASA Contractor Report No. 1420, titled Emissivity Coatings for Low-Temperature Space Radiators, by G. R. Cunningham, J. R. Grammer, and F. J. Smith, Lockheed Aircraft Corp., Sunnyvale, Calif., Sept. 1969, pp. 66 through 81.

The evaluated functions defined through Eqs. C.1, 4 and 5 are collocated by power polynomials of this form

$$f(T) = \sum_{i=0}^N a_i T^i \quad (C.6)$$

For the total hemispherical emittance, a fourth degree power polynomial was found to be satisfactory with

$$\begin{aligned} a_0 &= 0.8990103 \\ a_1 &= -0.1400633 \times 10^{-3} \\ a_2 &= 0.387900 \times 10^{-6} \\ a_3 &= -0.3937509 \times 10^{-9} \\ a_4 &= 0.1015627 \times 10^{-12} \end{aligned}$$

For the auxiliary functions XX and XXX the coefficients are

	XX	XXX
$a_0 =$	0.7804112	0.6538383
$a_1 =$	$-0.5527205 \times 10^{-4}$	$0.1144374 \times 10^{-3}$
$a_2 =$	$0.2530228 \times 10^{-6}$	$-0.2432286 \times 10^{-7}$
$a_3 =$	$-0.3229181 \times 10^{-9}$	$-0.1437500 \times 10^{-9}$
$a_4 =$	$0.8854202 \times 10^{-13}$	$0.4947915 \times 10^{-13}$

## APPENDIX D

The Simplified Radiator System Analysis

The simplified analysis is intended to support and to give guidance in the detailed and rigorous computer simulation of the Space Radiator Heat Rejection System. While it is possible to derive valuable insight in the radiator performance with relatively little effort, some results must be considered with great caution as presently it seems impossible to assess the extent of their validity in view of the simplifying assumptions made in the analysis.

Considered is a plane, rectangular panel, interrupted by  $n$  parallel tubes of length  $L$  and diameter  $d$ , all equally spaced and the distance  $(H/n + d)$  apart, where  $(d \ll H/n)$ . Assuming that the panel rejects heat from both sides into an environment at the "equivalent sink temperature"  $T_s$  and that the tube temperature is uniform and equal to  $T_b$ , one obtains for the power rejection from the panel area  $2A = 2HL$

$$\dot{Q} = 2A\epsilon\eta (T_b^4 - T_s^4) \quad (D.1)$$

Here, the fin effectiveness  $\eta$  is a known function  $\eta(N_c)$  of the conductance parameter

$$N_c = \epsilon\sigma T_b^3 (H/2n)^2 / (kt/2) \quad (D.2)$$

and the sink temperature  $T_s$ . In Eqs. 1 and 2,  $\epsilon$  represents the surface emittance, while  $k$  and  $t$ , in Eq. 2, stand for the thermal conductivity and the panel thickness, respectively.

The power rejected according to Eq. 1 must be transferred from the working fluid to the radiator tube. If the fluid enters the radiator at the rate  $\dot{M}$ , with the inlet temperature  $T_o$  and if the fluid has constant thermophysical properties then

$$\dot{Q} = \dot{M} c_p (T_o - T_b) (1 - e^{-u}) \quad (D.3)$$

where  $c_p$  represents the specific heat at constant pressure and

$$U = \frac{\pi d h L}{(\dot{M}/n) c_p} = \frac{4L}{d} \frac{N_{Nu}}{N_{Pr} N_{Re}} \quad (D.4)$$

which is a constant depending on tube geometry, fluid properties and operating conditions but not on radiator temperature (because the fluid properties are considered to be invariant along the tube). The Nusselt number  $N_{Nu}$  may be related to the Prandtl number  $N_{Pr}$  and the Reynolds number  $N_{Re}$  through

$$N_{Nu} = 5 + 0.025 (N_{Re} N_{Pr})^{0.8} \quad \text{for } N_{Pr} < 0.1, N_{Re} > 2300 \quad (D.5)$$

$$N_{Nu} = 0.023 (N_{Re}^{0.8} N_{Pr}^{0.3}) \quad \text{for } N_{Pr} > 0.1, N_{Re} > 2300 \quad (D.6)$$

$$N_{Nu} = 3.65 + \frac{0.0668 (N_{Re} N_{Pr} d/L)}{1 + 0.045 (N_{Re} N_{Pr} d/L)^{2/3}} \quad N_{Re} \leq 2300 \quad (D.7)$$

Equations 1 and 3 define the tube temperature or fin base temperature  $T_b$ . After introducing the characteristic length  $L_o$



$$L_o = \sqrt{\frac{\dot{M} c_p}{\epsilon \sigma T_o^3}}$$

one may first normalize all dimensions

$$L^* = \frac{L}{L_o}, \quad d^* = \frac{d}{L_o}, \quad H^* = \frac{H}{L_o}, \quad t^* = \frac{t}{L_o} \quad \text{and} \quad A^* = \frac{A}{L_o^2} \quad (D.9)$$

and then the temperatures with respect to the known inlet temperature  $T_o$ :

$$\theta_b = \frac{T_b}{T_o}, \quad \theta_s = \frac{T_s}{T_o} \quad (D.10)$$

Combining Eqs. 1 and 3 gives

$$2 A^* \eta (\theta_b^4 - \theta_s^4) - (1 - \theta_b) (1 - e^{-u}) = 0 \quad (D.11)$$

as the equation which defines  $\theta_b$ . However, the effectiveness  $\eta$  depends implicitly on  $\theta_b$  through its dependence on the conductance parameter

$$N_c = \theta_b^3 \phi \left( \frac{H^*}{n} \right)^2 \frac{1}{2t^*} \quad (D.12)$$

where  $\phi$  is the system parameter, defined by

$$\phi = \left[ \frac{\epsilon \sigma T_o^3 \dot{M} c_p}{k^2} \right]^{1/2} \quad (D.13)$$

The system parameter  $\phi$ , the characteristic length  $L_o$ , the Prandtl number  $N$  together with the ratio of fin to fluid thermal conductivities,  $k/k_{f1}$ , determine the tube temperature  $\theta_b$  for any given geometry ( $L^*$ ,  $H^*$ ,  $t^*$ ,  $d^*$ ;  $n$ )

and any given environment ( $\theta_s$ ). The Renyonlds number is given by

$$N_{Re} = \frac{4\phi}{\pi n d^*} \frac{k/k_{f1}}{N_{Pr}} \quad (D.14)$$

Once the tube temperature  $\theta_b$  is found then  $N_c$  and  $\eta$  are given through Eq. 12 and  $\eta(N_c)$ . The radiant power rejection can be expressed as the fraction of the "entrance enthalpy"

$$\dot{Q}^* = \frac{\dot{Q}}{\dot{M} c_p T_o} = 2\eta A^* (\theta_b^4 - \theta_s^4) \quad (D.15)$$

and the normalized fin weight by

$$\phi_f = \frac{M_f}{\rho_f L_o^3} = \frac{t^* \dot{Q}^*}{2\eta (\theta_b^4 - \theta_s^4)} \quad (D.16)$$

which leads to the fin mass requirement per power rejection

$$\psi_f = \frac{\phi_f}{\dot{Q}^*} = \frac{t^*}{2\eta (\theta_b^4 - \theta_s^4)} \quad (D.17)$$

and, finally, to the total mass requirement

$$\psi_{tot} = \psi_f \left[ 1 + F \frac{(d^*)^2}{H^* t^*} \right] \quad (D.18)$$

where the mass augmentation factor  $F$  is defined by

$$F = \pi \left[ \frac{\rho_t}{\rho_f} \frac{d_m}{d} \frac{s}{d} \left( 1 + \frac{\rho_p}{\rho_f} \frac{A_p}{A_t} \right) + \frac{\rho_{f1}}{4\rho_f} \right] \quad (D.19)$$

with  $\rho_t$  tube material density  
 $\rho_f$  fin material density

$d_m =$	$(d + d_o)/2$ , mean tube diameter
$d$	inner tube diameter
$s =$	$(d_o - d)/2$
$\rho_p$	density of protection layer
$A_p$	cross-sectional area of protection layer
$A_t =$	$s d_m$
$\rho_{fl}$	fluid density.

In summary, it can be seen that the performance of the heat rejection system considered here is determined by

(i) operating conditions

characterized by 
$$\phi = \sqrt{\frac{\epsilon \sigma T_o^3 \dot{M} c_p}{k^2}}$$

and 
$$L_o = \sqrt{\dot{M} c_p / (\epsilon \sigma T_o^3)}$$

(ii) geometry  $L^*$ ,  $H^*$ ,  $t^*$  and  $d^*$ ;  $n$

(iii) fluid property  $N_{pr}$ ,  $k/k_{fl}$

The determination of the complete mass requirement per unit of power rejection is possible by the additional specification of the

(iv) Mass augmentation factor  $F$

which determines the mass of tube, meteoroid protection and fluid, all per unit of corresponding fin mass.

The key equation, Eq. 11 has been programmed for numerical solution, using the programming language WIPL. Important quantities such as  $Q^*$ ,  $\psi_f$  and  $\psi_{tot}$  can also be evaluated. The fin effectiveness  $\eta(N_c)$  is taken from LOVE, Radiative Heat Transfer where the results of a numerical analysis by Reynolds\* are presented in graphical and numerical form for the case of

$\theta_s = 0$ . The small correction on  $\eta$  for  $0 \leq \theta_s \leq 0.8$  can be found in Radiation Heat Transfer by Sparrow and Cess.

Results obtained from this analysis at conditions for which large-scale computer calculations had been obtained are shown, for comparison, in Part I, Chapter C.

The WIPL code is listed below.

WIPL CODE  
SIMPLIFIED ANALYSIS

```

1.000 COMMENT FIN WEIGHT AND HEAT REJECTION-
1.001 DECLARE FCC[11],FFCC[11],ETAC[11],DETAC[11],FETAC[11]-
1.002 DECLARE FDETAC[11],F1[11],F2[11]-
1.003 SET NL = 0-
1.004 SET NPHI = 0-
1.005 SET NN = 0-
1.006 SET NT = 0-
1.010 TYPE "SPECIFY FC FOR ETA TABULATION"-
1.015 DO PART 2 FOR I=1,11-
1.020 TYPE "SPECIFY CORRESP. ETA-VALUES"-
1.025 DO PART 3 FOR I=1,11-
1.030 DO PART 4-
1.035 TYPE "SELECT RUNNING MODE"-
1.036 TYPE " M=0 FOR COMPUTING HEAT REJECTION"-
1.040 TYPE " M=1 FOR LENGTH VARIATION"-
1.041 TYPE " M=2 FOR SYST. PARAMETER VARIATION"-
1.042 TYPE " M=3 FOR TUBE NUMBER VARIATION"-
1.043 TYPE " M=4 FOR THICKNESS VARIATION"-
1.050 ACCEPT M-
1.055 IF M EQ 1 THEN GO TO 1.300-
1.060 TYPE "SPECIFY GEOMETRY, LENGTH, THICKNESS, DEPTH"-
1.065 ACCEPT L,T,H-
1.070 TYPE "SPECIFY NUMBER OF TUBES AND DIAMETER"-
1.075 ACCEPT N,D-
1.076 IF M EQ 2 THEN GO TO 1.400-
1.080 TYPE "SPECIFY SYSTEM PARAMETER"-
1.085 ACCEPT PHI-
1.090 TYPE "SPECIFY FLUID PRANDTL NO. AND"-
1.091 TYPE " RATIO OF FIN TO FLUID THERM. COND."-
1.095 ACCEPT PR,THCR-
1.096 TYPE "SPECIFY WEIGHT AUGMENTATION FACTOR"-
1.097 ACCEPT WA-
1.100 RE = 4.0*PHI*THCR/(3.141593*N*D*PR)-
1.101 LIMIT = 0-
1.105 IF PR LT 0.1 THEN GO TO 1.120-
1.106 IF (RE-2300.0) 1.107,1.110,1.110-
1.107 Z1 = PR*RE*D/L-
1.108 U = (3.65+(0.0668*Z1)/(1.0+0.045*Z1**0.667))*4.0/Z1-

```

```

1.109 GO TO 1.125-
1.110 U = 4.0*L/D*0.023*RE**(-0.2)*PR**(-0.7)-
1.115 GO TO 1.125-
1.120 U = 4.0*L/D*(5.0+0.025*(RE*PR)**0.8)/(RE*PR)-
1.125 A = L*H-
1.126 IF U LT 127.0 THEN GO TO 1.130-
1.127 B = 1.0-
1.128 GO TO 1.135-
1.130 B = 1.0-EXP(-U)-
1.135 SET THETA = 1.0-
1.140 FNC = PHI*(H*THETA/N)**2*THETA/(2.0*T)-
1.145 DO PART 6-
1.150 Y = 2.0*A*FETA(11)*THETA**4-(1.0-THETA)*B-
1.155 DEDT = FETA(11)*3.0*FNC/THETA-
1.160 DY = 2.0*A*THETA**3*(THETA*DEDT+4.0*FETA(11))+B-
1.165 DTHETA = Y/DY-
1.167 LIMIT = LIMIT+1-
1.168 IF LIMIT EQ 11 THEN STOP-
1.170 IF ABS(DTHETA) LT 1.0E-04 THEN GO TO 1.185-
1.175 THETA = THETA-DTHETA-
1.180 GO TO 1.140-
1.185 THETA0 = 1.0-THETA-
1.186 IF LIMIT EQ 1 THEN THETA0 = DTHETA-
1.190 Q = B*THETA0-
1.195 MASS = T*Q/(2.0*FETA(11)*THETA**4)-
1.200 MOQ = MASS/Q-
1.201 PSITOT = MOQ*(1.0+WA*D**2/(H*T))-
1.205 PRINT FORM 1.210, RE,U,THETA-
1.206 PRINT FORM 1.212, FNC,FETA(11),PSITOT-
1.207 PRINT FORM 1.211, Q,MASS,MOQ-
1.210 FORM RE=#####, U=#####, THETA=#####,-
1.211 FORM Q=#####, M=#####, MOQ=#####-
1.212 FORM NC=#####, ETA=#####, PSITOT=#####-
1.230 IF M EQ 0 THEN GO TO 1.035-
1.231 IF M GT 1 THEN GO TO 1.233-
1.232 IF MC LT NL THEN GO TO 1.345 ELSE STOP-
1.233 IF M GT 2 THEN GO TO 1.245-
1.234 IF MC LT NPHI THEN GO TO 1.445 ELSE STOP-
1.235 TYPE "MORE PROGRAMMING REQUIRED, SEE W. WULFF"-
1.299 STOP-

```

```

1.300 TYPE "SPECIFY THICKNESS, DEPTH, INITIAL * FINAL"-
1.305 TYPE "    LENGTHS AND NO. OF STEPS"-
1.310 ACCEPT T,H,LC,LEND,NL-
1.315 L = LC-
1.320 PRINT FORM 1.325, L-
1.325 FORM FOR L = @@@@@@@@@@-
1.330 DL = (LEND-LC)/(NL-1)-
1.335 MC = M-
1.340 GO TO 1.070-
1.345 MC = MC+1-
1.350 L = L+DL-
1.355 TYPE "    "-
1.360 GO TO 1.101-
1.400 TYPE "SPECIFY RANGE OF SYST. PARAMETER AND THE"-
1.405 TYPE "    NUMBER OF STEPS"-
1.410 ACCEPT PHIMAX,PHIMIN,NPHI-
1.415 PHI = PHIMIN-
1.430 DPHI = (PHIMAX-PHIMIN)/(NPHI-1)-
1.435 MC = M-1-
1.440 GO TO 1.090-
1.445 MC = MC+1-
1.450 PHI = PHI+DPHI-
1.455 TYPE "    "-
1.460 GO TO 1.100-
2.000 READ FC[1]-
3.000 READ ETA[1]-
4.000 H1 = FC[2]-FC[1]-
4.005 H2 = FC[3]-FC[1]-
4.010 X1 = (H2/H1)*ETA[2]-
4.015 X2 = -(H2/H1-H1/H2)*ETA[1]-
4.020 X3 = -H1/H2*ETA[3]-
4.025 DETA[1] = (X1+X2+X3)/(H2-H1)-
4.026 TYPE "    NC          ETA          DETA/DNC"-
4.027 FORM @@@@@@@@@@@@@, ....., @@@@@@@@@@@@@-
4.028 PRINT FORM 4.027, FC[1], ETA[1], DETA[1]-
4.030 DO PART 5 FOR I=2,10-
4.035 H1 = FC[10]-FC[9]-
4.040 H2 = FC[11]-FC[9]-
4.045 X1 = (2.0-H1/H2)*ETA[11]-
4.050 X2 = -H2/H1*ETA[10]-

```

```

4.055 X3 = -(2.0-H1/H2-H2/H1)*ETA[9]-
4.060 DETAC[11] = (X1+X2+X3)/(H2-H1)-
4.065 IF ETAC[11] LT 1.0E-06 THEN DETAC[11] = 0.0-
4.070 PRINT FORM 4.027, FC[11], ETAC[11], DETAC[11]-
4.075 X1X2 = FC[3]-FC[2]-
4.080 AAA = -(FC[2]*ETA[3]-FC[3]*ETA[2]+X1X2)-
4.085 AAA = AAA/(FC[2]*FC[3]*X1X2)-
4.090 BBB = (1.0-ETA[2]-AAA*FC[2]**2)/FC[2]-
4.095 TYPE "FOR FNC NEAR ZERO, 1-ETA = AAA*FNC**2+BBB*FNC"-
4.100 PRINT FORM 4.105, AAA, BBB-
4.105 FORM AAA = @@@@@@@@@@@@@, BBB = @@@@@@@@@@@@@-
5.000 H2 = FC[I+1]-FC[I-1]-
5.005 H1 = FC[I]-FC[I-1]-
5.010 X1 = (H1/H2)*ETA[I+1]-
5.015 X2 = -(2.0-H2/H1)*ETA[I]-
5.020 X3 = (2.0-H1/H2-H2/H1)*ETA[I-1]-
5.025 DETAC[I] = (X1+X2+X3)/(H2-H1)-
5.026 PRINT FORM 4.027, FC[I], ETAC[I], DETAC[I]-
6.000 IF FNC LT FC[3] THEN GO TO 6.020-
6.005 DO PART 7 FOR I=1,11-
6.010 DO PART 8 FOR I=1,11-
6.015 DO PART 9 FOR I=2,11-
6.016 GO TO 6.030-
6.020 FETAC[11] = -((AAA*FNC+BBB)*FNC-1.0)-
6.025 FDETA[11] = -(2.0*AAA*FNC+BBB)-
6.030 REM REURN-
7.000 FETAC[I] = ETAC[I]-
7.005 FDETA[I] = DETAC[I]-
8.000 FFC[I] = FC[I]-FNC-
9.000 DO PART 10 FOR J=I,11-
9.005 DO PART 11 FOR J=I,11-
10.000 XFC = FC[J]-FC[I-1]-
10.005 F1[J] = (FETAC[I-1]*FFC[J]-FETAC[J]*FFC[I-1])/XFC-
10.010 F2[J] = (FDETA[I-1]*FFC[J]-FDETA[J]*FFC[I-1])/XFC-
11.000 FETAC[J] = F1[J]-
11.005 FDETA[J] = F2[J]-
6.030 REM RETURN-

```



## APPENDIX E

The Fin-To-Tube Shape Factor

A closed-form integration for the view factor of the fin with respect to the tube was carried out by Mr. Yao. This view factor occurs in Eqs. 6.15 and 6.16. Only the final results are given here.

The reader should recognize that some of the symbols defined below (Eqs. D.1 through 6) apply only here.

Let  $(x_f, y_f, z_f)$  designate the position of the center of an area element  $A_f$  on the fin and  $z_m$  the same on the tube. Let  $r_e$ ,  $s_t$  and  $s_r$  represent, respectively, the outer tube radius, the fin tip and the fin root thickness, and let the fin height be given as  $H$ .

Then, with

$$\rho = \sqrt{x_f^2 + y_f^2} \quad (D.1)$$

$$\beta = \text{arc tg } \frac{y_f}{x_f} \quad (D.2)$$

$$\alpha = \text{arc tg } \frac{s_r - s_t}{2r_c} \quad (D.3)$$

$$\phi = \arctg \frac{s_r}{2r_e} \quad (D.4)$$

$$a = \rho^2 + r_e^2 + (z_f - z_m)^2 \quad (D.5)$$

$$\phi^* = \arcsin \frac{r_e}{\rho} \quad (D.6)$$

one obtains first

$$Z_1 = \ln \frac{a-2r_e \rho \cos(\phi^*-\beta)}{a-2r_e \rho \cos(\phi-\beta)} + \frac{a-2r_e^2 \cos(\alpha+\beta)}{a-2r_e \rho \cos(\phi^*-\beta)} - \frac{a-2r_e^2 \cos(\alpha+\beta)}{a-2r_e \rho \cos(\phi-\beta)} \quad (D.7)$$

$$Z_2 = \frac{8r_e^2 \rho^2 (z_f - z_m)^2 + 4\rho^2 r_e^2 a - a^3}{2(a^2 - 4r_e^2 \rho^2)^{3/2}} \quad (D.8)$$

$$Z_3 = \arcsin \frac{2r_e \rho - a \cos(\phi^*-\beta)}{a-2r_e \rho \cos(\phi^*-\beta)} - \arcsin \frac{2r_e \rho - a \cos(\phi-\beta)}{a-2r_e \rho \cos(\phi-\beta)} \quad (D.9)$$

$$Z_4 = \frac{2a(z_f - z_m)^2 - a^2 + 4r_e^2 \rho^2}{a^2 - 4r_e^2 \rho^2} \quad (D.10)$$

$$Z_5 = \frac{r_e \rho \sin(\phi^* - \beta)}{a - 2r_e \rho \cos(\phi^* - \beta)} - \frac{r_e \rho \sin(\phi - \beta)}{a - 2r_e \rho \cos(\phi - \beta)} \quad (D.11)$$

The final result is

$$SS = \Delta A_1 \Delta z \left\{ \frac{-Z_1}{4\pi\rho} + \frac{\sin(\alpha + \beta)}{2\pi\rho} \left[ \frac{\phi^* - \phi}{2} + Z_2 Z_3 Z_4 Z_5 \right] \right\} \quad (D.12)$$

This expression contains all the geometric relations that are required for fin-channel radiative interaction. It needs to be evaluated only once for every fin element.

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STUDY OF DESIGN PARAMETERS OF SPACE BASE  
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Final Report

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Part II: Space radiator simulation manual for  
computer code.

Part III: Space radiator simulation system  
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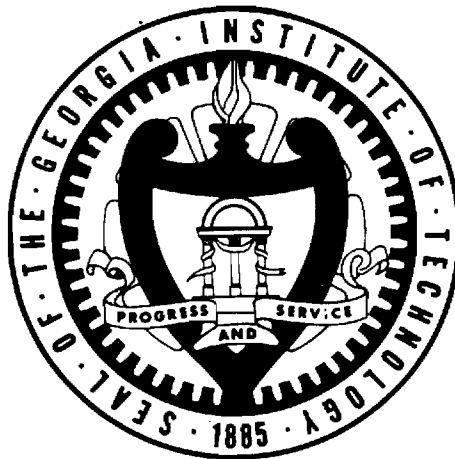
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April, 1972

GEORGIA INSTITUTE OF TECHNOLOGY  
School of Mechanical Engineering  
Atlanta, Georgia

Final Report  
Part I

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Stothe P. Kezios, Ph.D.  
Director, School of Mechanical Engineering

## FOREWORD

This report covers part of the work performed during the second year phase of a two year research project, at the School of Mechanical Engineering at the Georgia Institute of Technology in Atlanta, Georgia, for the NASA Manned Spacecraft Center, Houston, Texas. The contract designation is NAS 9-10415 and the project title is "Study of Design Parameters of Space Base and Shuttle Heat Rejection Systems." The project was monitored by Dr. W.E. Simon of the Power Generation Branch of NASA MSC, Houston, Texas, and was performed by Dr. W.Z. Black and Dr. W. Wulff as Co-Investigators. The project resulted in one Annual Report [2]\* and two Final Reports, the first one of which covers a detailed and rigorous space radiator simulation analysis [1] and includes a computer program users manual. The second Final Report is this report.

Although the study of system parameters and the system optimization were originally conceived to be performed on the basis of the rigorous radiator system simulation [1], the growing complexity of this simulation soon gave rise to the need for a simplified analysis. The simplified system simulation was later expanded into systematic optimization procedures. Both the simplified simulation and the optimization procedures serve to supplement and support the originally intended system parameter study [1].

The work presented here was supported by the contributions of computer coding by Mr. Richard J. Huntley and Mr. Wallace W. Carr, both Graduate Research Assistants and M.S. Candidates.

---

\*Numbers in brackets refer to the Bibliography at the end of this report.

## SUMMARY

A simplified radiator system analysis was performed to predict steady-state radiator system performance. The system performance was found to be describable in terms of five non-dimensional system parameters. The governing differential equations are integrated numerically to yield the enthalpy rejection for the coolant fluid.

The simplified analysis was extended to produce firstly the derivatives of the coolant exit temperature with respect to the governing system parameters and secondly a procedure to find the optimum set of system parameters which yields the lowest possible coolant exit temperature for either a given projected area or a given total mass. The process can be inverted to yield either the minimum area or the minimum mass, together with the optimum geometry, for a specified heat rejection rate.

The major accomplishments of the simplified radiator system analysis are:

- (1) the reduction of the number of necessary systems parameters from twelve or more to six,
- (2) the graphical representation of system performance in terms of non-dimensional groups, suitable to aid in the design of radiative heat rejection systems, and
- (3) an efficient computer code suitable for preliminary performance prediction.

The accomplishments of the systematic optimization analysis are two computer codes which perform iterative optimizations processes leading to the maximum heat rejection for either a given projected fin-plus-tube area or a given total system mass.

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# NOMENCLATURE

$A^*$	Normalized projected radiator area, Eq. 38
$A$	Coefficient matrix in Eq. 49
$A_{ij}$	Elements of coefficient matrix $A$
$a_i$	Coefficients of power polynomial
$B_{ij} = \partial^2 \theta_b / \partial x_i \partial x_j$	Second-order derivatives of $\theta_b$
$c$	Constant in Nusselt number relation, Eq. 35
$c_p$	Coolant fluid specific heat (Btu/(lbm R))
$d$	Tube diameter (ft)
$d^* = d/L_o$	Normalized tube diameter
$F_{fs}$	Fin to sink view factor, Eq. 15
$f$	Blockage coefficient, defined by Eq. 16
$H$	Fin height, from base to tip (ft)
$H^* = H/L_o$	Normalized fin height
$\bar{h}_c$	Average convective film coefficient for convection from fluid to tube (Btu/(hr ft <sup>2</sup> R))
$i$	Number of radiatively active sides
$k$	Thermal conductivity (Btu/(hr ft R)) of structural material
$k_f$	Thermal conductivity (Btu/(hr ft R)) of coolant fluid
$L$	Tube length (ft)
$L_o$	Reference length (ft), defined by Eq. 37
$L^* = L/L_o$	Normalized tube length
$M$	Parameter, defined by Eq. 12 (viewfactor augmentation)
$M$	Structural mass, per fin-tube element (lbm)
$M_o = \rho L_o^3$	Reference mass (lbm)

$M_f$	Coolant fluid mass (lbm)
$M^* = M / (M_o \phi_1)$	Normalized structural mass
$M_f^* = M_f / (M_o \phi_1)$	Normalized coolant mass
$M_{tot}^* = M_f^* + M^*$	Normalized total mass
$m$	Exponent on $N_{Re}$ in Nusselt number relation, Eq. 35
$\dot{m}$	Coolant mass flow rate (lbm/hr)
$N$	Tube to sink view factor, Eq. 17
$N_c = \bar{N}_c \Theta^3$	Conduction parameter, Eq. 14
$\bar{N}_c, N_c^*$	Reference conduction parameter, defined by Eqs. 20 and 58*
$N_{Gr}, N_{Gr}^*$	Graetz number, Eqs. 23 and 55
$N_{Nu}, N_{Nu}^*$	Nusselt number, Eqs. 8, 9, 10, 35 and 57*
$N_{Pr}$	Prandtl number, Eq. 7
$N_{Re}, N_{Re}^*$	Reynolds number, Eqs. 6 and 56*
$n$	Number of coolant channels
$n$	Exponent on $N_{Pr}$ in Nusselt number relation, Eq. 35
$p$	Exponent on $d/L$ -ratio in Nusselt number relation, Eq. 35
$q''$	Incident radiant heat flux (Btu/(hr ft <sup>2</sup> ))
$r$	Number of constraints
$T$	Absolute temperature (R)
$t$	Fin panel thickness (ft)
$t^* = t/L_o$	Normalized fin panel thickness
$U$	System parameter, defined by Eq. 22
$V$	System parameter, ratio of convective to radiative resistances, Eq. 21
$\tilde{X}$	Vector whose components are the system parameters $x_i$

---

\*Superscripted stars on symbol  $N$  indicate non-dimensional groups evaluated by replacing all dimensions  $L$ ,  $H$ ,  $d$  and  $t$  by  $L_o$ .



$x_i$	General system parameters, $x_1 = U$ , $x_2 = V$ , $x_3 = \bar{N}_c$ , $x_4 = \lambda$
$\Delta \underline{X}$	Computed changes of $\underline{X}$ , in iteration process
$Y$	Generalized distance from optimum, defined by Eqs. 49 and 50
$y_i$	Components of $\underline{Y}$ , defined by Eqs. 50
$z$	Distance along the tube (ft)

#### GREEK SYMBOLS

$\alpha_s$	Solar absorptance
$\delta$	Incremental component vector defining a neighborhood about potential optimum, Eq. 47
$\epsilon$	Surface emittance
$\zeta = z/L_o$	Normalized axial distance
$\eta$	Effectiveness of unobstructed fin
$\bar{\eta}$	Combined fin and tube effectiveness
$\theta = T/T_o$	Normalized temperature
$\lambda = 4H/d$	
$\sigma$	Stefan Beltzmann constant $\sigma = 0.1714 \times 10^{-8}$ Btu/(hr ft <sup>2</sup> R <sup>4</sup> )
$\mu$	Dynamic viscosity (lbm/(sec ft))
$\rho$	Structural material density (lbm/ft <sup>3</sup> )
$\rho_f$	Fluid density (lbm/ft <sup>3</sup> )
$\phi_1$	Parameter, defined by Eq. 42
$\phi_2$	Parameter, defined by Eq. 44
$\phi_3$	Parameter, defined by Eq. 63
$\psi_i$	Constraints, Eqs. 39, 40, 41 and 46

#### SUBSCRIPTS

$b$	Fin base
$c$	Conduction

e	Channel exit
f	Fluid, bulk property
i,j	Component subscripts
k	Iteration step counting index
m	at optimum
s	sink
w	Fluid, at tube wall
o	Channel inlet

## I. INTRODUCTION

A large-scale, complete and rigorous computer simulation of space radiator systems was developed under the same contract as the work presented here [1].\* The computer code consists of over fifty program units and is capable of simulating transient as well as steady-state radiator system performances under prescribed time-dependent operational and environmental conditions. The program accommodates both gaseous and liquid coolant fluids with any consistently prescribed set of thermodynamic and transport properties. In principle, this large-scale computer program could serve to not only predict radiator system performance but also to optimize certain design parameters via enumeration of performance characteristics, associated with selected parameter combinations

Any large computer simulation must be tested during its development; the greater the number of independent verification modes the greater will be the confidence in the program performance. In addition to the tests described in Reference [1], a simplified radiator system analysis was developed to verify the large-scale computer program performance. This simplified analysis served later (1) to extend the rigorous analysis to a wider class of radiator geometries and operating conditions [1] than would have been possible otherwise and (2) to develop systematic optimization procedures in support of system parameter studies on the basis of the rigorous computer simulation.

The essence of the simplified analysis lies in the recognition of the dominant performance characteristics and in the parameter reduction through normalization of the governing differential equations. The optimization is based on standard requirements of extrema subject to suitable constraints. These requirements are imposed on the governing differential equations prior to their numerical integration.

Chapter II below is a discussion of the simplified analysis and is followed by the presentation of the optimization analysis in Chapter III.

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\*Numbers in brackets refer to the Bibliography.

## II. SIMPLIFIED RADIATOR SYSTEM ANALYSIS

### A. Objective and Background

The purpose of this analysis is to provide an efficient process of least complexity which serves to describe the steady-state performance of a space radiator system. The radiator system is described in detail in the following chapter.

The analysis is intended to accommodate the essential features of the radiator system performance but to require less computational effort than the rigorous system simulation described in Reference [1]. Finally, the simplified analysis should serve as the basis for systematic optimization of the radiator system.

A first simplified analysis was developed during the first year of the contract period [2], also in support of the rigorous computer analysis, and for the purpose of treating approximately some radiator system geometries which deviate from the basic geometry underlying the rigorous system analysis, such as cylindrical shell radiator structures, asymmetrically loaded coolant channels etc. [1]. This analysis was based on two global energy balances, one for the coolant fluid and one for the radiator panel, and included the two thermal resistances associated with the energy transfer from the fluid to the tube and from the radiator into space. The mean tube wall temperature (and with it, the fin base temperature) was obtained from a single, transcendental equation (Eq. D.11, [2]) and the cooling capacity was found explicitly in terms of the mean tube wall temperature (Eq. 1.1 or 3, [2]). Agreement between this simplified analysis and the full-scale computer simulation was found to be approximately 4%.

The search for an optimum radiator geometry on the basis of the first simplified analysis lead to the conclusion that for optimum radiator geometries neighboring coolant channels should be expected to be close enough to each other so as to obstruct each other's, and the fin panel's, view of space. Moreover, the rigorous computer simulation indicated that the tube wall temperature changes significantly in the direction of the flow even though conduction parallel to the flow direction, both in the fluid, in the

tube wall and in the fin, is negligible. These effects are not accounted for in the first simplified analysis.

The new simplified analysis presented here, for the prediction of steady-state radiator heat rejection performance, takes into account

- (i) temperature variations within the fluid, tube and fin, along the direction of the flow,
- (ii) temperature variation within the fin, normal to the direction of the flow,
- (iii) direct radiative interaction between fin panel and space, tube and space, fin and tubes, tube and neighboring tubes (by approximation),

but does not take into account

- (i) thermal conduction in the direction of the coolant flow, both in the fin-tube structure and the fluid
- (ii) temperature variation of thermal properties,
- (iii) end effects at the coolant fluid manifold,
- (iv) cross-sectional changes in the fin panel,
- (v) secondary reflections of radiant heat.

The analysis reduces to an initial value problem consisting of two ordinary, first-order, coupled differential equations, with one of its initial conditions given through a transcendental equation. The new formulation is suitable for optimization.

### B. Description of Radiator System

The radiator system considered here consists of a given number of parallel coolant channels, equally spaced in one plane, and connected by rectangular fin panels which have constant thickness and are symmetric with respect to the plane through the tube areas.

The coolant channels are taken to be thin-walled tubes of circular cross-section, with diameter  $d$  and length  $L$ . Substitution of the hydraulic diameter for the diameter permits readily the inclusion of channels with non-circular cross-sections, except for possible effects from radiative interaction between tube, fin and environment.

Let the distance between tube centers be designated by  $d + 2H$  and the fin panel thickness by  $t$ . Then the radiator can be considered to consist of the given number  $n$  of identical fin elements as shown in Figure 1.

The coolant fluid enters the channel at  $z = 0$  with time-invariant temperature  $T_f = T_o$ . Both the fluid temperature  $T_f(z)$  and the fin base temperature  $T_b(z)$  decrease along the tube provided the equivalent sink temperature  $T_s$  is less than  $T_o$ . The coolant fluid emerges from the radiator at  $z = L$  with the exit temperature  $T_e$ .

The objective of the analysis is to predict the rate of heat rejection  $\dot{m} c_p (T_o - T_e)$  per tube for a given mass flow rate  $\dot{m}$  per tube and given fluid properties, namely density  $\rho$ , specific heat  $c_p$ , thermal conductivity  $k_f$ , dynamic viscosity  $\mu$ , and given properties of the fin, that is, surface emittance  $\epsilon$ , solar absorptance  $\alpha_s$ , and thermal conductivity  $k$ .

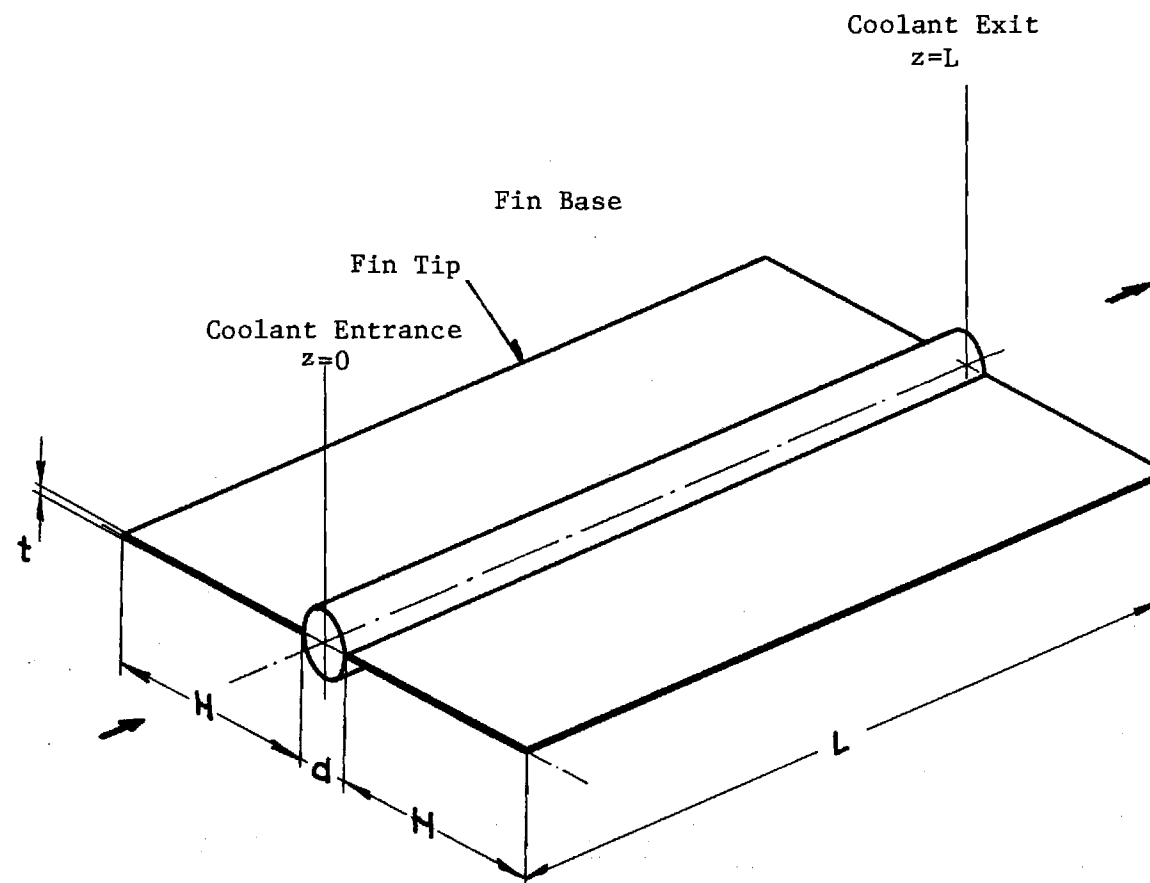


Figure 1. Geometry of Fin System Element

### C. Governing Equations

#### 1. Conservation of Energy

Setting the local change of the fluid enthalpy flux first equal to the local heat exchange per unit tube length between fluid and tube wall, and then equal to the radiative heat rejection per unit tube length one obtains, for  $i$  effective sides of the radiator:

$$- \dot{m} c_p \frac{dT_f}{dz} = \pi d \bar{h}_c (T_f - T_b) \quad (1)$$

$$\pi d \bar{h}_c (T_f - T_b) = 2iH\epsilon\sigma\bar{\eta}(T_b^4 - T_s^4) \quad (2)$$

where  $\bar{h}_c$  and  $\sigma$  represent, respectively, the convective film coefficient and the Stefan-Boltzmann constant. The symbol  $\bar{\eta}$  represents the overall tube-and-fin effectiveness and is evaluated in Section 3. The convective film coefficient is computed from the Nusselt number  $N_{Nu}$

$$\bar{h}_c = N_{Nu} \frac{k_f}{d} \quad (3)$$

the evaluation of which is deferred to Section 2. The equivalent sink temperature  $T_s$  is computed from the known incident normal radiant heat flux  $q''$  by

$$T_s = \sqrt[4]{\frac{\alpha_s}{\epsilon} \frac{q''}{\sigma}} \quad (4)$$

The first-order differential equation, Eq. 1, is subject to the initial condition at  $z = 0$

$$T_f(0) = T_o \quad (5)$$



Equations 1 and 2 determine the two dependent variables  $T_f(z)$  and  $T_b(z)$ . Integration of Eq. 1 from  $z = 0$  to  $z = L$  gives the unknown fluid exit temperature  $T_e$ .

## 2. Convective Film Coefficient

The Nusselt number  $N_{Nu}$  in Eq. 3 depends on the Reynolds number  $N_{Re}$ , the Prandtl number  $N_{Pr}$  and the  $d/L$  ratio.

$$N_{Re} = \frac{4\dot{m}}{\pi d \mu} \quad (6)$$

$$N_{Pr} = \frac{\mu c_p}{k_f} \quad (7)$$

For laminar flow,  $N_{Re} \leq 2300$ , of non-metallic fluids,  $N_{Pr} \geq 0.1$ , Hausen [3] established the relation

$$N_{Nu} = \left[ 3.65 + \frac{0.0668 N_{Re} N_{Pr} \frac{d}{L}}{1 + 0.045 (N_{Re} N_{Pr} \frac{d}{L})^{2/3}} \right] \left( \frac{\mu_f}{\mu_w} \right)^{0.14} \quad (8)$$

The nomogram in Figure 2, taken from the VDI Waermeatlas,\* facilitates the estimate of the Nusselt number in accordance with Eq. 8.

For turbulent flow,  $N_{Re} > 2300$ , of non-metallic fluids,  $N_{Pr} \geq 0.1$ , similarity considerations lead to

$$N_{Nu} = 0.116 \left[ 1 + \left( \frac{d}{L} \right)^{2/3} \right] (N_{Re}^{2/3} - 125) N_{Pr}^{1/3} \left( \frac{\mu_f}{\mu_w} \right)^{0.14} \quad (9)$$

which is also presented in a nomogram in Figure 3.

Liquid metal convective heat transfer in tubes,  $N_{Pr} < 0.1$ , may be represented by the result of the work by Seban and Shimazaki [4] which is

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\*VDI-Verlag GMBH, Düsseldorf, West Germany

valid for  $(N_{Re} N_{Pr}) > 100$  and  $L/d > 60$

$$N_{Nu} = 5.0 + 0.025 (N_{Re} N_{Pr})^{0.8} \quad (10)$$

Other Nusselt number relationships may be more suitable in a particular case. Their selection does not affect the ensuing analysis because the Nusselt number calculation is part of the input data preparation, prior to the numerical evaluation of the solution.

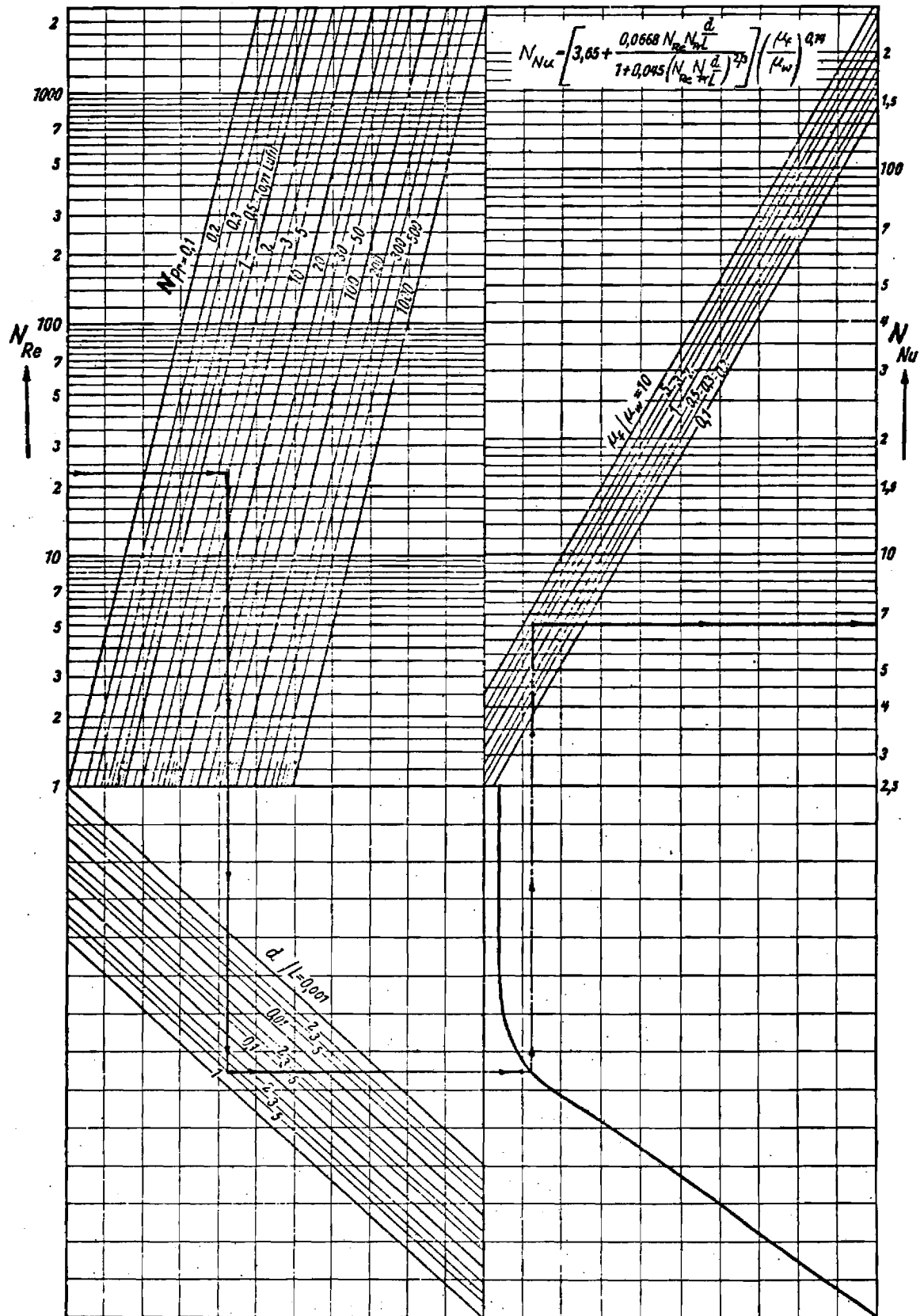


Figure 2. Nusselt Number Nomogram for Laminar Flow of Non-Metallic Coolant Fluids, Eq. 8. (VDI Waermeatlas)

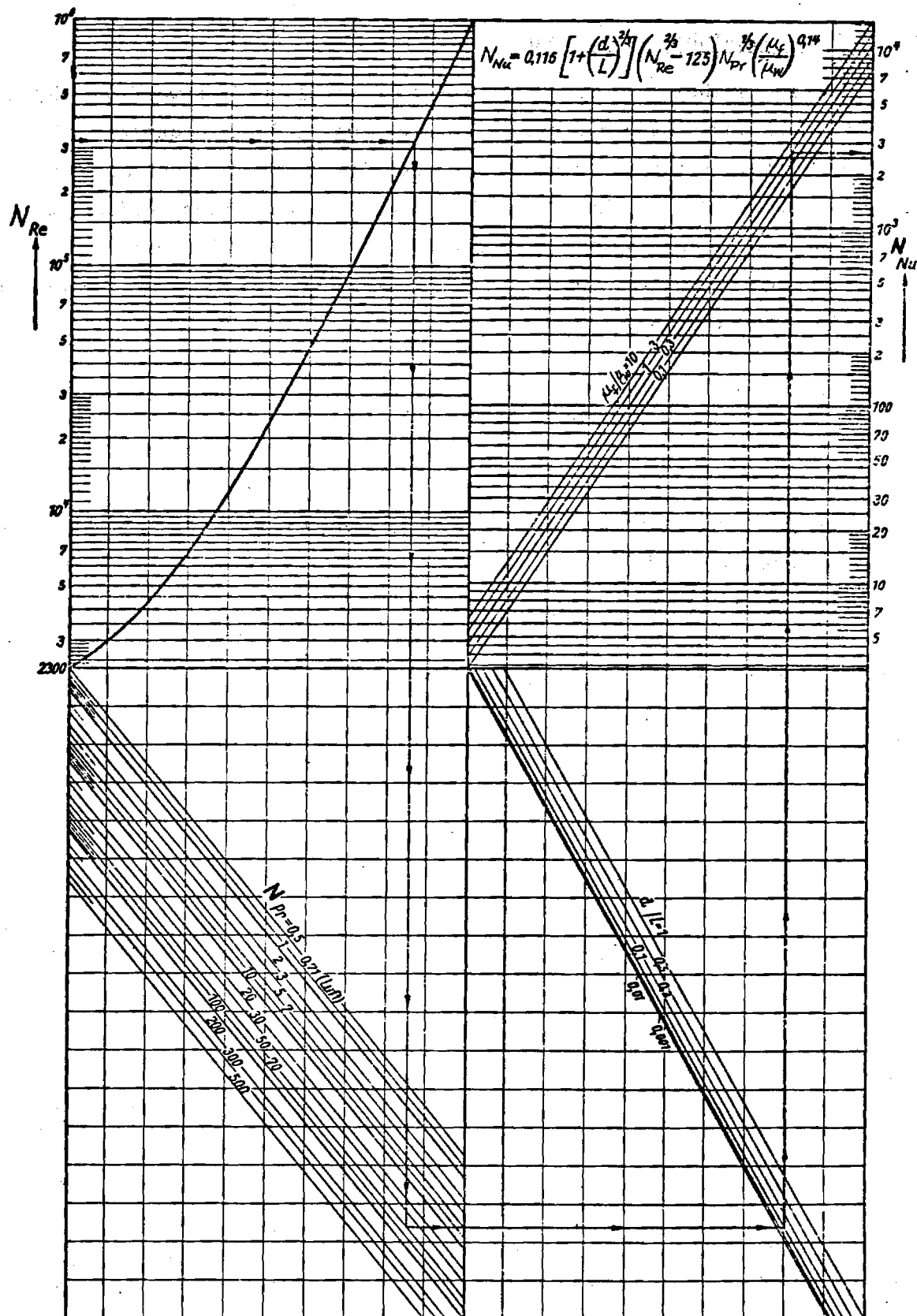


Figure 3. Nusselt Number Nomogram for Turbulent Flow of Non-Metallic Coolant Fluids, Eq. 9. (VDI Waermeatlas).

### 3. Total Fin-Tube Effectiveness

The symbol  $\bar{\eta}$  in Eq. 2 denotes the overall radiative fin-and-tube effectiveness and accounts not only for the non-uniform temperature distribution within the fin and normal to the flow direction, but also for the effects on the radiative heat transfer from the tube. These effects are due to the additional radiating tube area and the partial blockage of emission from the fin and the tubes by the tubes of the neighboring fin elements. Adding the contributions from the tube to that of the fin, one obtains

$$\bar{\eta}(z) = M(z) \cdot \eta(z) + N \quad (11)$$

Here,  $\eta(z)$  is the radiative fin effectiveness of the unobstructed fin, Eqs. 32, [5,6,7] and varies with the fin base temperature  $T_b(z)$ . The factor  $M$  is the product of the shape factor of the fin with respect to the sink,  $F_{fs}$ , and the corrective term  $f$  on  $\eta$  for the temperature profile distortion resulting from the tube-fin interaction [8]:

$$M = F_{fs}(\lambda) \cdot f(\lambda, N_c) \quad (12)$$

where

$$\lambda = \frac{4H}{d} \quad (13)$$

and

$$N_c = \frac{2\epsilon\sigma H^2}{kt} [T_b(z)]^3 \quad (14)$$

which is the well-known conduction parameter. Hottel's crossed-string method yields

$$F_{fs}(\lambda) = \left( \sqrt{\lambda(\lambda + 2)} + \arcsin \frac{1}{1 + \lambda} - \pi/2 \right) / \lambda \quad (15)$$

and the curve fit of data in Reference [8] resulted in

$$f(\lambda, N_c) = 1 - (N_c/\lambda)(0.1460 N_c - 0.02866) . \quad (16)$$

Finally, the symbol  $N$  in Eq. 11 represents the radiative heat rejection contributed by the tubes and is the shape factor of the tube with respect to the sink, multiplied by the tube-to-fin area ratio

$$N(\lambda) = \pi \left[ 1/2 + \left[ \lambda + 2 - \sqrt{\lambda(\lambda + 2)} - \arcsin \frac{1}{\lambda + 1} \right] / \pi \right] / \lambda \quad (17)$$

This completes the description of the overall fin-and-tube effectiveness. In summary it should be emphasized that axial conduction, radial tube wall temperature variation, kinetic and potential coolant energies, and end effects are ignored, that radiative interaction between tube and fin and between tubes is only approximate and that the representation of Eq. 16 is limited to  $T_s < 0.8 T_b$  (but could be extended, in principle).

#### D. Scaling Parameters

Effective analysis, numerical integration, graphical representation of numerical results and, ultimately, the system optimization makes the reduction of parameters mandatory. Introduce the non-dimensional

$$\text{axial distance} \quad \zeta = \frac{z}{L} \quad (18)$$

$$\text{temperature} \quad \theta = T/T_o \quad (19)$$

$$\begin{array}{l} \text{reference conduction} \\ \text{parameter} \end{array} \quad \bar{N}_c = \frac{2\epsilon\sigma T_o^3 H^2}{kt} \quad (20)$$

$$\begin{array}{l} \text{ratio of convective to} \\ \text{radiative resistance} \end{array} \quad V = \frac{1}{\pi} \frac{t}{H} \frac{k}{k_f} \frac{\bar{N}_c}{N_{Nu}} \quad (21)$$

$$\text{convection number} \quad U = \pi N_{Nu} / N_{Gz} \quad (22)$$

where the Graetz number  $N_{Gz}$  is defined by

$$N_{Gz} = (c_p \dot{m}) / (k_f L) = \frac{\pi}{4} \frac{d}{L} N_{Re} N_{Pr} \quad (23)$$

With these parameters and the  $\lambda$  defined by Eq. 13 one may recast the problem as previously established by Eqs. 1, 2 and 5 as given by Eqs. 24 through 26:

$$\frac{d\theta_f}{d\zeta} = -U(\theta_f - \theta_b) \quad (24)$$

$$\theta_f - \theta_b = V\bar{n}(\theta_b^4 - \theta_s^4) \quad (25)$$



subject to the initial condition at  $\zeta = 0$

$$\theta_f(0) = 1 \quad (26)$$

Integration of Eq. 24 subject to Eqs. 25 and 26 yields the coolant fluid exit temperature  $\theta_e$  sought

$$\theta_e = \theta_f(1) \quad (27)$$

and, consequently, this coolant fluid exit temperature is a function of five parameters

$$\theta_e = \theta_e(U, V, \bar{N}_c, \lambda; \theta_s) \quad (28)$$

The first four parameters correspond to the geometrical dimensions  $L$ ,  $H$ ,  $t$  and  $d$  of the radiator system and  $\theta_s$  represents the environment.

### E. Solution

Equation 25 is transcendental because of  $\bar{\eta}(\theta_b)$  and would require iterative solutions at every step of the numerical integration of Eq. 24. It is more economical to derive a second differential equation for  $\theta_b$  by differentiating Eq. 25

$$\frac{d\theta_b}{d\zeta} = \frac{-U}{\frac{1}{\theta_f - \theta_b} + \frac{1}{\bar{\eta}} \frac{d\bar{\eta}}{d\theta_b} + \frac{4\theta_b^3}{\theta_b^4 - \theta_s^4}} \quad (29)$$

and to obtain the initial condition for Eq. 29 through a single iterative process (Newton Raphson method) from

$$V\bar{\eta}(\theta_b^4 - \theta_s^4) + \theta_b = 1 \quad \text{at} \quad \zeta = 0 \quad (30)$$

- The derivative  $d\bar{\eta}/d\theta_b$  is obtained from Eqs. 11 and 16

$$\begin{aligned} \frac{d\bar{\eta}}{d\theta_b} &= \frac{d\bar{\eta}}{dN_c} \frac{dN_c}{d\theta_b} = 3 \bar{N}_c \theta_b^2 \frac{\partial}{\partial N_c} (Mn) \\ &= 3 \bar{N}_c \theta_b^2 \left\{ \eta_{Ffs} \left[ 0.02866 - 0.2920 \bar{N}_c \theta_b^3 \right] / \lambda \right. \\ &\quad \left. + M \frac{d\eta}{dN_c} \right\} \end{aligned} \quad (31)$$

which can be evaluated once the fin effectiveness  $\eta(N_c)$ , that is the ratio of the actual power loss from an unobstructed (tubless) fin panel to the power loss from an ideal, unobstructed fin of infinite thermal conductivity, is represented by a power polynomial:

$$\eta(N_c) = \sum_{i=0}^6 a_i (N_c)^i \quad (32a)$$

where

$$\begin{aligned} a_0 &= 1.000\ 000 \\ a_1 &= -1.163\ 143 \\ a_2 &= 1.478\ 836 \\ a_3 &= -1.267\ 550 \\ a_4 &= 0.632\ 522\ 3 \\ a_5 &= -0.162\ 706\ 7 \\ a_6 &= 0.016\ 542\ 23 \end{aligned}$$

Equation 32 is valid for  $0 \leq N_c \leq 2.5$ . For greater values,  $N_c > 2.5$

$$\eta(N_c) = 0.686\ 609\ 5\ e^{-0.229\ 771\ 8} \quad (32b)$$

Equations 24 and 29 have been integrated by a fourth-order Runge-Kutta procedure [2], subject to initial conditions given by Eqs. 26 and 30. The result of the integration yields primarily  $\Theta_e$  in the form of Eq. 28.

## F. Results

A typical computer print-out of the results is shown in Figure 4. First are presented the five parameters  $\{\theta_s; U, V, \bar{N}_c, \lambda\}$  of Eq. 28, as read-in. Following that is a table in which are listed, as functions of axial distance measured from the coolant fluid inlet, the fluid temperature  $\theta_f = \text{THETAF}$ , the fin base temperature  $\theta_b = \text{THETAB}$ , next the local conduction parameter  $N_c = \text{NC}$ , the local total fin-plus-tube effectiveness  $\bar{\eta} = \text{ETABAR}$ , the local fin effectiveness  $\eta = \text{ETA}$ , the nondimensional axial position  $\zeta = \text{ZETA}$  and, finally, the number of integration steps required to reach the particular axial position. The number of integration steps depends on the chosen error limits and the rates of variable changes.

The time rate of heat rejection per fin element (tube) is to be computed from

$$\dot{m} c_p (T_o - T_e) = \dot{m} c_p T_o (1 - \theta_e) \quad (32)$$

Parameter studies are presented in Figures 5 through 9. The non-dimensional fluid exit temperature is plotted versus non-dimensional fin panel area with the temperature  $\theta_s$  (Fig. 5), the non-dimensional tube spacing  $\lambda$  (Figs. 6 and 7) and the reference conduction parameter  $\bar{N}_c$  (Figs. 8 and 9) as parameters.

02 THETA-S = .800000  
 U = .100000+01  
 V = .200000+01  
 FNCBAR = .200000+00  
 LAMBDA = .250000+02

THETA1	THETA3	NC	ETABAR	ETA	ZETA	STEP NO
1.000000	.833159	.117763	.964240	.881541	.00	1
.984460	.835315	.116568	.965251	.882615	.10	2
.976137	.832677	.115468	.966185	.883569	.20	4
.966937	.830231	.114453	.967048	.884452	.30	5
.944763	.827954	.113518	.967846	.885267	.40	6
.933550	.825862	.112656	.968583	.886020	.50	7
.923266	.823916	.111861	.969263	.886716	.60	8
.913669	.822113	.111128	.969892	.887359	.70	9
.904875	.820443	.110453	.970473	.887953	.80	10
.896764	.818898	.109830	.971009	.888501	.90	11
.889284	.817467	.109253	.971505	.889008	1.00	12

FIGURE 4. TYPICAL COMPUTER PRINT-OUT FOR RADIATOR SYSTEM PERFORMANCE

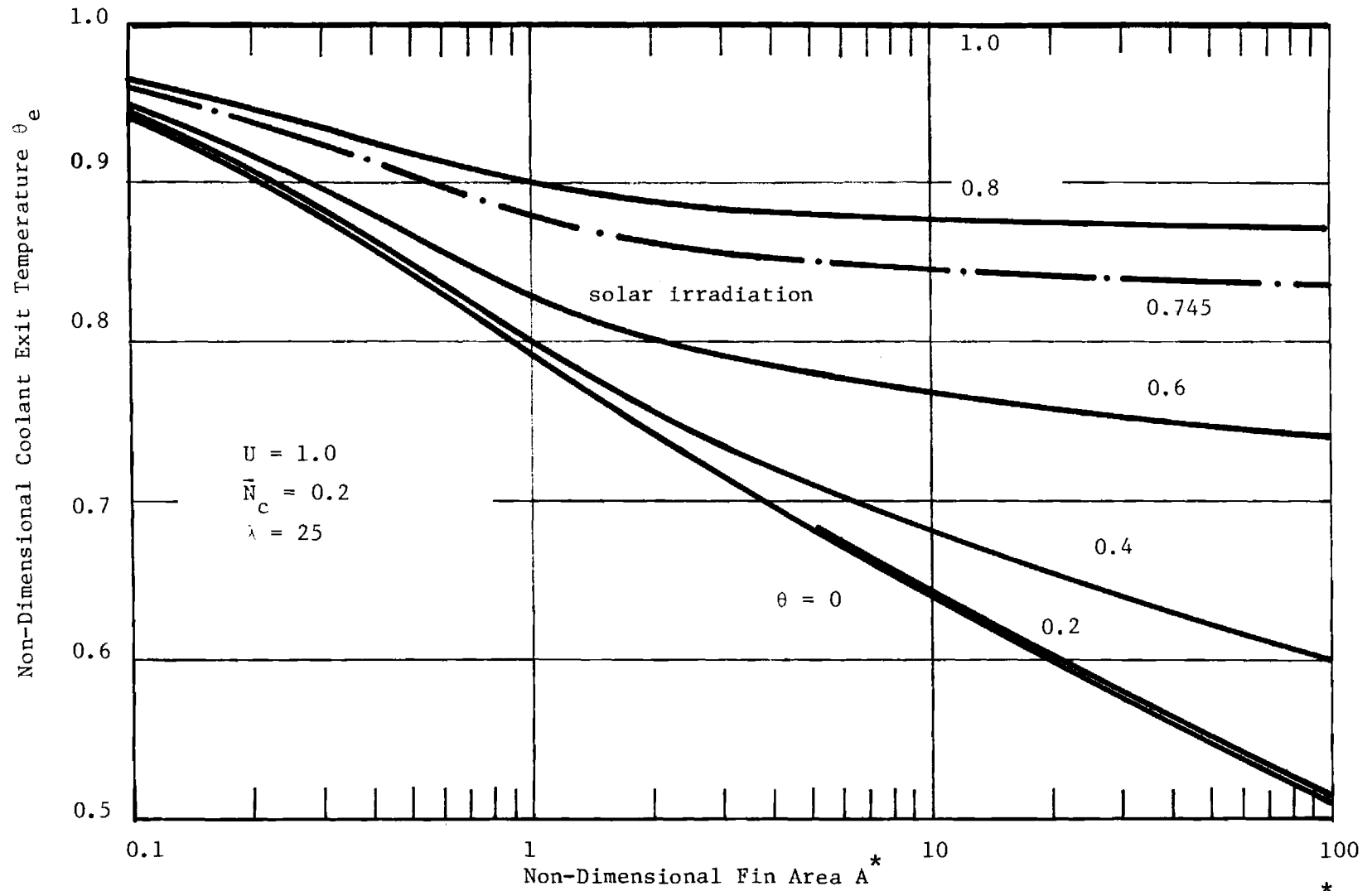


Figure 5. Non-Dimensional Coolant Exit Temperature  $\theta$  vs. Non-Dimensional Fin Panel Area  $A^*$  with Sink Temperature  $\theta_s$  As Parameter.

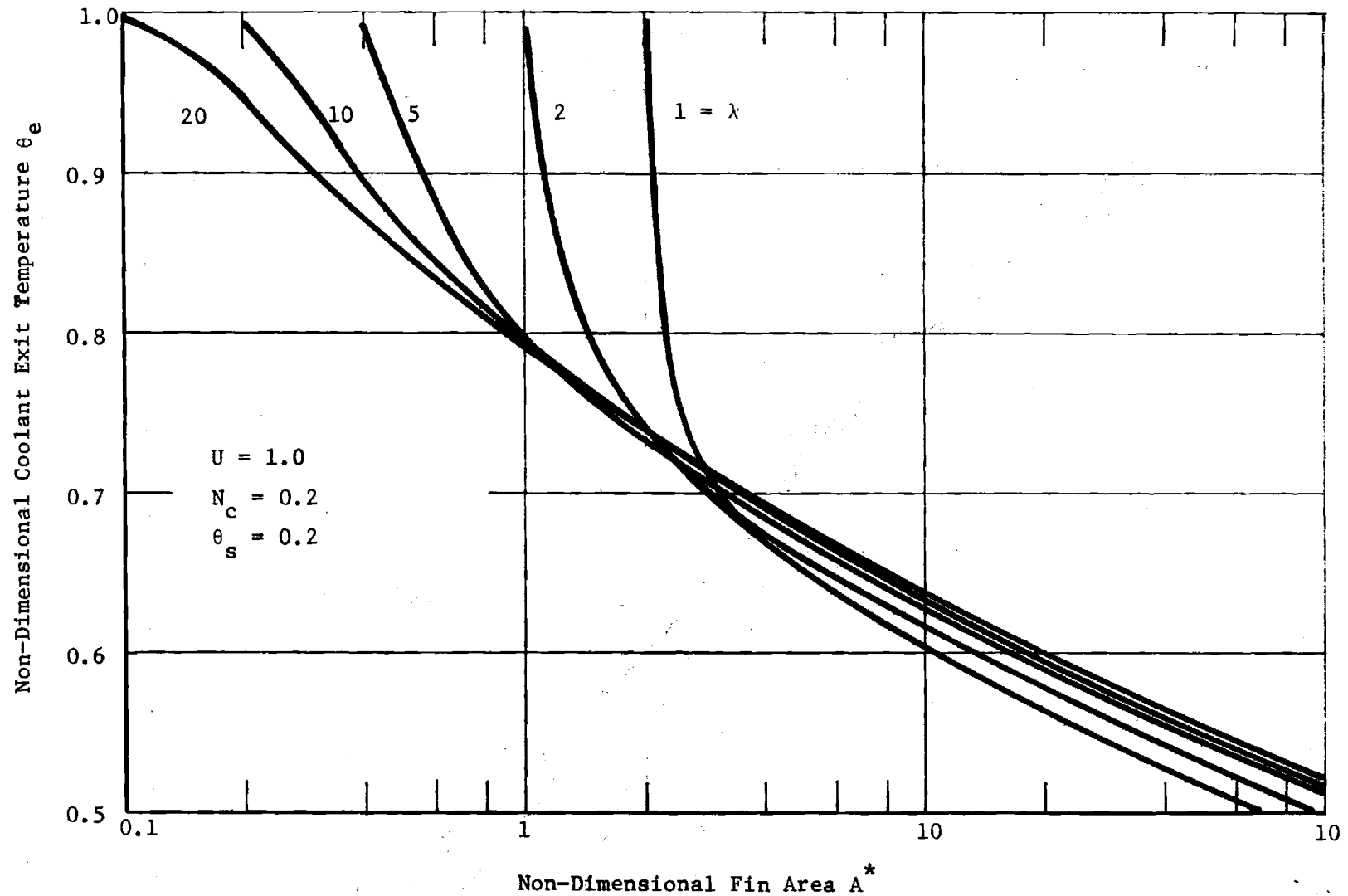


Figure 6. Non-Dimensional Coolant Exit Temperature vs. Non-Dimensional Fin Area For Constant  $U$  and Varying  $\lambda$

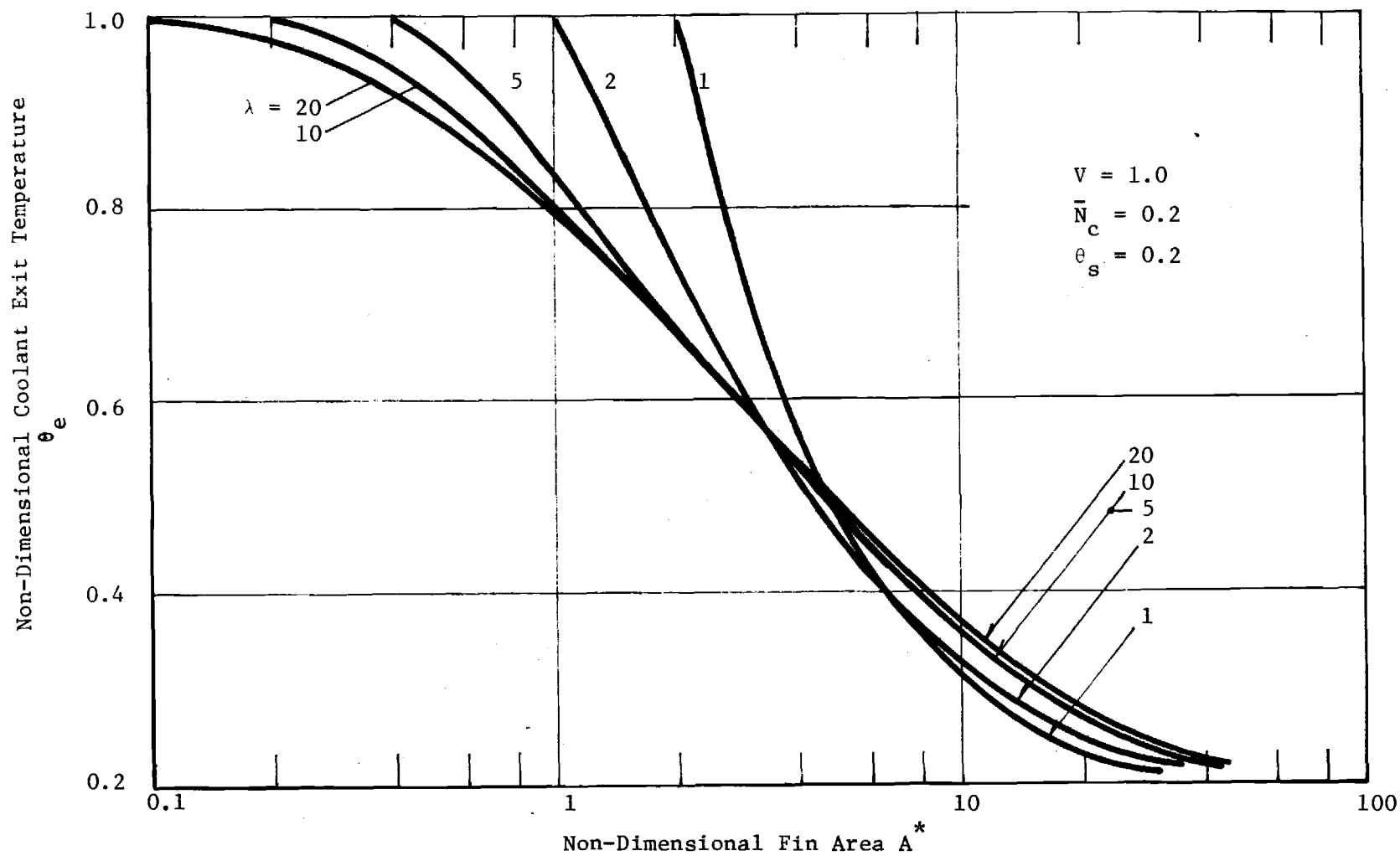


Figure 7. Non-Dimensional Coolant Exit Temperature vs. Non-Dimensional Fin Area For Constant  $V$  and Varying  $\lambda$



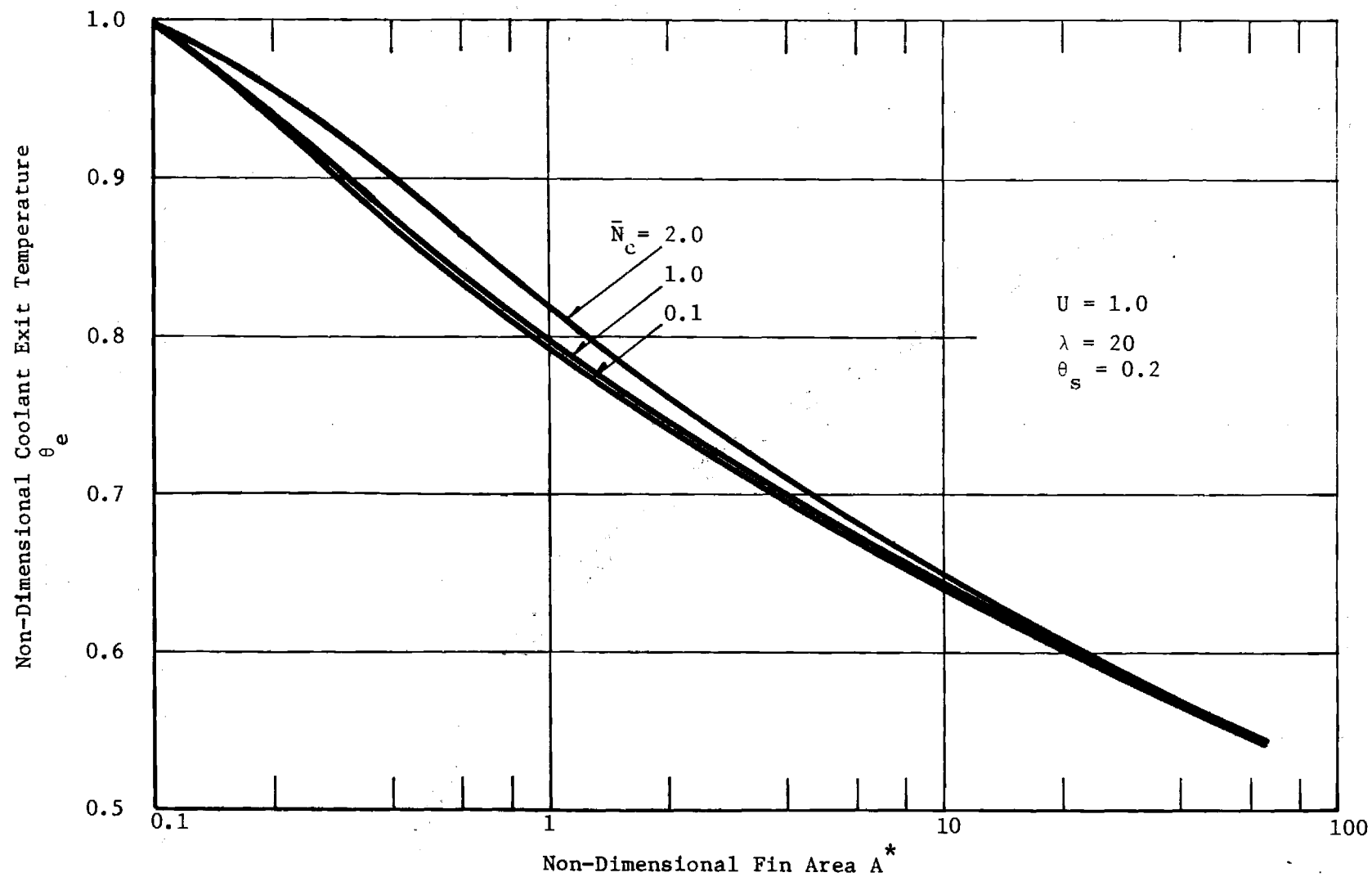


Figure 8. Non-Dimensional Coolant Exit Temperature VS. Non-Dimensional Fin Area For Varying Conduction Parameter and Long Fins.

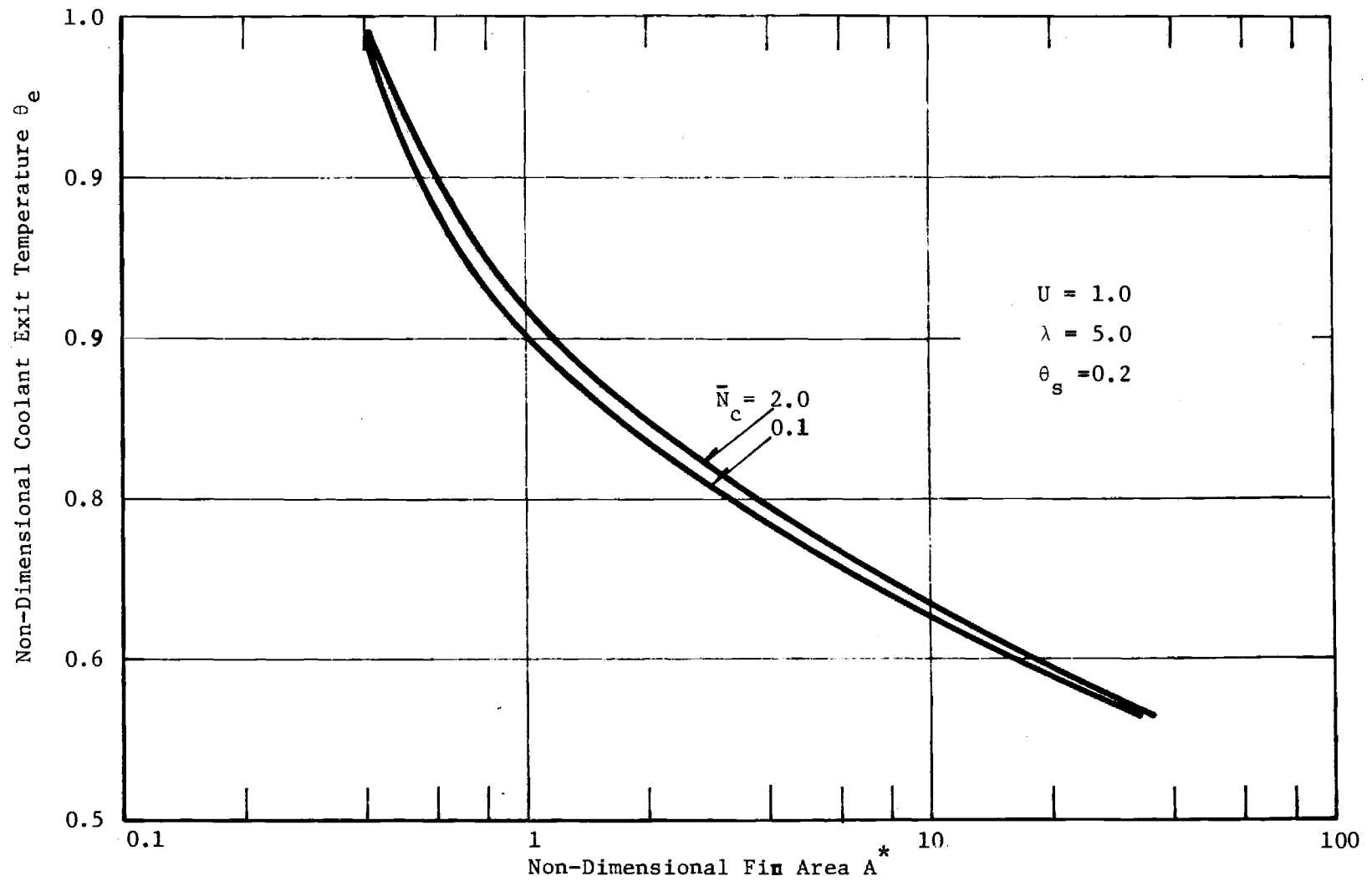


Figure 9. Non-Dimensional Coolant Exit Temperature VS. Non-Dimensional Fin Area For Varying Conduction Parameter and Short Fins

### III. OPTIMIZATION OF THE RADIATOR SYSTEM

#### A. Purpose

In the design of the radiator system all parameters describing the system objectives and the system environment are prescribed beforehand while, at some occasions and within certain limits, the geometry of the system may be selected in the process of the design. This possible freedom of choice leads to the optimization of the radiator system geometry.

The primary purpose of the radiator system is to lower the coolant fluid temperature. For a given set of constraints the radiator with the lowest fluid exit temperature is considered here to be an optimum radiator system.

The set of constraints to be imposed on the optimum radiator system must be selected for a particular set of circumstances. In this analysis two sets out of many possible constraints were selected, that is, weight (or volume) and projected area of the radiator are considered to be prescribed.

The process of optimization proposed here, namely to find the minimum coolant fluid exit temperature for either a fixed radiator area or a fixed mass may be inverted to yield the minimum radiator area or weight for a prescribed cooling rate.

The optimization process can be outlined in the following steps:

- (i) select, for given coolant exit temperature

$$\theta_e = \theta_e \{U, V, \bar{N}_c, \lambda; \theta_s\} \quad (28)$$

that optimum set  $\{U, V, \bar{N}_c, \lambda\}$  of geometric parameters which constitutes, under selected constraints, the system geometry for the lowest possible coolant exit temperature,

- (ii) transform the non-dimensional groups  $U$ ,  $V$  and  $\bar{N}_c$  into suitable, non-dimensional radiator length  $L^*$ , fin height  $H^*$  and panel thickness  $t^*$ ,

(iii) calculate the physical dimensions  $L$ ,  $H$ ,  $t$  and  $d$  of the optimum radiator system.

If all fin elements operate under optimum conditions then the radiator as a whole operates under optimum conditions.

## B. Problem Formulation

### 1. General

Of the five parameters  $U$ ,  $V$ ,  $\bar{N}_c$ ,  $\lambda$  and  $\theta_s$  which define  $\theta_e$  only the first four can be considered independent while  $\theta_s$  is determined by the incident radiant flux. For simplicity let us introduce the vector  $\underline{X}$  with components  $x_1 = U$ ,  $x_2 = V$ ,  $x_3 = \bar{N}_c$  and  $x_4 = \lambda$ . The minimum coolant fluid exit temperature  $\theta_e(\underline{X})$ , subject to  $r \leq 3$  constraints may be obtained from the necessary conditions for a relative extremum

$$\frac{\partial \theta_e}{\partial x_i} = 0 \quad i = 1, \dots, (4 - r) \quad (33)$$

and the  $r$  constraints

$$\varphi_j(\underline{X}) = 0 \quad j = 1, \dots, r \quad (34)$$

This problem formulation implies that the  $r$  constraint equations can be solved explicitly for  $r$  components of  $\underline{X}$  which can be substituted into Eqs. 33 to yield  $(4 - r)$  equations for the remaining  $(4 - r)$  components. This was in effect achieved by replacing the Nusselt number relationships given in Eqs. 8 and 9 by a simpler relation of the form

$$N_{Nu} = c N_{Re}^m N_{Pr}^n (d/L)^p \quad (35)$$

with appropriate constants  $c = 0.023$ ,  $m = 0.8$ ,  $n = 1/3$ ,  $p = 0$ , for  $N_{Re} \geq 2,300$ ; and  $c = 1.86$ ,  $m = n = p = 1/3$  for  $N_{Re} < 2,300$ . The method of Lagrangian multipliers, however, permits the retention of the original constraints at the expense of additional, computational efforts in the case of fixed-mass constraints, since Eqs. 34 need not to be solved explicitly for  $\underline{X}$ .

Two sets of constraints have been considered. Both sets have in common four sets of bounds

$$\left. \begin{aligned}
 U_{\min} &\leq x_1 \leq U_{\max} \\
 V_{\min} &\leq x_2 \leq V_{\max} \\
 (\bar{N}_c)_{\min} &\leq x_3 \leq (\bar{N}_c)_{\max} \\
 \lambda_{\min} &\leq x_4 \leq \lambda_{\max}
 \end{aligned} \right\} \quad (36)$$

which represent practical limits of the design parameters  $x_i$ ,  $i = 1, \dots, 4$ . Should the search procedure lead to the intersection of any of the above limits, then their appropriate equalities would enter the set of constraints. The other two constraints, namely of fixed volume or of fixed area, are discussed below.

## 2. Area Constraint

Introduce the characteristic length of a single fin element

$$L_o = \sqrt{\frac{\dot{m} c_p}{i \epsilon \sigma T_o^3}} \quad (37)$$

which may be interpreted as the length of a square fin element whose effectiveness is unity and which reduces the fluid temperature to zero. Dividing the physical dimensions  $L$ ,  $H$ ,  $t$  and  $d$  by the reference length  $L_o$  one obtains the non-dimensional geometric quantities  $L^*$ ,  $H^*$ ,  $t^*$  and  $d^*$ , respectively. The normalized projected fin element area is (for  $t \ll d$ )

$$A^* = A/L_o^2 = [2HL + dL]/L_o^2 \quad (38)$$

Consequently, the constraint of "fixed area," the first of Eq. 34, becomes

$$\phi_1 = 2H^*L^*[1 + 2/\lambda] - A^* = 0 \quad (39)$$

or

$$\phi_1(x_1, x_2, x_4) = x_1 x_2 (1 + 2/x_4) - A^* = 0 \quad (40)$$

This constraint equation can be solved explicitly for any one of its arguments, regardless of applicable Nusselt number relationship (see comments following Eq. 34).

The constant area constraint may be supplemented by an additional constraint reflecting the preference for a particular fin panel thickness, arising from manufacturing considerations. A fixed panel thickness  $t$ , or equivalently  $t^*$  would require

$$\phi_2(\underline{X}) = t^* - \phi_1 x_2^2 / x_3 \cdot N_{Nu}^2(\underline{X}) = 0 \quad (41)$$

where  $\phi_1$  is known beforehand

$$\phi_1 = (1/2i) (\pi k_f/k)^2 k^2 / (i \dot{m} C_p \epsilon \sigma T_o^3) \quad (42)$$

Equation 41 depends on the Nusselt number relationship used. It becomes a necessary constraint when the optimization tends toward panel thicknesses which approach the tube diameter.



### 3. Volume Constraint

The volume constraint is stipulated by the desire to design a radiator system of minimum mass. The coolant mass may (liquids) or may not (gases) contribute significantly to the total mass.

The mass of one fin element is, under the assumption of a tube wall thickness, equal to the fin panel thickness

$$M^* = \frac{M}{\phi_1 M_o} = L^* t^* [2H^* + \pi(d^* - t^*)] \approx \frac{UV^3}{\bar{N}_c} \left[ 1 + \frac{2\pi}{\lambda} \right] N_{Nu}^2 \quad (43)$$

provided  $t^* \ll d^*$ . Here  $M_o = \rho L_o^3$  is the reference mass and  $M$  the physical mass of the structure. With

$$\phi_2 = 2i(\rho_f/\rho)(k/k_f) \quad (44)$$

the coolant fluid mass may be expressed as

$$M_f^* = \frac{M_f}{\phi_1 M_o} = \phi_2 N_{Nu} \left( \frac{V}{\lambda} \right)^2 U \quad (45)$$

Consequently, the "volume constraint" becomes

$$\varphi_1(X) = M_{tot}^* - x_1 x_2^2 N_{Nu}(X) \left\{ x_2 [1 + 2\pi/x_4] N_{Nu}/x_3 + \phi_2/x_4^2 \right\} = 0 \quad (46)$$

where  $M_{tot}^* = M^* + M_f^*$ . Equation 46 reveals that while  $\phi_2 > 1$  the fluid mass contributes to the total mass only when  $x_4$  becomes small. For the expected cases of  $x_4 \approx 10$  Eq. 46 reduces to a volume constraint as the influence of the density ratio  $\rho/\rho_f$  vanishes.

#### 4. Sufficiency Requirements

An optimum radiator is found at  $\bar{X}_m$  when  $\theta_e(\bar{X}_m)$  is a minimum, that is, when in the vicinity of  $\bar{X}_m$

$$\theta_e(\bar{X}_m + \delta) > \theta_e(\bar{X}_m) \quad (47)$$

where  $\delta$  is a small vector, with components  $\{\Delta x_1, \dots, \Delta x_4\}$ , and with its endpoint on the hypersurface defined by Eqs. 34.

There exist analytical expressions for the sufficiency conditions in terms of second-order derivatives  $\partial^2 \theta_e / \partial x_i \partial x_j$  which are applicable when either all constraints can be solved explicitly as implied by Eqs. 33 and 34 or the number of constraints is at least two. Since, however,  $\theta_e$  is obtained through numerical integration of Eqs. 24 and 29 and the sufficiency test is to be performed only once the potential optimum is found, it appears economical to evaluate Eq. 47 directly. Further developments concerning the sufficiency criteria are necessary at this time.

### C. Solution

#### 1. The Optimum

The remaining task is to solve the system of Eqs. 33 as they are obtained after substitution of Eqs. 34. The solution is obtained through an iterative process based on the Newton-Raphson procedure. Starting with an estimated set of parameters  $\underline{X}_1$  the iteration is carried on according to

$$\underline{X}_{k+1} = \underline{X}_k + \Delta \underline{X}_k, \quad k = 1, 2, \dots \quad (48)$$

where  $\underline{X}_k$  represents the current,  $\underline{X}_{k+1}$  the future parameter set and the increments  $\Delta \underline{X}_k$  are the solution to the system of linear algebraic equations

$$\underline{Y}_k = (\underline{A})_k (\Delta \underline{X})_k. \quad (49)$$

The current components  $(y_i)_k$  of  $\underline{Y}_k$  are the current values of the derivatives

$$y_i = - \frac{\partial \theta}{\partial x_i} e, \quad i = 1, \dots, (4 - r). \quad (50)$$

The current elements  $\{A_{ij}\}_k$  of the square matrix  $(\underline{A})_k$  are the second-order derivatives

$$A_{ij} = \frac{\partial^2 \theta}{\partial x_i \partial x_j} e; \quad i, j = 1, \dots, (4 - r). \quad (51)$$

It is obvious that the matrix  $A_{ij}$  is symmetric and one needs to compute only  $(5 - r)(4 - r)/2$  independent, second-order derivatives.

The derivatives in Eq. 50 and 51 are obtained from Eqs. 24 and 25 by first differentiating with respect to  $x_i$ ,  $i = 1, \dots, (4 - r)$  and then interchanging the order of differentiation. This leads first to  $2(4 - r)$  first-order, ordinary, non-linear differential equations for

$$\begin{aligned}
-\frac{dy_i}{d\zeta} &= \frac{d}{d\zeta} \left( \frac{\partial \theta_f}{\partial x_i} \right), \quad i = 1, \dots, (4 - r) \\
-\frac{dy_i}{d\zeta} &= \frac{d}{d\zeta} \left( \frac{\partial \theta_b}{\partial x_i} \right), \quad i = (5 - r), \dots, (8 - 2r)
\end{aligned} \tag{52}$$

and then to  $(5 - r)(4 - r)$  first-order, ordinary non-linear differential equations for

$$\left. \begin{aligned}
\frac{dA_{ij}}{d\zeta} &= \frac{d}{d\zeta} \left( \frac{\partial^2 \theta_f}{\partial x_i \partial x_j} \right) \\
\frac{dB_{ij}}{d\zeta} &= \frac{d}{d\zeta} \left( \frac{\partial^2 \theta_b}{\partial x_i \partial x_j} \right)
\end{aligned} \right\} \tag{53}$$

The second sets of equations in Eqs. 52 and 53 are necessary because of the dependence of  $\theta_e$  on  $\theta_b$ . All initial conditions, at  $\zeta = 0$ , for Eqs. 52 and 53 can be derived from Eqs. 26 and 30

$$y_i(0) = (y_0)_i, \quad A_{ij}(0) = (A_0)_{ij}, \quad B_{ij}(0) = (B_0)_{ij}. \tag{54}$$

In summary, one needs to integrate, together with Eqs. 24 and 29, the Eqs. 52 and 53, subject to the initial conditions given by Eqs. 26, 30 and 54. The integration is carried out from  $\zeta = 0$  to  $\zeta = 1$  where it yields not only  $\theta_e$  and  $\theta_b(1)$  but also all derivatives in Eqs. 49. This system is solved for the components of  $\Delta \underline{X}$ , then a new set of parameters  $\underline{X}$  is computed from Eqs. 48 and the iterative cycle repeated. The repetitions are continued until either the first  $(4 - r)$   $y_i$ 's are small or one of the inequalities in Eqs. 36 is violated. Should that happen then an additional constraint is introduced,  $r$  is incremented by one, and the cycle is continued until either all limits in Eqs. 36 are reached or the remaining  $y_i$ 's are sufficiently small. The result may or may not be an optimum for the initially chosen  $A^*$  or  $M^*$ . Finally, the potential optimum is tested in accordance with Eq. 47.

## 2. Parameter Transformation and Physical Dimension of Optimum Radiator System.

Once the optimum set  $\{x_1, \dots, x_4\}_m$  is found it is necessary to compute from this the physical dimensions  $L$ ,  $H$ ,  $t$  and  $d$  which define the geometry of the radiator system. To accomplish this task the set  $\{x_1, \dots, x_4\}_m$  is first transformed into the previously introduced set of normalized dimensions  $\{L^*, H^*, t^*, d^*\}$ . This transformation is, in general, not possible in explicit form. However, when the simplified Nusselt number relation of the form given by Eq. 35 is substituted for the more general expressions in Eqs. 8 and 9 then  $L^*$ ,  $H^*$ ,  $t^*$  and  $d^*$  can be expressed explicitly in terms of the parameters  $\{x_1, \dots, x_4\}$ .

Define first the starred quantities.

$$N_{Gz}^* = \frac{c \dot{m}}{k_f L_o} \quad (55)$$

$$N_{Re}^* = \frac{4\dot{m}}{\pi \mu L_o} \quad (56)$$

$$N_{Nu}^* = c(N_{Re}^*)^m (N_{Pr})^n \quad (57)$$

$$N_c^* = \frac{2\epsilon \sigma T_o^3 L_o}{k} \quad (58)$$

which are essentially the unstarred quantities evaluated with the reference length instead of any other dimension and which are all known from system objectives and environmental conditions.

Next one needs to distinguish between laminar and turbulent coolant flow as the constants  $c$ ,  $m$ ,  $n$  and  $p$  are different for the two regimes.

For laminar flow,  $c = 1.86$ ,  $m = n = p = 1/3$

$$L^* = \frac{N_{Gz}^*}{2\pi \sqrt{c^3}} \sqrt{x_1^3} \quad (59)$$

$$H^* = \pi \sqrt{c^3 / N_{Gz}^*} \frac{x_2}{\sqrt{x_1}} \quad (60)$$

$$t^* = \pi^2 c N_c^* / (N_{Gz}^*)^2 \frac{x_2^2}{x_1 x_3} \quad (61)$$

and

$$d^* = 4\pi \sqrt{c^3} / N_{Gz}^* \frac{x_2}{x_4 \sqrt{x_1}} \quad (62)$$

For turbulent flow,  $c = 0.023$ ,  $m = 4/5$ ,  $n = 1/3$ ,  $p = 0$ .

With

$$\phi_3 = \pi N_{Nu}^* / N_{Gz}^* \quad (63)$$

one obtains

$$L^* = 1 / [2^5 \phi_3^{14}]^{1/9} x_1 \left[ \frac{x_2}{x_4} \right]^{4/9} \quad (64)$$

$$H^* = 1/2 [2^5 \phi_3^{14}]^{1/9} (x_2^5 x_4^4)^{1/9} \quad (65)$$

$$t^* = N_c^* / 4 [2^5 \phi_3^{14}]^{2/9} \frac{(x_2^5 x_4^4)^{2/9}}{x_3} \quad (66)$$

$$d^* = \left[ 2 \phi_3 \frac{x_2}{x_4} \right]^{5/9} \quad (67)$$

Finally, multiplication of the starred quantities  $L^*$ ,  $H^*$ ,  $t^*$  and  $d^*$  by the reference length  $L_0$  gives the dimension  $L$ ,  $H$ ,  $t$  and  $d$  of the fin radiator system with optimum performance.

#### D. Results

The ultimate presentation of the results from the radiator system optimization is the plot of the optimum non-dimensional geometric parameters  $L^*$ ,  $H^*$ ,  $t^*$ , and  $d^*$  versus the non-dimensional enthalpy rejection  $(1 - \theta_e)$ . These graphs would include three or more parameters, namely the sink temperature  $\theta_s$  and the parameters occurring in the constraint equations. However, the optimum geometry is not expected to depend strongly on all parameters because of physical considerations, and convenient graphical presentation should be possible.

As indicated in the Introduction, the systematic optimization based on analytical extrema search techniques was developed in addition to the original program objectives but could not be completed within the contract period. The graphical presentation of the optimum geometry in terms of intended enthalpy rejection is therefore not included in this report.

However, computer codes were written on the basis of the solution discussed in Section C. These codes produce the optimum radiator parameters  $L^*$ ,  $H^*$ ,  $t^*$  and  $d^*$  based on the Nusselt number relationship in Eq. 35, for any particular set of input parameters computed from mission requirements and environmental conditions. These results are described in detail in Chapter IV, Sections C. 4 and D. 4.

## IV. COMPUTER CODES

### A. Introduction

Three separate FORTRAN codes were developed, one for the simplified radiator system simulation, one for the minimum area optimization and one for the minimum mass system optimization.

The three codes have several subprograms in common and could be united into a single code to avoid duplication. The simulation program, however, is more efficient as a single program and will remain a tool by itself. The optimization programs, on the other hand, are means to develop a tool, namely suitable working charts from which to read optimum system parameters. Once these diagrams are obtained, the codes are no longer needed.

The three FORTRAN codes are discussed separately in the following three sections but reference is made in the description of subprograms which have previously been discussed.

### B. Radiator System Simulation

#### 1. Objective

The purpose of the Simplified Radiator System Simulation (SRSS) is to evaluate the analysis developed in Chapter II for determining the coolant exit temperature

$$\theta_e(U, V, \bar{N}_c, \lambda; \theta_s)$$

in terms of the five governing groups  $U$ ,  $V$ ,  $\bar{N}_c$ ,  $\lambda$  and  $\theta_s$  defined by Eqs. 13, 19 through 22.

The coolant exit temperature and also the fin base temperature are obtained by integrating simultaneously Eqs. 24 and 29, subject to the initial conditions given by Eqs. 27 and 30. The integration is performed by the Runge Kutta integration procedure described in Reference [1], under variable step size mode.



The individual program units are discussed in the subsequent sections to the extent deemed necessary for the proper utilization of the code. Following the program description are presented the input and output specifications.

## 2. Deck Assembly

The Simplified Radiator System Simulation (SRSS) code consists of:

one		main program	MAIN
three	SUBROUTINE	subprograms	RKS SDRV SCNTL
three	FUNCTION	subprograms	ETA POLY

A block diagram is shown in Figure 7 below.

a. The MAIN Program accepts data input and lists the accepted input data. Following the input data management, the initial conditions for integration are set, an initial step size is computed and control variables are set to control the integration procedure. Then the Runge Kutta procedure RKS is called which performs the integration and indirectly the listing of results. After return from the integration, control is transferred to the start of the program for arbitrarily many repetitions of the program execution. When all input data are exhausted then control is transferred to a normal exit.

The input data preparation is discussed in Section 3 below.

The initial conditions are given through Eqs. 27 and 30. The dependent variables  $\theta_f$  and  $\theta_b$  are, during integration, placed in the array Y;  $Y(1) = \theta_f$  and  $Y(2) = \theta_b$ . Equation 27 leads to the statement  $Y(1) = 1.0$  and Eq. 30 is solved for  $\theta_b$  by the Newton-Raphson iteration which follows the listing of accepted input data. The iteration process leads to the statement  $Y(2) = \text{THETAB} (= \theta_b)$ .

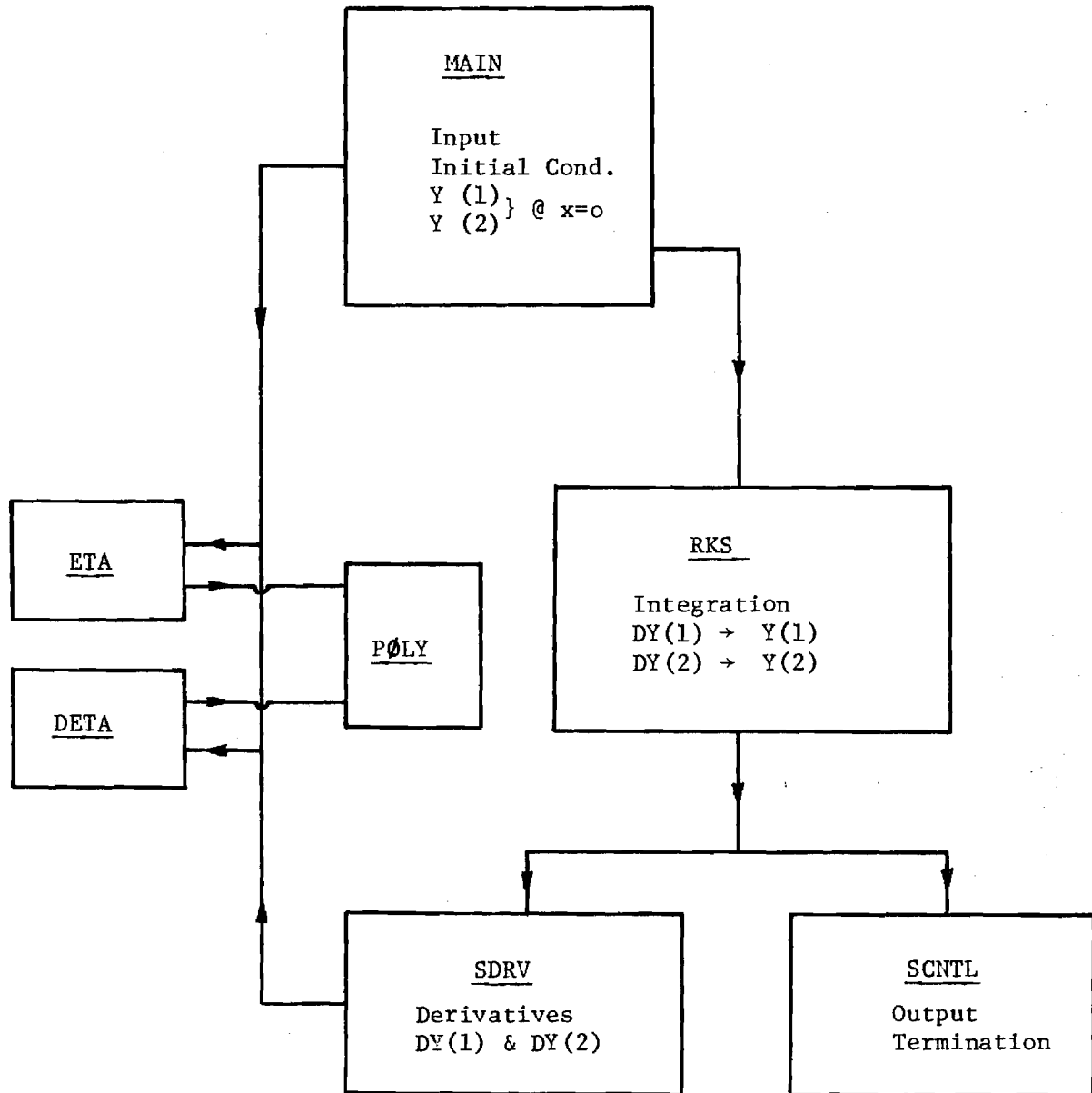


Figure 10. Block Diagram For Simplified Radiator System Simulation Code

Control variables which direct the Runge Kutta Integration

RKS are:

ZZZ = 0.0	initial value for independent variable
DX = $1/(U(1 - \theta_b))$	initial step size
IFVD = 0	variable step size selection
N = 2	two equations to be integrated
IBKP = 1	cut interval as required
NTRY = 1	normal integration mode
IERR = 0	normal error condition ( $DX \neq 0$ )
A(I) = $5 \times 10^{-5}$ ; I = 1,2	absolute error per step
R(I) = $5 \times 10^{-5}$	relative error per step

Control variables which direct output during the integration

are:

DLMT = DXWRT/500	error limit for output interval
XWRT = 0.0	first value of independent variable at first output listing
LSTEP = 0	controls change to fixed step, LSTEP = 1 once integration step exceeds write interval DXWRT
LCNT = 0	senses cut-back (LCNT = 1) of step size prior to listing of results; used to save uncut time step
ICNT = 0	integration step counter

The CALL statement for the Runge-Kutta SUB-ROUTINE is explained in Reference [1] together with the specification of necessary DIMENSION and EXTERNAL declarations. SDRV and SCNTL must be declared EXTERNAL. The necessary array allocations can be read from the listing of the RKS code: all arrays with variable dimension (N) must be declared in MAIN for N = 2 elements. Only the elements of Y, A and R must be specified prior to the CALL RKS statement.

b. The Runge-Kutta SUBROUTINE RKS is described in detail in Reference [1]. The user is not expected to alter this code in any way and needs to know only the control features discussed in paragraph a above and the fact that RKS calls two subroutines SDRV and SCNTL.

c. The SUBROUTINE SDRV serves to evaluate the derivatives  $d\theta_f/d\zeta = dY(1)$  and  $d\theta_b/d\zeta = dY(2)$  in accordance to Eqs. 24 and 29. The fin effectiveness  $\eta(N_c)$  and its derivative  $d\eta/dN_c$  are evaluated externally by the FUNCTION subprograms ETA and DETA.

SDRV is called from RKS and supplies the derivatives, via argument list, as functions of the independent variable  $X$  and dependent variables  $Y(1)$  and  $Y(2)$ , to the integrating procedure RKS. SDRV is called four times per integration step.

d. The SUBROUTINE SCNTL serves to provide

- (i) output listing at prescribed intervals DXWRT,
- (ii) transfer to fixed-step integration mode once the automatically selected (by RKS) step size DX exceeds the print-out interval DXWRT,
- (iii) termination of the integration procedure and RETURN to the MAIN program (via RKS). The same program, with different WRITE commands is used in the Minimum Area and Minimum Mass Optimization codes.

In the argument list are available

- (i) the dependent variables  $Y(1)$  and  $Y(2)$
- (ii) their derivatives  $DY(1)$  and  $DY(2)$
- (iii) the dependent variable  $X$
- (iv) the step size DX which may be altered for repeat of last step
- (v) NTRY = 1 normal continuation of RKS  
2 return from RKS to MAIN

- 3 repeat with new DX
- 4 restart
- (vi) IFVD = 0 variable integration step
- 1 fixed integration step

e. The FUNCTION ETA Subprogram computes the fin effectiveness  $\eta(N_c)$  for the unobstructed fin in accordance to Eqs. 32a and b. The power polynomial is evaluated by calling the FUNCTION PØLY subprogram [1].

FUNCTION ETA (X) receives the independent variable  $X = N_c$  and returns the fin effectiveness ETA. The two functions represented by Eqs. 32 a and b apply strictly to zero sink temperature, but the effectiveness is insensitive to the sink temperature and the equations hold sufficiently well for  $\theta_s < 0.8 \theta_b$ .

ETA (X) is called from MAIN and SDRV.

f. The FUNCTION DETA Subprogram computes the derivative  $d\eta/dN_c$  of the fin effectiveness  $\eta(N_c)$  for the unobstructed fin. The derivative is derived from Eqs. 32 a and b, and evaluated by calling the FUNCTION PØLY subprogram [1].

FUNCTION DETA (X) receives the independent variable  $X = N_c$  and returns  $DETA = d\eta/dN_c$ . DETA (X) is called from MAIN and SDRV.

g. The FUNCTION PØLY Subprogram [1] serves to evaluate power polynomials  $f(x)$  of any degree  $(N - 1)$ .

$$f(x) = \sum_{i=1}^N A_i X^i$$

by computing the recurrence expression

$$f_i = A_N$$

$$f_{k+1} = f_k X + A_{N-k}, k = 1, \dots, (N-1)$$

(N-1) times. The argument list of FUNCTION PØLY (N, A, X) transfers to the subprogram the number N of coefficients  $A_i$ , the one-dimensional array A(N) and the independent variable X; it returns PØLY = f(x). PØLY is called from ETA and DETA.

### 3. Input Data Preparation

The non-dimensional groups

$$\text{THETAS} = \theta_s = T_s/T_o \quad (19)$$

$$U = \pi N_{Nu} / N_{Gz} \quad (22)$$

$$V = (i/\pi)(t/H)(k/k_f)(\bar{N}_c/N_{Nu}) \quad (21)$$

$$\text{FNCBR} = \bar{N}_c = 2\epsilon\sigma T_o^3 H^2 / (kt) \quad (20)$$

$$\text{RHOD} = \lambda = 4H/d \quad (13)$$

$$\text{DXWRT} = \Delta z/L, \text{ output intervals}$$

are computed and punched, in the above sequence, on two cards in

FORMAT (5F16.8)

There may be arbitrarily many pairs of cards as specified above, one pair for each desired simulation. They will be executed in succession. The End-of-Job card will cause normal exit from the program.

The expected run time on the UNIVAC 1108 is 0.24 seconds per simulation plus necessary compiling and collecting times. There were 32 runs executed in 14.8 seconds.

### 4. Output Presentation

For each simulation are printed, on a separate page, first the governing parameters as read in:

THETAS =  $\theta_s$  , defined by Eqs. 4 & 19

U , " Eq. 22

V , " Eq. 21

FNCBAR =  $\bar{N}_c$  , " Eq. 20

LAMBDA =  $\lambda$  , " Eq. 13

and then a table consisting of seven columns. In the first two columns are listed, as functions of axial distance ZETA =  $\zeta$  at the selected intervals

$\Delta\zeta = DXWRT$ , the coolant fluid temperature  $THETA_F = \theta_f$  and the fin base temperature  $THETA_B = \theta_b$ . The positions  $ZETA = \zeta$  are listed in the sixth column.

The conduction parameter  $NC = N_c$ , the combined fin-plus-tube effectiveness  $ETABAR = \bar{\eta}$ , defined by Eq. 11, and the fin effectiveness (unobstructed fin, Eqs. 32 a and b)  $ETA = \bar{\eta}$ , are listed, respectively, in the third, fourth and fifth columns. The last columns indicate the integration steps required to reach the respective axial position  $\zeta$ . The desired coolant fluid exit temperature  $\theta_e (U, V, \bar{N}_c, \lambda; \theta_s)$  is found in the last entry of the first column.

Figure 11 shows a typical output listing

### C. Minimum Area Optimization

#### 1. Objective

The Minimum Area Optimization (MAO) code serves to find the minimum coolant fluid exit temperature for a given projected fin panel area, on the basis of the optimization analysis developed in Chapter III and the constraints as discussed in Section III-2. From the results, evaluated for a number of selected areas, one can determine the minimum area required for a chosen heat rejection rate. The optimum parameter set  $X_{\lambda}$  is sought which satisfies Eqs. 33, 36, 40 and 41 by performing the iterations specified in Eqs. 48 through 53 until Eq. 33 is satisfied.

From the optimum parameter set  $X_{\lambda}$  one computes the normalized system dimensions  $L^*$ ,  $H^*$ ,  $t^*$  and  $d^*$  in accordance to Eqs. 59 through 62 or Eqs. 64 through 67, depending on the flow regime in the coolant channels.

The necessary derivatives occurring in Eqs. 50 and 51 are obtained by Runge-Kutta Integration of the first-order, ordinary, non-linear differential equations given by Eqs. 52 and 53.

The individual program units are discussed in the following sections. Reference is made to program description in Section 3 of this Chapter and to Reference [1].

THETA-S = .200000  
 U = .100000+01  
 V = .100000+01  
 FRCBAR = .200000+00  
 LAMBDA = .250000+02

THETA-F	THETA-B	NC	ETAPAR	ETA	ZETA	STEP NO
1.000000	.724992	.076213	1.001229	.919403	.00	1
.973292	.713938	.072780	1.004461	.922709	.10	2
.948092	.703261	.069563	1.007515	.925832	.20	3
.924291	.692944	.066546	1.010401	.928785	.30	4
.901791	.682975	.063715	1.013130	.931576	.40	5
.880499	.673338	.061056	1.015711	.934216	.50	6
.860331	.664019	.058558	1.018154	.936714	.60	7
.841212	.655007	.056204	1.020465	.939079	.70	8
.823068	.646288	.053989	1.022655	.941319	.80	9
.805835	.637851	.051902	1.024729	.943441	.90	10
.789452	.629684	.049934	1.026696	.945453	1.00	11
						12

FIGURE 11. SAMPLE RESULTS OF SIMPLIFIED RADIATOR SYSTEM SIMULATION



## 2. Deck Assembly

In its current state the Minimum Area Optimization (MAO) code consists of

1		main program	MAIN		
14	SUBROUTINE	subprograms	RKS	}	integration
			SDRV		
			SCNTL		
			BNDEND	}	restraints
			RESBND		
			REST		
			NDERV	}	partial derivatives
			MDER		
			NCDERV		
			MIXDER		
			DERV		
			INITIAL	}	initial conditions
			INTMIX		
			FMINV		matrix inversion
5	FUNCTION	subprograms	ETA	}	$\eta$ and its derivatives
			DETA		
			DDETA		
			DDDETA		
			POLY		

The basic concept of the program is as described in Chapter B before and the block diagram in Figure 7 applies in principle. The exceptions are

- (i) there are 20 simultaneous differential equations to be solved:
  - 2 for  $\theta_f$  and  $\theta_b$
  - 6 for the derivatives of  $\theta_f$  and  $\theta_b$  with respect to  $U$ ,  $V$  and  $\bar{N}_c$
  - 12 for the second derivatives of  $\theta_f$  and  $\theta_b$  with respect to  $U$ ,  $V$ , and  $\bar{N}_c$ ,
- (ii) the initial conditions on the last 18 differential equations are obtained in subroutines,

- (iii) partial derivatives occurring in the differential equations for the last 18 of the above derivatives are computed by subroutines,
- (iv) at the end of each integration, the system of Eq. 49 is solved by FMINV and the integration repeated until the optimum is reached.

a. The MAIN Program performs the same functions as described in Chapter IV, Section B.2.a. and, in addition, calls INITIAL and INTMIX which compute the remaining 18 initial conditions for the partial derivatives as defined through Eq. 54. Moreover, instead of returning to the next input data set after completion of an integration, new system parameters  $\bar{X}$  are computed by solving, via FMINV, the system of Eq. 49. The results are tested, through SUBROUTINE BNDCND, to satisfy the Inequalities 36. If necessary, the newly computed parameters are adjusted to fall within the above limits, and the integration is repeated.

b. The SUBROUTINES RKS, SDRV and SCNTL perform the integration as explained in Sections B.2.b, c and d. There are, however, 18 additional dependent variables  $Y(3), \dots, Y(20)$  to be integrated by RKS and, consequently, 18 additional derivatives to be evaluated in accordance with Eqs. 52 and 53. These derivatives are evaluated by calling DERV for  $\partial\theta_f/\partial X_i$ ,  $\partial\theta_b/\partial X_i$ ,  $\partial^2\theta_f/\partial X_i^2$  and  $\partial^2\theta_b/\partial X_i^2$  and by calling MIXDER for  $\partial^2\theta_f/\partial X_i\partial X_j$  and  $\partial^2\theta_b/\partial X_i\partial X_j$ ,  $i \neq j$ . SCNTL contains a slightly altered output specification.

c. The Restraint SUBROUTINES BNDCND, RESEND and REST serve to, respectively,

- (i) test newly computed components  $\bar{X}$  for compliance with Inequalities 36.
- (ii) solve Eqs. 40 and 41 for  $x_3$  and  $x_4$ .

When newly computed components  $\bar{X}$  do not comply with Inequalities 36 then they are set to meet the minimum and maximum conditions and the system of Eq. 49 is solved again, all from within BNDCND.

SUBROUTINE REST is the only particular program which differentiates the two optimization modes, minimum area and minimum mass. It may be replaced to accommodate other constraints. BNDCND and REST are called from MAIN.

d. The SUBROUTINES NDERV, MDER, NCDER, MIXDER and DERV are used to compute, respectively,

- (i)  $N$  and its derivatives  $dN/d\lambda$ ,  $d^2N/d\lambda^2$  from Eq. 17,
- (ii)  $M$  and its derivatives, up to third order, with respect to its arguments  $\lambda$  and  $N_c$ , in accordance to Eqs. 12, 15 and 16,
- (iii)  $N_c$  and its derivatives  $\partial N_c / \partial \theta_b$ ,  $\partial^2 N_c / \partial \theta_b^2$  and  $\partial^2 N_c / \partial T \partial N_c$ ,
- (iv) the mixed derivatives defined by Eq. 53 for  $i \neq j$ ,
- (v) the derivatives defined by Eqs. 52 and 53.

e. The SUBROUTINES INITIAL and INTMLX serve to compute the initial conditions for the dependent variable  $Y(3), \dots, Y(20)$ , at the channel inlet  $\zeta = 0$  which represent simple and mixed derivatives  $\partial^2 \theta_f / \partial X_i \partial X_j$  and  $\partial^2 \theta_b / \partial X_i \partial X_j$  in Eq. 53.

f. The SUBROUTINE FMINV is capable of performing two related tasks:

- (i) to invert a square, invertible matrix  $\tilde{A}$
- (ii) to solve a non-trivial system of equations  $\tilde{A}\tilde{Z} = \tilde{X}$ .

The solutions are obtained through a sequence of elementary row operations which lead from the properly augmented coefficient matrix to the row-reduced echelon matrix, a standard procedure described in most elementary introductions to linear algebra [1].

SUBROUTINE FMINV ( $A, X, N, M$ ) accepts, through its argument list, the square, invertible matrix  $\tilde{A}$  of rank  $N$ , and if task (i) above is intended,  $M$  is set equal to  $M = 2N$ . Then, upon return from FMINV there will be the inverted matrix  $\tilde{A}^{-1}$  placed in  $XMAT(I, J)$  with  $I = 1, \dots, N, J = N + 1, N + 2, \dots, 2N$ . The two-dimensional array  $XMAT$  is transferred to the calling program via a COMMON declaration. When task (ii) above is intended,  $M$  is set equal to  $M = N + 1$  and, upon return from FMINV, the unknown vector  $\tilde{Z}$  is placed in the one-dimensional array  $\tilde{X}$ .

FMINV is used to solve Eq. 49 and is called from MAIN and from BNDCND.

g. The FUNCTION subprograms ETA, DETA and POLY are described in Sections B.2.e, f, and g of this chapter.

h. The FUNCTION subprograms DDETA and DDDETA are used to compute the second and third-order derivatives  $d^2\eta/dN_c^2$  and  $d^3\eta/dN_c^3$  as derived from Eqs. 32 a and b. The derivatives are evaluated via FUNCTION POLY subprogram.

### 3. Input Data Preparation

Each optimization run is carried out for a selected  $ASTAR = A^*$  (Eq. 38) and  $TSTAR = t^*$  (Eq. 41) as the independent variables. The necessity of specifying  $t^*$  is not yet completely established at this time but included in the program for additional flexibility and to prevent the search from reading excessively large values of  $t^*$ .

The optimization produces

- (i) the optimum coolant exit temperature
- (ii) the optimum parameters  $\underline{X} = \{x_1, \dots, x_4\}$  from which to compute  $L^*$ ,  $H^*$ ,  $d^*$  and  $\lambda$  (Eqs. 59-67).

For each optimization one computes

THETAS = $\theta_s$	,	defined by Eqs. 19 & 4
U, starting value	,	" " Eq. 22
V, starting value	,	" " Eq. 21
FNCBR = $\bar{N}_c$ starting value	,	" " Eq. 20
ASTAR = $A^*$	,	" " Eq. 40

and selects

DXWRT =  $\Delta\zeta$  the axial interval for which results are to be printed.

The first five values are punched on one card in

FORMAT (5F16.8).

The last value is punched on a second card in

FORMAT(F16.8)

There may be arbitrarily many pairs of cards as specified above, one pair for each optimization run. They will be carried out in succession. The End-of-Job card will cause normal exit from the program.

The expected run time per optimization is approximately 15 seconds plus necessary compilation and collection times.

#### 4. Presentation of Results

Each iteration produces one page of output, first the three parameters THETAS, ASTAR and TSTARI as read in. Next the current values of the system parameters  $U$ ,  $V$ ,  $FNCBAR = \bar{N}_c$ ,  $LAMBDA = \lambda$  and the identification of the flow regime, followed by the iteration count for establishing initial conditions

The table lists, as functions of  $\zeta = Z/L$ , in this order

THETAF =  $\theta_f(\zeta)$  coolant temperature

THETAB =  $\theta_b(\zeta)$  fin base temperature

NC =  $N_c(\zeta) = \bar{N}_c \theta_b^3$ , Eq. 14

ETABAR =  $\bar{\eta}$ , defined by Eq. 11

ETA =  $\eta$ , defined by Eq. 32

M defined by Eq. 12

N defined by Eq. 17

X =  $\zeta$ , the non-dimensional distance along the channel.

Following the table is a list of the three system parameters  $U$ ,  $V$ ,  $\bar{N}_c$  and the first and second derivatives of  $\theta_f$  with respect to these parameters. Finally are listed the newly computed changes of the above parameters.

A typical listing of the results is shown in Figure 9.

THETA-S = .70300000  
 ASTAR = .675000  
 U = 250.000000  
 V = .000050  
 NCBAR = .000100  
 LAMDA = .037736

NUMBER OF ITERATIONS = 14

THETA F	THETA A	NC	ETABAR	ETA	M	N	ZETA
1.0000000	.9479813	.0000094	53.0099905	.9998844	.1801204	53.8198819	.0000000
.9211307	.9180655	.0000776	53.0099919	.9999098	.1801264	53.8198819	.2000000
.8677819	.8669164	.0000652	53.0099928	.9999242	.1801247	53.8198819	.4000000
.8309971	.8303729	.0000573	53.0099933	.9999334	.1801236	53.8198819	.6000000
.8039970	.8035309	.0000519	53.0099938	.9999397	.1801229	53.8198819	.8000000
.7835871	.7832305	.0000480	53.0099938	.9999441	.1801224	53.8198819	1.0000000

# PARAMETERS AT END OF TUBE

THETA F = .783587

U = .250000+03

V = .500000-04

NCBAR = .100000-03

D2FDU = -.247174-05

D2FDV = .117912+00

D2FNCBAR = .771548-04

D2FD2U = .197315-07

D2FD2V = .979100+01

D2FD2NCBAR = .844942-03

D2FD0DV = .530919-05

D2FD0NCBAR = .105256-05

D2FD0VNCBAR = .525590+01

DELTA U = .126478+03

DELTA V = -.363925-04

DELTA NCBAR = -.224943-01

OPTIMUM REACHED

FIGURE 12. RESULTS LISTING OF MINIMUM AREA OPTIMIZATION



and selected

DXWRT =  $\Delta\zeta$  the axial interval for which results are to be printed.

DA, DR absolute and relative integration step errors.

The data values are punched on the data card in NAMELIST format. The NAMELIST is called INPUT.

There may be arbitrarily many additional input data cards as specified above, or with only one datum, one NAMELIST for each optimization run. They will be carried out in succession until an End-of-Job card causes normal exit from the program.

The expected run time per optimization is approximately 20 seconds plus necessary machine preparation times.

#### 4. Representation of Results

Results are presented in the format identical to that described in section C.4 above, except that instead of the single TSTAR1 are listed  $MSTAR = M^*$ ,  $PHI2 = \phi_2$  and  $FNUSTR = N^*_{Nu}$ , as read in. The representative output example is shown in Figure 10.



02 THETA-S = .60000000  
 U = 13.363489  
 V = 2.792685  
 FNCBR = 5.731686  
 TOTMAS = 95.000000  
 PHI2 = 2000.000000  
 ALPHA = 5.000000  
 LAMBDA = 31297273.250000

NUMBER OF ITERATIONS = 8

THETA F	THETA B	NC	ETABAR	ETA	M	N	X
1.0000000	.7686288	2.6027493	.3775592	.3775592	1.0000000	.0000001	.0000000
.7134382	.6486320	1.0316816	.5284999	.5284998	1.0000000	.0000001	.2000000
.6322974	.6138584	1.1674339	.5047780	.5047780	1.0000000	.0000001	.4000000
.6088749	.6038113	1.2169663	.4969006	.4969006	1.0000000	.0000001	.6000000
.6024210	.6010400	1.2322024	.4945601	.4945600	1.0000000	.0000001	.8000000
.6006592	.6002832	1.2364472	.4939149	.4939148	1.0000000	.0000001	1.0000000

PARAMETERS AT END OF TUBE

THETA F = .600659  
 U = .133635+02 V = .279269+01 NCBAR = .573169+01  
 DTFDU = -.496030-04 DTFDV = -.131301-03 DTFNCBAR = -.384840-04  
 D2TFD2U = .105462+03 D2TFD2V = .455954+04 D2TFD2NCBAR = .120270+03  
 D2TFDUDV = -.229946-02 D2TFDUDNCBAR = .294970-03 D2TFDUDNCBAR = -.740525+03  
 INCR. SOUGHT DELTA U = .100314+00 DELTA V = -.807482-01 DELTA NCBAR = -.497181+00  
 INCR. USED DELTA U = .100314+00 DELTA V = -.832025-01 DELTA NCBAR = -.497181+00

INSIGNIFICANT IMPROVEMENT OVER LAST STEP

FIGURE 13. RESULTS LISTING OF MINIMUM MASS OPTIMIZATION

## V. CONCLUSIONS

The work presented herein resulted in a simplified radiator system analysis and a systematic optimization procedure.

Comparison of the simplified with the rigorous [1] system analyses indicated that the agreement between the two analyses can be expected to be within approximately 5%.

The optimization procedures carried out lead frequently to an optimum on the boundaries of the parameter domain. Mass optimization tends toward widely spaced tubes between thin fins.

Optimization was originally intended to be achieved through the parametric study of trends rather than through analytical procedures. Within the resources provided by this contract two analytical iterative optimization procedures were developed beyond the original goal of work. These procedures lead to the maximum heat rejection for given system area or system mass. Future work should be aimed toward the development of

- (1) a parameter domain within which relative extrema exist,
- (2) suitable working diagrams, through repeated applications of the developed codes, which depict the optimum geometric system parameters and produce either the least area or the least weight requirements for a given heat rejection rate.

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## APPENDIX A

COMPUTER CODE FOR SIMPLIFIED

RADIATOR SYSTEM SIMULATION

OFOR-15 MAIN

FOR 59A-07/25/72-22:59:39 (.0)

MAIN PROGRAM

STORAGE USED: CODE(1) 000464; DATA(0) 000355; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 SDR 000014  
0004 MCT 000006

EXTERNAL REFERENCES (BLOCK, NAME)

0005 SDRV  
0006 SCNTL  
0007 ETA  
0010 DETA  
0011 RKS  
0012 NINTRS  
0013 NWDUS  
0014 NI02S  
0015 NR0US  
0016 SQRT  
0017 ASIN  
0020 NSTOPS

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000150	100	0001	000327	105L	0001	000345	110L	0000	000265	120F	0001	000405	232G
0001	000007	45L	0000	000233	5F	0000	000234	50F	0000	000236	55F	0001	000131	85L
0001	000145	90L	0000	000304	900F	0000	000311	905F	0001	000452	920L	0000	000313	925F
0001	000147	95L	0001	000460	950L	0000 R	000030	A	0000 R	000213	AYY	0000 R	000205	A1
0000 R	000214	BYX	0010 R	000000	DETA	0000 R	000220	DETZ	0003 R	000007	DFM	0003	000010	DFMX
0003 R	000011	DFNX	0004 R	000000	DLMT	0000 R	000060	DLY	0003	000012	DROU	0003	000013	DROV
0000 R	000224	DX	0004 R	000003	DXWRT	0000 R	000014	DY	0000 R	000140	DYST	0000 R	000221	DYY
0007 R	000000	ETA	0000 R	000216	ETY	0000 R	000215	ETYY	0003 R	000006	FM	0003 R	000005	FN
0000 R	000212	FNC	0003 R	000002	FNCBR	0000 I	000232	I	0000 I	000227	IRKP	0004 I	000001	ICNT
0000 I	000231	IER	0000 I	000225	IFVD	0004 I	000005	LCNT	0004 I	000004	LSTEP	0000 I	000226	N
0000 I	000207	NCT	0000 I	000230	NTRY	0000 R	000074	PD	0000 R	000044	R	0003 R	000004	RHOD
0006 R	000000	SCNTL	0000 R	000110	SD	0005 R	000000	SDRV	0000 R	000206	THETAB	0000 R	000204	THETAS
0003 R	000003	TH4	0000 R	000210	T2	0000 R	000211	T3	0003 R	000000	U	0003 R	000001	V
0004 R	000002	XWRT	0000 R	000000	Y	0000 R	000124	YS	0000 R	000170	YSIMP	0000 R	000154	YST
0000 R	000217	YY	0000 R	000222	ZZ	0000 R	000223	ZZZ						

00100 1\* C  
00100 2\* C  
00100 3\* C  
00100 4\* C  
00100 5\* C  
00100 6\* C

SIMPLIFIED RADIATOR SYSTEM SIMULATION

\*\*\*\*\*

THETAS, U, V, FNCBR, AND RHOD ARE SYSTEM PARAMETERS  
DXWRT IS THE INCREMENT OF AXIAL DISTANCE FOR OUTPUT LISTING

```

00100      7*      C      FORMAT (5F16.8)
00100      8*      C
00101      9*      EXTERNAL SDRV,SCNTL
00103     10*      DIMENSION Y(12),DY(12),A(12),R(12),DLY(12),PD(12),SD(12),YS(12),
00103     11*      1      DYST(12),YST(12),YSIMP(12)
00104     12*      COMMON /SDR/ U,V,FNCBR,TH4,RHOD,FN,FM,DFM,DFMX,DFNX,DROU,DROV/MCT/
00104     13*      1      DLMT,ICNT,XWRT,DXWRT,LSTEP,LCNT
00104     14*      C
00104     15*      C
00105     16*      FNN(X) = (X+3.570796-SQRT(X*(X+2.0))-ASIN(1.0/(1.0+X)))/X
00106     17*      FFS(X) = (SQRT(X*(X+2.0))+ASIN(1.0/(X+1.0))-1.570796)/X
00107     18*      FFN(X,Y)= 1.0-Y*(0.1460*Y-0.02866)/X
00110     19*      DFFN(X,Y)=(-0.2920*Y+0.02866)/X
00111     20*      DFFNX(X,Y)= Y*(0.1460*Y-0.02866)/(X*X)
00112     21*      DFFS(X) = -SQRT((X+2.0)/X)/(X*(X+1.0))+(1.5708-ASIN(1.0/(X+1.0)))/
00112     22*      1      (X*X)
00112     23*      C
00113     24*
00113     25*      WRITE(6,5)
00115     26*      5 FORMAT(1H1)
00115     27*      C
00116     28*      45 READ(5,50,END=950) THETAS,U,V,FNCBR,RHOD,DXWRT
00126     29*      50 FORMAT(5E16.8)
00127     30*      WRITE(6,55) THETAS,U,V,FNCBR,RHOD
00136     31*      55 FORMAT(1H0,10X,10HTHETA=,F19.6,/,
00136     32*      1      11X,10H      U = ,E20.6,/,
00136     33*      2      11X,10H      V = ,E20.6,/,
00136     34*      3      11X,10H FNCBR = ,E20.6,/,
00136     35*      4      11X,10H LAMBDA = ,E20.6,///)
00136     36*      C
00137     37*      IF(THETAS.GE. 1.0) GO TO 920
00141     38*      FN      = FNN(RHOD)
00142     39*      DFNX     = (-FN+1.0-SQRT(RHOD*(RHOD+2.0)))/RHOD
00143     40*      TH4      = THETAS**4
00144     41*      IF(V.GT.0.2) GO TO 90
00146     42*      A1      = 2.0+1.0/(V*ETA(FNCBR))
00147     43*      IF(A1.GE.2.45) GO TO 85
00151     44*      THETAB = 1.0-1.0/(2.0*A1)
00152     45*      GO TO 95
00153     46*      85 THETAB = 1.0-(A1-SQRT(A1**2-6.0))/6.0
00154     47*      GO TO 95
00155     48*      90 THETAB = 0.9
00156     49*      95 NCT      = 0
00157     50*      100 T2      = THETAB**2
00150     51*      T3      = T2*THETAB
00161     52*      FNC      = FNCBR*T3
00162     53*      FM      = FFN(RHOD,FNC)*FFS(RHOD)
00163     54*      DFM     = DFFN(RHOD,FNC)*FFS(RHOD)
00164     55*      AYY     = T3*THETAB-TH4
00165     56*      BYY     = 1.0-THETAB
00166     57*      ETTY    = ETA(FNC)
00167     58*      ETY     = ETTY*FM+FN
00170     59*      YY      = 1.0/V-ETTY*AYY/BYY
00171     60*      DETZ    = (DFM*ETYY+FM*DETA(FNC))*3.0*T2*FNCBR
00172     61*      DYY     = -AYY/BYY*(DETZ+ETY/BYY)-4.0*ETI*T3/BYY
00173     62*      ZZ      = THETAB-YY/DYY
00174     63*      IF(NCT.GT.20) WRITE(6,900)

```

```

00177 64*      IF(ZZ.LT.1.0) GO TO 105
00201 65*      THETAB = (THETAB+1.0)/2.0
00202 66*      NCT    = NCT+1
00203 67*      GO TO 100
00204 68*      105 IF(ABS(ZZ-THETAB)/ZZ.LT.1.0E-06) GO TO 110
00206 69*      NCT    = NCT+1
00207 70*      THETAB = ZZ
00210 71*      GO TO 100
00210 72*      C
00211 73*      110 Y(1)   = 1.0
00212 74*      Y(2)   = THETAB
00213 75*      ZZZ    = 0.0
00214 76*      DX     = 1.0/(U*(1.0-THETAB))
00215 77*      IFVD   = 0
00216 78*      IF(DX.GT.DXWRT) DX = DXWRT
00220 79*      N      = 2
00221 80*      IBKP   = 1
00222 81*      NTRY   = 1
00223 82*      IERR   = 0
00223 83*      C
00224 84*      DLMT   = DXWRT/500.0
00225 85*      XWRT   = 0.0
00226 86*      LSTEP  = 0
00227 87*      LCNT   = 0
00230 88*      ICNT   = 0
00231 89*      DO 115 I=1,12
00234 90*      A(I)   = 5.0E-05
00235 91*      115 R(I) = 5.0E-05
00235 92*      C
00237 93*      WRITE(6,120)
00241 94*      120 FORMAT(1H0,13X,6HTHETAB,14X,6HTHETAB,16X,2HNC,17X,6HETABAR,15X,
00241 95*      1      3HETA,8X,4HZETA,6X,7HSTEP NO,/)
00241 96*      C
00242 97*      CALL RK5(SDRV,SCNTL,Y,DY,A,R,ZZZ,DX,N,IFVD,IBKP,NTRY,IERR,
00242 98*      .1      DLY,PD,SD,YS,YST,DYST,YSIMP)
00243 99*      WRITE(6,905)
00245 100*     GO TO 45
00246 101*     900 FORMAT(1H0,20HNEWTON-RAPHSON FAILS)
00247 102*     905 FORMAT(1H ,/////)
00250 103*     920 WRITE(6,925)
00252 104*     925 FORMAT(1H0,37HRADIATIVE HEATING CANNOT BE SIMULATED)
00253 105*     GO TO 45
00254 106*     950 STOP
00255 107*     END

```

END OF COMPILATION: NO DIAGNOSTICS.

OFOR,IS SUB1

FOR 59A-07/25/72-23:00:14 (.0)

SUBROUTINE SDRV ENTRY POINT 000150

STORAGE USED: CODE(1) 000155; DATA(0) 000037; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 SDR 000014  
0004 SOC 000003

EXTERNAL REFERENCES (BLOCK, NAME)

0005 ETA  
0006 DETA  
0007 ASIN  
0010 SQRT  
0011 NERRJS

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000136	10L	0001	000103	5L	0000 R 000005	A0	0000 R 000006	A1	0000 R 000007	A2
0000 R	000010	A3	0000 R	000011	A4	0000 R	000012	A5	0006 R	000000	DETA
0003	000010	DFMX	0003	000011	DFNX	0003	000012	DROU	0003	000013	DROV
0004 R	000001	ETY	0004 R	000002	ETZ	0000 R	000003	FFN	0000 R	000004	FFS
0003 R	000005	FN	0003 R	000002	FNCBR	0000	000024	INJPS	0003 R	000004	RHOD
0003 R	000000	U	0003	000001	V	0000 R	000000	Y2	0000 R	000001	Y3
0004 R	000000	Z									Y4

00101	1*		SUBROUTINE SDRV(Y,DY,X)
00101	2*	C	
00101	3*	C	COMPUTES DERIVATIVES
00101	4*	C	
00103	5*		DIMENSION Y(12),DY(12)
00104	6*		COMMON /SDR/ U,V,FNCBR,TH4,RHOD,FM,FFN,DFM,DFMX,DFNX,DROU,DROV/SOC/
00104	7*	1	Z,ETY,ETZ
00104	8*	C	
00105	9*		Y2 = Y(2)**2
00106	10*		Y3 = Y(2)*Y2
00107	11*		Y4 = Y2*Y2
00110	12*		Z = FNCBR*Y3
00111	13*		ETY = ETA(Z)
00112	14*		FFN = 1.-Z*(0.1460+Z-0.02866)/RHOD
00113	15*		FFS = (SQRT(RHOD*(RHOD+2.))+ASIN(1./(RHOD+1.))-1.5708)/RHOD
00114	16*		FM = FFN*FFS
00115	17*		DFM = (-0.2920*Z+0.02866)/RHOD*FFS
00116	18*		ETZ = ETY*FM*FN
00117	19*		DY(1) = -U*(Y(1)-Y(2))
00120	20*		A0 = Y4-TH4



```

00121 21*      IF(A0.GT.1.0E-08) GO TO 5
00123 22*      DY(2) = 0.0
00124 23*      GO TO 10
00125 24*      5 A1 = Y(1)-Y(2)
00126 25*      A2 = (ETY+DFM+DETA(Z)*FM)
00127 26*      A3 = 3.*A2*Y2*FNCBR/ETZ
00130 27*      A4 = 4.0*Y3/A0
00131 28*      A5 = 1.0+A1*(A3+A4)
00132 29*      DY(2) = DY(1)/A5
00132 30*      C
00133 31*      10 CONTINUE
00134 32*      RETURN
00135 33*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

FOR:IS SUB2  
FOR 59A-07/25/72-23:01:21 (.0)

SUBROUTINE SCNTL ENTRY POINT 000146

STORAGE USED: CODE(1) 000173; DATA(0) 000016; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 SDC 000003  
0004 MCT 000005

EXTERNAL REFERENCES (BLOCK, NAME)

0005 NWDJS  
0006 NI025  
0007 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000054	5L	0001	000061	50L	0000	000002	55F	0001	000134	60L	0004	R	000000	DLMT				
0000	R	000000	DXSTR	0004	R	000003	DXWRT	0003	R	000001	ETY	0003	R	000002	ETZ	0004	I	000001	ICNT
0000		000010	I-JP%	0000	I	000001	LCNT	0004	I	000004	LSTEP	0004	R	000002	XWRT	0003	R	000000	Z

```

00101      1*      SUBROUTINE SCNTL(Y,DY,DX,X,NTRY,IFVD)
00101      2*      C
00101      3*      C CONTROLS INTEGRATION
00101      4*      C
00103      5*      DIMENSION Y(12),DY(12)
00104      6*      COMMON /SDC/ Z,ETY,ETZ /MCT/ DLMT,ICNT,XWRT,DXWRT,LSTEP
00105      7*      ICNT = ICNT+1
00106      8*      IF(DX.GE.DXWRT.AND.ICNT.GT.1) LSTEP = 1
00110      9*      IF(ABS(X-XWRT).LT.DLMT) GO TO 50
00112     10*      IF(XWRT.GT.X) GO TO 5
00112     11*      C
00114     12*      DXSTR = DX
00115     13*      DX = DX+XWRT-X
00116     14*      LCNT = 1
00117     15*      NTRY = 3
00120     16*      RETURN
00120     17*      C
00121     18*      5 NTRY = 1
00122     19*      RETURN
00122     20*      C
00123     21*      50 WRITE(6,55) Y(1),Y(2),Z,ETZ,ETY,X,ICNT
00134     22*      55 FORMAT(1H,5F20.6,F10.2,11Q)
00135     23*      IF(LCNT.EQ.1) DX = DXSTR
00137     24*      LCNT = 0
00140     25*      IF(ABS(1.0-XWRT).LE.DLMT) GO TO 60
00142     26*      XWRT = XWRT+DXWRT

```

00143 27\* NTRY = 1  
00144 28\* IF(LSTEP.EQ.0) RETURN  
00146 29\* DX = DXWRT  
00147 30\* IFVD = 1  
00150 31\* RETURN  
00150 32\* C  
00151 33\* 60 NTRY = 2  
00152 34\* RETURN  
00153 35\* END

END OF COMPILATION: NO DIAGNOSTICS.

DFOR,IS SUB3

FOR S9A-07/25/72-23:01:28 (.0)

FUNCTION ETA

ENTRY POINT 000036

STORAGE USED: COJE(1) 000044; DATA(0) 000022; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 EXP  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 IL 0000 R 000001 A 0000 R 000010 B 0000 R 000000 ETA 0000 000014 INJP5  
0003 R 000000 POLY

00101 1\* FUNCTION ETA(X)  
00101 2\* C  
00101 3\* C COMPUTES FIN EFFECTIVENESS ETA  
00101 4\* C  
00103 5\* DIMENSION A(7), B(2)  
00104 6\* DATA A(1),A(2),A(3),A(4),A(5),A(6),A(7)/0.10E+01, -0.1163143E+01,  
00104 7\* 1 0.1478836E+01, -0.1267550E+01, 0.6325223E+00, -0.1627067E+00,  
00104 8\* 2 0.1654223E-01/ B(1),B(2)/0.6866095E+00, -0.2297718E+00/  
00116 9\* IF(X.GT.2.5) GO TO 1  
00120 10\* ETA = POLY(7,A,X)  
00121 11\* RETURN  
00122 12\* 1 ETA = B(1)\*EXP(B(2)\*X)  
00123 13\* RETURN  
00124 14\* END

END OF COMPILATION: NO DIAGNOSTICS.

OFOR, IS SUB4

FOR S9A-07/25/72-23:01:35 (,0)

FUNCTION DATA

ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000021; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY

0004 EXP

0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 1L

0000 R 000001 A

0000 R 000007 B

0000 R 000000 DETA

0000 000013 INJPS

0003 R 000000 POLY

```
00101 1* FUNCTION DATA(X)
00101 2* C
00101 3* C COMPUTES FIRST DERIVATIVE DETA/DNC
00101 4* C
00103 5* DIMENSION A(6), B(2)
00104 6* DATA A(1),A(2),A(3),A(4),A(5),A(6)/-0.1163143E+01, 0.2957672E+01,
00104 7* 1 -0.3802650E+01, 0.2530089E+01, -0.8135335E+00, 0.9925338E-01/
00104 8* 2 B(1),B(2)/-0.1577635E+00, -0.2297718E+00/
00115 9* IF(X.GT.2.5) GO TO 1
00117 10* DETA = POLY(6,A,X)
00120 11* RETURN
00121 12* 1 DETA = B(1)*EXP(B(2)*X)
00122 13* RETURN
00123 14* END
```

END OF COMPILATION:

NO DIAGNOSTICS.

QFOR,IS SUBS

FOR S9A-07/25/72-23:01:39 (+0)

FUNCTION POLY

ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000012 1075 0000 000003 INJPS 0000 I 000002 K 0000 I 000001 L 0000 R 000000 POLY

00101	1*		FUNCTION POLY(N,A,X)
00101	2*	C	
00101	3*	C	EVALUATES POLYNOMIALS
00101	4*	C	
00103	5*		DIMENSION A(N)
00104	6*		POLY = 0.
00105	7*		L = N
00106	8*		DO 1 K=1,N
00111	9*		POLY = POLY*X+A(L)
00112	10*	1	L = L-1
00114	11*		RETURN
00115	12*		END

END OF COMPILATION:

NO DIAGNOSTICS.

OFOR:IS SUB6  
FOR 59A-07725/72-23:01:48 (.0)

SUBROUTINE RKS ENTRY POINT 000643

STORAGE USED: CODE(1) 0010401 DATA(0) 0000641 BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR2\$  
0004 NEXP5\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000010	10L	0001	000313	110L	0001	000333	120L	0001	000045	126G	0001	000343	130L
0001	000071	140G	0001	000355	140L	0001	000105	146G	0001	000130	156G	0001	000417	160L
0001	000150	164G	0001	000177	174G	0001	000500	185L	0001	000510	190L	0001	000013	20L
0001	000232	205G	0001	000270	217G	0001	000524	220L	0001	000530	230L	0001	000543	240L
0001	000374	243G	0001	000032	25L	0001	000552	250L	0001	000554	251L	0001	000572	257L
0001	000604	259L	0001	000623	270L	0001	000456	300L	0001	000615	336G	0001	000054	40L
0001	000060	45L	0001	000006	5L	0001	000076	50L	0001	000123	70L	0001	000135	80L
0000 R	000014	AM	0000 R	000007	AMAX	0000 R	000011	C	0000 R	000010	D	0000 R	000001	DDT
0000 R	000003	DELT	0000 R	000012	E	0000 R	000000	FR10	0000 I	000004	I	0000 I	000005	IFLAG
0000	000030	INJP\$	0000 I	000002	ISYMP	0000 I	000013	J	0000 R	000006	S			

00101	1*		SUBROUTINE RKSDERIV,CNTRL,Y,DY,A,R,T,DEL,N,IFVD,IBKP,NTRY,	D6000100
00101	2*		1IERR,DELY,PD,SD,YS,YST,DYST,YSIMP)	D6000200
00101	3*	C		
00101	4*	C	INTEGRATION	
00101	5*	C		
00103	6*		DIMENSION Y(N),DY(N),A(N),R(N),DELY(N),	D60 3
00103	7*		IPD(N),SD(N),YS(N),DYST(N),YST(N),YSIMP(N)	D600040
00104	8*		EXTERNAL DERIV, CNTRL	D6000500
00104	9*	C	FR10 IS FIFTH ROOT OF TEN	D600 60
00105	10*		FR10=1.5848932	D6000700
00106	11*		IERR=0	D6000800
00106	12*	C	YS CONTAINS Y VALUES AT LEFT END POINT OF INTEGRATION INTERVAL	D600090
00106	13*	C		D6001000
00106	14*	C	YSIMP CONTAINS Y FOR SIMPSONS RULE CHECK CHECK NOT MADE FOR	D6001100
00106	15*	C	FIXED STEP MODE ISYMP IS CONTROL PARAMETER =1, FIXED, 2 VAR	D600120
00106	16*	C		D6001300
00106	17*	C	IF FIXED STEP SIZE GO ONE INTERVAL OF LENGTH DELT AND RETURN TO	D600140
00106	18*	C	CNTRL, IF VAR GO TWO INTERVALS BEFORE RETURN TO CNTRL	D600150
00106	19*	C		D6001600
00106	20*	C	IFVD = 0 VARIABLE INTERVAL	D600170
00106	21*	C	= 1 FIXED	D6001800
00106	22*	C	IBKP = 0 CUT INTERVAL ONCE BEFORE REPEAT (UNDER IFVD=0 )	D600190
00106	23*	C	= 1 CUT AS REQUIRED	D60 200
00106	24*	C	NTRY = 1 CONTINUE INTEGRATING	D6002100

00106	25*	C	2	RETURN FROM RKS	D60 220
00106	26*	C	3	STEP REPEATED WITH NEW DELT	D6002300
00106	27*	C	4	RESTART	D600240
00106	28*	C	IERR = 0	NORMAL	D6002500
00106	29*	C	-1	DELT=0, RETURN FROM RKS	D6002600
00106	30*	C	1	A(I)+ R(I)*ABS(Y(I)) = 0. , RETURN FROM RKS	D6002700
00107	31*		5	IF(DELT) 20,10,20	D6002800
00112	32*		10	IERR=-1	D6002900
00113	33*			GO TO 270	D7003000
00114	34*		20	CALL DERIV(Y,DY,T)	D6003100
00115	35*			NTRY=1	D6003200
00116	36*			CALL CNTRL(Y,DY,DELT,T,NTRY,IFVD)	D6003300
00117	37*		25	DDT=DELT	D6003400
00120	38*			IF(IFVD) 40,30,40	D6003500
00123	39*		30	ISYMP=2	D6003600
00124	40*			DELT=DELT/2.	D600370
00125	41*			DO 31 I=1,N	D7003800
00130	42*		31	SD(I)=0.0	D600390
00132	43*			IFLAG=1	D600400
00133	44*			S=1.	D6004100
00134	45*			GO TO 45	D6004200
00135	46*		40	ISYMP=1	D6004300
00136	47*			DELT=DELT	D600440
00137	48*		45	DO 46 I=1,N	D6004500
00142	49*			YST(I)=Y(I)	D600460
00143	50*		46	DYST(I)=DY(I)	D60 470
00145	51*		50	DO 60 I=1,N	D6004800
00150	52*			DELY(I)=DELT*DY(I)	D6004900
00151	53*			PD(I)=DELY(I)	D600500
00152	54*		60	CONTINUE	D6005100
00154	55*			GO TO (80,70),ISYMP	D6005200
00155	56*		70	DO 71 I=1,N	D6005300
00160	57*		71	SD(I)=SD(I)+S*DY(I)	D6005400
00162	58*		80	T=T+DELT/2.	D6005500
00163	59*			DO 85 I=1,N	D6005600
00166	60*			YS(I)=Y(I)	D600570
00167	61*			Y(I)=YS(I)+DELY(I)/2.	D6005800
00170	62*		85	CONTINUE	D6005900
00172	63*			CALL DERIV(Y,DY,T)	D6006000
00173	64*			DO 90 I=1,N	D6006100
00176	65*			DELY(I)=DELT*DY(I)	D600620
00177	66*			PD(I)=PD(I)+2.*DELY(I)	D6006300
00200	67*			Y(I)=YS(I)+DELY(I)/2.	D600640
00201	68*		90	CONTINUE	D6006500
00203	69*			CALL DERIV(Y,DY,T)	D6006600
00204	70*			DO 95 I=1,N	D600670
00207	71*			DELY(I)=DELT*DY(I)	D600680
00210	72*			PD(I)=PD(I)+2.*DELY(I)	D6006900
00211	73*			Y(I)=YS(I)+DELY(I)	D600700
00212	74*		95	CONTINUE	D6007100
00214	75*			T=T+DELT/2.	D6007200
00215	76*			CALL DERIV(Y,DY,T)	D600730
00216	77*			DO 100 I=1,N	D6007400
00221	78*			DELY(I)=DELT*DY(I)	D6007500
00222	79*			PD(I)=PD(I)+DELY(I)	D600760
00223	80*			Y(I)=YS(I)+PD(I)/6.	D600770
00224	81*		100	CONTINUE	D6007800



00226	82*	GO TO (110,120),ISYMP	D6007900
00227	83*	110 NTRY=1	D6008000
00230	84*	CALL DERIV(Y,DY,T)	D6008100
00231	85*	CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)	D600820
00232	86*	GO TO 300	D6008300
00233	87*	120 GO TO (130,140),IFLAG	D6008400
00234	88*	130 S=4.	D6008500
00235	89*	IFLAG=2	D6008600
00236	90*	CALL DERIV(Y,DY,T)	D6008700
00237	91*	GO TO 50	D7008800
00240	92*	140 CALL DERIV(Y,DY,T)	D6008900
00241	93*	AMAX =0.0	D6009000
00242	94*	DO 180 I=1,N	D6009100
00245	95*	SD(I)=SD(I)+DY(I)	D6009200
00246	96*	YSIMP(I)=YST(I)+DELT*SD(I)/3.	D6009300
00247	97*	D =ABS(Y(I)-YSIMP(I))	D600940
00250	98*	C =A(I)+R(I)*ABS(Y(I))	D6009500
00251	99*	IF(C ) 160,150,160	D600960
00254	100*	150 IERR=1	D6009700
00255	101*	GO TO 270	D6009800
00256	102*	160 E =ABS(D /C )	D6009900
00257	103*	AMAX=AMAX1(AMAX,E)	D601000
00260	104*	180 CONTINUE	D7010100
00262	105*	IF(AMAX-1.) 215,215,230	D601020
00265	106*	215 NTRY= 1	D6010300
00266	107*	CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)	D6010400
00267	108*	300 IF(NTRY-1) 185,185,310	D601050
00272	109*	310 IF(NTRY-2) 270,270,330	D601060
00275	110*	330 IF(NTRY-3) 340,340,5	D601070
00300	111*	340 T=T-DOT	D601080
00301	112*	IF(DEL) 259,10,259	D6010900
00304	113*	185 GO TO (40,190),ISYMP	D601100
00305	114*	190 IF(AMAX-.75) 200,25,220	D6011100
00310	115*	200 IF(AMAX-.075) 210,25,25	D601120
00313	116*	210 DEL=DEL*FR10	D6011300
00314	117*	GO TO 25	D6011400
00315	118*	220 DEL=DEL/FR10	D6011500
00316	119*	GO TO 25	D6011600
00317	120*	230 I =1+ IBKP	D6011700
00320	121*	GO TO (240,250),I	D6011800
00321	122*	240 T=T-DEL	D6011900
00322	123*	DEL=DEL/FR10	D6012000
00323	124*	GO TO 259	D601210
00324	125*	250 J=1	D6012200
00325	126*	251 AM=AMAX/10.**J	D6012300
00326	127*	IF(1.-AM) 255,257,257	D601240
00331	128*	255 J=J+1	D6012500
00332	129*	GO TO 251	D601260
00333	130*	257 T=T-DEL	D601270
00334	131*	DEL=DEL/(FR10**J)	D601280
00335	132*	259 DO 245 I=1,N	D601290
00340	133*	DY(I)=DYST(I)	D6013000
00341	134*	245 Y(I)=YST(I)	D6013100
00343	135*	GO TO 25	D6013200
00344	136*	270 RETURN	D6013300
00345	137*	END	D6013400

END OF COMPILATION:

NO DIAGNOSTICS.

Q XOT

MAP 0023-07/25-23:02

ADDRESS LIMITS 001000 013206 040000 044714

STARTING ADDRESS 012523

WORDS DECIMAL 5255 IBANK 2509 DBANK

SEGMENT MAIN		001000 013206	040000 044714
NSWTC\$/FOR	1	001000 001021	
NRBLK\$/FOR	1	001022 001044	
NRWNO\$/FOR	1	001045 001124	2 040000 040011
NWEF\$/FOR	1	001125 001326	2 040012 040031
NFTCH\$/FOR	1	001327 001617	2 040032 040067
NBDCV\$/FOR	1	001620 001752	2 040070 040125
NFTV\$/FOR	1	001753 001775	
NCNVT\$/FOR	1	001776 002222	2 040126 040215
NCLOS\$/FOR	1	002223 002371	2 040216 040247
NwBLK\$/FOR	1	002372 002513	
NBSBL\$/FOR	1	002514 002550	
NUPJAS\$/FOR	1	002551 002603	
NBF00\$/FOR			2 040250 042451
NININ\$/FOR	1	002604 003014	2 042452 042463
NINPT\$/FOR	1	003015 003657	2 042464 042503
NOTIN\$/FOR	1	003660 004173	2 042504 042507
NOU\$/FOR	1	004174 005162	2 042510 042534
NFMIS\$/FOR	1	005163 006041	2 042535 042611
NIDERS\$/FOR	1	006042 006214	2 042612 042716
NFCMK\$/FOR	1	006215 007076	2 042717 043055
			4 043056 043127
NTAB\$/FOR			2 043130 043166
ERUS/MISC			
NIBJF\$/FOR	1	007077 007141	2 043167 043167
TIRS/TECH	1	007142 007626	0 043170 043220
			2 043221 043500
SQRT\$/FOR	1	007627 007667	2 043501 043512
ASINCOSS\$/FOR	1	007670 010104	0 043513 043540
NIER\$/FOR	1	010105 010166	2 043541 043674
NOBJF\$/FOR	1	010167 010233	
EXP\$/FOR	1	010234 010323	2 043675 043715
NEXP\$/FOR	1	010324 010411	2 043716 043725
NERR\$/FOR	1	010412 010736	2 043726 044071
SDR (COMMON BLOCK)			044072 044105
MCT (COMMON BLOCK)			044106 044113
SDC (COMMON BLOCK)			044114 044116
BLANK\$COMMON (COMMON BLOCK)			
SUB6	1	010737 011776	0 044117 044202
			2 BLANK\$COMMON

SUB5	1	011777 012042	0	044203 044217
02 SUB4	1	012043 012106	2	BLANKSCOMMON
			0	044220 044240
			2	BLANKSCOMMON
SUB3	1	012107 012152	0	044241 044262
			2	BLANKSCOMMON
SUB2	1	012153 012345	0	044263 044300
	3	SDC	2	BLANKSCOMMON
			4	MCT
SUB1	1	012346 012522	0	044301 044337
	3	SDR	2	BLANKSCOMMON
			4	SDC
MAIN	1	012523 013206	0	044340 044714
	3	SDR	2	BLANKSCOMMON
			4	MCT

SYSS\*RLIBS. LEVEL 63

END OF COLLECTION - TIME 1.677 SECONDS

## APPENDIX B

## COMPUTER CODE FOR MINIMUM

## AREA OPTIMIZATION

2 FOR, IS MAIN  
FOR S9A-07/27/72-17:43:15 (,0)

# MAIN PROGRAM

STORAGE USED: CODE(1) 001045; DATA(0) 000677; BLANK COMMON(2) 000000

## COMMON BLOCKS:

0003 BLK1 0000 4  
0004 BLK2 0000 2  
0005 BLK3 0000 5  
0006 BLK4 0000 2  
0007 BLK5 0000 2

## EXTERNAL REFERENCES (BLOCK, NAME)

0010 SERV  
0011 SCHTL  
0012 REST  
0013 WSERV  
0014 ETA  
0015 DETA  
0016 WSERV  
0017 WSER  
0020 INITIAL  
0021 INITIAL  
0022 RAS  
0023 FMINV  
0024 ENDEND  
0025 PENDING  
0026 WNTRE  
0027 ARJUF  
0030 N1025  
0031 WJUS  
0032 ASIN  
0033 SERT  
0034 WJUS  
0035 STOP1

## STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000002 IL	0001 000163 100L	0000 000421 1000F	0001 000333 105L	0001 000351 105L
0001 000354 109L	0001 000361 110L	0000 000423 2000F	0000 000454 2010F	0000 000467 2040F
0000 000535 205F	0000 000622 2050F	0000 000616 2070F	0001 001033 2080L	0000 000631 2085F
0000 000514 210F	0000 000521 215F	0000 000602 220F	0000 000533 230F	0001 000516 233G
0000 000547 240F	0000 000564 250F	0001 001024 300L	0001 001041 3000L	0001 000725 340G
0001 000020 50L	0001 000160 90L	0000 000462 900F	0000 R 000050 A	0000 R 000337 AV
0000 R 000353 ASTA	0000 R 000362 AYY	0004 000003 A1	0004 000004 A2	0004 000005 A3
0004 000100 A4	0004 000007 A5	0004 000010 A6	0000 R 000363 BYY	0015 R 000000 DETA
0000 R 000367 DET2	0003 000023 DE10L	0003 000021 DE40TH	0003 R 000013 DFM	0003 R 000012 DFMX
0003 R 000010 DFNX	0006 R 000006 DLNCS	0006 R 000001 DLDU	0006 R 000002 DLDV	0005 R 000000 DLMY
0000 R 000120 DLY	0007 000000 DNCSDV	0007 000001 DNCSDV	0003 R 000007 DNCDT1	0000 R 000405 DX
0005 R 000003 DXWRT	0000 R 000024 DY	0000 R 000240 DYST	0000 R 000370 DYY	0000 R 000373 D2CD2Y

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0000 R 000374 D2CT1B 0000 R 000402 D2E4LN 0000 R 000376 D2E4LU 0000 R 000400 D2E4LV 0000 R 000401 D2E4TN
0000 R 000375 D2E4TV 0000 R 000377 D2E4TV 0003 R 000016 D2FMLL 0006 R 000003 D2LDVV 0006 R 000004 D2LDIV
0006 R 000005 D2LDV 0006 R 000007 D2LNCR 0006 R 000010 D2LVNB 0006 R 000011 D2LVNB 0003 R 000015 D2MNC
0003 R 000014 D2VLIC 0003 R 000011 D2NDLL 0004 R 000001 D3M2LN 0004 R 000002 D3M2NL 0004 R 000011 E3
0014 R 000000 ETA 0000 R 000365 FTY 0000 R 000364 FTY 0003 R 000022 E3 0003 R 000002 F4
0003 R 000003 EN 0003 R 000005 FNC 0003 R 000004 FNCBR 0000 I 000403 I 0000 I 000410 IAKP
0005 I 000001 ICNT 0000 I 000412 JERR 0000 I 000406 IFVD 0000 I 000413 LCNT 0005 I 000004 LSTEP
0000 I 000407 N 0000 I 000357 NCT 0000 I 000354 NIT 0000 I 000414 NITMAX 0000 I 000411 NTRY
0000 R 000144 PD 0000 R 000074 R 0000 R 000355 RHOD 0006 R 000000 RLA 0011 R 000000 SCNTL
0000 R 000170 S2 0010 R 000000 SDRV 0003 R 000000 THETAB 0000 R 000372 THETA 0003 R 000001 THETAS
0000 R 000356 TH4 0000 R 000360 T2 0000 R 000361 T3 0004 R 000000 U 0003 R 000006 V
0000 R 000354 X1 0005 R 000002 XWRT 0000 R 000000 Y 0000 R 000214 YS 0000 R 000310 YSIMP
0000 R 000264 YST 0000 R 000366 YY 0003 R 000017 Y1 0003 R 000020 Y2 0000 R 000371 Z2
0000 R 000404 ZZZ 0000 R 000415 Z1 0010 R 000416 Z2 0000 R 000417 Z3 0000 R 000420 Z5

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00100 1* C
00100 2* C RADIATOR SYSTEMS OPTIMIZATION FOR MINIMUM AREA
00100 3* C
00101 4* EXTERNAL SDRV,SCNTL
00103 5* DIMENSION Y(20),DY(20),A(20),R(20),PLY(20),PD(20),SP(20),
00103 6* 1 YS(20),DYST(20),YST(20),YSIMP(20),XM(3),AM(3,4)
00104 7* COMMON /BLKT/THETAB,THETAS,FN,FNCBR,FNC,V,DNCOT1,DFMX,D2NDLL,
00104 8* 1 DFMX,DFM,D2MLNC,D2MNC,D2FMLL,Y1,Y2,DE4DTR,E3,DE1DL/
00104 9* 2 BLKD/U,D342LN,D342NL,A1,A2,A3,A4,A5,A6,EB
00104 10* 3 /NCT/PLMT,ICNT,XCRT,DXRT,LSTEP
00104 11* 4 /BLKP/RLAM,DL0U,DL0V,D2DUU,D2LDVV,D2LDNCB,
00104 12* 5 D2LNCR,D2LVNB,D2LVNB
00104 13* 6 /BLKP/INCRD1,INCRDV
00105 14* FFS(X) = (SORT(X*(X+2.0))+AS1*(1.0/(X+1.0))-1.570796)/X
00106 15* FFN(X,Y) = 1.0-Y*(0.1460*X-Y-0.02166)/X
00107 16* DFF(X,Y) = (-0.2920*Y+0.02866)/X
00110 17* 1 RLA(5,1000,E=3000) THETAS,U,V,FNCBR,ASTAR,DXWRT
00120 18* 100, FORMAT(5F16.8)
00121 19* NIT = 1
00122 20* 5 IF (U.LT.4.00E-06.OR.U.GT.250.1) GO TO 1
00124 21* IF (V.LT.4.00E-05.OR.V.GT.50.0) GO TO 1
00126 22* IF (FNCBR.LT.0.00009.OR.FNCBR.GT.4.0) GO TO 1
00130 23* 5, CALL REST(U,V,ASTAR,RLAM,DL0U,DL0V,D2DUU,D2LDVV,D2LDNCB,
00130 24* 10LDNCB,D2LNCR,D2LVNB,D2LVNB)
00131 25* CALL NDERV(RLAM,FN,DFMX,D2NDLL)
00132 26* R100 = RLA
00133 27* WRITE(6,200) THETAS,ASTAR,U,V,FNCBR,RLAM
00143 28* 200, FORMAT (1H1,10X,10THETA=5 = F20.8/
00143 29* 4 11X,10H ASTAR = F20.6//
00143 30* 1 11X,10H U = F20.6/
00143 31* 2 11X,10H V = F20.6/
00143 32* 3 11X,10H FNCBR = F20.6/
00143 33* 5 11X,10H LAMBDA = F20.6//)
00144 34* WRITE(6,201) NIT
00147 35* 201, FORMAT (5X,NUMBER OF ITERATIONS = I2)
00150 36* TH4 = THETAS**4
00151 37* IF (V.GT.0.2) GO TO 90
00153 38* IF (V.LE.1.0E-10) GO TO 108
00155 39* 9, THETAB = 0.9

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02 00156 40* 95 NCT = 0
00157 41* 00 T2 = THETAB**2
00160 42* T3 = T2*THETAB
00161 43* FNC = FNCDBR+T3
00162 44* FM = FEN(RH00,FNC)*FFS(RH00)
00163 45* DFM = DEFN(RH00,FNC)*FFS(RH00)
00164 46* AYY = T3*THETAB-TH4
00165 47* BYY = 1.0-THETAB
00166 48* ETTY = ETA(FNC)
00167 49* EY = ETTY+FM+FN
00170 50* YY = 1.0/V-ETTY+AYY/BYY
00171 51* DETZ = (DFM+ETTY+FM*DETA(FNC))*3.0+T2*FNCDBR
00172 52* DYY = -AYY/3YY*(DETZ+ETTY/BYY)-4.0*ETTY*T3/BYY
00173 53* ZZ = THETAB-YY/DYY
00174 54* IF(NCT.GT.20) GO TO 109
00176 55* IF(ZZ.LT.1.0) GO TO 105
00200 56* THETAB = (THETAB+1.0)/2.0
00201 57* NCT = NCT+1
00202 58* GO TO 100
00203 59* 05 IF(ABS(ZZ-THETAB)/ZZ.LT.1.0E-06) GO TO 110
00205 60* NCT = NCT+1
00206 61* THETAB = ZZ
00207 62* GO TO 100
00210 63* 08 THETAB = 1.0
00211 64* GO TO 110
00212 65* 09 WRITE(6,900)
00212 66* C
00214 67* 10 Y(1) = 1.0
00215 68* Y(2) = THETAB
00216 69* 0n FORMAT(1H0,20HVEYTON-RAPHSON FAILS)
00217 70* THETAF = Y(1)
00220 71* Y1 = Y(1)
00221 72* Y2 = Y(2)
00222 73* CALL NCDSRV(FNCBR,THETAB,FNC,NCDT1,D2C02T,D2CTNB)
00223 74* CALL MDER(FLAM,FNC,FM,DFMX,DFM,D2FMLL,D2MDNC,D2MLNC,D342LN,D3M2NL)
00224 75* CALL INTIAL(OLDV,D2LDVV,Y(3),Y(4),Y(5),Y(6),0.,1.,D2E4TU,D2E4LU)
00225 76* CALL INTIAL(OLDV,D2LDVV,Y(7),Y(8),Y(9),Y(10),0.,0.,D2E4TV,D2E4LV)
00226 77* CALL INTIAL(OLDV,D2LNCB,Y(11),Y(12),Y(13),Y(14),1.,1.,D2E4TN,
00226 78* D2E4LN)
00227 79* CALL INTMIX(D2E4TV,Y(4),OLDV,D2E4LV,D2LDVV,Y(16),Y(15))
00230 80* CALL INTMIX(D2E4TN,Y(4),OLDV,D2E4LN,D2LUNB,Y(18),Y(17))
00231 81* CALL INTMIX(D2E4TN,Y(8),OLDV,D2E4LN,D2LVNB,Y(20),Y(19))
00232 82* DO 115 I=1,20
00235 83* A(I) = 1.0E-05
00236 84* 15 R(I) = 1.0E-05
00236 85* C
00240 86* WRITE(6,2040)
00242 87* 2 40 FORMAT(/7X,6HTHETAF,9X,6HTHETAB,9X,2HNC,13X,6HETABAR,
00242 88* 1 9X,3HETA,13X,1HM,14X,1HN,14X,4HZETA/)
00242 89* C
00243 90* ZZ = 0.0
00244 91* DX = 1.0/(U*(1.0-THETAB))
00245 92* IFV0 = 0
00246 93* IF(DX.GT.DXWRT) DX = DXWRT
00250 94* N = 20
00251 95* IBKP = 1
00252 96* NTRY = 1

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C 2 00253 97* IERR = 0
00253 98* C
00254 99* CLMT = DXWRT/500.0
00255 100* XWRT = 0.0
00256 101* LSTEP = 0
00257 102* LCNT = 0
00260 103* ICNT = 0
00261 104* CALL RKS(SDRV,SCNTL,Y,DY,A,R,ZZ2,DX,N,IFVD,IBKP,NTRY,IERR,
00261 105* ,DLY,PD,SD,YS,YST,OYST,YSIMP)
00252 106* XM(1) = -Y(3)
00263 107* XM(2) = -Y(7)
00264 108* XM(3) = -Y(11)
00265 109* AM(1,1) = Y(5)
00266 110* AM(1,2) = Y(15)
00267 111* AM(1,3) = Y(17)
00270 112* AM(2,1) = Y(15)
00271 113* AM(2,2) = Y(9)
00272 114* AM(2,3) = Y(19)
00273 115* AM(3,1) = Y(17)
00274 116* AM(3,2) = Y(19)
00275 117* AM(3,3) = Y(13)
00276 118* CALL FMIN(Y,AM,XM,3,4)
00277 119* WRITE(6,205)
00301 120* 205 FORMAT (///45X,27H PARAMETERS AT END OF TUBE //)
00302 121* WRITE(6,210) Y(1)
00305 122* 210 FORMAT (1X,51(,8HTHETAF =,F10.6,/)
00306 123* WRITE(6,215) U,V,FNCBR
00313 124* 215 FORMAT (1X,23(,34U =,E14.6,15X,3HV =,E14.6,12X,7HNCBR =,E14.6,/)
00314 125* WRITE(6,230) Y(3),Y(7),Y(11)
00321 126* 230 FORMAT (1X,19(,7H2TFDU =,E14.6,12X,7H2TFDV =,E14.6,9X,
00321 127* 10H2FNCBR =,E14.6,/)
00322 128* WRITE(6,240) Y(5),Y(9),Y(13)
00327 129* 240 FORMAT (1X,17(,9H2TFD2U =,E14.6,10X,9H2TFD2V =,E14.6,6X,
00327 130* 13H2TFD2NCBR =,E14.6,/)
00330 131* WRITE(6,250) Y(15),Y(17),Y(19)
00335 132* 250 FORMAT (1X,15X,10H2TFDUDV =,E14.6,5X,14H2TFDUDNCBR =,E14.6,
00335 133* 5X,14H2TFDUDNCBR =,E14.6,/)
00336 134* WRITE(6,220) (XM(I),I=1,3)
00344 135* 220 FORMAT (/19X9DELTA U =,E14.6,10X,9HDELTA V =,E14.6,6X,
00344 136* 13HDELTA NCBR =,E14.6,/)
00345 137* NITMAX = 20
00346 138* 200 IF (NIT,GT,NITMAX) GO TO 300
00350 139* NIT = NIT+1
00351 140* Z1 = U
00352 141* Z2 = V
00353 142* Z3 = FNCBR
00354 143* CALL BNDCND(Y,U,V,FNCBR,YM)
00355 144* CALL RESBND(U,V,XM,ASTAR)
00356 145* Z5 = Y(1)-THETAS
00357 146* IF(Z5,LT,0.0001) GO TO 2080
00361 147* IF (ABS(XM(1))/Z1,GT,0.0001) GO TO 50
00363 148* IF (ABS(XM(2))/Z2,GT,0.0001) GO TO 50
00365 149* IF (ABS(XM(3))/Z3,GT,0.0001) GO TO 50
00367 150* WRITE(6,2070)
00371 151* 2070 FORMAT(1H0,15HOPTIMUM REACHED)
00372 152* GO TO 1
00373 153* 300 WRITE(6,2050) NITMAX

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00376	154*	2050	FORMAT (1X,30H NUMBER OF ITERATIONS EXCEEDS ,I2)
00377	155*		GO TO 1
00400	156*	2080	WRITE(6,2085)
00402	157*	2085	FORMAT(1H0,24HVANISHING HEAT REJECTION)
00403	156*		GO TO 1
00404	159*	3000	STOP
00405	160*		END

END OF COMPILATION: NO DIAGNOSTICS.

02

2FOR,IS RKS  
FOR 59A-07/27/72-17:43:28 (,0)

SUBROUTINE RKS ENTRY POINT 000643

STORAGE USED: CODE(1) 001040; DATA(0) 000064; BLANK COMMON(2) 000000

EXTERNAL REFERENCE (BLOCK, NAME)

0003 NERR2\$  
0004 NEXP5\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000010	10L	0001	000313	110L	0001	000333	120L	0001	000045	126L	0001	000343	130L
0001	000071	140L	0001	000355	140L	0001	000105	146L	0001	000130	156L	0001	000417	160L
0001	000150	164L	0001	000177	174L	0001	000500	185L	0001	000510	190L	0001	000013	20L
0001	000232	205L	0001	000270	217L	0001	000524	220L	0001	000530	230L	0001	000543	240L
0001	000374	243L	0001	000032	25L	0001	000552	250L	0001	000554	251L	0001	000572	257L
0001	000604	259L	0001	000623	270L	0001	000456	300L	0001	000615	336L	0001	000054	40L
0001	000060	45L	0001	000006	5L	0001	000076	50L	0001	000123	70L	0001	000135	80L
0000 R	000014	AM	0000 R	000007	AMAX	0000 R	000011	C	0000 R	000010	D	0000 R	000001	DDY
0000 R	000003	DEL	0000 R	000012	F	0000 R	000000	FR10	0000 I	000004	I	0000 I	000005	IFLAG
0000	000030	INJ	0000 I	000002	ISYMP	0000 I	000013	J	0000 R	000006	S			

00101	1*		SUBROUTINE RKS(deriv,ctrl,y,ty,a,r,t,del,n,ifvd,irkp,ntry,	
00101	2*		iterr,deley,pc,sn,ys,yst,dyst,ytimp)	0600020
00103	3*		dimension y(n),dy(n),a(n),r(n),deley(n),	060 3
00103	4*		ip(n),so(n),ys(n),dyst(n),yst(n),ysimp(n)	0600040
00104	5*		external deriv,ctrl	06000500
00104	6*	C	fr10 is fifth root of ten	0600 60
00105	7*		fr10=1.5848932	06000700
00106	8*		iterr=0	06000800
00106	9*	C	ys contains y values at left end point of integration interval	0600090
00106	10*	C		06001000
00106	11*	C	ysimp contains y for simpson's rule check check not made for	06001100
00106	12*	C	fixed step mode isymp is control parameter =1, fixed=2 vard	0600120
00106	13*	C		06001300
00106	14*	C	if fixed step size go one interval of length delt and return to	0600140
00106	15*	C	ctrl, if var go two intervals before return to ctrl	0600150
00106	16*	C		06001600
00106	17*	C	ifvd = 0 variable interval	0600170
00106	18*	C	= 1 fixed	06001800
00106	19*	C	irkp = 0 cut interval once before repeat (under ifvd=0)	0600190
00106	20*	C	= 1 cut as required	060 200
00106	21*	C	ntry = 1 continue integrating	06002100
00106	22*	C	2 return from rks	060 220
00106	23*	C	3 step repeated with new delt	06002300
00106	24*	C	4 restart	0600240

00106	25*	C	IERR = 0	NORMAL	06002500
00106	26*	C	-1	DEL=0, RETURN FROM RKS	06002600
00106	27*	C	1	A(I)+R(I)*ABS(Y(I)).= 0., RETURN FROM RKS	06002700
00107	28*		5	IF (DEL) 20,10,20	06002800
00112	29*		1	IERR=-1	06002900
00113	30*			GO TO 270	07003000
00114	31*		2	CALL DERIV(Y,DY,T)	06003100
00115	32*			NTRY=1	06003200
00116	33*			CALL CNTRL(Y,DY,DEL,T,NTRY,IFV)	06003300
00117	34*		2	DEL=DEL	06003400
00120	35*			IF (IFV) 40,30,40	06003500
00123	36*		3	ISYMP=2	06003600
00124	37*			DEL=DEL/2.	06003700
00125	38*			DO 31 I=1,N	07003800
00130	39*		3	SJ(I)=0.0	06003900
00132	40*			IFLAG=1	06004000
00133	41*			S=1.	06004100
00134	42*			GO TO 45	06004200
00135	43*		4	ISYMP=1	06004300
00136	44*			DEL=DEL	06004400
00137	45*		4	DO 46 I=1,N	06004500
00142	46*			YST(I)=Y(I)	06004600
00143	47*		4	DYST(I)=DY(I)	06004700
00145	48*		5	DO 60 I=1,N	06004800
00150	49*			DELY(I)=DEL+DY(I)	06004900
00151	50*			PD(I)=DELY(I)	06005000
00152	51*		6	CONTINUE	06005100
00154	52*			GO TO (80,70),ISYMP	06005200
00155	53*		7	DO 71 I=1,N	06005300
00160	54*		7	SJ(I)=SJ(I)+S*DY(I)	06005400
00162	55*		8	T=T+DEL/2.	06005500
00163	56*			DO 85 I=1,N	06005600
00166	57*			YS(I)=Y(I)	06005700
00167	58*			Y(I)=YS(I)+DELY(I)/2.	06005800
00170	59*		8	CONTINUE	06005900
00172	60*			CALL DERIV(Y,DY,T)	06006000
00173	61*			DO 90 I=1,N	06006100
00176	62*			DELY(I)=DEL+DY(I)	06006200
00177	63*			PD(I)=PD(I)+2.*DELY(I)	06006300
00200	64*			Y(I)=YS(I)+DELY(I)/2.	06006400
00201	65*		9	CONTINUE	06006500
00203	66*			CALL DERIV(Y,DY,T)	06006600
00204	67*			DO 95 I=1,N	06006700
00207	68*			DELY(I)=DEL+DY(I)	06006800
00210	69*			PD(I)=PD(I)+2.*DELY(I)	06006900
00211	70*			Y(I)=YS(I)+DELY(I)	06007000
00212	71*		9	CONTINUE	06007100
00214	72*			I=I+DEL/2.	06007200
00215	73*			CALL DERIV(Y,DY,T)	06007300
00216	74*			DO 120 I=1,N	06007400
00221	75*			DELY(I)=DEL+DY(I)	06007500
00222	76*			PD(I)=PD(I)+DELY(I)	06007600
00223	77*			Y(I)=YS(I)+PD(I)/6.	06007700
00224	78*		10	CONTINUE	06007800
00226	79*			GO TO (110,120),ISYMP	06007900
00227	80*		11	NTRY=1	06008000
00230	81*			CALL DERIV(Y,DY,T)	06008100

00231	82*	CALL CNTRL(Y,DY,DEL,T,NTRY,IFV)	D600820
00232	83*	GO TO 300	D6008300
00233	84*	12) GO TO (130,140),IFLAG	D6008400
00234	85*	13) C=4.	D6008500
00235	86*	IFLAG=2	D6008600
00236	87*	CALL DERIV(Y,DY,T)	D6008700
00237	88*	GO TO 50	D7008800
00240	89*	14) CALL DERIV(Y,DY,T)	D6008900
00241	90*	AMAX=0.0	D6009000
00242	91*	DO 180 J=1,N	D6009100
00245	92*	SD(I)=SD(I)+DY(I)	D6009200
00246	93*	YSIMP(I)=YST(I)+DEL*SD(I)/3.	D6009300
00247	94*	D=ABS(Y(I)-YSIMP(I))	D6009400
00250	95*	C=A(I)+R(I)*ABS(Y(I))	D6009500
00251	96*	IF(C) 160,150,160	D6009600
00254	97*	15) IERR=1	D6009700
00255	98*	GO TO 270	D6009800
00256	99*	16) E=ABS(D)/C	D6009900
00257	100*	AMAX=AMAX1(AMAX,E)	D6010000
00260	101*	18) CONTINUE	D7010100
00262	102*	IF(AMAX-1.) 215,215,230	D6010200
00265	103*	21) NTRY=1	D6010300
00266	104*	CALL CNTRL(Y,DY,DEL,T,NTRY,IFV)	D6010400
00267	105*	30) IF(NTRY-1) 185,185,310	D6010500
00272	106*	31) IF(NTRY-2) 270,270,330	D6010600
00275	107*	33) IF(NTRY-3) 340,340,5	D6010700
00300	108*	34) T=T-20T	D6010800
00301	109*	IF(DEL) 259,10,259	D6010900
00304	110*	18) GO TO (40,190),ISYVP	D6011000
00305	111*	19) IF(AMAX-.75) 200,25,220	D6011100
00310	112*	20) IF(AMAX-.075) 210,25,25	D6011200
00313	113*	21) DEL=DEL*FR10	D6011300
00314	114*	GO TO 25	D6011400
00315	115*	22) DEL=DEL/FR10	D6011500
00316	116*	GO TO 25	D6011600
00317	117*	23) I=1+I9KP	D6011700
00320	118*	GO TO (240,250),I	D6011800
00321	119*	24) T=T-DEL	D6011900
00322	120*	DEL=DEL/FR10	D6012000
00323	121*	GO TO 259	D6012100
00324	122*	25) J=1	D6012200
00325	123*	25) AM=AMAX/10.**J	D6012300
00326	124*	IF(1.-AM) 255,257,257	D6012400
00331	125*	255) J=J+1	D6012500
00332	126*	GO TO 251	D6012600
00333	127*	257) T=T-DEL	D6012700
00334	128*	DEL=DEL/(FR10**J)	D6012800
00335	129*	259) GO 245 I=1,N	D6012900
00340	130*	DY(I)=DYST(I)	D6013000
00341	131*	245) Y(I)=YST(I)	D6013100
00343	132*	GO TO 25	D6013200
00344	133*	270) RETURN	D6013300
00345	134*	END	D6013400

END OF COMPILATION: NO DIAGNOSTICS.

FOR IS SDRV  
FOR 59A-67/27/72-17:43:59 (.0)

SUBROUTINE SDRV ENTRY POINT 000601

STORAGE USED: CODE(1) 000605; DATA(0) 000066; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLK1 000024  
0004 BLK2 000012  
0005 BLK3 000012  
0006 VAB 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0007 ETA  
0010 MCDERV  
0011 MDER  
0012 BETA  
0013 DERV  
0014 MIXDER  
0015 MERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000003 AAR	0004 000003 A1	0005 000004 A2	0006 000005 A3	0007 000006 A4
0004 000007 A5	0004 000010 A6	0006 R 000005 B5B	0000 R 000037 DA1DN	0000 R 000007 DA1DU
0000 R 000023 DA1DV	0000 R 000040 DA2DN	0006 R 000010 DA2DU	0000 R 000024 DA2DV	0000 R 000041 DA3DN
0000 R 000011 DA2DU	0000 R 000025 DA3DV	0000 R 000042 DA4DN	0000 R 000012 DA4DU	0000 R 000026 DA4DV
0000 R 000043 DA5DN	0000 R 000013 DA5DU	0000 R 000027 DA5DV	0000 R 000044 DA6DN	0000 R 000014 DA6DU
0000 R 000030 DA6DV	0000 R 000045 DERDN	0000 R 000004 DERDTB	0000 R 000015 DERDU	0000 R 000031 DERDV
0000 R 000046 DET	0000 R 000047 DEPN	0000 R 000017 DEPU	0000 R 000033 DEPV	0012 R 000000 BETA
0000 R 000016 DEU	0000 R 000032 DEV	0003 000023 DEIDL	0003 000021 DE4DTB	0003 R 000013 DEM
0003 R 000012 DFVX	0003 000010 DFVX	0003 R 000006 DLDNCR	0005 R 000001 DLDU	0005 R 000002 DLDV
0000 R 000050 DMCN	0000 R 000020 DMDU	0000 R 000034 DMDV	0000 R 000036 DMDY	0003 R 000007 DMCDT1
0000 R 000006 DMCDU	0000 R 000022 DMCDV	0000 R 000001 D2CD2T	0000 R 000002 D2CTNB	0003 R 000016 D2FMI L
0005 R 000003 D2LDU	0005 R 000004 D2LDV	0003 R 000005 D2LDVV	0005 R 000007 D2LNCR	0005 R 000010 D2LUNB
0005 R 000011 D2LVN	0003 R 000015 D2MDCN	0003 R 000014 D2MLNC	0000 R 000051 D2MNCN	0000 R 000021 D2MNCU
0000 R 000035 D2MNCV	0003 000011 D2MDLL	0004 R 000001 D3M2LN	0004 R 000002 D3M2NL	0004 R 000011 ER
0006 R 000002 ET	0007 R 000000 ETA	0003 000022 E3	0003 R 000002 FM	0003 R 000003 FM
0003 R 000005 ENC	0003 R 000024 ENCB	0000 000056 INJPS	0005 R 000000 RLAM	0003 R 000000 THETA9
0000 R 000000 THETA9	0003 R 000001 THETAS	0004 R 000000 U	0003 000006 V	0003 R 000017 Y1
0003 R 000020 Y2	0006 R 000000 Z1	0006 R 000001 Z2		

00101	1*	SUBROUTINE SDRV(Y,DY,X)
00101	2*	C
00101	3*	CALCULATION OF DERIVATIVES FOR THE INTEGRATION SUBROUTINE
00101	4*	C
00103	5*	DIMENSION Y(20),DY(20)

```

02 00104 6* COMMON /BLK1/THETAB,THETAS,FM,FN,FNCBR,FNC,V,DNCDT1,DFNX,D2NDLL,
00104 7* 1 DFNX,DFM,D2MLNC,D2MNC,D2FMLL,Y1,Y2,DE4DTB,E3,DE1DL/
00104 8* 2 BLK0/U,D3M2LN,D3M2NL,A1,A2,A3,A4,A5,A6,E8
00104 9* 3 /BLK8/RLAM,DLDU,DLDV,D2LDU,D2LDV,D2LVNB,DLNCR,
00104 10* 4 D2LNCB,D2LUNB,D2LVNB
00104 11* 5 /VAB/ Z1,Z2,ET
00105 12* Z1 = FNC
00106 13* Z2 = FM*ETA(FNC)+FN
00107 14* ET = ETA(FNC)
00110 15* THETA9 = Y(1)
00111 16* THETA9 = Y(2)
00112 17* Y1 = Y(1)
00113 18* Y2 = Y(2)
00114 19* CALL NCDERV(FNCBR,THETAB,FNC,DNCDT1,D2C02T,D2CTNB)
00115 20* CALL MDER(RLAM,FNC,FM,DFMX,DFM,D2FMLL,D2MNC,D2MLNC,D3M2LN,D3M2NL)
00116 21* AAA = THETA9-THETAB
00117 22* DY(1) = -H*AAA
00120 23* EB = FM*ETA(FNC)+FN
00121 24* DEB0TB = 3*THETA3**2+FNCBR*(FM*DETA(FNC)+ETA(FNC)*DFM)
00122 25* BB3 = 4*THETA3**3/(THETA3**4-THETAS**4)
00123 26* DY(2) = -U/(1/AAA+DEB0TB/EB+BB3)
00124 27* CALL DERV(Y(3),Y(4),Y(5),Y(6),DLDU,D2LDU,DY(3),DY(4),DY(5),
00124 28* 1 DY(6),0.,1.,DNCDU,DA1DU,DA2DU,DA3DU,DA4DU,DA5DU,
00124 29* 2 DA6DU,DEB0U,DEU,DEU,DMDU,D2MNCU)
00125 30* CALL DERV(Y(7),Y(8),Y(9),Y(10),DLDV,D2LDV,DY(7),DY(8),DY(9),
00125 31* 1 DY(10),0.,0.,DNCDV,DA1DV,DA2DV,DA3DV,DA4DV,DA5DV,
00125 32* 2 DA6DV,DEB0V,DEV,DEV,DMDV,D2MNCV)
00126 33* CALL DERV(Y(11),Y(12),Y(13),Y(14),DLNCR,D2LNCB,DY(11),DY(12),
00126 34* 1 DY(13),DY(14),1.,0.,DNCDN,DA1DN,DA2DN,DA3DN,DA4DN,
00126 35* 2 DA5DN,DA6DN,DEB0N,DEU,DEU,DMDN,D2MNCN)
00127 36* CALL MIXDER(Y(4),Y(8),Y(7),Y(16),Y(15),DNCDU,DNCDV,DY(4),DA1DU,
00127 37* 1 DA1DV,DA2DU,DA2DN,DA3DU,DA3DV,DA4DV,DA5DU,DA5DV,
00127 38* 2 DA6DU,DA6DV,DLDU,DLDV,D2LDU,D2LDV,DEB0U,DEB0V,DEU,DEV,
00127 39* 3 DEU,DEV,DMDU,DMDV,D2MNCU,D2MNCV,1.,0.,DY(16),
00127 40* 4 DY(15))
00130 41* CALL MIXDER(Y(4),Y(12),Y(11),Y(18),Y(17),DNCDU,DNCDN,DY(4),
00130 42* 1 DA1DU,DA1DN,DA2DU,DA2DN,DA3DU,DA3DN,DA4DU,DA5DU,
00130 43* 2 DA5DN,DA6DU,DA6DN,DLDU,DLNCR,D2LVNB,DEB0N,DEB0V,
00130 44* 3 DEU,DEV,DMDU,DMDN,D2MNCU,D2MNCN,1.,1.,
00130 45* 4 DY(18),DY(17))
00131 46* CALL MIXDER(Y(8),Y(12),Y(11),Y(20),Y(19),DNCDV,DNCDN,DY(8),
00131 47* 1 DA1DV,DA1DN,DA2DV,DA2DN,DA3DV,DA3DN,DA4DV,DA5DV,
00131 48* 2 DA5DN,DA6DV,DA6DN,DLDV,DLNCR,D2LVNB,DEB0N,DEB0V,
00131 49* 3 DEV,DEU,DEV,DEU,DMDV,DMDN,D2MNCV,D2MNCN,0.,1.,
00131 50* 4 DY(20),DY(19))
00132 51* RETURN
00133 52* END

```

END OF COMPILATION: NO DIAGNOSTICS.

02 2FOR,IS SCNTL  
FOR 59A-07/27/72-17:43:50 (,0)

SUBROUTINE SCNTL ENTRY POINT 000147

STORAGE USED: CODE(1) 000174; DATA(0) 000014; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 MCT 000035  
0004 VAB 000043  
0005 BLKI 000024

EXTERNAL REFERENCES (BLOCK, NAME)

0006 NWJ03  
0007 NIQ23  
0010 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000002	2000	=	0001	000054	5L	0001	000061	50L	0001	000135	60L	0003	R	000000	DLMT			
0000	R	000000	DXST	0003	R	000003	DXWRT	0004	R	000002	ET	0003	I	000001	ICNT	0000	000006	INJPS	
0000	I	000001	LCNT	0003	I	000004	LSTEP	0005	R	000000	VAL	0003	R	000002	XWRT	0004	R	000000	Z1
0004	R	000001	Z2																

```

00101 1* SUBROUTINE SCNTL(Y,DY,DX,X,NTRY,IFVD)
00101 2* C
00101 3* C INTEGRATION CONTROL PROGRAM
00101 4* C
00103 5* DIMENSION Y(20),DY(20)
00104 6* COMMON /MCT/ DLMT,ICNT,XWRT,DXWRT,LSTEP
00104 7* 1 /VAB/ Z1,Z2,ET
00104 8* 2 /BLKI/VAL(20)
00105 9* ICNT = ICNT+1
00106 10* IF(DX.GE.DXWRT.AND.ICNT.GT.1) LSTEP = 1
00110 11* IF(ABS(X-XWRT).LT.DLMT) GO TO 50
00112 12* IF(XWRT.GT.X) GO TO 5
00112 13* C
00114 14* DXSTR = DX
00115 15* DX = DX+XWRT-X
00116 16* LCNT = 1
00117 17* NTRY = 3
00120 18* RETURN
00120 19* C
00121 20* 5 NTRY = 1
00122 21* RETURN
00122 22* C
00123 23* 50 WRITE(6,2000) Y(1),Y(2),Z1,Z2,ET,VAL(3),VAL(4),X
00135 24* 2000 FORMAT (8F15.7)

```



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02 00136 25*      IF(LCNT.EQ.1) DX = DXSTR
    00140 26*      LCNT = 0
    00141 27*      IF(ABS(1.0-YWRT).LE.DLMT) GO TO 60
    00143 28*      XWRT = XWRT+DXWRT
    00144 29*      NTRY = 1
    00145 30*      IF(ISTEP.EQ.0) RETURN
    00147 31*      DX = DXWRT
    00150 32*      IFV = 1
    00151 33*      RETURN
    00151 34*      C
    00152 35*      GO NTRY = 2
    00153 36*      RETURN
    00154 37*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

2FOR,IS BNDCND  
FOR 59A-07/27/72-17:43:54 (,0)

SUBROUTINE BNDCND ENTRY POINT 000350

STORAGE USED: CODE(1) 0004111 DATA(0) 000065; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLK3 00000:

EXTERNAL REFERENCES (BLOCK, NAME)

0004 FMINV  
0005 AERR35

STORAGE ASSIGNMENT BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000115	600L	0001	000130	601L	0001	000134	602L	0001	000137	603L	0001	000167	610L
0001	000202	611L	0001	000206	612L	0001	000211	613L	0001	000241	620L	0001	000254	621L
0001	000260	622L	0001	000263	623L	0001	000313	630L	0001	000040	700L	0000	R	000000
0003	000000	DNCB0	0003	000001	DNCB0V	0000	I	000017	I	0000	000044	INJP5	0000	I
0000	I	000021	K	0000	R	000014	XN	0000	R	000022	Z1	0000	R	000023
														0000
														0000
														0000
														0000

```

00101 1* SUBROUTINE BNDCND(Y,U,V,FNCBR,X")
00101 2* C
00101 3* C BNDCND INSURES U,V, AND NCBAR ARE WITHIN THEIR BOUNDARY CONDITIONS
00101 4* C
00103 5* DIMENSION AN(3,4),XN(3),XM(3),Y(20)
00104 6* COMMON /BLKR/DNCB0U,DNCB0V
00105 7* I = 0
00106 8* J = 0
00107 9* K = 0
00110 10* Z1 = 0
00111 11* Z2 = 0
00112 12* Z3 = FNCBR
00113 13* J = Z1+X"(1)
00114 14* J = Z2+X"(2)
00115 15* FNCBR = Z3+X"(3)
00116 16* AN(1,1) = Y(5)
00117 17* AN(1,2) = Y(15)
00120 18* AN(1,3) = Y(17)
00121 19* AN(2,1) = Y(15)
00122 20* AN(2,2) = Y(9)
00123 21* AN(2,3) = Y(19)
00124 22* AN(3,1) = Y(17)
00125 23* AN(3,2) = Y(19)
00126 24* AN(3,3) = Y(13)
00127 25* 700 IF (U.LT.4.00E-06.OR.U.GT.250.1) GO TO 600
00131 26* IF (V.LT.4.00E-05.OR.V.GT.50.01) GO TO 610

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00133 27* IF (FNCBR.LT.0.00009.OR.FNCBR.GT.4.01) GO TO 620
00135 28* RETURN
00136 29* 600 IF (U.LE.5.0E-06) GO TO 601
00140 30* IF (U.GE.250.) GO TO 602
00142 31* GO TO 610
00143 32* 601 XN(1) = 5.0E-06-71
00144 33* GO TO 603
00145 34* 600 XN(1) = 250.-71
00146 35* 600 AN(1*1) = 1.
00147 36* AN(1*2) = 0.
00150 37* AN(1*3) = 0.
00151 38* I = 1
00152 39* IF (I.EQ.0) YN(1) = -Y(3)
00154 40* IF (J.EQ.0) YN(2) = -Y(7)
00156 41* IF (K.EQ.0) XN(3) = -Y(11)
00160 42* CALL FMINV(AN,XN,3,4)
00161 43* 61 IF (V.LE.5.0E-05) GO TO 611
00163 44* IF (V.GE.50.) GO TO 612
00165 45* GO TO 620
00166 46* 61 XN(2) = 5.0E-05-72
00167 47* GO TO 613
00170 48* 61 XN(2) = 50.-72
00171 49* 61 AN(2*1) = 0.
00172 50* AN(2*2) = 1.
00173 51* AN(2*3) = 0.
00174 52* J = 1
00175 53* IF (I.EQ.0) XN(1) = -Y(3)
00177 54* IF (J.EQ.0) YN(2) = -Y(7)
00201 55* IF (K.EQ.0) YN(3) = -Y(11)
00203 56* CALL FMINV(AN,XN,3,4)
00204 57* 62 IF (FNCBR.LE.1.0E-04) GO TO 62.
00206 58* IF (FNCBR.GE.4.00) GO TO 622
00210 59* GO TO 630
00211 60* 62 XN(3) = 1.0E-04-73
00212 61* GO TO 623
00213 62* 62 XN(3) = 4.00-73
00214 63* 62 AN(3*1) = 0.
00215 64* AN(3*2) = 0.
00216 65* AN(3*3) = 1.
00217 66* I = 1
00220 67* IF (I.EQ.0) XN(1) = -Y(3)
00222 68* IF (J.EQ.0) YN(2) = -Y(7)
00224 69* IF (K.EQ.0) XN(3) = -Y(11)
00226 70* CALL FMINV(AN,XN,3,4)
00227 71* 63 XN(1) = XN(1)
00230 72* XN(2) = XN(2)
00231 73* XN(3) = XN(3)
00232 74* U = 71+XN(1)
00233 75* V = 72+XN(2)
00234 76* FNCBR = 73+XN(3)
00235 77* 66 GO TO 700
00236 78* END

```

END OF COMPILATION: NO DIAGNOSTICS.

FOR, IS RESND  
FOR 59A-07/27/72-17:44:04 (,0)

SUBROUTINE RESND ENTRY POINT 000203

STORAGE USED: CODE( ) 000232; DATA(0) 000053; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 SGR  
0004 NWJJS  
0005 N1025  
0006 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000026	1L	0001	000126	2L	0000	000007	2000F	0001	000144	3L	0001	000161	4L
0000	R	000003	A	0000	R	000004	B	0000	R	000005	C	0000	R	000006
0000	R	000000	U2	0000	R	000001	V2	0000	R	000002	Z	0000	R	000037
														INJPS

```

00101      1*      SUBROUTINE RESND(U,V,XM,ASTAR)
00101      2*      C
00101      3*      RESND CHECKS THE BOUNDARY OF LAMBDA AND KEEPS ASTAR
00101      4*      C      GREATER THAN U*V
00101      5*      C
00103      6*      DIMENSION XM(3)
00104      7*      U2 = U-X1(1)
00105      8*      V2 = V-X1(2)
00106      9*      Z = ASTAR-U*V
00107      10*     IF (Z.LT.2.*U*V*1.0E-06) GO TO 1
00111      11*     RETURN
00111      12*      C
00112      13*     A = X1(1)*XM(2)
00113      14*     B = U2*XM(2)+V2*XM(1)
00114      15*     C = U2*V2-0.99*ASTAR
00115      16*     IF (ABS(C).LE.1.0E-05.AND.XM(2).LT.0.) GO TO 2
00117      17*     IF (ABS(C).LE.1.0E-05.AND.XM(1).LT.0.) GO TO 3
00117      18*      C
00121      19*     EPS = (-B+SGRT(B**2.-4.*A*C))/2./A
00122      20*     X1(1) = EPS*XM(1)
00123      21*     X1(2) = EPS*XM(2)
00124      22*     U = U2+XM(1)
00125      23*     V = V2+XM(2)
00126      24*     GO TO 4
00127      25*     EPS = (0.99*ASTAR-U2*V)/(V*XM(1))
00130      26*     X1(1) = EPS*XM(1)
00131      27*     U = U2+XM(1)
00132      28*     GO TO 4
00133      29*     EPS = (0.99*ASTAR-U*V2)/(U*XM(2))
00134      30*     XM(2) = EPS*XM(2)

```

```

00135 31*      V      = V2+X*(2)
00135 32*      C
00136 33*      4      WRITE(6,2000) X*(1),X*(2),EPS
00143 34*      2000  FORMAT (//5X'EPSILON CALCULATED' DELTAU ='E14.6/
00143 35*      1      31X' DELTAV ='E14.6/31X'EPSILON = 'E14.6)
00143 36*      C
00144 37*      RETURN
00145 38*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

02 2FOR.15 REST  
FOR 59A-07/27/72-17:44:10 (,0)

SUBROUTINE REST ENTRY POINT 000066

STORAGE USED: CODE(1) 000120; DATA(0) 000023; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 A1 0000 000003 INJP5

00101	1*		SUBROUTINE REST(U,V,ASTAR,RLAM,DLDU,DLDV,D2LDUU,D2LDUV,D2LDVV,
00101	2*		DLDNCA,D2LNCB,D2LUNB,D2LVNB)
00101	3*	C	
00101	4*	C	CONSTANT AREA RESTRAINT
00101	5*	C	
00103	6*		A1 = ASTAR-U*V
00104	7*		RLAM = 2*U*V/A1
00105	8*		DLDU = 2.*ASTAR*V/A1**2.
00106	9*		DLDV = 2.*ASTAR*U/A1**2.
00107	10*		D2LDUU = 4.*ASTAR*V**2./A1**3.
00110	11*		D2LDUV = 2.*ASTAR*(ASTAR+U*V)/A1**3.
00111	12*		D2LDVV = 4.*ASTAR*U**2./A1**3.
00112	13*		DLDNCA = 0.
00113	14*		D2LNCB = 0.
00114	15*		D2LUNB = 0.
00115	16*		D2LVNB = 0.
00116	17*		RETURN
00117	18*		END

END OF COMPILATION: NO DIAGNOSTICS.

02

DFOR, IS NDERV  
FOR 59A-07/27/72-17:44:13 (,0)

SUBROUTINE NDERV ENTRY POINT 000056

STORAGE USED: CODE(1) 000070; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 SORT  
0004 ASIN  
0005 NERR3%

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 C4      0000 R 000001 C5      0000 R 000002 C6      0000 000006 INJP%

00101	1*		SUBROUTINE NDERV(RLAM, FN, DFNX, D2NDLL)
00101	2*	C	
00101	3*	C	N AND ITS DERIVATIVES
00101	4*	C	
00103	5*		C4 = RLAM+1.0
00104	6*		C5 = RLAM+2.0
00105	7*		C6 = SORT(RLAM+C5)
00106	8*		FN = (1.5707963+C5-C6-ASIN(1.0/C4))/RLAM
00107	9*		DFNX = -(FN+C6/C4-1.0)/RLAM
00110	10*		D2NDLL = -(2.0*DFNX+1.0/(C4*C4*C6))/RLAM
00111	11*		RETURN
00112	12*		END

END OF COMPILATION: NO DIAGNOSTICS.

158671

FOR IS MDER  
FOR S9A-07/27/72-17144:16 (1.0)

SUBROUTINE MDER ENTRY POINT 000251

STORAGE USED: CODE(1) 000304; DATA(0) 000966; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (LOCK, NAME)

0003 SGR  
0004 ASIN  
0005 NERR35

STORAGE ASSIGNMENT (LOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000011 CFM	0000 R 000030 C1	0000 R 000017 C10	0000 R 000020 C11	0000 R 000022 C12
0000 R 000001 C2	0000 R 000003 C3	0000 R 000002 C4	0000 R 000004 C5	0000 R 000005 C6
0000 R 000006 C7	0000 R 000007 C8	0000 R 000010 C9	0000 R 000015 DCFDNC	0000 R 000013 DCFNCL
0000 R 000014 DFFSOL	0000 R 000016 D2CFLL	0000 R 000025 D2CFNC	0000 R 000026 D2CLNC	0000 R 000023 D2C2LL
0000 R 000021 D2C3LL	0000 R 000024 D2FSLL	0000 R 000027 D3C2LN	0000 R 000030 D3C2NL	0000 R 000012 FFS
0000 000040 1NJP5				

00101 1\* SUBROUTINE MDER(RLAM,FNC,FM,OFMX,DFM,D2FMLL,D2MDNC,D2MLNC,D3M2LN,

00101 2\* 1 3M2NL)

00101 3\* C

00101 4\* C AND ITS DERIVATIVES

00101 5\* C

00103 6\* C1 = 0.1460\*FNC-0.02866

00104 7\* C2 = SGR(RLAM\*(2.+RLAM))

00105 8\* C4 = 1.0/(RLAM+1.0)

00106 9\* C3 = ASIN(C4)

00107 10\* C5 = 1/(C4+C2)

00110 11\* C6 = (C4\*\*2)/SGRT(1-C4\*\*2)

00111 12\* C7 = 1/(RLAM\*\*2)

00112 13\* C8 = FNC/RLAM

00113 14\* C9 = 2.\*C7+C8

00114 15\* CFN = 1.0-C1\*C8

00115 16\* FFS = (C2+C3-1.57079635)/RLAM

00116 17\* FM = CFN+FFS

00117 18\* DCFDNC = C1\*C7+FNC

00120 19\* DFFSOL = (C5-C6-FFS)/RLAM

00121 20\* FMX = CFN+DFFSOL+FFS+DCFNDL

00122 21\* DCFDNC = -(0.1460\*FNC+C1)/RLAM

00123 22\* FM = FFS+DCFNDL

00124 23\* D2CFLL = -C1\*C9

00125 24\* C10 = 1-C4\*\*2

00126 25\* C11 = +1.+0.5/C10+C4\*\*2

00127 26\* D2C3LL = 2\*(C4\*\*3)\*C11/SGRT(C10)

00130 27\* C12 = 1/C2-C2\*(C4\*\*2)

00131 28\* D2C2LL = -C12/(C4+C2)\*\*2



00132	29*	D2FSLL	= (D2C2LL+D2C3LL-2.*DFFSOL)/RLAM
00133	30*	D2FVLL	= CFN*D2FSLL+2.*DFFSOL*D2CFNDL+FFS*D2CFLL
00134	31*	D2CFNC	= -0.2920/RLAM
00135	32*	D24DNC	= FFS*D2CFNC
00136	33*	D2CLNC	= (0.146*FNC+C1)*C7
00137	34*	D2MLNC	= DCFDNC*DFFSOL+D2CLNC*FFS
00140	35*	D3C2LN	= -2.*D2CLNC/RLAM
00141	36*	D342LN	= DCFDNC*D2FSLL+2.*DFFSOL*D2CLNC+FFS*D3C2LN
00142	37*	D3C2NL	= 0.2920*C7
00143	38*	D342NL	= FFS*D3C2NL+DFFSOL*D2CFNC
00144	39*	RETURN	
00145	40*	END	

END OF COMPILATION: NO DIAGNOSTICS.

02 2FOR, IS NCDERV  
FOR S9A-07/27/72-17:44:19 (,0)

SUBROUTINE NCDERV ENTRY POINT 000031

STORAGE USED: CODE(1) 000044; DATA(0) 000013; BLANK COMMON(2) 000000

EXTERNAL REFERENCE (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 C00004 INJ 5 0000 R 000000 T2 0000 R 000001 T3

00101	1*		SUBROUTINE NCDERV(FNCBR, THETA3, FNC, DNCOT1, D2CD2T, D2CTNB)
00101	2*	C	
00101	3*	C	MC AND ITS DERIVATIVES
00101	4*	C	
00103	5*		T2 = THETA3**2
00104	6*		T3 = T2*THETA3
00105	7*		FNC = FNCBR*T3
00106	8*		DNCOT1 = 3.*FNCBR*T2
00107	9*		D2CD2T = 6.*FNCBR*THETA3
00110	10*		D2CTNB = 3.*T2
00111	11*		RETURN
00112	12*		END

END OF COMPILATION: NO DIAGNOSTICS.

158675

2FOR, IS MIXDER  
FOR 59A-07/27/72-17:44:49 (,0)

SUBROUTINE MIXDER ENTRY POINT 000347

STORAGE USED: CODE(1) 000454; DATA(1) 000104; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLKI 00002  
0004 BLKD 000012

EXTERNAL REFERENCES (BLOCK, NAME)

0005 DDETA  
0006 DETA  
0007 DODETA  
0010 ETA  
0011 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0004 R 000003 A1	0004 R 000004 A2	0004 R 000005 A3	0004 R 000006 A4	0004 R 000007 A5
0004 R 000010 Y6	0007 R 000000 DODETA	0005 R 000000 DDETA	0006 R 000000 DETA	0003 000023 DE101
0003 000021 DE4DT	0003 R 000013 DFM	0005 R 000012 DFMX	0003 R 000010 DFNX	0003 000007 DFCOT1
0000 R 000006 DNDXY	0000 R 000003 D2A1XY	0000 R 000020 D2A2XY	0000 R 000021 D2A3XY	0000 R 000022 D2A4XY
0000 R 000015 D2A5XY	0000 R 000004 D2A6XY	0000 R 000017 D2E8XY	0000 R 000012 D2EPXY	0000 R 000011 D2EXY
0003 R 000016 D2FML	0000 R 000013 D2M0LY	0000 R 000015 D2M0NC	0000 R 000014 D2M2XY	0003 R 000014 D2MLNC
0003 R 000011 D2NDL	0000 R 000016 D2NDXY	0000 R 000007 D3MNL	0000 R 000010 D3MNX	0004 R 000001 D3M2LN
0004 R 000002 D3M2NY	0000 R 000005 D3M2NY	0000 R 000011 ER	0010 R 000000 ETA	0003 000022 F3
0003 R 000002 FM	0003 000003 FN	0003 R 000005 FNC	0003 R 000004 FNCBR	0000 000026 Y4DP
0003 000000 THETA	0003 000001 THETAS	0004 R 000000 U	0003 000006 V	0000 R 000000 Y
0003 000017 Y1	0003 R 000020 Y2	0000 R 000002 Z		

00101 1\* SUBROUTINE MIXDER(DT3DY,DT3DY,DTFDY,D2T8XY,D2TFXY,DNCDY,DNCDY,  
00101 2\* D2T8YE,DA1DX,DA1DY,DA2DX,DA2DY,DA3DX,DA3DY,  
00101 3\* DA4DY,DA5DX,DA5DY,DA6DX,DA6DY,DLOX,DLOY,  
00101 4\* D2LOXY,DE3DY,DE3DX,DEY,DEPX,DEPY,DMDX,  
00101 5\* DMXY,D2MNCX,D2MNCY,R,C,D3T8YE,DTFDXY)  
00101 6\* C  
00101 7\* C FOR MIXED DERIVATIVES OF U AND V SET B=1. AND C= 0.  
00101 8\* C FOR MIXED DERIVATIVES OF U AND FNCBR SET B=1. AND C= 1.  
00101 9\* C FOR MIXED DERIVATIVES OF V AND FNCBR SET B=0. AND C= 1.  
00101 10\* C  
00103 11\* DIMENSION Y(2)  
00104 12\* COMMON /BLKI/THETAS,THETAS,FM,FN,FNCBR,FNC,V,DNCDT1,DFNX,D2NDLL,  
00104 13\* DFMX,DFM,D2MLNC,D2M0NC,D2FMLL,Y1,Y2,DE4DT8,E3,DE1DL/  
00104 14\* BLKD/U,D3M2LN,D3M2NL,A1,A2,A3,A4,A5,A6,EB  
00105 15\* Y(2) = Y2  
00106 16\* Z = FNC

00107	17*	D1F0XY = U*(D2T8XY-D2TFXY)+9*(D1B0Y-D1F0Y)
00110	18*	D2A1XY = 2*(D2A1DX+D2A1DY/A1-A1**2*(D2TFXY-D2TRXY)
00111	19*	D2A6XY = 6*(FNCBR*D2T8DX+D2T8DY+6*Y(2)*FNCBR*D2T8XY+C*6*Y(2)*D2B0X
00112	20*	D3M2XY = DLDY+D3M2NL
00113	21*	DNC0XY = 6*(FNCBR*Y(2)*D2B0X+D2T8DY+3*(FNCBR*(Y(2)**2)*D2T8XY+
00113	22*	C*3*(Y(2)**2)*D2T8DY
00114	23*	D3M4LY = DLDY+D3M4LN+DNC0Y+D3M4NL
00115	24*	D3M4XY = D3M4LY+D2LX+D2MLNC+D2LDXY+D3M4NY+DNC0X+D2MDNC+DNC0XY
00116	25*	D2EXY = DDETA(Z)*DNC0X+DNC0Y+DETA(Z)*DNC0XY
00117	26*	D2EPXY = DDETA(Z)*DNC0X+DNC0Y+DDETA(Z)*DNC0XY
00120	27*	D2M0LY = DLDY+D2F0LL+DNC0Y+D2M0LNC
00121	28*	D2M0XY = DLDX+D2M0LY+D2FX+D2LX+D2M0NY+DNC0X+D2M0LNC+DNC0XY
00122	29*	D2A5XY = FX+D2EPXY+D2PX+D2MY+D2PY+D2MX+DETA(Z)*D2M0XY+DEY+D2MNCX+
00122	30*	ETA(Z)*D3M4XY+DEX+D2M0NY+D2FX+D2EXY
00123	31*	D2M0XY = D2FX+D2L0XY+D2LX+D2L0Y+D2M0LL
00124	32*	D2E0XY = FX+D2EXY+D2DY+DEX+ETA(Z)*D2M0XY+DEY+D2M0X+D2M0XY
00125	33*	D2A2XY = -(D2B0Y+D2D0X+D2B0Y+D2D0Y+A2+D2B0XY-(A6+D2A5XY+
00125	34*	D2A6XY+D2A5XY+D2A5DY+D2A5DX+A5+D2A6XY)/E8
00126	35*	D2A3XY = D2A3DX+D2A3DY/A3+D2T8XY+D2A3DX/D2T8DX+
00126	36*	A3*D2T8DX*(-3*D2T8DY/Y(2)**2-D2A3DY)
00127	37*	D2A4XY = D2A1XY+D2A2XY+D2A3XY
00130	38*	D3TYE = U+D2A4XY/A4**2-D2A4DY*(2*A4*D2T8XE+B)/A4**2
00131	39*	RETURN
00132	40*	END

END OF COMPILATION:

NO DIAGNOSTICS.

158677

02 2FOR,IS DERV  
FOR S9A-07/27/72-17:44:54 (,0)

SUBROUTINE DERV ENTRY POINT 000604

STORAGE USED: CODE(1) 000721; DATA(0) 000130; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 ELKI 000024  
0004 BLKD 000012

EXTERNAL REFERENCES (BLOCK, NAME)

0005 ETA  
0006 DETA  
0007 DDETA  
0010 DDDETA  
0011 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0004 R 000003 A1	0004 R 000004 A2	0004 R 000005 A3	0004 R 000006 A4	0004 R 000007 A5
0004 R 000010 A6	0000 R 000051 A7	0000 R 000063 DA7DU	0010 R 000000 DDETA	0007 R 000000 DDETA
0006 R 000000 DETA	0003 R 000023 DE1DL	0003 R 000021 DE4DTA	0003 R 000013 DFM	0003 R 000012 DFMX
0003 R 000010 DFNX	0003 R 000007 DDCDT1	0000 R 000052 DNDU	0000 R 000024 DY	0000 R 000053 D2A11U
0000 R 000066 D2A2UU	0000 R 000067 D2A3UU	0000 R 000070 D2A4UU	0000 R 000061 D2A5UU	0000 R 000062 D2A6UU
0000 R 000065 D2E8UU	0000 R 000055 D2EPUU	0000 R 000060 D2EUU	0003 R 000016 D2FMLL	0003 R 000015 D2MDNC
0003 R 000014 D2MLNC	0000 R 000056 D2MUU	0000 R 000054 D2NCUU	0003 R 000011 D2NDLL	0000 R 000064 D2NUU
0000 R 000057 D3'NUU	0004 R 000001 D3M2LN	0004 R 000002 D3M2NL	0004 R 000011 ER	0005 R 000000 ETA
0003 R 000022 E3	0003 R 000002 FM	0003 R 000003 FN	0003 R 000005 FNC	0003 R 000004 FNCBR
0000 R 000077 INJPS	0003 R 000000 THETAB	0003 R 000001 THETAS	0004 R 000000 U	0003 R 000006 V
0000 R 000000 Y	0003 R 000017 Y1	0003 R 000020 Y2	0000 R 000050 Z	

00101 1\* SUBROUTINE DERV(Y3,Y4,Y5,Y6,DLDU,D2LDU,DY3,DY4,DY5,DY6,B,C,  
00101 2\* 1 DNCDD,DA1DU,DA2DU,DA3DU,DA4DU,DA5DU,DA6DU,  
00101 3\* 2 DEPDU,DEU,DEPU,DYDU,D2MNCU)  
00101 4\* C  
00101 5\* C DERIVATIVES  
00101 6\* C  
00101 7\* C FOR FIRST AND SECOND DERIVATIVES OF U SET C=1 AND B=0  
00101 8\* C FOR FIRST AND SECOND DERIVATIVES OF V SET C=0 AND B=0  
00101 9\* C FOR FIRST AND SECOND DERIVATIVES OF FNCBR SET C=0 AND B=1  
00101 10\* C  
00103 11\* DIMENSION Y(20),DY(20)  
00104 12\* COMMON /BLKI/THETAB,THETAS,FM,FN,FNCBR,FNC,V,DNCDT1,DFNX,D2NDLL,  
00104 13\* 1 DFMX,DFM,D2MLNC,D2MDNC,D2FMLL,Y1,Y2,DE4DTA,E3,DE1DL/  
00104 14\* 2 BLKD/U,D3M2LN,D3M2NL,A1,A2,A3,A4,A5,A6,EB  
00105 15\* Y(1) = Y1  
00106 16\* Y(2) = Y2

00107	17*	Y(3)	= Y3
00110	18*	Y(4)	= Y4
00111	19*	Y(5)	= Y5
00112	20*	Y(6)	= Y6
00113	21*	Z	= FNC
00114	22*	F3	= FV*ETA(Z)+FN
00115	23*	A1	= 1./(Y(1)-Y(2))
00116	24*	A5	= FV*DETA(Z)+ETA(Z)*DF4
00117	25*	A6	= -3*Y(2)**2*FICRR
00120	26*	A2	= A6*A5/EB
00121	27*	A3	= 4*Y(2)**3/(Y(2)**4-THETAS**4)
00122	28*	A4	= A1+A2+A3
00123	29*	DA1DU	= -A1**2*(Y(3)-Y(4))
00124	30*	DICDU	= A6*Y(4)+B*Y(2)**3
00125	31*	DEPU	= DDETA(Z)*DNCDU
00126	32*	DJDU	= DFVX*DLDU+DFM*DNCDU
00127	33*	D2MNCU	= D2MLNC*DLDU+D2MNC*DNCDU
00130	34*	DEU	= DETA(Z)*DNCDU
00131	35*	DA5DU	= FV*DEPU+DETA(Z)*DMDU+ETA(Z)*D2MNCU+DFM*DEU
00132	36*	DA6DU	= 6*Y(2)*FICRR+Y(4)+B*Y(2)**2
00133	37*	A7	= A6*DA5DU+A5*DA6DU
00134	38*	DLDU	= DFVX*DLDU
00135	39*	DEBDU	= FV*DEU+ETA(Z)*DMDU+DNCDU
00136	40*	DA2DU	= (A7-A2*DEBDU)/EB
00137	41*	DA3DU	= A3*Y(4)*(3/Y(2)-A3)
00140	42*	DA4DU	= DA1DU+DA2DU+DA3DU
00141	43*	Y(4)	= -C/A4+U/A4**2+DA4DU
00142	44*	DA1UU	= -A1**2*(Y(5)-Y(6))-2*A1*DA1DU*(Y(3)-Y(4))
00143	45*	D2NCUU	= A6*Y(5)+DA6DU*Y(4)+B*3*Y(4)*Y(2)**2
00144	46*	DEPUU	= DDETA(Z)*D2NCUU+DNCDU**2+DDDETA(Z)
00145	47*	D2MUU	= DFVX*D2L2UU+DLDU*(D2F4LL*DLDU+D2MLNC*DNCDU)+DFM*D2NCUU
00145	48*	1	+DNCDU*(D2MLNC*DLDU+D2MNC*DNCDU)
00146	49*	D3MNUU	= D2MLNC*D2L3UU+DLDU*(D3Y2LN*DLDU+D3M2NL*DNCDU)
00146	50*	1	+D2M2NL*D2NCUU+DNCDU*D2M2NL*DLDU
00147	51*	DEUU	= DETA(Z)*D2NCUU+DETA(Z)*DNCDU**2
00150	52*	DA5UU	= FV*D2EPUU+2*DEPU*DMDU+DETA(Z)*D2MUU+ETA(Z)*D3MNUU
00150	53*	1	+2*DEU*D2MNCU+DFM*D2EPU
00151	54*	DA6UU	= 6*Y(2)*FICRR+Y(6)+6*FICRR*Y(4)**2+B*12*Y(2)*Y(4)
00152	55*	DA7DU	= A6*DA5UU+DA5DU*DA6DU+5*DA6UU+DA6DU*DA5DU
00153	56*	D2NUU	= DFVX*D2L2UU+DLDU*(D2F4LL*DLDU+D2MLNC*DNCDU)+DFM*D2NCUU
00154	57*	DEBUU	= FV*D2EPUU+2*DEPU*DMDU+DETA(Z)*D2MUU+D2NUU
00155	58*	DA2UU	= (DA7DU-A2*D2EPUU-2*DEBUU*DA2DU)/EB
00156	59*	DA3UU	= A3*((3/Y(2)-A3)*Y(6)+Y(4)*(-3/Y(2)**2*Y(4)-DA3DU)
00156	60*	1	+(1/13*DA3DU)**2)
00157	61*	DA4UU	= DA1UU+DA2UU+DA3UU
00160	62*	Y(6)	= -2/A4*Y(4)+DA4DU+U/A4**2+DA4UU
00161	63*	Y(3)	= U*(Y(4)-Y(3))+Y(2)-Y(3))*C
00162	64*	Y(5)	= U*(Y(6)-Y(5))+2*(Y(4)-Y(3))*C
00163	65*	Y3	= DY(3)
00164	66*	Y4	= DY(4)
00165	67*	Y5	= DY(5)
00166	68*	Y6	= DY(6)
00167	69*	RETURN	
00170	70*	END	

END OF COMPILATION: NO DIAGNOSTICS.

2 FOR.IS INITIAL  
FOR S9A-07/27/72-17:44:59 (.0)

SUBROUTINE INITIAL ENTRY POINT 000470

STORAGE USED: CODE(1) 000522; DATA(0) 000123; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLKI 00002

EXTERNAL REFERENCES (BLOCK, NAME)

0004 ETA  
0005 DETA  
0006 DDETA  
0007 MERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000037 A1	0000 R 000040 A2	0000 R 000041 A3	0000 R 000053 B1	0000 R 000054 B2
0006 R 000000 DDETA	0000 R 000051 DDETDU	0000 R 000000 DETA	0000 R 000046 DETAQU	0003 R 000023 DE1DL
0000 R 000032 DE1DU	0000 R 000027 DE1DTB	0000 R 000047 DE1DU	0000 R 000030 DE2DTB	0000 R 000034 DE2DU
0000 R 000031 DE3DT	0000 R 000035 DE3DU	0000 R 000021 DE4DTB	0000 R 000033 DE4DU	0000 R 000044 DECNQU
0003 R 000013 DF4	0000 R 000045 DF4DU	0000 R 000012 DFMX	0000 R 000043 DFNQU	0003 R 000010 DFNX
0003 R 000007 DNCDT	0000 R 000061 D1	0000 R 000062 D2	0000 R 000065 D2E1CR	0000 R 000060 D2E1LU
0000 R 000055 D2E1T	0000 R 000036 D2E2TU	0000 R 000042 D2E3TU	0000 R 000066 D2E4CR	0000 R 000071 D2E4LU
0003 R 000016 D2FML	0000 R 000056 D2MDLU	0003 R 000015 D2MDNC	0003 R 000014 D2MLNC	0000 R 000052 D2MNCU
0000 R 000067 D2NCH	0000 R 000050 D2NCTU	0003 R 000011 D2NDLL	0000 R 000057 D2NDLU	0000 R 000063 D3
0000 R 000064 D4	0004 R 000000 ETA	0000 R 000025 E1	0000 R 000026 E2	0003 R 000022 E3
0003 R 000002 FM	0003 R 000003 FN	0003 R 000005 FNC	0003 R 000004 FNCBR	0000 R 000100 INJPE
0000 R 000070 TERMN	0003 R 000000 THETAB	0003 R 000001 THETAS	0003 R 000006 V	0000 R 000000 Y
0003 R 000017 Y1	0003 R 000020 Y2	0000 R 000024 Z		

00101	1*		SUBROUTINE INITIAL (DLDU, D2LDUU, Y3, Y4, Y5, Y6, B, C, D2E4TU, D2E4LU)
00101	2*	C	
00101	3*	C	INITIAL CONDITIONS
00101	4*	C	
00101	5*	C	FOR FIRST AND SECOND DERIVATIVES OF U SET C=1 AND B=0
00101	6*	C	FOR FIRST AND SECOND DERIVATIVES OF V SET C=0 AND B=0
00101	7*	C	FOR FIRST AND SECOND DERIVATIVES OF FNCBR SET C=1 AND B=1
00101	8*	C	
00103	9*		DIMENSION Y(20)
00104	10*		COMMON /BLKI/THETAB, THETAS, FM, FN, FNCBR, FNC, V, DNCDT1, DFNX, D2NDLL,
00104	11*		DFMX, DFM, D2MLNC, D2MDNC, D2FMLL, Y1, Y2, DE4DTB, E3, DE1DL
00105	12*		Y(1) = Y1
00106	13*		Y(2) = Y2
00107	14*		Z = FNC
00110	15*		E1 = FM*ETA(Z)+FN
00111	16*		E2 = THETAB**4-THETAS**4

00112	17*	E3	= E2/(1-THETAB)
00113	18*	Y(3)	= 0.
00114	19*	DE1DTB	= DNCOT1*(DFM*ETA(Z)+FM*DETA(Z))
00115	20*	DE2DTB	= 4.*(THETAB**3)
00116	21*	DE3DTB	= E3*(DE2DTB+E3)/E2
00117	22*	DE4DTB	= E1*DE3DTB+E3*DE1DTB
00120	23*	DE1DL	= ETA(Z)*DFMX+DFNX
00121	24*	DE1DNC	= ETA(Z)*DFM*DETA(Z)+FM
00122	25*	DE4DU	= E1*DE1DL*DLBU+R*E3*DE1DNC*THETAB**3
00123	26*	Y(4)	= -(1-C1)/V**2*DE4DU/DE4DTB
00124	27*	Y(5)	= 0.
00125	28*	DE2DU	= DE2DTB*Y(4)
00126	29*	DE3DU	= E3*(E3*Y(4)+DE2DU)/E2
00127	30*	DE2TU	= 12.*Y(4)*(THETAB**2)
00130	31*	A1	= E3*(DE2TU+DE3DU)/E2
00131	32*	A2	= DE2TB+E3
00132	33*	A3	= -E3*DE2DU/(E2**2)+DE3DU/E2
00133	34*	DE3TU	= A1*A2**3
00134	35*	DFNDU	= DFM*DLBU
00135	36*	DFCNDU	= DNCOT1*Y(4)+B*Y(2)**3
00136	37*	DFNDU	= DFM*DLBU+DFM*DFCNDU
00137	38*	DE1DU	= DETA(Z)*DFCNDU
00140	39*	DE1DU	= FM*DE1DU+ETA(Z)*DFNDU+DFNDU
00141	40*	D2NCTU	= 6*E1CBB*Y(2)*Y(4)+B**3*Y(2)**2
00142	41*	D2ETDU	= DETA(Z)*DFCNDU
00143	42*	D2MNCU	= D2MLNC*DLBU+D2MNC*DFNDU
00144	43*	B1	= DFM*DE1DU+ETA(Z)*D2MNCU+FM*DDETDU+DETA(Z)*DFNDU
00145	44*	B2	= DE1DTB*D2NCTU/DNCOT1
00146	45*	D2E1TU	= DNCOT1*B1+B2
00147	46*	D2NDLU	= D2E1LL*DLBU+D2MLNC*DFNDU
00150	47*	D2NDLU	= D2NDLL*DLBU
00151	48*	D2E1LU	= ETA(Z)*D2NDLU+DE1DU*DFMX+D2NDLU
00152	49*	D2E4LU	= E3*D2E1LU+DE1DL*DE3DU
00153	50*	D1	= DETA(Z)*DFCNDU*DFM
00154	51*	D2	= ETA(Z)*D2MNCU
00155	52*	D3	= DETA(Z)*FM*DFCNDU
00156	53*	D4	= DETA(Z)*DFNDU
00157	54*	D2E1CB	= D1*D2+D3*D4
00160	55*	D2E4CB	= DE3DU*DE1DNC+E3*D2E1C
00161	56*	D2NCHB	= 3*Y(4)*Y(2)**2
00162	57*	TERMNB	= 9*(E3*DE1DNC*D2NCHB+Y(2)**3*D2E4CB)
00163	58*	D2E4UU	= E3*DE1DL*D2LDUU+DLBU*D2E4LU+TERMNB
00164	59*	D2E4TU	= E1*D2E3TU+DE3DTB*DE1DU+E3*D2E1TU+DE1DTB*DE3DU
00165	60*	Y(6)	= -(Y(4)*D2E4TU+D2E4UU)*2*(C-1)/V**3/DE4DTB
00166	61*	Y3	= Y(3)
00167	62*	Y4	= Y(4)
00170	63*	Y5	= Y(5)
00171	64*	Y6	= Y(6)
00172	65*	RETURN	
00173	66*	END	

END OF COMPILATION: NO DIAGNOSTICS.



02 2FOR,IS INTMIX  
FOR S9A-07/27/72-17:45:03 (,0)

SUBROUTINE INTMIX ENTRY POINT 000024

STORAGE USED: CODE(1) 000032; DATA(0) 000007; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLKI 000024

EXTERNAL REFERENCES (BLOCK, NAME)

0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0003 R 000023 DE1DL	0000 R 000000 DE4DL	0003 R 000021 DE4DTB	0000 R 000001 DE4DXY	0003 000013 DEF
0003 000012 DFMX	0003 000010 DFNX	0003 000007 DNCOT1	0003 000016 D2FMML	0003 000015 D2MDNC
0003 000014 D2MLNC	0003 000011 D2MDLL	0003 R 000022 E3	0003 000002 FM	0003 000003 FN
0003 000005 FNC	0003 000004 FNCBR	0000 000002 INJP5	0003 000000 THETAB	0003 000001 THETAS
0003 000006 V	0003 000017 Y1	0003 000020 Y2		

00101	1*	SUBROUTINE INTMIX(D2E4TY,DTBDX,DLOX,D2E4LY,D2LOXY,D2TBXY,D2TFXY)
00101	2*	C
00101	3*	C INITIAL MIXED DERIVATIVES
00101	4*	C
00103	5*	COMMON /BLKI/THETAB,THETAS,FM,FN,FNCBR,FNC,V,DNCOT1,DFNX,D2MDLL,
00103	6*	1 DFMX,DFM,D2MLNC,D2MDNC,D2FMML,Y1,Y2,DE4DTB,E3,DE1DL
00104	7*	D2TFXY = 0.
00105	8*	DE4DL = E3*DE1DL
00106	9*	DE4DXY = D2E4LY+DLOX+DE4DL*D2LOXY
00107	10*	D2TBXY = -(DE4DXY+DTBDX*D2E4TY)/DE4DTB
00110	11*	RETURN
00111	12*	END

END OF COMPILE: NO DIAGNOSTICS.

158881

2 FOR IS FMINV  
FOR S9A-07/27/72-17:45:07 (,0)

SUBROUTINE FMINV ENTRY POINT 000254

STORAGE USED: CODE(1) 000306; DATA(0) 001260; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000346	1056	0001	000151	111L	0001	000051	111G	0001	000071	117G	0001	000104	123G
0001	000125	133G	0001	000130	137G	0001	000172	150G	0001	000175	154G	0001	000227	164G
0001	000212	20L	0000	R	001214	AA	0000	R	001217	B	0000	I	001212	I
0000	I	001215	I1	0000	I	001220	I2	0000	I	001213	J	0000	I	001216
												0000	R	000000
														XMAT

00101	1*		SUBROUTINE FMINV (I,X,N,M)	1
00101	2*	C		
00101	3*	C	MATRIX INVERSION AND SOLUTION OF SIMULTANEOUS EQUATIONS	
00101	4*	C		
00103	5*		DIMENSION A(N,N),X(N),XMAT(25,26)	2
00104	6*		DO 1 I=1,N	3
00107	7*		XMAT(I,M) = X(I)	4
00110	8*		DO 1 J=1,N	5
00113	9*		XMAT(I,J) = A(I,J)	6
00116	10*		DO 20 I=1,N	7
00121	11*		AA = XMAT(I,I)	8
00122	12*		DO 5 J=1,M	9
00125	13*		XMAT(I,J) = XMAT(I,J)/AA	10
00127	14*		IF (I.EQ.1) GO TO 11	11
00131	15*		I1 = I-1	12
00132	16*		DO 10 K=1,I1	13
00135	17*		B = XMAT(K,I)	14
00136	18*		DO 10 J=1,M	15
00141	19*	1	XMAT(K,J) = XMAT(K,J) - XMAT(I,J) * B	16
00144	20*		IF (I.EQ.N) GO TO 20	17
00146	21*	1	I2 = I+1	18
00147	22*		DO 15 K=I2,N	1
00152	23*		B = XMAT(K,I)	20
00153	24*		DO 15 J=1,M	21
00156	25*	1	XMAT(K,J) = XMAT(K,J) - XMAT(I,J) * B	22
00161	26*	2	CONTINUE	23
00163	27*		DO 25 I=1,N	24
00166	28*	2	X(I) = XMAT(I,M)	25
00170	29*		RETURN	26
00171	30*		END	

END OF COMPILATION:

NO DIAGNOSTICS.

2 FOR IS ETA  
FOR S9A-07/27/72-17:45:11 (,0)

FUNCTION ETA ENTRY POINT 000036

STORAGE USED: CODE( ) 000044; DATA(0) 000022; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 EXP  
0005 NERR3s

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 1L 0000 R 000001 A 0000 R 000010 R 0000 R 000000 ETA 0000 000014 INJP  
0003 R 000000 POLY

00101	1*		FUNCTION ETA(X)
00101	2*	C	
00101	3*	C	FIN EFFECTIVENESS OF THE UNOSTRUCTED FIN
00101	4*	C	
00103	5*		DIMENSION A(7), B(2)
00104	6*		DATA A(1),A(2),A(3),A(4),A(5),A(6),A(7)/0.10E+01, -0.1163143E+01,
00104	7*		1 0.1470836E+01, -0.1267550E+01, 0.6325223E+00, -0.1627067E+00,
00104	8*		2 0.1654223E-01/ B(1),B(2)/0.6866095E+00, -0.2297718E+00/
00116	9*		IF(X-3I,2,5) GO TO 1
00120	10*		ETA = POLY(7,A,X)
00121	11*		RETURN
00122	12*		ETA = B(1)*EXP(B(2)*X)
00123	13*		RETURN
00124	14*		END

END OF COMPILATION: NO DIAGNOSTICS.

158683

2 FOR IS DETA  
FOR 59A-07/27/72-17:45:15 (.0)

FUNCTION DETA ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000021; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

n003 POLY  
n004 EXP  
n005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

n001 000016 IL 0000 R 000001 A 0000 R 000007 B 0000 R 000000 DETA 0000 000013 INJP\$  
n003 R 000000 POLY

00101	1*		FUNCTION DETA(X)
00101	2*	C	
00101	3*	C	DETA/DNC
00101	4*	C	
00103	5*		DIMENSION A(6), B(2)
00104	6*		DATA A(1),A(2),A(3),A(4),A(5),A(6)/-0.1163143E+01, 0.2957672E+01,
00104	7*	1	-0.3802650E+01, 0.2530089E+01, -0.8135335E+00, 0.9925338E-01/
00104	8*	2	B(1),B(2)/-0.1577635E+00, -0.2297718E+00/
00115	9*		IF(X.GT.2.5) GO TO 1
00117	10*		DETA = POLY(6,A,X)
00120	11*		RETURN
00121	12*	1	DETA = B(1)*EXP(B(2)*X)
00122	13*		RETURN
00123	14*		END

END OF COMPILATION: NO DIAGNOSTICS.

2  
 2 FOR IS DDETA  
 FOR 59A-07/27/72-17:45:23 (,0)

FUNCTION DDETA ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000020; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

n003 POLY  
 n004 EXP  
 n005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

n001 000016 1L 0000 R 000001 A 0000 R 000006 B 0000 R 000000 DDETA 0000 000012 INJP  
 n003 R 000000 POLY

```

00101 1* FUNCTION DDETA(X)
00101 2* C
00101 3* C DDETA/D2NC
00101 4* C
00103 5* DIMENSION A(5), B(2)
00104 6* DATA A(1),A(2),A(3),A(4),A(5)/0.2957672E+01, -0.7605300E+01,
00104 7* 1 0.7590257E+01, -0.3254134E+01, 0.4962669E+00/ B(1),B(2)/
00104 8* 2 0.3624960E-01, -0.2297718E+00/
00114 9* IF(X.GT.2.5) GO TO 1
00116 10* DDETA = POLY(5,A,X)
00117 11* RETURN
00120 12* 1 DDETA = B(1)*EXP(B(2)*X)
00121 13* RETURN
00122 14* END
  
```

END OF COMPILATION: NO DIAGNOSTICS.

2  
 3FOR,IS DDDETA  
 FOR 59A-07/27/72-17:45:35 (.0)

FUNCTION DDDETA ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
 0004 EXP  
 0005 MERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 1L 0000 R 000001 A 0000 R 000005 B 0000 R 000000 DDDETA 0000 000011 INJP  
 0003 R 000000 POLY

00101	1*		FUNCTION DDDETA(X)
00101	2*	C	
00101	3*	C	DDDETA/D3MC
00101	4*	C	
00103	5*		DIMENSION A(4), B(2)
00104	6*		DATA A(1),A(2),A(3),A(4)/-0.7405300E+01,0.1518053E+02,
00104	7*		1-0.9762402E+01,0.1095068E+01/
00104	8*		2 B(1),B(2)/-0.8329136E+02,-0.2497718E+00/
00113	9*		IF(X.GT.2.5) GO TO 1
00115	10*		DDDETA = POLY(4,A,X)
00116	11*		RETURN
00117	12*		DDDETA = B(1)*EXP(B(2)*X)
00120	13*		RETURN
00121	14*		END

END OF COMPILATION: NO DIAGNOSTICS.

2 FOR.I5 POLY  
FOR S9A-07/27/72-17:45:42 (,0)

FUNCTION POLY ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000012 1076 0000 000003 INJP5 0000 I 000002 K 0000 I 000001 L 0000 R 000000 POLY

00101	1*		FUNCTION POLY(N,A,X)
00101	2*	C	
00101	3*	C	POLYNOMIAL EVALUATION $Y = A(N) + X * A(N-1) + X ** 2 * A(N-2) + \dots$
00101	4*	C	
00103	5*		DIMENSION A(N)
00104	6*		POLY = 0.
00105	7*		L = N
00106	8*		DO 1 K=1,N
00111	9*		POLY = POLY * X + A(L)
00112	10*	1	L = L-1
00114	11*		RETURN
00115	12*		END

END OF COMPILATION: NO DIAGNOSTICS.

02

QXQT

MAP 0023-07/27-17:45

ADDRESS LIMITS 001000 020432 040000 047526  
 STARTING ADDRESS 017366

WORDS DECIMAL 7363 IBANK 3927 DBANK

SEGMENT	1A11	001000 020432	040000 047526
NSWTS/FOR	1	001000 001021	
NRCLKS/FOR	1	001022 001044	
NRWNS/FOR	1	001045 001124	2 040000 040011
NWFS/FOR	1	001125 001326	2 040012 040031
NFTCS/FOR	1	001327 001517	2 040032 040067
NBCLS/FOR	1	001520 001752	2 040070 040125
NFTVS/FOR	1	001753 001775	
NCWVS/FOR	1	001776 002222	2 040126 040215
NCLOS/FOR	1	002223 002371	2 040216 040247
NWCLKS/FOR	1	002372 002513	
NBSHS/FOR	1	002514 002550	
NUPDS/FOR	1	002551 002603	
NBFCS/FOR			2 040250 042451
NIMIS/FOR	1	002604 003014	2 042452 042463
NINPS/FOR	1	003015 003657	2 042464 042503
NOTIS/FOR	1	003658 004173	2 042504 042507
NOUTS/FOR	1	004174 005152	2 042510 042534
NFATS/FOR	1	005153 006041	2 042535 042611
NIDERS/FOR	1	006042 006214	2 042612 042716
NFCHK/FOR	1	006215 007076	2 042717 043055
			4 043056 043127
NTBS/FOR			2 043130 043166
ERUS/MISC			
NISUES/FOR	1	007077 007141	2 043167 043167
TIRS/TECH	1	007142 007626	0 043170 043220
			2 043221 043500
NEXPS/FOR	1	007627 007714	2 043501 043510
NIRS/FOR	1	007715 007776	2 043511 043544
NOSUES/FOR	1	007777 010043	
ASINCS/FOR	1	010044 010260	0 043645 043672
SORTS/FOR	1	010261 010321	2 043673 043704
EXPS/FOR	1	010322 010411	2 043705 043725
NERRS/FOR	1	010412 010736	2 043726 044071
BLKR (COMMON BLOCK)			044072 044103
VAB (COMMON BLOCK)			044104 044106
MCT (COMMON BLOCK)			044107 044113
BLKB (COMMON BLOCK)			044114 044115
BLKD (COMMON BLOCK)			044116 044127
BLKI (COMMON BLOCK)			044130 044153



✓ 2	BLANK\$COMMON (COMMON BLOCK)				
	POLY	1	010737 011002	0	044154 044170
✓	DDDETA	1	011003 011046	2	BLANK\$COMMON
✓	DDDETA	1	011047 011112	0	044171 044207
✓	DETA	1	011113 011156	2	BLANK\$COMMON
✓	ETA	1	011157 011222	0	044210 044227
✓	FMINV	1	011223 011530	2	BLANK\$COMMON
✓	INTMIX	1	011531 011562	0	044230 044250
✓	INITIAL	3	BLKI	2	BLANK\$COMMON
✓	DERV	1	011563 012304	0	044251 044272
✓	DERV	3	BLKI	2	BLANK\$COMMON
✓	DERV	3	BLKI	2	BLANK\$COMMON
✓	MIXDER	1	013226 013701	0	044273 045552
✓	MIXDER	3	BLKI	2	BLANK\$COMMON
✓	NDODRV	1	013702 013745	0	045553 045561
✓	MDER	1	013746 014251	2	BLANK\$COMMON
✓	NDODRV	1	014252 014341	0	045562 045704
✓	REST	1	014342 014461	2	BLANK\$COMMON
✓	RESAND	1	014462 014713	0	045705 046034
✓	BNDCND	1	014714 015324	2	BLANK\$COMMON
✓	SCNTL	3	BLK3	4	BLKD
✓	SDRV	1	015325 015520	0	046035 046140
✓	SDRV	3	BLKI	2	BLANK\$COMMON
✓	SDRV	5	BLKR	4	BLKD
✓	RAS	1	016326 017365	0	046141 046153
✓	MAIN	1	017366 020432	2	BLANK\$COMMON
✓	MAIN	3	BLKI	0	046154 046241
✓	MAIN	5	MCT	2	BLANK\$COMMON
✓	MAIN	7	BLKB	6	BLKR

SYSS\$RLIBS. LEVEL 63  
END OF COLLECTION - TIME 2.084 SECONDS

APPENDIX C

COMPUTER CODE FOR MINIMUM

MASS OPTIMIZATION

QFOR, IS MAIN  
FOR S9A=07/27/72-19:37:17 (1.0)

MAIN PROGRAM

STORAGE USED: CODE(1) 001035; DATA(0) 000776; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLKI 000324  
0004 BLKD 000312  
0005 MCT 000005  
0006 BLKR 000312

EXTERNAL REFERENCES (BLOCK, NAME)

0007 SDRV  
0010 SCNTL  
0011 CONST  
0012 NDERV  
0013 ETA  
0014 DETA  
0015 NCDERV  
0016 MDER  
0017 INTIAL  
0020 INT4IX  
0021 RKS  
0022 FMINV  
0023 BNDCND  
0024 NINTR5  
0025 NRNL5  
0026 NWD05  
0027 NI025  
0030 SORT  
0031 ASIN  
0032 NI015  
0033 NSTOPS

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000002 1L	0001	000122 100L	0001	000273 105L	0001	000311 108L	0001	000314 109L
0001	000321 110L	0000	000466 2000F	0000	000526 2010F	0000	000541 2040F	0000	000557 205F
0000	000716 2050F	0000	000712 2060F	0001	001015 2070L	0000	000725 2075F	0001	001023 2090L
0000	000733 2095F	0000	000566 210F	0000	000573 215F	0000	000654 220F	0000	000673 221F
0000	000605 230F	0001	000462 2336	0000	000621 240F	0000	000636 250F	0001	001007 300L
0001	001031 3000L	0001	000703 3466	0001	000725 3556	0001	000024 50L	0001	000117 90L
0000	000534 900F	0000 R	000050 A	0000 R	000355 ALPHA	0000 R	000337 AM	0000 R	000372 ATY
0004	000003 A1	0004	000004 A2	0004	000005 A3	0004	000006 A4	0004	000007 A5
0004	000010 A6	0000 R	000373 BYT	0000 R	000356 DA	0014 R	000000 DETA	0000 R	000377 DETZ
0003	000023 DE1DL	0003	000021 DE4DTB	0003 R	000013 DFM	0003 R	000012 DFMX	0003 R	000010 DFNX
0006 R	000006 DLDNCB	0006 R	000001 DLDV	0006 R	000002 DLDV	0005 R	000000 DLMY	0000 R	000120 DLY
0003 R	000007 DNCDT1	0000 R	000357 DR	0000 R	000414 DX	0005 R	000003 DXWRT	0000 R	000024 DY
0000 R	000240 DYST	0000 R	000400 DYY	0000 R	000403 DZCD2T	0000 R	000404 DZCTNB	0000 R	000412 DZE4LN
0000 R	000406 DZE4LU	0000 R	000410 DZE4LV	0000 R	000411 DZE4TN	0000 R	000405 DZE4TU	0000 R	000407 DZE4TV

c 2

0003 R 000016 D2FILL	0006 R 000003 D2LDUU	0006 R 000004 D2LDUV	0006 R 000005 D2LDVV	0006 R 000007 D2LNCB
0006 R 000010 D2LUNB	0006 R 000011 D2LVNB	0003 R 000015 D2MONC	0003 R 000014 D2MLNC	0003 R 000011 D2NDLL
0004 R 000001 D3M2LN	0004 R 000002 D3M2NL	0004 R 000011 EB	0013 R 000000 ETA	0000 R 000375 ETY
0000 R 000374 EYYY	0003 R 000022 E3	0003 R 000002 FM	0003 R 000003 FN	0003 R 000005 FNC
0003 R 000004 FNCBR	0000 I 000415 I	0000 I 000420 IBKP	0005 I 000001 ICNT	0000 I 000422 IERR
0000 I 000416 IFVD	0000 R 000426 INPUT	0000 I 000424 LCNT	0005 R 000004 LSTEP	0000 I 000423 LSTEP
0000 I 000417 N	0000 I 000367 NCT	0000 I 000360 NIT	0000 I 000421 NTRY	0000 R 000144 PD
0000 R 000354 PHI2	0000 R 000074 R	0000 R 000365 RHOD	0006 R 000000 RLAM	0010 R 000000 SCNTL
0000 R 000170 SD	0007 R 000000 SDRV	0000 R 000364 TFLEXT	0003 R 000000 THETAB	0000 R 000402 THETAF
0003 R 000001 THETAS	0000 R 000366 TH4	0000 R 000353 TOTMAS	0000 R 000370 T2	0000 R 000371 T3
0004 R 000000 U	0003 R 000006 V	0000 R 000334 XM	0005 R 000002 XWRT	0000 R 000000 Y
0000 R 000214 YS	0000 R 000310 YSIMP	0000 R 000264 YST	0000 R 000376 YY	0003 R 000017 Y1
0003 R 000020 Y2	0000 R 000425 ZS	0000 R 000401 ZZ	0000 R 000413 ZZZ	0000 R 000361 Z1
0000 R 000362 Z2	0000 R 000363 Z3			

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00100      1*      C
00100      2*      C
00100      3*      C      RADIATOR SYSTEMS OPTIMIZATION FOR MINIMUM MASS
00100      4*      C
00100      5*      C
00101      6*      EXTERNAL SDRV,SCNTL
00103      7*      DIMENSION Y(20),DY(20),A(20),R(20),DLY(20),PD(20),SD(20),
00103      8*      1      YS(20),DYST(20),YST(20),YSIMP(20),XM(3),AM(3,4)
00104      9*      COMMON /BLKI/THETAB,THETAS,FM,FN,FNCBR,FNC,V,DNCOT1,DFNX,D2NDLL,
00104     10*      1      DFMX,DFM,D2MLNC,D2MDNC,D2FMLL,Y1,Y2,DE4DT8,E3,DE1DL/
00104     11*      2      BLKD/U,D3M2LN,D3M2NL,A1,A2,A3,A4,A5,A6,EB
00104     12*      3      /MCT/DLNT,ICNT,XWRT,DXWRT,LSTEP
00104     13*      4      /BLKR/RLAM,DLDU,DLDV,D2LDUU,D2LDUV,D2LDVV,D2LNCB,
00104     14*      5      D2LNCB,D2LUNB,D2LVNB
00105     15*      NAMELIST /INPUT/ THETAS,U,V,FNCBR,TOTMAS,PHI2,ALPHA,DXWRT,DA,DR
00105     16*      C
00106     17*      FFS(X) = (SQRT(X*(X+2.0))+ASIN(1.0/(X+1.0))-1.570796)/X
00107     18*      FFN(X,Y) = 1.0-Y*(0.1460+Y-0.02866)/X
00110     19*      DFFN(X,Y) = (-0.2920+Y+0.02866)/X
00111     20*      1 READ(5,INPUT,END=3000)
00114     21*      NIT      = 1
00115     22*      Z1      = U
00116     23*      Z2      = V
00117     24*      Z3      = FNCBR
00120     25*      XM(1)   = 0.
00121     26*      XM(2)   = 0.
00122     27*      XM(3)   = 0.
00123     28*      TFLEXT  = 2.0
00124     29*      50 CALL CONST(TOTMAS,PHI2,ALPHA,U,V,FNCBR,RLAM,DLDU,DLDV,D2LNCB,
00124     30*      1      D2LDUU,D2LDUV,D2LUNB,D2LDVV,D2LVNB,D2LNCB)
00125     31*      CALL NDERV(RLAM,FN,DFNX,D2NDLL)
00126     32*      RHOD     = RLAM
00127     33*      WRITE(6,2000) THETAS,U,V,FNCBR,TOTMAS,PHI2,ALPHA,RLAM
00141     34*      2000 FORMAT (1H1,10X,10HTHETA-S = F20.8/
00141     35*      1      11X,10H      U = F20.6/
00141     36*      2      11X,10H      V = F20.6/
00141     37*      3      11X,10H FNCBR = F20.6/
00141     38*      4      11X,10H TOTMAS = F20.6/
00141     39*      5      11X,10H PHI2 = F20.6/

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00141 40*      6      11X.10H ALPHA = F20.6/
00141 41*      9      11X.10H LAMBDA = F20.6777/
00142 42*      WRITE(6,2010) NIT
00145 43*      2010 FORMAT(15X,NUMBER OF ITERATIONS = 'I2)
00146 44*      TH4 = THETAS**4
00147 45*      IF(V.GT.0.2) GO TO 90
00151 46*      IF(V.LE.1.0E-10) GO TO 108
00153 47*      90 THETAB = 0.9
00154 48*      95 NCT = 0
00155 49*      100 T2 = THETAB**2
00156 50*      T3 = T2*THETAB
00157 51*      FNC = FNCBR*T3
00160 52*      FM = FFN(RHOD,FNC)*FFS(RHOD)
00161 53*      DFM = OFFN(RHOD,FNC)*FFS(RHOD)
00162 54*      AYY = T3*THETAB-TH4
00163 55*      BYY = 1.0-THETAB
00164 56*      ETTY = ETA(FNC)
00165 57*      ETY = ETTY*FM+FN
00166 58*      YY = 1.0/V-ETY*AYY/BYY
00167 59*      DETZ = (DFM*ETYY+FM*DETA(FNC))*3.0*T2*FNCBR
00170 60*      DYY = -AYY/BYY*(DETZ+ETY/BYY)-4.0*ETY*T3/BYY
00171 61*      ZZ = THETAB-YY/DYY
00172 62*      IF(NCT.GT.20) GO TO 109
00174 63*      IF(ZZ.LT.1.0) GO TO 105
00176 64*      THETAB = (THETAB+1.0)/2.0
00177 65*      NCT = NCT+1
00200 66*      GO TO 100
00201 67*      105 IF(ABS(ZZ-THETAB)/ZZ.LT.1.0E-06) GO TO 110
00203 68*      NCT = NCT+1
00204 69*      THETAB = ZZ
00205 70*      GO TO 100
00206 71*      108 THETAB = 1.0
00207 72*      GO TO 110
00210 73*      109 WRITE(6,900)
00210 74*      C
00212 75*      110 Y(1) = 1.0
00213 76*      Y(2) = THETAB
00214 77*      900 FORMAT(1H0,20HNEWTON-RAPHSON FAILS)
00215 78*      THETAF = Y(1)
00216 79*      Y1 = Y(1)
00217 80*      Y2 = Y(2)
00220 81*      CALL NCDERV(FNCBR,THETAB,FNC,DVCDT1,D2CD2T,D2CTNR)
00221 82*      CALL MDER(RLAM,FNC,FM,DFMX,DFM,D2FMLL,D2MDNC,D2MLNC,D3M2LN,D3M2NL)
00222 83*      CALL INTIAL(DLDU,D2LDUU,Y(3),Y(4),Y(5),Y(6),0.,1.,D2E4TU,D2E4LU)
00223 84*      CALL INTIAL(DLDV,D2LDVV,Y(7),Y(8),Y(9),Y(10),0.,0.,D2E4TV,D2E4LV)
00224 85*      CALL INTIAL(DLONCB,D2LNCB,Y(11),Y(12),Y(13),Y(14),1.,1.,D2E4TN,
00224 86*      D2E4LN)
00225 87*      CALL INTMIX(D2E4TV,Y(4),DLDU,D2E4LV,D2LDVV,Y(16),Y(15))
00226 88*      CALL INTMIX(D2E4TN,Y(4),DLDU,D2E4LN,D2LVNB,Y(18),Y(17))
00227 89*      CALL INTMIX(D2E4TV,Y(8),DLDV,D2E4LN,D2LVNB,Y(20),Y(19))
00230 90*      ZZZ = 0
00231 91*      DLMT = DX/500.
00232 92*      DO 115 I=1,20
00233 93*      AT(I) = DA
00236 94*      115 R(I) = DR
00238 95*      C
00240 96*      WRITE(6,2040)

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00242 97* 2040 FORMAT(/7X,6HTHETA,10X,6HTHETAB,10X,2HNC,13X,6HETABAR,
00242 98* 1 11X,3HETA,13X,1HM,14X,1HN,14X,1HX/)
00242 99* C
00243 100* ZZZ = 0.0
00244 101* DX = 1.0/(U*(1.0-THETAB))
00245 102* IFVD = 0
00246 103* IF(DX.GT.DXWRT) DX = DXWRT
00250 104* N = 20
00251 105* IBKP = 1
00252 106* NTRY = 1
00253 107* IERR = 0
00253 108* C
00254 109* DLMT = DXWRT/500.0
00255 110* XWRT = 0.0
00256 111* LUTEP = 0
00257 112* LCNT = 0
00260 113* ICNT = 0
00261 114* CALL RK5(SDRV,SCNLC,Y,DY,A,R,ZZZ,DX,N,IFVD,IBKP,NTRY,IERR,
00261 115* 1 DLY,PD,SD,YS,YST,DYST,YSIMP)
00262 116* XM(1) = -Y(3)
00263 117* XM(2) = -Y(7)
00264 118* XM(3) = -Y(11)
00265 119* AM(1,1) = Y(5)
00266 120* AM(1,2) = Y(15)
00267 121* AM(1,3) = Y(17)
00270 122* AM(2,1) = Y(15)
00271 123* AM(2,2) = Y(9)
00272 124* AM(2,3) = Y(19)
00273 125* AM(3,1) = Y(17)
00274 126* AM(3,2) = Y(19)
00275 127* AM(3,3) = Y(13)
00276 128* CALL FMINV(AM,XM,3,4)
00277 129* 205 FORMAT (///45X,27H PARAMETERS AT END OF TUBE //)
00300 130* 210 FORMAT (1X,51X,8HTHETA,=,F10.6,/)
00301 131* 215 FORMAT (1X,23X,3HU =,E14.6,16X,3HV =,E14.6,12X,7HNGBAR =,E14.6,/)
00302 132* 230 FORMAT (1X,19X,7HDTFDO =,E14.6,12X,7HDTFDOV =,E14.6,8X,
00302 133* 1 11HDTFNGBAR =,E14.6,/)
00303 134* 240 FORMAT (1X,17X,9HDTFDO2U =,E14.6,10X,9HDTFDO2V =,E14.6,6X,
00303 135* 1 13HDTFDO2NGBAR =,E14.6,/)
00304 136* 250 FORMAT (1X,16X,10HDTFDO2U =,E14.6,5X,14HDTFDO2NGBAR =,E14.6,
00304 137* 1 5X,14HDTFDO2NGBAR =,E14.6,/)
00305 138* 220 FORMAT (7,3X,24HINCR. SOUGHT DELTA U =,E14.6,10X,9HDELTA V =,
00305 139* 1 E14.6,6X,13HDELTA NGBAR =,E14.6,/)
00306 140* 221 FORMAT (7,3X,24HINCR. USED DELTA U =,E14.6,10X,9HDELTA V =,
00306 141* 1 E14.6,6X,13HDELTA NGBAR =,E14.6,/)
00307 142* NIT = NIT+1
00310 143* Z1 = U
00311 144* Z2 = V
00312 145* Z3 = FNGBR
00313 146* WRITE (6,205)
00315 147* WRITE (6,210) Y(1)
00320 148* WRITE (6,215) Z1,Z2,Z3
00325 149* WRITE (6,230) Y(3),Y(7),Y(11)
00332 150* WRITE (6,240) Y(5),Y(9),Y(13)
00337 151* WRITE (6,250) Y(15),Y(17),Y(19)
00344 152* WRITE(6,220) (XM(I),I=1,3)
00352 153* CALL BNDEND(U,V,FNGBR,XM,TOTMAS,ALPHA)

```

```

00353 154*      WRITE(6,221) (XM(I),I=1,3)
00351 155*      200 IF (INIT .GT. 20) GO TO 300
00363 156*      ZS      = Y(1)-THETAS
00364 157*      IF(ZS.LT.0.0001) GO TO 2070
00366 158*      IF(ABS(Y(1)-TFLEX)/Y(1) .LE. 5.0E-04) GO TO 2090
00370 159*      TFLEX = Y(1)
00371 160*      IF (ABS(XM(1))/Z1.GT.0.0001) GO TO 50
00373 161*      IF (ABS(XM(2))/Z2.GT.0.0001) GO TO 50
00375 162*      IF (ABS(XM(3))/Z3.GT.0.0001) GO TO 50
00377 163*      WRITE(6,2060)
00401 164*      2060 FORMAT(1H0,15HOPTIMUM REACHED)
00402 165*      GO TO 1
00403 166*      300 WRITE (6,2050)
00405 167*      2050 FORMAT (1X,32H NUMBER OF ITERATIONS EXCEEDS 20 )
00406 168*      GO TO 1
00407 169*      2070 WRITE(6,2075)
00411 170*      2075 FORMAT(1H0,24HVANISHING HEAT REJECTION)
00412 171*      GO TO 1
00413 172*      2090 WRITE(6,2095)
00415 173*      2095 FORMAT(1H0,40HINSIGNIFICANT IMPROVEMENT OVER LAST STEP)
00416 174*      GO TO 1
00417 175*      3000 STOP
00420 176*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QFOR,IS CONST  
FOR S9A=07/27/72=19:37:44 (70)

SUBROUTINE CONST ENTRY POINT 000373

STORAGE USED: CODE(1) 000446; DATA(0) 000071; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 SQRT  
0004 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000003 A	0000 R 000002 AZ	0000 R 000004 B	0000 R 000005 C	0000 R 000006 D
0000 R 000010 DAN	0000 R 000007 DAV	0000 R 000014 DBN	0000 R 000013 DBV	0000 R 000017 DCU
0000 R 000020 DCV	0000 R 000026 DDN	0000 R 000024 DDV	0000 R 000025 DDV	0000 R 000023 DZ
0000 R 000012 DZANN	0000 R 000031 DZAVN	0000 R 000011 DZAVV	0000 R 000015 DZBNN	0000 R 000032 DZBVN
0000 R 000015 DZBVV	0000 R 000021 DZCUU	0000 R 000033 DZCUV	0000 R 000022 DZCVV	0000 R 000030 DZDNN
0000 R 000034 DZDVN	0000 R 000027 DZDVV	0000 000043 INJP\$	0000 R 000000 PI	0000 R 000001 Z

00101 1\* SUBROUTINE CONST(TOTMAS,PHI2,ALPHA,U,V,FNCBR,MOD,DLDU,DLDV,DLNCB,  
00101 2\* DZLDUU,DZLDVV,DZLUNB,DZLDVV,DZLVNB,DZLNCB)

00101 3\* C

00103 4\* PI = 3.1415926

00104 5\* Z = V\*\*3/FNCBR

00105 6\* AZ = ALPHA\*ALPHA

00106 7\* A = TOTMAS-AZ\*Z

00107 8\* IF (A.LT. 1.0E-06) A = 1.0E-06

00107 9\* C

00111 10\* 50 B = PI\*ALPHA\*Z

00112 11\* C = PHI2\*ALPHA\*V\*\*2\*SQRT(U)

00113 12\* D = SQRT(B\*B+A\*C)

00114 13\* MOD = (B+D)/A

00114 14\* C

00115 15\* DAV = -3.0\*B\*ALPHA/(PI\*V)

00116 16\* DAN = AZ\*Z/FNCBR

00117 17\* DZAVV = -6.0\*AZ\*V/FNCBR

00120 18\* DZANN = -2.0\*DAN/FNCBR

00120 19\* C

00121 20\* DBV = 3.0\*B/V

00122 21\* DBN = -B/FNCBR

00123 22\* DZBVV = 6.0\*B/(V\*V)

00124 23\* DZBNN = 2.0\*B/(FNCBR\*FNCBR)

00124 24\* C

00125 25\* DCU = C/(2.0\*U)

00126 26\* DCV = 2.0\*C/V

00127 27\* DZCUU = -DCU/(2.0\*U)

00130 28\* DZCVV = DCV/V

00130 29\* C



00131	30*	DZ	= 2.0*D
00132	31*	DDU	= A*DCU/DZ
00133	32*	DDV	= (2.0*B*DBV+A*DCV+C*DAV)/DZ
00134	33*	DDN	= (2.0*B*DBN+C*DAN)/DZ
00135	34*	D2DVV	= (-DDV*DDV+DBV*DBV+B*D2BVV+DAV*DCV+(D2CVV*A+D2AVV*C)/2.0)
00135	35*		/D
00136	36*	D2DNN	= (-DDN*DDN+DBN*DBN+B*D2BNN+C*D2ANN/2.0)/D
00136	37*		
00137	38*	D2AVN	= -3.0*DBN*ALPHA/(PI*V)
00140	39*	D2BVN	= 3.0*DBN/V
00141	40*	D2CUV	= C/(U+V)
00142	41*	D2DVN	= (-DDN*DDV+DBV*DBN+B*D2BVN+(DCV*DAN+C*D2AVN)/2.0)/D
00142	42*		
00143	43*	DLDU	= DCU/DZ
00144	44*	DLDV	= (-HOD*DAV+DBV*DDV)/A
00145	45*	DLNLCB	= (-HOD*DAN+DBN*DDN)/A
00145	46*		
00146	47*	D2LDUU	= (D2CUU-DDU*DCU/D)/DZ
00147	48*	D2LDVV	= (-2.0*DLDV*DAV-HOD*D2AVV+D2BVV+D2DVV)/A
00150	49*	D2LNLCB	= (-2.0*DLNLCB*DAN-HOD*D2ANN+D2BNN+D2DNN)/A
00150	50*		
00151	51*	D2LDUV	= (D2CUV-2.0*DLDU*DDV)/DZ
00152	52*	D2LUNB	= -DLDU*DDN/D
00153	53*	D2LVNB	= (-DLDV*DAN-DLNLCB*DAV-HOD*D2AVN+D2BVN+D2DVN)/A
00154	54*	RETURN	
00155	55*	END	

END OF COMPILATION: NO DIAGNOSTICS.

QFOR, IS BNDEND  
FOR S9A=07/27/72=19:37:47 (,0)

02

SUBROUTINE BNDEND ENTRY POINT 000154

STORAGE USED: CODE(1) 000207; DATA(0) 000027; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NEXP6\$  
0004 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000011	500L	0000	000014	INJP\$	0000	I	000000	L	0000	R	000004	X1	0000	R	000005	X2
0000	R	000001	Z1	0000	R	000002	Z2	0000	R	000003	Z3						

```

00101 1* SUBROUTINE BNDEND(U,V,FNCBR,XM,TOTMAS,ALPHA)
00101 2* C
00101 3* C BNDEND INSURES U,V, AND NCBAR ARE WITHIN THEIR BOUNDARY CONDITIONS
00101 4* C
00103 5* DIMENSION XM(3)
00104 6* L = 1
00105 7* Z1 = U
00106 8* Z2 = V
00107 9* Z3 = FNCBR
00110 10* 500 U = Z1+XM(1)
00111 11* V = Z2+XM(2)
00112 12* FNCBR = Z3+XM(3)
00113 13* IF(L.EQ.2) RETURN
00113 14* C
00115 15* IF(U.LT.5.0E-06) XM(1) = 5.0E-06-Z1
00117 16* IF(U.GT.250.0) XM(1) = 250.0-Z1
00121 17* IF(V.LT.5.0E-06) XM(2) = 5.0E-06-Z2
00123 18* IF(V.GT. 50.0) XM(2) = 50.0-Z2
00125 19* IF(FNCBR.LT.1.0E-04) XM(3) = 1.0E-04-Z3
00127 20* IF(FNCBR.GT.6.0) XM(3) = 6.0-Z3
00131 21* X1 = TOTMAS*FNCBR
00132 22* X2 = ALPHA**2
00133 23* IF(X1-X2+V**3.LT.0.0) XM(2) = (X1/X2)**(1.0/3.0)-Z2
00135 24* L = 2
00136 25* GO TO 500
00137 26* END

```

END OF COMPILATION: NO DIAGNOSTICS.

QFOR:IS NDERV  
FOR 59A-07/27/72-19:37:49 (1.0)

SUBROUTINE NDERV ENTRY POINT 000056

STORAGE USED: CODE(1) 000070; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 SQRT  
0004 ASIN  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 C4      0000 R 000001 C5      0000 R 000002 C6      0000 000006 INJP\$

00101	1*		SUBROUTINE NDERV(RLAM, FN, DFNX, D2NDLL)
00101	2*	C	
00101	3*	C	N AND ITS DERIVATIVES
00101	4*	C	
00103	5*		C4 = RLAM+1.0
00104	6*		C5 = RLAM*2.0
00105	7*		C6 = SQRT(RLAM+C5)
00106	8*		FN = (1.5707963+C5-C6-ASIN(1.0/C4))/RLAM
00107	9*		DFNX = -(FN+C6/C4-1.0)/RLAM
00110	10*		D2NDLL = -(2.0*DFNX+1.0/(C4*C4*C6))/RLAM
00111	11*		RETURN
00112	12*		END

END OF COMPILATION: NO DIAGNOSTICS.

2  
2  
QFOR:IS MDER

FOR S9A-07/27/72-19:37:57 (10)

SUBROUTINE MDER

ENTRY POINT 000251

STORAGE USED: CODE(1) 0003041 DATA(0) 0000661 BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 SQRT  
0004 ASIN  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000011 CFN	0000 R 000000 C1	0000 R 000017 C10	0000 R 000020 C11	0000 R 000022 C12
0000 R 000001 C2	0000 R 000003 C3	0000 R 000002 C4	0000 R 000004 C5	0000 R 000005 C6
0000 R 000006 C7	0000 R 000007 C8	0000 R 000010 C9	0000 R 000015 DCFDNC	0000 R 000013 DCFNDL
0000 R 000014 DFFSDL	0000 R 000016 D2CFLL	0000 R 000025 D2CFNC	0000 R 000026 D2CLNC	0000 R 000023 D2C2LL
0000 R 000021 D2C3LL	0000 R 000024 D2FSLL	0000 R 000027 D3C2LN	0000 R 000030 D3C2NL	0000 R 000012 FFS
0000 000040 INCP\$				

00101	1*	SUBROUTINE MDER(RLAM,FNC,FM,DFMX,DFM,D2CFLL,D2MDNC,D2MLNC,D3M2LN,
00101	2*	D3M2NL)
00101	3*	C
00101	4*	C
00101	5*	C
00103	6*	C1 = 0.1460*FNC-0.02866
00104	7*	C2 = SQRT(RLAM*(2.+RLAM))
00105	8*	C4 = 1.0/(RLAM+1.0)
00106	9*	C3 = ASIN(C4)
00107	10*	C5 = 1/(C4*C2)
00110	11*	C6 = (C4**2)/SQRT(1-C4**2)
00111	12*	C7 = 1/(RLAM**2)
00112	13*	C8 = FNC/RLAM
00113	14*	C9 = 2.*C7*C8
00114	15*	CFN = 1.0-C1*C8
00115	16*	FFS = (C2+C3-1.57079635)/RLAM
00116	17*	FM = CFN*FFS
00117	18*	DCFNDL = C1*C7*FNC
00120	19*	DFFSDL = (C5-C6-FFS)/RLAM
00121	20*	DFMX = CFN*DFFSDL+FFS*DCFNDL
00122	21*	DCFDCNC = -(0.1460*FNC+C1)/RLAM
00123	22*	DFM = FFS*DCFDCNC
00124	23*	D2CFLL = -C1*C9
00125	24*	C10 = 1-C4**2
00126	25*	C11 = +1.+0.5/C10+C4**2
00127	26*	D2C3LL = 2*(C4**3)*C11/SQRT(C10)
00130	27*	C12 = 1/C2-C2*(C4**2)
00131	28*	D2C2LL = -C12/(C4*C2)**2

00132	29*	D2FSL	= (D2C2LL+D2C3LL-2.*DFFSDL)/RLAM
00133	30*	D2FMLL	= CFN*D2FSL+2.*DFFSDL*DCFNDL*FFS*D2CFLL
00134	31*	D2CFNC	= -0.2920/RLAM
00135	32*	D2WDNC	= FFS*D2CFNC
00136	33*	D2CLNC	= (0.146*FNC+C1)*C7
00137	34*	D2MLNC	= DCFDNC*DFFSDL+D2CLNC*FFS
00140	35*	D3C2LN	= -2.*D2CLNC/RLAM
00141	36*	D3M2LN	= DCFDNC*D2FSL+2.*DFFSDL*D2CLNC*FFS*D3C2LN
00142	37*	D3C2NL	= 0.2920*C7
00143	38*	D3M2NL	= FFS*D3C2NL+DFFSDL*D2CFNC
00144	39*	RETURN	
00145	40*	END	

END OF COMPILATION: NO DIAGNOSTICS.

QFOR,IS INTIAL  
FOR 59A-07/27/72-19:38:01 (70)

SUBROUTINE INTIAL ENTRY POINT 000470

STORAGE USED: CODE(1) 000522/ DATA(0) 000123/ BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLK1 000004

EXTERNAL REFERENCES (BLOCK, NAME)

0004 ETA  
0005 DETA  
0006 DDETA  
0007 NERR34

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000037 A1	0000 R 000040 A2	0000 R 000041 A3	0000 R 000053 B1	0000 R 000054 B2
0006 R 000000 DDETA	0000 R 000051 DDETDU	0005 R 000000 DETA	0000 R 000046 DETADU	0003 R 000023 DE1DL
0000 R 000032 DE1DTC	0000 R 000027 DE1DTB	0000 R 000047 DE1DU	0000 R 000030 DE2DTB	0000 R 000034 DE2DU
0000 R 000031 DE3DTB	0000 R 000035 DE3DU	0003 R 000021 DE4DTB	0000 R 000033 DE4DU	0000 R 000044 DFCNDU
0007 R 000013 DFM	0000 R 000045 DFM DU	0003 R 000012 DFMX	0000 R 000043 DFN DU	0003 R 000010 DFNX
0003 R 000007 DDCDT1	0000 R 000051 D1	0000 R 000062 D2	0000 R 000065 D2E1CB	0000 R 000060 D2E1LU
0000 R 000055 D2E1TU	0000 R 000036 D2E2TU	0000 R 000042 D2E3TU	0000 R 000066 D2E4CB	0000 R 000071 D2E4UU
0003 R 000016 D2FM L	0000 R 000056 D2MDLU	0003 R 000015 D2MDNC	0003 R 000014 D2MLNC	0000 R 000052 D2MNCU
0000 R 000067 D2NCTB	0000 R 000050 D2NCTU	0003 R 000011 D2NDLL	0000 R 000057 D2NDLU	0000 R 000063 D3
0000 R 000064 D4	0004 R 000000 ETA	0000 R 000025 E1	0000 R 000026 E2	0003 R 000022 E3
0003 R 000002 FM	0003 R 000003 FN	0003 R 000005 FNC	0003 R 000004 FNCBR	0000 000100 INJPs
0000 R 000070 TERM:B	0003 R 000000 THETAB	0003 R 000001 THETAS	0003 R 000006 V	0000 R 000000 Y
0003 R 000017 Y1	0003 R 000020 Y2	0000 R 000024 Z		

00101	1*		SUBROUTINE INTIALC DLUU,D2LDUU,Y3,Y4,Y5,Y6,B,C,D2E4TU,D2E4LU)
00101	2*	C	
00101	3*	C	INITIAL CONDITIONS
00101	4*	C	
00101	5*	C	FOR FIRST AND SECOND DERIVATIVES OF U SET C=1 AND B=0
00101	6*	C	FOR FIRST AND SECOND DERIVATIVES OF V SET C=0 AND B=0
00101	7*	C	FOR FIRST AND SECOND DERIVATIVES OF FNCBR SET C=1 AND B=1
00101	8*	C	
00103	9*		DIMENSION Y(20)
00104	10*		COMMON /BLK1/THETAB,THETAS,FM,FN,FNCBR,FNC,V,DNCDT1,DFNX,D2NDLL,
00104	11*		DFMX,DFM,D2MLNC,D2MDNC,D2FM L,Y1,Y2,DE4DTB,E3,DE1DL
00105	12*		Y(1) = Y1
00106	13*		Y(2) = Y2
00107	14*		Z = FNC
00110	15*		E1 = FM*ETA(Z)*FN
00111	16*		E2 = THETAB**4-THETAS**4

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00112 17* E3 = E2/(1-THETAB)
00113 18* Y(3) = 0.
00114 19* DE1DTB = DNCOT1*(DFM*ETA(Z)+FM*DETA(Z))
00115 20* DE2DTB = 4*(THETAB**3)
00116 21* DE3DTB = E3*(DE2DTB+E3)/E2
00117 22* DE4DTB = E1*DE3DTB+E3*DE1DTB
00120 23* DE1DL = ETA(Z)*DFMX+DFNX
00121 24* DE1DNC = ETA(Z)*DFM+DETA(Z)*FM
00122 25* DE4DU = E3*DE1DL*DLDU+B*E3*DE1DNC*THETAB**3
00123 26* Y(4) = -((1-C17V**2+DE4DU)/DE4DTB
00124 27* Y(5) = 0.
00125 28* DE2DU = DE2DTB*Y(4)
00126 29* DE3DU = E3*(E3*Y(4)+DE2DU)/E2
00127 30* D2E2TU = 12*Y(4)*(THETAB**2)
00130 31* A1 = E3*(D2E2TU+DE3DU)/E2
00131 32* A2 = DE2DTB+E3
00132 33* A3 = -E3*DE2DU/(E2**2)+DE3DU/E2
00133 34* D2E3TU = A1+A2+A3
00134 35* DFNDU = DFNX*DLDU
00135 36* DFCNDU = DNCOT1*Y(4)+B*Y(2)**3
00136 37* DFMU = DFMX*DLDU+DFM*DFCNDU
00137 38* DETADU = DETA(Z)*DFCNDU
00140 39* DE1DU = FM*DETADU+ETA(Z)*DFMU+DFNDU
00141 40* D2VCTU = 6*FNCBR*Y(2)*Y(4)+B*3*Y(2)**2
00142 41* DDETDU = DDETA(Z)*DFCNDU
00143 42* D2MNCU = D2MLNC*DLDU+D2MNC*DFCNDU
00144 43* B1 = DFM*DETADU+ETA(Z)*D2MNCU+FM*DDETDU+DETA(Z)*DFMU
00145 44* B2 = DE1DTB*D2NCTU/DNCOT1
00146 45* D2E1TU = DNCOT1*B1+B2
00147 46* D2MDLU = D2FLL*DLDU+D2MLNC*DFCNDU
00150 47* D2NDLU = D2NOLL*DLDU
00151 48* D2E1LU = ETA(Z)*D2MDLU+DETADU*DFMX+D2NDLU
00152 49* D2E4LU = E3*D2E1LU+DE1DL*DE3DU
00153 50* D1 = DETA(Z)*DFCNDU*DFM
00154 51* D2 = ETA(Z)*D2MNCU
00155 52* D3 = DDETA(Z)*FM*DFCNDU
00156 53* D4 = DETA(Z)*DFMU
00157 54* D2E1CB = D1+D2+D3+D4
00160 55* D2E4CB = DE3DU*DE1DNC+E3*D2E1CB
00161 56* D2NCNB = 3*Y(4)*Y(2)**2
00162 57* TERMNB = B*(E3*DE1DNC*D2NCNB+Y(2)**3*D2E4CB)
00163 58* D2E4UU = E3*DE1DNC*D2LUU*DLDU+D2E4LU*TERMNB
00164 59* D2E4TU = E1*D2E3TU+DE3DTB*DE1DU+E3*D2E1TU+DE1DTB*DE3DU
00165 60* Y(6) = -(Y(4)*D2E4TU+D2E4UU+2*(C=1)/V**3)/DE4DTB
00166 61* Y3 = Y(3)
00167 62* Y4 = Y(4)
00170 63* Y5 = Y(5)
00171 64* Y6 = Y(6)
00172 65* RETURN
00173 66* END

```

END OF COMPILATION: NO DIAGNOSTICS.

QFOR,IS DERV  
FOR S9A-07/27/72-19:38:07 (70)

SUBROUTINE DERV ENTRY POINT 000604

STORAGE USED: CODE(1) 000721, DATA(0) 000130, BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLK1 000024  
0004 BLKD 000012

EXTERNAL REFERENCES (BLOCK, NAME)

0005 ETA  
0006 DETA  
0007 DDETA  
0010 DODETA  
0011 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0004 R 000003 A1	0004 R 000004 A2	0004 R 000005 A3	0004 R 000006 A4	0004 R 000007 A5
0004 R 000010 A6	0000 R 000051 A7	0000 R 000063 DA7DU	0010 R 000000 DODETA	0007 R 000000 DDETA
0006 R 000000 DETA	0003 000023 DE1DL	0003 000021 DE4DTB	0003 R 000013 DFM	0003 R 000012 DFMX
0003 R 000010 DFNX	0003 000007 DNCOT1	0000 R 000052 DNDU	0000 R 000024 DY	0000 R 000053 D2A1UU
0000 R 000066 D2A2UU	0000 R 000067 D2A3UU	0000 R 000070 D2A4UU	0000 R 000061 D2A5UU	0000 R 000062 D2A6UU
0000 R 000065 D2E2UU	0000 R 000055 D2EPUU	0000 R 000060 D2E2UU	0003 R 000016 D2FMCL	0003 R 000015 D2MDNC
0003 R 000014 D2MLNC	0000 R 000056 D2MUU	0000 R 000054 D2NCUU	0003 R 000011 D2NDLL	0000 R 000064 D2NUU
0000 R 000057 D3M2UU	0004 R 000001 D3M2LN	0004 R 000002 D3M2NL	0004 R 000011 EB	0005 R 000000 ETA
0003 000022 E3	0003 R 000002 FM	0003 R 000003 FN	0003 R 000005 FNC	0003 R 000004 FNCBR
0000 000077 INJP5	0003 000000 THETAB	0003 R 000001 THETAS	0004 R 000000 U	0003 000006 V
0000 R 000000 Y	0003 R 000017 Y1	0003 R 000020 Y2	0000 R 000050 Z	

00101	1*	SUBROUTINE DERV(Y3,Y4,Y5,Y6,DL0U,D2LD0U,DY3,DY4,DY5,DY6,B,C,
00101	2*	1 DNC0U,DA1DU,DA2DU,DA3DU,DA4DU,DA5DU,DA6DU,
00101	3*	2 DEBDU,DEU,DEPU,DMDU,D2MNCU)
00101	4*	C
00101	5*	C DERIVATIVES
00101	6*	C
00101	7*	C FOR FIRST AND SECOND DERIVATIVES OF U SET C=1 AND B=0
00101	8*	C FOR FIRST AND SECOND DERIVATIVES OF V SET C=0 AND B=0
00101	9*	C FOR FIRST AND SECOND DERIVATIVES OF FNCBR SET C=0 AND B=1
00101	10*	C
00103	11*	DIMENSION Y(20),DY(20)
00104	12*	COMMON /BLK1/THETAB,THETAS,FM,FN,FNCBR,FNC,V,DNCOT1,DFNX,D2NDLL,
00104	13*	1 DFMX,DFM,D2MLNC,D2MDNC,D2FMCL,Y1,Y2,DE4DTB,E3,DE1DL/
00104	14*	2 BLKD/U,D3M2LN,D3M2NL,A1,A2,A3,A4,A5,A6,EB
00105	15*	Y(1) = Y1
00106	16*	Y(2) = Y2



02

00107	17*	Y(3)	= Y3
00110	18*	Y(4)	= Y4
00111	19*	Y(5)	= Y5
00112	20*	Y(6)	= Y6
00113	21*	Z	= FNC
00114	22*	EB	= FM*ETA(Z)*FN
00115	23*	A1	= 1./(Y(1)-Y(2))
00116	24*	A5	= FM*DETA(Z)*ETA(Z)*DFM
00117	25*	A6	= 3*Y(2)**2*FNGBR
00120	26*	A2	= A6*A5/EB
00121	27*	A3	= 4*Y(2)**3/(Y(2)**4-THETAS**4)
00122	28*	A4	= A1+A2+A3
00123	29*	DA1DU	= -A1**2*(Y(3)-Y(4))
00124	30*	DNCDU	= A6*Y(4)+B*Y(2)**3
00125	31*	DEPU	= DDETA(Z)*DNCDU
00126	32*	DMDU	= DFMX*DLDU+DFM*DNCDU
00127	33*	D2MNCU	= D2MLNC*DLDU+D2MNC*DNCDU
00130	34*	DEU	= DETA(Z)*DNCDU
00131	35*	DA5DU	= FM*DEPU+DETA(Z)*DMDU+ETA(Z)*D2MNCU+DFM*DEU
00132	36*	DA6DU	= 6*Y(2)*FNGBR*Y(4)+B*3*Y(2)**2
00133	37*	A7	= A6*DA5DU+A5*DA6DU
00134	38*	DNDU	= DFMX*DLDU
00135	39*	DEBDU	= FM*DEU+ETA(Z)*DMDU+DNDU
00136	40*	DA2DU	= (A7-A2*DEBDU)/EB
00137	41*	DA3DU	= A3*Y(4)*(3/Y(2)-A3)
00140	42*	DA4DU	= DA1DU+DA2DU+DA3DU
00141	43*	DY(4)	= -C/A4+U/A4**2*DA4DU
00142	44*	D2A1UU	= -A1**2*(Y(5)-Y(6))-2*A1*DA1DU*(Y(3)-Y(4))
00143	45*	D2NCUU	= A6*Y(6)+DA6DU*Y(4)+B*3*Y(4)*Y(2)**2
00144	46*	D2EPUU	= DDETA(Z)*D2NCUU+DNCDU**2*DDETA(Z)
00145	47*	D2MUU	= DFMX*D2L0UU+DLDU*(D2FLL*DLDU+D2MLNC*DNCDU)+DFM*D2NCUU
00145	48*	1	+DNCDU*(D2MLNC*DLDU+D2MNC*DNCDU)
00146	49*	D3MNUU	= D2MLNC*D2L0UU+DLDU*(D3M2LN*DLDU+D3M2NL*DNCDU)
00146	50*	1	+D2M0VC*D2NCUU+D1CDU+D3M2NL*DLDU
00147	51*	D2EUU	= DETA(Z)*D2NCUU+DDETA(Z)*DNCDU**2
00150	52*	D2A5UU	= FM*D2EPUU**2*DEPU+DMDU*DETA(Z)*D2MUU+ETA(Z)*D3MNUU
00150	53*	1	+2*DEU*D2MNCU+DFM*D2EUU
00151	54*	D2A6U	= 6*Y(2)*FNGBR*Y(6)+6*FNGBR*Y(4)**2+B*12*Y(2)*Y(4)
00152	55*	DA7DU	= A6*D2A5UU+DA5DU*DA6DU+A5*D2A6U+DA6DU*DA5DU
00153	56*	D2NUU	= DFMX*D2L0UU+D2N0LL*DLDU**2
00154	57*	D2E8UU	= FM*D2EUU+2*DEU*DA4DU+ETA(Z)*D2MUU+D2NUU
00155	58*	D2A2UU	= (DA7DU-A2*D2E8UU-2*DEBDU*DA2DU)/EB
00156	59*	D2A3UU	= A3*((3/Y(2)-A3)*Y(6)+Y(4))*(-3/Y(2)**2*Y(4)-DA3DU)
00156	60*	1	+(1/A3*DA3DU)**2)
00157	61*	D2A4UU	= D2A1UU+D2A2UU+D2A3UU
00150	62*	DY(6)	= -2/A4*DY(4)*DA4DU+U/A4**2*D2A4UU
00161	63*	DY(3)	= U*(Y(4)-Y(3))+Y(2)-Y(1))*C
00162	64*	DY(5)	= U*(Y(6)-Y(5))+2*(Y(4)-Y(3))*C
00163	65*	DY3	= DY(3)
00164	66*	DY4	= DY(4)
00165	67*	DY5	= DY(5)
00166	68*	DY6	= DY(6)
00167	69*	RETURN	
00170	70*	END	

END OF COMPILATION: NO DIAGNOSTICS.

QFOR:IS INTMIX  
FOR 59A=07/27/72=19:38:11 (.0)

SUBROUTINE INTMIX ENTRY POINT 000024

STORAGE USED: CODE(1) 000032; DATA(0) 000007; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLK1 000024

EXTERNAL REFERENCES (BLOCK, NAME)

0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0003 R 000023 DE1DL	0000 R 000000 DE4DL	0003 R 000021 DE4DTB	0000 R 000001 DE4DXY	0003 000013 DFM
0003 000012 DFM	0003 000010 DFNX	0003 000007 DNC DT1	0003 000016 D2FM LL	0003 000015 D2MDNC
0003 000014 D2MLNC	0003 000011 D2NDLL	0003 R 000022 E3	0003 000002 FM	0003 000003 FN
0003 000005 FNC	0003 000004 FNCBR	0000 000002 INJPS	0003 000000 THETAB	0003 000001 THETAS
0003 000006 V	0003 000017 Y1	0003 000020 Y2		

00101	1*	SUBROUTINE INTMIX(D2E4TY,DTBOX,DLOX,D2E4LY,D2LDXY,D2TBXY,D2YFXY)
00103	2*	COMMON /BLK1/THETAB,THETAS,FM,FN,FNCBR,FNC,V,DNC DT1,DFNX,D2NDLL,
00103	3*	1 DFMX,DFM,D2MLNC,D2MDNC,D2FM LL,Y1,Y2,DE4DTB,E3,DE1DL
00104	4*	D2YFXY = 0.
00105	5*	DE4DL = E3*DE1DL
00106	6*	DE4DXY = D2E4LY*DLOX+DE4DL*D2LDXY
00107	7*	D2TBXY = -(DE4DXY+DTBOX*D2E4TY)/DE4DTB
00110	8*	RETURN
00111	9*	END

END OF COMPILATION: NO DIAGNOSTICS.

QFOR,IS MIXDER  
FOR S9A-0772772-19:38:18 (70)

SUBROUTINE MIXDER ENTRY POINT 000347

STORAGE USED: CODE(1) 000454; DATA(0) 000104; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLKI 000024  
0004 BLKD 000012

EXTERNAL REFERENCES (BLOCK, NAME)

0005 DDETA  
0006 DETA  
0007 DODETA  
0010 ETA  
0011 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0004 R 000003 A1	0004 R 000004 A2	0004 R 000005 A3	0004 R 000006 A4	0004 R 000007 A5
0004 R 000010 A6	0007 R 000000 DODETA	0005 R 000000 DDETA	0006 R 000000 DETA	0003 000023 DE1DL
0001 000021 DE4D19	0003 R 000013 DFM	0003 R 000012 DFMX	0003 R 000010 DFNX	0003 000007 DNCOT1
0000 R 000006 DNCOT1	0000 R 000003 D2A1XY	0000 R 000020 D2A2XY	0000 R 000021 D2A3XY	0000 R 000022 D2A4XY
0000 R 000015 D2A5XY	0000 R 000004 D2A6XY	0000 R 000017 D2E8XY	0000 R 000012 D2EPXY	0000 R 000011 D2EXY
0003 R 000016 D2FMLL	0000 R 000013 D2MOLY	0003 R 000015 D2M0NC	0000 R 000014 D2MDXY	0003 R 000014 D2MLNC
0003 R 000011 D2NDLL	0000 R 000016 D2NDXY	0000 R 000007 D3MNLV	0000 R 000010 D3MNXV	0004 R 000001 D3M2LN
0004 R 000002 D3M2NL	0000 R 000005 D3M2NY	0004 R 000011 EB	0010 R 000000 ETA	0003 000022 E3
0003 R 000002 FM	0003 000003 FN	0003 R 000005 FNC	0003 R 000004 FNCBR	0000 000026 INJP3
0003 000000 THETAB	0003 000001 THETAS	0004 R 000000 U	0003 000006 V	0000 R 000000 Y
0003 000017 Y1	0003 R 000020 Y2	0000 R 000002 Z		

00101	1*		SUBROUTINE MIXDER(DTBDY,DTBDY,DTFDY,D2TBXY,D2TFXY,DNCDX,DNCDY,
00101	2*	1	D2TBXE,DA1DX,DA1DY,DA2DX,DA2DY,DA3DX,DA3DY,
00101	3*	2	DA4DY,DA5DX,DA5DY,DA6DX,DA6DY,DLDX,DLDY,
00101	4*	3	D2LDXY,DEBDY,DEBDX,DEX,DEY,DEPX,DEPY,DMDX,
00101	5*	4	DMDY,D24NCX,D2MNCY,B,C,D3TXE,DTFDXY)
00101	6*	C	
00101	7*	C	FOR MIXED DERIVATIVES OF U AND V SET B=1. AND C= 0.
00101	8*	C	FOR MIXED DERIVATIVES OF U AND FNCBR SET B=1. AND C= 1.
00101	9*	C	FOR MIXED DERIVATIVES OF V AND FNCBR SET B=0. AND C= 1.
00101	10*	C	
00103	11*		DIMENSION Y(2)
00104	12*		COMMON /BLKI/THETAB,THETAS,FM,FN,FNCBR,FNC,V,DNCDT1,DFNX,D2NDLL,
00104	13*	1	DFMX,DFM,D2MLNC,D2M0NC,D2FMLL,Y1,Y2,DE4DTB,E3,DE1DL,
00104	14*	2	BLKD/U,D3M2LN,D3M2NL,A1,A2,A3,A4,A5,A6,EB
00105	15*	Y(2)	= Y2
00106	16*	Z	= FNC

00107	17*	DTFDXY = U*(D2T8XY-D2TFXY)+B*(DT8DY-DTFDY)
00110	18*	D2A1XY = 2*DA1DX*DA1DY/A1-A1**2*(D2TFXY-D2T8XY)
00111	19*	D2A6XY = 6*FNCBR*DT8DX*DT8DY+6*Y(2)*FNCBR*D2T8XY+C*6*Y(2)*DT8DX
00112	20*	D3M2NY = DLDY*D3M2NL
00113	21*	DNCDXY = 6*FNCBR*Y(2)*DT8DX*DT8DY+3*FNCBR*(Y(2)**2)*D2T8XY+
00113	22*	I C*3*(Y(2)**2)*DT8DX
00114	23*	D3MNLX = DLDY*D3M2LN+DNCDY*D3M2NL
00115	24*	D3MNLX = D3MNLX+DLDX*D2MLNC*D2LDXY+D3M2NY*DNCDX+D2MNC*DNCDXY
00116	25*	D2EXY = DDETA(Z)*DNCDX*DNCDY+DETA(Z)*DNCDXY
00117	26*	D2EPXY = DDETA(Z)*DNCDX*DNCDY+DDETA(Z)*DNCDXY
00120	27*	D2MDLY = DLDY*D2FMLL+DNCDY*D2MLNC
00121	28*	D2MDXY = DLDX*D2MDLY+DFMX*D2LDXY+D2MNCY*DNCDX+DFM*DNCDXY
00122	29*	D2A5XY = FM*D2EPXY+DEPX*DMDY+DEPY*DMDX+DETA(Z)*D2MDXY+DEY*D2MNCX+
00122	30*	I EYA(Z)*D3MNLX+DEX*D2MVCY+DFM*D2EXY
00123	31*	D2NDXY = DFNX*D2LDXY+DLDX*DLDY+D2NDLL
00124	32*	D2EBXY = FM*D2EXY+DMDY*DEX+ETA(Z)*D2MDXY+DEY*DMDX+D2NDXY
00125	33*	D2A2XY = -(DEBDY*DA2DX+DEBDX*DA2DY+A2*D2EBXY-(A6*D2A5XY+
00125	34*	I DA6DY*DA5DX+DA5DY*DA6JX+A5*D2A6XY))/EB
00126	35*	D2A3XY = DA3DX*DA3DY/A3+D2T8XY*DA3DX/DT8DX+
00126	36*	I A3*DT8DX*(-3*DT8DY/Y(2)**2-DA3DY)
00127	37*	D2A4XY = D2A1XY+D2A2XY+D2A3XY
00130	38*	D3TXYE = U*D2A4XY/A4**2-DA4DY*(2*A4*D2T8XE+B)/A4**2
00131	39*	RETURN
00132	40*	END

END OF COMPILATION: NO DIAGNOSTICS.

QFOR,IS ETA  
FOR S9A=0772772=19:38:27 (.0)

FUNCTION ETA ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000022; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 EXP  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 IL 0000 R 000001 A 0000 R 000010 B 0000 R 000000 ETA 0000 000014 INJPS  
0003 R 000000 POLY

```

00101 1* FUNCTION ETA(X)
00103 2* DIMENSION A(7), B(2)
00104 3* DATA A(1),A(2),A(3),A(4),A(5),A(6),A(7)/0.10E+01, -0.1163143E+01,
00104 4* 1 0.1478836E+01, -0.1267550E+01, 0.6325223E+00, -0.1627067E+00,
00104 5* 2 0.1654223E+01, B(1),B(2)/0.6866095E+00, -0.2297718E+00/
00116 6* IF(X.GT.2.5) GO TO 1
00120 7* ETA = POLY(7,A,X)
00121 8* RETURN
00122 9* 1 ETA = B(1)*EXP(B(2)*X)
00123 10* RETURN
00124 11* END

```

END OF COMPILATION: NO DIAGNOSTICS.

OFOR, IS DETA  
FOR 59A-0772772-19:38:44 (,0)

FUNCTION DETA ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000021; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 EXP  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 IL 0000 R 000001 A 0000 R 000007 B 0000 R 000000 DETA 0000 000013 INJPS  
0003 R 000000 POLY

```

00101 1* FUNCTION DETA(X)
00103 2* DIMENSION A(6); B(2)
00104 3* DATA A(1),A(2),A(3),A(4),A(5),A(6)/=0.1163143E+01, 0.2957672E+01,
00104 4* 1 -0.3802650E+01, 0.2530089E+01, -0.8135335E+00, 0.9925338E-01/
00104 5* 2 B(1),B(2)/=0.1577635E+00, -0.2297718E+00/
00115 6* IF(X.GT.2.5) GO TO 1
00117 7* DETA = POLY(6,A,X)
00120 8* RETURN
00121 9* 1 DETA = B(1)*EXP(B(2)*X)
00122 10* RETURN
00123 11* END

```

END OF COMPILATION: NO DIAGNOSTICS.

2FOR:IS DDETA  
FOR S9A=07/27/72=19:39:00 (170)

FUNCTION DDETA ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000020; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 EXP  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 1L 0000 R 000001 A 0000 R 000006 B 0000 R 000000 DDETA 0000 000012 INCP5  
0003 R 000000 POLY

00101 1\* FUNCTION DDETA(X)  
00103 2\* DIMENSION A(5), B(2)  
00104 3\* DATA A(1),A(2),A(3),A(4),A(5)/0.2957672E+01, -0.7605300E+01,  
00104 4\* 1 0.7590267E+01, -0.3254134E+01, 0.4962669E+00/ B(1),B(2)/  
00104 5\* 2 0.3624960E+01, -0.2297718E+00/  
00114 6\* IF(X.GT.2.5) GO TO 1  
00116 7\* DDETA = POLY(5,A,X)  
00117 8\* RETURN  
00120 9\* 1 DDETA = B(1)\*EXP(B(2)\*X)  
00121 10\* RETURN  
00122 11\* END

END OF COMPILATION: NO DIAGNOSTICS.

QFOR,IS DDDETA  
FOR S9A\*07/27/72-19139:17 (.0)

FUNCTION DDDETA ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 EXP  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 IL 0000 R 000001 A 0000 R 000005 B 0000 R 000000 DDDETA 0000 000011 INJPS  
0003 R 000000 POLY

```

00101 1* FUNCTION DDDETA(X)
00103 2* DIMENSION A(4), B(2)
00104 3* DATA AT(1),A(2),AT(3),A(4)/-0.7605300E+01,0.1518053E+02,
00104 4* 1-0.9762402E+01,0.1985068E+01/
00104 5* 2 B(1),B(2)/-0.8329136E-02,-0.8297718E+00/
00113 6* IF(X.GT.2.5) GO TO 1
00115 7* DDDETA = POLY(4,A,X)
00116 8* RETURN
00117 9* 1 DDDETA = B(1)*EXP(B(2)*X)
00120 10* RETURN
00121 11* END

```

END OF COMPILATION: NO DIAGNOSTICS.



QFOR,IS POLY  
FOR S9A-07/27/72-19:39:21 (70)

FUNCTION POLY ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000013; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000012 1076 0000 000003 INJPS 0000 I 000002 K 0000 I 000001 L 0000 R 000000 POLY

00101	1*	FUNCTION POLY(N,A,X)
00103	2*	DIMENSION A(N)
00104	3*	POLY = 0.
00105	4*	L = N
00106	5*	DO 1 K=1,N
00111	6*	POLY = POLY*X+A(L)
00112	7*	L = L-1
00114	8*	RETURN
00115	9*	END

END OF COMPILATION: NO DIAGNOSTICS.

QFOR:IS SDRV  
FOR S9A-07/27/72-19:39:24 (70)

SUBROUTINE SDRV ENTRY POINT 000601

STORAGE USED: CODE(1) 000605; DATA(0) 000066; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BLK1 000024  
0004 BLK2 000012  
0005 BLK3 000012  
0006 VAB 000003

EXTERNAL REFERENCES (BLOCK, NAME)

0007 ETA  
0010 NCDEV  
0011 MDER  
0012 DETA  
0013 DERV  
0014 MIXDER  
0015 NERX35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000003 /AA	0004 000003 A1	0004 000004 A2	0004 000005 A3	0004 000006 A4
0004 000007 /S	0004 000010 A6	0000 R 000005 BBA	0000 R 000037 DA1DN	0000 R 000007 DA1DU
0000 R 000023 LA1DV	0000 R 000040 DA2DN	0000 R 000010 DA2DU	0000 R 000024 DA2DV	0000 R 000041 DA3DN
0000 R 000011 LA3DU	0000 R 000025 DA3DV	0000 R 000042 DA4DN	0000 R 000012 DA4DU	0000 R 000026 DA4DV
0000 R 000043 LA5DN	0000 R 000013 DA5DU	0000 R 000027 DA5DV	0000 R 000044 DA6DN	0000 R 000014 DA6DU
0000 R 000030 LA6DV	0000 R 000045 DEBDN	0000 R 000004 DEBOTB	0000 R 000015 DEBDU	0000 R 000031 DEBDV
0000 R 000046 DEN	0000 R 000047 DEPN	0000 R 000017 DEPU	0000 R 000033 DEPV	0012 R 000000 DETA
0000 R 000016 LEU	0000 R 000032 DEV	0003 000023 DEIDL	0003 000021 DE4DTB	0003 R 000013 DFM
0003 R 000012 DFMX	0003 000010 DFNX	0005 R 000006 DLNDCB	0005 R 000001 DLDU	0005 R 000002 DLDV
0000 R 000050 DMDN	0000 R 000020 DMDU	0000 R 000034 DMDV	0000 R 000036 DNCDN	0003 R 000007 DNCOT1
0000 R 000006 DNCDU	0000 R 000022 DNCDV	0000 R 000001 D2CD2T	0000 R 000002 D2CTNB	0003 R 000016 D2FMLL
0005 R 000003 D2LDUU	0005 R 000004 D2LDUV	0005 R 000005 D2LDVV	0005 R 000007 D2LNCH	0005 R 000010 D2LVNB
0005 R 000011 D2LVNB	0003 R 000015 D2MNC	0003 R 000014 D2MLNC	0000 R 000051 D2MNCN	0000 R 000021 D2MNCU
0000 R 000035 D2MNCV	0003 000011 D2NDLL	0004 R 000001 D3M2LN	0004 R 000002 D3M2NL	0004 R 000011 E8
0006 R 000002 ET	0007 R 000000 ETA	0003 000022 E3	0003 R 000002 FM	0003 R 000003 FN
0003 R 000005 FNC	0003 R 000004 FNCR	0000 000056 INJP5	0005 R 000000 RLAM	0003 R 000000 THETAB
0000 R 000000 THETAF	0003 R 000001 THETAS	0004 R 000000 U	0003 000006 V	0003 R 000017 Y1
0003 R 000020 Y2	0006 R 000000 Z1	0006 R 000001 Z2		

00101 1\* SUBROUTINE SDRV(Y,DY,X)  
00103 2\* DIMENSION Y(20),DY(20)  
00104 3\* COMMON /BLK1/THETAB,THETAS,FM,FN,FNCR,FNC,V,DNCDT1,DFNX,D2NDLL,  
00104 4\* DFMX,DFN,D2MLNC,D2MNC,D2FMLL,Y1,Y2,DE4DTB,E3,DEIDL/  
00104 5\* 2 BLKD/U,D3M2LN,D3M2NL,A1,A2,A3,A4,A5,A6,E8

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00104      6*      3      /BLKR/RLAM,DL0U,DL0V,D2LD0U,D2LD0V,D2LD0V,D2LD0V,DL0NCB,
00104      7*      4      D2LNCB,D2LUNB,D2LVNB
00104      8*      5      /VAB/ Z1,Z2,ET
00105      9*      Z1      = FNC
00106     10*      Z2      = FM*ETA(FNC)+FN
00107     11*      ET      = ETA(FNC)
00110     12*      THETAF = Y(1)
00111     13*      THETAB = Y(2)
00112     14*      Y1      = Y(1)
00113     15*      Y2      = Y(2)
00114     16*      CALL NCOERV(FNCBR,THETAB,FNC,DNCDT1,D2CD2T,D2CTNB)
00115     17*      CALL MDER(RLAM,FNC,FM,DF4X,DF4,D2FMLL,D2MDNC,D2MLNC,D3M2LN,D3M2NL)
00116     18*      AAA      = THETAF-THETAB
00117     19*      DY(1)     = -U*AAA
00120     20*      EB      = FM*ETA(FNC)+FN
00121     21*      DEBDYB = 3*THETAB**2+FNCBR*(FM*ETA(FNC)+ETA(FNC)*DFM)
00122     22*      BBB      = 4*THETAB**3/(THETAB**4-THETAS**4)
00123     23*      DY(2)     = -U/(1/AAA*DEBDYB/EB+BBB)
00124     24*      CALL DERV(Y(3),Y(4),Y(5),Y(6),DL0U,D2LD0U,DY(3),DY(4),DY(5),
00124     25*      1      DY(6),0.,1.,DNCDU,DA10U,DA20U,DA30U,DA40U,DA50U,
00124     26*      2      DA60U,DEB0U,DEU,DEPV,DMDU,D2MNCU)
00125     27*      CALL DERV(Y(7),Y(8),Y(9),Y(10),DL0V,D2LD0V,DY(7),DY(8),DY(9),
00125     28*      1      DY(10),0.,0.,DNCDV,DA10V,DA20V,DA30V,DA40V,DA50V,
00125     29*      2      DA60V,DEB0V,DEV,DEPV,DMDV,D2MNCV)
00126     30*      CALL DERV(Y(11),Y(12),Y(13),Y(14),DL0NCB,D2LNCB,DY(11),DY(12),
00126     31*      1      DY(13),DY(14),1.,0.,DNCDN,DA10N,DA20N,DA30N,DA40N,
00126     32*      2      DA50N,DA60N,DEB0N,DEN,DEPN,DMDN,D2MNCN)
00127     33*      CALL MIXDER(Y(4),Y(8),Y(7),Y(16),Y(15),DNCDU,DNCDV,DY(4),DA10U,
00127     34*      1      DA10V,DA20U,DA20V,DA30U,DA30V,DA40V,DA50U,DA50V,
00127     35*      2      DA60U,DA60V,DL0U,DL0V,D2LD0U,DEB0V,DEB0U,DEU,DEV,
00127     36*      3      DEPV,DEPN,DMDU,DMDV,D2MNCU,D2MNCV,1.,0.,DY(16),
00127     37*      4      DY(15))
00130     38*      CALL MIXDER(Y(4),Y(12),Y(11),Y(18),Y(17),DNCDU,DNCDN,DY(4),
00130     39*      1      DA10U,DA10N,DA20U,DA20N,DA30U,DA30N,DA40N,DA50U,
00130     40*      2      DA50N,DA60U,DA60N,DL0U,DL0NCB,D2LUNB,DEB0N,DEB0U,
00130     41*      3      DEU,DEN,DEPV,DEPN,DMDU,DMDN,D2MNCU,D2MNCN,1.,1.,
00130     42*      4      DY(18),DY(17))
00131     43*      CALL MIXDER(Y(8),Y(12),Y(11),Y(20),Y(19),DNCDV,DNCDN,DY(8),
00131     44*      1      DA10V,DA10N,DA20V,DA20N,DA30V,DA30N,DA40N,DA50V,
00131     45*      2      DA50N,DA60V,DA60N,DL0V,DL0NCB,D2LVNB,DEB0N,DEB0V,
00131     46*      3      DEV,DEN,DEPV,DEPN,DMDV,DMDN,D2MNCV,D2MNCN,0.,1.,
00131     47*      4      DY(20),DY(19))
00132     48*      RETURN
00133     49*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

FOR IS SCNTL  
FOR S9A-07/27/72=19:40:52 (10)

SUBROUTINE SCNTL ENTRY POINT 000147

STORAGE USED: CODE(1) 000174; DATA(0) 000014; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 MCT 000005  
0004 VAB 000003  
0005 BLKI 000024

EXTERNAL REFERENCES (BLOCK, NAME)

0006 NWBUS  
0007 NI025  
0010 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000002	2000	0001	000054	5L	0001	000061	50L	0001	000135	60L	0003	R	000000	DLMT				
0000	R	000000	DXSTR	0003	R	000003	DXWRT	0004	R	000002	ET	0003	I	000001	ICNT	0000	000006	INJPS	
0000	I	000001	LCNT	0003	I	000004	LSTEP	0005	R	000000	VAL	0003	R	000002	XWRT	0004	R	000000	Z1
0004	R	000001	Z2																

```

00101 1* SUBROUTINE SCNTL(Y,DY,DX,X,NTRY,IFVD)
00103 2* DIMENSION Y(20),DY(20)
00104 3* COMMON /MCT/ DLMT,ICNT,XWRT,DXWRT,LSTEP
00104 4* 1 /VAB/ Z1,Z2,ET
00104 5* 2 /BLKI/VAL(20)
00105 6* ICNT = ICNT+1
00106 7* IF(DX.GE.DXWRT.AND.ICNT.GT.1) LSTEP = 1
00110 8* IF(ABS(X-XWRT).LT.DLMT) GO TO 50
00112 9* IF(XWRT.GT.X) GO TO 5
00112 10* C
00114 11* DXSTR = DX
00115 12* DX = DX+XWRT-X
00116 13* LCNT = 1
00117 14* NTRY = 3
00120 15* RETURN
00120 16* C
00121 17* 5 NTRY = 1
00122 18* RETURN
00122 19* C
00123 20* 50 WRITE(6,2000) Y(1),Y(2),Z1,Z2,ET,VAL(3),VAL(4),X
00135 21* 2000 FORMAT (8F15.7)
00136 22* IF(LCNT.EQ.1) DX = DXSTR
00140 23* LCNT = 0
00141 24* IF(ABS(1.0-XWRT).LE.DLMT) GO TO 60

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00143 25*      XWRT  = XWRT+DXWRT
00144 26*      NTRY  = 1
00145 27*      IF(LSTEP.EQ.0) RETURN
00147 28*      DX    = DXWRT
00150 29*      IFVD  = 1
00151 30*      RETURN
00151 31*      C
00152 32*      60 NTRY  = 2
00153 33*      RETURN
00154 34*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QFOR,IS RKS

FOR 59A=07/27/72-19:41:32 (70)

SUBROUTINE RKS

ENTRY POINT 000643

STORAGE USED: CODE(1) 001040; DATA(1) 000064; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR2\$

0004 NEXP5\$

0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000010	10L	0001	000313	110L	0001	000333	120L	0001	000045	1266	0001	000343	130L	
0001	000071	140\$	0001	000355	140L	0001	000105	1466	0001	000130	1566	0001	000417	160L	
0001	000150	154\$	0001	000177	1746	0001	000500	185L	0001	000510	190L	0001	000013	20L	
0001	000232	205\$	0001	000270	2176	0001	000524	220L	0001	000530	230L	0001	000543	240L	
0001	000374	243\$	0001	000032	25L	0001	000552	250L	0001	000554	251L	0001	000572	257L	
0001	000604	259L	0001	000623	270L	0001	000456	300L	0001	000615	3366	0001	000054	40L	
0001	000050	45L	0001	000006	5L	0001	000076	50L	0001	000123	70L	0001	000135	80L	
0000	R	000014	AM	0000	R	000007	AMAX	0000	R	000011	C	0000	R	000010	D
0000	R	000003	DEL	0000	R	000012	E	0000	R	000000	FR10	0000	I	000004	I
0000		000030	INJ1\$	0000	I	000002	ISYMP	0000	I	000013	J	0000	R	000006	S

00101	1*	SUBROUTINE RKS(deriv,cntrl,y,dy,a,r,t,del,n,ifvd,ibkp,ntry,	
00101	2*	i1err,deley,pd,sd,ys,yst,dyst,ysimp)	D6000200
00103	3*	DIMENSION Y(N),DY(N),A(N),R(N),DELY(N),	D60 3
00103	4*	1PD(N),SD(N),YS(N),DYST(N),YST(N),YSIMP(N)	D600040
00104	5*	EXTERNAL DERIV, CNTRL	D6000500
00104	6*	C FR10 IS FIFTH ROOT OF TEN	D600 60
00105	7*	FR10=1.5848932	D6000700
00106	8*	IERR=0	D6000800
00106	9*	C YS CONTAINS Y VALUES AT LEFT END POINT OF INTEGRATION INTERVAL	D600090
00106	10*	C	D6001000
00106	11*	C YSIMP CONTAINS Y FOR SIMPSON'S RULE CHECK CHECK NOT MADE FOR	H6001100
00106	12*	C FIXED STEP MODE ISYMP IS CONTROL PARAMETER =1, FIXED, 2 VAR	D6001200
00106	13*	C	D6001300
00106	14*	C IF FIXED STEP SIZE GO ONE INTERVAL OF LENGTH DELT AND RETURN TO	D600140
00106	15*	C CNTRL, IF VAR GO TWO INTERVALS BEFORE RETURN TO CNTRL	D600150
00106	16*	C	D6001600
00106	17*	C IFVD = 0 VARIABLE INTERVAL	D600170
00106	18*	C = 1 FIXED	D6001800
00106	19*	C IBKP = 0 CUT INTERVAL ONCE BEFORE REPEAT (UNDER IFVD=0)	D600190
00106	20*	C = 1 CUT AS REQUIRED	D60 200
00106	21*	C NTRY = 1 CONTINUE INTEGRATING	D6002100
00106	22*	C 2 RETURN FROM RKS	D60 220
00106	23*	C 3 STEP REPEATED WITH NEW DELT	D6002300
00106	24*	C 4 RESTART	D600240

00106	25*	C	IERR = 0	NORMAL	D6002500
00106	26*	C	-1	DELT=0, RETURN FROM RKS	D6002600
00106	27*	C	1	A(I)+R(I)*ABS(Y(I)) = 0., RETURN FROM RKS	D6002700
00107	28*		5	IF(DEL) 20,10,20	D6002800
00112	29*		10	IERR=-1	D6002900
00113	30*			GO TO 270	D7003000
00114	31*		20	CALL DERIV(Y,DY,T)	D6003100
00115	32*			NTRY=1	D6003200
00116	33*			CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)	D6003300
00117	34*		25	DDT=DEL	D6003400
00120	35*			IF(IFVD) 40,30,40	D6003500
00123	36*		30	ISYMP=2	D6003600
00124	37*			DELT=DEL/2.	D6003700
00125	38*			DO 31 I=1,N	D7003800
00130	39*		31	SD(I)=0.0	D6003900
00132	40*			IFLAG=1	D6004000
00133	41*			S=1.	D6004100
00134	42*			GO TO 45	D6004200
00135	43*		40	ISYMP=1	D6004300
00136	44*			DELT=DEL	D6004400
00137	45*		45	DO 46 I=1,N	D6004500
00142	46*			YST(I)=Y(I)	D6004600
00143	47*		46	OYST(I)=DY(I)	D60 470
00145	48*		50	DO 60 I=1,N	D6004800
00150	49*			DELY(I)=DELT*DY(I)	D6004900
00151	50*			PD(I)=DELY(I)	D6005000
00152	51*		60	CONTINUE	D6005100
00154	52*			GO TO (80,70),ISYMP	D6005200
00155	53*		70	DO 71 I=1,N	D6005300
00160	54*		71	SD(I)=SD(I)+S*DY(I)	D6005400
00162	55*		80	T=T+DELT/2.	D6005500
00163	56*			DO 85 I=1,N	D6005600
00166	57*			YS(I)=Y(I)	D6005700
00167	58*			Y(I)=YS(I)+DELY(I)/2.	D6005800
00170	59*		85	CONTINUE	D6005900
00172	60*			CALL DERIV(Y,DY,T)	D6006000
00173	61*			DO 90 I=1,N	D6006100
00176	62*			DELY(I)=DELT*DY(I)	D6006200
00177	63*			PD(I)=PD(I)+2.*DELY(I)	D6006300
00200	64*			Y(I)=YS(I)+DELY(I)/2.	D6006400
00201	65*		90	CONTINUE	D6006500
00203	66*			CALL DERIV(Y,DY,T)	D6006600
00204	67*			DO 95 I=1,N	D6006700
00207	68*			DELY(I)=DELT*DY(I)	D6006800
00210	69*			PD(I)=PD(I)+2.*DELY(I)	D6006900
00211	70*			Y(I)=YS(I)+DELY(I)	D6007000
00212	71*		95	CONTINUE	D6007100
00214	72*			T=T+DELT/2.	D6007200
00215	73*			CALL DERIV(Y,DY,T)	D6007300
00216	74*			DO 100 I=1,N	D6007400
00221	75*			DELY(I)=DELT*DY(I)	D6007500
00222	76*			PD(I)=PD(I)+DELY(I)	D6007600
00223	77*			Y(I)=YS(I)+PD(I)/6.	D6007700
00224	78*		100	CONTINUE	D6007800
00226	79*			GO TO (110,120),ISYMP	D6007900
00227	80*		110	NTRY=1	D6008000
00230	81*			CALL DERIV(Y,DY,T)	D6008100

00231	82*	CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)	D600820
00232	83*	GO TO 300	D6008300
00233	84*	120 GO TO (130,140),IFLAG	D6008400
00234	85*	130 S=4.	D6008500
00235	86*	IFLAG=2	D6008600
00236	87*	CALL DERIV(Y,DY,T)	D6008700
00237	88*	GO TO 50	D7008800
00240	89*	140 CALL DERIV(Y,DY,T)	D6008900
00241	90*	AMAX =0.0	D6009000
00242	91*	DO 180 I=1,N	D6009100
00245	92*	SD(I)=SD(I)+DY(I)	D6009200
00246	93*	YSIMP(I)=YST(I)+DEL*SD(I)/3.	D6009300
00247	94*	D =ABS(Y(I)-YSIMP(I))	D600940
00250	95*	C =A(I)+RT(I)*ABS(Y(I))	D6009500
00251	96*	IF(C ) 160,150,160	D600960
00254	97*	150 IERR=1	D6009700
00255	98*	GO TO 270	D6009800
00256	99*	160 E =ABS(D /C )	D6009900
00257	100*	AMAX=AMAX1(AMAX,E)	D601000
00260	101*	180 CONTINUE	D7010100
00262	102*	IF(AMAX-1.) 215,215,230	D601020
00265	103*	215 NTRY= 1	D6010300
00266	104*	CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)	D6010400
00267	105*	300 IF(NTRY-1) 185,185,310	D601050
00272	106*	310 IF(NTRY-2) 270,270,330	D601060
00275	107*	330 IF(NTRY-3) 340,340,5	D601070
00300	108*	340 T=T-DDT	D601080
00301	109*	IF(DEL) 259,10,259	D6010900
00304	110*	185 GO TO (40,190),ISYMP	D601100
00305	111*	190 IF(AMAX-.75) 200,25,220	D6011100
00310	112*	200 IF(AMAX-.075) 210,25,25	D601120
00313	113*	210 DEL=DEL*FR10	D6011300
00314	114*	GO TO 25	D6011400
00315	115*	220 DEL=DEL/FR10	D6011500
00316	116*	GO TO 25	D6011600
00317	117*	230 I =1+ I\$KP	D6011700
00320	118*	GO TO (240,250),I	D6011800
00321	119*	240 T=T-DEL	D6011900
00322	120*	DEL=DEL/FR10	D6012000
00323	121*	GO TO 259	D601210
00324	122*	250 J=1	D6012200
00325	123*	251 AM=AMAX/10.**J	D6012300
00326	124*	IF(1.-AM) 255,257,257	D601240
00331	125*	255 J=J+1	D6012500
00332	126*	GO TO 251	D601260
00333	127*	257 T=T-DEL	D601270
00334	128*	DEL=DEL/(FR10**J)	D601280
00335	129*	259 DO 245 I=1,N	D601290
00340	130*	DY(I)=DYST(I)	D6013000
00341	131*	245 Y(I)=YST(I)	D6013100
00343	132*	GO TO 25	D6013200
00344	133*	270 RETURN	D6013300
00345	134*	END	D6013400

END OF COMPILATION:

NO DIAGNOSTICS.



QFOR,IS FMINV  
FOR S9A=0772772-19:41:58 (10)

SUBROUTINE FMINV ENTRY POINT 000254

STORAGE USED: CODE(1) 000306; DATA(0) 001260; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000046	1053	0001	000151	111L	0001	000051	117G	0001	000071	117G	0001	000104	123G	
0001	000125	1333	0001	000130	137G	0001	000172	150G	0001	000175	154G	0001	000227	164G	
0001	000212	20L	0000	R	001214	AA	0000	R	001217	B	0000	I	001212	I	
0000	I	001215	11	0000	I	001220	12	0000	I	001213	J	0000	I	001216	K
												0000	R	000000	XMAT

00101	1*	SUBROUTINE FMINV (A,X,N,M)	1
00103	2*	DIMENSION A(N,N),X(N),XMAT(25,26)	2
00104	3*	DO 1 I=1,N	3
00107	4*	XMAT(I,M) = X(I)	4
00110	5*	DO 1 J=1,M	5
00113	6*	1 XMAT(I,J) = A(I,J)	6
00116	7*	DO 20 I=1,N	7
00121	8*	AA = XMAT(I,1)	8
00122	9*	DO 5 J=1,M	9
00125	10*	5 XMAT(I,J) = XMAT(I,J)/AA	10
00127	11*	IF (1.EQ.1) GO TO 11	11
00131	12*	11 = I-1	12
00132	13*	DO 10 K=1,I1	13
00135	14*	B = XMAT(K,I)	14
00136	15*	DO 10 J=1,M	15
00141	16*	10 XMAT(K,J) = XMAT(K,J) - XMAT(I,J) * B	16
00144	17*	IF (1.EQ.N) GO TO 20	17
00146	18*	11 12 = I+1	18
00147	19*	DO 15 K=12,N	19
00152	20*	B = XMAT(K,I)	20
00153	21*	DO 15 J=1,M	21
00156	22*	15 XMAT(K,J) = XMAT(K,J) - XMAT(I,J) * B	22
00161	23*	20 CONTINUE	23
00163	24*	DO 25 I=1,N	24
00166	25*	25 X(I) = XMAT(I,M)	25
00170	26*	RETURN	26
00171	27*	END	

END OF COMPILATION: NO DIAGNOSTICS.

9FOR,IS NCDERV  
FOR 59A-07/27/72-19:42:07 (10)

SUBROUTINE NCDERV ENTRY POINT 000031

STORAGE USED: CODE(1) 0000447 DATA(0) 0000131 BLANK COMMON(2) 0000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000004 I JPS 0000 R 000000 I2 0000 R 000001 I3

00101	1*		SUBROUTINE NCDERV(FNCBR,THETAB,FNC,DNCDT1,D2CD2T,D2CTNB)
00101	2*	C	
00101	3*	C	NE AND ITS DERIVATIVES
00101	4*	C	
00103	5*		T2 = THETAB**2
00104	6*		T3 = T2*THETAB
00103	7*		FNC = FNCBR*T3
00106	8*		DNCDT1 = 3.*FNCBR*T2
00107	9*		D2CD2T = 6.*FNCBR*THETAB
00110	10*		D2CTNB = 3.*T2
00111	11*		RETURN
00112	12*		END

END OF COMPILATION: NO DIAGNOSTICS.

QXQT

MAP 0023-07/27-19:42

ADDRESS LIMITS 001000 021317 040000 047773

STARTING ADDRESS 020263

WORDS DECIMAL 8400 IBANK 4092 DBANK

177961

SEGMENT MAIN		001000 021317	040000 047773
NSWTS\$/FOR	1	001000 001021	
NRBLK\$/FOR	1	001022 001044	
NRWDS\$/FOR	1	001045 001124	2 040000 040011
NWEP\$/FOR	1	001125 001326	2 040012 040031
NFTCH\$/FOR	1	001327 001617	2 040032 040067
NBJCV\$/FOR	1	001620 001752	2 040070 040125
NFTVS\$/FOR	1	001753 001775	
NCLDS\$/FOR	1	001776 002144	2 040126 040157
NWBLK\$/FOR	1	002145 002266	
NBSBL\$/FOR	1	002267 002323	
NUPDS\$/FOR	1	002324 002356	
NBFDS\$/FOR			2 040160 042361
NCNVT\$/FOR	1	002357 002603	2 042362 042451
NINVS\$/FOR	1	002604 003014	2 042452 042463
NOTINS\$/FOR	1	003015 003330	2 042464 042467
NOUT\$/FOR	1	003331 004317	2 042470 042514
NFMT\$/FOR	1	004320 005176	2 042515 042571
NIDRS\$/FOR	1	005177 005351	2 042572 042676
NFCHK\$/FOR	1	005352 006233	2 042677 043035
			4 043036 043107
NTA3\$/FOR			2 043110 043146
ERUS\$/MISC			
NLINP\$/FOR	1	006234 007747	2 043147 043330
TIRS\$/TECH	1	007750 010434	0 043331 043361
			2 043362 043641
NEXP6\$/FOR	1	010435 010631	2 043642 043713
ASINCS\$/FOR	1	010632 011046	0 043714 043741
SQRT\$/FOR	1	011047 011107	2 043742 043753
EXP\$/FOR	1	011110 011177	2 043754 043774
NTER\$/FOR	1	011200 011261	2 043775 044130
NOJFS\$/FOR	1	011262 011326	
NEXP5\$/FOR	1	011327 011414	2 044131 044140
NERR\$/FOR	1	011415 011741	2 044141 044304
BLKR (COMMON BLOCK)			044305 044316
BLKD (COMMON BLOCK)			044317 044330
BLKI (COMMON BLOCK)			044331 044354
VAB (COMMON BLOCK)			044355 044357
MCT (COMMON BLOCK)			044360 044364
BLANK\$COMMON (COMMON BLOCK)			

NCDERV	1	011742 012005	0	044365 044377
PMINV	1	012006 012313	2	BLANKSCOMMON
RKS	1	012314 013353	0	044400 045657
SCNTL	1	013354 013547	2	BLANKSCOMMON
	3	MCT	0	045660 045743
	5	BLKI	2	BLANKSCOMMON
SURV	1	013550 014354	0	045744 045757
	3	BLKI	2	BLANKSCOMMON
	5	BLKR	4	VAB
POLY	1	014355 014420	6	VAB
DDETA	1	014421 014464	0	046046 046062
DDETA	1	014465 014530	2	BLANKSCOMMON
DETA	1	014531 014574	0	046063 046101
ETA	1	014575 014640	2	BLANKSCOMMON
MIXDER	1	014641 015314	0	046102 046121
	3	BLKI	2	BLANKSCOMMON
INTNIX	1	015315 015346	4	BLKD
DERV	1	015347 016267	0	046271 046277
	3	BLKI	2	BLANKSCOMMON
INITIAL	1	016270 017011	4	BLKD
	3	BLKI	0	046430 046552
MDER	1	017012 017315	2	BLANKSCOMMON
NDERV	1	017316 017405	0	046553 046640
BNDEND	1	017406 017614	2	BLANKSCOMMON
CONST	1	017615 020262	0	046641 046655
MAIN	1	020263 021317	2	BLANKSCOMMON
	3	BLKI	0	046656 046704
	5	MCT	2	BLANKSCOMMON
			4	BLKD
			6	BLKR

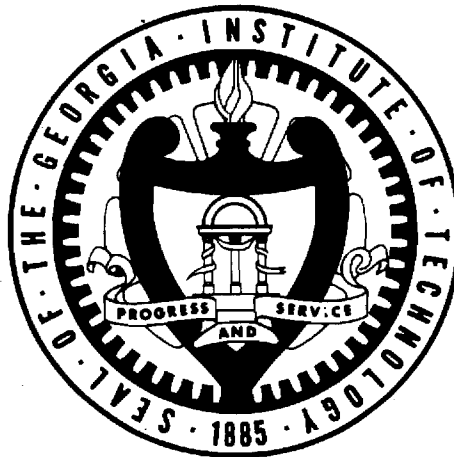
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GEORGIA INSTITUTE OF TECHNOLOGY  
School of Mechanical Engineering  
Atlanta, Georgia

*Final Report*  
*Pt. 2*

SPACE RADIATOR SIMULATION  
MANUAL FOR COMPUTER CODE



Contract No. NAS 9-10415  
by  
William Z. Black and Wolfgang Wulff

Sponsored by the  
Power Generation Branch  
Manned Spacecraft Center  
NATIONAL AERONAUTICS AND SPACE ADMINISTRATION  
Houston, Texas

April 1972

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SCHOOL OF MECHANICAL ENGINEERING  
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## FOREWORD

This report constitutes the Users Manual that accompanies the computer simulation of a space radiator system. It summarizes a two year project carried out by the School of Mechanical Engineering at the Georgia Institute of Technology, Atlanta, Georgia for the NASA Manned Spacecraft Center, Houston, Texas under Contract No. NAS 9-10415. The results of the analysis phase of the program are contained in a separate report entitled "Space Radiator Simulation - System Analysis". A third report submitted under this contract covers the development of a simplified system simulation and the initial phases of a system optimization procedure.

The contract was entitled "Study of Design Parameters of Space Base and Space Shuttle Heat Rejection Systems". The work was monitored by Dr. W. E. Simon of the Power Generation Branch of NASA MSC, Houston, Texas, and was carried out by Dr. W. Z. Black and Dr. W. Wulff as Co-Investigators and Mr. S. M. Morcos, Mr. S. L. Yao and Mr. R. M. Hinson graduate students in the School of Mechanical Engineering under the direction of Dr. S. P. Kezios. Mr. C. R. Caldarale of the Rich Electronic Computing Center also contributed to this report.

The Work carried out by Dr. W. Z. Black is reflected in Section B.2 a and b and Section B.3, d through f, of the Permanent Part of the program. The program units of the Selective part described in Section B.3 were completed by Mr. Morcos under the direction of the Co-Investigators.

Dr. Black is responsible for the coding of programs No. 14 through 22, for the supervision of coding programs No. 23 through 30 which was carried out by Mr. Caldarale and Mr. Hinson and for the supervision of coding programs No. 51 through 60 which was carried out by Mr. Morcos. Dr. W. Wulff is responsible for the general program structure



described in Section B.1 and the program units described in the Sections B.3a through c and B.3g of the Permanent Part of the program. Dr. Wulff coded programs 1 through 7, 31 through 38 and 40 through 42 and supervised Mr. Yao in the coding of programs No. 8 through 13 and No. 39 and supervised Mr. Morcos in the coding of programs No. 43 through 50 and No. 61 and 62.

## SUMMARY

This manual describes a computer program that simulates the performance of a space radiator. The program basically consists of a rigorous analysis which analyzes a symmetrical fin panel and an approximate analysis that predicts system characteristics for cases of non-symmetrical operation. The rigorous analysis accounts for both transient and steady state performance including aerodynamic and radiant heating of the radiator system. The approximate analysis considers only steady state operation with no aerodynamic heating.

The first section of this manual contains a description of the radiator system and instructions to the user for program operation. The input required for the execution of all program options is described. Several examples of program output are contained in this section. Sample output includes the radiator performance during ascent, reentry and orbit.

The second section of the user's manual describes the structure of the entire program as well as the function of each program unit. All interfaces between the various program units are described so that the user may remove a single unit or group of program units that have general utility for use in other analyses. This section concludes with a program listing.

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## A. USERS INSTRUCTIONS

### 1. Program Objectives and Capabilities

The program units described in this manual constitute a transient heat transfer analysis of a space radiator heat rejection system. The analysis is intended to simulate the radiator system of the space base and the space shuttle for conditions of arbitrarily prescribed combinations of aerodynamic heating and incident solar, earth albedo and planetary irradiation.

An exact analysis is performed for the radiator system consisting of equally spaced parallel coolant channels existing in one plane connected by plane fin panels of trapezoidal cross-section. The midplane between tubes and a parallel plane passing through the tube axis are assumed to be planes of symmetry for the purpose of the exact analysis. The program results are extended over the entire radiator panel on the basis of this symmetry.

An approximate analysis is used to extend the exact program simulation to cover the situation of the non-symmetrical radiator panel. This extension predicts on the assumption of one-dimensional analysis, the system performance when the radiator panel is curved and when the tubes are unequally spaced, non-symmetrically loaded or U-shaped.

The exact system analysis predicts both transient and steady-state, two-dimensional temperature profiles, local and total heat transfer rates and coolant fluid temperature and pressure profiles in the flow channel. The analysis also predicts the protection layer thickness which covers the coolant channel and the inlet and exit manifold tubing. Total system mass and projected area are calculated.

The program units described in this manual serve to simulate the radiator system performance. Even though successive runs with system parameter variations are feasible, the task of system optimization is dealt with in the simplified analysis which is described in detail in the Simplified Analysis Manual.

The purpose of this manual is to describe the program units that make up the computer simulation, the necessary input parameters for program execution

and the typical program output for simulated radiator operation. The deck composition is described in part 2, and the input data set is explained in part 3. Part 4 is devoted to a description of the program output and part 5 gives examples of typical radiator performance for ascent, reentry and orbital conditions. Section B describes each subprogram in detail by describing their input and output variables as well as the function of each program unit.

The simulation program is separated into modular units or subprograms so that units of general utility may be removed and run separately. This arrangement also facilitates program modifications if future changes are required. The subprogram concept also reduces the extent of program checking when new program units are added to the existing program package.

The entire program consists of 62 separate program units and the Data Set. The source language is FORTRAN V. The program is executed from catalogued files, using temporary data files as batch processing on the UNIVAC 1108 computer operating under EXEC 8, Version 27.20.143, at the Georgia Tech Rich Electronic Computer Center.

Except for the control statements necessary to collect the program into elements to be stored on mass storage for subsequent program execution from mass storage, the Executive Control Statements are not discussed. Control statements may be freely chosen by the user in accordance with the Executive Control Language. All statements are in 026 key punch code.

#### a. System Description

The exact simulation program predicts the performance characteristics of the typical coolant channel and fin segment shown in Fig. 1. The fin profile has a trapezoidal cross-sectional area. The parallel plane passing through the tube axis and the plane bisecting two adjacent tubes are assumed to be planes of symmetry.

The program has property options that allow the user to select from a number of different coolant fluids, structural and meteoroid protection materials and surface coatings. The materials available for the protection layer, tube and fin are beryllium, copper or aluminum in any combination. The coolant fluids may be selected from a gas, several liquids or a liquid metal.





Possible choices are helium, silicon oil, the two fluorochemicals FC-43 and FC-75 and the liquid metal NaK. The entire radiator system is covered with the same passive thermal coating Z-93 which has optically diffuse but wavelength and temperature dependent optical properties.

The approximate analysis predicts the performance of the non-symmetrical radiator panels shown in Fig. 2. This one-dimensional analysis predicts the approximate location of the adiabatic plane separating two adjacent coolant channels when the radiator panel is curved (Fig. 2a) or the coolant channels are shaped in form of the letter U (Fig. 2b). Other situations which the approximate analysis is capable of treating involve the radiator panel with non-uniformly spaced coolant channels and coolant channels that do not all have identical inlet fluid mass flow rates or identical inlet fluid temperatures.

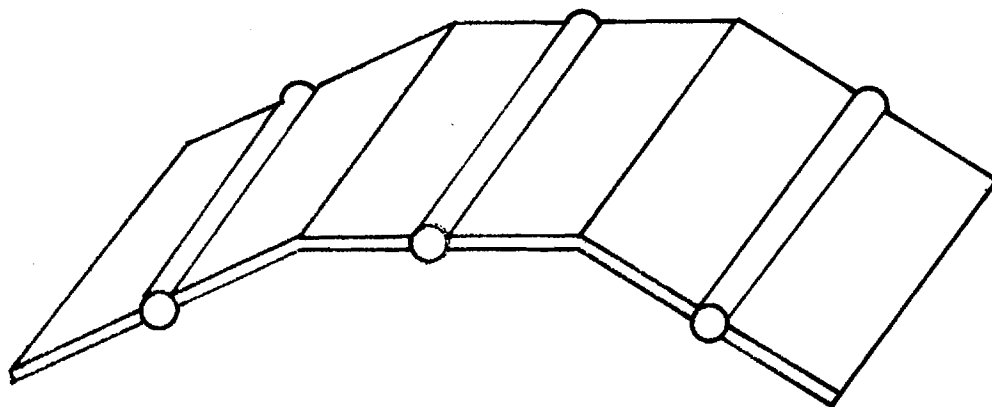
#### b. Operating Conditions

The program simulates radiator characteristics for ascent and reentry of the shuttle radiator system as well as orbital conditions for the shuttle and space base radiator. Typical system performance for these conditions are shown in part 5 of this manual.

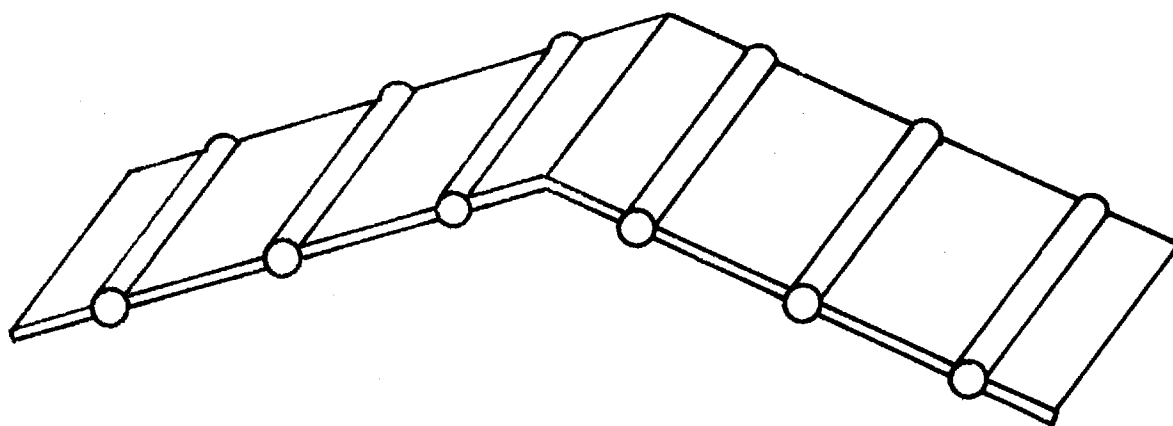
The altitude and velocity profiles of the shuttle vehicle during ascent and reentry are supplied by the user. These profiles are read into the program from card images.

The irradiation data to be supplied by the user is subdivided into three basic categories: solar, earth albedo and planetary irradiation. This data may be either read from card images or from a previously prepared data file which contains the irradiation data in the format specified by the MRI radiant flux program.

The program is capable of system simulation for both transient and steady state conditions. Under the steady state option, integration of the system differential equations proceeds from the user supplied initial condition until steady state is reached. All environmental conditions such as incident radiant flux remain constant during the integration process. Under the transient option, the environmental conditions are allowed to change with time as the integration process proceeds.



(a) Curved Radiator Panel



(b) U Shaped Coolant Channels

Fig. 2 Radiator Panel For Approximate Analysis

Provision has been made within the program to allow for the variation in coolant fluid temperature and mass flow at the inlet plane of the coolant channel. These variations must be supplied by the user as sets of discrete data values punched on cards.

## 2. The Deck Composition

The program structure is dictated by the calling sequence of the Runge-Kutta-Simpson integration procedure (see Section III-2 of the System Analysis Manual). The Source Deck consists of two parts. The first part, referred to as the Permanent Part, requires no attention from the user unless he plans major program modifications, for instance to accommodate geometrical configurations other than the one considered here. The Permanent Part contains the first 42 program units.

The second part, called the Selective Part, is assembled from case to case by the user in accordance with his choice of the

- (i) coolant fluid
- (ii) flow channel material
- (iii) meteoroid protection layer material
- (iv) fin panel material
- (v) thermal control coating

There are always 20 program units in the Selective Part.

Following the Source Deck is the Data Set which is discussed in Section A-3. It consists of six groups, one for each of the following input specifications:

- (i) parameters which specify non-symmetrical radiator panel
- (ii) ascent and reentry profiles
- (iii) solar, albedo and planetary irradiation
- (iv) coolant fluid inlet temperature and mass flux histories
- (v) system specifications
- (vi) program options.

Most of the user's activities will be reflected in his manipulation of the Data Set, especially of the data in groups (v) and (vi) above.

### 3. The Input Data Set

Input data are divided into six groups and assembled in the order of increasing frequency of changes so that data records most likely to be changed are at the end of the Data Set. This enables the user to keep large portions of the Job Deck on mass storage. The six groups in the Data Set define

- (i) the parameters which specify the non-symmetrical radiator panel
- (ii) the ascent and reentry profiles
- (iii) the incident thermal radiant flux as a function of time
- (iv) the coolant fluid inlet temperature and mass flux histories
- (v) the system specifications, other than material selections
- (vi) the options of program execution.

The program may be executed successively after reading new data records beginning at any of the six data groups listed above. Provisions have been also made for restarting the program by reading new data records from one group and skipping the following data group or only skipping portions of the succeeding data groups. The restart options are determined by the value of the restart integer MRSTRT. A description of this integer is given in part (f) of this section.

None of the data groups contains a fixed number of records. The number of records in groups (ii), (iii) and (iv) are determined and specified to the computer by the user. Groups (i), (v) and (vi) contain a minimum of records as specified below; the number of additional records in these groups and of additional sets of groups (ii), (iii) and (iv) depend on the number of additional executions and parameter variations during the same job. In the description of data records (card images) the numbering starts from one in each group.

a) Alphabetical List of Input Data

All input variables needed for program operation are listed below with their units and a brief description of the meaning of the variable.

VARIABLE NAME	UNITS	REFERENCED IN	DESCRIPTION
ABEMF		FINA	ratio of solar absorptivity to infrared emissivity of coating material
AL	ft	TUBEA	array of tube lengths
ALIMIT		RUNOPT	absolute error limit per integration step
ALPHA	gm/(day ft <sup>2</sup> )	PROTLR	constant relating meteoroid flux to mass
ALTA	ft	ascent and reentry data	elements in altitude array used for ascent altitude profile of shuttle
ALTR	ft	ascent and reentry data	elements in altitude array used for reentry altitude profile of shuttle
AMAN	ft <sup>2</sup>	MANIFD	total manifold area projected into plane of fin
AN		PROTLR	see equation 8.6 System Analysis Manual
ATK		PROTLR	see equation 8.8 in System Analysis Manual
BTA		PROTLR	constant relating meteoroid flux to mass
D	in	TUBEA	tube internal diameter
DITBI	in	TUBE	internal tube diameter
DTWRTE	hr	RUNOPT	fixed time interval between data printout
GAMMA		PROTLR	constant used to adjust observed to predicted meteoroid penetration depth

H	in	FINA	array of fin heights
HFNI	in	FIN	fin height from root to tip
ICASE		QNML,MRI	integer value which specifies set of irradiation values on MRI file
ISYM		SYSTEM	integer which specifies type of non-symmetrical loading of tubes
ITAPE		QNML,MRI	integer value to specify card input (= 0) or MRI file input ( $\neq 0$ ) for irradiation data
LFLD		RUNOPT	control integer used to select between variable or fixed inlet fluid properties
LIMWRT		RUNOPT	maximum number of data recordings during integration toward steady state
LTS		RUNOPT	control integer used to select initial system temperature
LTT		RUNOPT	control integer used to select between temperature dependent or independent surface coating properties
M	lbm/hr	FLUIDA	array of mass flow rates at inlets to tubes
MDOTI	lbm/hr	FLOW	total coolant mass flow rate
MRSTRT		GINPT	control integer used for selection of reentry point for restarting program
MSTOTR		RUNOPT	control integer for selecting transient or steady state computation
MZ		TUBE	number of nodal points along tube axis ( $\leq 10$ )
NA		ascent and reentry data	number of instances for specified ascent velocity and altitude data
NCNV		RUNOPT	control integer for selection of orbit, ascent or reentry

NFLD		inlet coolant fluid history	number of mass flow rate and temperature data values
NR		ascent and reentry data	number of instances for specified reentry velocity and altitude data
NRMP		PROTLR	number of radial nodal points in the protection layer ( $\leq 5$ )
NRTBI		TUBE	number of radial nodal points in tube wall ( $\leq 5$ )
NSRD		SYSTEM	number of sides of the radiator panel which radiate (1 or 2)
NT		SYSTEM	number of flat symmetrically loaded panels
NTBS		TUBE	number of flow channels
NTM		QNML,MRI	number of instances for specified irradiation data ( $< 100$ )
NX		FIN	number of nodal points on fin perpendicular to flow channel ( $\leq 10$ )
PHI		PROTLR	see equation 8.1 System Analysis Manual
PHIN	degrees	MRI	array of angles that relate fin panel orientation with MRI data
PHN	degrees	QNML	angle of panel relative to MRI data
PIN	lbf/ft <sup>2</sup>	FLUIDA	array of pressures at inlet to tubes
PO	lbf/ft <sup>2</sup>	FLOW	entrance pressure into flow channel
PROB		PROTLR	probability of no damage caused by meteoroid impact
RHOFNI	lbm/ft <sup>3</sup>	FIN	fin material density
RHOMET	g/cm <sup>3</sup>	PROTLR	meteoroid density
RHOMPI	lbm/ft <sup>3</sup>	PROTLR	protection layer density



RHØTBI	lbm/ft <sup>3</sup>	TUBE	tube wall density
RLIMIT		RUNØPT	relative error per integration step
SRØØTI	in	FIN	fin root thickness
STAGX	ft	FIN	distance from stagnation point on shuttle to center of radiator panel along a streamline
STBI	in	TUBE	tube wall thickness
STIPI	in	FIN	fin tip thickness
TA	sec	ascent and reentry data	elements in time array used for ascent velocity and elevation profiles
TAU	days	PRØTLR	time radiator is exposed to meteoroid environment
TEND	hr	RUNØPT	termination time for transient performance calculation
TH	in	FINA	array of fin thickness
THETA		PRØTLR	see equation 8.1 System Analysis Manual
TI	R	RUNØPT	initial system temperature
TIFLD	R	coolant fluid inlet temperature history	array of coolant fluid inlet temperatures
TIN	R	FLUIDA	array of temperatures at inlets to tubes
TO	R	FLØW	entrance temperature into flow channel
TØ	min	QNML,MRI	time in minutes at which irradiation data is required
TR	sec	ascent and reentry data	elements in time array used for reentry velocity and elevation profiles
TSTAR	R	SYSTEM	array of sink temperatures

VELA	ft/sec	ascent and reentry data	elements in velocity array used for ascent velocity profile of shuttle
VELM	ft/sec	PRØTLR	average meteoroid velocity
VELR	ft/sec	ascent and reentry data	elements in velocity array used for reentry velocity profile of shuttle
VERTX	ft	FIN	total dimension of radiator panel in direction parallel to acceleration of gravity
W1	in	TUBEA	array of distances between side segments of tubes when U-shaped
W2	in	TUBEA	array of distances between base segments of tubes when U-shaped
WIFLD	lbm/hr	coolant fluid inlet mass flow rate history	array of coolant fluid inlet mass flow rates
XL	ft	TUBE	tube length

## b) Parameters Specifying Non-Symmetrical Radiator Panels

Normal program operation assumes that radiator conditions are symmetrical about each coolant fluid channel. Several factors may destroy this symmetry. One may be non-uniform spacing of the tubes. Another factor may be different inlet temperatures or mass flow rates into adjacent tubes. A third factor could be attributed to a curved panel resulting in a variation to sink temperature between adjacent fin segments. These factors are specified in the NAMELIST's that follow.

Numerical values for a non-symmetrical panel, for irradiation data and for system specifications are read into the computer in NAMELIST format, one NAMELIST each with name and input data as follows:

- i) for non-symmetrical panel specifications (see Section (b))
  - SYSTEM for selection of system parameters when panels and/or flow conditions are non-symmetrical
  - TUBEA for non-symmetrical tube specifications
  - FINA for non-symmetrical fin specifications
  - FLUIDA for non-symmetrical fluid specifications
  - MRI for non-symmetrical irradiation data
- ii) for irradiation specifications (see Section (c))
  - QNML for irradiation data under symmetrical radiator panel conditions
- iii) for system specifications (see Section (f))
  - TUBE for flow channel specifications
  - FLØW for coolant fluid flow specifications
  - FIN for fin parameter specifications
  - PRØTLR for the protection layer specifications
  - MANIFD for the manifold input data.

Each input data record under NAMELIST format has

- (i) a blank in its first column,
- (ii) a \$ - sign in its second column,
- (iii) the NAMELIST name, starting in the third column, followed by one or more blanks,

(iv) the string of data in free field format as follows:

$$A = b ,$$

where (A) is the variable name and (b) is a number; imbedded blanks between (A), (=), (b), ( , ) and a following (A) are permissible,

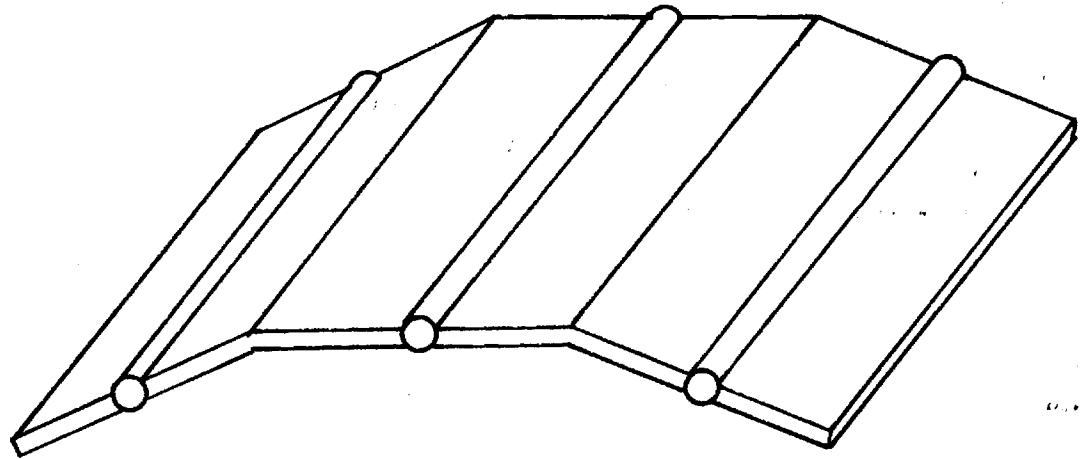
(v) the comma of the last datum is to be placed by a \$ - sign which delimits the NAMELIST data string.

For more details concerning NAMELIST formatting, the reader may consult standard texts on FORTRAN PROGRAMMING. The NAMELIST data strings must be sequenced in the order in which they are listed above.

i) NAMELIST/SYSTEM/contains 4 variables

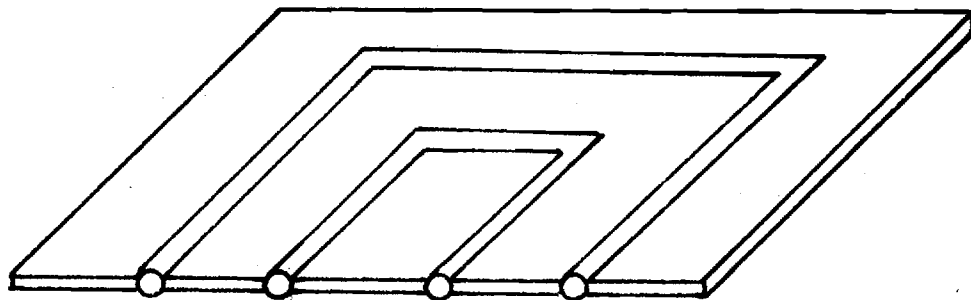
NT	number of flat symmetrically loaded panels, each panel containing NTBS symmetrically loaded tubes (see Fig.3 )
NSRD	= 1 for one side of the panel radiating = 2 for both sides of the panel radiating Note: if panel is curved NSRD is set equal to 1
ISYM	= 0 for symmetrical panel in one plane and straight parallel equally spaced tubes = 1 for non-symmetrical curved panel consisting of straight fin segments and straight parallel tubes = 2 for non-symmetrical panel in one plane with U-shaped tubes = 3 for non-symmetrical curved panel consisting of straight fin segments and U-shaped tubes = 4 for non-symmetrical panel in one plane with straight parallel tubes but non-symmetrical loading
TSTAR	an (NT,3) array of sink temperatures in °R. These values are ignored if MRI file is used for incident flux data (see Section (d)). Also: If the radiator panel is flat only (1, NSRD) of the array is required.

The following four NAMELIST's are not read and should not be placed in the program if ISYM = 0. If ISYM ≠ 0 these four NAMELIST's should appear in the order shown.



NT = 3

NTBS = 1



NT = 2

NTBS = 3

Fig. 3

Examples of the Input Parameters NT and NTBS

ii) NAMelist/TUBEA/contains 4 variables

D            the single value of all tube internal diameters  
in inches

AL           an array of NT tube lengths in feet. These  
values are used only for straight tubes. For  
curved tubes, the lengths are calculated from  
the values of W1 and W2 which are read in below

W1           distance between U-shaped tubes along inlet and  
exit portion of the tube in inches (i.e., both  
side segments of the U) NT + 1 values

W2           distance between U-shaped tubes along bottom  
segment of U in inches, NT + 1 values (see Fig.  
4 for details of dimensions W1 and W2)

iii) NAMelist/FINA/contains 3 variables

H            NT + 1 values of the fin height in inches from  
tube centerline to midplane between tubes. This  
value is used on for the case of straight tubes.  
Fin height for cases of the U tube is calculated  
from W1 and W2 in NAMelist TUBEA

TH           NT values for the fin thickness in inches

ABEMF       ratio of the solar absorptivity to infrared  
emissivity of the surface of the radiator system

iv) NAMelist/FLUIDA/contains 3 variables

TIN          NT values for the inlet fluid temperatures in R

PIN          NT values for the inlet fluid pressure in lbf/ft<sup>2</sup>

M            NT values for the inlet fluid mass flow rate in  
lbm/hr

v) NAMelist/MRI/contains 5 variables

ITAPE = 0   to read irradiation data from cards

         > 0   to read irradiation data from MRI file. Value of  
ITAPE designates the input unit used to read file  
data

ICASE       integer used to designate the case number of  
stored data to be used for input. ICASE = 3,  
for example, would designate the third set of  
data stored on the MRI file to be used for  
current input. If ITAPE = 0, this integer is  
ignored.

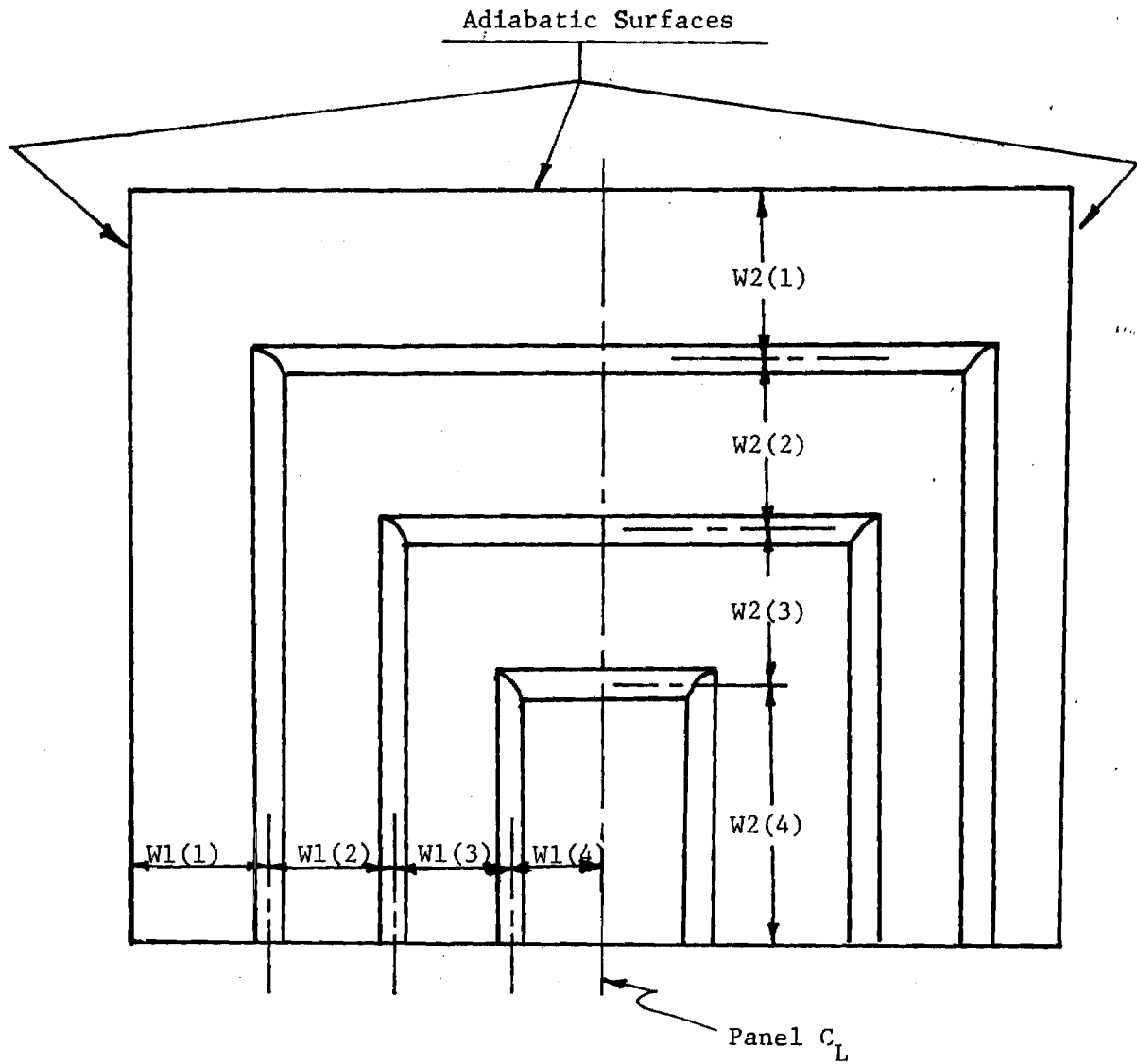


Fig. 4 The Dimensions  $W1(I)$  and  $W2(I)$   
for U Shaped Coolant Channels

NTM            number of instances at which incident flux values  
on to be specified if either card or file data is  
used;  $NTM \leq 100$ . If ITAPE > 0, NTM must be the  
number of data values stored on the MRI file

TØ            time in minutes during orbit at which irradiation  
data is required. Time is measured from zero time  
specified on MRI output data. If ITAPE = 0 this  
value is ignored

PHIN           array of NT values of the angles in degrees  
between the normals to the NT flat radiator  
segments and the reference line used as the zero  
angle reference on the MRI program

### c) Ascent and Reentry Data

Velocity and altitude profiles for the shuttle vehicle during both ascent and reentry are read from cards which have the format listed in the table below. Selection of ascent or reentry phases is determined by the control integer NCØNV (see Section (e) which explains the Execution Control Options).

Card No.	FØRMAT	Variable
1	2I10	NA,NR the number of ordered pairs of elements in the velocity-time and altitude-time arrays for ascent and reentry profiles, respectively
$\text{INT}^* \begin{matrix} \text{through} \\ (NA/10 + 0.9) \\ + 1 = N_1 \end{matrix}$	10F8.3	TA, elements of time array (NA values) in seconds, selected for ascent velocity and elevation profiles
$\begin{matrix} N_1 + 1 \\ \text{through} \\ N_1 + \text{INT}(NR/10 + 0.9) \\ = N_2 \end{matrix}$	10F8.3	TR, elements of time array (NR values) in seconds, selected for reentry velocity and elevation profiles

\* INT(x) = Integer part of x



$N_2 + 1$ through $N_2 + \text{INT}(NA/8+0.875)$ $= N_3$	8E10.3	VELA, elements of velocity array (NA values) in ft/sec, selected for ascent velocity profile of orbiter
$N_3 + 1$ through $N_3 + \text{INT}(NR/8+0.875)$ $= N_4$	8E10.3	VELR, elements of velocity array (NR values) in ft/sec, selected for reentry velocity profile of orbiter
$N_4 + 1$ through $N_4 + \text{INT}(NA/8+0.875)$ $= N_5$	8E10.3	ALTA, elements of altitude array (NA values) in ft, selected for ascent elevation profile of orbiter
$N_5 + 1$ through $N_5 + \text{INT}(NR/8+0.875)$	8E10.3	ALTR, elements of altitude array (NR values) in ft, selected for reentry elevation profile of orbiter

#### d) Irradiation Data

Incident radiant flux on the space base during orbit may be either read from card images or it may be read from a user supplied data file (MRI program). The selection of card or file input is determined by the value of the control integer ITAPE. If ITAPE = 0, the irradiation data is read from cards in the format shown below. If ITAPE > 0, the irradiation data is read from unit ITAPE. The data is assumed to be in the format specified in the MRI manual "Orbiting Satellite Surface Temperature Prediction and Analysis," MRI Project No. NAS9-1059.

The input variables used to input the irradiation data are to be supplied in the NAMELIST format called QNML, if ISYM = 0. The input strings must be in the sequence shown below. If ISYM  $\neq$  0, these values are supplied from the NAMELIST called MRI

NAMELIST/QNML/contains 5 variables

ITAPE = 0 to read irradiation data from cards  
 > 0 to read irradiation data from MRI tape. Value of ITAPE designates the input unit used to read file data

ICASE integer used to designate the case number of stored data to be used for input. ICASE = 3,

for example, would designate the third set of data stored on the MRI file as the irradiation data to be used for current input. If ITAPE = 0, this integer is ignored

- NTM            number of instances at which incident fluxes are to be specified if either card or file data is used;  $NTM \leq 100$ . If ITAPE > 0, NTM must be the number of data values stored on the MRI file
- TØ            time in minutes during orbit at which irradiation data is required. Time is measured from zero time specified on MRI output data. If ITAPE = 0 this value is ignored
- PHN            single angle in degrees between normal to panel and reference line used as a zero angle reference in MRI program.

If the irradiation data is to be read from cards, it must be in the format shown below with the cards immediately following the NAMELIST input cards for QNML.

Card No.	FØRMAT	Variable
1 through NTM	10F8.3	<p>On each card:</p> <div><div><div>(i) time TM in seconds</div><div>(ii) 3 values of solar</div><div>(iii) 3 values of albedo</div><div>(iv) 3 values of planetary</div></div><div>}</div><div>irradiation (Btu/hr ft<sup>2</sup>)</div></div> <p>Incident at TM and each coming from one of three directions in this order:</p> <div><div><div>(i) -30°</div><div>(ii) 0°</div><div>(iii) +30°</div></div><div>}</div><div>from the panel normal (upper side)</div></div>
NTM + 1 through 2*NTM	10F8.3	Same data as above, now for lower fin panel side, only if NSRAD = 2

e) Coolant Fluid Temperature and Mass Flux Histories

When the temperature and mass flow rates entering the coolant channels vary with respect to time, the variation in these quantities are entered from cards in the format indicated below.

Card No.	FORMAT	Variable
1	I10	NFLDTA the number of data values of temperature and mass flow rate to be entered on each card:
2 through NFLD + 1	3F20.6	i) TMEFLD the time in hours corresponding to the given temperature and mass flow rate ii) TIFLD coolant fluid temperature in R at time NFLDTA iii) WIFLD coolant mass flow rate in lbm/hr at time NFLDTA

f) System Specifications

The system specifications define the radiator system and follow immediately after the ascent and coolant fluid temperature and mass flux histories.

The first four cards in the group of the system specification data contain the names of

- (i) the tube material
- (ii) the fin material
- (iii) the coolant fluid
- (iv) the meteoroid protection layer material

in alphanumeric form, beginning with an alphabetic character. Each name is punched on a separate card, beginning in the first column and extending no further than through the first twelve columns. These names serve to identify the printout of results. The user must supply these names in agreement with the composition of the Selective Part described in Chapter I.A. The subsequent data records contain numerical data.

(i) NAMelist/TUBE/contains 7 variables

DITBI internal (hydraulic) tube diameter, in inch,  
 STBI tube wall thickness, in inch,  
 XL tube length, in ft,  
 RHØTBI tube wall density, in lbm/ft<sup>3</sup>,  
 MZ number of nodal points along the tube axis  
 (no. of intervals plus one),  $\leq 10$ ,  
 NRTBI number of nodal points in radial direction, in  
 the tube wall,  $\leq 5$ ,  
 NTBS number of flow channels.

(ii) NAMelist/FLOW/contains 3 variables

MDØTI total coolant mass flow rate, in lbm/hr,  
 TO entrance (and reference) temperature in degrees  
 R,  
 PO entrance (and reference) pressure in lbf/ft<sup>2</sup>.

(iii) NAMelist/FIN/contains 7 variables

NX number of nodal points on the fin, perpendicular  
 to the channel axis,  $NX \leq 10$ ,  
 SRØØTI fin root thickness, in inch,  
 HFNI fin height, from root to tip, in inch,  
 STIPI fin tip thickness, in inch,  
 RHØFNI fin material density, in lbm/ft<sup>3</sup>,  
 STAGX distance from stagnation point on shuttle vehicle  
 to the center of radiator system measured along  
 a streamline, in ft,  
 VERTX total dimension of radiator panel in the direction  
 parallel to the acceleration of gravity, in ft.

(iv) NAMelist/PRØTLR/contains 13 variables

NRMP number of radial nodal points in the protection  
 layer,  $\leq 5$ ,  
 RHØMPI protection layer material density, in lbm/ft<sup>3</sup>,  
 RHØMET meteoroid density in g/cm<sup>3</sup>,  
 VELM meteoroid velocity in ft/sec,  
 TAU time that vulnerable area is exposed to meteoroid  
 environment, in days,  
 PRØB probability of no damage caused by meteoroid  
 impacts, dimensionless,

ALPHA	experimental constant that relates meteoroid flux and mass (see Eq. 8.4) in $\text{gm}/(\text{day ft}^2)$ ,
BTA	experimental constant that relates meteoroid flux and mass (see Eq. 8.4) dimensionless,
GAMMA	empirical constant used to adjust predicted penetration depths to one observed experimentally (see Eq. 8.1) dimensionless,
PHI	empirical constant (see Eq. 8.1) dimensionless,
THETA	empirical constant (see Eq. 8.1) dimensionless,
ATK	empirical constant used to account for spalling on a target of finite thickness (see Eq. 8.8) dimensionless,
AN	experimental constant that describes penetration depth as a function of angle of incidence (see Eq. 8.6) dimensionless.

(v) NAMelist/MANIFD/contains

AMAN	total manifold area, projected into the fin plane, in $\text{ft}^2$ .
------	---

Any one of these variables may be changed for successive program executions by entering them in NAMelist/GINPT/ as discussed below.

g) Execution Control Options

The user has the choice of options for the execution of the first as well as successive computer runs.

Two data records are considered to be execution controls: RUNOPT and GINPT.

(vi) NAMelist/RUNOPT/contains 11 variables

MSTOPT	= 1 to compute steady state conditions
	= 2 to simulate transient system performance,
DTWRTE	fixed time interval, in hr, between data printout during integration,
TEND	termination time, in hr, for transient performance calculation,
ALIMIT	absolute error limit per integration step, see Eq. 15.3, in Systems Analysis Manual

RLIMIT      relative error limit per integration step, see  
Eq. 15.3, in Systems Analysis Manual.

TI            initial temperature, in degrees R,

LIMWRT      maximum number of data recording during integra-  
tion toward steady state, exclusive of initial  
conditions record and steady state record,

NCØNV      = 0    no aerodynamic heating  
             = 1    ascent  
             = 2    reentry,

LTT          = 0    optical properties of the surface coating are  
                 independent of temperature  
             = 1    optical properties of the surface coating are  
                 dependent on temperature  
             = 2    irradiation on fin on tube is zero but subroutine  
                 QINCID is called,

LFLD        = 1    coolant fluid inlet temperature and mass flow  
                 rate are constant  
             = 2    temperature and mass flow rate of inlet coolant  
                 fluid is variable,

LTS          = 0    uniform initial system temperature equal to TI  
             = 1    initial system temperature taken to be equal to  
                 previously obtained values.

(vii) NAMelist/GINPT/contains every variable name listed in items  
(i) through (vi) in addition to NSRD and the following control  
integer.

MRSTRT = 1    new velocity and altitude profiles  
             = 2    new irradiation history  
             = 3    both new velocity, altitude profiles plus new  
                 irradiation history  
             = 4    new coolant fluid inlet conditions  
             = 5    new coolant fluid inlet conditions, plus new  
                 velocity and altitude profiles  
             = 6    new coolant fluid inlet conditions, plus new  
                 irradiation history  
             = 7    new coolant fluid inlet conditions, plus new  
                 velocity and altitude profiles, plus new  
                 irradiation history  
             = 8    the only new input variables are those specified  
                 in GINPT.

There may be as many NAMELIST/GINPT/ records as computing time permits. If GINPT is omitted, the program will terminate.

## 4. Results

### a. Information Produced

The results produced by the program can be divided into three categories each of which is listed on a separate output page. All input variables that specify the system and select the program options are printed on the first page of output (see Fig. 5 ). The second page contains all initial line conditions and system parameters which remain constant during the integration process (see Fig. 6 ). The succeeding pages contain the current system description printed during the integration process (see Fig. 7 ). If the system is non-symmetrical an additional page of output is printed. This page (see Fig. 8 ) lists system parameters when the radiator panel is curved, the flow channels are not identically loaded, or the flow channels are U-shaped.

### b. Output Format

The first output record consists of the list of all system specifications and execution control data as listed in Sections 3(d) and 3(e). This output appears with all of the variables in the NAMELISTS TUBE, FLOW, FIN, PRØTLR, MANIFD and RUNØPT listed in the NAMELIST format. A sample of this output is shown in Fig. 5 .

On the second page of output the initial coolant fluid flow properties are printed in non-dimensional form (see Fig. 6 ). This information includes non-dimensional pressure, velocity, temperature, and tube wall temperature. The reference quantities are listed below the initial line conditions. They are inlet fluid pressure, velocity and temperature. Next are printed several dimensionless parameters such as the Reynold's number, Prandtl number and Nusselt number for the initial coolant fluid flow conditions. The quantity DELTA is the ratio of the internal tube diameter to the tube length. The relative pressure is given by equation 3.31 in the System Analysis Manual. The Biot number of the tube is also printed.



02 QXQT N.VER1

\$TUBE

DITBI = .25000000E+00

STBI = .10000000E+00

XL = .20000000E+01

RHOTBI = .16859200E+03

MZ = +5

NRTBI = +3

NTBS = +12

\$END

\$FLQW

MDOTI = .50000000E+03

ID = .70000000E+03

PO = .25000000E+04

\$END

\$FIN

NX = +5

SRQOTI = .50000000E-01

HFNI = .12000000E+02

STIPI = .50000000E-01

RHOFNI = .16859200E+03

STAGX = .20000000E+02

VERTX = .10000000E+02

FIGURE 5

INPUT VARIABLES

\$END

\$PROTLR

NRMP = +3

RHOWPI = .11357400E+03

RHOWET = .50000000E+00

VELM = .65600000E+05

TAU = .36500000E+04

PROB = .99000000E+00

ALPHA = .18800000E-09

BTA = .12130000E+01

GAMMA = .15000000E+01

PHI = .50000000E+00

THETA = .66666700E+00

ATK = .17500000E+01

AN = .10000000E+01

\$END

\$MANIFD

AMAN = .43000000E+02

\$END

\$RUNOPT

MSTOTR = +2

DTWRT = .83330000E-02

TEND = .20000000E+00

ALIMIT = .50000000E-04

RLIMIT = .10000000E-04

TI = .66000000E+03

LIMWRT = +20

NCONV = +0

LTT = +0

LFLD = +2  
LTS = +0

END

# INITIAL LINE CONDITIONS

\*\*\*\*\*

(ALL QUANTITIES ARE NORMALIZED)

PT.NO.	POSITION Z	PRESSURE P	VELOCITY W	FLUID TEMPERATURE T	WALL TEMPERATURE TWI
1	.000	1.000000	.974335	.942857	.942857
2	.250	.998340	.974339	.942857	.942857
3	.500	.996680	.974342	.942858	.942857
4	.750	.995020	.974346	.942858	.942857
5	1.000	.993359	.974350	.942858	.942857

INLET PRESSURE P0 = 2500.000 LBF/SQ.FT  
REF. VELOCITY W0 = .69331 FT/SEC  
REF. TEMPERATURE T00 = 700.000 R

REYNOLDS NO = .13191+03  
PRANDTL NO = 95.079041  
DELTA = .010417  
REL.PRESSURE IS .170849+04

INIT. NUSSELT NO. NU = .768940+01  
WALL BIOT NO. BI = .362036-02

## SYSTEM PARAMETERS

\*\*\*\*\*

TUBE LENGTH, XL = 2.000 FT  
INTERNAL DIAMETER, DITB = .250 IN  
WALL THICKNESS, STB = .100 IN  
MATERIAL ALUMINUM  
MASS (ALL TUBES), MTB = 3.0896 LBM  
NUMBER OF TUBES, NTBS = 12

FIN HEIGHT, HFN = 12.000 IN  
THICKNESS AT ROOT, SROOT = .050 IN  
THICKNESS AT TIP, STIP = .050 IN  
MATERIAL ALUMINUM  
MASS (ALL FINS), MFN = 33.7184 LBM  
NO. OF FIN SIDES RADIATING = 1

COOLANT FLUID IS SILICONE OIL  
MASS (IN ALL TUBES), MFL = 4.9345 LBM

PROTECTION LAYER THICKNESS, SMP = .063 IN  
MASS, MMP = 1.804 LBM  
MATERIAL IS BERYLLIUM

TOTAL MASS (EXCL. MANIFLD.) MTOT = 43.5462 LBM  
TOTAL AREA (SINGLE NORMAL PROJECTION),  
ATOT = 48.9000 SQ FT

FIGURE 6

INITIAL LINE CONDITIONS AND SYSTEM PARAMETERS

ELAPSED TIME IS .0000 HR  
RELATIVE TIME IS .0000

\*\*IN ORBIT\*\*

1 INTEGR. STEPS

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 700.000 R  
REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .9877+04 BTU/(HR FT)  
TOT. RADIANT REJECTION QTOT = .1295+05 BTU/HR  
COND. FROM MANIFOLDS, CONDMF = .0000 BTU/HR  
INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
AERODYN. HEATING POWER, QCONV = .0000 BTU/HR  
ENERGY STORAGE RATE, STORG = -.1295+05 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T DISTANCE NORMAL TO FLOW DIRECTION X				
Z		.00000	.25000	.50000	.75000	1.00000
.000	.6564	.9429	.9429	.9429	.9429	.9429
.250	.6552	.9429	.9429	.9429	.9429	.9429
.500	.6551	.9429	.9429	.9429	.9429	.9429
.750	.6552	.9429	.9429	.9429	.9429	.9429
1.000 (EXIT)	.6564	.9429	.9429	.9429	.9429	.9429

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 700.000 R  
REFERENCE PRESSURE, P00 = 2500.000 LBF/SQ.FT  
REFERENCE VELOCITY, W00 = .693 FT/SEC  
COOLANT POWER, INLET AT T=T0 H0 = .433766+05 BTU/HR  
EXIT CURRENTLY EI = .361473+05 BTU/HR  
INLET CURRENTLY HI = .361472+05 BTU/HR  
TOT. REJECTION DH = -.108887+00 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	.9743	.9429	.9429	.9429	-.000057	.00000
.250	.99834	.9743	.9429	.9429	.9429	.000283	.05725
.500	.99668	.9743	.9429	.9429	.9429	.000548	.25331
.750	.99502	.9743	.9429	.9429	.9429	.000803	.56830
1.000 (EXIT)	.99336	.9743	.9429	.9429	.9429	.001039	1.00000

FIGURE 7

CURRENT SYSTEM DESCRIPTION

ELAPSED TIME IS  
RELATIVE TIME IS

.0083 HR  
10.3992

\*\*IN ORBIT\*\*

23 INTEGR. STEPS

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 700.000 R

REF. RADIANT HEAT FLUX

PER UNIT AXIAL LENGTH, QREF = .9877+04 BTU/(HR FT)

TOT. RADIANT REJECTION QTOT = .1198+05 BTU/HR

COND. FROM MANIFOLDS, CONDMF = .4641+03 BTU/HR

INCIDENT SOLAR FLUX,

QSOLR = .1000+03 BTU/(HR SQ FT)

INCIDENT INFRARED FLUX,

QIRED = .2000+01 BTU/(HR SQ FT)

AERODYN. HEATING POWER,

QCONV = .0000 BTU/HR

ENERGY STORAGE RATE,

STORG = -.1085+05 BTU/HR

AXIAL  
DIST.

RELATIVE  
RAD. HEAT  
REJECTION

RELATIVE TEMPERATURE OF FIN, T

DISTANCE NORMAL TO FLOW DIRECTION

Z

Q

X

.00000 .25000 .50000 .75000 1.00000

.000	.6304	.9342	.9342	.9342	.9342	.9342
.250	.6024	.9332	.9248	.9233	.9232	.9233
.500	.6010	.9331	.9244	.9228	.9226	.9228
.750	.6022	.9331	.9248	.9233	.9232	.9233
1.000 (EXIT)	.6292	.9338	.9338	.9338	.9338	.9338

## FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 700.000 R

REFERENCE PRESSURE, P00 = 2500.000 LBF/SQ.FT

REFERENCE VELOCITY, W00 = .693 FT/SEC

COOLANT POWER, INLET AT T=0 H0 = .433766+05 BTU/HR

EXIT CURRENTLY EI = .378522+05 BTU/HR

INLET CURRENTLY HI = .385222+05 BTU/HR

TOT. REJECTION DH = .670021+03 BTU/HR

AXIAL  
DIST.

PRESSURE  
P

VELOCITY  
W

FLUID  
TEMP

WALL  
TEMP

PROTECT.  
LAYER  
TEMP

ENTHALPY REJECTION  
PER UNIT TUBE  
LENGTH  
BTU/(HR FT)

FRACTION  
OF  
TOTAL

.000	1.00000	1.0046	.9539	.9342	.9342	25.000915	.00000
.250	.99824	1.0046	.9525	.9332	.9332	24.412890	.27294
.500	.99648	1.0046	.9512	.9331	.9331	22.919608	.53364
.750	.99473	1.0046	.9499	.9331	.9331	21.380966	.77689
1.000 (EXIT)	.99297	1.0046	.9488	.9338	.9338	18.983883	1.00000

FIGURE 7 CONCLUDED

FLAT PANEL  
3 STRAIGHT TUBES  
2 RADIATING SIDES

TUBE SPECIFICATIONS 2.5000-01 INCHES INSIDE DIAMETER

TUBE LENGTH (FEET)

2.0000+00 2.0000+00 2.0000+00

FIN SPECIFICATIONS

FIN THICKNESS (INCHES)

5.0000-02 5.0000-02 5.0000-02

FIN HALF-WIDTH (INCHES)

1.1000+01 1.1500+01 1.2000+01 1.2500+01

SINK TEMPERATURE (DEGREES R)

4.9391+02 4.9391+02 4.9391+02  
5.8736+02 5.8736+02 5.8736+02

FLUID SPECIFICATIONS

INLET TEMPERATURE (DEGREES R)

7.0000+02 7.0000+02 7.0000+02

INLET PRESSURE (LBF/SQ.FT.)

2.5000+03 2.5000+03 2.5000+03

MASS FLOW RATE (LBM/HR)

5.0000+02 5.0000+02 5.0000+02

FIGURE 8  
SYSTEM PARAMETERS FOR NON-SYMMETRICAL HEATING

TUBE	BULK TEMP	INCHES TO ADIABATIC PLANE		PER CENT FIN EFFICIENCY		FIN TIP TEMP DEG R		OUTLET TEMP	HEAT REJECTED
	DEG R	LEFT	RIGHT	LEFT	RIGHT	LEFT	RIGHT	DEG R	BTU/HR
1	6.0619+02	1.1000+01	1.1531+01	5.9360+01	5.7589+01	5.7493+02	5.7314+02	6.0619+02	1.4465+03
2	6.0570+02	1.1469+01	1.2028+01	5.7816+01	5.6025+01	5.7314+02	5.7135+02	6.0570+02	1.4542+03
3	6.0524+02	1.1972+01	1.2500+01	5.6226+01	5.4593+01	5.7135+02	5.6975+02	6.0524+02	1.4613+03

A listing of system parameters follows the initial line condition. The system parameters include the pertinent geometry parameters, the materials and the mass of the tube, fin, coolant fluid and protection layer materials. In addition, the thickness of the meteoroid protection layer, the total system mass excluding the mass of the manifolds and the total projected areas are listed at the bottom of the page.

Following the preliminary results is a series of current system descriptions printed out during the integration process (see Fig. 7). The first items printed are the current real time and the relative time which is the ratio of the current time to the time required for a fluid particle to pass through the flow channel. If the calculations are for the space base in orbit, the words "in orbit" are printed at the top of the page. If the calculations are for either ascent or reentry of the space shuttle, the current shuttle velocity and altitude are printed along with the current atmospheric temperature. For either the sub-orbital or orbital case, the cumulative number of integration steps from the initial conditions is printed at the top of the page.

The next data printed are a summary of the heat flux terms and a listing of the non-dimensional fin temperature distribution. The reference temperature is the inlet fluid temperature and the reference radiant heat flux is the black body emissive power at the inlet coolant fluid temperature evaluated for a unit length of fin and tube segment. The total radiant rejection and the conduction into the fin from both the inlet and exit manifolds for the entire radiator system are also printed. The external heat fluxes incident on the system that are printed are the infrared irradiation from earth albedo, solar irradiation and the convective flux on the radiator system of the shuttle vehicle during ascent and reentry. The energy storage rate for the entire system is also listed along with the external heat fluxes.

The final table listed on the page contains coolant fluid properties as functions of axial distance as well as the local cooling rates. All properties are tabulated in non-dimensional form and the table is preceded by a list of the necessary reference quantities. Properties of the coolant fluid that are printed are pressure, velocity and bulk temperature. The

non-dimensional interior tube wall temperature and the normalized temperature of the meteoroid protection layer temperature are also listed as functions of axial distance down the tube. The enthalpy rejection of the fluid and fraction of the total enthalpy drop are listed as functions down the tube. The latter parameter is helpful in determining the effectiveness of the various segments of the tube length in their capacity to reject heat. Below the reference properties used for the fluid properties are the inlet and exit fluid stagnation enthalpies as well as the total change in stagnation enthalpy. These quantities are labeled respectively,  $H_I$ ,  $H_E$  and  $DH$ . Also for comparison is listed the enthalpy flux at the reference state ( $T_O, P_O$ ) and the reference velocity  $W_O$ .

The last table of current output data constitutes either the system conditions at the specified termination time,  $TEND$  in hours, or the steady state condition depending on whether  $MSTOTR = 1$  or  $MSTOTR = 2$ , respectively. For conditions of steady state, the words "STEADY STATE IS REACHED" are printed after the final data printout.

When input parameters lead to a condition for which the tube centerline or the midplane separating two tubes are not lines of symmetry, then a single page of output is printed prior to all other output. This page contains a written description of tube arrangement on the radiator and a printout describing the panel as either flat or curved. The number of non-symmetrical tubes and the number of sides of the panel radiating are also printed at the top of the page.

The next output consists of a description of the sink temperature data including orientation of the fin segments relative to the reference vector used in the MRI program.

The tube specifications follow the sink temperature data. The data listed are the internal tube diameter and the tube lengths when the tubes are straight. If the tubes are U-shaped, the spacings at the inlet and outlet segments as well as the crossover segments of the tubes are listed instead of the tube lengths.



The fin specifications follow the tube data. The data listed are fin thicknesses and the distance between the tube centerline and the midplane separating two tubes. The effective sink temperature for each fin segment is also printed under fin specifications. When both sides of the panel radiate, two lines of sink temperatures are printed corresponding to the sink temperature both radiator sides.

The coolant fluid specifications follow the fin data. Inlet fluid temperature, pressure and mass flow rates are printed for each non-symmetrical tube.

The final data to be printed on the page is a summary of the system conditions for all of the non-symmetrical fin segments. The data printed includes the fin bulk temperature, position of the adiabatic plane, the fin effectiveness, fin tip temperature, fluid outlet temperature and total heat rejected by the fluid in each of the tubes.

This single page of output is then followed by the usual system printout as described above, repeated once for each non-symmetrical tube.

### c. Output Listing

Figures 5, 6 and 7 in this section are typical output listings produced by the simulation program. Figure 5 provides a listing of all input parameters that appear in the NAMELIST format. Figure 6 shows initial line conditions and system parameters and Figure 7 shows a typical printout of the current radiator system parameters. These three figures are typical of program output when the radiator panel is symmetrical about each flow channel. If non-symmetrical conditions exist, output similar to that shown in Figure 8 is produced.

## 5. Typical Radiator System Performance

This section summarizes typical program results for expected radiator input conditions during ascent and reentry for the shuttle vehicle and during both transient and steady state performance of the radiator system in a simulated orbit.

### a. Ascent

#### (i) Mission Description

The ascent altitude and velocity profiles used for the system output shown in Figures 10 and 11 is given in Figure 9. Other input parameters were selected as reasonable conditions expected for a typical space shuttle radiator system. The total fin area of 124.5 ft<sup>2</sup> is subdivided by 10 equally spaced 12-foot long flow channels. The total fin height between flow channels is 1 ft. The inlet coolant fluid temperature and pressure are 200 F and 60 lb<sub>f</sub>/in<sup>2</sup>, respectively. The system materials are aluminum tubes, fins and meteoroid protection material with silicon oil as the coolant fluid. Other system parameters are printed as input listing which precedes the current system parameters shown in Figure 10a. The panel radiates only from one side. Incident radiant fluxes are constant during ascent at 100 Btu/hr ft<sup>2</sup> for solar irradiation, 2.0 Btu/hr ft<sup>2</sup> for earth albedo. There is no infrared earth irradiation.

#### (ii) System Response

The response of the radiator system during ascent is summarized in Figures 10 and 11. A selected number of output listings of current system parameters during ascent are shown in Figure 10a. Figure 10b is a plot of the total heat rejection from the entire radiator surface during ascent as well as the outlet coolant fluid temperature. Figure 11 is a plot of the aerodynamic heating at various times during the ascent phase of the mission.

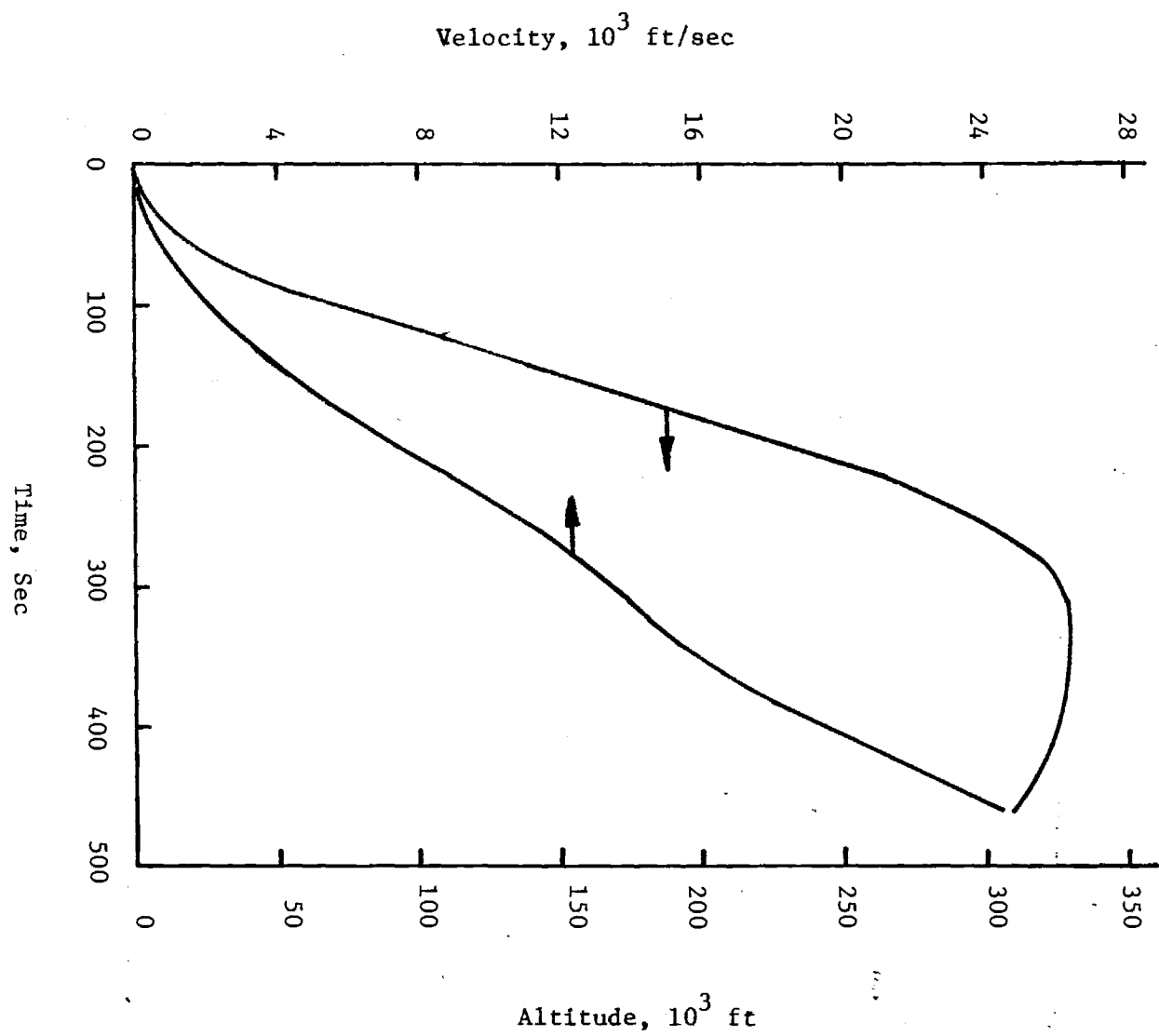


Fig. 9 Ascent Velocity and Altitude Profiles

\$QNML

ITAPE = +0

ICASE = +0

NTM = +4

TO = .00000000E+00

PHN = .00000000E+00

\$END

\$TUBE

DITBI = .25000000E+00

STBI = .10000000E+00

XL = .12000000E+02

RHOTBI = .16859200E+03

HZ = +9

NRTBI = +3

NTBS = +10

\$END

\$FLOW

MOOTI = .50000000E+03

TO = .65967000E+03

PO = .86400000E+04

\$END

\$FIN

NX = +5

SROOTI = .50000000E-01

HFNI = .60000000E+01

STIPI = .50000000E-01

RHOFNI = .16859200E+03

STAGX = .20000000E+02

VERTX = .10000000E+02

FIGURE 10(A) SYSTEM RESPONSE FOR ASCENT  
PROGRAM OUTPUT

\$END

\$PROTLR

NRMP = +3

RHOMPI = .16859200E+03

RHOMET = .50000000E+00

VELM = .65600000E+05

TAU = .36500000E+04

PKGB = .99000000E+00

ALPHA = .18800000E-09

BTA = .12130000E+01

GAMMA = .15000000E+01

PHI = .50000000E+00

THETA = .66666700E+00

ATK = .17500000E+01

AN = .10000000E+01

\$END

\$MANIFD

AMAN = .43000000E+02

\$END

\$RUNOPT

MSTOTR = +2

2 DTWRT = .9333000E-02  
TEND = .1270000E+00  
ALIMIT = .5000000E-04  
RLIMIT = .1000000E-04  
TI = .65967000E+03  
LIMWT = +25  
NCONV = +1  
LTI = +0  
LFLO = +1  
LTS = +0

SEND

FIGURE 10(A) CONTINUED

## INITIAL LINE CONDITIONS

\*\*\*\*\*

(ALL QUANTITIES ARE NORMALIZED)

PT.NO.	POSITION Z	PRESSURE P	VELOCITY W	FLUID TEMPERATURE T	WALL TEMPERATURE TWI
1	.000	1.000000	1.000000	1.000000	1.000000
2	.125	.999810	1.000000	1.000000	1.000000
3	.250	.999621	1.000001	1.000000	1.000000
4	.375	.999431	1.000001	1.000000	1.000000
5	.500	.999241	1.000001	1.000000	1.000000
6	.625	.999052	1.000001	1.000000	1.000000
7	.750	.998862	1.000001	1.000000	1.000000
8	.875	.998673	1.000002	1.000000	1.000000
9	1.000	.998483	1.000002	1.000000	1.000000

INLET PRESSURE  $P_0$  = 8640.000 LBF/SQ.FT  
 REF. VELOCITY  $W_0$  = .87128 FT/SFC  
 REF. TEMPERATURE  $T_{00}$  = 659.670 R

REYNOLDS NO = .30017+04  
 PRANDTL NO = 8.628066  
 DELTA = .001736  
 REL.PRESSURE IS .391534+04

INIT. NUSSLT NO.  $NU$  = .201856+02  
 WALL BIOT NO.  $BI$  = .485314-02

FIGURE 10(A) CONTINUED

## SYSTEM PARAMETERS

\*\*\*\*\*

TUBE LENGTH,  $XL$  = 12.000 FT  
 INTERNAL DIAMETER,  $D_{ITB}$  = .250 IN  
 WALL THICKNESS,  $STB$  = .100 IN  
 MATERIAL ALUMINUM  
 MASS (ALL TUBES),  $MTB$  = 15.4480 LBM  
 NUMBER OF TUBES,  $NTBS$  = 10

FIN HEIGHT,  $HFN$  = 6.000 IN  
 THICKNESS AT ROOT,  $S_{ROOT}$  = .050 IN  
 THICKNESS AT TIP,  $STIP$  = .050 IN  
 MATERIAL ALUMINUM  
 MASS (ALL FINS),  $MFN$  = 84.2960 LBM  
 NO. OF FIN SIDES RADIATING = 1

COOLANT FLUID IS SILICONE OIL  
 MASS (IN ALL TUBES),  $MFL$  = 19.1289 LBM

PROTECTION LAYER THICKNESS,  $SMP$  = .011 IN  
 MASS,  $MMP$  = 2.038 LBM  
 MATERIAL IS BERYLLIUM

TOTAL MASS (EXCL. MANIFLD.)  $MTOT$  = 120.9104 LBM

TOTAL AREA (SINGLE NORMAL PROJECTION) ,

ATOT = 124,5000 SQ FT

FIGURE 10(A) CONTINUED

ELAPSED TIME IS .0000 HR , RELATIVE TIME IS .0000 1 INTEGR. STEPS  
 ALTITUDE IS .00 FT , VELOCITY IS .00 FT/SEC , ATM. TEMPERATURE IS 518.67 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*  
 REFERENCE TEMPERATURE, T00 = 659.670 R ,  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT) , INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .3289+05 BTU/HR , INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 COND. FROM MANIFOLDS, CONDMF = .0000 BTU/HR , AERODYN. HEATING POWER, QCONV = -.1924+05 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.5213+05 BTU/HR

AXIAL RELATIVE RELATIVE TEMPERATURE OF FIN, T  
 DIST. RAD. HEAT  
 REJECTION  
 Z 0 DISTANCE NORMAL TO FLOW DIRECTION  
 X

Z	0	.00000	.25000	.50000	.75000	1.00000
.000	.8477	1.0000	1.0000	1.0000	1.0000	1.0000
.125	.8451	1.0000	1.0000	1.0000	1.0000	1.0000
.250	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.375	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.500	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.625	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.750	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.875	.8451	1.0000	1.0000	1.0000	1.0000	1.0000
1.000	.8477	1.0000	1.0000	1.0000	1.0000	1.0000

(EXIT)

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*  
 REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .276866+05 BTU/HR TOT. REJECTION DH = .500488+01 BTU/HR

AXIAL	PRESSURE	VELOCITY	FLUID	WALL	PROTECT.	ENTHALPY REJECTION	
DIST.	P	W	TEMPERATURES	TEMPERATURES	LAYER	PER UNIT TUBE	FRACTION
			TF	TWI	TMP	LENGTH	OF
						BTU/(HR FT)	TOTAL
.000	1.00000	1.0000	1.0000	1.0000	1.0000	.000000	.00000
.125	.99981	1.0000	1.0000	1.0000	1.0000	.000138	.02563
.250	.99962	1.0000	1.0000	1.0000	1.0000	.000242	.09431
.375	.99943	1.0000	1.0000	1.0000	1.0000	.000311	.19375
.500	.99924	1.0000	1.0000	1.0000	1.0000	.000380	.31625
.625	.99905	1.0000	1.0000	1.0000	1.0000	.000450	.46386
.750	.99886	1.0000	1.0000	1.0000	1.0000	.000484	.63147
.875	.99867	1.0000	1.0000	1.0000	1.0000	.000519	.80933
1.000	.99848	1.0000	1.0000	1.0000	1.0000	.000553	1.00000

(EXIT)

FIGURE 10(A) CONTINUED



ELAPSED TIME IS .0250 HR , RELATIVE TIME IS 6.5344 38 INTEGR. STEPS  
 ALTITUDE IS 56495.86 FT , VELOCITY IS 1449.87 FT/SEC , ATM. TEMPERATURE IS 389.97 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 659.670 R ,  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .2115+05 BTU/HR , INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 COND. FROM MANIFOLDS, CONDMF = .2443+04 BTU/HR , AERODYN. HEATING POWER, QCONV = -.1634+05 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.3076+05 BTU/HR

AXIAL DIST. Z	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	1.2195	1.0879	1.0879	1.0879	1.0879	1.0879
.125	.6981	.9517	.8626	.8047	.7764	.7689
.250	.3854	.9572	.8687	.8112	.7832	.7759
.375	.3856	.9550	.8680	.8110	.7832	.7758
.500	.3860	.9574	.8691	.8115	.7834	.7760
.625	.2817	.9560	.8704	.8141	.7866	.7793
.750	.3870	.9585	.8697	.8117	.7835	.7760
.875	.7019	.9489	.8620	.8044	.7762	.7687
1.000 (EXIT)	1.2263	1.0893	1.0893	1.0893	1.0893	1.0893

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .234041+05 BTU/HR TOT. REJECTION OH = .428260+04 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TF	WALL TEMP TWI	PROTECT. LAYER TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	1.0879	1.0879	-186.124290	.00000
.125	.99961	.9956	.9867	.9517	.9517	74.285834	-.11604
.250	.99962	.9928	.9871	.9572	.9572	63.325497	.30202
.375	.99942	.9913	.9760	.9550	.9550	44.420708	.52840
.500	.99923	.9896	.9815	.9574	.9574	50.998492	.75807
.625	.99904	.9891	.9689	.9560	.9560	27.315979	.98675
.750	.99885	.9881	.9807	.9585	.9585	46.887507	1.14491
.875	.99866	.9868	.9641	.9489	.9489	32.308490	1.38225
1.000 (EXIT)	.99847	.9970	.9754	1.0893	1.0893	-241.334110	1.00000

FIGURE 10(A) CONTINUED

ELAPSED TIME IS .0500 HR , RELATIVE TIME IS 13.0687 55 INTEGR. STEPS  
 ALTITUDE IS 194986.66 FT , VELOCITY IS 6199.57 FT/SEC , ATM. TEMPERATURE IS 462.40 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R ,  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .2358+05 BTU/HR , INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 COND. FROM MANIFOLDS, CONDMF = .2680+03 BTU/HR , AERODYN. HEATING POWER, QCONV = .2487+05 BTU/HR  
 ENERGY STORAGE RATE, STORG = .1333+05 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION X				
Z		.00000	.25000	.50000	.75000	1.00000
.000	.7697	.9783	.9783	.9783	.9783	.9783
.125	.6604	.9172	.9160	.9160	.9152	.9147
.250	.5928	.9214	.9231	.9249	.9252	.9250
.375	.5873	.9133	.9177	.9216	.9231	.9233
.500	.5825	.9130	.9182	.9223	.9238	.9240
.625	.5725	.9076	.9160	.9225	.9254	.9261
.750	.5751	.9070	.9147	.9204	.9227	.9231
.875	.5904	.8927	.9012	.9077	.9106	.9112
1.000 (EXIT)	.6184	.9315	.9315	.9315	.9315	.9315

FIGURE 10(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .159106+05 BTU/HR TOT. REJECTION OH = .117761+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMP TF	WALL TEMP TWI	PROTECT. LAYER TEMP TMP	ENTHALPY REJECTION	
						PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00030	1.0000	1.0000	.9783	.9783	42.947949	.00000
.125	.99981	.9927	.9814	.9172	.9172	126.966851	.14939
.250	.99961	.9874	.9750	.9214	.9214	106.037679	.33239
.375	.99942	.9827	.9603	.9133	.9133	92.991148	.47533
.500	.99923	.9784	.9561	.9130	.9130	85.266257	.61012
.625	.99904	.9748	.9429	.9076	.9076	69.837278	.72965
.750	.99885	.9714	.9416	.9070	.9070	68.399486	.83144
.875	.99866	.9677	.9290	.8927	.8927	71.936090	.93790
1.000 (EXIT)	.99847	.9672	.9252	.9315	.9315	-12.536525	1.00000

ELAPSED TIME IS .0750 HR , RELATIVE TIME IS 19.6031 80 INTEGR. STEPS  
 ALTITUDE IS 308492.96 FT , VELOCITY IS 11899.46 FT/SEC , ATM. TEMPERATURE IS 334.32 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 659.670 R ,  
 REF. RADIANT HEAT FLUX  
 PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .2231+05 BTU/HR , INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 COND. FROM MANIFOLDS, CONDMF = .2343+03 BTU/HR , AERODYN. HEATING POWER, QCONV = .6611+04 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.4995+04 BTU/HR

46

AXIAL DIST. Z	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.7313	.9671	.9671	.9671	.9671	.9671
.125	.6328	.9301	.9147	.9054	.9007	.8993
.250	.5703	.9322	.9199	.9128	.9093	.9084
.375	.5614	.9246	.9138	.9078	.9051	.9044
.500	.5533	.9224	.9127	.9075	.9051	.9046
.625	.5419	.9172	.9092	.9053	.9039	.9035
.750	.5395	.9146	.9068	.9030	.9015	.9012
.875	.5462	.9035	.8954	.8913	.8897	.8894
1.000	.5620	.9121	.9121	.9121	.9121	.9121
(EXIT)						

FIGURE 10(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LRF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .172146+05 BTU/HR TOT. REJECTION DH = .104721+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMP TMP	ENTHALPY REJECTION	
						PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9671	.9671	67.116564	.00000
.125	.99911	.9939	.9852	.9301	.9301	112.356394	.15252
.250	.99961	.9892	.9780	.9322	.9322	93.483206	.31570
.375	.99942	.9850	.9670	.9246	.9246	86.350813	.44887
.500	.99923	.9811	.9610	.9224	.9224	78.735278	.57788
.625	.99904	.9778	.9515	.9172	.9172	69.991269	.69352
.750	.99885	.9746	.9473	.9146	.9146	66.729493	.79852
.875	.99866	.9712	.9386	.9035	.9035	71.472905	.90460
1.000	.99847	.9690	.9329	.9121	.9121	42.267320	1.00000
(EXIT)							

ELAPSED TIME IS .1000 HR , RELATIVE TIME IS 26.1375 96 INTEGR. STEPS  
 ALTITUDE IS 330000.00 FT , VELOCITY IS 16499.06 FT/SEC , ATM. TEMPERATURE IS 376.50 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*  
 REFERENCE TEMPERATURE, T00 = 659.670 R ,  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .2110+05 BTU/HR , INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 COND. FROM MANIFOLDS, CONDMF = .2680+03 BTU/HR , AERODYN. HEATING POWER, QCONV = .8142+04 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.1412+04 BTU/HR

AXIAL DIST. Z	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.7192	.9634	.9634	.9634	.9634	.9634
.125	.6096	.9257	.9074	.8948	.8876	.8852
.250	.5396	.9264	.9112	.9010	.8951	.8933
.375	.5289	.9186	.9048	.8955	.8902	.8885
.500	.5189	.9153	.9026	.8941	.8892	.8876
.625	.5029	.9096	.8984	.8910	.8867	.8854
.750	.5014	.9060	.8952	.8880	.8838	.8825
.875	.5144	.8959	.8848	.8772	.8728	.8714
1.000 (EXIT)	.5421	.9050	.9050	.9050	.9050	.9050

FIGURE 10(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*  
 REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .164041+05 BTU/HR TOT. REJECTION DH = .112820+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMP	ENTHALPY REJECTION	
						PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9634	.9634	73.935921	.00000
.125	.99981	.9996	.9850	.9257	.9257	119.899873	.15022
.250	.99961	.9885	.9766	.9264	.9264	101.527077	.31116
.375	.99942	.9840	.9651	.9186	.9186	94.038617	.44474
.500	.99923	.9798	.9582	.9153	.9153	86.704713	.57414
.625	.99904	.9760	.9482	.9096	.9096	78.045426	.69177
.750	.99885	.9726	.9429	.9060	.9060	74.709270	.79958
.875	.99866	.9690	.9337	.8959	.8959	70.516509	.90673
1.000 (EXIT)	.99847	.9666	.9276	.9050	.9050	45.841238	1.00000

ELAPSED TIME IS .1250 HR , RELATIVE TIME IS 32.6718 108 INTEGR. STEPS  
 ALTITUDE IS 313506.32 FT , VELOCITY IS 23548.28 FT/SEC , ATM. TEMPERATURE IS 341.94 R

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R ,  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .2112+05 BTU/HR , INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 COND. FROM MANIFOLDS, CONDMF = .2058+03 BTU/HR , AERODYN. HEATING POWER, QCONV = .2673+05 BTU/HR  
 ENERGY STORAGE RATE, STORG = .1783+05 BTU/HR

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AXIAL RELATIVE RELATIVE TEMPERATURE OF FIN, T  
 DIST. RAD. HEAT REJECTION  
 2 0  
 DISTANCE NORMAL TO FLOW DIRECTION  
 X

		.00000	.25000	.50000	.75000	1.00000
.000	.7023	.9583	.9583	.9583	.9583	.9583
.125	.6094	.9225	.9108	.9011	.8948	.8926
.250	.5488	.9222	.9136	.9061	.9013	.8995
.375	.5362	.9141	.9067	.9002	.8959	.8943
.500	.5244	.9098	.9036	.8979	.8940	.8926
.625	.5097	.9035	.8988	.8941	.8909	.8897
.750	.5033	.8991	.8949	.8905	.8874	.8862
.875	.5052	.8892	.8849	.8803	.8771	.8759
1.000	.5154	.8951	.8951	.8951	.8951	.8951

FIGURE 10(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY H1 = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .156841+05 BTU/HR TOT. REJECTION DM = .120026+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMPERATURE TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9583	.9583	83.797215	.00000
.125	.99981	.9934	.9847	.9225	.9225	124.993712	.14961
.250	.99961	.9880	.9755	.9222	.9222	107.109274	.30702
.375	.99942	.9832	.9637	.9141	.9141	99.836724	.43992
.500	.99923	.9787	.9559	.9098	.9098	92.636692	.56879
.625	.99904	.9747	.9455	.9035	.9035	84.442323	.68700
.750	.99885	.9709	.9393	.8991	.8991	80.657710	.79626
.875	.99866	.9670	.9298	.8892	.8892	81.486121	.90382
1.000	.99847	.9642	.9229	.8951	.8951	55.786302	1.00000

(EXIT)

ELAPSED TIME IS .1270 HR , RELATIVE TIME IS 33.1959 110 INTEGR. STEPS  
 ALTITUDE IS 310980.00 FT , VELOCITY IS 24234.00 FT/SEC , ATM. TEMPERATURE IS 337.93 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION RTOT = .2123+05 BTU/HR, INCIDENT INFRARED FLUX, QIHED = .2000+01 BTU/(HR SQ FT)  
 CONU. FROM MANIFOLDS, CONDMF = .1911+03 BTU/HR, AERODYN. HEATING POWER, QCONV = .3037+05 BTU/HR  
 ENERGY STORAGE RATE, STORG = .2135+05 BTU/HR

AXIAL DIST. RELATIVE RAD. HEAT REJECTION  
 0  
 RELATIVE TEMPERATURE OF FIN, T  
 DISTANCE NORMAL TO FLOW DIRECTION  
 X

	0.00000	.25000	.50000	.75000	1.00000
.000	.7001	.9576	.9576	.9576	.9576
.125	.6115	.9226	.9122	.9032	.8973
.250	.5534	.9222	.9149	.9081	.9036
.375	.5406	.9140	.9081	.9022	.8982
.500	.5286	.9097	.9049	.8998	.8962
.625	.5145	.9034	.9000	.8960	.8930
.750	.5072	.8989	.8961	.8923	.8895
.875	.5066	.8891	.8862	.8823	.8793
1.000 (EXIT)	.5127	.8941	.8941	.8941	.8941

FIGURE 10(A) CONCLUDED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 840.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .156649+05 BTU/HR  
 INLET CURRENTLY HI = .276867+05 BTU/HR  
 TOT. REJECTION DH = .120218+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TW	PROTECT. LAYER TEMPERATURE TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9576	.9576	85.173653	.00000
.125	.99981	.9934	.9847	.9226	.9226	124.974236	.14983
.250	.99961	.9880	.9755	.9222	.9222	107.177940	.30672
.375	.99942	.9832	.9638	.9140	.9140	99.976870	.43945
.500	.99923	.9787	.9558	.9097	.9097	92.755548	.56813
.625	.99904	.9746	.9455	.9034	.9034	84.717443	.68622
.750	.99885	.9708	.9392	.8989	.8989	80.865383	.79553
.875	.99866	.9670	.9297	.8891	.8891	81.721740	.90304
1.000 (EXIT)	.99847	.9641	.9227	.8941	.8941	57.510110	1.00000

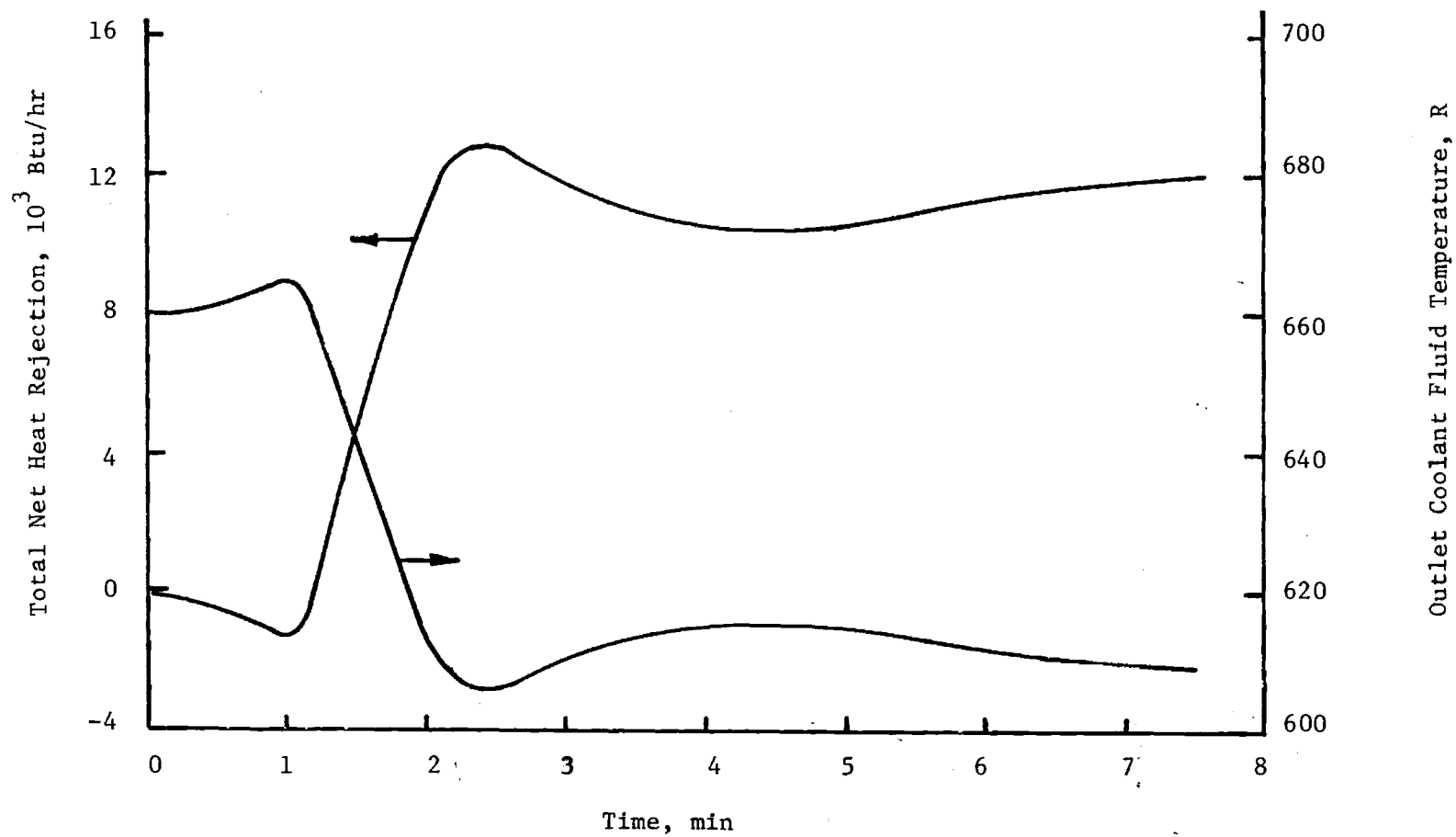


Fig. 10(b) System Response For Ascent—System Heat Rejection And Outlet Fluid Temperature

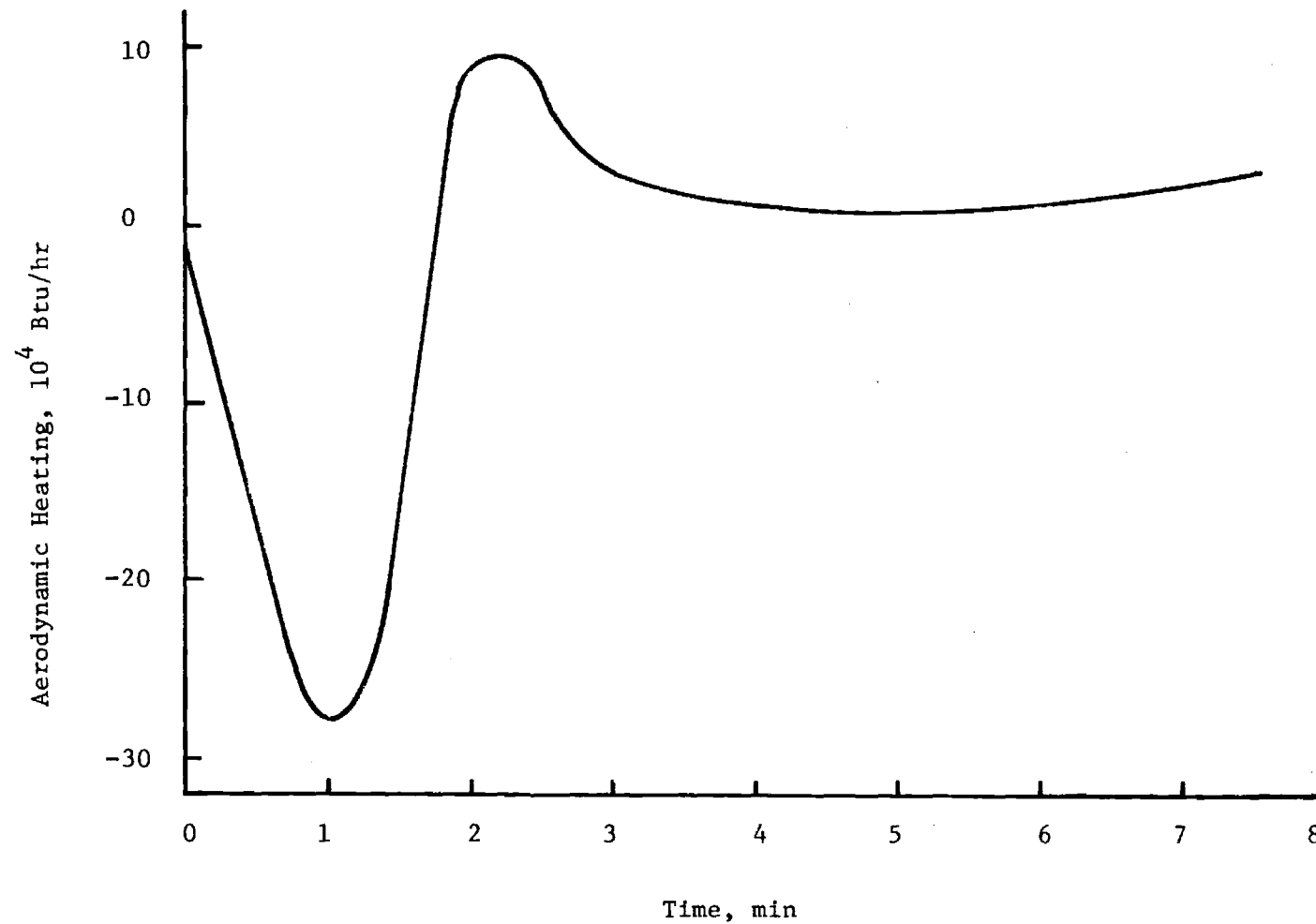


Fig. 11 Aerodynamic Heating For Ascent



b. Reentry

(i) Mission Description

The reentry altitude and velocity profiles used for the system output shown in Figures 13 and 14 are given in Figure 12.

Input parameters for reentry are identical to those given in the mission description for ascent except for IC0NV which is set equal to 2 to designate reentry. The system parameters for reentry are printed prior to the current system output shown in Figure 13a.

(ii) System Response

The response of the radiator system during reentry is summarized in Figures 13 and 14. Figure 13 (a) provides selected program output showing current system conditions at various times during the reentry phase of the mission. Figure 13 (b) is a plot of the total heat rejection from the radiator system and the temperature of the coolant fluid at the exit plane of the flow channel. Figure 14 is a graph of the aerodynamic heating of the system as a function of time during the reentry phase of the mission.

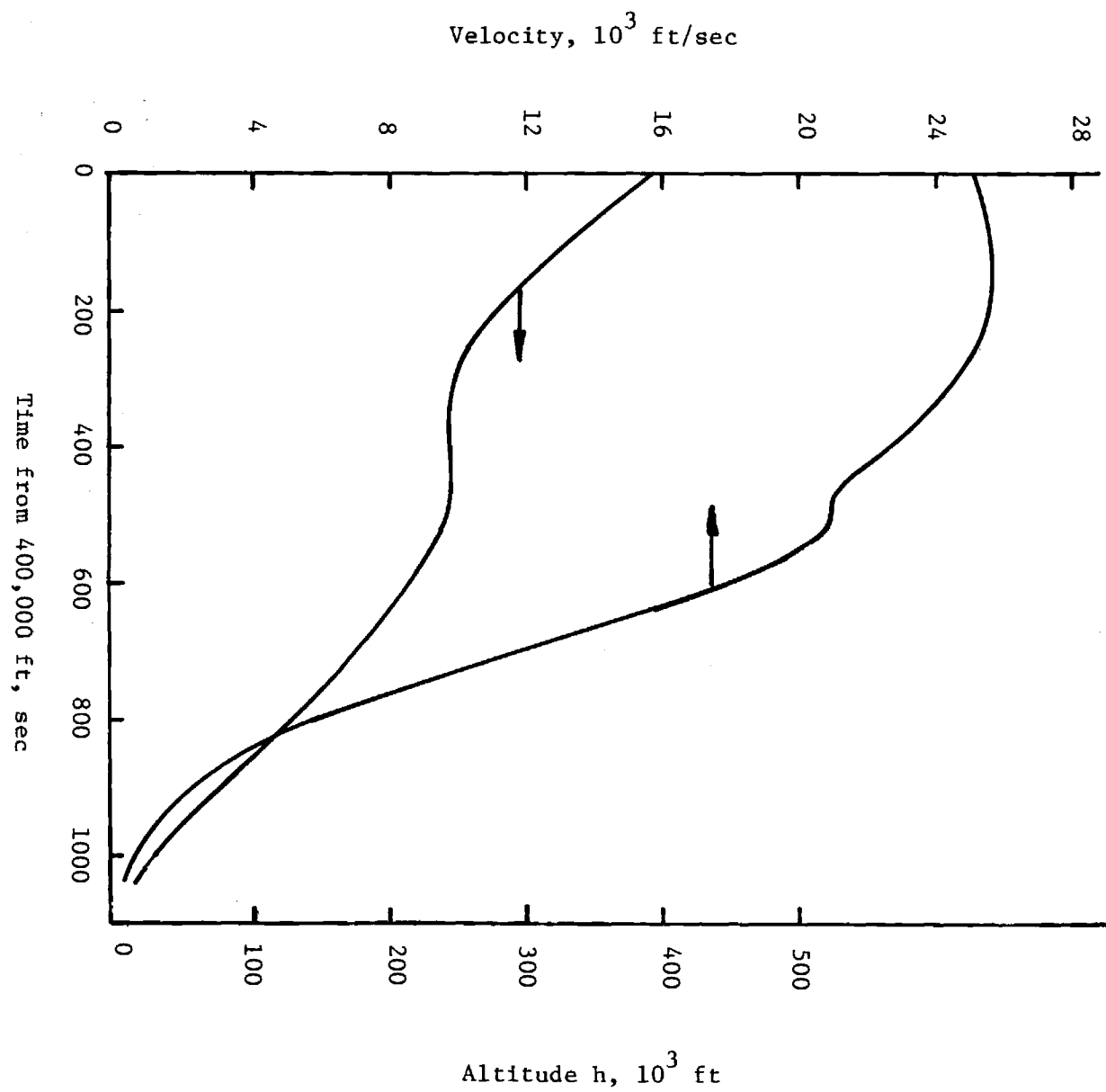


Fig. 12 Reentry Velocity and Altitude Profiles

02 SQNML  
 ITAPE = +0  
 ICASE = +0  
 NTM = +4  
 IQ = .00000000E+00  
 PHN = .00000000E+00

SEND  
 \$TUBE  
 OITBI = .25000000E+00  
 STBI = .10000000E+00  
 XL = .12000000E+02  
 RHOTBI = .16859200E+03  
 MZ = +9  
 NRTBI = +3  
 NTBS = +10

SEND  
 \$FLOW  
 MDOTI = .50000000E+03  
 IQ = .65967000E+03  
 PO = .86400000E+04

FIGURE 13(A) SYSTEM RESPONSE FOR REENTRY  
 PROGRAM OUTPUT

SEND  
 \$FIN  
 NX = +5  
 SROOTI = .50000000E-01  
 HFNT = .60000000E+01  
 STIPI = .50000000E-01  
 RHOFNI = .16859200E+03  
 STAGX = .20000000E+02  
 VERTX = .10000000E+02

SEND  
 \$PROILR  
 NRMP = +3  
 RHOMPI = .16859200E+03  
 RHOMET = .50000000E+00  
 VELM = .65600000E+05  
 TAU = .36500000E+04  
 PROB = .99000000E+00  
 ALPHA = .18800000E-09  
 BTA = .12130000E+01  
 GAMMA = .15000000E+01  
 PHI = .50000000E+00  
 THETA = .66666700E+00  
 ATK = .17500000E+01  
 AN = .10000000E+01

SEND  
 \$MANIED  
 AMAN = .43000000E+02

SEND  
 \$RUNOPT  
 MSTOTR = +2

DTWRTE	==	.16667000E-01
TEND	==	.29400000E+00
ALIMIT	==	.50000000E-04
RLIMIT	==	.10000000E-04
TI	==	.65967000E+03
LIMWRT	==	+25
NCONV	==	+2
LTT	==	+0
LFLD	==	+1
LTS	==	+0

SEND

FIGURE 13(A) CONTINUED

## INITIAL LINE CONDITIONS

\*\*\*\*\*

(ALL QUANTITIES ARE NORMALIZED)

PT.NO.	POSITION Z	PRESSURE P	VELOCITY W	FLUID TEMPERATURE T	WALL TEMPERATURE TWI
1	.000	1.000000	1.000000	1.000000	1.000000
2	.125	.999810	1.000000	1.000000	1.000000
3	.250	.999621	1.000001	1.000000	1.000000
4	.375	.999431	1.000001	1.000000	1.000000
5	.500	.999241	1.000001	1.000000	1.000000
6	.625	.999052	1.000001	1.000000	1.000000
7	.750	.998862	1.000001	1.000000	1.000000
8	.875	.998673	1.000002	1.000000	1.000000
9	1.000	.998483	1.000002	1.000000	1.000000

INLET PRESSURE P0 = 8640.000 LBF/SQ.FT

REF. VELOCITY W0 = .87128 FT/SEC

REF. TEMPERATURE T00 = 659.670 R

REYNOLDS NO = .30017+04

PRANDTL NO = 8.628066

DELTA = .001736

REL. PRESSURE IS .391534+04

INIT. NUSSELT NO. NU = .201856+02

WALL BIOT NO. BI = .485314+02

FIGURE 13(A) CONTINUED

## SYSTEM PARAMETERS

\*\*\*\*\*

TUBE LENGTH, XL = 12.000 FT  
 INTERNAL DIAMETER, DITB = .250 IN  
 WALL THICKNESS, STB = .100 IN  
 MATERIAL ALUMINUM  
 MASS (ALL TUBES), MTB = 15.4480 LBM  
 NUMBER OF TUBES, NTBS = 10

FIN HEIGHT, HFN = 6.000 IN  
 THICKNESS AT ROOT, SROOT = .050 IN  
 THICKNESS AT TIP, STIP = .050 IN  
 MATERIAL ALUMINUM  
 MASS (ALL FINS), MFN = 84.2960 LBM  
 NO. OF FIN SIDES RADIATING = 1

COOLANT FLUID IS SILICONE OIL  
 MASS (IN ALL TUBES), MFL = 19.1289 LBM

PROTECTION LAYER THICKNESS, SMP = .011 IN  
 MASS, MMP = 2.038 LBM  
 MATERIAL IS BERYLLIUM

TOTAL MASS (EXCL. MANIFLD.) MTOT = 120.9104 LBM

TOTAL AREA (SINGLE NORMAL PROJECTION) ,  
ATOT = 124.5000 SQ FT

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FIGURE 13(A) CONTINUED

ELAPSED TIME IS .0000 HR , RELATIVE TIME IS .0000 1 INTEGR. STEPS  
 ALTITUDE IS 394000.00 FT , VELOCITY IS 25400.00 FT/SEC , ATM. TEMPERATURE IS 661.41 R

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX  
 PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT)  
 TOT. RADIANT REJECTION QTOT = .3289+05 BTU/HR  
 CONU. FROM MANIFOLDS, CONDMF = .0000 BTU/HR  
 INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 AERODYN. HEATING POWER, QCONV = .6237+04 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.2665+05 BTU/HR

AXIAL DIST. Z	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.8477	1.0000	1.0000	1.0000	1.0000	1.0000
.125	.8451	1.0000	1.0000	1.0000	1.0000	1.0000
.250	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.375	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.500	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.625	.8430	1.0000	1.0000	1.0000	1.0000	1.0000
.750	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.875	.8451	1.0000	1.0000	1.0000	1.0000	1.0000
1.000	.8477	1.0000	1.0000	1.0000	1.0000	1.0000
(EXIT)						

FIGURE 13(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=10 HO = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .276866+05 BTU/HR  
 INLET CURRENTLY HI = .276867+05 BTU/HR  
 TOT. REJECTION OH = .500488-01 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID		PROTECT. LAYER	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR.FT)	FRACTION OF TOTAL
			TEMPERATURE TF	WALL TEMP TW	TEMP TMP		
.000	1.00000	1.0000	1.0000	1.0000	1.0000	.000000	.000000
.125	.99981	1.0000	1.0000	1.0000	1.0000	.000138	.02563
.250	.99962	1.0000	1.0000	1.0000	1.0000	.000242	.09431
.375	.99943	1.0000	1.0000	1.0000	1.0000	.000311	.19375
.500	.99924	1.0000	1.0000	1.0000	1.0000	.000380	.31625
.625	.99905	1.0000	1.0000	1.0000	1.0000	.000450	.46386
.750	.99886	1.0000	1.0000	1.0000	1.0000	.000484	.63147
.875	.99867	1.0000	1.0000	1.0000	1.0000	.000519	.80933
1.000	.99848	1.0000	1.0000	1.0000	1.0000	.000553	1.00000
(EXIT)							

DTWRT	=	.16667000E-01
TEND	=	.29400000E+00
ALIMIT	=	.50000000E-04
RLIMIT	=	.10000000E-04
TI	=	.65967000E+03
LIMWRT	=	+25
NCONV	=	+2
LTT	=	+0
LFLD	=	+1
LTS	=	+0

SEND

FIGURE 13(A) CONTINUED



# INITIAL LINE CONDITIONS

\*\*\*\*\*

(ALL QUANTITIES ARE NORMALIZED)

PT.NO.	POSITION Z	PRESSURE P	VELOCITY W	FLUID TEMPERATURE T	WALL TEMPERATURE TWI
1	.000	1.000000	1.000000	1.000000	1.000000
2	.125	.999810	1.000000	1.000000	1.000000
3	.250	.999621	1.000001	1.000000	1.000000
4	.375	.999431	1.000001	1.000000	1.000000
5	.500	.999241	1.000001	1.000000	1.000000
6	.625	.999052	1.000001	1.000000	1.000000
7	.750	.998862	1.000001	1.000000	1.000000
8	.875	.998673	1.000002	1.000000	1.000000
9	1.000	.998483	1.000002	1.000000	1.000000

INLET PRESSURE  $P_0 = 8640.000$  LBF/SQ.FT  
 REF. VELOCITY  $W_0 = .87128$  FT/SEC  
 REF. TEMPERATURE  $T_{00} = 659.670$  R

REYNOLDS NO  $= .30017+04$   
 PRANDTL NO  $= 8.628066$   
 DELTA  $= .001736$   
 REL. PRESSURE IS  $.391534+04$

INIT. NUSSELT NO.  $NU = .201856+02$   
 WALL BIOT NO.  $BI = .485314-02$

FIGURE 13(A) CONTINUED

## SYSTEM PARAMETERS

\*\*\*\*\*

TUBE LENGTH,  $XL = 12.000$  FT  
 INTERNAL DIAMETER,  $D_{ITB} = .250$  IN  
 WALL THICKNESS,  $STB = .100$  IN  
 MATERIAL ALUMINUM  
 MASS (ALL TUBES),  $MTB = 15.4480$  LBM  
 NUMBER OF TUBES,  $NTBS = 10$

FIN HEIGHT,  $HFN = 6.000$  IN  
 THICKNESS AT ROOT,  $SROOT = .050$  IN  
 THICKNESS AT TIP,  $STIP = .050$  IN  
 MATERIAL ALUMINUM  
 MASS (ALL FINS),  $MFN = 84.2960$  LBM  
 NO. OF FIN SIDES RADIATING  $= 1$

COOLANT FLUID IS SILICONE OIL  
 MASS (IN ALL TUBES),  $MFL = 19.1289$  LBM

PROTECTION LAYER THICKNESS,  $SMP = .011$  IN  
 MASS,  $MMP = 2.038$  LBM  
 MATERIAL IS BERYLLIUM

TOTAL MASS (EXCL. MANIFLD.)  $MTOT = 120.9104$  LBM

TOTAL AREA (SINGLE NORMAL PROJECTION) ,  
ATOT = 124,5000 SQ FT

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FIGURE 13(A) CONTINUED

ELAPSED TIME IS .0000 HR , RELATIVE TIME IS .0000 1 INTEGR. STEPS  
 ALTITUDE IS 394000.00 FT , VELOCITY IS 25400.00 FT/SEC , ATM. TEMPERATURE IS 661.41 R

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R ,  
 REF. RADIANT HEAT FLUX  
 PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .3289+05 BTU/HR , INCIDENT INERARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 CONU. FROM MANIFOLDS, CONDMF = .0000 BTU/HR , AERODYN. HEATING POWER, QCONV = .6237+04 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.2665+05 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION	RELATIVE TEMPERATURE OF FIN, T				
Z	Q	DISTANCE NORMAL TO FLOW DIRECTION				
		X				
		.00000	.25000	.50000	.75000	1.00000
.000	.8477	1.0000	1.0000	1.0000	1.0000	1.0000
.125	.8451	1.0000	1.0000	1.0000	1.0000	1.0000
.250	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.375	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.500	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.625	.8430	1.0000	1.0000	1.0000	1.0000	1.0000
.750	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.875	.8451	1.0000	1.0000	1.0000	1.0000	1.0000
1.000	.8477	1.0000	1.0000	1.0000	1.0000	1.0000
(EXIT)						

FIGURE 13(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .276866+05 BTU/HR TOT. REJECTION DH = .500488+01 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMP TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR.FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	1.0000	1.0000	.000000	.00000
.125	.99981	1.0000	1.0000	1.0000	1.0000	.000138	.02563
.250	.99962	1.0000	1.0000	1.0000	1.0000	.000242	.09431
.375	.99943	1.0000	1.0000	1.0000	1.0000	.000311	.19375
.500	.99924	1.0000	1.0000	1.0000	1.0000	.000380	.31625
.625	.99905	1.0000	1.0000	1.0000	1.0000	.000450	.46386
.750	.99886	1.0000	1.0000	1.0000	1.0000	.000484	.63147
.875	.99867	1.0000	1.0000	1.0000	1.0000	.000519	.80933
1.000	.99848	1.0000	1.0000	1.0000	1.0000	.000553	1.00000
(EXIT)							

ELAPSED TIME IS .0500 HR , RELATIVE TIME IS 13.0695 36 INTEGR. STEPS  
 ALTITUDE IS 292998.55 FT , VELOCITY IS 25799.98 FT/SEC , ATM. TEMPERATURE IS 325.17 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT)  
 TOT. RADIANT REJECTION QTOT = .2773+05 BTU/HR  
 COND. FROM MANIFOLDS, CONDMF = -.9819+02 BTU/HR  
 INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 AERODYN. HEATING POWER, QCONV = .5638+05 BTU/HR  
 ENERGY STORAGE RATE, STORG = .3510+05 BTU/HR

AXIAL DIST. Z RELATIVE RAD. HEAT REJECTION Q RELATIVE TEMPERATURE OF FIN, T  
 DISTANCE NORMAL TO FLOW DIRECTION X

Z	Q	.00000	.25000	.50000	.75000	1.00000
.000	.7159	.9625	.9625	.9625	.9625	.9625
.125	.7255	.9584	.9678	.9711	.9720	.9719
.250	.7293	.9556	.9662	.9703	.9716	.9717
.375	.7242	.9527	.9643	.9690	.9707	.9709
.500	.7198	.9500	.9626	.9679	.9699	.9702
.625	.7259	.9478	.9610	.9668	.9691	.9694
.750	.7126	.9458	.9598	.9662	.9687	.9691
.875	.6797	.9441	.9589	.9657	.9684	.9690
1.000 (EXIT)	.6273	.9345	.9345	.9345	.9345	.9345

FIGURE 13(A) CONTINUED

## FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=TO H0 = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .211383+05 BTU/HR  
 INLET CURRENTLY HI = .276867+05 BTU/HR  
 TOT. REJECTION DH = .654839+04 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TF	WALL TEMPERATURES TWI	PROTECT. LAYER TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9625	.9625	80.776558	.00000
.125	.99981	.9963	.9925	.9584	.9584	73.419073	.17633
.250	.99962	.9931	.9856	.9556	.9556	64.706697	.33361
.375	.99943	.9902	.9799	.9527	.9527	58.594118	.47323
.500	.99924	.9877	.9742	.9500	.9500	51.918591	.59957
.625	.99905	.9854	.9698	.9478	.9478	47.317073	.71200
.750	.99886	.9834	.9649	.9458	.9458	41.271317	.81355
.875	.99867	.9816	.9619	.9441	.9441	36.321868	.90322
1.000 (EXIT)	.99848	.9794	.9578	.9345	.9345	50.009891	1.00000

ELAPSED TIME IS .1000 HR , RELATIVE TIME IS 26.1390 60 INTEGR. STEPS  
 ALTITUDE IS 245000.71 FT , VELOCITY IS 23799.82 FT/SEC , ATM. TEMPERATURE IS 362.54 R

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX  
 PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .5506+05 BTU/HR, INCIDENT INFRARED FLUX, QIKED = .2000+01 BTU/(HR SQ FT)  
 CONJ. FROM MANIFOLDS, CONDMF = -.1573+03 BTU/HR, AERODYN. HEATING POWER, QCONV = .1775+06 BTU/HR  
 ENERGY STORAGE RATE, STORR = .1193+06 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
Z		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.6299	.9354	.9354	.9354	.9354	.9354
.125	1.2481	1.0371	1.1389	1.2022	1.2359	1.2459
.250	1.6155	1.0278	1.1252	1.1849	1.2163	1.2254
.375	1.6061	1.0297	1.1254	1.1843	1.2153	1.2243
.500	1.5974	1.0254	1.1221	1.1816	1.2130	1.2222
.625	1.7098	1.0252	1.1178	1.1745	1.2043	1.2130
.750	1.5816	1.0219	1.1189	1.1789	1.2106	1.2200
.875	1.2127	1.0341	1.1332	1.1958	1.2296	1.2398
1.000 (EXIT)	.6032	.9265	.9265	.9265	.9265	.9265

FIGURE 13(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=10, HQ = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .307355+05 BTU/HR TOT. REJECTION DH = -.304877+04 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMP TF	WALL TEMP TW	PROTECT. LAYER TEMP TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9354	.9354	160.289324	.00000
.125	.99981	1.0040	1.0125	1.0371	1.0371	-61.108924	-.30040
.250	.99962	1.0060	1.0110	1.0278	1.0278	-41.625156	.64967
.375	.99944	1.0070	1.0198	1.0297	1.0297	-24.422209	1.00147
.500	.99925	1.0081	1.0143	1.0254	1.0254	-27.635271	1.36675
.625	.99906	1.0079	1.0242	1.0252	1.0252	-2.552186	1.66864
.750	.99887	1.0084	1.0127	1.0219	1.0219	-23.023775	1.76029
.875	.99868	1.0095	1.0268	1.0341	1.0341	-18.077758	2.15741
1.000 (EXIT)	.99850	1.0004	1.0171	.9265	.9265	224.826389	1.00000

ELAPSED TIME IS .1500 HR , RELATIVE TIME IS 39.2085 86 INTEGR. STEPS  
 ALTITUDE IS 234996.24 FT , VELOCITY IS 20199.73 FT/SEC , ATM. TEMPERATURE IS 384.00 R

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .6486+05 BTU/HR , INCIDENT INFRARED FLUX, QIREN = .2000+01 BTU/(HR SQ FT)  
 COND. FROM MANIFOLDS, CONDMF = .4866+03 BTU/HR , AERODYN. HEATING POWER, QCONV = .1582+06 BTU/HR  
 ENERGY STORAGE RATE, STORG = .8051+05 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION X				
Z		.00000	.25000	.50000	.75000	1.00000
.000	.7198	.9636	.9636	.9636	.9636	.9636
.125	1.4150	1.0926	1.1935	1.2578	1.2935	1.3045
.250	1.8367	1.0752	1.1646	1.2196	1.2491	1.2579
.375	1.8485	1.0866	1.1724	1.2249	1.2530	1.2614
.500	1.8592	1.0847	1.1701	1.2224	1.2504	1.2587
.625	1.9854	1.0895	1.1675	1.2141	1.2385	1.2456
.750	1.8776	1.0901	1.1738	1.2249	1.2523	1.2605
.875	1.5358	1.1232	1.2130	1.2699	1.3012	1.3108
1.000 (EXIT)	.9599	1.0290	1.0290	1.0290	1.0290	1.0290

FIGURE 13(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .410317+05 BTU/HR TOT. REJECTION DH = -.133450+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID WALL TEMPERATURES		PROTECT. LAYER TMP	ENTHALPY REJECTION	
			TF	TWI		PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9636	.9636	99.048074	.00000
.125	.99982	1.0106	1.0293	1.0926	1.0926	-172.390614	.11927
.250	.99963	1.0169	1.0327	1.0752	1.0752	-115.685914	.35437
.375	.99944	1.0217	1.0514	1.0866	1.0866	-95.888686	.48609
.500	.99926	1.0265	1.0507	1.0847	1.0847	-92.660016	.62721
.625	.99907	1.0295	1.0694	1.0895	1.0895	-54.798841	.74575
.750	.99888	1.0332	1.0604	1.0901	1.0901	-80.840606	.83365
.875	.99869	1.0390	1.0841	1.1232	1.1232	-106.374774	.97867
1.000 (EXIT)	.99850	1.0341	1.0833	1.0290	1.0290	147.963610	1.00000

ELAPSED TIME IS .2000 HR , RELATIVE TIME IS 52.2780 112 INTEGR. STEPS  
 ALTITUDE IS 171993.57 FT , VELOCITY IS 10399.29 FT/SEC , ATM. TEMPERATURE IS 487.17 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT)  
 TOT. RADIANT REJECTION QTOT = .7602+05 BTU/HR  
 COND. FROM MANIFOLDS, CONDMF = .6873+03 BTU/HR  
 INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 AERODYN. HEATING POWER, QCONV = .1290+06 BTU/HR  
 ENERGY STORAGE RATE, STORG = .3334+05 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
Z		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.7486	.9722	.9722	.9722	.9722	.9722
.125	1.6110	1.1342	1.2450	1.3218	1.3667	1.3813
.250	2.1382	1.1092	1.2023	1.2645	1.2997	1.3108
.375	2.1632	1.1263	1.2145	1.2735	1.3059	1.3174
.500	2.1871	1.1246	1.2119	1.2703	1.3035	1.3139
.625	2.3489	1.1324	1.2094	1.2598	1.2879	1.2966
.750	2.2311	1.1341	1.2190	1.2759	1.3083	1.3186
.875	1.8337	1.1830	1.2782	1.3448	1.3840	1.3968
1.000	1.1567	1.0745	1.0745	1.0745	1.0745	1.0745
(EXIT)						

FIGURE 13(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .479690+05 BTU/HR  
 INLET CURRENTLY HI = .276867+05 BTU/HR  
 TOT. REJECTION DH = -.202823+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMP TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9722	.9722	81.004254	.00000
.125	.99982	1.0161	1.0436	1.1342	1.1342	-263.760815	.13900
.250	.99963	1.0255	1.0499	1.1092	1.1092	-172.715717	.35778
.375	.99945	1.0330	1.0764	1.1263	1.1263	-145.424294	.48221
.500	.99926	1.0403	1.0770	1.1246	1.1246	-138.439034	.61517
.625	.99907	1.0450	1.1030	1.1324	1.1324	-85.539244	.72615
.750	.99889	1.0508	1.0923	1.1341	1.1341	-121.559476	.81088
.875	.99870	1.0600	1.1250	1.1830	1.1830	-168.841410	.94894
1.000	.99851	1.0553	1.1270	1.0745	1.0745	152.930897	.00000
(EXIT)							

ELAPSED TIME IS .2500 HR , RELATIVE TIME IS 65.3475 138 INTEGR. STEPS  
 ALTITUDE IS 74990.18 FT , VELOCITY IS 2199.64 FT/SEC , ATM. TEMPERATURE IS 394.97 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*  
 REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT)  
 TOT. RADIANT REJECTION QTOT = .6239+05 BTU/HR  
 COND. FROM MANIFOLDS, CONDMF = .1221+04 BTU/HR  
 INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 AERODYN. HEATING POWER, QCONV = -.2252+05 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.1037+06 BTU/HR

AXIAL RELATIVE RELATIVE TEMPERATURE OF FIN, T  
 DIST. RAD. HEAT REJECTION  
 Z Q DISTANCE NORMAL TO FLOW DIRECTION  
 X

Z	Q	.00000	.25000	.50000	.75000	1.00000
.000	.7413	.9700	.9700	.9700	.9700	.9700
.125	1.3374	1.1293	1.1889	1.2386	1.2700	1.2810
.250	1.7043	1.1026	1.1505	1.1899	1.2143	1.2227
.375	1.7279	1.1216	1.1633	1.1992	1.2217	1.2295
.500	1.7503	1.1191	1.1605	1.1961	1.2186	1.2263
.625	1.8472	1.1295	1.1613	1.1902	1.2035	1.2149
.750	1.7916	1.1300	1.1681	1.2021	1.2237	1.2313
.875	1.5837	1.1812	1.2224	1.2614	1.2871	1.2965
1.000	1.2233	1.0887	1.0887	1.0887	1.0887	1.0887

FIGURE 13(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*  
 REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .476853+05 BTU/HR  
 INLET CURRENTLY HI = .276867+05 BTU/HR  
 TOT. REJECTION DH = -.199986+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TE	WALL TEMPERATURES TWI	PROTECT. LAYER TEMPERATURES TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9700	.9700	86.688877	.00000
.125	.99982	1.0156	1.0433	1.1293	1.1293	-248.916988	.13119
.250	.99963	1.0244	1.0473	1.1026	1.1026	-160.148674	.34140
.375	.99945	1.0316	1.0739	1.1216	1.1216	-138.130276	.45934
.500	.99926	1.0386	1.0735	1.1191	1.1191	-132.202616	.58956
.625	.99907	1.0434	1.0996	1.1295	1.1295	-86.335973	.69966
.750	.99889	1.0491	1.0895	1.1300	1.1300	-117.345061	.78642
.875	.99870	1.0585	1.1210	1.1812	1.1812	-174.303867	.92608
1.000	.99851	1.0560	1.1255	1.0887	1.0887	106.599895	1.00000

(EXIT)



ELAPSED TIME IS .2667 HR , RELATIVE TIME IS 69.7040 148 INTEGR. STEPS  
 ALTITUDE IS 48989.54 FT , VELOCITY IS 1199.81 FT/SEC , ATM. TEMPERATURE IS 389.97 R

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .3745+05 BTU/HR, INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
 CONV. FROM MANIFOLDS, CONDMF = .1843+04 BTU/HR, AERODYN. HEATING POWER, QCONV = -.1705+06 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.2212+06 BTU/HR

AXIAL DIST. Z	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.8631	1.0041	1.0041	1.0041	1.0041	1.0041
.125	.8816	1.0874	1.0392	1.0268	1.0269	1.0292
.250	.8987	1.0677	1.0163	1.0006	.9982	.9996
.375	.9144	1.0850	1.0277	1.0084	1.0042	1.0050
.500	.9291	1.0834	1.0259	1.0064	1.0022	1.0031
.625	.8896	1.0947	1.0311	1.0076	1.0010	1.0010
.750	.9553	1.0951	1.0334	1.0117	1.0065	1.0070
.875	1.1262	1.1330	1.0672	1.0442	1.0398	1.0396
1.000 (EXIT)	1.4023	1.1242	1.1242	1.1242	1.1242	1.1242

FIGURE 13(A) CONCLUDED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .427089+05 BTU/HR TOT. REJECTION DH = -.150223+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID WALL PROTECT. LAYER			ENTHALPY REJECTION	
			TEMPERATURES			PER UNIT TUBE LENGTH BTU/(HR.FT)	FRACTION OF TOTAL
			TF	TWI	TMP		
.000	1.00000	1.0000	1.0000	1.0041	1.0041	-11.239679	.00000
.125	.99982	1.0103	1.0296	1.0874	1.0874	-156.783590	.13991
.250	.99963	1.0160	1.0308	1.0677	1.0677	-99.922624	.30879
.375	.99944	1.0212	1.0497	1.0850	1.0850	-95.677561	.41267
.500	.99926	1.0262	1.0508	1.0834	1.0834	-88.377534	.53137
.625	.99907	1.0306	1.0683	1.0947	1.0947	-71.594559	.63305
.750	.99888	1.0348	1.0667	1.0951	1.0951	-77.058642	.72233
.875	.99869	1.0421	1.0839	1.1330	1.1330	-133.142769	.84743
1.000 (EXIT)	.99850	1.0474	1.0955	1.1242	1.1242	-78.039950	1.00000

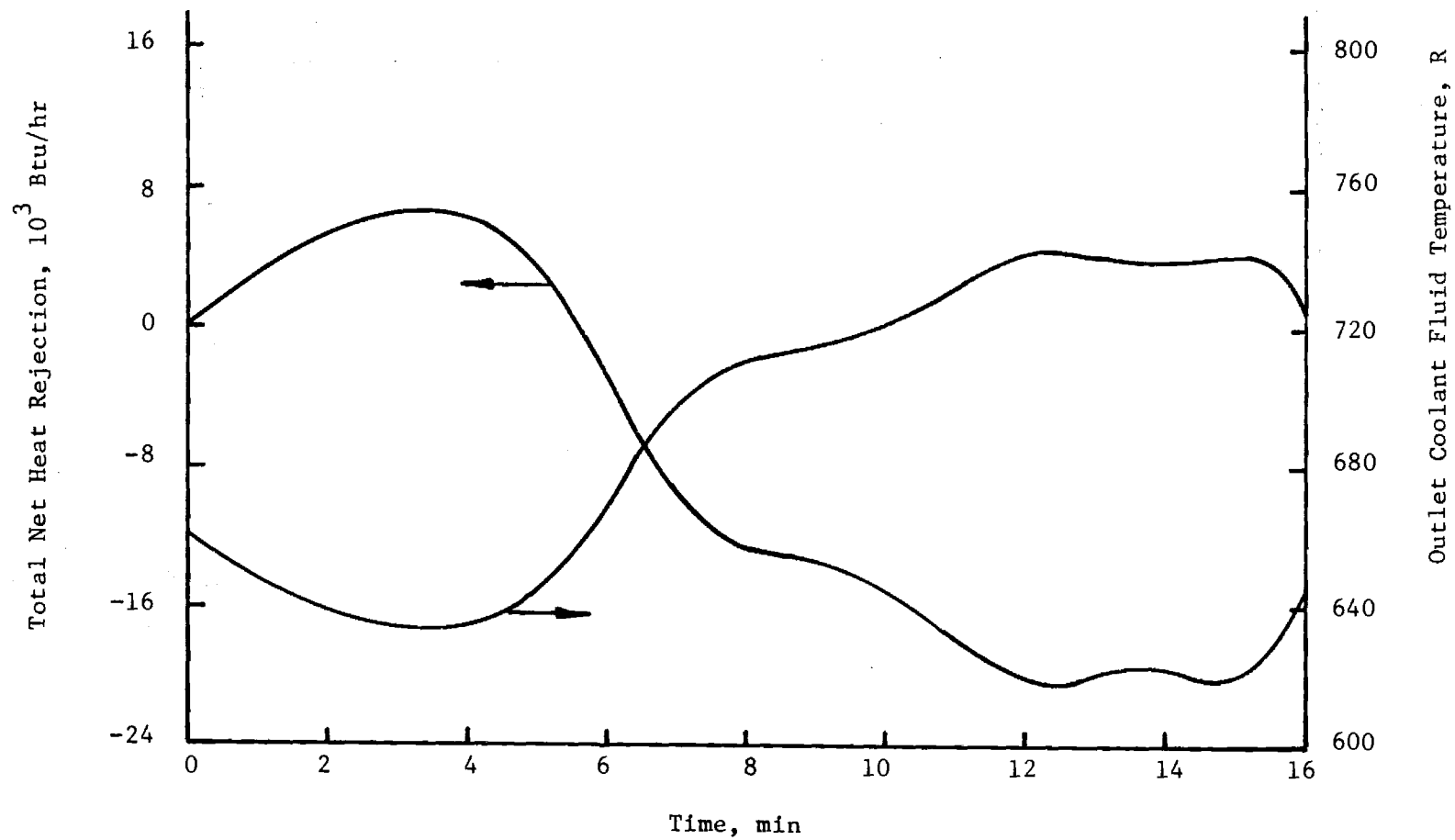


Fig. 13(b) System Response For Reentry-Heat Rejection And Outlet Fluid Temperature

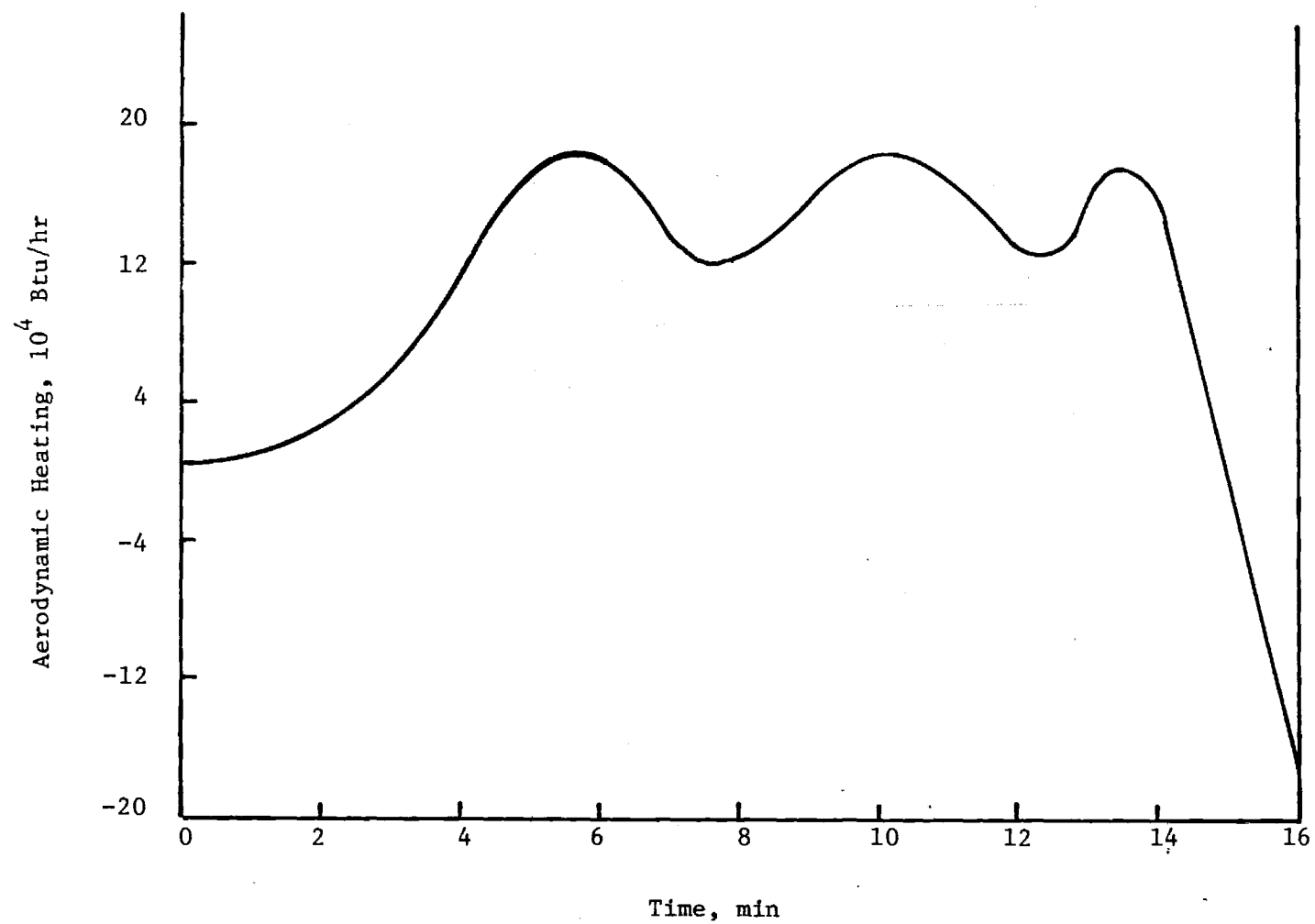


Fig.14 Aerodynamic Heating For Reentry

### c. Orbital Mission

#### (i) Mission Description

Transient system performance is predicted in Figure 16 for a typical space base orbital mission. The mission is described by a 270 nautical mile circular orbit inclined at  $55^\circ$ . Incident flux values for solar irradiation, earth albedo and earth irradiation were supplied by a previously executed MRI program with the flux data being transferred from magnetic tape.

Incident radiant flux values for a typical fin element are plotted as a function of time into orbit in Figure 15. Other input parameters which specify system geometry, material properties and fluid flow conditions are identical to those used for the ascent and reentry phases that were previously reported in sections 5a and 5b.

#### (ii) System Response

The response of the radiator system during the first twenty minutes of the orbit is summarized in Figures 16(a) and 16(b). Input parameters and a selected number of output listings of current radiator conditions are shown in Figure 16(a). Figure 16(b) graphically summarizes the outlet coolant fluid temperature and the total net heat rejection from the radiator for the same period during orbit.

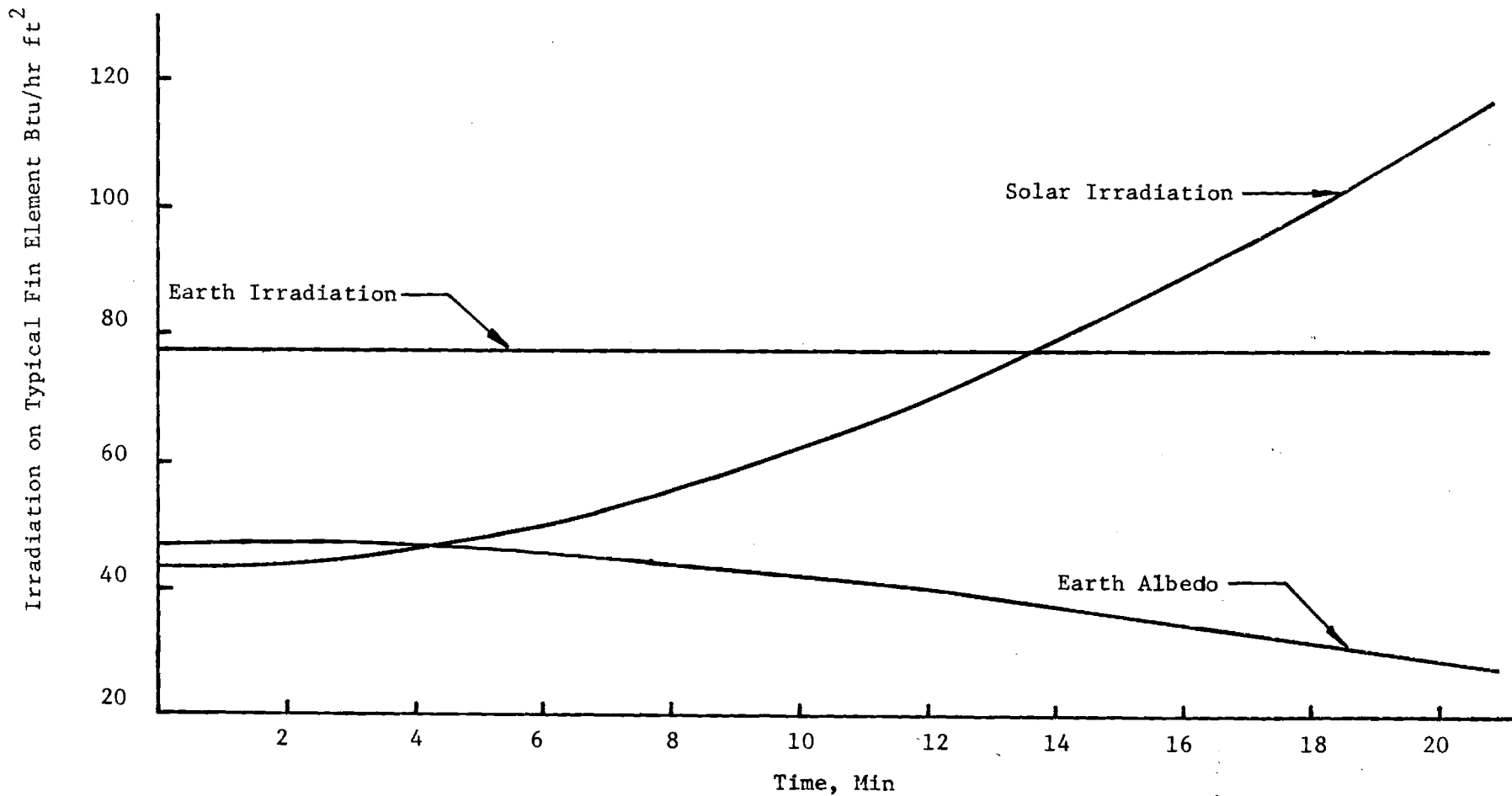


Fig. 15 Orbital Incident Fluxes

```

SQNML
ITAPE  = +3
ICASE  = +1
NTM    = +73
TO     = .00000000E+00
PHN    = .00000000E+00

```

```

SEND
STUBE
GITBI  = .25000000E+00
STBI   = .10000000E+00
XL     = .12000000E+02
RHOTBI = .16859200E+03
MZ     = +9
NKTBI  = +3
NTBS   = +10

```

```

SEND
$FLOW
MDOTI  = .50000000E+03
TO     = .65967000E+03
PO     = .86400000E+04

```

FIGURE 16(A) SYSTEM RESPONSE FOR ORBITAL MISSION  
PROGRAM OUTPUT

```

SEND
$FIN
NX      = +5
SROOTI  = .50000000E-01
RFNI    = .60000000E+01
STPI    = .50000000E-01
RHOFNI  = .16859200E+03
STAGX   = .20000000E+02
VERTX   = .10000000E+02

```

```

SEND
$PROTLR
NKMP    = +3
RHOMPI  = .16859200E+03
RHOMET  = .50000000E+00
VELM    = .65600000E+05
TAU     = .36500000E+04
PROB    = .99000000E+00
ALPHA   = .18800000E-09
BTA     = .12130000E+01
GAMMA   = .15000000E+01
PHI     = .50000000E+00
THETA   = .66666700E+00
ATK     = .17500000E+01
AN      = .10000000E+01

```

```

SEND
$MANIFD
AMAN    = .43000000E+02

```

```

SEND
$RUNOPT
MSTOTR  = +2

```

02 DTWRITE = .35000000E-01  
TEND = .70000000E+00  
ALIMIT = .50000000E-04  
RLIMIT = .10000000E-04  
TI = .65967000E+03  
LIMWT = +25  
NCONV = +0  
LTT = +0  
LFLD = +1  
LTS = +0

FIGURE 16(A) CONTINUED

SEND

## INITIAL LINE CONDITIONS

\*\*\*\*\*

(ALL QUANTITIES ARE NORMALIZED)

PT.NO.	POSITION Z	PRESSURE P	VELOCITY W	FLUID TEMPERATURE T	WALL TEMPERATURE TWI
1	.000	1.000000	1.000000	1.000000	1.000000
2	.125	.999810	1.000000	1.000000	1.000000
3	.250	.999621	1.000001	1.000000	1.000000
4	.375	.999431	1.000001	1.000000	1.000000
5	.500	.999241	1.000001	1.000000	1.000000
6	.625	.999052	1.000001	1.000000	1.000000
7	.750	.998862	1.000001	1.000000	1.000000
8	.875	.998673	1.000002	1.000000	1.000000
9	1.000	.998483	1.000002	1.000000	1.000000

INLET PRESSURE P0 = 8640.000 LBF/SQ.FT  
 REF. VELOCITY W0 = .87128 FT/SEC  
 REF. TEMPERATURE T00 = 659.670 R

REYNOLDS NO = .30017+04  
 PRANDTL NO = 8.628066  
 DELTA = .001736  
 REL.PRESSURE IS .391534+04

FIGURE 16(A) CONTINUED

INIT. NUSSELT NO. NU = .201856+02  
 WALL BIOT NO. BI = .485314-02

## SYSTEM PARAMETERS

\*\*\*\*\*

TUBE LENGTH, XL = 12.000 FT  
 INTERNAL DIAMETER, DITB = .250 IN  
 WALL THICKNESS, STB = .100 IN  
 MATERIAL ALUMINUM  
 MASS (ALL TUBES), MTB = 15.4480 LBM  
 NUMBER OF TUBES, NTBS = 10

FIN HEIGHT, HFN = 6.000 IN  
 THICKNESS AT ROOT, SROOT = .050 IN  
 THICKNESS AT TIP, STIP = .050 IN  
 MATERIAL ALUMINUM  
 MASS (ALL FINS), MFN = 84.2960 LBM  
 NO. OF FIN SIDES RADIATING = 1

COOLANT FLUID IS SILICONE OIL  
 MASS (IN ALL TUBES), MFL = 19.1289 LBM

PROTECTION LAYER THICKNESS, SMP = .011 IN  
 MASS, MMP = 2.038 LBM  
 MATERIAL IS BERYLLIUM

TOTAL MASS (EXCL. MANIFLD.) MTOT = 120.9104 LBM



TOTAL AREA (SINGLE NORMAL PROJECTION) :

ATOT = 124.5000 SQ FT

72

52

FIGURE 16(A) CONTINUED

ELAPSED TIME IS  
RELATIVE TIME IS

.0000 HR  
.0000

\*\*IN ORBIT\*\*

1 INTEGR. STEPS

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R

REF. RADIANT HEAT FLUX

PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT)

TOT. RADIANT REJECTION QTOT = .2114+05 BTU/HR

CONU. FROM MANIFOLDS, CONDMF = .0000 BTU/HR

INCIDENT SOLAR FLUX, QSOLR = .4326+03 BTU/(HR SQ FT)

INCIDENT INFRARED FLUX, QIRED = .1380+02 BTU/(HR SQ FT)

AERODYN. HEATING POWER, QCONV = .0000 BTU/HR

ENERGY STORAGE RATE, STORG = -.2114+05 BTU/HR

AXIAL RELATIVE RELATIVE TEMPERATURE OF FIN, T  
DIST. RAD. HEAT REJECTION  
Z Q X  
DISTANCE NORMAL TO FLOW DIRECTION

Z	Q	X	.00000	.25000	.50000	.75000	1.00000
.000	.5473	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000
.125	.5439	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000
.250	.5418	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000
.375	.5418	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000
.500	.5418	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000
.625	.5412	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000
.750	.5418	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000
.875	.5439	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000
1.000	.5473	1.0000	1.0000	1.0000	1.0000	1.0000	1.0000

(EXIT)

FIGURE 16(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R

REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT

REFERENCE VELOCITY, W00 = .871 FT/SEC

COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR

EXIT CURRENTLY EI = .276866+05 BTU/HR

INLET CURRENTLY HI = .276867+05 BTU/HR

TOT. REJECTION DH = .500488+01 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TF	WALL TEMPERATURES TWI	PROTECT. LAYER TEMPERATURES TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	1.0000	1.0000	.000000	.00000
.125	.99981	1.0000	1.0000	1.0000	1.0000	.000138	.02563
.250	.99962	1.0000	1.0000	1.0000	1.0000	.000242	.09431
.375	.99943	1.0000	1.0000	1.0000	1.0000	.000311	.19375
.500	.99924	1.0000	1.0000	1.0000	1.0000	.000380	.31625
.625	.99905	1.0000	1.0000	1.0000	1.0000	.000450	.46386
.750	.99886	1.0000	1.0000	1.0000	1.0000	.000484	.63147
.875	.99867	1.0000	1.0000	1.0000	1.0000	.000519	.80933
1.000	.99848	1.0000	1.0000	1.0000	1.0000	.000553	1.00000

(EXIT)

ELAPSED TIME IS .0700 HR  
RELATIVE TIME IS 18.2969

\*\*IN ORBIT\*\*

44 INTEGR. STEPS

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT),  
TOT. RADIANT REJECTION QTOT = .1368+05 BTU/HR  
COND. FROM MANIFOLDS, CONDMF = .1925+03 BTU/HR  
INCIDENT SOLAR FLUX, QSOLR = .4326+03 BTU/(HR SQ FT)  
INCIDENT INFRARED FLUX, QIKED = .1313+02 BTU/(HR SQ FT)  
AERODYN. HEATING POWER, QCONV = .0000 BTU/HR  
ENERGY STORAGE RATE, STORG = -.7238+04 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
Z		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.4812	.9816	.9816	.9816	.9816	.9816
.125	.3944	.9592	.9448	.9351	.9276	.9278
.250	.3403	.9581	.9453	.9368	.9320	.9305
.375	.3352	.9549	.9431	.9352	.9308	.9294
.500	.3306	.9527	.9415	.9341	.9299	.9286
.625	.3178	.9504	.9403	.9337	.9299	.9287
.750	.3230	.9485	.9386	.9320	.9283	.9271
.875	.3463	.9447	.9351	.9287	.9250	.9238
1.000 (EXIT)	.3877	.9538	.9538	.9538	.9538	.9538

FIGURE 16(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
REFERENCE VELOCITY, W00 = .871 FT/SEC  
COOLANT POWER, INLET AT T=T0 HO = .276867+05 BTU/HR  
EXIT CURRENTLY EI = .214331+05 BTU/HR  
INLET CURRENTLY HI = .276867+05 BTU/HR  
TOT. REJECTION DH = .625357+04 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMP TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9816	.9816	39.721291	.00000
.125	.99981	.9963	.9919	.9592	.9592	70.315785	.15369
.250	.99962	.9933	.9863	.9581	.9581	60.747243	.32489
.375	.99943	.9907	.9801	.9549	.9549	54.229073	.46643
.500	.99924	.9883	.9758	.9527	.9527	49.817427	.59891
.625	.99905	.9862	.9704	.9504	.9504	43.049000	.71844
.750	.99886	.9842	.9676	.9485	.9485	41.208278	.82419
.875	.99867	.9824	.9626	.9447	.9447	38.468452	.92672
1.000 (EXIT)	.99848	.9816	.9600	.9538	.9538	13.438381	1.00000

ELAPSED TIME IS .1400 HR  
 RELATIVE TIME IS 36.5939 \*\*IN ORBIT\*\* 74 INTEGR. STEPS

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT); INCIDENT SOLAR FLUX, QSOLR = .4326+03 BTU/(HR SQ FT)  
 TOT. RADIANT REJECTION QTOT = .1151+05 BTU/HR; INCIDENT INFRARED FLUX, QIRED = .1164+02 BTU/(HR SQ FT)  
 COND. FROM MANIFOLDS, CONDMF = .2601+03 BTU/HR; AERODYN. HEATING POWER, QCONV = .0000 BTU/HR  
 ENERGY STORAGE RATE, STORG = -.2599+04 BTU/HR

AXIAL RELATIVE RELATIVE TEMPERATURE OF FIN, T  
 DIST. RAD. HEAT REJECTION  
 Z Q  
 DISTANCE NORMAL TO FLOW DIRECTION  
 X  
 .00000 .25000 .50000 .75000 1.00000

.000	.4690	.9777	.9777	.9777	.9777	.9777
.125	.3587	.9453	.9273	.9150	.9078	.9054
.250	.2888	.9446	.9290	.9184	.9122	.9102
.375	.2791	.9392	.9247	.9148	.9091	.9072
.500	.2702	.9357	.9219	.9125	.9070	.9053
.625	.2516	.9316	.9191	.9107	.9058	.9043
.750	.2545	.9282	.9157	.9073	.9024	.9008
.875	.2787	.9212	.9089	.9005	.8955	.8940
1.000	.3245	.9331	.9331	.9331	.9331	.9331
(EXIT)						

FIGURE 16(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
 REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
 REFERENCE VELOCITY, W00 = .871 FT/SEC  
 COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
 EXIT CURRENTLY EI = .190346+05 BTU/HR TOT. REJECTION OH = .865213+04 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TF	WALL TEMPERATURES TW	PROTECT. LAYER TEMPERATURES TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9777	.9777	46.776074	.00000
.125	.99981	.9952	.9889	.9453	.9453	91.621171	.14379
.250	.99962	.9912	.9821	.9446	.9446	78.693769	.30749
.375	.99942	.9877	.9736	.9392	.9392	72.155310	.44290
.500	.99923	.9844	.9680	.9357	.9357	67.887362	.57417
.625	.99904	.9815	.9603	.9316	.9316	60.128966	.69518
.750	.99885	.9787	.9564	.9282	.9282	59.147844	.80515
.875	.99867	.9759	.9488	.9212	.9212	57.910906	.91549
1.000	.99848	.9745	.9446	.9331	.9331	24.112062	1.00000
(EXIT)							

ELAPSED TIME IS  
RELATIVE TIME IS

.2100 HR  
54.8908

\*\*IN ORBIT\*\*

107 INTEGR. STEPS

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

REFERENCE TEMPERATURE, T00 = 659.670 R

REF. RADIANT HEAT FLUX

PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT)

TOT. RADIANT REJECTION QTOT = .1081+05 BTU/HR

COND. FROM MANIFOLDS, CONDMF = .2873+03 BTU/HR

INCIDENT SOLAR FLUX,

QSOLR = .4326+03 BTU/(HR SQ FT)

INCIDENT INFRARED FLUX,

QIRED = .9903+01 BTU/(HR SQ FT)

AERODYN. HEATING POWER,

QCONV = .0000 BTU/HR

ENERGY STORAGE RATE,

STORG = -.9825+03 BTU/HR

AXIAL  
DIST.

RELATIVE  
RAD. HEAT  
REJECTION  
Q

RELATIVE TEMPERATURE OF FIN, T

DISTANCE NORMAL TO FLOW DIRECTION

Z

X

.00000 .25000 .50000 .75000 1.00000

.000	.4659	.9763	.9763	.9763	.9763	.9763
.125	.3485	.9402	.9212	.9079	.9002	.8976
.250	.2736	.9401	.9236	.9123	.9057	.9036
.375	.2620	.9337	.9182	.9077	.9015	.8995
.500	.2510	.9297	.9149	.9048	.8989	.8970
.625	.2295	.9247	.9113	.9022	.8969	.8953
.750	.2309	.9206	.9071	.8979	.8925	.8908
.875	.2552	.9118	.8983	.8889	.8834	.8817
1.000	.3023	.9252	.9252	.9252	.9252	.9252

(EXIT)

FIGURE 16(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

REFERENCE TEMPERATURE, T00 = 659.670 R

REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT

REFERENCE VELOCITY, W00 = .871 FT/SEC

COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR

EXIT CURRENTLY EI = .181485+05 BTU/HR

INLET CURRENTLY HI = .276867+05 BTU/HR

TOT. REJECTION DH = .953815+04 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMP TMP	ENTHALPY REJECTION	
						PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9763	.9763	49.157877	.00000
.125	.99981	.9947	.9878	.9402	.9402	98.755130	.14067
.250	.99962	.9905	.9807	.9401	.9401	84.352570	.30112
.375	.99942	.9867	.9713	.9337	.9337	78.174984	.43370
.500	.99923	.9831	.9654	.9297	.9297	74.254160	.56407
.625	.99904	.9799	.9567	.9247	.9247	66.467727	.68532
.750	.99885	.9768	.9525	.9206	.9206	66.269156	.79686
.875	.99866	.9736	.9438	.9118	.9118	66.394603	.91070
1.000	.99848	.9719	.9389	.9252	.9252	28.582312	1.00000

(EXIT)

ELAPSED TIME IS .2800 HR  
RELATIVE TIME IS 73.1878

\*\*IN ORBIT\*\*

136 INTEGR. STEPS

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R

REF. RADIANT HEAT FLUX  
PER UNIT AXIAL LENGTH, GREF = .3246+04 BTU/(HR FT),

TOT. RADIANT REJECTION GTOT = .1059+05 BTU/HR

COND. FROM MANIFOLDS, CONDMF = .2989+03 BTU/HR

INCIDENT SOLAR FLUX, GSOLR = .4326+03 BTU/(HR SQ FT)

INCIDENT INFRARED FLUX, GIREP = .7787+01 BTU/(HR SQ FT)

AERODYN. HEATING POWER, GCONV = .0000 BTU/HR

ENERGY STORAGE RATE, STORG = -.4030+03 BTU/HR

AXIAL  
DIST.  
Z  
RELATIVE  
RAD. HEAT  
REJECTION  
θ

RELATIVE TEMPERATURE OF FIN, T

DISTANCE NORMAL TO FLOW DIRECTION  
X

Z	θ	.00000	.25000	.50000	.75000	1.00000
.000	.4658	.9758	.9758	.9758	.9758	.9758
.125	.3461	.9382	.9187	.9051	.8972	.8946
.250	.2695	.9384	.9216	.9101	.9033	.9012
.375	.2570	.9315	.9158	.9049	.8986	.8966
.500	.2451	.9274	.9123	.9019	.8958	.8939
.625	.2224	.9220	.9083	.8989	.8935	.8917
.750	.2230	.9176	.9037	.8941	.8886	.8868
.875	.2472	.9080	.8939	.8841	.8783	.8765
1.000 (EXIT)	.2947	.9219	.9219	.9219	.9219	.9219

FIGURE 16(A) CONTINUED

## FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R

REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT

REFERENCE VELOCITY, W00 = .871 FT/SEC

COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR

EXIT CURRENTLY EI = .177966+05 BTU/HR

INLET CURRENTLY HI = .276867+05 BTU/HR

TOT. REJECTION DH = .989009+04 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMP TF	WALL TEMP TWI	PROTECT. LAYER TEMP TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9758	.9758	50.129854	.00000
.125	.99981	.9946	.9873	.9382	.9382	101.470659	.13955
.250	.99961	.9903	.9802	.9384	.9384	86.378242	.29864
.375	.99942	.9863	.9704	.9315	.9315	80.419735	.42994
.500	.99923	.9826	.9644	.9274	.9274	76.643045	.55984
.625	.99904	.9793	.9553	.9220	.9220	68.930267	.68103
.750	.99885	.9760	.9510	.9176	.9176	69.137488	.79315
.875	.99866	.9727	.9418	.9080	.9080	70.083364	.90853
1.000 (EXIT)	.99848	.9708	.9367	.9219	.9219	30.468460	1.00000

ELAPSED TIME IS .3150 HR  
RELATIVE TIME IS 82.3363

\*\*IN ORBIT\*\*

151 INTEGR. STEPS

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R	INCIDENT SOLAR FLUX, QSOLR = .4326+03 BTU/(HR SQ FT)
REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT),	INCIDENT INFRARED FLUX, QIREN = .6522+01 BTU/(HR SQ FT)
TOT. RADIANT REJECTION QTOT = .1056+05 BTU/HR	AERODYN. HEATING POWER, QCONV = .0000 BTU/HR
CONV. FROM MANIFOLDS, CONDMF = .3020+03 BTU/HR	ENERGY STORAGE RATE, STORG = -.2744+03 BTU/HR

78

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION	RELATIVE TEMPERATURE OF FIN, T				
Z	θ	DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000

.000	.4664	.9756	.9756	.9756	.9756	.9756
.125	.3461	.9377	.9180	.9044	.8954	.8937
.250	.2690	.9380	.9211	.9095	.9027	.9005
.375	.2564	.9309	.9151	.9042	.8979	.8958
.500	.2442	.9268	.9116	.9011	.8950	.8931
.625	.2211	.9213	.9074	.8980	.8925	.8908
.750	.2216	.9168	.9028	.8931	.8875	.8857
.875	.2457	.9069	.8927	.8828	.8770	.8751
1.000 (EXIT)	.2634	.9211	.9211	.9211	.9211	.9211

FIGURE 16(A) CONCLUDED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R	
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT	
REFERENCE VELOCITY, W00 = .871 FT/SEC	
COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR	INLET CURRENTLY HI = .276867+05 BTU/HR
EXIT CURRENTLY EI = .177034+05 BTU/HR	TOT. REJECTION DH = .998328+04 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TF	WALL TEMPERATURES TWI	PROTECT. LAYER TEMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9756	.9756	50.420656	.00000
.125	.99981	.9945	.9871	.9377	.9377	102.200647	.13930
.250	.99961	.9902	.9800	.9380	.9380	86.914325	.29805
.375	.99942	.9862	.9701	.9309	.9309	81.018184	.42902
.500	.99923	.9825	.9642	.9268	.9268	77.254283	.55679
.625	.99904	.9791	.9549	.9213	.9213	69.565755	.67993
.750	.99885	.9759	.9506	.9168	.9168	69.883926	.79218
.875	.99866	.9724	.9413	.9069	.9069	71.081735	.90794
1.000 (EXIT)	.99848	.9706	.9361	.9211	.9211	30.978268	1.00000

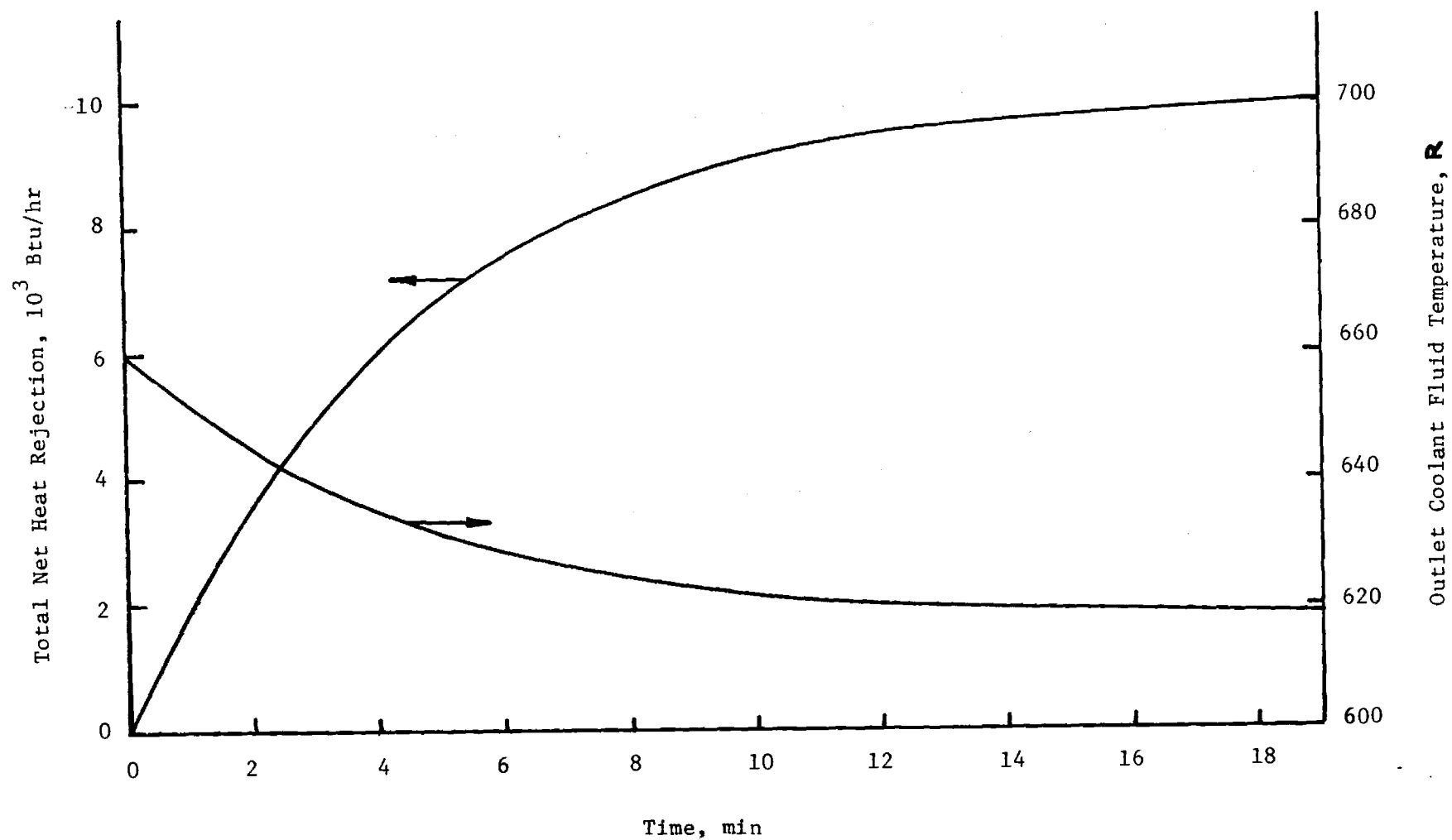


Fig. 16(b) System Response For Orbital Mission-Heat Rejection And Outlet Fluid Temperature



#### d. Steady State Environment

##### (i) Description of Environment

Program output has been produced for steady state system performance to a time invariant environmental conditions. The entire radiator system is initially assumed to be at a uniform temperature of 200°F (TI = 659.67R). The integration process is started and it proceeds until steady state operation of the system is achieved. The environmental conditions during the integration process remain constant. For the output of Figure 17, incident radiant flux values were read in from cards according to the format described in section A.3(d). These flux values were 100 Btu/hr. ft.<sup>2</sup> for solar irradiation, 2.0 Btu/hr. ft.<sup>2</sup> for earth albedo. There was assumed to be no infrared earth irradiation. There was no aerodynamic heating of the radiator panel. All other input parameters are the same as those used for program output during ascent (see section A.5(a)).

##### (ii) System Response

The response of the radiator system during the period leading to steady state operation is shown in Figure 17. Figure 17(a) shows the input parameters and a selected number of output listings of the system performance. Figure 17(b) graphically summarizes the resulting outlet coolant fluid temperature and the total heat rejection of the radiator system as the system approaches steady state operation.

```

$QNML
ITAPE  = +0
ICASE  = +0
NTM    = +4
TO     = .00000000E+00
PHN    = .00000000E+00

```

```

$END
$TUBE
GITBI  = .25000000E+00
STBI   = .10000000E+00
XL     = .12000000E+02
RHOTBI = .16859200E+03
MZ     = +5
NRTBI  = +3
NTBS   = +10

```

```

$END
$FLOW
MCOTI  = .50000000E+03
TO     = .65967000E+03
PO     = .86400000E+04

```

FIGURE 17(A) SYSTEM RESPONSE FOR STEADY STATE ENVIRONMENT  
PROGRAM OUTPUT

```

$END
$FIN
NX      = +5
SROOTI  = .50000000E-01
MFNI    = .60000000E+01
STIPI   = .50000000E-01
KHOFNI  = .16859200E+03
STAGX   = .20000000E+02
VERTA   = .10000000E+02

```

```

$END
$PROTLR
NRMP    = +3
RHOMPI  = .16859200E+03
KHONET  = .50000000E+00
VELM    = .65600000E+05
TAU     = .36500000E+04
PROB    = .99000000E+00
ALPHA   = .18800000E-09
BTA     = .12130000E+01
GAMMA   = .15000000E+01
PHI     = .50000000E+00
THETA   = .66666700E+00
ATK     = .17500000E+01
AN      = .10000000E+01

```

```

$END
$MANIFD
AMAN    = .43000000E+02

```

```

$END
$RUNOPT
MSTOTR  = +1

```

DTWRTE = .33333000E-01  
TEND = .00000000E+00  
ALIMIT = .50000000E-04  
RLIMIT = .10000000E-04  
TI = .65967000E+03  
LIMWT = +25  
NCONV = +0  
LTY = +0  
LFLD = +1  
LTS = +0

SEND

FIGURE 17(A) CONTINUED

## INITIAL LINE CONDITIONS

\*\*\*\*\*

(ALL QUANTITIES ARE NORMALIZED)

PT.NO.	POSITION Z	PRESSURE P	VELOCITY W	FLUID TEMPERATURE T	WALL TEMPERATURE TWI
1	.000	1.000000	1.000000	1.000000	1.000000
2	.250	.999621	1.000001	1.000000	1.000000
3	.500	.999241	1.000001	1.000000	1.000000
4	.750	.998862	1.000001	1.000000	1.000000
5	1.000	.998483	1.000002	1.000000	1.000000

INLET PRESSURE P0 = 8640.000 LBF/SQ.FT  
 REF. VELOCITY W0 = .87128 FT/SFC  
 REF. TEMPERATURE T00 = 659.670 R

REYNOLDS NO = .30017+04  
 PRANDTL NO = 8.628066  
 DELTA = .001736  
 REL. PRESSURE IS .391534+04

FIGURE 17(A) CONTINUED

INIT. NUSSLETT NO. NU = .201856+02  
 WALL BIOT NO. BI = .485314-02

## SYSTEM PARAMETERS

\*\*\*\*\*

TUBE LENGTH, XL = 12.000 FT  
 INTERNAL DIAMETER, DITB = .250 IN  
 WALL THICKNESS, STB = .100 IN  
 MATERIAL ALUMINUM  
 MASS (ALL TUBES), MTB = 15.4480 LBM  
 NUMBER OF TUBES, NTBS = 10

FIN HEIGHT, HFN = 6.000 IN  
 THICKNESS AT ROOT, SROOT = .050 IN  
 THICKNESS AT TIP, STIP = .050 IN  
 MATERIAL ALUMINUM  
 MASS (ALL FINS), MFN = 84.2960 LBM  
 NO. OF FIN SIDES RADIATING = 1

COOLANT FLUID IS SILICONE OIL  
 MASS (IN ALL TUBES), MFL = 19.1289 LBM

PROTECTION LAYER THICKNESS, SMP = .011 IN  
 MASS, MMP = 2.038 LBM  
 MATERIAL IS BERYLLIUM

TOTAL MASS (EXCL. MANIFLD.) MTOT = 120.9104 LBM  
 TOTAL AREA (SINGLE NORMAL PROJECTION),  
 ATOT = 124.5000 SQ FT

ELAPSED TIME IS .0000 HR  
RELATIVE TIME IS .0000

\*\*IN ORBIT\*\*

1 INTEGR. STEPS

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*  
REFERENCE TEMPERATURE, T00 = 659.670 R  
REF. RADIANT HEAT FLUX  
PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
TOT. RADIANT REJECTION QTOT = .3288+05 BTU/HR, INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
COND. FROM MANIFOLDS, CONDMF = .0000 BTU/HR, AERODYN. HEATING POWER, QCONV = .0000 BTU/HR  
ENERGY STORAGE RATE, STORG = -.3288+05 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION				
		X				
		.00000	.25000	.50000	.75000	1.00000
.000	.8477	1.0000	1.0000	1.0000	1.0000	1.0000
.250	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.500	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
.750	.8436	1.0000	1.0000	1.0000	1.0000	1.0000
1.000 (EXIT)	.8477	1.0000	1.0000	1.0000	1.0000	1.0000

FIGURE 17(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*  
REFERENCE TEMPERATURE, T00 = 659.670 R  
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
REFERENCE VELOCITY, W00 = .871 FT/SEC  
COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
EXIT CURRENTLY EI = .276866+05 BTU/HR TOT. REJECTION DH = .498047-01 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TF	WALL TEMPERATURES TWI	PROTECT. LAYER TEMPERATURES TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	1.0000	1.0000	.000000	.00000
.250	.99962	1.0000	1.0000	1.0000	1.0000	.000277	.10085
.500	.99924	1.0000	1.0000	1.0000	1.0000	.000415	.33176
.750	.99886	1.0000	1.0000	1.0000	1.0000	.000519	.63619
1.000 (EXIT)	.99848	1.0000	1.0000	1.0000	1.0000	.000588	1.00000

ELAPSED TIME IS .1000 HR  
RELATIVE TIME IS 26.1382

\*\*IN ORBIT\*\*

52 INTEGR. STEPS

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*  
REFERENCE TEMPERATURE, T00 = 659.670 R  
REF. RADIANT HEAT FLUX  
PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
TOT. RADIANT REJECTION QTOT = .1977+05 BTU/HR, INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
COND. FROM MANIFOLDS, CONDMF = .1545+03 BTU/HR, AERODYN. HEATING POWER, QCONV = .0000 BTU/HR  
ENERGY STORAGE RATE, STORG = -.7621+04 BTU/HR

AXIAL RELATIVE RELATIVE TEMPERATURE OF FIN, T  
DIST. RAD. HEAT  
REJECTION  
Z 0  
DISTANCE NORMAL TO FLOW DIRECTION  
X

Z	0	.00000	.25000	.50000	.75000	1.00000
.000	.7309	.9669	.9669	.9669	.9669	.9669
.250	.5016	.9203	.8992	.8851	.8770	.8744
.500	.4808	.9091	.8903	.8777	.8704	.8681
.750	.4615	.8974	.8815	.8709	.8647	.8628
1.000	.5476	.9070	.9070	.9070	.9070	.9070
(EXIT)						

FIGURE 17(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*  
REFERENCE TEMPERATURE, T00 = 659.670 R  
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
REFERENCE VELOCITY, W00 = .871 FT/SEC  
COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
EXIT CURRENTLY EI = .156871+05 BTU/HR TOT. REJECTION DH = .119996+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TF	WALL TEMPERATURES TWI	PROTECT. LAYER TEMPERATURES TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9669	.9669	69.442389	.00000
.250	.99962	.9884	.9740	.9203	.9203	112.865777	.27736
.500	.99923	.9788	.9559	.9091	.9091	98.273507	.58267
.750	.99886	.9709	.9354	.8974	.8974	79.855536	.82863
1.000	.99848	.9668	.9238	.9070	.9070	35.347144	1.00000
(EXIT)							

ELAPSED TIME IS  
RELATIVE TIME IS

.2000 HR  
52.2765

\*\*IN ORBIT\*\*

79 INTEGR. STEPS

86

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R	INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)
REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT),	INCIDENT INFRARED FLUX, QIREP = .2000+01 BTU/(HR SQ FT)
TOT. RADIANT REJECTION QTOT = .1740+05 BTU/HR	AERODYN. HEATING POWER, QCONV = .0000 BTU/HR
CONU. FROM MANIFOLDS, CONDMF = .1896+03 BTU/HR	ENERGY STORAGE RATE, STORG = -.2265+04 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
Z	Q	DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.7183	.9632	.9632	.9632	.9632	.9632
.250	.4485	.9041	.8798	.8630	.8533	.8501
.500	.4166	.8883	.8657	.8502	.8411	.8381
.750	.3838	.8698	.8501	.8366	.8287	.8261
1.000 (EXIT)	.4785	.8810	.8810	.8810	.8810	.8810

FIGURE 17(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R	
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT	
REFERENCE VELOCITY, W00 = .871 FT/SEC	
COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR	INLET CURRENTLY HI = .276867+05 BTU/HR
EXIT CURRENTLY EI = .127453+05 BTU/HR	TOT. REJECTION DH = .149414+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES TF	WALL TEMPERATURES TWI	PROTECT. LAYER TEMPERATURES TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9632	.9632	75.726097	.00000
.250	.99961	.9863	.9690	.9041	.9041	133.561531	.25844
.500	.99923	.9745	.9478	.8883	.8883	122.299504	.55811
.750	.99886	.9641	.9208	.8698	.8698	104.992007	.81310
1.000 (EXIT)	.99848	.9585	.9050	.8810	.8810	49.425550	1.00000

ELAPSED TIME IS  
RELATIVE TIME IS.3000 HR  
78.4147

\*\*IN ORBIT\*\*

104 INTEGR. STEPS

87

## FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R

REF. RADIANT HEAT FLUX

PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT),

TOT. RADIANT REJECTION QTOT = .1685+05 BTU/HR

COND. FROM MANIFOLDS, CONDMF = .1986+03 BTU/HR

INCIDENT SOLAR FLUX,

QSOLR = .1000+03 BTU/(HR SQ FT)

INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)

AERODYN. HEATING POWER, QCONV = .0000 BTU/HR

ENERGY STORAGE RATE, STORG = -.9773+03 BTU/HR

AXIAL  
DIST.  
RELATIVE  
RAD. HEAT  
REJECTION

RELATIVE TEMPERATURE OF FIN, T

DISTANCE NORMAL TO FLOW DIRECTION

Z

Q

X

.00000 .25000 .50000 .75000 1.00000

.000	.7158	.9624	.9624	.9624	.9624	.9624
.250	.4374	.9005	.8755	.8582	.8481	.8448
.500	.4025	.8836	.8600	.8437	.8342	.8311
.750	.3645	.8625	.8417	.8273	.8189	.8162
1.000 (EXIT)	.4617	.8743	.8743	.8743	.8743	.8743

FIGURE 17(A) CONTINUED

## FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R

REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT

REFERENCE VELOCITY, W00 = .871 FT/SEC

COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR

INLET CURRENTLY HI = .276867+05 BTU/HR

EXIT CURRENTLY EI = .120168+05 BTU/HR

TOT. REJECTION DH = .156699+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID WALL PROTECT. TEMPERATURES LAYER			ENTHALPY REJECTION	
			TF	TWI	TMP	PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9624	.9624	76.930037	.00000
.250	.99961	.9858	.9678	.9005	.9005	137.760096	.25316
.500	.99923	.9735	.9462	.8836	.8836	128.067226	.55047
.750	.99886	.9625	.9173	.8625	.8625	112.156114	.80811
1.000 (EXIT)	.99848	.9565	.9003	.8743	.8743	53.326069	1.00000



ELAPSED TIME IS .4000 HR  
RELATIVE TIME IS 104.5529

\*\*IN ORBIT\*\*

130 INTEGR. STEPS

# FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R	INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)
REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT),	INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)
TOT. RADIANT REJECTION QTOT = .1671+05 BTU/HR	AERODYN. HEATING POWER, QCONV = .0000 BTU/HR
COND. FROM MANIFOLDS, CONDMF = .2010+03 BTU/HR	ENERGY STORAGE RATE, STORG = -.6569+03 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
Z		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.7152	.9622	.9622	.9622	.9622	.9622
.250	.4348	.8997	.8745	.8571	.8469	.8436
.500	.3993	.8825	.8587	.8422	.8326	.8294
.750	.3595	.8607	.8395	.8249	.8163	.8135
1.000 (EXIT)	.4574	.8725	.8725	.8725	.8725	.8725

FIGURE 17(A) CONTINUED

# FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R	
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT	
REFERENCE VELOCITY, W00 = .871 FT/SEC	
COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR	INLET CURRENTLY HI = .276867+05 BTU/HR
EXIT CURRENTLY EI = .118339+05 BTU/HR	TOT. REJECTION DH = .158528+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMPERATURE TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9622	.9622	77.176887	.00000
.250	.99961	.9857	.9675	.8997	.8997	138.666939	.25172
.500	.99923	.9733	.9458	.8825	.8825	129.416553	.54823
.750	.99886	.9621	.9165	.8607	.8607	114.102509	.80655
1.000 (EXIT)	.99848	.9560	.8992	.8725	.8725	54.379859	1.00000

ELAPSED TIME IS .5000 HR  
RELATIVE TIME IS 130.6912

\*\*IN ORBIT\*\*

157 INTEGR. STEPS

02

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
REF. RADIANT HEAT FLUX  
PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT), INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
TOT. RADIANT REJECTION QTOT = .1668+05 BTU/HR, INCIDENT INFRARED FLUX, QIREQ = .2000+01 BTU/(HR SQ FT)  
COND. FROM MANIFOLDS, CONDMF = .2017+03 BTU/HR, AERODYN. HEATING POWER, QCONV = .0000 BTU/HR  
ENERGY STORAGE RATE, STORG = -.5757+03 BTU/HR

AXIAL RELATIVE RELATIVE TEMPERATURE OF FIN, T  
DIST. RAD. HEAT  
REJECTION  
Z Q DISTANCE NORMAL TO FLOW DIRECTION  
X

Z	Q	.00000	.25000	.50000	.75000	1.00000
.000	.7150	.9622	.9622	.9622	.9622	.9622
.250	.4342	.8994	.8742	.8568	.8466	.8433
.500	.3985	.8822	.8584	.8419	.8322	.8290
.750	.3582	.8602	.8390	.8243	.8156	.8128
1.000 (EXIT)	.4564	.8721	.8721	.8721	.8721	.8721

FIGURE 17(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R  
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
REFERENCE VELOCITY, W00 = .871 FT/SEC  
COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR INLET CURRENTLY HI = .276867+05 BTU/HR  
EXIT CURRENTLY EI = .117874+05 BTU/HR TOT. REJECTION DH = .158993+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMPERATURE TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9622	.9622	77.232511	.00000
.250	.99961	.9857	.9674	.8994	.8994	138.877901	.25135
.500	.99923	.9732	.9457	.8822	.8822	129.726835	.54762
.750	.99886	.9620	.9163	.8602	.8602	114.610315	.80610
1.000 (EXIT)	.99848	.9558	.8989	.8721	.8721	54.654328	1.00000

ELAPSED TIME IS .6000 HR  
RELATIVE TIME IS 156.8294

\*\*IN ORBIT\*\*

184 INTEGR. STEPS

FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*  
REFERENCE TEMPERATURE, T00 = 659.670 R  
REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT),  
TOT. RADIANT REJECTION QTOT = .1667+05 BTU/HR  
COND. FROM MANIFOLDS, CONDMF = .2018+03 BTU/HR  
INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)  
INCIDENT INFRARED FLUX, QIRED = .2000+01 BTU/(HR SQ FT)  
AERODYN. HEATING POWER, QCONV = .0000 BTU/HR  
ENERGY STORAGE RATE, STORG = -.5547+03 BTU/HR

AXIAL DIST. Z	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000
.000	.7150	.9622	.9622	.9622	.9622	.9622
.250	.4340	.8994	.8741	.8567	.8465	.8432
.500	.3983	.8622	.8583	.8418	.8321	.8289
.750	.3579	.8601	.8388	.8241	.8155	.8127
1.000 (EXIT)	.4561	.8720	.8720	.8720	.8720	.8720

FIGURE 17(A) CONTINUED

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*  
REFERENCE TEMPERATURE, T00 = 659.670 R  
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT  
REFERENCE VELOCITY, W00 = .871 FT/SEC  
COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR  
EXIT CURRENTLY EI = .117753+05 BTU/HR  
INLET CURRENTLY HI = .276867+05 BTU/HR  
TOT. REJECTION DH = .159114+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURES		PROTECT. LAYER TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
			TF	TWI			
.000	1.00000	1.0000	1.0000	.9622	.9622	77.246378	.00000
.250	.99961	.9857	.9674	.8994	.8994	138.930130	.25127
.500	.99923	.9732	.9457	.8822	.8822	129.798357	.54747
.750	.99886	.9620	.9162	.8601	.8601	114.739276	.80599
1.000 (EXIT)	.99848	.9558	.8988	.8720	.8720	54.722584	1.00000

ELAPSED TIME IS .6183 HR  
RELATIVE TIME IS 161.6107

\*\*IN ORBIT\*\*

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FIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT REJECTION

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R	INCIDENT SOLAR FLUX, QSOLR = .1000+03 BTU/(HR SQ FT)
REF. RADIANT HEAT FLUX PER UNIT AXIAL LENGTH, QREF = .3246+04 BTU/(HR FT),	INCIDENT INFRARED FLUX, QIREN = .2000+01 BTU/(HR SQ FT)
TOT. RADIANT REJECTION QTOT = .1667+05 BTU/HR	AERODYN. HEATING POWER, QCONV = .0000 BTU/HR
CONJ. FROM MANIFOLDS, CONDMF = .2019+03 BTU/HR	ENERGY STORAGE RATE, STORG = -.5533+03 BTU/HR

AXIAL DIST.	RELATIVE RAD. HEAT REJECTION Q	RELATIVE TEMPERATURE OF FIN, T				
Z		DISTANCE NORMAL TO FLOW DIRECTION X				
		.00000	.25000	.50000	.75000	1.00000

FIGURE 17(A) CONCLUDED

.000	.7150	.9622	.9622	.9622	.9622	.9622
.250	.4340	.8994	.8741	.8567	.8465	.8432
.500	.3983	.8822	.8583	.8418	.8321	.8289
.750	.3579	.8600	.8388	.8241	.8155	.8126
1.000 (EXIT)	.4561	.8720	.8720	.8720	.8720	.8720

FLUID PROPERTIES AND SURFACE TEMPERATURE OF METEOROID PROTECTION LAYER

\*\*\*\*\*

REFERENCE TEMPERATURE, T00 = 659.670 R	
REFERENCE PRESSURE, P00 = 8640.000 LBF/SQ.FT	
REFERENCE VELOCITY, W00 = .871 FT/SEC	
COOLANT POWER, INLET AT T=T0 H0 = .276867+05 BTU/HR	INLET CURRENTLY H1 = .276867+05 BTU/HR
EXIT CURRENTLY E1 = .117746+05 BTU/HR	TOT. REJECTION DH = .159121+05 BTU/HR

AXIAL DIST.	PRESSURE P	VELOCITY W	FLUID TEMPERATURE TF	WALL TEMPERATURE TWI	PROTECT. LAYER TEMPERATURE TMP	ENTHALPY REJECTION PER UNIT TUBE LENGTH BTU/(HR FT)	FRACTION OF TOTAL
.000	1.00000	1.0000	1.0000	.9622	.9622	77.247403	.00000
.250	.99961	.9857	.9674	.8994	.8994	138.934032	.25126
.500	.99923	.9732	.9457	.8822	.8822	129.805758	.54745
.750	.99886	.9620	.9162	.8600	.8600	114.752011	.80597
1.000 (EXIT)	.99848	.9558	.8988	.8720	.8720	54.731251	1.00000

STEADY STATE IS REACHED

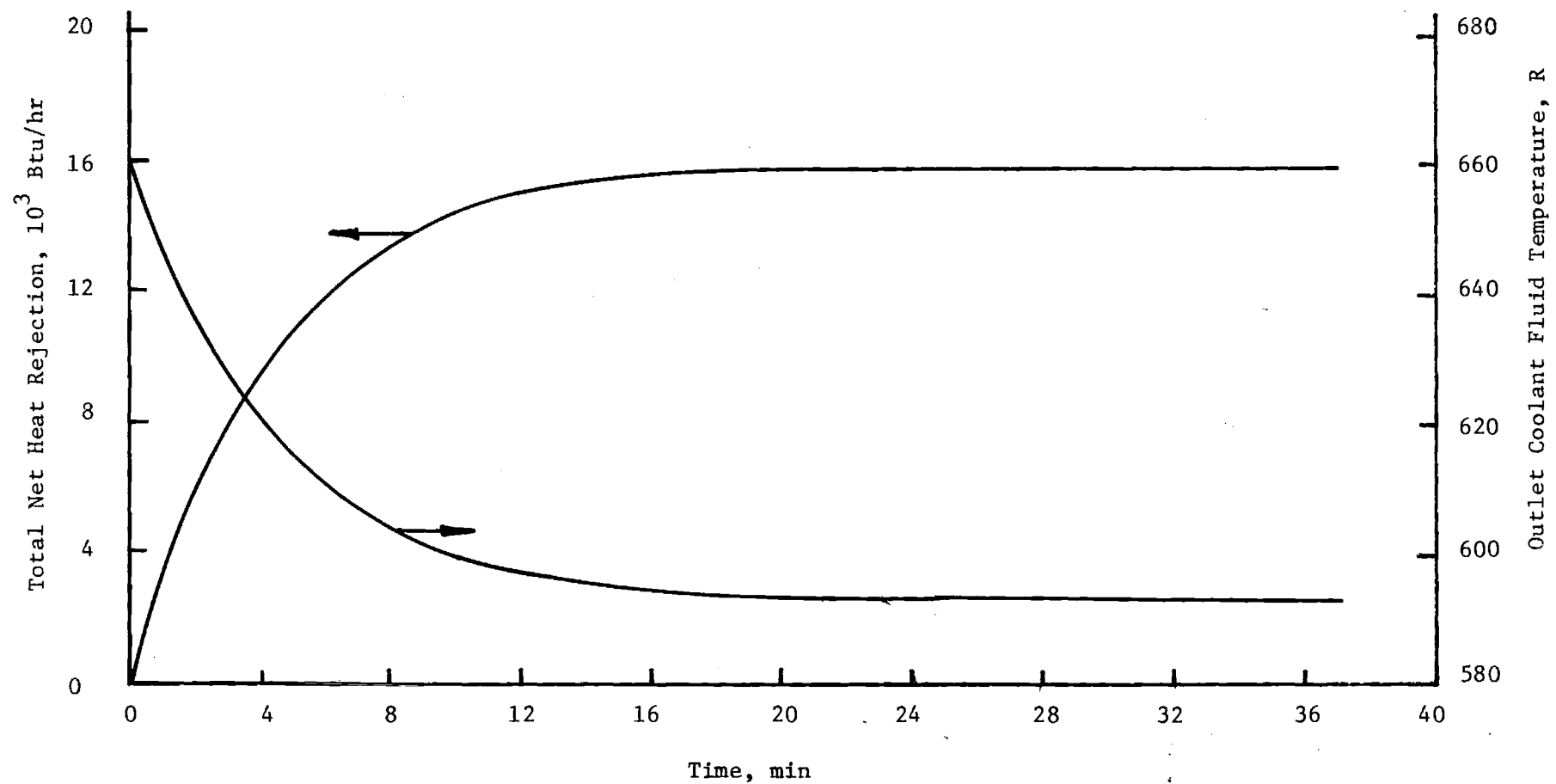


Fig. 17(b) System Response for Steady State Environment-System Heat Rejection And Outlet Fluid Temperature

## B. Program Description

### 1. Structure

The program structure is dictated by the calling sequence of the Runge-Kutta-Simpson integration procedure (see Section III-2 of the System Analysis Manual). The Source Deck consists of two parts. The first part is referred to as the Permanent Part and the second part is called the Selective Part. The Input Data Set follows the Selective Part.

#### a. The Permanent Part

The Permanent Part requires no attention from the user unless he plans major program modifications such as consideration of geometrical configurations other than the one considered on the present analysis. The Permanent Part contains the first 42 program units.

#### b. The Selective Part

The Selective part contains the options for selection of system materials. It is assembled from case to case by the user in accordance with his choice of the coolant fluid material, flow channel material, meteoroid protection layer material, fin panel material and thermal control coating material. There are always 20 program units in the Selective Part.

#### c. The Input Data Set

The Input Data Set follows immediately after the Source Deck. As discussed in Section A-3 the Data Set consists of six groups that specify the non-symmetrical panel conditions, ascent and reentry profiles, the irradiation on the panel, inlet coolant fluid conditions, system specifications and finally the program options. These separate data units are arranged in an order of increasing possibility of input changes so that data units most likely to be changed are at the end of the Data Set.

## 2. Fastrand Execution

A run procedure was written for the purpose of entering the entire program, including property subprograms, on FASTRAND drums and collecting, or mapping, the program into convenient elements. The elements are chosen so that they form commonly used groups of subprograms which then may be selectively collected into a complete program. The advantages of this technique are the significant reduction in the compilation and collection time and the elimination of submitting the program in card form.

The entire program is mapped into two basic elements: the permanent element and the property elements. The permanent element (called PERM), consists of all subroutines listed in the permanent part of the program; the property elements consist of the material properties of the surface coating, fin material, tube material, meteoroid protection material, and coolant fluid.

Each property element has a unique name. The first two letters of the name are a prefix which indicates the function of the particular material. For example, a property element starting with the prefix CF indicates that those properties to be used for the coolant fluid. The remaining letters of the element name are a suffix which describes the particular material to be used. For example, the suffix HE indicates the material helium. The prefix-suffix combination CFHE, therefore, denotes the coolant fluid of helium.

The sixteen names designating the permanent part of the program and the property elements are as follows:

### i) Permanent Elements

1. Element PERM contains all subroutines in the permanent part of the program. (See Part B.3 a through g)

### ii) Property Elements

#### A. Coolant Fluid Properties (Prefix CF)

2. Element CFHE contains all helium properties
3. Element CFSIL contains all silicon oil properties
4. Element CFNAK contains all NaK properties

5. Element CFFC43 contains all properties of the 3-M fluoro-chemical liquid FC43

6. Element CFFC75 contains all properties of the 3-M fluoro-chemical liquid FC75

B. Fin Properties (Prefix FN)

7. Element FNAL contains all aluminum properties

8. Element FNCU contains all copper properties

9. Element FNBR contains all beryllium properties

C. Tube Properties (Prefix TB)

10. Element TBAL contains all aluminum properties

11. Element TBCU contains all copper properties

12. Element TBBR contains all beryllium properties

D. Meteoroid Protection Properties (Prefix MP)

13. Element MPAL contains all aluminum properties

14. Element MPCU contains all copper properties

15. Element MPBR contains all beryllium properties

E. Surface Coating Properties (Prefix SC)

16. Element SCZ93 contains all zinc oxide coating (Z93) properties

a. Control Statements

The run deck consists of the following control statements:

The catalogue card

@ CAT,P NASA.,F2

is used to create a file, called here NASA. This card is to be used only the first time the FASTRAND program is run. It must be omitted when subsequent runs are made because the file NASA should be created only once.

The assign card

@ ASG,AX NASA.

is used to name an external file which has been previously catalogued and to cause its assignment to the run. The option A specifies that



the file is currently catalogued and the option X specifies that the run is to have exclusive use of the file until the run terminates so that no other run may interfere with the program operation.

The use card

@ USE N.,NASA.

allows the user to refer to the previously catalogued file called NASA by the single letter N. The abbreviated file name will save a great deal of card punching for the subsequent portions of the run.

The Fortran compiler cards, one of which would be

@ FØR,IS N.MAIN

creates Fortran symbolic and relocatable elements both called MAIN which are to be entered into the file NASA. The I option causes the insertion of the element MAIN and the S option produces a listing of the element. There must be a FØR card for each subprogram that is to be entered into the file NASA.

After all of the subprograms have been filed, they are mapped into separate elements. This is achieved by the MAP and IN card.

The MAP cards, one of which would be

@ MAP,ISR ,N.PERM

specifies that the collector should combine a set of relocatable elements from the file NASA into a single relocatable element called PERM. There must be a MAP card for each program element that the user expects to utilize during program operation. A listing of these seventeen elements is given at the beginning of this section.

Following each MAP card is a list of the separate subprograms that are to be placed in the element declared by the MAP card. The placing of each subprogram into the element is achieved by an IN card.

The IN card, one of which would be

IN N.MAIN

places the element MAIN from the file NASA in the element name declared by the previous MAP statement. There must be an IN card for each program element that is to be entered into the element declared by the previous MAP statement.

As an example, if the user wishes to collect the subprograms called AA, BB and CC which are located in the file NASA and place them in an element called ELEM, the proper cards would be:

```
@ MAP,ISR ,NASA.ELEM
IN NASA.AA
IN NASA.BB
IN NASA.CC
```

It is necessary to place a define card at the end of each mapped element.

A define card, one of which could be

```
DEF THCFN, DTHCFN, CPFN
```

is used to list those external definitions to be retained by the resulting relocatable element. The DEF statement causes the collector to construct a table defining the entry points to the element. There should be a DEF statement following each MAP card which declares the entry point to each element collected by the MAP card.

#### b. Input Data

The runstream described above enters all subprograms into the file called NASA and then subdivides the program into convenient groupings of elements. Once this run has been made, it becomes a relatively simple task to collect and run a typical system simulation. As an example, assume the user wishes output for a radiator system consisting of an aluminum fin, copper tube, beryllium meteoroid protection layer, silicon oil coolant fluid and Z93 surface coating. Assuming the user has previously declared the file NASA and he has mapped the sixteen elements discussed above, then this combination of materials could be mapped into an absolute element called VER1 by the runstream consisting of

(Run cards appropriate to the users system)

```
@ ASG,AX NASA.
@ USE N.,NASA.
@ MAP,IS ,N.VER1
IN N.PERM
```

```

IN N.FNAL
IN N.TBCU
IN N.MPBR
IN N.CFSIL
IN N.SCZ93
END
@ XQT N.VER1

```

This deck must be followed by the data set described in Section A-3.

As a second example, if the user wishes to run the program for a system consisting of an aluminum tube, fin and protection layer with helium coolant fluid and Z93 surface coating the deck would consist of

(Run cards appropriate to the users system)

```

@ ASG,AX NASA.
@ USE N.,NASA.
@ MAP,IS ,N.VER2
IN N.PERM
IN N.FNAL
IN N.TBAL
IN N.MPAL
IN N.CFHE
IN N.SCZ93
END
@ XQT N.VER2

```

This deck must be followed by the data deck described in Section A-3. In this example the element PERM plus the property elements are mapped into an absolute element called VER2. It should be pointed out that once these absolute elements have been mapped they are available for subsequent runs without being re-mapped. In future runs, if VER2 is called upon to be executed, the element consisting of PERM plus is the property elements listed above are automatically used.

### 3. Description of Program Units

This section describes each subprogram which makes up the Permanent and Selective Part of the program.

The description of each subprogram follows the same format. The name of the subprogram is given and it is categorized as either a FUNCTION or SUBROUTINE subprogram. The argument list is also stated.

The objective of each of the subprograms is stated. All input and output variables in the argument list are defined and units are given for all dimensional input and output quantities.

The calling subprograms and those subprograms called are listed for each program unit.

The program units are subdivided into groups of subprograms each of which has a common overall function. For example, all subprograms that are used for the calculation of the aerodynamic heating of the radiator panel are grouped into a common block of programs. In all, there are 62 program units with 42 units on the permanent part and 20 units in the Selective Part. The Permanent Part contains 42 program units grouped in seven sets from (a) through (g) as follows:

a) the principal integration program set with

1. MAIN, the main program unit which

- |          |  |
|----------|--|
| provides | α) input and output  |
|          | β) unit conversion   |
|          | γ) computation of design parameters                                  |
|          | δ) preparations for principal integration,                           |
| calls    | α) RKS, DERIVM, CNTLM, for principal integration                     |
|          | β) FLSTRT to establish the initial flow field                        |
|          | γ) FUNCTION subprograms for fluid and structural material properties |
|          | δ) QINCID, SHAPEF (see group c)                                      |
|          | ε) mathematical procedure subprograms,                               |

2. SUBROUTINE DERIVM (Y,DY,TIME) which is called from RKS,

- |          |  |
|----------|--|
| receives | α) the current system state variables Y                              |
|          | β) the current time TIME,  |
| provides | α) all derivatives of the variables Y with respect to time,          |
| calls    | α) CONVEC, QRAD  |
|          | β) FLSTRT to establish the flow field                                |
|          | γ) FUNCTION subprograms for fluid and structural material properties |
|          | δ) mathematical procedure subprograms,                               |

3. SUBROUTINE CNTLM (Y,DY,DX,X,NTRY,IFVD) which

- |                |  |
|----------------|--|
| is called from | RKS,                                   |
| receives       | α) all system state variables Y        |
|                | β) their derivatives DY                |
|                | γ) current time X, step size DX,       |
| provides       | α) output during integration           |
|                | β) integration step size control       |
|                | γ) integration termination criteria,   |
| calls          | α) HFL, fluid enthalpy subprogram      |
|                | β) mathematical procedure subprograms, |

b) the secondary integration program set, necessary to establish the dynamic fluid flow field, with

4. SUBROUTINE FLSTRT (RE,PR,DELTA) which

is called from	MAIN,DERIVM,
receives	$\alpha$ ) RE, the mean Reynolds number $\beta$ ) PR, the mean Prandtl number $\gamma$ ) DELTA, the diameter-to-length ratio for the tube,
provides	$\alpha$ ) fluid flow (initial and current) conditions $\beta$ ) table heading for initial conditions,
calls	$\alpha$ ) TRNSPT to compute friction factor and convective film coefficient (in non-dimensional form) $\beta$ ) RKSF, DERIVL, CNTLN $\gamma$ ) fluid flow property subprograms $\delta$ ) mathematical procedure subprograms,

5. SUBROUTINE DERIVL (Y,DY,X) which

is called from	RKSF,
receives	$\alpha$ ) local fluid flow variables Y $\beta$ ) position along flow channel axis, X,
provides	$\alpha$ ) spatial derivatives DY of Y,

6. SUBROUTINE CNTLN (Y,DY,DX,X,NTRY,IFVD) which

is called from	RKSF,
receives	$\alpha$ ) the flow variables Y $\beta$ ) their spatial derivatives DY $\gamma$ ) position X, interval DX along channel axis,
provides	$\alpha$ ) output when initial conditions are established $\beta$ ) termination criteria for RKSF integration,

7. SUBROUTINE TRNSPT (RE,PRL,DELTA,FR,FNU) which computes non-dimensional friction factor and convective film coefficient, Eqs. 3.7 through 9 and Eqs. 3.16 through 18, and

is called from	FLSTRT,
receives	$\alpha$ ) Reynolds number RE

- provides
  - β) Prandtl number PRL
  - γ) diameter-to-length ratio DELTA,
  - α)  $FR = 4f/\delta$ , Eq. 3.32
  - β)  $FNU = 4 N_{Nu}/(\delta N_{Re} N_{Pr})$ , Eq. 3.37,
- c) The program set used to calculate the incident net radiant flux, with,
8. SUBROUTINE QINCID which
- is called from MAIN,  
 receives
  - α) MSTØTR control option for simulation of steady state (= 1) or transient (= 2) conditions
  - β) NSRD the number of radiator sides that are exposed to the environment
  - γ) PHN the angle between the reference used in the MRI file data and the normal to the fin panel,
- provides
  - α) data transfer from the MRI Incident Radiation Computer program or card input
  - β) averaging of radiant flux over circumference of tube cross-section,
9. SUBROUTINE SHAPEF which
- is called from MAIN,  
 provides exchange function SS as defined by Eq. 6.15, see Appendix D,
10. SUBROUTINE TRMATX which
- is called from QRAD,  
 provides the evaluation of the transfer matrix  $M_{ij}$ , defined by Eq. 6.7,
11. SUBROUTINE EXITAV which
- is called from QRAD,  
 provides the evaluation of the excitation vector  $P_j$  in Eq. 6.18,
12. SUBROUTINE ABSØRB which
- is called from QRAD,  
 receives
  - α) total hemispherical emittance  $\epsilon$ , see Eq. 6.5
  - β) temperature distributions on the fin and tube surfaces
  - γ) exchange function SS

provides                       $\delta$ ) functions  $x_{ij}$  and  $x_{ijk}$  defined by Eqs. 6.10 and 6.11,  
 calls                            adsorptance matrix  $\alpha_{ij}$  as defined by Eq. 6.9,  
                                   $\alpha$ ) SHAPEF for SS  
                                   $\beta$ ) EMIT for (T)  
                                   $\gamma$ ) AVGEMT for  $x_{ij}$  and  $x_{ijk}$   
                                   $\delta$ ) DEFINT for integration,

13. SUBROUTINE QRAD which

is called from                DERIVM,  
 receives                       $\alpha$ ) transfer matrix and excitation vector  $M_{ij}$  and  $P_j$ , respectively,  
 provides                       $\alpha$ ) inversion of transfer matrix  
                                   $\beta$ ) solution of radiosity equations  
                                   $\gamma$ ) grid mapping,  
 calls                             $\alpha$ ) TRMATX  
                                   $\beta$ ) EXITAV  
                                   $\gamma$ ) YINT, INTERP,

d) the program set used to calculate the aerodynamic heating during ascent and reentry, consisting of

14. FUNCTION ALTVEL (TIME, IOPTN, AVA, AVR, TA, TR, NA, NR) which interpolates user-supplied altitude and velocity profiles for a given instant, and

is called from                CONVEC,  
 receives                       $\alpha$ ) current real time, TIME, in seconds, measured from start of transition phase, i.e., ascent: TIME = 0, ALTVEL = AVA (1)  
                                  reentry: TIME = 0, ALTVEL = AVR (1)  
                                   $\beta$ ) ordered pairs of time and altitude (TA,AVA) and (TR,AVR) for ascent and reentry, respectively  
                                   $\gamma$ ) NA and NR, the number of ordered pairs and elements in TA,AVA and TR,AVR, respectively



- 6) IØPTN = 0 for ascent  
       = 1 for reentry,
- provides                    data interpolation,  
 calls                      YINT for interpolation,
15. SUBROUTINE ATMØS (ELEV, TATM, CATM) which computes the atmospheric temperature and the speed of sound as functions of altitude and
- is called from            CØNVEC,
- receives                α) current shuttle elevation, ELEV, in feet
- provides                atmospheric temperature, TATM in degrees R and speed of sound, CATM, in ft/sec,
- calls                    YINT for interpolation, PØLY for evaluation of polynomials,
16. SUBROUTINE REFP (ELEV, REFTP, REFPR, REFVIS, REFRHØ, REFK, REFCP, REFGAM) which computes the atmospheric air properties as a function of elevation and of high speed reference temperature and
- is called from            CØNVEC,
- receives                α) current orbiter elevation, ELEV, in feet  
                           β) current high speed reference temperature REFTP, in degrees R,
- provides                atmospheric Prandtl number REFPR (dimensionless), dynamic viscosity REFVIS in lbm/(ft sec), density REFRHØ in lbm/ft<sup>3</sup>, thermal conductivity REFK in Btu/(sec ft R), specific heat at constant pressure REFCP in Btu/(lbm/R) and the ratio of specific heats REFGAM (dimensionless),
- calls                    α) YINT for interpolation  
                           β) PØLY for polynomial expansion,
17. SUBROUTINE NUS (MACHNØ, TATM, CATM, TIN, PRATM, VISATM, RHØATM, STAGX, VERTX, NSRAD, NUS1) which provides a single Nusselt number for the calculation of the aerodynamic heating of the orbiter radiator system at the current time and
- is called from            CØNVEC,
- receives                α) current orbiter Mach number MACHNØ (dimensionless)  
                           β) current atmospheric temperature TATM, in degrees R

- γ) current atmospheric velocity of sound, CATM, in ft/sec
- ε) temperature of the coolant fluid at the inlet plane TIN, in degrees R
- ζ) atmospheric Prandtl number PRATM, evaluated at the high speed reference temperature (dimensionless)
- η) atmospheric density RHØATM, evaluated at the high speed reference temperature in lbm/ft<sup>3</sup>
- θ) distance from the stagnation point on the shuttle vehicle to the midpoint of the radiator panel measured along a streamline, STAGX, in feet
- ι) overall dimension of the radiator panel measure in a direction parallel to the acceleration of gravity, VERTX, in feet
- κ) integer value indicating number of non-adiabatic sides of the radiator panel, NSRAD, i.e., NSRAD = 1 for single non-adiabatic surface or NSRAD = 2 for both sides of radiator being non-adiabatic,

provides mean Nusselt number NUS1, for meteoroid layer and fin at current orbiter elevation during either ascent or reentry (dimensionless),

calls YINT for interpolation,

18. SUBROUTINE CØNVEC (TIME) which computes normalized aerodynamic heating of fin and meteoroid protection layer for both ascent and reentry phases of orbiter operation and

is called from DERIVM,

receives α) current real time TIME, in seconds, measured from start of transition phase; i.e., TIME must be measured with respect to same reference point as user supplied time arrays in FUNCTION ALTSH and VELSH,

provides normalized convective flux from fin surface CØNFN (I,J), and from meteoroid protection surface CØNMP (I),

calls α) ALTVEL for altitude and velocity of orbiter  
 β) ATMØS for atmospheric properties  
 γ) REFP for atmospheric properties evaluated at high speed reference temperature

δ) YINT for interpolation

ε) NUS for Nusselt number,

19. FUNCTION CPAIR (T) which computes the specific heat at constant pressure for atmospheric air and

is called from            ATMØS, REFTP and CØNVEC,  
 receives                absolute temperature T in R,  
 provides                specific heat at constant pressure in  
                           Btu/lbmR,  
 calls                    PØLY for polynomial expansion,

20. FUNCTION ENTAIR (T) which computes the enthalpy of atmospheric air and

is called from            CØNVEC,  
 receives                absolute temperature T in R,  
 provides                enthalpy in Btu/lbm,

21. FUNCTION TNH (ENT) which computes the temperature of atmospheric air for a given enthalpy and

is called from            CØNVEC,  
 receives                enthalpy ENT in Btu/lbm,  
 provides                absolute temperature in R,  
 calls                    α) CPAIR for the specific heat at constant  
                           pressure of atmospheric air  
                           β) ENTAIR for the enthalpy of air,

- e) the program to compute the thickness of the meteoroid protection layer

22. FUNCTION TK (GAMMA, A, BETA, DENSM, THETA, PHI, AN ALPHA, VELM, PØ, TAU, DENST, W, TNN, AMAN, TIN, RØUT) which computes the thickness of the meteoroid protection layer for given environmental conditions, tube and manifold areas, experimental constants and protection layer properties and

is called from            MAIN,  
 receives                α) nondimensional experimental constant,  
                           GAMMA, see Eq. 8.9

- β) nondimensional experimental constant A, see Eq. 8.9
  - γ) nondimensional constant which relates meteoroid flux to mass, BETA
  - δ) density of the meteoroid particle, DENSM, in gm/cm<sup>3</sup>
  - ε) nondimensional experimental constant THETA, see Eq. 8.9
  - ζ) nondimensional experimental constant PHI, see Eq. 8.9
  - η) nondimensional constant AN, used to describe penetration depth as a function of angle of incidence of meteoroid particle
  - θ) velocity of meteoroid particle relative to radiator panel, VELM, in ft/sec
  - ι) probability of no damage caused by impact of meteoroid, PØ
  - κ) time TAU, the radiator panel is exposed to meteoroid environment, in days
  - λ) density of protection material DENST, in lbm/ft<sup>3</sup>
  - μ) axial length of single flow channel exposed to meteoroid environment, W, in inches
  - ν) integer number of flow channels, TNN,
  - ξ) area of the manifold that is exposed, AMAN, in ft<sup>2</sup>
  - φ) temperature of coolant fluid at inlet plane of flow channel, TIN, in degrees R
  - π) outside radius RØUT of the unprotected flow channel, in inches,
- provides meteoroid protection thickness TK, in inches,
- calls ELAS for modulus of elasticity of protection material,

f) the program set which calculates the location of the adiabatic plane between two flow channels under conditions of non-symmetrical loading of the tubes and for curved radiator panels, consisting of

23. SUBROUTINE SHADE which zeros those elements in the array Q which correspond to portions of the radiator panel that do not receive solar, albedo and planetary irradiation and

is called from TCALC and QINCID,  
 receives  $\alpha$ ) NS, number of flat sides to the polygon  
 used as a reference body by MRI program  
 for calculating incident radiant flux  
 $\beta$ ) radiant flux array Q in Btu/hr ft<sup>2</sup>.  
 Array must have NS or greater values,  
 provides zeros in part of the Q array which correspond  
 to fin elements in the shade,

24. SUBROUTINE TCALC which calculates the equivalent sink temperatures  
 from incident flux data and,

is called from MAIN,  
 receives NTF the number of flat fin segments.

provides TSTAR the equivalent sink temperature in R,  
 calls SHADE for incident flux data,

25. SUBROUTINE EFFICY which evaluates the one-dimensional fin  
 efficiency and its derivative as a function of the fin base  
 temperature and

is called from TTIPS,  
 receives  $\alpha$ ) the dimensionless conductance parameter NC  
 $\beta$ ) the derivative of the conductance parameter  
 with respect to the base temperature NCPR  
 in R<sup>-1</sup>  
 $\gamma$ ) the ratio of the sink to base temperature  
 TS/TB  
 $\delta$ ) the fin base temperature TB in R,  
 provides  $\alpha$ ) the fin efficiency ETA  
 $\beta$ ) the derivative of the fin efficiency with  
 respect to the base temperature ETAPR in  
 R<sup>-1</sup>,  
 calls POLY for polynomial expansion,

26. SUBROUTINE TTIP which computes the tip temperature of a one-  
 dimensional fin and

is called from ADIABH,  
 receives  $\alpha$ ) the dimensionless conductance parameter NC

- provides                       $\beta$ ) the ratio of the sink to base temperature  $TS\phi TB$ ,  
                                  the ratio of the difference between the fin tip temperature and sink temperature to the difference between the base and sink temperature,  
 calls                             $P\phi LY$  for polynomial expansion,
27. FUNCTION NUSA which evaluates the Nusselt number for the coolant fluid flow for both laminar and turbulent flow and  
 is called from                 $ADIABH$  and  $TTIPS$ ,  
 receives                       $\alpha$ ) the Reynold number for the fluid  $REY$   
                                   $\beta$ ) the Prandtl number for the fluid  $PR$   
                                   $\gamma$ ) the ratio of the internal tube diameter to tube length  $D\phi L$ ,  
 provides                      the fluid Nusselt number,
28. SUBROUTINE TTIPS which uses the Newton-Raphson method to numerically solve for the base temperature of the one-dimensional fin and  
 is called from                 $ADIABH$ ,  
 receives                       $\alpha$ ) the integer  $IT$  used to specify the tube number for which the fin tip temperatures are calculated. Numbering system is consistent with the input parameters  $TIN$  (inlet temperature),  $PIN$  (inlet pressure),  $T$  (fin thickness),  $AL$  (tube length) and  $M$  (mass flow rate)  
                                   $\beta$ ) the integer  $ITST$  which designates the sink temperature data stored in the array  $TSTAR$   
                                   $\gamma$ ) the distance  $H1$  to the adiabatic plane for the fin attached to the left of the tube number  $IT$ , in inches  
                                   $\delta$ ) the distance  $H2$  to the adiabatic plane for the fin attached to the right of the tube number  $IT$ , in inches,  
 provides                       $\alpha$ ) the adiabatic plane temperature for the fin attached to the right of tube  $IT$  in  $R$   
                                   $\beta$ ) the adiabatic plane temperature for the fin attached to the left of tube  $IT$  in  $R$ ,  
 calls                             $\alpha$ )  $NUS$  for the coolant fluid Nusselt number  
                                   $\beta$ )  $EFFICY$  for fin efficiency  
                                   $\gamma$ )  $TTIP$  for the fin tip temperature,

29. SUBROUTINE ADIABH which determines the position of the adiabatic planes so that there is a continuity in adiabatic plane temperature of the one-dimensional fin and so that the heat convected from the fluid equals the heat radiated from the fin and
- is called from           MAIN,
- receives                the integer ITST which designates sink temperature data stored in the array TSTAR,
- provides                printout of table headings and system parameters when radiator is non-symmetrically loaded and diagnostic printout if distance to adiabatic plane exceeds the spacing between the tubes. For this case one tube is transferring heat to the fin while the adjacent tube is gaining heat. TTIPS also provides the distances to all the adiabatic planes within the non-symmetrical radiator section,
- calls                   α) property subroutines for fin and fluid properties
- β) NUS for the coolant fluid Nusselt number
- γ) TTIPS for the tip temperature of the fin
- δ) PDERIV for determination of the elements of the matrix  $P_{ij}$
- ε) FMINV for matrix inversion,
30. FUNCTION PDERIV which calculates the partial derivative  $\partial \delta_i / \partial h_j$  (see Section II-14 of the System Analysis Manual) and
- is called from           ADIABH,
- receives                α) the integer J denotes the tube under consideration,  $J = -1$  denotes tube IT - 1;  $J = 0$  denotes tube IT and  $J = +1$  denotes tube IT + 1
- β) the integer IT which specifies the tube number to be considered
- γ) the integer ITST which designates sink temperature data stored in the array TSTAR,
- provides                the partial derivative  $\partial h_i / \partial h_j$
- calls                   TTIPS for evaluation of the temperature at the adiabatic plane,

g) the program set consisting of all the mathematical procedures discussed in detail in Part III of the System Analysis Manual

31. SUBROUTINE RKSF (see Section III.15) which

is called from            FLSTRT, for dynamic fluid flow field,  
calls                        DERIVL, CNTLN,

32. SUBROUTINE RKS (see Section III.15) which

is called from            MAIN, for principal integration,  
calls                        DERIVM, CNTLM,

33. FUNCTION POLY (X,A,M) (see Section III.16),

34. FUNCTION YINT (X,Y,M,N,P) (see Section III.17),

35. SUBROUTINE DDX (Y,DY,DX,N),

(see Section III.18)

36. SUBROUTINE D2DX2 (Y,D2Y,DX,N),

37. FUNCTION DEFINT (Y,DX,N),

(see Section III.19)

38. SUBROUTINE FINT (Y,YO,DX,N,F),

39. SUBROUTINE INTERP (NX1,MZ1,XX1,ZZ1,YY1,NX2,MZ2,XX2,ZZ2,YY2) maps  
a two-dimensional function YY1 from one grid (XX1,ZZ1) onto  
another grid (XX2,ZZ2) and

is called from            QRAD,

receives            α) the number of nodal points (NX1,MZ1) of  
the original grid

β) the function YY1

γ) the number of nodal points (NX2,MZ2) of  
new grid,

provides            the function YY2 at the new grid,

calls                YINT,

is restricted to        NX1,MZ1,NX2,MZ2 ≤ 10,

40. FUNCTION DEFNT serves to integrate by the trapezoidal rule and  
is used where random error accumulation is more critical than  
truncation errors, and V accepts through its argument list  
(Y, DX, N)



- $\alpha$ ) the array Y of N equally spaced ordinates
  - $\beta$ ) the interval DX
  - $\gamma$ ) the integer constant N, representing the number of nodal points.
- returns the integral of YCS) as DEFNT.

41. SUBROUTINE MTXINV (N,M) performs two related tasks:

- (i) to solve a system of N linear algebraic equations, when  $M = N + 1$ ,
- (ii) to invert an  $N \times N$  invertible matrix, when  $M = 2N$ .

For task (i) it

accepts through COMMON TRMTX (k,1) the augmented coefficient matrix (see Chapter III.7). Here k and 1 represent integer constants,  $k = N$ ,  $1 = N + 1$ .

through the argument list (N,M) the rank N of the coefficient matrix and  $M = N + 1$ , two integer constants.

returns the solution in TRMTX (I,J),  $I = 1, \dots, k$ ,  $J = k$ .

For task (ii) it

accepts through COMMON TRMTX (k,1)

the  $k \times k$  matrix in the first k columns of TRMTX (I,J),  $I = 1, \dots, k$ ;  $J = 1, \dots, k$  and the  $k \times k$  identity matrix in the second k columns of TRMTX (I,J)  $I = 1, \dots, k$ ;  $J = k + 1, \dots, 2k$ . Hence  $1 = 2k$ .

through the argument list (N,M) the rank N of the coefficient matrix and  $M = 2N$ , both integer constants.

returns the inverted matrix in the second k columns of TRMTX (I,J),  $I = 1, \dots, k$ ;  $J = k + 1, \dots, 2k$ .

42. SUBROUTINE FMINV serves to solve N linear algebraic equations by the method used for SUBROUTINE MTXINV. It's calling provisions are designed to accommodate the requirements of ADIABH. It

receives, through its argument list (A,X,N,M)

$\alpha$ ) the (N x N), invertible coefficient matrix A

$\beta$ ) the know, N-dimensional vector X

$\gamma$ ) the rank (N) of A, an integer constant

$\delta$ )  $M = N + 1$

where  $N \leq 25$

returns the solution  $Y = A^{-1}X$  placed in the array X

is called from ADIABH.

The Selective Part contains five groups of thermophysical properties, with the total of 20 program units. Each group constitutes a property package, one each for

- (i) the coolant fluid (8 program units)
- (ii) the flow channel material (3 program units)
- (iii) the meteoroid protection layer (4 program units)
- (iv) the fin material (3 program units)
- (v) the thermal control coating (2 program units).

The choice of a particular material combination must be reflected in the corresponding data specification of the Input Data Set, (see Section 3f of Chapter A).

- a) The coolant property subprogram set consists of eight elements, 43 through 50
- 43. FUNCTION RHOF (P,T) which computes the fluid density RHOF in slug/ft<sup>3</sup> as a function of pressure P in lbf/ft<sup>2</sup>, and absolute temperature T in R,
  - 44. FUNCTION BETA (RHOF,T) which computes the isobaric expansion coefficient  $\beta$ , Section II.C.11, in 1/R as a function of density RHOF in slug/ft<sup>3</sup>, and temperature T in R,
  - 45. FUNCTION CAPPA (P,T) which computes the isothermal compressibility, Eq. 11.3, in ft<sup>2</sup>/lbf, as a function of pressure P in lbf/ft<sup>2</sup>, and temperature T in R,
  - 46. FUNCTION CPF (RHOF,T) which computes the specific heat at constant pressure CPF, in Btu/(slug R), as a function of density RHOF in slug/ft<sup>3</sup>, and temperature T, in R,
  - 47. FUNCTION HFL (RHOF,T) which computes the fluid enthalpy HFL, in Btu/slug, as a function of density RHOF in slug/ft<sup>3</sup>, and temperature T, in R,
  - 48. FUNCTION VISC (RHOF,T) which computes the fluid dynamic viscosity VISC, in slug/(ft sec) as a function of density RHOF in slug/ft<sup>3</sup>, and temperature T, in R,

49. FUNCTION THCF (RHØ,T) which computes the fluid thermal conductivity THCF in Btu/(hr ft R) as a function of density RHØ in slug/ft<sup>3</sup>, and temperature T, in R,
50. FUNCTION PF (RHØ,T) which computes the fluid pressure PF in lbf/ft<sup>2</sup> as a function of density RHØ in slugs/ft<sup>3</sup>, and absolute temperature T, in R. For specific details of the coolant fluid properties, see Section II.11 and 12 and Appendix B of the System Analysis Manual.

b) The coolant channel material property subprogram set consists of three elements, 51 through 53.

51. FUNCTION THCTB (T) computes the thermal conductivity THCTB in Btu/(hr ft R), of the tube wall material as a function of the absolute temperature T, in R,
52. FUNCTION DTHCTB (T) computes the relative change of thermal conductivity k with temperature T, namely  $(dk_w/dT)/k_w$  in 1/R, as a function of temperature T, in R,
53. FUNCTION CPTB (T) computes the specific heat at constant pressure CPTB, in Btu/(slug R) as a function of T, in R.

For specific details of the coolant channel properties, see Appendix A of the System Analysis Manual.

c) the meteoroid protection layer material properties are coded in a set of four elements, 54 through 57

54. FUNCTION THCMP (T),
55. FUNCTION DTHCMP (T),
56. FUNCTION CPMP (T),
57. FUNCTION ELAS (T) computes the modulus of elasticity ELAS in lbf/in<sup>2</sup> as a function of temperature T, in R.

For specific details of the meteoroid protection properties, see Appendix A of the System Analysis Manual.

d) The fin material properties required are coded in three FUNCTION subprograms, numbered 58 through 60

58. FUNCTION THCFN (T) computes the thermal conductivity THCFN, in Btu/(hr ft R) for the fin material as a function of absolute temperature T, in R,
59. FUNCTION DTHCFN (T) computes the relative change of thermal conductivity, namely  $(dk_f/dT)/k_f$  in 1/R, as a function of temperature T, in R,
60. FUNCTION CPFN (T) computes the specific heat at constant pressure CPFN, in Btu/(slug R) as a function of temperature T, in R.

For specific details of the fin material properties, see Appendix A of the System Analysis Manual.

- e) Optical properties of the thermal control coating are coded in two subprograms, numbered 61 and 62

61. FUNCTION EMIT (T) computes the total hemispherical emittance in accordance with Eq. 6.5 of the System Analysis Manual as a function of the surface temperature T in R, and is called by ABSORB and EXITAV,
62. SUBROUTINE AVGEMT (T,XX,XXX,N) evaluates the expressions for  $x_{ij}$  and  $x_{ijk}$  given as Eqs. 6.10 and 11 in the System Analysis Manual for N interacting surface elements. It is called by ABSORB (see program No. 12).

For specific details of surface coating properties, see Appendix C of the System Analysis Manual.

This completes the Source Deck discussion. Recall that the Source Deck composition implies a particular selection of coolant fluid and structural materials.

#### 4. Program Listing

This section contains the entire program listing in alphabetical order of subprogram name.

Labelled common blocks and external program references and printed out prior to the listing of each program unit.

All program units have been documented with comment cards for the convenience of the user.

All program units are summarized in a table of contents which follows the final unit listing. The table of contents lists all program elements as either symbolic or relocatable and it also prints the time and date when the program element was most recently stored on FASTRAND drum.

\*\*\*\*\* ABSORB \*\*\*\*\*

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DEFOR,S ME\*NASA5.ABSORB,ME\*NASA5.ABSORB  
FOR S9A-07/13/72-20:52:53 (0,)

SUBROUTINE ABSORB ENTRY POINT 000552

STORAGE USED: CODE(1) 000600; DATA(0) 000115; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 GRD 003721  
0004 ABSRST 000601  
0005 AVGABS 000251

EXTERNAL REFERENCES (BLOCK, NAME)

0006 EMIT  
0007 DEFNT  
0010 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000032	1166	0001	000042	1236	0001	000062	1326	0001	000065	1356	0001	000067	1406
0001	000137	1526	0001	000155	1616	0001	000156	1646	0001	000244	1746	0001	000251	1776
0001	000260	2026	0001	000271	2066	0001	000352	2206	0001	000355	2236	0001	000357	2266
0001	000430	2406	0004 R	000000	ALPHFN	0004 R	000226	ALPHMP	0000 R	000000	AUX1	0000 R	000005	AUX2
0000 R	000012	AUX3	0000 R	000017	AUX4	0003 R	003465	CSF	0007 R	000000	DEFNT	0003 R	003464	DXX2
0003	003472	DXX21	0003	003463	DZMFN	0003 R	003410	EBFN	0003 R	003441	EBMP	0006 R	000000	EMIT
0004 R	000461	EMITFN	0004 R	000512	EMITMP	0003 R	003467	EXTIFN	0003 R	003471	EXTIMP	0003 R	003466	EXTSFN
0003 R	003470	EXTSMP	0004	000517	EXTVCT	0000 I	000032	I	0000 I	000037	IL	0000	000057	INJP5
0000 I	000034	J	0000 I	000040	JL	0000 I	000036	K	0000 I	000043	L	0003	003461	LCT
0003	003462	LTT	0003	003460	MCVRD	0003 R	003473	SS	0003 R	003670	SSTT	0005 R	000100	TFIN
0000 R	000035	TFIN	0000 R	000033	TM	0005 R	000244	TPR	0003	000000	TRMTX	0000 R	000041	X
0005 R	000062	XXFN	0005 R	000000	XXMP	0005 R	000071	XXXFN	0005 R	000031	XXXMP	0003	003446	XX2
0000 R	000042	Y	0000 R	000026	ZCF	0000 R	000031	ZCM	0000 R	000024	ZF	0000 R	000025	ZFM
0000 R	000027	ZM	0000 R	000030	ZMM	0003	003453	ZZ2						

```

00101      1*      SUBROUTINE ABSORB (TO,XCFN,XCMP,XXCFN,XXCMP)
00101      2*      C
00101      3*      C THIS SUBROUTINE COMPUTES :
00101      4*      C THE ABSORBTANCE MATRIX REPRESENTING THE MAIN ABSORBTANCE AFTER
00101      5*      C INTERNAL INTEGRATIONS
00101      6*      C
00103      7*      COMMON /GRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZ2(5),
00103      8*      1 MCVRD,LCT,LTT,DZMFN,
00103      9*      2 DXX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DXX21,SS(5,5,5)
00103     10*      3 ,SSTT(5,5)
00104     11*      COMMON /ABSRST/ALPHFN(5,5,6),ALPHMP(5,31),EMITFN(5,5),EMITMP(5),
00104     12*      1 EXTVCT(50)
00105     13*      COMMON/AVGABS/XXMP(5,5),XXMP(5,5),XXFN(7),XXFN(7),
00105     14*      1 TFIN(10,10),TPR(5)

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\*\*\*\*\* ABSORB \*\*\*\*\*

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00106 15* DIMENSION AUX1(5),AUX2(5),AUX3(5),AUX4(5)
00106 16* C
00107 17* ZF = EXTSMF+EXTIFN
00110 18* ZFM = ZF*(1-XCFN)
00111 19* ZCF = ZF*(XCFN-XXCFN)
00112 20* ZM = EXTSMF+EXTIMP
00113 21* ZMM = ZM*(1-XCMP)
00114 22* ZCM = ZM*(XCMP-XXCMP)
00114 23* C
00114 24* C FIN
00114 25* C
00115 26* DO 10 I=1,5
00120 27* TM = T0*TPR(I)
00121 28* EMITMP(I) = EMIT(TM)
00122 29* DO 10 J=1,5
00125 30* TFINN = T0*TFIN(J,I)
00126 31* 10 EMITFN(J,I) = EMIT(TFINN)
00131 32* DO 60 K=1,5
00134 33* DO 30 IL=1,5
00137 34* DO 20 JL=1,5
00142 35* X = CSF*SS(JL,IL,K)*EBFN(IL,JL)
00143 36* AUX1(JL) = X*(XXMP(IL,JL)-XXXMP(IL,JL))
00144 37* 20 AUX3(JL) = X*(EMITFN(IL,JL)-XXMP(IL,JL))
00146 38* AUX2(IL) = DEFNT (AUX1,DXX2,5)+AUX1(1)*DXX2/2.0
00147 39* 30 AUX4(IL) = DEFNT (AUX3,DXX2,5)+AUX3(1)*DXX2/2.0
00151 40* DO 35 IL=1,5
00154 41* Y = SSTT(IL,K)*EBMP(IL)
00155 42* AUX1(IL) = Y*(XXFN(IL)-XXXFN(IL))
00156 43* 35 AUX3(IL) = Y*(EMITMP(IL)-XXFN(IL))
00160 44* DO 60 I=1,5
00163 45* DO 60 J=1,5
00166 46* ALPHFN(I,J,K) = (XXFN(K)*EBMP(K)+DEFNT(AUX1,0.25,5)+DEFNT(AUX2,
00166 47* 1 0.25,5)+ZCM)/(EMITMP(K)*EBMP(K)+DEFNT(AUX3,0.25,
00166 48* 2 5)+DEFNT(AUX4,0.25,5)+ZMM)
00166 49* C
00166 50* C
00167 51* 60 ALPHFN(I,J,6) = XCFN
00167 52* C
00167 53* C CHANNEL
00167 54* C
00173 55* DO 120 K=1,5
00176 56* DO 90 I=1,5
00201 57* DO 90 J=1,5
00204 58* L = J+5*(I-1)
00205 59* DO 80 JL=1,5
00210 60* X = SS(I,J,JL)*EBMP(JL)
00211 61* AUX1(JL) = X*(XXFN(JL)-XXXFN(JL))
00212 62* 80 AUX3(JL) = X*(EMITMP(JL)-XXFN(JL))
00212 63* C
00214 64* 90 ALPHMP(K,L) = (XXMP(J,I)*EBFN(J,I)+DEFNT(AUX1,0.25,5)+ZCF)/
00214 65* 1 (EMITFN(J,I)*EBFN(J,I)+DEFNT(AUX3,0.25,5)+ZFM)
00214 66* C
00217 67* DO 95 I=1,5
00222 68* DO 93 IL=1,5
00225 69* DO 92 JL=1,5
00230 70* X = CSF*SS(JL,IL,K)*EBFN(IL,JL)
00231 71* AUX1(JL) = X*(XXMP(IL,JL)-XXXMP(IL,JL))

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00232 72* 92 AUX3(JL) = X*(EMITFN(IL,JL)-XXMP(IL,JL))
00234 73* AUX2(IL) = DEFNT (AUX1,DXX2,5)+AUX1(1)*DXX2/2.0
00235 74* 93 AUX4(IL) = DEFNT (AUX3,DXX2,5)+AUX3(1)*DXX2/2.0
00237 75* DO 94 IL=1,5
00242 76* Y = SSTT(IL,I)*EBMP(IL)
00243 77* AUX1(IL) = Y*(XXFN(IL)-XXXFN(IL))
00244 78* 94 AUX3(IL) = Y*(EMITMP(IL)-XXFN(IL))
00246 79* 95 ALPHMP(K,25+I) = (XXFN(I)*EBMP(I)+DEFNT(AUX1,0.25,5)+DEFNT(AUX2,
00246 80* 1 0.25,5)+ZCM)/(EMITMP(I)*EBMP(I)+DEFNT(AUX3,0.25,
00246 81* 2 5)+DEFNT(AUX4,0.25,5)+ZMM)
00246 82* C
00250 83* 120 ALPHMP(K,31) = XCMP
00252 84* RETURN..
00253 85* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG.P \*\*\*\*\* ADIABH \*\*\*\*\*

\*\*\*\*\* ADIABH \*\*\*\*\*

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QFOR: S ME\*NASA5.ADIABH, ME\*NASA5.ADIABH  
FOR S9A-07/13/72-20:52:57 (0,)

SUBROUTINE ADIABH ENTRY POINT 000506

STORAGE USED: CODE(1) 000522; DATA(0) 000451; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 ADBH 000352

EXTERNAL REFERENCES (BLOCK, NAME)

0004 NUSA  
0005 RHOF  
0006 VISC  
0007 CPF  
0010 THCF  
0011 THCFN  
0012 EMIT  
0013 TTIPS  
0014 PDERIV  
0015 FMINV  
0016 NPRTS  
0017 NI02S  
0020 NSTOPS  
0021 EXP  
0022 NWDUS  
0023 NERR3S

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000174 1005F	0001 000004 1136	0001 000074 1316	0000 000373 15F	0001 000124 1506
0001 000161 1576	0001 000167 1656	0001 000201 1736	0001 000271 2056	0001 000326 2316
0001 000401 2556	0001 000420 2636	0001 000445 2766	0000 000316 5F	0001 000077 55L
0001 000115 60L	0001 000322 76L	0001 000346 81L	0001 000336 815L	0001 000356 82L
0001 000362 86L	0001 000433 89L	0001 000441 895L	0000 R 000000 A	0003 R 000001 AL
0003 R 000161 CONFL	0003 R 000173 CONFNA	0003 R 000147 CP	0007 R 000000 CPF	0003 R 000000 D
0000 R 000144 DHAB	0003 R 000326 DTL	0003 R 000244 EFF	0003 R 000205 EMIS	0012 R 000000 EMIT
0003 R 000013 H	0003 R 000231 HAB	0000 R 000170 HABR	0000 I 000156 I	0000 I 000163 ICNT
0000 000425 INJPS	0000 I 000164 IT	0000 I 000173 J	0000 I 000171 J1	0000 I 000172 J2
0003 R 000064 M	0000 I 000160 NSTOP	0003 I 000076 NT	0000 I 000162 NTM1	0000 I 000161 NT1
0004 I 000000 NUSA	0014 R 000000 PDERIV	0003 R 000052 PIN	0003 R 000314 QOUT	0000 R 000157 RHO
0005 R 000000 RHOF	0003 R 000217 TB	0000 R 000167 TF	0003 000026 TH	0010 R 000000 THCF
0011 R 000000 THCFN	0003 R 000040 TIN	0003 R 000270 TL	0000 R 000165 TL1	0000 R 000166 TL2
0003 R 000340 TOUT	0003 000077 TSTAR	0006 R 000000 VISC	0003 R 000135 VSC	

00101 1\* SUBROUTINE AOIABH(ITST)  
00103 2\* PARAMETER NTP = 10  
00104 3\* PARAMETER NTP1 = NTP+1

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00105      4*      PARAMETER NTPSQ = NTP*NTP
00106      5*      REAL M
00107      6*      COMMON /ADBH/ D,AL(NTP),H(NTP1),TH(NTP),TIN(NTP),PIN(NTP),
00107      7*      1 M(NTP),NT,TSTAR(NTP,3),VSC(NTP),CP(NTP),CONFL(NTP),
00107      8*      2 CONFNA(NTP),EMIS(NTP),TB(NTP),HAB(NTP1),EFF(NTP,2),TL(NTP,2),
00107      9*      3 QOUT(NTP),DTL(NTP),TOUT(NTP)
00110     10*      DIMENSION A(NTPSQ),DHAB(NTP)
00111     11*      DEFINE U(I) = 3.1415927*CONFL(I)*AL(I)*NUSA(48.0*M(I))/(3.1415927*
00111     12*      1D*VSC(I))*VSC(I)*CP(I)/CONFL(I)/D/(12*AL(I))/(M(I)*CP(I))
00112     13*      DO 10 I = 1,NT
00115     14*      RHO = RHOF(PIN(I),TIN(I))
00116     15*      VSC(I) = VISC(RHO,TIN(I))
00117     16*      CP(I) = CPF(RHO,TIN(I))
00120     17*      CONFL(I) = THCF(RHO,TIN(I))
00121     18*      CONFNA(I) = THCFN(TIN(I))
00122     19*      10 EMIS(I) = EMIT(TIN(I))
00124     20*      NSTOP=20
00125     21*      NT1=NT+1
00126     22*      NTM1=NT-1
00127     23*      ICNT=1
00130     24*      DO 2 IT=1,NT1
00133     25*      2 HAB(IT)=H(IT)
00135     26*      55 CONTINUE
00136     27*      ICNT=ICNT+1
00137     28*      IF(ICNT.LE.NSTOP)GO TO 60
00141     29*      PRINT 1005,NSTOP
00144     30*      1005 FORMAT('PROGRAM STOPPED IN SUBROUTINE ADIABH BECAUSE THE DISTANCE
00144     31*      > TO AN',/,,' ADIABATIC PLANE FAILED TO CONVERGE WITHIN THE ALLOWABL
00144     32*      >E RANGE OF
00144     33*      >',/,,' VALUES(2.5% AND 97.5% OF TOTAL FIN WIDTH) AFTER',I3,', ITERAT
00144     34*      >IONS.',/,,' THIS IS PROBABLY BECAUSE THE INPUT CONDITIONS DO NOT AL
00144     35*      >LOW A SOLUTION',/,,' IN THIS RANGE. HOWEVER THE NUMBER OF ITERATION
00144     36*      >S CAN BE INCREASED ',/,,' BY ASSIGNING A LARGER VALUE TO NSTOP IN S
00144     37*      >UBROUTINE ADIABH.')
00145     38*      STOP
00146     39*      60 CONTINUE
00147     40*      DO 6 IT=1,NT
00152     41*      CALL TTIPS(IT,ITST,HAB(IT),H(IT+1)*2.0-HAB(IT+1),TL1,TL2)
00153     42*      TL(IT,1) = TL1
00154     43*      6 TL(IT,2) = TL2
00156     44*      DO 7 IT=1,NTM1
00161     45*      DTL(IT)=TL(IT,2)-TL(IT+1,1)
00162     46*      7 CONTINUE
00164     47*      DO 75 IT=1,NTM1
00167     48*      75 IF(ABS(DTL(IT)).GT.0.05)GO TO 76
00172     49*      DO 752 IT = 1,NT
00175     50*      TF = (TIN(IT)-TB(IT))*(1.0-EXP(-U(IT)))
00176     51*      QOUT(IT) = M(IT)*CP(IT)*TF
00177     52*      752 TOUT(IT) = TIN(IT)-TF
00201     53*      WRITE(6,5)
00203     54*      5 FORMAT(////,7X,'BULK TEMP',5X,'INCHES TO ADIABATIC PLANE',
00203     55*      1 5X,'PER CENT FIN EFFICIENCY',5X,'FIN TIP TEMP DEG R',
00203     56*      2 5X,'OUTLET TEMP',5X,'HEAT REJECTED',/,1X,'TUBE',4X,
00203     57*      3 'DEG R',11X,'LEFT',8X,'RIGHT',12X,2('LEFT',8X,'RIGHT',9X),
00203     58*      4 'DEG R',11X,'BTU/HR',/)
00204     59*      DO 100 I = 1,NT
00207     60*      HABR = H(I+1)*2.0-HAB(I+1)

```

```

00210 61* 100 WRITE(6,15) I,TB(I),HAB(I),HABR,EFF(I,1),EFF(I,2),TL(1,1),
00210 62* 1 TL(I,2),TOUT(I),QOUT(I)
00225 63* 15 FORMAT(1X,I2,3X,1PE10.4,6X,1PE10.4,2X,1PE10.4,7X,
00225 64* 1 2(1PE10.4,2X,1PE10.4,4X),1PE10.4,7X,1PE10.4)
00226 65* RETURN
00227 66* 76 CONTINUE
00230 67* DO 88 IT=1,NTM1
00233 68* IF(NT.NE.2)GO TO 815
00235 69* J1=0
00236 70* J2=0
00237 71* GO TO 86
00240 72* 815 IF(IT.NE.1)GO TO 81
00242 73* J1=0
00243 74* J2=1
00244 75* GO TO 86
00245 76* 81 IF(IT.NE.NTM1)GO TO 82
00247 77* J1=-1
00250 78* J2=0
00251 79* GO TO 86
00252 80* 82 J1=-1
00253 81* J2=1
00254 82* 86 DO 88 J=J1,J2
00257 83* 88 A(IT*NTM1-1*NTM1+J*NTM1+IT)=PDERIV(J,IT,ITST)
00262 84* DO 8 IT=1,NTM1
00265 85* 8 DHAB(IT)=-DTL(IT)
00267 86* IF(NT.NE.2)GO TO 89
00271 87* DHAB(1)=-DTL(1)/A(1)
00272 88* GO TO 895
00273 89* 89 CALL FMINV(A,DHAB,NTM1,NT)
00274 90* 895 CONTINUE
00275 91* DO 9 IT=2,NT
00300 92* HAB(IT)=HAB(IT)+DHAB(IT-1)
00301 93* HAB(IT)=MIN(HAB(IT),1.95*H(IT))
00302 94* HAB(IT)=MAX(.05*H(IT),HAB(IT))
00303 95* 9 CONTINUE
00305 96* GO TO 55
00306 97* END

```

END OF COMPILATION: NO DIAGNOSTICS.

OHDS.P \*\*\*\*\* ALTVEL \*\*\*\*\*

\*\*\*\*\* ALTVEL \*\*\*\*\*

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QFOR,S ME\*NASAS,ALTVEL,ME\*NASAS,ALTVEL  
FOR S9A-07/13/72-20:53:00 (0,)

FUNCTION ALTVEL ENTRY POINT 000151

STORAGE USED: CODE(1) 000207; DATA(0) 000016; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 YINT  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000052 1L	0001	000120 2L	0001	000124 3L	0001	000133 4L	0000 R 000000 ALTVEL
0000	000002 INJPS	0000	I 000001 M	0003	R 000000 YINT			

```
00101 1*      FUNCTION ALTVEL (TIME,IOPTN,AVA,AVR,TA,TR,NA,NR)
00101 2*      C
00101 3*      C THIS SUBROUTINE COMPUTES :
00101 4*      C ALTITUDE AND VELOCITY OF ORBITER AS A FUNCTION OF TIME FROM
00101 5*      C LAUNCH (IOPTN = 0 ) OR TIME FROM PASSING THROUGH 400,000 FEET
00101 6*      C FOR REENTRY ( IOPTN = 0 )
00101 7*      C TIME IN SECS, ALTITUDE IN FEET
00101 8*      C AND VELOCITY IN FEET/SEC
00101 9*      C
00103 10*     DIMENSION TA(NA),TR(NR),AVA(NA),AVR(NR)
00104 11*     IF (TIME .GT. TA(NA) .AND. IOPTN .EQ. 0) GO TO 3
00106 12*     IF (TIME .GT. TR(NR) .AND. IOPTN .EQ. 1) GO TO 4
00110 13*     M = 2
00111 14*     1 IF (IOPTN .EQ. 0) ALTVEL = YINT(TA,AVA,NA,M,TIME)
00113 15*     IF (IOPTN .EQ. 1) ALTVEL = YINT(TR,AVR,NR,M,TIME)
00115 16*     IF (ALTVEL .LT. 0.0 .AND. M .GT. 2) GO TO 2
00117 17*     RETURN
00120 18*     2 M = M-1
00121 19*     GO TO 1
00122 20*     3 ALTVEL = AVA(NA)
00123 21*     RETURN
00124 22*     4 ALTVEL = AVR(NR)
00125 23*     RETURN
00126 24*     END
```

END OF COMPILATION: NO DIAGNOSTICS.

QMDG,P \*\*\*\*\* ATMOS \*\*\*\*\*

\*\*\*\*\* ATMOS \*\*\*\*\*

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DFOR,S ME\*NASAS.ATMOS,ME\*NASAS.ATMOS  
FOR S9A-07/13/72-20:53:03 (0.)

SUBROUTINE ATMOS ENTRY POINT 000237

STORAGE USED: CODE(1) 000261; DATA(0) 000211; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 CPAIR  
0005 NWDUS  
0006 NIO2S  
0007 SQRT  
0010 NERR3S

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000105	10L	0001	000137	15L	0001	000070	2L	0001	000161	20L	0001	000057	2316
0001	000012	30L	0000	000116	35F	0001	000022	40L	0000	000136	45F	0001	000032	60L
0000 R	000041	A	0000 R	000055	B	0000 R	000074	CF1	0004 R	000000	CPAIR	0000 R	000107	CPATM
0000 R	000104	DH	0000 R	000100	ELEV	0000 R	000110	GAMATM	0000 R	000102	GEOH	0000 R	000101	GEOHM
0000 R	000007	GH	0000 R	000031	GT	0000 R	000000	H	0000 I	000103	I	0000	000201	INJP
0000 I	000105	J	0000 R	000077	MWT	0000 R	000113	PATM	0003 R	000000	POLY	0000 R	000075	PSL
0000 R	000112	PZ	0000 R	000115	RHOATM	0000 R	000076	RSL	0000 R	000072	RU	0000 R	000073	RUI
0000 R	000114	RZ	0000 R	000020	T	0000 R	000106	TMK	0000 R	000071	WMO	0000 R	000111	Z

00101 1\*  
00101 2\* C  
00101 3\* C  
00101 4\* C  
00101 5\* C  
00101 6\* C  
00101 7\* C  
00101 8\* C  
00101 9\* C  
00101 10\* C  
00101 11\* C  
00101 12\* C  
00101 13\* C  
00101 14\* C  
00101 15\* C  
00103 16\*  
00104 17\*  
00104 18\*  
00104 19\*  
00104 20\*  
00104 21\*  
00104 22\*  
00104 23\*

SUBROUTINE ATMOS (ELEV,TATM,CATM)

THIS SUBROUTINE COMPUTES :

PROPERTIES FOR ATMOSPHERIC AIR UP TO 528,000 FEET (100 MILES OR 161 KM)  
PROPERTIES FOR AN ALTITUDE LESS THAN 301,000 FEET TAKEN FROM  
U. S. STANDARD ATMOSPHERE FOR 1966. PROPERTIES FOR ALTITUDES BETWEEN  
301,000 FEET AND 528,000 FEET ARE TAKEN FROM APPROXIMATE ANALYSIS GIVEN  
IN THE SAME REFERENCE. ERRORS IN PRESSURE AND DENSITY ARE NO LARGER  
THAN 5 PERCENT FOR ALTITUDES GREATER THAN 301,000 FEET AND LESS  
THAN 528,000 FEET.

ELEVATION IN FEET  
TEMPERATURE IN DEG R.  
SPECIFIC HEAT IN BTU/LB\* R  
VEL OF SOUND IN FT/SEC

DIMENSION H(7),GH(9),T(9),GT(8),A(12),B(12)  
DATA H(1),H(2),H(3),H(4),H(5),H(6),H(7) /0.0,1.0,-1.5731262E-07,2.  
14656553E-14,-3.8667054E-21,6.0621354E-28,-9.5013649E-35/,GH(1),GH(  
22),GH(3),GH(4),GH(5),GH(6),GH(7),GH(8),GH(9) /0.0,11.0,20.0,32.0,4  
37.0,52.0,61.0,79.0,90.0/,T(1),T(2),T(3),T(4),T(5),T(6),T(7),T(8),T  
4(9) /288.15,216.65,216.65,228.65,270.65,270.65,252.65,180.65,180.6  
55/,GT(1),GT(2),GT(3),GT(4),GT(5),GT(6),GT(7),GT(8) /-6.5,0.0,1.0,2  
6.8,0.0,-2.0,-4.0,0.0/,WMO,RU,RUI/ 28.9644,8314.32,1545.31/

\*\*\*\*\* ATMOS \*\*\*\*\*

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```

00104 24* 7 CF1/ 1000.0/,PSL,RSL /2116.22657,0.07647438/
00154 25* DATA A(1),A(2),A(3),A(4),A(5),A(6),A(7),A(8),A(9),A(10),A(11),
00154 26* 1A(12)/
00154 27* 1 0.10E+01, 0.3533367370E-01, -0.7474788290E-03,
00154 28* 2 0.2121572232E-03, -0.1325255219E-04, 0.5344159692E-06,
00154 29* 3 -0.1322745646E-07, 0.1965359762E-09, -0.1723714966E-11,
00154 30* 4 0.8707590786E-14, -0.2341816445E-16, 0.2597772972E-19,
00154 31* 5B(1),B(2),B(3),B(4),B(5),B(6),B(7),B(8),B(9),B(10),B(11),B(12)/
00154 32* 6 0.10E+01, 0.3393495800E-01, -0.3433553057E-02,
00154 33* 7 0.5497466428E-03, -0.3228358326E-04, 0.1106617734E-05,
00154 34* 8 -0.2291755793E-07, 0.2902146443E-09, -0.2230070938E-11,
00154 35* 9 0.1010575266E-13, -0.2482089627E-16, 0.2548769715E-19/
00205 36* REAL MWT
00206 37* IF (ELEV .LT. 0.0) GO TO 30
00210 38* IF (ELEV .GT. 528000.) GO TO 40
00212 39* GO TO 60
00213 40* 30 WRITE (6,35)
00215 41* 35 FORMAT (1X,'AN ATTEMPT HAS BEEN MADE TO EVALUATE ATMOSPHERIC PROPE
00215 42* RTIES FOR NEGATIVE ALTITUDES')
00216 43* RETURN
00217 44* 40 WRITE (6,45)
00221 45* 45 FORMAT (1X,'AN ATTEMPT HAS BEEN MADE TO EVALUATE ATMOSPHERIC PROPE
00221 46* RTIES FOR AN ALTITUDE EXCEEDING 100 MILES')
00222 47* RETURN
00223 48* 60 IF (ELEV .GT. 301000.) GO TO 20
00225 49* ELEV = ELEV*0.3048
00226 50* GEOHM = POLY(7,H,ELEV)
00227 51* GEOH = GEOHM/0.3048
00230 52* DO 1 I=1,9
00233 53* DH = GH(I)-GEOHM/CF1
00234 54* IF (DH.GT.0.0) GO TO 2
00236 55* 1 CONTINUE
00240 56* 2 J = I-1
00241 57* DH = GH(J)-GEOHM/CF1
00242 58* TMK = T(J)-GT(J)*DH
00243 59* TATM = TMK*1.8
00244 60* 10 CPATM = CPAIR(TATM)
00245 61* IF (ELEV .GT. 301000.) GO TO 15
00247 62* GAMATM = CPATM/(CPATM-0.0686)
00250 63* CATM = SQRT(1.4*RU*TMK/WMO)/0.3048
00251 64* RETURN
00252 65* 15 GAMATM = CPATM/(CPATM-(1.98585/MWT))
00253 66* CATM = SQRT(GAMATM*RU*TMK/MWT)/0.3048
00254 67* RETURN
00255 68* 20 Z = ELEV/(3280.8399)
00256 69* MWT = 28.9644-0.0309491*(Z-90.0)
00257 70* PZ = POLY(12,A,Z)
00260 71* PATM = PSL/(PZ**4)
00261 72* RZ = POLY(12,B,Z)
00262 73* RHOATM = RSL/(RZ**4)
00263 74* TATM = PATM*MWT/(RHOATM*1545.31)
00264 75* TMK = TATM/1.8
00265 76* GO TO 10
00266 77* END

```

END OF COMPILATION: NO DIAGNOSTICS.

\*\*\*\*\* ATMOS \*\*\*\*\*

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QHD6:P \*\*\*\*\* AV6EMT/SCZ93 \*\*\*\*\*



\*\*\*\*\* AVGEMT/SCZ93 \*\*\*\*\*

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1

QFOR S ME\*NASA5.AVGEMT/SCZ93,ME\*NASA5.AVGEMT/SCZ93  
FOR 59A-07/13/72-20:53:05 (0.)

SUBROUTINE AVGEMT ENTRY POINT 000110

STORAGE USED: CODE(1) 000121; DATA(0) 000040; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003- AVGABS 000251

EXTERNAL REFERENCES (BLOCK, NAME)

0004 POLY  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000004	1246	0001	000024	1326	0000 R 000000 A	0000 R 000005 B	0000 I 000016 I
0000	000025	INJPS	0000	I 000014	J	0004 R 000000 POLY	0000 R 000013 TALB	0003 R 000100 TFIN
0000 R	000017	TFNX	0000 R	000015	TMPX	0003 R 000244 TPR	0000 R 000012 TSOL	0003 R 000062 XXFN
0003 R	000000	XXMP	0003 R	000071	XXXFN	0003 R 000031 XXXMP		

```

00101      1*      SUBROUTINE AVGEMT(10)
00101      2*      C
00101      3*      C THIS SUBROUTINE COMPUTES :
00101      4*      C THE NECESSARY VARIABLES FOR THE COMPUTATIONS OF THE AVERAGE
00101      5*      C SURFACE COATING EMITTANCE
00101      6*      C
00103      7*      COMMON/AVGABS/XXMP(5,5),XXXMP(5,5),XXFN(7),XXXFN(7),
00103      8*      1 TFIN(10,10),TPR(5)
00104      9*      DIMENSION A(5), B(5)
00105     10*      DATA A(1),A(2),A(3),A(4),A(5)/0.7804112E+00, -0.5527205E-04,
00105     11*      1 0.2530228E-06, -0.3229181E-09, 0.8854202E-13/
00105     12*      2 B(1),B(2),B(3),B(4),B(5)/0.6538383E+00, 0.1144374E-03,
00105     13*      3 -0.2432286E-07, -0.1437500E-09, 0.4947915E-13/
00120     14*      DATA TSOL,TALB/10400.0,480.0/
00123     15*      DO 1 J = 1,5
00126     16*      TMPX = TPR(J)*T0
00127     17*      XXFN(J) = POLY(5,A,TMPX)
00130     18*      XXXFN(J) = POLY(5,B,TMPX)
00131     19*      DO 1 I = 1,5
00134     20*      TFINX = TFIN(J,I)*T0
00135     21*      XXMP(J,I) = POLY(5,A,TFINX)
00136     22*      1 XXXMP(J,I) = POLY(5,B,TFINX)
00141     23*      XXFN(6) = POLY(5,A,TALB)
00142     24*      XXXFN(6) = POLY(5,B,TALB)
00143     25*      XXFN(7) = 0.07156
00144     26*      XXXFN(7) = 0.030562
00145     27*      RETURN

```

\*\*\*\*\* AVGENT/SC293 \*\*\*\*\*

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00146 28\* END

END OF COMPILATION: NO DIAGNOSTICS.

BMDG/P \*\*\*\*\* BETA/CFFC43 \*\*\*\*\*

\*\*\*\*\* BETA/CFFC43 \*\*\*\*\*

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PAGE 1

QFOR,S ME\*NASA5.BETA/CFFC43,ME\*NASA5.BETA/CFFC43  
FOR S9A-07/13/72-20:53:07 (0,)

FUNCTION BETA ENTRY POINT 000015

STORAGE USED: CODE(1) 000017; DATA(0) 000010; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 BETA 0000 000003 INJPS 0000 R 000001 X1 0000 R 000002 X2

```
00101 1* FUNCTION BETA(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C COEFFICIENT OF THERMAL EXPANSION AS A FUNCTION OF DENSITY
00101 5* C (SLUG/QU.FT) AND TEMPERATURE (R) OF FC-43
00101 6* C UNITS 1/R
00101 7* C
00103 8* DATA X1,X2 /157.0883,-0.076167/
00106 9* BETA = -X2/(X1+X2*T)
00107 10* RETURN
00110 11* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* BETA/CFFC75 \*\*\*\*\*

\*\*\*\*\* BETA/CFFC75 \*\*\*\*\*

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PAGE 1

QFOR/S ME\*NASA5.BETA/CFFC75,ME\*NASA5.BETA/CFFC75  
FOR S9A-07/13/72-20:53:09 (0,)

FUNCTION BETA ENTRY POINT 000015

STORAGE USED: CODE(1) 000017; DATA(0) 000010; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 BETA 0000 000003 INJP\$ 0000 R 000001 X1 0000 R 000002 X2

```
00101      1*      FUNCTION BETA(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      COEFFICIENT OF THERMAL EXPANSION AS A FUNCTION OF DENSITY
00101      5*      C      (SLUG/QU.FT) AND TEMPERATURE (R) OF FC-75
00101      6*      C      UNITS 1/R
00101      7*      C
00103      8*      DATA X1,X2 /155.522,-0.085/
00106      9*      BETA = -X2/(X1+X2*T)
00107     10*      RETURN
00110     11*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QMDG\*P \*\*\*\*\* BETA/CFHE \*\*\*\*\*

\*\*\*\*\* BETA/CFHE \*\*\*\*\*

DATE 071372

PAGE 1

DFOR S ME\*NASAS.BETA/CFHE,ME\*NASAS.BETA/CFHE  
FOR S9A-07/13/72-20:53:10 (0,)

FUNCTION BETA ENTRY POINT 000106

STORAGE USED: CODE(1) 000112; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000005 A	0000 R 000014 ALPHA	0000 R 000002 A1	0000 R 000000 BETA	0000 R 000003 B1
0000 R 000004 C	0000 R 000017 DPDRHO	0000 R 000015 DPDT	0000 R 000016 DPDV	0000 000026 INJPS
0000 R 000001 R	0000 R 000006 RHOX	0000 R 000013 TM2	0000 R 000007 TX	0000 R 000010 V
0000 R 000011 VM2	0000 R 000012 VM3			

```

00101      1*      FUNCTION BETA(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      COEFFICIENT OF THERMAL EXPANSION AS A FUNCTION OF DENSITY
00101      5*      C      (SLUG/QU.FT) AND TEMPERATURE (R) OF HELIUM
00101      6*      C      UNITS 1/R
00101      7*      C
00103      8*      DATA R=A1,B1,C,A /2077.02,136.9595,3.5002295E-03,10.000658,1.49610
00103      9*      13E-02/
00111     10*      RHOX = RHO*515.4275
00112     11*      TX   = T/1.8
00113     12*      V    = 1.0/RHOX
00114     13*      VM2   = RHOX*RHOX
00115     14*      VM3   = RHOX*VM2
00116     15*      TM2   = 1.0/(TX*TX)
00117     16*      ALPHA = C*RHOX/(TX**3)
00120     17*      DPDT  = R*(V+B1)*VM2*(1.0+2.0*ALPHA)
00121     18*      DPDV  = -R*TX*VM2*(1.0+2.0*RHOX*B1)+A1*VM3*(2.0-3.0*A*RHOX)+R*C*VM3
00121     19*      1*TM2*(2.0+3.0*B1*RHOX)
00122     20*      DPDRHO=-DPDV/VM2
00123     21*      BETA  = DPDT/(DPDRHO*RHOX*1.8)
00124     22*      RETURN
00125     23*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* BETA/CFNAK \*\*\*\*\*

\*\*\*\*\* BETA/CFNAK \*\*\*\*\*

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PAGE 1

QFOR S ME\*NASAS.BETA/CFNAK ME\*NASAS.BETA/CFNAK  
FOR S9A-07/13/72-20:53:13 (0.)

FUNCTION BETA ENTRY POINT 000015

STORAGE USED: CODE(1) 000017; DATA(0) 000010; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 BETA 0000 000003 INJP5 0000 R 000001 X1 0000 R 000002 X2

00101	1*		FUNCTION BETA(RHO,T)
00101	2*	C	
00101	3*	C	THIS FUNCTION SUBPROGRAM COMPUTES :
00101	4*	C	COEFFICIENT OF THERMAL EXPANSION AS A FUNCTION OF DENSITY
00101	5*	C	(SLUG/QU.FT) AND TEMPERATURE (R) OF NAK 78.6
00101	6*	C	UNITS 1/R
00101	7*	C	
00103	8*		DATA X1,X2 /58.773064,-0.008433/
00106	9*		BETA = -X2/(X1+X2*T)
00107	10*		RETURN
00110	11*		END

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* BETA/CFSIL \*\*\*\*\*

\*\*\*\*\* BETA/CFSIL \*\*\*\*\*

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QFOR,S ME\*NASAS.BETA/CFSIL,ME\*NASAS.BETA/CFSIL  
FOR S9A-07/13/72-20:53:15 (0,)

FUNCTION BETA ENTRY POINT 000162

STORAGE USED: CODE(1) 000177; DATA(0) 000076; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 PF  
0004 NWQJ\$  
0005 NIO2\$  
0006 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000030	1F	0000	R	000001	A1	0000	R	000022	A11	0000	R	000002	A2	0000	R	000003	A3	
0000	R	000004	A4	0000	R	000005	A5	0000	R	000000	BETA	0000	R	000006	B1	0000	R	000023	B11
0000	R	000007	B2	0000	R	000010	B3	0000	R	000011	B4	0000	R	000012	B5	0000	R	000024	C
0000	R	000013	C1	0000	R	000025	C11	0000	R	000014	C2	0000	R	000017	DT	0000		000067	INJP\$
0000	R	000026	P	0003	R	000000	PF	0000	R	000027	P1	0000	R	000020	THETA	0000	R	000021	THETA1
0000	R	000015	T0	0000	R	000016	T01												

```

00101      1*      FUNCTION BETA(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      COEFFICIENT OF THERMAL EXPANSION AS A FUNCTION OF DENSITY
00101      5*      C      (SLUG/CU.FT) AND TEMPERATURE (R) OF DOW CORNING 200
00101      6*      C      SILICON OIL (1 CS)
00101      7*      C      UNITS 1/R
00101      8*      C      TEMPERATURE .GE. 359.67 AND .LE. 859.67
00101      9*      C
00103      10*     DATA A1,A2,A3,A4,A5 /12.35,2.98333,1.1,-0.48333,0.1/,B1,B2,B3,B4,B
00103      11*     15 /-1.5,-0.01333,-1.18,0.57333,-0.1/,C1,C2 /0.7767,-0.0288/,T0,T01
00103      12*     2,DT /559.67,609.67,50.0/
00123      13*     THETA = (T-T0)/DT
00124      14*     THETA1 = (T-T01)/DT
00125      15*     A11 = (((4.0*A5*THETA+3.0*A4)*THETA+2.0*A3)*THETA+A2)*((1.0E-06)/DT
00126      16*     B11 = (((4.0*B5*THETA+3.0*B4)*THETA+2.0*B3)*THETA+B2)*((1.0E-09)/DT
00127      17*     C = C1+C2*THETA1
00130      18*     C11 = C2/DT
00131      19*     P = PF(RHO,T)
00132      20*     P1 = P/144.0-14.696
00133      21*     BETA = -C11/C-A11*P1-B11*P1*P1/2.0
00134      22*     IF (T.LT.360.67.OR.T.GT.860.67.OR.P.GT.146116.224.OR.P.GT.110116.2
00134      23*     124.AND.T.LT.460.67) WRITE(6,1) T,P
00141      24*     1 FORMAT (1H0.67HCOEFFICIENT OF THERMAL EXPANSION OF SILICON OIL, OU
00141      25*     1T OF RANGE, T = ,F10.5,6H, P = ,F15.5,/)
00142      26*     RETURN
00143      27*     END

```

\*\*\*\*\* BETA/CFSIL \*\*\*\*\*

DATE 071372

PAGE 2

END OF COMPILATION: NO DIAGNOSTICS.

BHOG:P \*\*\*\*\* CAPPA/CFFC43 \*\*\*\*\*



\*\*\*\*\* CAPPA/CFFC43 \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASAS.CAPPA/CFFC43,ME\*NASAS.CAPPA/CFFC43  
FOR S9A-07/13/72-20:53:17 (0,)

FUNCTION CAPPA ENTRY POINT 000010

STORAGE USED: CODE(1) 000012; DATA(0) 000006; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 CAPPA 0000 000001 INJP5

```
00101 1* FUNCTION CAPPA(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C ISOTHERMAL COMPRESSIBILITY AS A FUNCTION OF DENSITY
00101 5* C (SLUG/CU.FT) AND TEMPERATURE (R) OF FC-43
00101 6* C UNITS SQ.FT/LBF
00101 7* C
00103 8* CAPPA = 0.0
00104 9* RETURN
00105 10* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHD6,P \*\*\*\*\* CAPPA/CFFC75 \*\*\*\*\*

\*\*\*\*\* CAPPA/CFFC75 \*\*\*\*\*

DATE 071372

PAGE 1

DFOR:5 ME\*NASA5.CAPPA/CFFC75:ME\*NASA5.CAPPA/CFFC75  
FOR 59A-07/13/72-20:53:19 (0:)

FUNCTION CAPPA ENTRY POINT 000010

STORAGE USED: CODE(1) 000012: DATA(0) 000006: BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 CAPPA 0000 000001 INJPS

00101	1*		FUNCTION CAPPA(RHO,T)
00101	2*	C	
00101	3*	C	THIS FUNCTION SUBPROGRAM COMPUTES :
00101	4*	C	ISOTHERMAL COMPRESSIBILITY AS A FUNCTION OF DENSITY
00101	5*	C	(SLUG/CU.FT) AND TEMPERATURE (R) OF FC-75
00101	6*	C	UNITS SQ.FT/LBF
00101	7*	C	
00103	8*		CAPPA = 0.0
00104	9*		RETURN
00105	10*		END

END OF COMPILATION: NO DIAGNOSTICS.

DH06:P \*\*\*\*\* CAPPA/CFHE \*\*\*\*\*

\*\*\*\*\* CAPPA/CFHE \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASA5.CAPPA/CFHE,ME\*NASA5.CAPPA/CFHE  
FOR S9A-07/13/72-20:53:21 (0,)

FUNCTION CAPPA ENTRY POINT 000064

STORAGE USED: CODE(1) 000067; DATA(0) 000031; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000005 A	0000 R 000002 A1	0000 R 000003 B1	0000 R 000004 C	0000 R 000000 CAPPA
0000 R 000013 DPDV	0000 000022 INJP5	0000 R 000001 R	0000 R 000006 RHOX	0000 R 000012 TM2
0000 R 000007 TX	0000 R 000010 VM2	0000 R 000011 VM3		

```

00101      1*      FUNCTION CAPPA(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPRDGRAM COMPUTES :
00101      4*      C      ISOTHERMAL COMPRESSIBILITY AS A FUNCTION OF DENSITY
00101      5*      C      (SLUG/CU.FT) AND TEMPERATURE (R) OF HELIUM
00101      6*      C      UNITS SQ.FT/LBF
00101      7*      C
00103      8*      DATA R,A1,B1,C,A /2077.02,136.9595,3.5002295E-03,10.000658,1.49610
00103      9*      13E-02/
00111     10*      RHOX = RHO*515.4275
00112     11*      TX   = T/1.8
00113     12*      VM2  = RHOX*RHOX
00114     13*      VM3  = RHOX*VM2
00115     14*      TM2  = 1.0/(TX*TX)
00116     15*      DPDV =-R*TX*VM2*(1.0+2.0*RHOX*B1)+A1*VM3*(2.0-3.0*A*RHOX)+R*C*VM3
00116     16*      1*TM2*(2.0+3.0*B1*RHOX)
00117     17*      CAPPA =-47.872*RHOX/DPDV
00120     18*      RETURN
00121     19*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* CAPPA/CFNAK \*\*\*\*\*

\*\*\*\*\* CAPPA/CFNAK \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASA5.CAPPA/CFNAK,ME\*NASA5.CAPPA/CFNAK  
FOR S9A-07/13/72-20:53:25 (0.)

FUNCTION CAPPA ENTRY POINT 000052

STORAGE USED: CODE(1) 000055; DATA(0) 000032; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000017 BETA	0000 R 000000 CAPPA	0000 R 000015 CP	0000 R 000010 C1	0000 R 000011 C2
0000 R 000007 DT	0000 000023 INJPS	0000 R 000016 RHOF	0000 R 000014 THETA	0000 R 000006 T0
0000 R 000020 VEL5	0000 R 000012 V1	0000 R 000013 V2	0000 R 000001 X1	0000 R 000002 X2
0000 R 000003 X3	0000 R 000004 X4	0000 R 000005 X5		

```

00101      1*      FUNCTION CAPPA(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      ISOTHERMAL COMPRESSIBILITY AS A FUNCTION OF DENSITY
00101      5*      C      (SLUG/CU.FT) AND TEMPERATURE (R) OF NAK 78.6
00101      6*      C      UNITS SQ.FT/LBF
00101      7*      C
00103      8*      DATA X1,X2,X3,X4,X5 /0.2255,-0.016292,0.005396,-0.000758,0.000054/
00103      9*      1,T0,DT /659.67,300.0/,C1,C2 /58.773064,-0.008433/,V1,V2 /6296.9267
00103     10*      2,0.99/
00117     11*      THETA = (T-T0)/DT
00120     12*      CP = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00121     13*      RHOF = C1+C2*T
00122     14*      BETA = -C2/RHOF
00123     15*      VEL5 = V1+V2*T
00124     16*      CAPPA = (32.174/((VEL5*VEL5)+BETA*BETA*T/(CP*778.26)))/RHOF
00125     17*      RETURN
00126     18*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* CAPPA/CFSIL \*\*\*\*\*

\*\*\*\*\* CAPPA/CFSIL \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASAS.CAPPA/CFSIL,ME\*NASAS.CAPPA/CFSIL  
FOR S9A-07/13/72-20:54:12 (0,)

FUNCTION CAPPA ENTRY POINT 000132

STORAGE USED: CODE(1) 000146; DATA(0) 000064; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 PF  
0004 NWDS  
0005 NIO2\$  
0006 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000023	1F	0000 R 000016 A	0000 R 000001 A1	0000 R 000002 A2	0000 R 000003 A3
0000 R	000004	A4	0000 R 000005 A5	0000 R 000017 B	0000 R 000006 B1	0000 R 000007 B2
0000 R	000010	B3	0000 R 000011 B4	0000 R 000012 B5	0000 R 000000 CAPPA	0000 R 000014 DT
0000	000055	INJP\$	0000 R 000020 P	0003 R 000000 PF	0000 R 000021 P1	0000 R 000015 THETA
0000 R	000022	TK	0000 R 000013 T0			

```

00101      1*      FUNCTION CAPPA(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      ISOTHERMAL COMPRESSIBILITY AS A FUNCTION OF DENSITY
00101      5*      C      (SLUG/CU.FT) AND TEMPERATURE (R) OF DOW CORNING
00101      6*      C      200 SILICON OIL (1 CS)
00101      7*      C      UNITS SQ.FT/LBF
00101      8*      C      TEMPERATURE .GE. 359.67 AND .LE. 859.67
00101      9*      C
00103      10*     DATA A1,A2,A3,A4,A5 /12.35,2.98333,1.1,-0.48333,0.1/,B1,B2,B3,B4,
00103      11*     185 /-1.5,-0.01333,-1.18,0.57333,-0.1/,T0,DT /559.67,50.0/
00120      12*     THETA = (T-T0)/DT
00121      13*     A = (((A5*THETA+A4)*THETA+A3)*THETA+A2)*THETA+A1)*1.0E-06
00122      14*     B = (((B5*THETA+B4)*THETA+B3)*THETA+B2)*THETA+B1)*1.0E-09
00123      15*     P = PF(RHO,T)
00124      16*     P1 = P/144.0-14.696
00125      17*     TK = A+B*P1
00126      18*     CAPPA = TK/144.0
00127      19*     IF (T.LT.360.67.OR.T.GT.860.67.OR.P.GT.146116.224.OR.P.GT.110116.2
00127      20*     124.AND.T.LT.460.67) WRITE(6,1) T,P
00134      21*     1 FORMAT (1H0,61HISOTHERMAL COMPRESSIBILITY OF SILICON OIL, OUT OF R
00134      22*     1ANGE, T = ,F10.5,6H, P = ,F15.5,/)
00135      23*     RETURN
00136      24*     END

```

END OF COMPILATION: NO DIAGNOSTICS.

\*\*\*\*\* CAPPA/CFSIL \*\*\*\*\*

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QHDG:P \*\*\*\*\* CNTLM \*\*\*\*\*

\*\*\*\*\* CNTLM \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME NASAS.CNTLM ME NASAS.CNTLM  
FOR S9A-07/13/72-20:54:17 (0.)

SUBROUTINE CNTLM ENTRY POINT 001771

STORAGE USED: CODE(1) 002060; DATA(0) 003423; BLANK COMMON(2) 001115

COMMON BLOCKS:

0003 SRTCNV 000003  
0004 QRD 003670  
0005 QIN 001610  
0006 GEOM 000020  
0007 FLDINL 000457  
0010 DVC4FL 000002

EXTERNAL REFERENCES (BLOCK, NAME)

0011 THCFN  
0012 CPFN  
0013 DEFIN  
0014 YINT  
0015 HFL  
0016 CPTB  
0017 CPMP  
0020 FINT  
0021 SQRT  
0022 NERR2\$  
0023 AL0\$  
0024 NWOU\$  
0025 NI02\$  
0026 NI01\$  
0027 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000206 10L	0001 000471 100L	0001 000506 105L	0001 000162 1476	0001 000227 15L
0001 000253 1776	0001 000110 2L	0001 000244 20L	0001 000347 2306	0001 000311 25L
0001 000127 3L	0001 000332 30L	0001 000617 3216	0001 000632 3306	0001 000670 3436
0001 000703 3466	0001 001061 3746	0001 001255 4226	0001 001274 4266	0001 001333 4406
0001 001403 4536	0001 001437 4646	0001 000166 5L	0001 000340 50L	0001 000517 500L
0001 000522 505L	0001 000542 506L	0001 000555 507L	0000 002504 509F	0000 002554 510F
0001 001537 5116	0001 001002 513L	0001 001054 514L	0000 002605 515F	0000 003031 516F
0001 001553 5226	0000 003057 530F	0001 001562 5306	0000 003064 535F	0001 001664 543L
0001 001670 544L	0001 000405 55L	0000 003310 550F	0000 003317 555F	0000 003322 560F
0001 001653 5606	0001 001176 600L	0001 000416 65L	0001 000147 7L	0001 000421 70L
0000 R 002430 ASTR1	0000 R 002431 ASTR2	0000 R 000310 AUX1	0002 R 000562 AUX2	0000 R 000322 AUX3
0000 R 000334 AUX4	0000 R 000346 AUX5	0002 000752 COB	0002 R 001107 COH	0000 R 002446 CONDMF
0000 R 002454 COND1	0000 R 002455 COND2	0002 R 000574 CONFN	0000 R 002456 CONMF1	0002 R 000740 CONMP
0012 R 000000 CPFN	0000 R 002467 CPFN1	0017 R 000000 CPMP	0000 R 002474 CPMP1	0016 R 000000 CPTB
0000 R 002472 CPTB1	0002 001070 CPO	0004 003465 CSF	0000 R 002502 DEDOT	0000 R 002501 DEDT
0013 R 000000 DEFIN	0002 001072 DELTA	0000 R 002464 DENTH	0000 R 002503 DIFE	0002 000764 D0B
0000 R 002354 DSTFN	0000 R 002412 DSTMP	0000 R 002400 DSTTB	0000 R 000360 DT021	0000 R 000372 DT022

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0002 R 001110 DXI      0002 001053 DXIMP      0002 001052 DXITB      0004 003464 DXX2      0004 003472 DXX21
0000 R 000000 DYI      0002 R 001051 DZ      0004 003463 DZMFN      0004 003410 EBFN      0004 003441 EBMF
0003 R 000000 ELEV      0000 R 002463 ENTHE      0000 R 002467 EXTIFN      0004 003471 EXTIMP
0004 R 003466 EXTISFN      0004 003470 EXTSMF      0002 R 001103 FENTH      0002 001045 FFO      0007 R 000456 FLDINT
0006 000016 FLVASS      0002 R 001101 FLUX      0010 000001 FLUXI      0002 001041 FMACH      0000 R 002470 FNM
0002 001040 FNU      0002 001046 FOF      0002 R 000012 FP      0002 001042 FPR      0000 R 000404 FPRME
0000 R 002451 FPRM1      0000 R 002450 FPRM2      0002 001037 FR      0002 001050 FRAD      0002 001111 FRD
0002 001036 FRE      0002 R 001104 FREJ      0002 001043 FRL      0002 001044 FRM      0002 R 000036 FT
0002 R 000024 FW      0002 001113 FXHW      0002 001112 FXOH      0002 001047 FZ      0015 R 000000 HFL
0002 R 001105 HFN      0000 I 002432 I      0002 001003 IFLOW      0000 I 002465 IM2      0000 003361 INJPS
0000 I 002442 J      0000 I 002443 K      0002 001024 LC      0004 003461 LCT      0007 I 000001 LFLD
0002 I 001032 LIM      0002 I 001031 LIMWRT      0002 001011 LL1      0002 001012 LL2      0002 001013 LL3
0002 I 001014 LL4      0002 I 001015 LL5      0002 001016 LL6      0002 001017 LL7      0002 001006 LMP
0000 I 002500 LSKIP      0002 I 001004 LT      0002 I 001005 LTB      0002 001007 LTBMZ      0002 001010 LTB2MZ
0004 003462 LTT      0004 003460 MCVRD      0002 001030 MM1      0002 I 001033 MOD      0002 I 001025 MSTOTR
0002 I 000777 MZ      0002 001034 NCCZ      0002 I 001035 NCONV      0002 001002 NCTL      0002 I 001026 NCTM
0002 I 001023 NEQUS      0007 000000 NFLDTA      0002 001027 NM1      0002 I 001001 NRMP      0002 001020 NRMP1
0002 001022 NRVP2      0002 001000 NRTB      0002 001021 NRTB1      0005 001605 NSRD      0006 I 000012 NTBS
0005 001604 NTM      0002 I 000776 NX      0002 001063 PHIF      0002 001062 PHIM      0002 001071 PI
0006 R 000015 PLMASS      0002 R 001065 PO      0000 R 002445 QAERO      0005 000454 QIFN      0000 R 002425 QRED
0005 001274 QITB      0002 R 001102 GREF      0002 R 000360 QRFN      0002 R 000524 GRMP      0005 000144 QSFN
0000 R 002424 QSOLR      0005 000764 QSTB      0005 R 001606 QTO      0000 R 002444 QTOT      0002 R 001074 RDTWRT
0002 R 001114 RHOFN      0002 R 001067 RHOO      0002 R 001073 RLIMIT      0000 R 002461 RRR      0002 R 001100 RTEND
0006 R 000000 SFN      0004 003473 SS      0000 R 002471 STFN      0000 R 002366 STFNX      0000 R 002475 STMP
0000 R 002477 STORG      0006 R 000017 STR      0000 R 002473 STTB      0000 R 002441 TABS      0003 R 000002 TATM
0006 R 000014 TBVASS      0002 R 000132 TEMP      0000 R 002452 TFNX1      0000 R 002453 TFNX2      0000 R 002426 TFN1
0000 R 002427 TFN2      0011 R 000000 THCFN      0002 001077 TI      0007 000146 TIFLD      0002 001076 TINTL
0005 000000 TM      0007 000002 TMEFLD      0002 R 000276 TMP      0006 R 000013 TNXL      0002 R 001075 TREF
0004 000000 TRMTX      0000 R 002466 TT      0000 R 002457 TTT      0002 R 000050 TW      0000 R 002440 TWRT
0005 001607 TX      0002 R 001064 TO      0003 R 000001 VELS      0000 R 002435 W      0007 000312 WIFLD
0010 R 000000 WRAT      0000 R 002460 WWW      0000 R 002447 WWX      0002 R 001066 WO      0002 R 000536 XIFN
0002 000005 XIMP      0002 000000 XITB      0002 R 001106 XL      0000 R 002476 XLTW      0002 001054 XRE
0000 R 001370 XTI4E      0002 001057 XX10      0002 001060 XX11      0002 001061 XX12      0004 003446 XX2
0002 001055 XX3      0002 001056 XX4      0014 R 000000 YINT      0000 R 002434 Z      0000 R 002436 ZCK
0000 R 002433 ZE      0002 R 000550 ZETA      0000 R 002437 ZT      0004 003453 ZZ2

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00101 1* SUBROUTINE CNTLM(Y,DY,DX,X,NTRY,IFVD)
00101 2* C
00101 3* C THIS SUBROUTINE :
00101 4* C 1- CONTROLS MAIN INTEGRATION
00101 5* C 2- CONTROLS OUTPUT OF MAIN INTEGRATION
00101 6* C 3- PROVIDES SYSTEM PERFORMANCE CHARACTERISTICS
00101 7* C IS PART OF RKS AND DERIVM SUBROUTINES
00101 8* C
00103 9* COMMON XITB(5),XIMP(5),FP(10),FW(10),FT(10),TW(10,5),TEMP(10,10),
00103 10* 1 TMP(10,5),GRFN(10,10),GRMP(10),XIFN(10),ZETA(10),AUX2(10),
00103 11* 2 CONFN(10,10),CONMP(10),COB(10),DOB(10)
00104 12* COMMON NX,MZ,NRTB,NRMP,NCTL,IFLOW,LT,LTB,LMP,LTBMZ,LTB2MZ,LL1,
00104 13* 1 LL2,LL3,LL4,LL5,LL6,LL7,NRMP1,NRTB1,NRMP2,NEQUS,LC,MSTOTR,
00104 14* 2 NCTM,NM1,MM1,LIMWRT,LIM,MOD,NCCZ,NCONV
00105 15* COMMON FRE,FR,FNU,FMACH,FPR,FRL,FRM,FFO,FOF,FZ,FRAD,DZ,DXITB,DXIMP
00105 16* 1 ,XRE,XX3,XX4,XX10,XX11,XX12,PHIM,PHIF,TO,PO,W0,RHOO,CP0,PI
00105 17* 2 ,DELTA,RLIMIT,ROTWRT,TREF,TINTL,TI,RTEND,FLUX,QREF,FENTH,
00105 18* 3 FREJ,HFN,XL,COH,DXI,FRD,FXOH,FXHW,RHOFN

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00106 19* COMMON /SRTCNV/ ELEV,VELS,TATM
00107 20* COMMON /QRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZ2(5),
00107 21* 1 MCVRD,LCT,LT,T,DZMFN,
00107 22* 2 DXX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DXX21,SS(5,5,5)
00110 23* COMMON /QIN/TM(100), QSFN(100,2), QIFN(100,2), QSTB(100,2),
00110 24* 1 QITB(100,2), NTM, NSRD, QTO, TX
00111 25* COMMON /GEOM/ SFN(10),NTBS,TNXL,TBMASS,PLMASS,FLMASS,STR
00112 26* COMMON /FLDINL/ NFLDTA,LFLD,TMEFLD(100),TIFLD(100),WIFLD(100),
00112 27* 1 FLDINT
00113 28* COMMON /DVCWFL/ WRAT,FLUXI
00113 29* C
00114 30* DIMENSION Y(200),DY(200),DY1(200),AUX1(10),AUX3(10),AUX4(10),
00114 31* 1 AUX5(10),DTDZ1(10),DTDZ2(10),FPRME(500),XTIME(500)
00115 32* DIMENSION DSTFN(10),STFNX(10),DSTTB(10),DSTMP(10)
00115 33* C
00116 34* QSOLR = EXTSFN*QTO
00117 35* QIRED = EXTIFN*QTO
00117 36* C
00120 37* NCTM = NCTM+1
00120 38* C
00121 39* TFN1 = TEMP(1,1)*T0
00122 40* TFN2 = TEMP(MZ,1)*T0
00123 41* ASTR1 = STR*SQRT(THCFN(TFN1)*CPFN(TFN1))
00124 42* ASTR2 = STR*SQRT(THCFN(TFN2)*CPFN(TFN2))
00125 43* IF (NCTM .GT. 500) GO TO 7
00125 44* C
00127 45* IF (X .LT. XTIME(NCTM-1)) GO TO 2
00131 46* FPRME(NCTM) = ASTR1*DY(1)+ASTR2*DY(MZ)
00132 47* XTIME(NCTM) = X
00133 48* GO TO 3
00134 49* 2 FPRME(NCTM) = FPRME(NCTM-1)
00135 50* XTIME(NCTM) = XTIME(NCTM-1)
00136 51* FPRME(NCTM-1) = ASTR1*DY(1)+ASTR2*DY(MZ)
00137 52* XTIME(NCTM-1) = X
00137 53* C
00140 54* 3 IF (NCTM .GT. 1) GO TO 5
00142 55* FPRME(NCTM) = ASTR1*DY(1)+ASTR2*DY(MZ)
00143 56* XTIME(NCTM) = X
00143 57* C
00144 58* 7 IF (NCTM .GT. 1) GO TO 5
00146 59* DO 4 I = 1,NEQUS
00151 60* 4 DY1(I) = DY(I)
00153 61* GO TO 500
00154 62* 5 IF(IFVD.EQ.1) GO TO 10
00156 63* IF(NTRY.EQ.3) GO TO 30
00160 64* GO TO (50,100)*MSTOTR
00160 65* C
00161 66* 10 IF(MSTOTR.EQ.1) GO TO 20
00163 67* ZE = X-RTEND
00164 68* IF(ABS(ZE)/RDTWRT.LE.0.0005) GO TO 65
00166 69* IF(ZE.GT.0.0) GO TO 105
00170 70* 15 IFVD = 1
00171 71* DX = RDTWRT
00172 72* IF(LIM.GT.LIMWRT) RETURN
00174 73* GO TO 500
00175 74* 20 Z = 0.0
00176 75* DO-25 I = 1,NEQUS

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00201 76* IF(ABS(DY(I)).LT.1.0E-30) GO TO 25
00203 77* W = ABS(DY1(I)/DY(I))
00204 78* IF(W.LE.0.0) GO TO 25
00206 79* ZCK = ALOG(W)
00207 80* IF(ABS(ZCK).LT.1.0E-06) GO TO 25
00211 81* Z = AMAX1(Z,ABS(DY(I)/ZCK)*DX)
00212 82* 25 DY1(I) = DY(I)
00214 83* IF(Z.LE.RLIMIT) GO TO 65
00216 84* IF(LIM.LE.LIMWRT) GO TO 15
00220 85* IFVD = 0
00221 86* RETURN
00222 87* 30 NTRY = 1
00223 88* IF(MOD.EQ.1) GO TO 10
00225 89* GO TO 500
00225 90* C
00225 91* C
00226 92* 50 Z = 0.0
00227 93* DO 55 I = 1,NEQUS
00232 94* IF(ABS(DY(I)).LT.1.0E-30) GO TO 55
00234 95* W = ABS(DY1(I)/DY(I))
00235 96* IF(W.LE.0.0) GO TO 55
00237 97* ZCK = ALOG(W)
00240 98* IF(ABS(ZCK).LT.1.0E-06) GO TO 55
00242 99* Z = AMAX1(Z,ABS(DY(I)/ZCK)*DX)
00243 100* 55 DY1(I) = DY(I)
00245 101* IF(Z.LE.RLIMIT) GO TO 65
00247 102* GO TO 70
00247 103* C
00250 104* 65 NTRY = 2
00251 105* GO TO 505
00251 106* C
00252 107* 70 IF(LIM.GT.LIMWRT) RETURN
00254 108* ZT = X-TWRT
00255 109* IF(ABS(ZT)/RDTWRT.LE.0.0005) GO TO 500
00257 110* IF(ZT.LT.0.0) RETURN
00261 111* NTRY = 3
00262 112* IF(DX.GE.RDTWRT) MOD = 1
00264 113* LX = DX+TWRT-X
00265 114* RETURN
00265 115* C
00266 116* 100 ZE = X-RTEND
00267 117* IF(ABS(ZE)/RDTWRT.LE.0.0005) GO TO 65
00271 118* IF(ZE.LT.0.0) GO TO 70
00273 119* 105 NTRY = 3
00274 120* LX = DX+RTEND-X
00275 121* RETURN
00275 122* C
00275 123* C
00276 124* 500 TWRT = X+RDTWRT
00277 125* 505 TABS = X*TREF
00300 126* IF(NCONV.GT.0) GO TO 506
00302 127* WRITE(6,510) TABS,X,NCTM
00307 128* GO TO 507
00310 129* 506 WRITE(6,509) TABS,X,NCTM,ELEV,VELS,TATM
00310 130* C
00310 131* C
00320 132* 507 DO 508 J=1,MZ

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00323 133*      FT(J) = Y(LT+J)
00324 134*      TW(J,1) = Y(LT8+J)
00325 135*      TMP(J,NRMP) = Y(LL5+J)
00326 136*      TEMP(J,1) = Y(LL4+J)
00327 137*      DO 508 I=2,NX
00332 138*          K = MZ*(I-2)+J
00333 139*      508 TEMP(J,I) = Y(K)
00336 140*      IF (LFLD.EQ. 2) FT(1) = FLDINT
00336 141*      C
00336 142*      C
00340 143*      509 FORMAT(1H1,5X,20HRELAPSED TIME IS ,F12.4,6H HR , ,
00340 144*          1 6X,20HRELATIVE TIME IS ,F12.4,
00340 145*          2 6X,17,14H INTEGR. STEPS,/,
00340 146*          3 6X,20HALTITUDE IS ,F10.2,8H FT , ,
00340 147*          4 6X,20HVELOCITY IS ,F10.2,10H FT/SEC , ,
00340 148*          5 6X,22HATM. TEMPERATURE IS ,F8.2,2H R,/)
00341 149*      510 FORMAT(1H1,5X,25HRELAPSED TIME IS ,F12.4,11H HR , ,
00341 150*          1 6X,25HRELATIVE TIME IS ,F12.4,
00341 151*          2 18X,12H**IN ORBIT**,20X,17,14H INTEGR. STEPS,/)
00341 152*      C
00341 153*      C LISTING OF FIN TEMPERATURE DISTRIBUTION
00341 154*      C
00342 155*      DO 512 J=1,MZ
00345 156*      DO 511 I=1,NX
00350 157*      AUX4(I) = CONFN(J,I)
00351 158*      511 AUX1(I) = GRFN(J,I)
00353 159*      AUX5(J) = DEFINT(AUX4,DXI,NX)+CONVP(J)*COH
00354 160*      512 AUX3(J) = DEFINT(AUX1,DXI,NX)+GRWP(J)*COH
00356 161*      GTOT = DEFINT(AUX3,DZ,MZ)*GREF*XL
00357 162*      GAERO = DEFINT(AUX5,DZ,MZ)*GREF*XL
00357 163*      C
00357 164*      C CONDUCTION FROM MANIFOLD
00357 165*      C
00360 166*      IF (NCTM.GT. 500) GO TO 514
00362 167*      IF (NCTM.GT. 3) GO TO 513
00364 168*      CONDMF = 0.66667*FPRME(1)*SQRT(XTIME(NCTM))
00365 169*      GO TO 600
00366 170*      513 WWW = X/3.0
00367 171*      FPRM2 = YINT(XTIME,FPRME,NCTM,3,WWW)
00370 172*      WWW = 2.0*WWW
00371 173*      FPRM1 = YINT(XTIME,FPRME,NCTM,3,WWW)
00372 174*      CONDMF = SQRT(XTIME(NCTM))/105.0*(68.0*FPRME(NCTM)+90.0*FPRM1+
00372 175*      1 36.0*FPRM2+16.0*FPRME(1))
00372 176*      C
00373 177*      514 DO 580 I=1,NX
00376 178*      TFNX1 = TEMP(1,I)*T0
00377 179*      DTDZ1(I) = -THCFN(TFNX1)*SFN(I)*(-3.0*TEMP(1,I)+4.0*TEMP(2,I)
00377 180*      1 -TEMP(3,I))
00400 181*      TFNX2 = TEMP(MZ,I)*T0
00401 182*      580 DTDZ2(I) = THCFN(TFNX2)*SFN(I)*(3.0*TEMP(MZ,I)-4.0*TEMP(MZ-1,I)+
00401 183*      1 TEMP(MZ-2,I))
00403 184*      COND1 = DEFINT(DTDZ1,DXI,NX)
00404 185*      COND2 = DEFINT(DTDZ2,DXI,NX)
00405 186*      CONMF1 = (COND1+COND2)*NTBS*T0*HFN/(DZ*XL)
00406 187*      IF (CONDMF.LE. CONMF1.OR. NCTM.GT. 500) CONDMF = CONMF1
00406 188*      C
00406 189*      C FLUID ENTHALPY REJECTION

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00406 190* C
00410 191* 600 TTT = FT(1)*T0
00411 192* WWW = FW(1)*W0
00412 193* RRR = RH00/FW(1)
00413 194* ENTHI = FLUX*(HFL(RRR,TTT)+WWW**2/1556.36)*WRAT
00414 195* ITT = FT(MZ)*T0
00415 196* WWW = FW(MZ)*W0
00416 197* RRR = RH00/FW(MZ)
00417 198* ENTHE = FLUX*(HFL(RRR,TTT)+WWW**2/1556.36)*WRAT
00420 199* DENTH = ENTHI-ENTHE
00420 200* C
00420 201* C ENERGY STORAGE RATE
00420 202* C
00420 203* C FIN
00420 204* C
00421 205* DO 700 I=2,NX
00424 206* IM2 = I-2
00425 207* DO 700 J=1,MZ
00430 208* K = IM2*MZ+J
00431 209* TT = T0*TEMP(J,I)
00432 210* CPFN1 = CPFN(TT)
00433 211* DSTFN(J) = CPFN1*SFN(I)*DY(K)
00434 212* 700 STFNX(I) = DEFINT(DSTFN,DZ,MZ)
00437 213* DO 705 J=1,MZ
00442 214* K = LTB+J
00443 215* TT = T0*TEMP(J,1)
00444 216* CPFN1 = CPFN(TT)
00445 217* DSTFN(J) = CPFN1*DY(K)
00446 218* 705 STFNX(1) = DEFINT(DSTFN,DZ,MZ)*SFN(1)
00450 219* FNM = 2.0*RHO*FN*TNXL*HFN
00451 220* STFN = DEFINT(STFNX,DXI,NX)*FNM
00451 221* C
00451 222* C TUBE
00451 223* C
00452 224* DO 710 J=1,MZ
00455 225* K = LTB+J
00456 226* TT = T0*TEMP(J,1)
00457 227* CPTB1 = CPTB(TT)
00460 228* 710 DSTTB(J) = CPTB1*DY(K)
00462 229* STTB = DEFINT(DSTTB,DZ,MZ)*TBMAS
00462 230* C
00462 231* C PROTECTION LAYER
00462 232* C
00463 233* DO 715 J=1,MZ
00466 234* K = LTB+J
00467 235* TT = T0*TEMP(J,1)
00470 236* CPMP1 = CPMP(TT)
00471 237* 715 DSTMP(J) = CPMP1*DY(K)
00473 238* STMP = DEFINT(DSTMP,DZ,MZ)*PLMASS
00473 239* C
00474 240* XLTW = XL/(T0*W0*3600.0)
00475 241* STORG = (STFN+STTB+STMP)/XLTW
00476 242* STORG = DENTH+CONDMF*QAERO-QTOT
00476 243* C
00477 244* WRITE(6,515) T0,QSOLR,QREF,QIRED,QTOT,QAERO,CONDMF,STORG,
00477 245* 1(XIFN(I),I=1,NX)
00515 246* 515 FORMAT(1H0,15X,55HFIN TEMPERATURE DISTRIBUTION AND RADIANT HEAT RE

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00515 247* 1JECTION,/,16X,55H*****
00515 248* 2*****//,
00515 249* 3 2X,33HREFERENCE TEMPERATURE, T00 =,F10.3,13H R //,
00515 250* 4 2X,56HREF. RADIANT HEAT FLUX
00515 251* 5 8X,33HINCIDENT SOLAR FLUX, QSOLR. =,E10.4,15H BTU/(HR SQ FT),
00515 252* 6 //,
00515 253* 7 2X,33H PER UNIT AXIAL LENGTH, QREF =,E10.4,13H BTU/(HR FT),
00515 254* 8 8X,33HINCIDENT INFRARED FLUX, QIRED =,E10.4,15H BTU/(HR SQ FT),
00515 255* 9 //,
00515 256* 1 2X,33HTOT. RADIANT REJECTION QTOT =,E10.4,13H BTU/HR ,
00515 257* 2 8X,33HAERODYN. HEATING POWER, QCONV =,E10.4,7H BTU/HR,/,
00515 258* 3 2X,33HCOND. FROM MANIFOLDS, CONDMF =,E10.4,13H BTU/HR ,
00515 259* 4 8X,33HENERGY STORAGE RATE, STORG =,E10.4,7H BTU/HR,/,
00515 260* 5/,6H AXIAL,4X,8HRELATIVE,14X,30HRELATIVE TEMPERATURE OF FIN, T,/,6
00515 261* 6H DIST.,4X,9HRAD. HEAT,/,10X,9HREJECTION,15X,33HDISTANCE NORMAL TO
00515 262* 7 FLOW DIRECTION,/,3H 2,9X,1H0,31X,1HX,/,20X,10F9.5,)
00516 263* WRITE(6,516)
00520 264* 516 FORMAT(1H,120H-----)
00520 265* 1-----
00520 266* 2---)
00521 267* DO 525 J=1,MZ
00524 268* 525 WRITE(6,530) ZETA(J),AUX3(J),(TEMP(J,I),I=1,NX)
00535 269* 530 FORMAT(1H,F5.3,F11.4,2X,10F9.4)
00536 270* WRITE(6,555)
00536 271* C
00540 272* WRITE(6,535) T0,P0,W0,FENTH,ENTHI,ENTHE,DENTH
00551 273* 535 FORMAT(1H0,15X,70HFLUID PROPERTIES AND SURFACE TEMPERATURE OF METE
00551 274* 10ROID PROTECTION LAYER,/,16X,70H*****
00551 275* 2*****//,
00551 276* 3 10X,33HREFERENCE TEMPERATURE, T00 =,F15.3,12H R //,
00551 277* 4 10X,33HREFERENCE PRESSURE, P00 =,F15.3,12H LRF/SQ.FT //,
00551 278* 5 10X,33HREFERENCE VELOCITY, V00 =,F15.3,12H FT/SEC //,
00551 279* 6 6X,36HCOOLANT POWER, INLET AT T=T0 HO =,E11.6,7H BTU/HR,
00551 280* 7 6X,36H INLET CURRENTLY HI =,E11.6,7H BTU/HR,/,
00551 281* 8 6X,36H EXIT CURRENTLY EI =,E11.6,7H BTU/HR,
00551 282* 9 6X,36H TOT. REJECTION DH =,E11.6,7H BTU/HR,/,
00551 283* 9 58X,8HPROTECT.,10X,18HENTHALPY REJECTION,/,36X,5HFLUID,7X,4HWALL,
00551 284* 17X,5HLAYER,7X,13HPER UNIT TUBE,5X,8HFRACTION,/,6H AXIAL,5X,8HPRESS
00551 285* 2URE,4X,8HVELOCITY,9X,23HT E M P E R A T U R E S,12X,6HLENGTH,11X,2
00551 286* 3HOF,/,6H DIST.,8X,1HP,11X,1HW,11X,2HTF,9X,3HTWI,9X,3HTMP,9X,
00551 287* 411HBTU/(HR FT),7X,5HTOTAL,/)
00551 288* C
00551 289* C POWER FLUX, FLUID FLOW
00551 290* C
00552 291* LSKIP = 1
00553 292* CALL FINT(AUX2,0.0,DZ,MZ,AUX1)
00554 293* DEDT = AUX1(1)-AUX1(MZ)
00555 294* IF(ABS(DEDT).LT.1.0E-08) LSKIP = 2
00557 295* DO 545 J=1,MZ
00562 296* DEDOT = 0.0
00563 297* GO TO (543,544), LSKIP
00564 298* 543 DEDOT = (AUX1(1)-AUX1(J))/DEDT
00565 299* 544 DIFE = AUX2(J)*FREJ
00566 300* 545 WRITE(6,550) ZETA(J),FP(J),FW(J),FT(J),TW(J,1),TMP(J,NRMP),DIFE,
00566 301* 1 DEDOT
00601 302* 550 FORMAT(1H,F5.3,F12.5,F11.4,3F12.4,F17.6,F14.5)
00602 303* WRITE(6,555)

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00604 304* 555 FORMAT(1H ,7H (EXIT))
00604 305* C
00605 306* LIM = LIM+1
00606 307* IF (MSTOTR .EQ. 1 .AND. NTRY .EQ. 2) WRITE (6,560)
00611 308* 560 FORMAT (1H1,25H STEADY STATE IS REACHED)
00612 309* RETURN
00613 310* END
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END OF COMPILATION: NO DIAGNOSTICS.

BHGDG:P \*\*\*\*\* CNTLN \*\*\*\*\*

\*\*\*\*\* CNTLN \*\*\*\*\*

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QFOR S ME\*NASA5.CNTLN ME\*NASA5.CNTLN  
FOR S9A-07/13/72-20:54:26 (0.)

SUBROUTINE CNTLN ENTRY POINT 000075

STORAGE USED: CODE(1) 000111; DATA(0) 000023; BLANK COMMON(2) 001115

COMMON BLOCKS:

0003 DRVLCM 000024  
0004 DRVLCN 000001

EXTERNAL REFERENCES (BLOCK, NAME)

0005 NWDJ\$  
0006 NIO1\$  
0007 NIO2\$  
0010 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000001	IF	0001	000025	1206	0002	000562	AUX2	0004	R	000000	B2	0002	000752	C08	
0002	001107	COH	0002	000574	CONFN	0002	000740	CONMP	0002		001070	CP0	0002	001072	DELTA	
0002	000764	DOE	0002	001110	DXI	0002	001053	DXIMP	0002		001052	DXITB	0002	001051	DZ	
0002	001103	FENTH	0002	001045	FF0	0002	001101	FLUX	0002		001041	FMACH	0002	001040	FNU	
0002	001046	F0F	0002	R	000012	FP	0002	001042	FPR	0002		001037	FR	0002	001050	FRAD
0002	001111	FRD	0002	001036	FRE	0002	001104	FREJ	0002		001043	FRL	0002	001044	FRM	
0002	R	000036	FT	0002	R	000024	F#	0002	001113	FXHW	0002	001112	FXOH	0002	001047	FZ
0002	001105	HFN	0000	I	000000	I	0002	I	001003	IFLOW	0000	000007	INJPS	0002	001024	LC
0002	001032	LIV	0002	001031	LIMWRT	0002	001011	LL1	0002		001012	LL2	0002	001013	LL3	
0002	001014	LL4	0002	001015	LL5	0002	001016	LL6	0002		001017	LL7	0002	001006	LMP	
0002	001004	LT	0002	001005	LTB	0002	001007	LTMZ	0002		001010	LTMZM	0002	001030	MM1	
0002	001033	MOE	0002	001025	MSTOTR	0002	I	000777	MZ	0002		001034	NCCZ	0002	001035	NCONV
0002	I	001002	NCTL	0002	001026	NCTM	0002	001023	NEQUS	0002		001027	NM1	0002	001001	NRMP
0002	001020	NRMP1	0002	001022	NRMP2	0002	001000	NRTB	0002		001021	NRTB1	0002	000776	NX	
0002	001063	PHIF	0002	001062	PHIM	0002	001071	PI	0002		001065	P0	0002	001102	QREF	
0002	000360	QRFN	0002	000524	QRMP	0002	001074	ROTWRT	0002		001114	RHOFN	0002	001067	RH00	
0002	001073	RLIMIT	0002	001100	RTEND	0003	R	000000	STRGE1	0003	R	000012	STRGE2	0002	000132	TEMP
0002	001077	TI	0002	001076	TINTL	0002	000276	TMP	0002		001075	TREF	0002	R	000050	TW
0002	001064	T0	0002	001066	W0	0002	000536	XIFN	0002		000005	XIMP	0002	000000	XITB	
0002	001106	XL	0002	001054	XRE	0002	001057	XX10	0002		001060	XX11	0002	001061	XX12	
0002	001055	XX3	0002	001056	XX4	0002	000550	ZETA								

00101 1\* SUBROUTINE CNTLN (Y,DY,DX,X,NTRY,IFVD)  
00101 2\* C  
00101 3\* C THIS SUBROUTINE CONTROLS :  
00101 4\* C 1- INTEGRATION OF COOLANT FLUID PRESSURE AND VELOCITY FIELDS  
00101 5\* C 2- INTEGRATION FOR FLUID INITIAL CONDITIONS  
00101 6\* C IS PART OF RKSF AND DERIVL SUBROUTINES  
00101 7\* C

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00103      8*      COMMON XITB(5),XIMP(5),FP(10),FW(10),FT(10),TW(10,5),TEMP(10,10),
00103      9*      1      TMP(10,5),GRFN(10,10),GRMP(10),XIFN(10),ZETA(10),AUX2(10),
00103     10*      2      CONFN(10,10),CONMP(10),COB(10),DOB(10)
00104     11*      COMMON NX,MZ,NRTB,NRMP,NCTL,IFLOW,LT,LTB,LMP,LTBMZ,LTBMZ,LL1,
00104     12*      1      LL2,LL3,LL4,LL5,LL6,LL7,NRMP1,NRTB1,NRMP2,NEQUS,LC,MSTOTR,
00104     13*      2      NCTM,NM1,M41,LIMWRT,LIM,MOD,NCCZ,NCONV
00105     14*      COMMON FRE,FR,FNU,FMACH,FPR,FRL,FRM,FFO,FOF,FZ,FRAD,DZ,DXITB,DXIMP
00105     15*      1      ,XRE,XX3,XX4,XX10,XX11,XX12,PHIM,PHIF,T0,P0,W0,RH00,CPO,PI
00105     16*      2      ,DELTA,RLIMIT,ROTWRT,TREF,TINTL,TI,RTEND,FLUX,QREF,FENTH,
00105     17*      3      FREJ,HFN,XL,COM,DXI,FRD,FXOH,FXHW,RHOFN
00106     18*      COMMON /DRVLCM/ STRGE1(10),STRGE2(10)
00107     19*      COMMON /DRVLCN/ B2
00110     20*      DIMENSION Y(3),DY(3)
00110     21*      C
00111     22*      STRGE1(NCTL) = B2
00112     23*      STRGE2(NCTL) = DY(3)
00113     24*      IF(IFLOW.EQ.1) WRITE (6,1) NCTL,X,(Y(I),I=1,3),TW(NCTL,1)
00125     25*      IF(NCTL.EQ.MZ) NTRY=2
00127     26*      FP(NCTL) = Y(1)
00130     27*      FW(NCTL) = Y(2)
00131     28*      IF (IFLOW .EQ. 1) FT(NCTL) = Y(3)
00133     29*      NCTL = NCTL+1
00134     30*      RETURN
00134     31*      C
00135     32*      1 FORMAT (15X,I5,F10.3,4F20.6)
00136     33*      END

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END OF COMPILATION: NO DIAGNOSTICS.

QHD6,P \*\*\*\*\* CONVEC \*\*\*\*\*



\*\*\*\*\* CONVEC \*\*\*\*\*

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QFOR S ME\*NASAS.CONVEC ME\*NASAS.CONVEC  
FOR S9A-07/13/72-20:54:33 (0.)

SUBROUTINE CONVEC ENTRY POINT 000543

STORAGE USED: CODE(1) 000561; DATA(0) 000144; BLANK COMMON(2) 001115

COMMON BLDCK5:

0003 VELALT 001132  
0004 CNV 000010  
0005 SRTCNV 000003  
0006 GRD 003721  
0007 SSF 000005

EXTERNAL REFERENCES (BLOCK, NAME)

0010 ALTVEL  
0011 ATMOS  
0012 ENTAIR  
0013 TNH  
0014 CPAIR  
0015 REFP  
0016 NUS  
0017 NEXP65  
0020 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000031 122G	0001 000033 125G	0001 000425 150L	0001 000401 211G	0001 000416 220G
0001 000417 223G	0001 000431 231G	0001 000507 243G	0001 000510 246G	0001 000122 31L
0001 000236 35L	0001 000253 37L	0001 000267 47L	0001 000045 5L	0001 000277 51L
0002 001105 AL	0007 000002 ALPFN	0003 R 000310 ALTA	0003 R 000454 ALTR	0010 R 000000 ALTVEL
0002 000562 AUX2	0000 R 000020 CATM	0002 000752 COB	0000 R 000043 COEF	0004 000007 COEFF
0002 001107 COH	0002 R 000574 CONFN	0002 R 000740 CONMP	0014 R 000000 CPAIR	0000 R 000054 CPFIN
0000 R 000045 CPMP	0002 001070 CPO	0006 003465 CSF	0002 001072 DELTA	0002 000764 DOB
0004 000003 DX	0002 001110 DXI	0002 001053 DXIMP	0002 001052 DXITB	0006 003464 DXX2
0006 003472 DXX21	0002 001051 DZ	0006 003463 DZMFN	0006 003410 EBFN	0006 003441 EBMF
0005 R 000000 ELEV	0012 R 000000 ENTAIR	0000 R 000021 ENTATM	0000 R 000042 ENTAW	0000 R 000030 ENTFN
0006 003467 EXTIFN	0006 003471 EXTIMP	0006 003466 EXTSMF	0006 003470 EXTSMF	0002 001103 FENTH
0002 001045 FFO	0002 001101 FLUX	0002 001041 FMACH	0002 001040 FNU	0002 001046 FOF
0002 000012 FP	0002 001042 FPR	0002 001037 FR	0002 001050 FRAD	0002 001111 FRD
0002 001036 FRE	0002 001104 FREJ	0002 001043 FRL	0002 001044 FRM	0002 000036 FT
0002 000024 FW	0002 001113 FXHW	0002 001112 FXOH	0002 001047 FZ	0002 001105 HFN
0000 I 000051 I	0002 001003 IFLOW	0000 I 000024 II	0000 000116 INJPS	0004 I 000002 IOPTN
0000 I 000050 J	0000 I 000025 JJ	0000 I 000053 L	0002 001024 LC	0006 003461 LCT
0002 001032 LIM	0002 001031 LIMWRT	0002 001011 LL1	0002 001012 LL2	0002 001013 LL3
0002 001014 LL4	0002 001015 LL5	0002 001016 LL6	0002 001017 LL7	0002 001006 LMP
0002 001004 LT	0002 001005 LTB	0002 001007 LTRMZ	0002 001010 LTR2MZ	0006 003462 LTT
0000 I 000026 M	0000 R 000000 MACHNO	0006 I 003460 MCVRD	0000 I 000017 M4	0002 001030 MM1
0002 001033 MOD	0002 001025 MSTOTR	0000 R 000002 MWT	0002 I 000777 MZ	0003 I 001130 NA
0002 001034 NCCZ	0002 001035 NCONV	0002 001002 NCTL	0002 001026 NCTM	0002 001023 NEQUS
0002 001027 NM1	0000 I 000016 NN	0003 I 001131 NR	0002 I 001001 NRMP	0002 001020 NRMP1

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0002	001022	NRMP2	0002	001000	NRTB	0002	001021	NRTB1	0004	I	000004	NSRAD	0000	R	000001	NUS1
0002	I	000776	NX	0002	001063	PHIF	0002	001062	PHIM	0002	001071	PI	0002		001065	P0
0002	001102	QREF	0002	000360	QRFN	0002	000524	QRMP	0002	001074	ROTWRT	0000	R	000031	RECFAC	
0000	R	000037	REFCP	0000	R	000032	REFENT	0000	R	000023	REFGAM	0000	R	000036	REFK	
0000	R	000041	REFRHO	0000	R	000003	REFT	0000	R	000033	REFTP	0000	R	000035	REFVIS	
0002	001067	RH00	0002	001073	RLIMIT	0002	001100	RTEND	0007	000003	R1	0007		000000	R2	
0004	000005	SMPC	0006	003473	SS	0006	003670	SST	0004	R	000000	STAGX	0004	R	000006	ST4
0007	000001	T	0003	R	000000	TA	0005	R	000002	TATM	0002	R	000132	TEMP	0000	R
0004	000005	THICK	0002	001077	TI	0002	R	001064	TIN	0002	001076	TINTL	0000	R	000044	TMET
0000	R	000034	TMK	0002	R	000276	TMP	0000	R	000027	TMPE	0013	R	000000	TNH	0003
0002	001075	TREF	0006	000000	TRMTX	0002	000050	TW	0002	001064	T0	0002		000620	VELA	
0003	R	000764	VELR	0005	R	000001	VELS	0004	R	000001	VERTX	0002	001066	W0	0002	000536
0002	000005	XIMP	0002	000000	XITB	0002	001106	XL	0002	001054	XRE	0007	R	000004	XSPM	
0000	R	000046	XXX	0002	001057	XX10	0002	001060	XX11	0002	001061	XX12	0006	003446	XX2	
0002	001055	XX3	0002	001056	XX4	0000	R	000047	YYY	0000	R	000040	Z	0002	000550	ZETA
0000	R	000055	ZZZ	0006	003453	ZZZ										

```

00101      1*      SUBROUTINE CONVEC (TIME)
00101      2*      C
00101      3*      C THIS SUBROUTINE COMPUTES :
00101      4*      C CONVECTIVE HEAT FLUX ON THE RADIATOR SYSTEM OF THE SHUTTLE
00101      5*      C DURING EITHER ASCENT OR REENTRY
00101      6*      C
00103      7*      COMMON XITB(5),XIMP(5),FP(10),FW(10),FT(10),TW(10,5),TEMP(10,10),
00103      8*      1 TMP(10,5),QRFN(10,10),QRMP(10),XIFN(10),ZETA(10),AUX2(10),
00103      9*      2 CONFN(10,10),CONMP(10),COS(10),DOB(10)
00104     10*      COMMON NX,MZ,NRTB,NRMP,NCTL,IFLOW,LT,LTB,LMP,LTBMZ,LTBMZ,LL1,
00104     11*      1 LL2,LL3,LL4,LL5,LL6,LL7,NRMP1,NRTB1,NRMP2,NEQUS,LC,MSTOTR,
00104     12*      2 NCTM,NM1,MN1,LIMWRT,LIM,MOD,NCCZ,NCONV
00105     13*      COMMON FRE,FR,FNU,FMACH,FPR,FRL,FRM,FFO,FOF,FZ,FRAD,DZ,DXITB,DXIMP
00105     14*      1 ,XRE,XX3,XX4,XX10,XX11,XX12,PHIM,PHIF,T0,P0,W0,RH00,CP0,PI
00105     15*      2 ,DELTA,RLIMIT,ROTWRT,TREF,TINTL,TI,RTEND,FLUX,QREF,FENTH,
00105     16*      3 FREJ,HFN,XL,C0H,DXI,FRU,FXOH,FXHW,RHOFN
00106     17*      COMMON/VELALT/TA(100),TR(100),ALTA(100),ALTR(100),VELA(100),
00106     18*      1 VELR(100),NA,NR
00107     19*      COMMON/CNV/STAGX,VERTX,IOPTN,DX,NSRAD,SMPC,ST4,COEFF
00110     20*      COMMON /SRTCNV/ ELEV,VELS,TATM
00111     21*      COMMON /GRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZ2(5),
00111     22*      1 MCVRD,LCT,LTT,DZMFN,
00111     23*      2 DXX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DXX21,SS(5,5,5)
00111     24*      3 ,SSTT(5,5)
00112     25*      COMMON /SSF/R2, T, ALPFN,R1,XSPM
00113     26*      EQUIVALENCE (SMPC,THICK),(TIN,T0),(AL,HFN)
00114     27*      REAL MACHNO,NUS1,MWT
00115     28*      DIMENSION REFT (11)
00116     29*      ELEV = ALTVEL(TIME,IOPTN,ALTA,ALTR,TA,TR,NA,NR)
00117     30*      IF (ELEV .LT. 528000.) GO TO 5
00121     31*      DO 2 NN = 1,NX
00124     32*      DO 2 MM = 1,MZ
00127     33*      CONMP(MM) = 0.0
00130     34*      2 CONFN(MM,NN) = 0.0
00133     35*      RETURN
00134     36*      5 VELS = ALTVEL(TIME,IOPTN,VELA,VELR,TA,TR,NA,NR)
00135     37*      CALL ATMOS (ELEV,TATM,CATM)

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\*\*\*\*\* CONVEC \*\*\*\*\*

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00136 38*      ENTATM = ENTAIR(TATM)
00137 39*      20 MACHNO = VEL5/CATM
00140 40*      REFPR = 0.72
00141 41*      REFGAM = 1.3
00142 42*      II = MZ/2
00143 43*      JJ = NX/2
00144 44*      M = 1
00145 45*      TMPE = TEMP(II,JJ)*TIN
00146 46*      ENTFN = ENTAIR(TMPE)
00147 47*      31 RECFAC = REFPR*((8.0+0.528*MACHNO**2)/(22.0+MACHNO**2))
00150 48*      33 RESENT = 0.5*(ENTATM+ENTFN)+0.11*RECFAC*
00150 49*      1 (REFGAM-1.0)*MACHNO**2*ENTATM
00151 50*      REFT(M) = INH(REFENT)
00152 51*      REFTP = REFT(M)
00153 52*      TMK = REFTP/1.8
00154 53*      REFVIS = ((1.458E-06*TMK**1.5)/(TMK+110.4))*0.671969
00155 54*      REFK = ((6.325E-07*TMK**1.5)/(TMK+245.4*10.0**(-12.0/TMK)))
00155 55*      1 *0.671969
00156 56*      REFCP = CPAIR(REFTP)
00157 57*      REFPR = REFVIS*REFCP/REFK
00160 58*      IF (ELEV .GT. 301000.) GO TO 35
00162 59*      REFGAM = REFCP/(REFCP-0.0686)
00163 60*      GO TO 37
00164 61*      35 Z = ELEV/(3280.8399)
00165 62*      MWT = 28.9644-0.0309491*(Z-90.0)
00166 63*      REFGAM = REFCP/(REFCP-(1.98585/MWT))
00167 64*      37 IF (M .EQ. 1) GO TO 47
00171 65*      IF (ABS(REFT(M)-REFT(M-1)) .LE. 5.0) GO TO 51
00173 66*      47 M = M+1
00174 67*      IF (M .GT. 10) GO TO 51
00176 68*      GO TO 31
00177 69*      51 CONTINUE
00200 70*      CALL REFP (ELEV,REFTP,REFPR,REFVIS,REFRHO,REFK,REFCP,
00200 71*      1 REFGAM)
00201 72*      ENTAW = ENTATM+RECFAC*VEL5**2/50062.744
00202 73*      CALL NUS(MACHNO,TATM,CATM,TIN,REFPR,REFVIS,REFRHO,
00202 74*      1 STAGX,VERTX,NSRAD,NUS1)
00203 75*      COEF = (3600.*REFK)/(STAGX*REFCP)
00203 76*      C
00203 77*      C DETERMINATION OF CONVECTION LOSS FROM METEOROID PROTECTION NODES
00203 78*      C
00204 79*      TMET = TMP(II,NRMP)*TIN
00205 80*      CPMP = CPAIR(TMET)
00206 81*      XXX = COEF*NUS1
00207 82*      YYY = CPMP*TIN
00210 83*      DO 80 J = 1,MZ
00213 84*      80 CONMP(J) = XXX*(ENTAW-YYY*TMP(J,NRMP))/(ST4*2.)
00213 85*      C
00213 86*      C DETERMINATION OF FRACTION OF FIN NODES COVERED BY PROTECTION LAYER
00213 87*      C
00215 88*      IF (MCVRD .EQ. 0) GO TO 150
00217 89*      DO 140 I = 1,MCVRD
00222 90*      DO 140 J = 1,MZ
00225 91*      140 CONFN(J,I) = 0.0
00230 92*      150 DO 160 J = 1,MZ
00233 93*      TFIN = TEMP(J,MCVRD+1)*TIN
00234 94*      ENTFN = ENTAIR(TFIN)

```

\*\*\*\*\* CONVEC \*\*\*\*\*

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```
00235 95* 160 CONFN(J,MCVRD+1) = XSPM*COEF*NUS1*(ENTAW-ENTFN)/ST4
00237 96* L = MCVRD+2
00237 97* C
00237 98* C DETERMINATION OF CONVECTION LOSS FROM FIN NODES
00237 99* C
00240 100* CPFIN = CPAIR(TMPE)
00241 101* ZZZ = CPFIN*TIN
00242 102* DO 180 I = L,NX
00245 103* DO 180 J = 1,MZ
00250 104* 180 CONFN(J,I) = XXX*(ENTAW-ZZZ*TEMP(J,I))/ST4
00253 105* 1000 RETURN
00254 106* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* CPAIR \*\*\*\*\*

\*\*\*\*\* CPAIR \*\*\*\*\*

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QFOR,S ME\*NASA5.CPAIR,ME\*NASA5.CPAIR  
FOR S9A-07/13/72-20:54:38 (0.)

FUNCTION CPAIR ENTRY POINT 000074

STORAGE USED: CODE(1) 000110; DATA(0) 000035; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 SQRT  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000054	10L	0001	000016	5L	0000	R	000001	A	0000	R	000000	CPAIR	0000	R	000012	CPN		
0000	R	000013	CPO	0000	R	000007	FMN	0000	R	000006	FMO	0000		000027	INJP\$	0003	R	000000	POLY
0000	R	000011	WMN	0000	R	000010	WMO												

```
00101      1*      FUNCTION CPAIR(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THE SPECIFIC HEAT OF AIR /S A FUNCTION OF TEMPERATURE (R)
00101      5*      C      UNITS BTU/(LBM.R)
00101      6*      C
00103      7*      DIMENSION A(5)
00104      8*      DATA FMO,FMN,WMO,WMN / 0.234559 ,0.765441,31.9988,28.0134/
00111      9*      DATA A(1),A(2),A(3),A(4),A(5)/0.34240E+00,-0.95225E-03,
00111     10*      1 0.31862E-05,-0.45750E-08,0.23750E-11/
00117     11*      IF (T .GT. 600.) GO TO 5
00121     12*      CPAIR = POLY (S,A,T)
00122     13*      RETURN
00123     14*      5 CPN = 9.47-3470./T+1160000./T**2
00124     15*      CPO = 11.151-172./SQRT(T)+1530./T
00125     16*      IF (T .LT. 5000.) GO TO 10
00127     17*      CPO = CPO+5.0E-05*(T-4000.)
00130     18*      10 CPAIR = FMO*CPO/WMO+FMN*CPN/WMN
00131     19*      RETURN
00132     20*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QMDG,P \*\*\*\*\* CPF/CFFC43 \*\*\*\*\*

\*\*\*\*\* CPF/CFFC43 \*\*\*\*\*

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QFOR: S ME\*NASA5.CPF/CFFC43, ME\*NASA5.CPF/CFFC43  
FOR S9A-07/13/72-20:54:41 (0.)

FUNCTION CPF ENTRY POINT 000014

STORAGE USED: CODE(1) 000016; DATA(0) 000011; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 CPF 0000 000004 INJPS 0000 R 000001 X1 0000 R 000002 X2

```
00101 1* FUNCTION CPF(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C SPECIFIC HEAT AT CONSTANT PRESSURE AS A FUNCTION OF
00101 5* C DENSITY (SLUG/CU.FT) AND TEMPERATURE (R) OF FC-43
00101 6* C UNITS BTU/(SLUG.R)
00101 7* C
00103 8* DATA X1,X2 /-0.020092,5.4054E-04/
00106 9* CPF = (X1+X2*T)*32.174
00107 10* RETURN
00110 11* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* CPF/CFFC75 \*\*\*\*\*

\*\*\*\*\* CPF/CFFC75 \*\*\*\*\*

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DFOR.S ME\*NASA5.CPF/CFFC75,ME\*NASA5.CPF/CFFC75  
FOR S9A-07/13/72-20:54:43 (0.)

FUNCTION CPF ENTRY POINT 000014

STORAGE USED: CODE(1) 000016; DATA(0) 000011; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 CPF 0000 000004 INJP5 0000 R 000001 X1 0000 R 000002 X2

```
00101 1* FUNCTION CPF(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C SPECIFIC HEAT AT CONSTANT PRESSURE AS A FUNCTION OF
00101 5* C DENSITY (SLUG/CU.FT) AND TEMPERATURE (R) OF FC-75
00101 6* C UNITS BTU/(SLUG.R)
00101 7* C
00103 8* DATA X1,X2 /0.115082,2.4333E-04/
00106 9* CPF = (X1+X2*T)*32.174
00107 10* RETURN
00110 11* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG.P \*\*\*\*\* CPF/CFHE \*\*\*\*\*

\*\*\*\*\* CPF/CFHE \*\*\*\*\*

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QFOR: S ME\*NASA5.CPF/CFHE, ME\*NASA5.CPF/CFHE  
FOR S9A-07/13/72-20:54:45 (0,)

FUNCTION CPF ENTRY POINT 000057

STORAGE USED: CODE(1) 000071; DATA(0) 000031; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 BETA  
0004 CAPPA  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000005 A	0000 R 000010 ALPHA	0000 R 000002 A1	0003 R 000000 BETA	0000 R 000003 B1
0000 R 000004 C	0004 R 000000 CAPPA	0000 R 000000 CPF	0000 R 000011 CV	0000 000024 INJPS
0000 R 000001 R	0000 R 000006 RHOX	0000 R 000007 TX	0000 R 000012 X1	

00101 1\* FUNCTION CPF(RHO,T)  
00101 2\* C  
00101 3\* C  
00101 4\* C THIS FUNCTION SUBPROGRAM COMPUTES :  
00101 5\* C SPECIFIC HEAT AT CONSTANT PRESSURE AS A FUNCTION OF  
00101 6\* C DENSITY (SLUG/CU.FT) AND TEMPERATURE (R) OF HELIUM  
00101 7\* C UNITS BTU/(SLUG.R)  
00103 8\* DATA R:A1,B1,C:A /2077.02,136.9595,3.5002295E-03,10.000658,1.49610  
00103 9\* 13E-02/  
00111 10\* RHOX = RHO\*515.4275  
00112 11\* TX = T/1.8  
00113 12\* ALPHA = C\*RHOX/(TX\*\*3)  
00114 13\* CV = R\*(1.5+6.0\*ALPHA\*(1.0+RHOX\*B1\*0.5))  
00115 14\* X1 = BETA(RHO,T)\*1.8  
00116 15\* CPF = (CV+(TX\*X1\*X1\*47.872)/(RHOX\*CAPPA(RHO,T)))\*76.8624E-04  
00117 16\* RETURN  
00120 17\* END

END OF COMPILATION: NO DIAGNOSTICS.

DHDG:P \*\*\*\*\* CPF/CFNAK \*\*\*\*\*



\*\*\*\*\* CPF/CFNAK \*\*\*\*\*

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QFOR/S ME\*NASA5.CPF/CFNAK,ME\*NASA5.CPF/CFNAK  
FOR S9A-07/13/72-20:54:47 (0.)

FUNCTION CPF ENTRY POINT 000071

STORAGE USED: CODE(1) 000104; DATA(0) 000032; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 CAPP  
0004 PF  
0005 ALOG  
0006 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0003 R 000000 CAPP	0000 R 000014 CP	0000 R 000000 CPF	0000 R 000010 C1	0000 R 000011 C2
0000 R 000007 DT	0000 000023 INJPS	0004 R 000000 PF	0000 R 000012 P0	0000 R 000016 P1
0000 R 000013 THETA	0000 R 000015 TK	0000 R 000006 T0	0000 R 000001 X1	0000 R 000002 X2
0000 R 000003 X3	0000 R 000004 X4	0000 R 000005 X5		

```

00101      1*      FUNCTION CPF(RHO,T)
00101      2*      C
00101      3*      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      SPECIFIC HEAT AT CONSTANT PRESSURE AS A FUNCTION OF
00101      5*      C      DENSITY (SLUG/CU.FT) AND TEMPERATURE (R) OF NAK 78.6
00101      6*      C      UNITS BTU/(SLUG.R)
00101      7*      C
00103      8*      DATA X1,X2,X3,X4,X5 /0.2255,-0.016292,0.005396,-0.000758,0.000054/
00103      9*      1,T0,DT /659.67,300.0/,C1,C2 /58.773064,-0.008433/,P0 /2116.224/
00116     10*      THETA = (T-T0)/DT
00117     11*      CP = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00120     12*      TK = CAPP(RHO,T)
00121     13*      P1 = PF(RHO,T)-P0
00122     14*      CPF = (CP-(2.0*T*C2*C2*ALOG(1.0+TK*P1)))/(TK*778.26*(C1+C2*T)**3)
00122     15*      1)*32.174
00123     16*      RETURN
00124     17*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* CPF/CFSIL \*\*\*\*\*

\*\*\*\*\* CPF/CFSIL \*\*\*\*\*

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QFOR S ME\*NASAS.CPF/CFSIL,ME\*NASAS.CPF/CFSIL  
FOR S9A-07/13/72-20:54:50 (0.)

FUNCTION CPF ENTRY POINT 000350

STORAGE USED: CODE(1) 000372; DATA(0) 000133; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 PF  
0004 POLY  
0005 NWDUS  
0006 NIO25  
0007 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000055 1F	0000 R 000040 A	0000 R 000007 A1	0000 R 000041 A11	0000 R 000010 A2
0000 R	000042 A22	0000 R 000011 A3	0000 R 000012 A4	0000 R 000013 A5	0000 R 000043 B
0000 R	000014 B1	0000 R 000044 B11	0000 R 000015 B2	0000 R 000045 B22	0000 R 000016 B3
0000 R	000017 B4	0000 R 000020 B5	0000 R 000046 C	0000 R 000037 CP	0000 R 000000 CPF
0000 R	000021 C1	0000 R 000047 C11	0000 R 000022 C2	0000 R 000025 DT	0000 R 000033 DT2
0000 R	000050 E1	0000 R 000051 E2	0000 R 000052 E3	0000 R 000054 FP	0000 000113 INJPs
0000 R	000053 P	0003 R 000000 PF	0004 R 000000 POLY	0000 R 000034 THETA	0000 R 000035 THETA1
0000 R	000036 THETA2	0000 R 000023 T0	0000 R 000024 T01	0000 R 000032 T02	0000 R 000026 X1
0000 R	000027 X2	0000 R 000030 X3	0000 R 000031 X4	0000 R 000001 Z	

00101 1\* FUNCTION CPF(RHO,T)  
00101 2\* C  
00101 3\* C THIS FUNCTION SUBPROGRAM COMPUTES :  
00101 4\* C SPECIFIC HEAT AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND  
00101 5\* C TEMPERATURE (R) OF DOW CORNING 200 SILICON OIL (1 CS)  
00101 6\* C UNITS BTU/(SLUG.R)  
00101 7\* C TEMPERATURE .GE. 359.67 AND .LE. 859.67  
00101 8\* C  
00103 9\* DIMENSION Z(6)  
00104 10\* DATA A1,A2,A3,A4,A5 /12.35,2.9B333,1.1,-0.48333,0.1/,B1,B2,B3,B4,B  
00104 11\* 15 /-1.5,-0.01333,-1.18,0.57333,-0.1/,C1,C2 /0.7767,-0.0288/,T0,T01  
00104 12\* 2,DT /559.67,609.67,50.0/,X1,X2,X3,X4 /0.46,0.00471,0.00141,0.00004  
00104 13\* 33/,T02,DT2 /539.67,80.0/  
00132 14\* THETA = (T-T0)/DT  
00133 15\* THETA1 = (T-T01)/DT  
00134 16\* THETA2 = (T-T02)/DT2  
00135 17\* CP = ((X4\*THETA2+X3)\*THETA2+X2)\*THETA2+X1  
00136 18\* A = (((A5\*THETA+A4)\*THETA+A3)\*THETA+A2)\*THETA+A1)\*1.0E-06  
00137 19\* A11 = (((4.0\*A5\*THETA+3.0\*A4)\*THETA+2.0\*A3)\*THETA+A2)\*((1.0E-06)/DT  
00140 20\* A22 = ((12.0\*A5\*THETA+6.0\*A4)\*THETA+2.0\*A3)\*((1.0E-06)/(DT\*DT)  
00141 21\* B = (((B5\*THETA+B4)\*THETA+B3)\*THETA+B2)\*THETA+B1)\*1.0E-09  
00142 22\* B11 = (((4.0\*B5\*THETA+3.0\*B4)\*THETA+2.0\*B3)\*THETA+B2)\*((1.0E-09)/DT  
00143 23\* B22 = ((12.0\*B5\*THETA+6.0\*B4)\*THETA+2.0\*B3)\*((1.0E-09)/(DT\*DT)

\*\*\*\*\* CPF/CFSIL \*\*\*\*\*

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00144 24* C = C1+C2*THETA1
00145 25* C11 = C2/DT
00146 26* E1 = A11*A11-B22/2.0*B11*C11/C
00147 27* E2 = 2.0*A11*C11/C-A22
00150 28* E3 = A*A-B
00151 29* P = PF(RHO,T)/144.0-14.696
00152 30* Z(1) = 0.0
00153 31* Z(2) = 2.0*(C11/C)**2
00154 32* Z(3) = (E2-A*Z(2))/2.0
00155 33* Z(4) = (E1-A*E2+E3*Z(2)/2.0)/3.0
00156 34* Z(5) = (A11*B11-A*E1+E3*E2/2.0+A*B*Z(2)/2.0)/4.0
00157 35* Z(6) = (B11*B11/2.0-2.0*A*A11*B11+E1*E3+A*B*E2+B*B*Z(2)/2.0)/10.0
00160 36* FP = POLY(6,Z,P)
00161 37* CPF = (CP-T*FP/(C*337.37))*32.174
00162 38* IF (T.GE.360.67.AND.T.LE.860.67) RETURN
00164 39* WRITE (6,1) T
00167 40* 1 FORMAT (1H0,5H5SPECIFIC HEAT OF SILICON OIL, TEMP. OUT OF RANGE, T
00167 41* 1 = ,F8.3,/)
00170 42* RETURN
00171 43* END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* CPFN/FNAL \*\*\*\*\*

\*\*\*\*\* CPFN/FNAL \*\*\*\*\*

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PAGE 1

QFOR,S ME,NASA5.CPFN/FNAL,ME,NASA5.CPFN/FNAL  
FOR S9A-07/13/72-20:54:54 (0,)

FUNCTION CPFN ENTRY POINT 000056

STORAGE USED: CODE(1) 000062; DATA(0) 000040; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ\$  
0004 NIQ2\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000012 1F	0000 R 000011 CP	0000 R 000000 CPFN	0000 R 000007 DT	0000 000032 INJP\$
0000 R	000010 THETA	0000 R 000006 T0	0000 R 000001 X1	0000 R 000002 X2	0000 R 000003 X3
0000 R	000004 X4	0000 R 000005 X5			

```
00101 1*      FUNCTION CPFN(T)
00101 2*      C
00101 3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4*      C      SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF ALUMINUM 7075
00101 5*      C      UNITS BTU/(SLUG.R)
00101 6*      C      TEMP. 6E. 300.0.AND.LE. 1200.0 R
00101 7*      C
00103 8*      DATA X1,X2,X3,X4,X5 /0.182,0.03616,-0.011417,0.00233,-0.000083/,T0
00103 9*      1,DT /400.0,200.0/
00113 10*     THETA = (T-T0)/DT
00114 11*     CP = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00115 12*     CPFN = CP*32.174
00116 13*     IF (T-6E.300.0.AND.T-LE.1200.0) RETURN
00120 14*     WRITE (6,1) T
00123 15*     1 FORMAT (1H0,51H$SPECIFIC HEAT OF ALUMINUM, TEMP. OUT OF RANGE, T =
00123 16*     1,F8.3,/)
00124 17*     RETURN
00125 18*     END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHD8,P \*\*\*\*\* CPFN/FNBR \*\*\*\*\*

\*\*\*\*\* CPFN/FNBR \*\*\*\*\*

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QFOR,S ME\*NASAS.CPFN/FNBR,ME\*NASAS.CPFN/FNBR  
FOR S9A-07/13/72-20:54:57 (0.)

FUNCTION CPFN ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NI025  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011	1F	0000 R 000010 CP	0000 R 000000 CPFN	0000 R 000006 DT	0000	000031	INJPS
0000 R	000007	THETA	0000 R 000005 T0	0000 R 000001 X1	0000 R 000002 X2	0000 R	000003	X3
0000 R	000004	X4						

```
00101      1*      FUNCTION CPFN(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF BERYLIUM WITH
00101      5*      C      0.84-1.68 BE0
00101      6*      C      UNITS BTU/(SLUG.R)
00101      7*      C      TEMP. GE. 400.0 AND LE. 1700.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4 /0.536,0.05667,-0.0085,0.00083/,T0,DT /800.,200./
00112     10*      THETA = (T-T0)/DT
00113     11*      CP = ((X4*THETA+X3)*THETA+X2)*THETA+X1
00114     12*      CPFN = CP*32.174
00115     13*      IF (T.GE.400.0.AND.T.LE.1700.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,51HSPECIFIC HEAT OF BERYLIUM= TEMP. OUT OF RANGE, T =
00122     16*      1,F8.3,/)
00123     17*      RETURN
00124     18*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* CPFN/FNCU \*\*\*\*\*

\*\*\*\*\* CPFN/FNCU \*\*\*\*\*

DATE 071372

PAGE 1

QFOR: S ME\*NASA5.CPFN/FNCU,ME\*NASA5.CPFN/FNCU  
FOR S9A-07/13/72-20:54:59 (0.)

FUNCTION CPFN ENTRY POINT 000045

STORAGE USED: CODE(1) 000051; DATA(0) 000031; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NI023  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000004 1F 0000 R 000003 CP 0000 R 000000 CPFN 0000 000023 INJP5 0000 R 000001 X1  
0000 R 000002 X2

00101 1\* FUNCTION CPFN(T)  
00101 2\* C  
00101 3\* C THIS FUNCTION SUBPROGRAM COMPUTES :  
00101 4\* C SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF COPPER  
00101 5\* C UNITS BTU/(SLUG.R)  
00101 6\* C TEMP. GE. 400.0 AND LE. 1800.0 R  
00101 7\* C  
00103 8\* DATA X1,X2 /0.08375,1.375E-05/  
00106 9\* CP = X2\*T+X1  
00107 10\* CPFN = CP\*32.174  
00110 11\* IF (T.GE.400.0.AND.T.LE.1800.0) RETURN  
00112 12\* WRITE (6,1) T  
00115 13\* 1 FORMAT (1H0,49HSPECIFIC HEAT OF COPPER, TEMP. OUT OF RANGE, T = ,  
00115 14\* 1F8.3,/)   
00116 15\* RETURN  
00117 16\* END

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* CPMP/MPAL \*\*\*\*\*

\*\*\*\*\* CPMP/MPAL \*\*\*\*\*

DATE 071372

PAGE 1

2FOR S ME\*NASA5.CPMP/MPAL,ME\*NASA5.CPMP/MPAL  
FOR S9A-07/13/72-20:55:01 (0,)

FUNCTION CPMP ENTRY POINT 000056

STORAGE USED: CODE(1) 000062; DATA(0) 000040; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ\$  
0004 NIO2\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000012 1F	0000 R 000011 CP	0000 R 000000 CPMP	0000 R 000007 DT	0000	000032 INJP\$
0000 R	000010 THETA	0000 R 000006 T0	0000 R 000001 X1	0000 R 000002 X2	0000 R	000003 X3

R		R	
0000	00000 <sub>4</sub> X <sub>4</sub>	0000	00000 <sub>5</sub> X <sub>5</sub>

```

00101      1*      FUNCTION CPMP(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF ALUMINUM 7075
00101      5*      C      UNITS BTU/(SLUG.R)
00101      6*      C      TEMP. GE. 300.0.AND.LE. 1200.0 R
00101      7*      C
00103      8*      DATA X1,X2,X3,X4,X5 /0.182,0.03616,-0.011417,0.00233,-0.000083/,T0
00103      9*      1,DT /400.0,200.0/
00113     10*      THETA = (T-T0)/DT
00114     11*      CP = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00115     12*      CPMP = CP*32.174
00116     13*      IF (T.GE.300.0.AND.T.LE.1200.0) RETURN
00120     14*      WRITE (6,1) T
00123     15*      1 FORMAT (1H0,51HSPECIFIC HEAT OF ALUMINUM, TEMP. OUT OF RANGE, T =
00123     16*      1,F8.3,/)
00124     17*      RETURN
00125     18*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHOG,P \*\*\*\*\* CPMP/MPBR \*\*\*\*\*

\*\*\*\*\* CPMP/MPBR \*\*\*\*\*

DATE 071372

PAGE 1

QFOR: S ME\*NASA5.CPMP/MPBR ME\*NASA5.CPMP/MPBR  
FOR S9A-07/13/72-20:55:03 (0.)

FUNCTION CPMP ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ\$  
0004 NI02\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011 1F	0000 R 000010 CP	0000 R 000000 CPMP	0000 R 000006 DT	0000	000031 INJP\$
0000 R	000007 THETA	0000 R 000005 T0	0000 R 000001 X1	0000 R 000002 X2	0000 R	000003 X3
0000 R	000004 X4					

```

00101 1*      FUNCTION CPMP(T)
00101 2*      C
00101 3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4*      C      SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF BERYLIUM WITH
00101 5*      C      0.84-1.68 BE0
00101 6*      C      UNITS BTU/(SLUG.R)
00101 7*      C      TEMP. GE. 400.0 AND LE. 1700.0 R
00101 8*      C
00103 9*      DATA X1,X2,X3,X4 /0.536,0.05667,-0.0085,0.00083/,T0,DT /800.,200./
00112 10*     THETA = (T-T0)/DT
00113 11*     CP = ((X4*THETA+X3)*THETA+X2)*THETA+X1
00114 12*     CPMP = CP*32.174
00115 13*     IF (T.GE.400.0.AND.T.LE.1700.0) RETURN
00117 14*     WRITE (6,1) T
00122 15*     1 FORMAT (1H0,51HSPECIFIC HEAT OF BERYLIUM= TEMP. OUT OF RANGE, T =
00122 16*     1,F8.3,/)
00123 17*     RETURN
00124 18*     END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* CPMP/MPCU \*\*\*\*\*



\*\*\*\*\* CPMP/MPCU \*\*\*\*\*

DATE 071372

PAGE

1

QFOR,S ME\*NASA5.CPMP/MPCU,ME\*NASA5.CPMP/MPCU  
FOR S9A-07/13/72-20:55:06 (0,)

FUNCTION CPMP ENTRY POINT 000045

STORAGE USED: CODE(1) 000051; DATA(0) 000031; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ\$  
0004 NIO2\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000004 1F 0000 R 000003 CP 0000 R 000000 CPMP 0000 000023 INJP\$ 0000 R 000001 X1  
0000 R 000002 X2

00101 1\* FUNCTION CPMP(T)  
00101 2\* C  
00101 3\* C THIS FUNCTION SUBPROGRAM COMPUTES :  
00101 4\* C SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF COPPER  
00101 5\* C UNITS BTU/(SLUG.R)  
00101 6\* C TEMP. GE. 400.0 AND LE. 1800.0 R  
00101 7\* C  
00103 8\* DATA X1,X2 /0.08375,1.375E-05/  
00106 9\* CP = X2\*T\*X1  
00107 10\* CPMP = CP\*32.174  
00110 11\* IF (T,GE,400.0,AND,T,LE,1800.0) RETURN  
00112 12\* WRITE (6,1) T  
00115 13\* 1 FORMAT (1H0,49HSPECIFIC HEAT OF COPPER, TEMP. OUT OF RANGE, T = ,  
00115 14\* 1F8.3,/)   
00116 15\* RETURN  
00117 16\* END

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* CPTB/TBAL \*\*\*\*\*

\*\*\*\*\* CPTB/TBAL \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASAS.CPTB/TBAL,ME\*NASAS.CPTB/TBAL  
FOR S9A-07/13/72-20:55:08 (0.)

FUNCTION CPTB ENTRY POINT 000056

STORAGE USED: CODE(1) 000062; DATA(0) 000040; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ5  
0004 NI025  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000012 1F	0000 R 000011 CP	0000 R 000000 CPTB	0000 R 000007 DT	0000 000032 INJP5
0000 R	000010 THETA	0000 R 000006 T0	0000 R 000001 X1	0000 R 000002 X2	0000 R 000003 X3
0000 R	000004 X4	0000 R 000005 X5			

```
00101 1*      FUNCTION CPTB(T)
00101 2*      C
00101 3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4*      C      SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF ALUMINUM 7075
00101 5*      C                      UNITS BTU/(SLUG.R)
00101 6*      C      TEMP. GE. 300.0.AND.LE. 1200.0 R
00101 7*      C
00103 8*      DATA X1,X2,X3,X4,X5 /0.182,0.03616,-0.011417,0.00233,-0.000083/,T0
00103 9*      1,DT /400.0,200.0/
00113 10*     THETA = (T-T0)/DT
00114 11*     CP = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00115 12*     CPTB = CP*32.174
00116 13*     IF (T.GE.300.0.AND.T.LE.1200.0) RETURN
00120 14*     WRITE (6,1) T
00123 15*     1 FORMAT (1H0,51HSPECIFIC HEAT OF ALUMINUM, TEMP. OUT OF RANGE, T =
00123 16*     1,F8.3,/)
00124 17*     RETURN
00125 18*     END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* CPTB/TBBR \*\*\*\*\*

\*\*\*\*\* CPTB/TBBR \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME NASAS.CPTB/TBBR ME NASAS.CPTB/TBBR  
FOR S9A-07/13/72-20:55:10 (0.)

FUNCTION CPTB ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NI025  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011	1F	0000 R 000010 CP	0000 R 000000 CPTB	0000 R 000006 DT	0000	000031	INJP5
0000 R	000007	THETA	0000 R 000005 T0	0000 R 000001 X1	0000 R 000002 X2	0000 R	000003	X3
0000 R	000004	X4						

```
00101      1*      FUNCTION CPTB(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF BERYLIUM WITH
00101      5*      C      0.84-1.68 BEQ
00101      6*      C      UNITS BTU/(SLUG.R)
00101      7*      C      TEMP. GE. 400.0 AND LE. 1700.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4 /0.536,0.05667,-0.0085,0.00083/,T0,DT /800.,200./
00112     10*      THETA = (T-T0)/DT
00113     11*      CP = ((X4*THETA+X3)*THETA+X2)*THETA+X1
00114     12*      CPTB = CP*32.174
00115     13*      IF (T.GE.400.0.AND.T.LE.1700.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,51HSPECIFIC HEAT OF BERYLIUM= TEMP. OUT OF RANGE, T =
00122     16*      1,F8.3,/)
00123     17*      RETURN
00124     18*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG P \*\*\*\*\* CPTB/TBCU \*\*\*\*\*

\*\*\*\*\* CPTB/TBCU \*\*\*\*\*

DATE 071372

PAGE 1

QFOR: S ME\*NASA5.CPTB/TBCU, ME\*NASA5.CPTB/TBCU  
FOR S9A-07/13/72-20:55:13 (0,)

FUNCTION CPTB ENTRY POINT 000045

STORAGE USED: CODE(1) 000051; DATA(0) 000031; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NIO25  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000004 1F 0000 R 000003 CP 0000 R 000000 CPTB 0000 000023 INJPS 0000 R 000001 X1  
0000 R 000002 X2

```
00101      1*      FUNCTION CPTB(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      SPECIFIC HEAT AS A FUNCTION OF TEMPERATURE (R) OF COPPER
00101      5*      C                        UNITS BTU/(SLUG.R)
00101      6*      C      TEMP. GE. 400.0 AND LE. 1800.0 R
00101      7*      C
00103      8*      DATA X1,X2 /0.08375,1.375E-05/
00106      9*      CP = X2*T+X1
00107     10*      CPTB = CP*32.174
00110     11*      IF (T.GE.400.0.AND.T.LE.1800.0) RETURN
00112     12*      WRITE (6,1) T
00115     13*      1 FORMAT (1H0,49HSPECIFIC HEAT OF COPPER, TEMP. OUT OF RANGE, T = ,
00115     14*      1F8.3,/)
00116     15*      RETURN
00117     16*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* DDX \*\*\*\*\*

\*\*\*\*\* DDX \*\*\*\*\*

DATE 071372

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QFOR S ME\*NASA5.DDX ME\*NASA5.DDX  
FOR S9A-07/13/72-20:55:15 (0,)

SUBROUTINE DDX ENTRY POINT 000127

STORAGE USED: CODE(1) 000153; DATA(0) 000046; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ5  
0004 NI025  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000061 1126 0001 000102 2L 0000 000003 3F 0000 R 000000 DX1 0000 I 000002 I  
0000 000016 INJP5 0000 I 000001 NI

00101	1*		SUBROUTINE DDX (Y,DY,DX,N)	1
00101	2*	C		
00101	3*	C	THIS SUBROUTINE COMPUTES :	
00101	4*	C	THE FIRST DERIVATIVE OF THE EQUALLY-SPACED ARRAY Y(N)	
00101	5*	C		
00103	6*		DIMENSION Y(N),DY(N)	2
00104	7*		IF (N.LT.3) GO TO 2	
00106	8*		DX1 = 2.0*DX	5
00107	9*		DY(1) = (-3.0*Y(1)+4.0*Y(2)-Y(3))/DX1	
00110	10*		N1 = N-1	6
00111	11*		DO 1 I=2,N1	7
00114	12*	1	DY(I) = (Y(I+1)-Y(I-1))/DX1	8
00116	13*		DY(N) = (Y(N-2)-4.0*Y(N-1)+3.0*Y(N))/DX1	
00117	14*		RETURN	10
00120	15*	2	WRITE (6,3)	11
00122	16*	3	FORMAT (25HDDX NOT ENOUGH NODAL PTS.,/)	12
00123	17*		RETURN	13
00124	18*		END	14

END OF COMPILATION: NO DIAGNOSTICS.

QHDG P \*\*\*\*\* DEFINIT \*\*\*\*\*

\*\*\*\*\* DEFINIT \*\*\*\*\*

DATE 071372

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1

QFOR,S ME\*NASA5.DEFINIT,ME\*NASA5.DEFINIT  
FOR S9A-07/13/72-20:55:18 (0,)

FUNCTION DEFINIT ENTRY POINT 000203

STORAGE USED: CODE(1) 000225; DATA(0) 000046; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000166	10L	0001	000117	1316	0001	000126	1366	0001	000032	2L	0001	000054	4L
0000	R	000000	DEFINT	0000	000024	INJP5	0000	I	000011	K	0000	I	000005	M
0000	I	000010	NEVE	0000	I	000007	NODD	0000	I	000003	N1	0000	I	000004
0000	R	000002	SODD									0000	R	000001
														SEVE

```

00101      1*      FUNCTION DEFINIT(Y,DX,N)
00101      2*      C
00101      3*      C THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C THE DEFINITE INTEGRAL BY SIMPSON'S RULE OF THE EQUALLY-SPACED
00101      5*      C ARRAY Y(N)
00101      6*      C
00103      7*      DIMENSION Y(N)
00104      8*      IF(N-3)10,1,2
00107      9*      1 DEFINIT = (DX/3.)*(Y(1)+4.*Y(2)+Y(3))
00110     10*      RETURN
00111     11*      2 IF(N-4)10,3,4
00114     12*      3 DEFINIT = (3.*DX/8.)*(Y(1)+3.*(Y(2)+Y(3))+Y(4))
00115     13*      RETURN
00116     14*      4 SEVE = 0.
00117     15*      SODD = 0.
00120     16*      N1 = N/2
00121     17*      N2 = 2*N1
00122     18*      M = N
00123     19*      NC = M-N2
00124     20*      IF(NC.EQ.0) M=N-1
00126     21*      NODD = M-1
00127     22*      NEVE = M-2
00130     23*      DO 7 K=2,NODD,2
00133     24*      7 SODD = SODD+Y(K)
00135     25*      DO 8 K=3,NEVE,2
00140     26*      8 SEVE = SEVE+Y(K)
00142     27*      DEFINIT=(DX/3.)*(Y(1)+Y(M)+4.*SODD+2.*SEVE)
00143     28*      IF(NC)10,9,10
00146     29*      9 DEFINIT = DEFINIT+(DX/24.)*(9.*Y(N)+19.*Y(N-1)-5.*Y(N-2)+Y(N-3))
00147     30*      10 RETURN
00150     31*      END

```

\*\*\*\*\* DEFINT \*\*\*\*\*

DATE 071372

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END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* DEFINT \*\*\*\*\*

\*\*\*\*\* DEFNT \*\*\*\*\*

DATE 071372

PAGE 1

DEFOR S ME\*NASAS.DEFNT,ME\*NASAS.DEFNT  
FOR S9A-07/13/72-20:55:22 (0,)

FUNCTION DEFNT ENTRY POINT 000046

STORAGE USED: CODE(1) 000060; DATA(0) 000023; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000020 1076 0000 R 000000 DEFNT 0000 I 000003 I 0000 000007 INJP5 0000 I 000001 L1  
0000 R 000002 Z

```
00101 1* FUNCTION DEFNT(Y,DX,N)
00101 2* C
00101 3* C THIS SUBROUTINE COMPUTES :
00101 4* C THE DEFINITE INTEGRAL BY TRAPEZOIDAL RULE OF THE EQUALLY-SPACED
00101 5* C ARRAY Y(N)
00101 6* C
00103 7* DIMENSION Y(N)
00104 8* L1 = N-1
00105 9* Z = 0.0
00106 10* DO 5 I= 2,L1
00111 11* 5 Z = Z+Y(I)
00113 12* DEFNT = DX*(Y(1)+2.0*Z+Y(N))/2.0
00114 13* RETURN
00115 14* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* DERIVL \*\*\*\*\*



\*\*\*\*\* DERIVL \*\*\*\*\*

DATE 071372

PAGE 1

OFOR S ME\*NASA5.DERIVL ME\*NASA5.DERIVL  
FOR S9A-07/13/72-20:55:25 (0.)

SUBROUTINE DERIVL ENTRY POINT 000245

STORAGE USED: CODE(1) 000257; DATA(0) 000053; BLANK COMMON(2) 001115

COMMON BLOCKS:

0003 FLCMP 000024

EXTERNAL REFERENCES (BLOCK, NAME)

0004 RHOF  
0005 CAPPA  
0006 BETA  
0007 CPF  
0010 NERR2\$  
0011 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000177 19L	0001 000132 5L	0002 000562 AUX2	0000 R 000012 A11	0000 R 000013 A13
0000 R 000017 A21	0000 R 000020 A23	0000 R 000014 A31	0000 R 000015 A32	0000 R 000016 A33
0006 R 000000 BETA	0000 R 000021 B1	0000 R 000022 B2	0005 R 000000 CAPPA	0002 000752 C0B
0002 001107 C0H	0002 000574 CONFN	0002 000740 CONMP	0007 R 000000 CPF	0002 R 001070 CP0
0000 R 000023 C1	0000 R 000024 C2	0000 R 000025 C3	0002 001072 DELTA	0002 000764 D0B
0003 R 000000 DTHETA	0002 001110 DXI	0002 001053 DXIMP	0002 001052 DXITB	0002 001051 DZ
0002 001103 FENTH	0002 001045 FFO	0002 001101 FLUX	0002 R 001041 FMACH	0002 R 001040 FNU
0002 001046 FOF	0002 000012 FP	0002 R 001042 FPR	0002 R 001037 FR	0002 001050 FRAD
0002 001111 FRD	0002 001036 FRE	0002 001104 FREJ	0002 001043 FRL	0002 001044 FRM
0002 R 000036 FT	0002 000024 FW	0002 001113 FXHW	0002 001112 FXOH	0002 R 001047 FZ
0000 R 000010 GBETA	0000 R 000011 GCP	0000 R 000007 GKAPPA	0002 001105 HFN	0002 I 001003 IFLOW
0000 000030 INJPS	0002 001024 LC	0002 001032 LIM	0002 001031 LIMWRT	0002 001011 LL1
0002 001012 LL2	0002 001013 LL3	0002 001014 LL4	0002 001015 LL5	0002 001016 LL6
0002 001017 LL7	0002 001006 LMP	0002 001004 LT	0002 001005 LTR	0002 001007 LTRM2
0002 001010 LTB2MZ	0002 001030 MM1	0002 001033 MOD	0002 001025 MSTOTR	0002 000777 MZ
0002 001034 MCCZ	0002 001035 NCONV	0002 I 001002 NCTL	0002 001026 NCTM	0002 001023 NEQUS
0002 001027 M1	0002 001001 NRMP	0002 001020 NRMP1	0002 001022 NRMP2	0002 001000 NRTB
0002 001021 NRTB1	0002 000776 NX	0000 R 000000 P	0002 001063 PHIF	0002 001062 PHIM
0002 001071 PI	0000 R 000003 PP	0002 R 001065 PO	0002 001102 GREF	0002 000360 GRFN
0002 000524 CRMP	0002 001074 ROTWRT	0000 R 000006 RHO	0004 R 000000 RHOF	0002 001114 RHOFN
0002 R 001067 RH00	0002 001073 RLIMIT	0000 R 000005 RRHO	0002 001100 RTEND	0000 R 000002 T
0002 000132 TEMP	0002 001077 TI	0002 001076 TINTL	0002 000276 TMP	0002 001075 TREF
0000 R 000004 TT	0002 R 000050 TW	0002 R 001064 TO	0000 R 000001 W	0002 001066 W0
0002 000536 XIFN	0002 000005 XIMP	0002 000000 XITB	0002 001106 XL	0002 001054 XRE
0002 001057 XX10	0002 001060 XX11	0002 001061 XX12	0002 001055 XX3	0002 001056 XX4
0002 000550 ZETA				

00101

1\*

SUBROUTINE DERIVL (Y,DY,X)

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00101 2* C
00101 3* C THIS SUBROUTINE COMPUTES :
00101 4* C SPATIAL DERIVATIVES DY OF Y
00101 5* C
00103 6* COMMON XITB(5),XIMP(5),FP(10),FW(10),FT(10),TW(10,5),TEMP(10,10),
00103 7* 1 TMP(10,5),GRFN(10,10),GRMP(10),XIFN(10),ZETA(10),AUX2(10),
00103 8* 2 CONFN(10,10),CONMP(10),COB(10),DOB(10)
00104 9* COMMON NX,MZ,NRTB,NRMP,NCTL,IFLOW,LT,LTB,LMP,LB2MZ,LTB2MZ,LL1,
00104 10* 1 LL2,LL3,LL4,LL5,LL6,LL7,NRMP1,NRTB1,NRMP2,NEQUS,LC,MSTOTR,
00104 11* 2 NCTV,NM1,NM1,LIMWRT,LIM,MOD,NCCZ,NCONV
00105 12* COMMON FRE,FR,FNU,FMACH,FPR,FRL,FRM,FFO,FOF,FZ,FRAD,DZ,DXITB,DXIMP
00105 13* 1 ,XRE,XX3,XX4,XX10,XX11,XX12,PHIM,PHIF,T0,P0,W0,RHO0,CP0,PI
00105 14* 2 ,DELTA,RLIMIT,ROTWRT,TREF,TINTL,TI,RTEND,FLUX,GREF,FENTH,
00105 15* 3 FREJ,HFN,XL,COH,DXI,FRD,FXOH,FXHW,RHOFN
00106 16* COMMON/FLCMP/DTHETA(20)
00106 17* C
00107 18* DIMENSION Y(3),DY(3)
00107 19* C
00110 20* P = Y(1)
00111 21* W = Y(2)
00112 22* T = Y(3)
00113 23* IF (IFLOW.EQ. 2) T = FT(NCTL)
00113 24* C
00115 25* PP = P0*P
00116 26* TT = T0*T
00117 27* RRHO = RHOF(PP,TT)
00120 28* RHO = RRHO/RHO0
00121 29* GKAPPA = CAPP(ARRHO,TT)*P0
00122 30* GBETA = BETA(RRHO,TT)*T0
00123 31* GCP = CPF(RRHO,TT)/CP0
00123 32* C
00124 33* A11 = FPR
00125 34* A13 = W*RHO
00126 35* A31 = GKAPPA
00127 36* A32 = -GBETA
00130 37* A33 = 1.0/W
00131 38* A21 = FZ*(1.0+T*A32)/(RHO*GCP)
00132 39* A23 = W*FMACH/GCP
00132 40* C
00133 41* B1 = (-FR*RHO*W**2)/2.0
00134 42* B2 = FNU*(TW(NCTL,1)-T)
00135 43* GO TO (5,10), IFLOW
00135 44* C
00136 45* 5 C1 = A21*A32-A31
00137 46* C2 = A23*A32-A33
00140 47* C3 = A11*C2-A13*C1
00140 48* C
00141 49* DY(1) = (B1*C2-A13*A32*B2)/C3
00142 50* DY(2) = (A11*A32*B2-B1*C1)/C3
00143 51* DY(3) = -(A31*DY(1)+A33*DY(2))/A32
00144 52* RETURN
00144 53* C
00144 54* C
00145 55* 10 C1 = DTHETA(NCTL)*A32
00146 56* C2 = A11*A33-A13*A31
00146 57* C
00147 58* DY(1) = (A13*C1+A33*B1)/C2

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00150 59\* DY(2) = -(A11\*C1+A31\*B1)/C2  
00151 60\* RETURN  
00152 61\* END

END OF COMPILATION: • NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* DERIVM \*\*\*\*\*

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QFOR S ME\*NASAS.DERIVM,ME\*NASAS.DERIVM  
FOR S9A-07/13/72-20155:28 (0.)

SUBROUTINE DERIVM ENTRY POINT 002331

STORAGE USED: CODE(1) 002360; DATA(0) 001061; BLANK COMMON(2) 001115

COMMON BLOCKS:

0003 DVM 000002  
0004 QIN 001610  
0005 FLDINL 000457  
0006 DVCNFL 000002

EXTERNAL REFERENCES (BLOCK, NAME)

0007 CONVEC  
0010 GRAD  
0011 DDX  
0012 D2DX2  
0013 THCFN  
0014 DTHCFN  
0015 CPFN  
0016 YINT  
0017 RHOF  
0020 VISC  
0021 THCF  
0022 CPF  
0023 FLSTRT  
0024 CAPPA  
0025 BETA  
0026 THCTB  
0027 CPTB  
0030 DTHCTB  
0031 THCMP  
0032 CPMP  
0033 DTHCMP  
0034 EXP  
0035 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000041 1146	0001 000047 1216	0001 000061 1256	0001 000173 130L	0001 000075 133G
0001 000220 140L	0001 000111 1426	0001 000124 1466	0001 000141 1546	0001 000146 1606
0001 000204 1706	0001 000243 2056	0001 000263 2106	0001 000310 2176	0001 000323 2266
0001 000341 2316	0001 000366 2406	0001 000401 2476	0001 000420 2536	0001 000536 2716
0001 000575 2776	0001 000772 3366	0001 001121 3666	0001 001137 3716	0001 001154 3776
0001 001174 4056	0001 001211 4146	0001 001235 4206	0001 001272 4316	0001 001361 4436
0001 001377 4466	0001 001414 4546	0001 001434 4626	0001 001451 4716	0001 001475 4756
0001 000633 502L	0001 001534 5066	0001 000760 506L	0001 001055 510L	0001 001056 515L
0001 001731 5276	0001 001755 5376	0001 002116 5556	0001 002262 6056	0001 002112 900L
0001 002177 901L	0001 002215 902L	0001 002221 903L	0002 R 000562 AUX2	0025 R 000000 BETA
0024 R 000000 CAPPA	0000 R 000722 CCP	0002 R 000752 COB	0002 001107 COH	0000 R 000676 CON

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0000 R 000702 CONJ	0002 R 000574 CONFN	0002 R 000740 CONMP	0000 R 000701 CPARM	0022 R 000000 CPF
0015 R 000000 CPFN	0032 R 000000 CPMF	0027 R 000000 CPTB	0002 R 001070 CP0	0000 R 000677 DCON
0002 R 001072 DELTA	0000 R 000700 DIFF	0002 R 000764 D0B	0000 R 000024 DP	0000 R 000012 DTF
0014 R 000000 DTHCFN	0033 R 000000 DTHCMP	0030 R 000000 DTHCTB	0000 R 000036 DW	0002 R 001110 DXI
0002 R 001053 DXIMP	0002 R 001052 DXITB	0000 R 000050 DXTW	0002 R 001051 DZ	0000 R 000214 DZTW
0000 R 000000 D2TF	0000 R 000360 D2XTW	0000 R 000524 D2ZTW	0002 R 001103 FENTH	0002 R 001045 FFO
0005 R 000456 FLDINT	0002 R 001101 FLUX	0006 R 000001 FLUXI	0002 R 001041 FMACH	0002 R 001040 FNU
0002 R 001046 FOF	0002 R 000012 FP	0002 R 001042 FPR	0002 R 001037 FR	0002 R 001050 FRAD
0002 R 001111 FRD	0002 R 001036 FRE	0002 R 001104 FREJ	0002 R 001043 FRL	0002 R 001044 FRM
0002 R 000036 FT	0002 R 000024 FW	0002 R 001113 FXHW	0002 R 001112 FXOH	0002 R 001047 FZ
0000 R 000724 GBETA	0000 R 000725 GCP	0000 R 000733 GOTH	0000 R 000734 GFNU	0000 R 000732 GFO
0000 R 000723 GKA2PA	0002 R 001105 HFN	0000 I 000671 I	0002 R 001003 IFLOW	0000 I 000674 IM2
0000 R 001014 INJ25	0000 I 000670 J	0000 I 000672 K	0000 I 000761 KZ	0000 I 000760 K1
0002 I 001024 LC	0005 I 000001 LFLD	0002 R 001032 LIM	0002 R 001031 LIMWRT	0000 I 000726 LLT
0002 I 001011 LL1	0002 I 001012 LL2	0002 I 001013 LL3	0002 I 001014 LL4	0002 I 001015 LL5
0002 I 001016 LL6	0002 I 001017 LL7	0002 I 001006 LMP	0002 I 001004 LT	0002 I 001005 LTB
0002 I 001007 LTB2MZ	0002 I 001010 LTB2MZ	0002 I 001030 MM1	0002 R 001033 MOD	0002 R 001025 MSTOTR
0002 I 000777 MZ	0002 R 001034 NCCZ	0002 I 001035 NCONV	0002 R 001002 NCTL	0002 R 001026 NCTM
0002 R 001023 NEQJS	0005 I 000000 NFLOTA	0002 I 001027 NM1	0000 I 000703 NM2	0002 I 001001 NRMP
0002 I 001020 NR421	0002 I 001022 NRMP2	0002 I 001000 NRTB	0002 I 001021 NRTB1	0004 R 001605 NSRD
0004 R 001604 NTM	0002 I 000776 NX	0000 R 000707 PFLC	0002 R 001063 PHIF	0002 R 001062 PHIM
0002 R 001071 PI	0000 R 000717 PP	0000 R 000714 PRTR	0002 R 001065 PO	0000 R 000756 GFN
0004 R 000454 QIFN	0004 R 001274 QITB	0000 R 000757 QMP	0002 R 001102 QREF	0002 R 000360 GRFN
0002 R 000524 QRV2	0004 R 000144 QSFN	0004 R 000764 QSTB	0004 R 001606 QTO	0002 R 001074 ROTWRT
0000 R 000713 RETR	0000 R 000720 RHO	0000 R 000710 RHOC	0017 R 000000 RHOF	0002 R 001114 RHOFN
0002 R 001067 RH00	0002 R 001073 RLIMIT	0000 R 000721 RRHO	0002 R 001100 RTEND	0000 R 000673 TABS
0002 R 000132 TEM2	0000 R 000706 TFLC	0000 R 000715 TFLIN	0021 R 000000 THCF	0013 R 000000 THCFN
0031 R 000000 THCMP	0026 R 000000 THCTB	0002 R 001077 TI	0005 R 000146 TIFLD	0002 R 001076 TINTL
0004 R 000000 TM	0005 R 000002 TMEFLD	0002 R 000276 TMP	0000 R 000755 TMPGRD	0002 R 001075 TREF
0000 R 000675 TT	0000 R 000731 TTB	0000 R 000750 TTHCMP	0000 R 000736 ITM	0000 R 000747 TTMBAR
0002 R 000050 TW	0004 R 001607 TX	0002 R 001064 TO	0020 R 000000 VISC	0005 R 000312 WIFLD
0000 R 000705 WIFLDT	0006 R 000000 WRAT	0002 R 001066 WO	0002 R 000536 XIFN	0002 R 000005 XIMP
0002 R 000000 XIT3	0002 R 001106 XL	0002 R 001054 XRE	0000 R 000737 XT	0000 R 000712 XTHCF
0000 R 000741 XTHCT	0000 R 000704 XTME	0000 R 000711 XVISC	0000 R 000742 XX1	0002 R 001057 XX10
0002 R 001060 XX11	0002 R 001061 XX12	0000 R 000743 XX2	0003 R 000000 XX20	0002 R 001055 XX3
0002 R 001056 XX4	0003 R 000001 XX48	0000 R 000744 XX5	0000 R 000745 XX6	0000 R 000716 X00
0000 R 000753 X01	0000 R 000754 X03	0000 R 000727 X1	0000 R 000730 X2	0000 R 000740 X3
0000 R 000751 X4	0000 R 000752 X5	0016 R 000000 YINT	0000 R 000746 YY1	0000 R 000735 Y1
0002 R 000550 ZETA				

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00101 1* SUBROUTINE DERIVM(Y,DY,TIME)
00101 2* C
00101 3* C THIS SUBROUTINE COMPUTES :
00101 4* C ALL DERIVATIVES OF THE VARIABLE Y WITH RESPECT TO TIME
00101 5* C
00103 6* COMMON XITB(5),XIMP(5),FP(10),FW(10),FT(10),TW(10,5),TEMP(10,10),
00103 7* 1 TMP(10,5),GRFN(10,10),GRMP(10),XIFN(10),ZETA(10),AUX2(10),
00103 8* 2 CONFN(10,10),CONMP(10),COB(10),DOB(10)
00104 9* COMMON NX,MZ,NRTB,NRMP,NCTL,IFLOW,LT,LTB,LMP,LTBMZ,LTB2MZ,LL1,
00104 10* 1 LL2,LL3,LL4,LL5,LL6,LL7,NRMP1,NRTB1,NRMP2,NEQUS,LC,MSTOTR,
00104 11* 2 NCTM,NM1,MM1,LIMWRT,LIM,MOD,NCCZ,NCONV
00105 12* COMMON FRE,FR,FNU,FMACH,FPR,FRL,FRM,FFO,F0F,FZ,FRAD,DZ,DXITB,DXIMP,
00105 13* 1 XRE,XX3,XX4,XX10,XX11,XX12,PHIM,PHIF,TO,PO,W0,RH00,CP0,PI
00105 14* 2 DELTA,RLIMIT,ROTWRT,TREF,TINTL,TI,RTEND,FLUX,QREF,FENTH,

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00105 15*      3      FREJ,HFN,XL,COH,DXI,FRD,FXOH,FXHW,RHOFN
00106 16*      COMMON/DVM/XX20,XX48
00107 17*      COMMON /GIN/TM(100), QSFN(100,2), QIFN(100,2), QSTB(100,2),
00107 18*      1      QITB(100,2), NTM, NSRD, QTO, TX
00110 19*      COMMON /FLOINL/ NFLDTA,LFLD,TMEFLD(100),TIFLD(100),WIFLD(100),
00110 20*      1      FLOINT
00111 21*      COMMON /DVCML/ WRAT,FLUXI
00111 22*      C
00112 23*      DIMENSION Y(200),DY(200),D2TF(10),DTF(10),DP(10),DW(10),
00112 24*      1      DXTW(10,10),DZTW(10,10),D2XTW(10,10),D2ZTW(10,10)
00112 25*      C
00113 26*      DO 100 J=1,MZ
00116 27*      100 TEMP(J,1) = Y(LL4+J)
00120 28*      DO 105 I=2,NX
00123 29*      K = MZ*(I-2)
00124 30*      DO 105 J=1,MZ
00127 31*      105 TEMP(J,I) = Y(K+J)
00127 32*      C
00132 33*      DO 110 J=1,MZ
00135 34*      110 TMP(J,1) = Y(LL4+J)
00137 35*      IF(NRTB.EQ.0) GO TO 130
00141 36*      DO 115 I=2,NRMP
00144 37*      K = LMP+(I-2)*MZ
00145 38*      DO 115 J=1,MZ
00150 39*      115 TMP(J,I) = Y(K+J)
00153 40*      DO 120 J=1,MZ
00156 41*      TW(J,NRTB) = Y(LL4+J)
00157 42*      DO 120 I=1,NRTB1
00162 43*      K = LTB+(I-1)*MZ+J
00163 44*      120 TW(J,I) = Y(K)
00163 45*      C
00166 46*      GO TO 140
00167 47*      130 DO 135 J=1,MZ
00172 48*      K = LL4+J
00173 49*      TW(J,1) = Y(K)
00174 50*      TMP(J,NRMP) = Y(K)
00175 51*      135 TEMP(J,1) = Y(K)
00175 52*      C
00177 53*      140 TABS = TIME*TREF*3600.0
00200 54*      TX = TABS/60.0
00201 55*      IF(NCONV.GT.0) CALL CONVEC (TABS)
00203 56*      CALL GRAD
00203 57*      C
00203 58*      C
00203 59*      C      FIN
00203 60*      C
00203 61*      C
00204 62*      DO 2 I=2,NM1
00207 63*      DO 1 J=1,MZ
00212 64*      1 DTF(J) = TEMP(J,1)
00214 65*      CALL ODX (DTF,DP,DZ,MZ)
00215 66*      CALL D2OX2(DTF, DW,DZ,MZ)
00216 67*      DO 2 J=2,MM1
00221 68*      DZTW(J,1) = DP(J)
00222 69*      2 D2ZTW(J,1) = DW(J)
00222 70*      C
00225 71*      DO 4 J=2,MM1

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00230 72* DO 3 I=1,NX
00233 73* 3 DTF(I) = TEMP(J,I)
00235 74* CALL DDX (DTF,DP,DXI,NX)
00236 75* CALL D2DX2(DTF,DW,DXI,NX)
00237 76* DO 4 I=2,NM1
00242 77* DXTW(J,I) = DP(I)
00243 78* 4 D2XTW(J,I) = DW(I)
00243 79* C
00246 80* DO 9 I=2,NM1
00251 81* IM2 = I-2
00252 82* DO 9 J=2,MM1
00255 83* K = IM2*MZ+J
00256 84* TT = T0+TEMP(J,I)
00257 85* CON = THCFN(TT)
00260 86* DCON = DTHCFN(TT)*T0
00261 87* DIFF = CON/(RHOFN*CPFN(TT))*FXHW
00262 88* CPARM = FRD/CON
00262 89* C
00263 90* COND = D2XTW(J,I)+FXOH*D2XTW(J,I)-COB(I)*DXTW(J,I)
00263 91* 1 +DCON*(DXTW(J,I)**2+FXOH*D2XTW(J,I)**2)
00264 92* 9 DY(K) = DIFF*(COND-DOB(I)*CPARM*(GRFN(J,I)-CONFN(J,I)))
00264 93* C
00264 94* C TIP OF FIN
00264 95* C
00267 96* NM2 = NX-2
00270 97* DO 11 J=2,MM1
00273 98* K = NM2*MZ+J
00274 99* 11 DY(K) = (18.*DY(K-MZ)-9.*DY(K-2*MZ)+2.*DY(K-3*MZ))/11.
00274 100* C
00274 101* C
00274 102* C
00274 103* C
00274 104* C FLUID FLOW VARIABLES
00274 105* C
00274 106* C
00276 107* DO 500 I=1,MZ
00301 108* 500 FT(I) = Y(I+LT)
00301 109* C
00303 110* IF (LFLD.EQ. 1) GO TO 502
00305 111* XTME = TIME*TREF
00306 112* FT(1) = YINT(TMEFLD,TIFLD,NFLDTA,3,XTME)/T0
00307 113* FLDINT = FT(1)
00310 114* WIFLOT = YINT(TMEFLD,WIFLD,NFLDTA,3,XTME)
00311 115* WRAT = WIFLOT/FLUXI
00311 116* C
00312 117* 502 TFLC = FT(LC)*T0
00313 118* PFLC = FP(LC)*P0
00314 119* RHOC = RHOF(PFLC,TFLC)
00315 120* XVISC = VISC(RHOC,TFLC)
00316 121* XTHCF = THCF(RHOC,TFLC)
00317 122* RETR = XRE/XVISC
00320 123* PRTR = (XVISC*CPF(RHOC,TFLC))/XTHCF
00321 124* TFLIN = Y(LT+1)
00322 125* CALL FLSTRT(RETR,PRTR,DELTA)
00323 126* X00 = DELTA*RETR*PRTR*XTHCF/4.0
00323 127* C
00324 128* CALL DDX(FW,DW,DZ,MZ)

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00325 129* CALL DDX(FT,DTF,DZ,MZ)
00325 130* C
00326 131* DY(LT+1) = 0.0
00327 132* IF (LFLD.EQ. 2) GO TO 506
00331 133* IF (TIME.LE.2.5) DY(LT+1) = 7.0*EXP(-7.0*TIME)*(1.0-TINTL)
00333 134* 506 AUX2(1) = FNU*(TW(1,1)-FT(1))
00334 135* AUX2(1) = -X00*AUX2(1)
00334 136* C
00335 137* DO 520 I=2,MZ
00340 138* PP = P0*FP(I)
00341 139* TT = T0*FT(I)
00342 140* RHO = 1.0/FW(I)
00343 141* RRHO = RH00*RHO
00344 142* CCP = CPF(RRHO,TT)
00344 143* C
00345 144* GKAPPA = CAPPA(RRHO,TT)*P0
00346 145* GBETA = BETA(RRHO,TT)*T0
00347 146* GCP = CCP/CP0
00350 147* AUX2(I) = FNU*(TW(I,1)-FT(I))
00351 148* LLT = LT+I
00352 149* IF (GKAPPA.LT. 1.0E-08) GO TO 510
00354 150* X1 = FZ*GBETA*FT(I)/(GKAPPA*GCP*RHO)
00355 151* GO TO 515
00356 152* 510 X1 = 0.0
00357 153* 515 X2 = 1.0-X1*GBETA
00360 154* DY(LLT) = ((AUX2(I)+FMACH*FR*(FW(I)**2)/2.0)/(RHO*GCP)-X1*DW(I))/
00360 155* 1 X2-FW(I)*DTF(I)
00361 156* 520 AUX2(I) = -X00*AUX2(I)
00361 157* C
00361 158* C
00361 159* C TUBE WALL TEMPERATURE
00361 160* C
00363 161* IF(NRTB.EQ.0) GO TO 900
00365 162* DO 625 J=1,MZ
00370 163* DO 615 I=1,NRTB
00373 164* 615 DTF(I) = TW(J,I)
00375 165* CALL DDX(DTF,D2TF,DXITB,NRTB)
00376 166* DO 620 I=1,NRTB
00401 167* 620 DP(I) = XITB(I)*D2TF(I)
00403 168* CALL DDX(DP,DTF,DXITB,NRTB)
00404 169* DO 625 I=2,NRTB1
00407 170* D2XTW(J,I) = DTF(I)/XITB(I)
00410 171* 625 DXTW(J,I) = (D2TF(I))**2
00410 172* C
00413 173* DO 630 I=2,NRTB1
00416 174* K = LTB+(I-1)*MZ
00417 175* DO 630 J=1,MZ
00422 176* TTB = TW(J,I)*T0
00423 177* GFO = THCTB(TTB)/CPTB(TTB)*FF0
00424 178* GDTHC = T0*DTHTCB(TTB)
00425 179* 630 DY(K+J) = GFO*(D2XTW(J,I)+GDTHC*(DXTW(J,I)))
00430 180* DO 635 J=1,MZ
00433 181* TTB = TW(J,1)*T0
00434 182* GFNU = XTHCF*FNU/(4.0*THCTB(TTB))*RETR*PRTR*DELTA*DXITB
00435 183* Y1 = TW(J,1)-FT(J)
00436 184* X1 = 4.0*DY(LTB*MZ+J)-DY(LTB*MZ+J)
00437 185* X2 = 3.0*GFNU*(1.0-T0*DTHTCB(TTB)*Y1)

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00440 186* 635 DY(LTB+J) = (X1+GFNU*DY(LT+J))/X2
00440 187* C
00440 188* C
00440 189* C
00440 190* C PROTECTION LAYER TEMPERATURE
00440 191* C
00440 192* C
00442 193* DO 730 J=1,MZ
00445 194* DO 720 I=1,NRMP
00450 195* 720 DTF(I) = TMP(J,I)
00452 196* CALL DDX(DTF,D2TF,DXIMP,NRMP)
00453 197* DO 725 I=1,NRMP
00456 198* 725 DP(I) = XIMP(I)*D2TF(I)
00460 199* CALL DDX(DP,DTF,DXIMP,NRMP)
00461 200* DO 730 I=2,NRMP1
00464 201* D2XTW(J,I) = DTF(I)/XIMP(I)
00465 202* 730 DXTW(J,I) = D2TF(I)**2
00465 203* C
00470 204* DO 735 I=2,NRMP1
00473 205* K = LMP+(I-2)*MZ
00474 206* DO 735 J=1,MZ
00477 207* TTM = TMP(J,I)*T0
00500 208* GFO = THCMP(TTM)/CPMP(TTM)*FOF
00501 209* GDTHC = T0*DTHCM(TTM)
00502 210* 735 DY(K+J) = GFO*(D2XTW(J,I)+GDTHC*(DXTW(J,I)))
00502 211* C
00502 212* C
00502 213* C
00502 214* C FIN - TUBE - PROTECTION LAYER INTERFACE
00502 215* C
00505 216* DO 740 J=1,MZ
00510 217* XT = TW(J,1)*T0
00511 218* X1 = DTHCTB(XT)
00512 219* X2 = DTHCMP(XT)
00513 220* X3 = DTHCFN(XT)
00514 221* XTHCT = THCTB(XT)
00515 222* XX1 = PHIM*THCMP(XT)/XTHCT
00516 223* XX2 = PHIF*THCFN(XT)/XTHCT
00517 224* XX5 = (X2-X1)*XX3*(-3.0*TW(J,NRTB)+4.0*TMP(J,2)-TMP(J,3))*T0
00520 225* XX6 = (X3-X1)*XX4*(-3.0*TW(J,NRTB)+4.0*TEMP(J,2)-TEMP(J,3))*T0
00520 226* C
00521 227* YY1 = 4.0*DY(LL1+J)-DY(LL2+J)
00522 228* YY1 = YY1+XX1*XX3*(4.0*DY(LMP+J)-DY(LL3+J))
00523 229* YY1 = YY1+XX2*XX4*(4.0*DY(J)-DY(MZ+J))
00524 230* 740 DY(LL4+J) = YY1/(3.0*XX1*(3.0*XX3-XX5)+XX2*(3.0*XX4-XX6))
00524 231* C
00524 232* C
00524 233* C MANIFOLD-FIN INTERFACE
00524 234* C
00524 235* C
00526 236* DO 750 I=1,NM1
00531 237* K = I*MZ-(MZ-1)
00532 238* DY(K) = DY(LL4+1)
00533 239* K = I*MZ
00534 240* 750 DY(K) = DY(LL4+MZ)
00534 241* C
00534 242* C

```

\*\*\*\*\* DERIV \*\*\*\*\*

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```

00534 243* C
00534 244* C
00534 245* C OUTER BOUNDARY OF PROTECTION LAYER
00534 246* C
00534 247* C
00536 248* DO 810 J=1,MZ
00541 249* TTMBAR = (8.0*TMP(J,NRMP)+5.0*TMP(J,NRMP1)-TMP(J,NRMP2))*T0/12.
00542 250* TTHCMP = THCMP(TTM)
00543 251* GFO = TTHCMP*F0F/CPMP(TTMBAR)
00544 252* GOTHC = T0*DTTHCMP(TTM)
00545 253* X1 = TMP(J,NRMP)-TMP(J,NRMP1)
00546 254* X2 = 1.0-GOTHC*(3.0*TMP(J,NRMP)-4.0*TMP(J,NRMP1)+
00546 255* 1 TMP(J,NRMP2))/4.0
00547 256* X4 = FRAD*XX12*(GRMP(J)+CONMP(J))/TTHCMP
00550 257* X5 = -X1*XX11*X2-X4
00551 258* 810 DY(LL5+J) = (12.0*X5*GFO-5.0*DY(LL6+J)+DY(LL7+J))/8.0
00551 259* C
00553 260* RETURN
00553 261* C
00554 262* 900 DO 905 J=1,MZ
00557 263* K = LL4+J
00560 264* TT = Y(K)*T0
00561 265* TFLC = T0*FT(J)
00562 266* RHOC = RH00/FW(J)
00563 267* X01 = XX10*CPTB(TT)+XX11*CPMP(TT)+XX12*CPFN(TT)
00564 268* X03 = XX4*THCFN(TT)
00565 269* IF(J.NE.1) GO TO 901
00567 270* TMPGRD = (5.0*(TEMP(2,2)-TEMP(2,1))+TEMP(3,1)-TEMP(3,2))/XX48
00570 271* GO TO 903
00571 272* 901 IF(J.NE.MZ) GO TO 902
00573 273* TMPGRD = (5.0*(TEMP(MM1,2)-TEMP(MM1,1))+TEMP(LL6,1)
00573 274* 1 -TEMP(LL6,2))/XX48
00574 275* GO TO 903
00575 276* 902 TMPGRD = (TEMP(J,2)-TEMP(J,1))/DXI
00576 277* 903 QFN = QRFN(J,1)+CONFN(J,1)
00577 278* QMP = GRMP(J)+CONMP(J)
00600 279* 905 DY(K) = (AUX2(J)-FRAD*(QMP+XX20*QFN)+X03*TMPGRD)/X01
00600 280* C
00602 281* K1 = LL4+1
00603 282* KZ = LL4+MZ
00604 283* DO 910 I=1,NM1
00607 284* K = I*MZ
00610 285* DY(K) = DY(KZ)
00611 286* K = K-MZ+1
00612 287* 910 DY(K) = DY(K1)
00614 288* RETURN
00615 289* END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG.P \*\*\*\*\* DTHCFN/FNAL \*\*\*\*\*

\*\*\*\*\* DTHCFN/FNAL \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME NASAS.DTHCFN/FNAL ME NASAS.DTHCFN/FNAL  
FOR S9A-07/13/72-20:55:36 (0.)

FUNCTION DTHCFN ENTRY POINT 000033

STORAGE USED: CODE(1) 000040; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCFN  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 R 000011 DTHC	0000 R 000000 DTHCFN	0000 000012 INJPS	0000 R 000010 THC
0003 R 000000 THCFN	0000 R 000007 THETA	0000 R 000005 TO	0000 R 000001 X1	0000 R 000002 X2
0000 R 000003 X3	0000 R 000004 X4			

```
00101 1*      FUNCTION DTHCFN (T)
00101 2*      C
00101 3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4*      C      (1/K)(DK/DT) AS A FUNCTION OF TEMPERATURE (R) OF ALUMINUM 7075
00101 5*      C      UNITS (1/R)
00101 6*      C      TEMP. GE. 300.0 AND LE. 1200.0 R
00101 7*      C
00103 8*      DATA X1,X2,X3,X4 /13.0665,0.6655,-5.25,1.015/,TO,DT /400.0,200.0/
00112 9*      THETA = (T-TO)/DT
00113 10*     THC = THCFN(T)
00114 11*     DTHC = (((X4*THETA+X3)*THETA+X2)*THETA+X1)/DT
00115 12*     DTHCFN = DTHC/THC
00116 13*     RETURN
00117 14*     END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHD6 P \*\*\*\*\* DTHCFN/FNBR \*\*\*\*\*

\*\*\*\*\* DTHCFN/FNBR \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME NASAS.DTHCFN/FNBR ME NASAS.DTHCFN/FNBR  
FOR 59A-07/13/72-20:55:39 (0.)

FUNCTION DTHCFN ENTRY POINT 000033

STORAGE USED: CODE(1) 000040 DATA(0) 000017 BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCMP  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 R 000011 DTHC	0000 R 000000 DTHCFN	0000 000012 INJPS	0000 R 000010 THC
0003 R 000000 THCMP	0000 R 000007 THETA	0000 R 000005 TD	0000 R 000001 X1	0000 R 000002 X2
0000 R 000003 X3	0000 R 000004 X4			

```
00101 1*      FUNCTION DTHCFN(T)
00101 2*      C
00101 3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4*      C      (1/K)(DK/DT) AS A FUNCTION OF TEMPERATURE (R) OF BERYLIUM WITH
00101 5*      C      0.84-1.68 BE0
00101 6*      C      UNITS (1/R)
00101 7*      C      TEMP. GE. 400.0 AND LE. 1700.0 R
00101 8*      C
00103 9*      DATA X1,X2,X3,X4 /-10.5643,1.65366,-0.60501,0.080668/,T0,DT /400.0
00103 10*     1,200.0/
00112 11*     THETA = (T-T0)/DT
00113 12*     THC   = THCMP(T)
00114 13*     DTHC  = (((X4*THETA+X3)*THETA+X2)*THETA+X1)/DT
00115 14*     DTHCFN = DTHC/THC
00116 15*     RETURN
00117 16*     END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG P \*\*\*\*\* DTHCFN/FNBR \*\*\*\*\*

\*\*\*\*\* DTHCFN/FNCJ \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASAS.DTHCFN/FNCU ME\*NASAS.DTHCFN/FNCU  
FOR S9A-07/13/72-20:55:41 (0.)

FUNCTION DTHCFN ENTRY POINT 000030

STORAGE USED: CODE(1) 000035; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCTB  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000004 DT	0000 R 000007 DTHC	0000 R 000000 DTHCFN	0000 000010 INJPS	0000 R 000006 THC
0003 R 000000 THCTB	0000 R 000005 THETA	0000 R 000003 T0	0000 R 000001 X1	0000 R 000002 X2

```
00101 1* FUNCTION DTHCFN(T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C (1/K)(DK/DJ) AS A FUNCTION OF TEMPERATURE (R) OF COPPER
00101 5* C UNITS (1/R)
00101 6* C TEMP. GE. 500.0 AND LE. 1800.0 R
00101 7* C
00103 8* DATA X1,X2 /-2.62067,-0.121/.T0/DT /600.0/200.0/
00110 9* THETA = (T-T0)/DT
00111 10* THC = THCTB(T)
00112 11* DTHC = (X2*THETA*THETA+X1)/DT
00113 12* DTHCFN = DTHC/THC
00114 13* RETURN
00115 14* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG P \*\*\*\*\* DTHCMP/MPAL \*\*\*\*\*

\*\*\*\*\* DTHCMP/MPAL \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASA5.DTHCMP/MPAL ME\*NASA5.DTHCMP/MPAL  
FOR S9A-07/13/72-20:55:44 (0,)

FUNCTION DTHCMP ENTRY POINT 000033

STORAGE USED: CODE(1) 000040; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCFN  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 R 000011 DTHC	0000 R 000000 DTHCMP	0000 000012 INJPS	0000 R 000010 THC
0003 R 000000 THCFN	0000 R 000007 THETA	0000 R 000005 T0	0000 R 000001 X1	0000 R 000002 X2
0000 R 000003 X3	0000 R 000004 X4			

00101	1*		FUNCTION DTHCMP(T)
00101	2*	C	
00101	3*	C	THIS FUNCTION SUBPROGRAM COMPUTES :
00101	4*	C	(1/K)(DK/DT) AS A FUNCTION OF TEMPERATURE (R) OF ALUMINUM 7075
00101	5*	C	UNITS (1/R)
00101	6*	C	TEMP. GE. 300.0 AND LE. 1200.0 R
00101	7*	C	
00103	8*		DATA X1,X2,X3,X4 /13.0665,0.6655,-5.25,1.015/,T0,DT /400.0,200.0/
00112	9*		THETA = (T-T0)/DT
00113	10*		THC = THCFN(T)
00114	11*		DTHC = (((X4*THETA+X3)*THETA+X2)*THETA+X1)/DT
00115	12*		DTHCMP = DTHC/THC
00116	13*		RETURN
00117	14*		END

END OF COMPILATION: NO DIAGNOSTICS.

QHDG P \*\*\*\*\* DTHCMP/MPBR \*\*\*\*\*

\*\*\*\*\* DTHCMP/MPBR \*\*\*\*\*

DATE 071372

PAGE 1

QFOR: S ME\*NASA5.DTHCMP/MPBR; ME\*NASA5.DTHCMP/MPBR  
FOR S9A-07/13/72-20:55:46 (0.)

FUNCTION DTHCMP ENTRY POINT 000033

STORAGE USED: CODE(1) 000040; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCMP  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 R 000011 DTHC	0000 R 000000 DTHCMP	0000 000012 INJPS	0000 R 000010 THC
0003 R 000000 THCMP	0000 R 000007 THETA	0000 R 000005 T0	0000 R 000001 X1	0000 R 000002 X2
0000 R 000003 X3	0000 R 000004 X4			

```
00101 1* FUNCTION DTHCMP (T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C (1/K)(DK/DT) AS A FUNCTION OF TEMPERA3URE (R) OF BERYLIUM WITH
00101 5* C 0.84-1.68 BEQ
00101 6* C UNITS (1/R)
00101 7* C TEMP. GE. 400.0 AND LE. 1700.0 R
00101 8* C
00103 9* DATA X1,X2,X3,X4 /-10.5643,1.65366,-0.60501,0.080668/,T0,DT /400.0
00103 10* 1,200.0/
00112 11* THETA = (T-T0)/DT
00113 12* THC = THCMP(T)
00114 13* DTHC = (((X4+THETA+X3)*THETA+X2)*THETA+X1)/DT
00115 14* DTHCMP = DTHC/THC
00116 15* RETURN
00117 16* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* DTHCMP/MPCU \*\*\*\*\*

\*\*\*\*\* DTHCMP/MPCU \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASA5.DTHCMP/MPCU,ME\*NASA5.DTHCMP/MPCU  
FOR S9A-07/13/72-20:55:50 (0,)

FUNCTION DTHCMP ENTRY POINT 000030

STORAGE USED: CODE(1) 000035; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCTB  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000004 DT	0000 R 000007 DTHC	0000 R 000000 DTHCMP	0000 000010 INJPS	0000 R 000006 THC
0003 R 000000 THCTB	0000 R 000005 THETA	0000 R 000003 T0	0000 R 000001 X1	0000 R 000002 X2

```
00101      1*      FUNCTION DTHCMP(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      (1/K)(DK/DT) AS A FUNCTION OF TEMPERATURE (R) OF COPPER
00101      5*      C      UNITS (1/R)
00101      6*      C      TEMP. GE. 500.0 AND LE. 1800.0 R
00101      7*      C
00103      8*      DATA X1,X2 /-2.62067,-0.121/,T0,DT /600.0,200.0/
00110      9*      THETA = (T-T0)/DT
00111     10*      THC   = THCTB(T)
00112     11*      DTHC  = (X2*THETA*THETA+X1)/DT
00113     12*      DTHCMP = DTHC/THC
00114     13*      RETURN
00115     14*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* DTHCTB/TBAL \*\*\*\*\*



\*\*\*\*\* DTHCTB/TBAL \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASAS.DTHCTB/TBAL,ME\*NASAS.DTHCTB/TBAL  
FOR S9A-07/13/72-20:55:53 (0,)

FUNCTION DTHCTB ENTRY POINT 000033

STORAGE USED: CODE(1) 000040; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCFN  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 R 000011 DTHC	0000 R 000000 DTHCTB	0000 000012 INJPS	0000 R 000010 THC
0003 R 000000 THCFN	0000 R 000007 THETA	0000 R 000005 T0	0000 R 000001 X1	0000 R 000002 X2
0000 R 000003 X3	0000 R 000004 X4			

```
00101      1*      FUNCTION DTHCTB(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      (1/K)(DK/DT) AS A FUNCTION OF TEMPERATURE (R) OF ALUMINUM 7075
00101      5*      C      UNITS (1/R)
00101      6*      C      TEMP. GE. 300.0,AND,LE. 1200.0 R
00101      7*      C
00103      8*      DATA X1,X2,X3,X4 /13.0665,0.6655,-5.25,1.015/,T0,DT /400.0,200.0/
00112      9*      THETA = (T-T0)/DT
00113     10*      THC   = THCFN(T)
00114     11*      DTHC  = (((X4*THETA+X3)*THETA+X2)*THETA+X1)/DT
00115     12*      DTHCTB = DTHC/THC
00116     13*      RETURN
00117     14*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHD6,P \*\*\*\*\* DTHCTB/TB8R \*\*\*\*\*

\*\*\*\*\* DTHCTB/TBBR \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASA5.DTHCTB/TBBR ME\*NASA5.DTHCTB/TBBR  
FOR S9A-07/13/72-20:55:56 (0.)

FUNCTION DTHCTB ENTRY POINT 000033

STORAGE USED: CODE(1) 000040; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCMP  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 R 000011 DTHC	0000 R 000000 DTHCTB	0000 000012 INJPS	0000 R 000010 THC
0003 R 000000 THCMP	0000 R 000007 THETA	0000 R 000005 T0	0000 R 000001 X1	0000 R 000002 X2
0000 R 000003 X3	0000 R 000004 X4			

```

00101      1*      FUNCTION DTHCTB(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      (1/K)(DK/DT) AS A FUNCTION OF TEMPERA3URE (R) OF BERYLIUM WITH
00101      5*      C      0.84-1.68 BEO
00101      6*      C      UNITS (1/R)
00101      7*      C      TEMP. GE. 400.0 AND LE. 1700.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4 /-10.5643,1.65366,-0.60501,0.080668/,T0,DT /400.0
00103     10*      1,200.0/
00112     11*      THETA = (T-T0)/DT
00113     12*      THC = THCMP(T)
00114     13*      DTHC = (((X4*THETA+X3)*THETA+X2)*THETA+X1)/DT
00115     14*      DTHCTB = DTHC/THC
00116     15*      RETURN
00117     16*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG P \*\*\*\*\* DTHCTB/TBCU \*\*\*\*\*

\*\*\*\*\* DTHCTB/TBCU \*\*\*\*\*

DATE 071372

PAGE

1

QFOR S ME\*NASA5.DTHCTB/TBCU ME\*NASA5.DTHCTB/TBCU  
FOR 59A-07/13/72-20:55:58 (0.)

FUNCTION DTHCTB ENTRY POINT 000030

STORAGE USED: CODE(1) 000035; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 THCTB  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000004 DT	0000 R 000007 DTHC	0000 R 000000 DTHCTB	0000 000010 INJPS	0000 R 000006 THC
0003 R 000000 THCTB	0000 R 000005 THETA	0000 R 000003 T0	0000 R 000001 X1	0000 R 000002 X2

```
00101 1*      FUNCTION DTHCTB (T)
00101 2*      C
00101 3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4*      C      (1/K)(DK/DT) AS A FUNCTION OF TEMPERATURE (R) OF COPPER
00101 5*      C      UNITS (1/R)
00101 6*      C      TEMP. GE. 500.0 AND LE. 1800.0 R
00101 7*      C
00103 8*      DATA X1,X2 /-2.62067,-0.121/T0,DT /600.0,200.0/
00110 9*      THETA = (T-T0)/DT
00111 10*     THC = THCTB(T)
00112 11*     DTHC = (X2*THETA*THETA*X1)/DT
00113 12*     DTHCTB = DTHC/THC
00114 13*     RETURN
00115 14*     END
```

END OF COMPILATION: NO DIAGNOSTICS.

DHDG:P \*\*\*\*\* D2DX2 \*\*\*\*\*

\*\*\*\*\* D2DX2 \*\*\*\*\*

DATE 071372

PAGE 1

QFOR:5 ME\*NASA5.D2DX2,ME\*NASA5.D2DX2  
FOR 59A-07/13/72-20:56:00 (0,)

SUBROUTINE D2DX2 ENTRY POINT 000137

STORAGE USED: CODE(1) 000163; DATA(0) 000046; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ\$  
0004 NI02\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000064 112G 0001 000112 2L 0000 000003 3F 0000 R 000000 DX2 0000 I 000002 I  
0000 000016 INJP\$ 0000 I 000001 N1

00101	1*	SUBROUTINE D2DX2 (Y,D2Y,DX,N)	1
00101	2*		
00101	3*	C THIS SUBROUTINE COMPUTES :	
00101	4*	C THE SECOND DERIVATIVE OF THE EQUALLY-SPACED ARRAY Y(N)	
00101	5*	C	
00103	6*	DIMENSION Y(N),D2Y(N)	2
00104	7*	IF (N.LT.4) GO TO 2	3
00106	8*	DX2 = DX**2	4
00107	9*	D2Y(1) = (2.0*Y(1)-5.0*Y(2)+4.0*Y(3)-Y(4))/DX2	5
00110	10*	N1 = N-1	6
00111	11*	DO 1 I=2,N1	7
00114	12*	1 D2Y(I) = (Y(I+1)-2.0*Y(I)+Y(I-1))/DX2	8
00116	13*	D2Y(N) = (2.0*Y(N)-5.0*Y(N-1)+4.0*Y(N-2)-Y(N-3))/DX2	9
00117	14*	RETURN	10
00120	15*	2 WRITE (6,3)	11
00122	16*	3 FORMAT (27H02DX2 NOT ENOUGH NODAL PTS.,/)	12
00123	17*	RETURN	13
00124	18*	END	14

END OF COMPILATION: NO DIAGNOSTICS.

QHD6.P \*\*\*\*\* EFFICY \*\*\*\*\*

\*\*\*\*\* EFFICY \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASA5.EFFICY,ME\*NASA5.EFFICY  
FOR S9A-07/13/72-20:56:02 (0,)

SUBROUTINE EFFICY ENTRY POINT 000134

STORAGE USED: CODE(1) 000161; DATA(0) 000131; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000007	116G	0001	000011	121G	0001	000040	132G	0001	000042	135G	0001	000073	146G	
0000	R	000064	A	0000	R	000104	AK	0000	R	000073	APR	0000	R	000000	C
0000	R	000055	COEFF	0000	R	000043	COF	0000	I	000102	I	0000	000114	INJP5	
0000	I	000105	K	0003	R	000000	POLY					0000	I	000103	J

```

00101      1*      SUBROUTINE EFFICY (NC,NCPR,TSOTB,TB,ETA,ETAPR)
00103      2*      DIMENSION C(5,7),COF(5),COEFF(7),A(7),APR(7)
00104      3*      REAL NC,NCPR
00105      4*      DATA(C(I,2),I=1,5)/-1.1632,.1633405,-1.629484,3.735489,-2.75355/,
00105      5*      1 (C(I,3),I=1,5)/1.438967,-.5483718,5.091788,-11.26815,7.905002/,
00105      6*      2 (C(I,4),I=1,5)/-1.158833,.7783245,-6.866955,14.98945,-10.35581/,
00105      7*      3 (C(I,5),I=1,5)/.537,-.5282332,4.460133,-9.675815,6.65502/,
00105      8*      4 (C(I,6),I=1,5)/-.1294667,.1666654,-1.360822,2.94164,-2.020815/,
00105      9*      5 (C(I,7),I=1,5)/.01253334,-.01962952,.1562953,-.3370346,.2314797/
00114     10*      A(1) = 1.0
00115     11*      DO 20 I=2,7
00120     12*      DO 10 J=1,5
00123     13*      10 COEF(J) = C(J,I)
00125     14*      20 A(I) = POLY (5,COEF,TSOTB)
00127     15*      ETA = POLY (7,A,NC)
00130     16*      APR(1) = 0.0
00131     17*      DO 40 I=2,7
00134     18*      DO 30 J=1,5
00137     19*      AK = J-1
00140     20*      30 COF(J) = AK*C(J,I)
00142     21*      40 APR(I) = -(1.0/TB)*POLY(5,COF,TSOTB)
00144     22*      COEFF(7)= APR(7)*NC**4
00145     23*      DO 50 I=1,6
00150     24*      K = I+1
00151     25*      AK = I
00152     26*      50 COEFF(I)= APR(I)*AK*A(K)*NCPR
00154     27*      ETAPR = POLY (7,COEFF,NC)
00155     28*      RETURN
00156     29*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

\*\*\*\*\* EFFICY \*\*\*\*\*

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QHDG:P \*\*\*\*\* ELAS/MPAL \*\*\*\*\*

\*\*\*\*\* ELAS/MPAL \*\*\*\*\*

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PAGE 1

QFOR: S ME\*NASAS.ELAS/MPAL,ME\*NASAS.ELAS/MPAL  
FOR S9A-07/13/72-20:56:06 (0,)

FUNCTION ELAS ENTRY POINT 000055

STORAGE USED: CODE(1) 000061; DATA(0) 000040; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NI02\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000010	IF	0000	R	000006	DT	0000	R	000000	ELAS	0000	000032	INJPS	0000	R	000007	THETA		
0000	R	000005	TO	0000	R	000001	X1	0000	R	000002	X2	0000	R	000003	X3	0000	R	000004	X4

```
00101      1*      FUNCTION ELAS(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      MODULUS OF ELASTICITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      C      ALUMINUM 7075
00101      6*      C      UNITS LBF/SQ.FT
00101      7*      C      TEMP. GE. 300.0.AND.LE. 1200.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4 /10.71,-0.63,-0.115,-0.06/,T0,DT /459.67,200.0/
00112     10*      THETA = (T-T0)/DT
00113     11*      ELAS = (((X4*THETA+X3)*THETA+X2)*THETA+X1)*1.0E06
00114     12*      ELAS = ELAS*144.0
00115     13*      IF (T.GE.300.0.AND.T.LE.1200.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,59HMODULUS OF ELASTICITY OF ALUMINUM, TEMP. OUT OF RAN
00122     16*      1GE, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* ELAS/MPBR \*\*\*\*\*

\*\*\*\*\* ELAS/MPBR \*\*\*\*\*

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PAGE 1

QFOR,S ME\*NASA5.ELAS/MPBR,ME\*NASA5.ELAS/MPBR  
FOR S9A-07/13/72-20:56:10 (0,)

FUNCTION ELAS ENTRY POINT 000055

STORAGE USED: CODE(1) 000061; DATA(0) 000040; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NIO2S  
0005 NERR3S

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000010	IF	0000	R	000006	DT	0000	R	000000	ELAS	0000	000032	INJPS	0000	R	000007	THETA		
0000	R	000005	T0	0000	R	000001	X1	0000	R	000002	X2	0000	R	000003	X3	0000	R	000004	X4

```
00101      1*      FUNCTION ELAS(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      MODULUS OF ELASTICITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      C      BERYLIUM WITH 0.84-1.68 BE0
00101      6*      C      UNITS LBF/SQ.FT
00101      7*      C      TEMP. GE. 400.0 AND LE. 1700.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4 /44.36,-3.755,0.335,-0.53/,T0,DT /459.67,400.0/
00112     10*      THETA = (T-T0)/DT
00113     11*      ELAS = (((X4*THETA+X3)*THETA+X2)*THETA+X1)*1.0E06
00114     12*      ELAS = ELAS*144.0
00115     13*      IF (T.GE.400.0.AND.T.LE.1700.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,59HMODULUS OF ELASTICITY OF BERYLIUM, TEMP. OUT OF RAN
00122     16*      1GE, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* ELAS/MPCU \*\*\*\*\*



\*\*\*\*\* ELAS/MPCU \*\*\*\*\*

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PAGE 1

QFOR:5 ME\*NASAS.ELAS/MPCU,ME\*NASAS.ELAS/MPCU  
FOR S9A-07/13/72-20:56:13 (0.)

FUNCTION ELAS ENTRY POINT 000055

STORAGE USED: CODE(1) 000061; DATA(0) 000040; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ\$  
0004 NI02\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000010 IF	0000 R 000006 DT	0000 R 000000 ELAS	0000	000032 INJP\$	0000 R 000007 THETA
0000 R	000005 TO	0000 R 000001 X1	0000 R 000002 X2	0000 R	000003 X3	0000 R 000004 X4

```
00101      1*      FUNCTION ELAS(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      MODULUS OF ELASTICITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      C      COPPER
00101      6*      C      UNITS LBF/SQ.FT
00101      7*      C      TEMP. GE. 400.0 AND LE. 1800.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4 /16.55,-0.4933,-1.935,0.2283/,T0,DT /459.67,400./
00112     10*      THETA = (T-T0)/DT
00113     11*      ELAS = (((X4*THETA+X3)*THETA+X2)*THETA+X1)*1.0E06
00114     12*      ELAS = ELAS*144.0
00115     13*      IF (T.GE.400.0.AND.T.LE.1800.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,57HMODULUS OF ELASTICITY OF COPPER, TEMP. OUT OF RANGE
00122     16*      1, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* EMIT/SCZ93 \*\*\*\*\*

\*\*\*\*\* EMIT/SCZ93 \*\*\*\*\*

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PAGE 1

QFOR,S ME\*NASAS.EMIT/SCZ93,ME\*NASAS.EMIT/SCZ93  
FOR S9A-07/13/72-20:56:16 (0,)

FUNCTION EMIT ENTRY POINT 000030

STORAGE USED: CODE(1) 000035; DATA(0) 000016; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000016 1L 0000 R 000001 A 0000 R 000000 EMIT 0000 000011 INJPS 0003 R 000000 POLY

```
00101 1* FUNCTION EMIT(T)
00101 2* C
00101 3* C THIS SUBROUTINE COMPUTES :
00101 4* C THE TOTAL HEMISPHERICAL EMITTANCE AS A FUNCTION OF THE SURFACE
00101 5* C TEMPERATURE (R)
00101 6* C
00103 7* DIMENSION A(5)
00104 8* DATA A(1),A(2),A(3),A(4),A(5)/0.8990103E+00, -0.1400633E-03,
00104 9* 1 0.3879700E-06, -0.3937509E-09, 0.1015627E-12/
00112 10* IF(T.GE.1600.0) GO TO 1
00114 11* EMIT = POLY(5,A,T)
00115 12* RETURN
00116 13* 1 EMIT = 0.228
00117 14* RETURN
00120 15* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHOG,P \*\*\*\*\* ENTAIR \*\*\*\*\*

\*\*\*\*\* ENTAIR \*\*\*\*\*

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PAGE 1

QFOR,S ME\*NASAS.ENTAIR,ME\*NASAS.ENTAIR  
FOR S9A-07/13/72-20:56:20 (0,)

FUNCTION ENTAIR ENTRY POINT 000204

STORAGE USED: CODE(1) 000225; DATA(0) 000071; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 ALOG  
0004 SQRT  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000157	10L	0001	000103	5L	0000	R	000001	A	0000	R	000000	ENTAIR	0000	R	000032	ENTN		
0000	R	000033	ENTJ	0000	R	000013	ENT0	0000	R	000015	ENT00	0000	R	000007	FMN	0000	R	000006	FMO
0000	000053	INJPS	0000	R	000031	TLOG	0000	R	000030	TMT	0000	R	000026	TMO	0000	R	000027	TOT	
0000	R	000012	TTO	0000	R	000014	T00	0000	R	000022	T02	0000	R	000023	T03	0000	R	000024	T04
0000	R	000025	T05	0000	R	000016	T2	0000	R	000017	T3	0000	R	000020	T4	0000	R	000021	T5
0000	R	000011	WMN	0000	R	000010	WMO												

```

00101      1*      FUNCTION ENTAIR(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      ENTHALPY OF AIR AS A FUNCTION OF TEMPERATURE (R)
00101      5*      C      UNITS BTU/LBM
00101      6*      C
00103      7*      DIMENSION A(5)
00104      8*      DATA A(1),A(2),A(3),A(4),A(5)/0.34240E+00,-0.95225E-03,
00104      9*      1 0.31862E-05,-0.45750E-08,0.23750E-11/
00112     10*      DATA FMO,FMN,WMO,WMN,TTO,ENT0/0.234559,0.765441,31.9988,28.0134,
00112     11*      1 600.,143.47 /
00121     12*      DATA T00,ENT00/300.,71.61/
00124     13*      T2 = T**2
00125     14*      T3 = T2*T
00126     15*      T4 = T2*T2
00127     16*      T5 = T2*T3
00130     17*      T02 = T00**2
00131     18*      T03 = T02*T00
00132     19*      T04 = T02*T02
00133     20*      T05 = T03*T02
00134     21*      TMO = T-T00
00135     22*      TOT = T/TTO
00136     23*      TMT = T-TTO
00137     24*      TLOG = LOG(TOT)
00140     25*      IF (T .GT. 600.) GO TO 5
00142     26*      ENTAIR = ENT00+A(1)*TMO+(A(2)/2.)*(T2-T02)+(A(3)/3.)*
00142     27*      1 (T3-T03)+(A(4)/4.)*(T4-T04)+(A(5)/5.)*(T5-T05)
00143     28*      RETURN

```

\*\*\*\*\* ENTAIR \*\*\*\*\*

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```
00144 29* 5 ENTN = 9.47*TMT-3470.*TLOG-1.16E+06*(1./T-1./TT0)
00145 30* ENTO = 11.515*TMT-344.*(SQRT(T)-SQRT(TT0))+1530.*TLOG
00146 31* IF (T .LT. 5000.) GO TO 10
00150 32* ENTO = ENTO+5.E-05*((T2-TT0**2)/2.-4000.*TMT)
00151 33* 10 ENTAIR = ENTO+FM0*ENTO/WM0+FMN*ENTN/WMN
00152 34* RETURN
00153 35* END
```

END OF COMPILATION:

NO DIAGNOSTICS.

ENDG.P \*\*\*\*\* EXITAV \*\*\*\*\*

\*\*\*\*\* EXITAV \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASAS.EXITAV,ME\*NASAS.EXITAV  
FOR S9A-07/13/72-20:56:23 (0,)

SUBROUTINE EXITAV ENTRY POINT 000076

STORAGE USED: CODE(1) 000110; DATA(0) 000030; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 QRD 003721  
0004 QIN 001610  
0005 ABSRST 000601

EXTERNAL REFERENCES (BLOCK, NAME)

0006 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000012	1116	0001	000021	1146	0005 R	000000	ALPHFN	0005 R	000226	ALPHMP	0003	003465	CSF
0003	003464	DXX2	0003	003472	DXX21	0003	003463	DZMFN	0003 R	003410	EBFN	0003 R	003441	EBMP
0005 R	000461	EMITFN	0005 R	000512	EMITMP	0003 R	003467	EXTIFN	0003 R	003471	EXTIMP	0003 R	003466	EXTSFN
0003 R	003470	EXTSMP	0005 R	000517	EXTVCT	0000 I	000002	I	0000	000010	INJP\$	0000 I	000003	J
0000 I	000004	K	0003	003461	LCT	0003	003462	LTT	0003	003460	MCVRD	0004	001605	NSRD
0004	001604	NTM	0004	000454	QIFN	0004	001274	QITB	0004	000144	QSFN	0004	000764	QSTB
0004	001606	QTO	0003	003473	SS	0003	003670	SSTT	0004	000000	TM	0003	000000	TRMTX
0004	001607	TX	0003	003446	XX2	0000 R	000000	ZFN	0000 R	000001	ZMP	0003	003453	ZZZ

```

00101      1*      SUBROUTINE EXITAV
00101      2*      C
00101      3*      C THIS SUBROUTINE COMPUTES :
00101      4*      C THE EXITATION VECTOR NECESSARY FOR THE CALCULATION OF THE INCIDENT
00101      5*      C NET RADIANT FLUX
00101      6*      C
00103      7*      COMMON /QRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZZ(5),
00103      8*      1      MCVRD,LCT,LTT,DZMFN,
00103      9*      2      DXX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DXX21,SS(5,5,5)
00103     10*      3      ,SSTT(5,5)
00104     11*      COMMON /QIN/TM(100), QSFN(100,2), QIFN(100,2), QSTB(100,2),
00104     12*      1      QITB(100,2), NTM, NSRD, QTO, TX
00105     13*      COMMON /ABSRST/ALPHFN(5,5,6),ALPHMP(5,31),EMITFN(5,5),EMITMP(5),
00105     14*      1      EXTVCT(50)
00106     15*      ZFN   = EXTSFN+EXTIFN
00107     16*      ZMP   = EXTSMP+EXTIMP
00110     17*      DO 16 I=1,5
00113     18*      DO 15 J=1,5
00116     19*      K     =(I-1)*5 +J
00117     20*      15 EXTVCT(K) = EMITFN(J,I)*EBFN(J,I)+(1.0-ALPHFN(I,J,6))*ZFN
00121     21*      K     =I+25
00122     22*      16 EXTVCT(K) = EMITMP(I)*EBMP(I)+(1.0-ALPHMP(I,31))*ZMP

```

\*\*\*\*\* EXITAV \*\*\*\*\*

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00124 23\* RETURN  
00125 24\* END

END OF COMPILATION: NO DIAGNOSTICS.

QH06P \*\*\*\*\* FINT \*\*\*\*\*

\*\*\*\*\* FINT \*\*\*\*\*

DATE 071372

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1

QFOR,S ME\*NASAS.FINT,ME\*NASAS.FINT  
FOR 59A-07/13/72-20:56:25 (0,)

SUBROUTINE FINT ENTRY POINT 000075

STORAGE USED: CODE(1) 000111; DATA(0) 000027; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000043 1116 0000 R 000000 DDX 0000 000010 INJPS 0000 I 000001 J

```
00101 1* SUBROUTINE FINT(Y,Y0,DX,N,F)
00101 2* C
00101 3* C THIS SUBROUTINE COMPUTES :
00101 4* C THE INDEFINITE INTEGRAL BY SIMPSON'S RULE OF THE EQUALLY-SPACED
00101 5* C ARRAY Y(N)
00101 6* C
00103 7* DIMENSION Y(N),F(N)
00104 8* F(1) = Y0
00105 9* DDX = DX/24.
00106 10* F(2) = F(1)+DDX*(9.*Y(1)+19.*Y(2)-5.*Y(3)+Y(4))
00107 11* F(3) = F(2)+DDX*(-Y(1)+13.*Y(2)+Y(3))-Y(4))
00110 12* DO 1 J=4,N
00113 13* 1 F(J) = F(J-1)+DDX*(9.*Y(J)+19.*Y(J-1)-5.*Y(J-2)+Y(J-3))
00115 14* RETURN
00116 15* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* FLSTRT \*\*\*\*\*

\*\*\*\*\* FLSTRT \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME NASAS FLSTRT ME NASAS FLSTRT  
FOR S9A-07/13/72-20:56:28 (0.)

SUBROUTINE FLSTRT ENTRY POINT 000231

STORAGE USED: CODE(1) 000245; DATA(0) 000233; BLANK COMMON(2) 001115

COMMON BLOCKS:

0003 DVCNFL 000002  
0004 FLCNP 000024

EXTERNAL REFERENCES (BLOCK, NAME)

0005 DERIVL  
0006 CNTLN  
0007 RHOF  
0010 CPF  
0011 TRNSPT  
0012 DDX  
0013 RKSF  
0014 NERR2S  
0015 NWOUS  
0016 NIO2S  
0017 NERR3S

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000046 10L	0000 000051 115F	0000 000127 120F	0001 000076 15L	0001 000120 20L
0001 000143 25L	0001 000010 5L	0000 R 000006 AL	0002 000562 AUX2	0006 R 000000 CNTLN
0002 000752 COB	0002 001107 COH	0002 000574 CONFN	0002 000740 CONMP	0010 R 000000 CPF
0002 R 001070 CPD	0002 001072 DELTA	0000 R 000014 OELY	0005 R 000000 DERIVL	0002 000764 DOB
0004 R 000000 DTHETA	0002 001110 DXI	0002 001053 DXIMP	0002 001052 DXITB	0000 R 000003 DY
0000 R 000030 DYST	0002 R 001051 OZ	0002 001103 FENTH	0002 001045 FFO	0002 001101 FLUX
0003 000001 FLUXI	0002 R 001041 FMACH	0002 R 001040 FNU	0002 001046 FOF	0002 000012 FP
0002 R 001042 FPR	0002 R 001037 FR	0002 001050 FRAD	0002 001111 FRO	0002 001036 FRE
0002 001104 FREJ	0002 001043 FRL	0002 001044 FRM	0002 R 000036 FT	0000 R 000047 FT1
0002 000024 FW	0002 001113 FXHW	0002 001112 FXOH	0002 R 001047 FZ	0002 001105 HFN
0000 I 000043 IBKP	0000 I 000045 IERR	0002 I 001003 IFLOW	0000 000223 INJPS	0000 I 000042 ISTP
0002 001024 LC	0002 001032 LIM	0002 001031 LIMWRT	0002 001011 LL1	0002 001012 LL2
0002 001013 LL3	0002 001014 LL4	0002 001015 LL5	0002 001016 LL6	0002 001017 LL7
0002 001006 LMP	0002 001004 LT	0002 001005 LTB	0002 001007 LTBMZ	0002 001010 LTB2MZ
0002 001030 MM1	0002 001033 MOD	0002 001025 MSTOTR	0002 I 000777 MZ	0002 001034 NCCZ
0002 001035 NCONV	0002 I 001002 NCTL	0002 001026 NCTM	0000 I 000046 NEQ	0002 001023 NEQUS
0002 001027 NM1	0002 001001 NRMP	0002 001020 NRMP1	0002 001022 NRMP2	0002 001000 NRTB
0002 001021 NRTB1	0000 I 000044 NTRY	0002 000776 NX	0000 R 000017 PD	0002 001063 PHIF
0002 001062 PHIM	0002 001071 PI	0002 R 001065 P0	0002 001102 QREF	0002 000360 QRFN
0002 000524 QRMF	0002 001074 RDTWRT	0007 R 000000 RHOF	0002 001114 RHOFN	0002 R 001067 RH00
0000 R 000011 RL	0002 001073 RLIMIT	0002 001100 RTEND	0000 R 000022 SO	0002 000132 TEMP
0002 R 001077 TI	0002 001076 TINTL	0002 000276 TYP	0002 001075 TREF	0002 000050 TW
0002 R 001064 TO	0003 R 000000 WRAT	0002 R 001066 W0	0002 000536 XIFN	0002 000005 XIMP
0002 000000 XITB	0002 001106 XL	0000 R 000050 XP	0002 001054 XRE	0002 001057 XX10
0002 001060 XX11	0002 001061 XX12	0002 001055 XX3	0002 001056 XX4	0000 R 000000 Y



\*\*\*\*\* FLSTRT \*\*\*\*\*

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0000 R 000025 YS

0000 R 000036 YSIMP

0000 R 000033 YST

0000 R 000041 Z

0002 000550 ZETA

```

00101      1*      SUBROUTINE FLSTRT(RE,PRL,DLTA)
00101      2*      C
00101      3*      C THIS SUBROUTINE PROVIDES :
00101      4*      C 1- FLUID FLOW (INITIAL AND CURRENT) CONDITIONS
00101      5*      C 2- TABLE HEADING FOR INITIAL CONJITIONS
00101      6*      C
00103      7*      COMMON XITB(5),XIMP(5),FP(10),FW(10),FT(10),TW(10,5),TEMP(10,10),
00103      8*      1 TMP(10,5),QRFN(10,10),GRMP(10),XIFN(10),ZETA(10),AUX2(10),
00103      9*      2 CONFN(10,10),CONMP(10),COB(10),DOB(10)
00104     10*      COMMON NX,MZ,NRTB,NRMP,NCTL,IFLOW,LT,LTB,LMP,LTBMZ,LTB2MZ,LL1,
00104     11*      1 LL2,LL3,LL4,LL5,LL6,LL7,NRMP1,NRTB1,NRMP2,NEQUS,LC,MSTOTR,
00104     12*      2 NCTM,NM1,M41,LIMWRT,LIM,MOD,NCCZ,NCONV
00105     13*      COMMON FRE,FR,FNU,FMACH,FPR,FRL,FRM,FFO,FOF,FZ,FRAD,DZ,DXITB,DXIMP
00105     14*      1 ,XRE,XX3,XX4,XX10,XX11,XX12,PHIM,PHIF,T0,P0,W0,RH00,CP0,PI
00105     15*      2 ,DLTA,RLIMIT,ROTWRT,TREF,TINTL,TI,RTEND,FLUX,QREF,FENTH,
00105     16*      3 FREJ,MFN,XL,COH,DXI,FRD,FXOH,FXHW,RHOFN
00106     17*      COMMON /DVC4FL/ WRAT,FLUXI
00107     18*      COMMON /FLC4P/ DTHETA(20)
00107     19*      C
00110     20*      DIMENSION Y(3),DY(3),AL(3),RL(3),DELY(3),PD(3),SD(3),YS(3),
00110     21*      1 DYST(3),YST(3),YSIMP(3)
00111     22*      C EXTERNAL DERIVL,CNTLN
00111     23*      C
00112     24*      GO TO (5,10),IFLOW
00113     25*      5 RH00 = RHOF(P0,T0)
00114     26*      CP0 = CPF(RH00,T0)
00115     27*      FPR = P0/(RH00*W0**2)
00116     28*      FZ = P0/(RH00*T0*CP0*778.16)
00117     29*      FMACH = W0**2/(T0*CP0*778.16)
00120     30*      10 CALL TRNSPT (RE,PRL,DLTA,FR,FNU)
00120     31*      C
00120     32*      C INITIALIZATION
00121     33*      Z = 0.0
00122     34*      ISTD = 1
00123     35*      IBKP = 1
00124     36*      NTRY = 1
00125     37*      IERR = 0
00126     38*      NCTL = 1
00127     39*      Y(1) = 1.0
00130     40*      GO TO (15,20),IFLOW
00131     41*      15 Y(2) = RH00/RHOF(P0,TI)*WRAT
00132     42*      Y(3) = FT(1)
00133     43*      NEQ = 3
00134     44*      WRITE (6,115)
00136     45*      GO TO 25
00137     46*      20 FT1 = FT(1)*T0
00140     47*      Y(2) = RH00/RHOF(P0,FT1)*WRAT
00141     48*      NEQ = 2
00142     49*      CALL DDX(FT,DTHETA,DZ,MZ)
00143     50*      25 CALL RKSF (DERIVL,CNTLN,Y,DY,AL,RL,Z,DZ,NEQ,ISTD,IBKP,NTRY,
00143     51*      1 IERR,DELY,PD,SD,YS,YST,DYST,YSIMP)
00144     52*      IF(IFLOW.EQ.2) RETURN

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\*\*\*\*\* FLSTRT \*\*\*\*\*

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00146	53*	XP = FPR/2.0	032
00147	54*	WRITE (6,120) P0,W0,T0,RE,PRL,DLTA,XP	
00160	55*	IFLOW = 2	
00161	56*	RETURN	041
00162	57*	115 FORMAT(11H,15X,23HINITIAL LINE CONDITIONS,/,16X,23H***** **** **	25
00162	58*	1*****/,25X,31H(ALL QUANTITIES ARE NORMALIZED),/,83X,5HFLUID,15	026
00162	59*	2X,4HWALL,/,15X,16HPT.NO. POSITION,11X,8HPRESSURE,12X,8HVELOCITY,	027
00162	60*	311X,11HTEMPERATURE,9X,11HTEMPERATURE,/,26X,1HZ,18X,1HP,19X,1HW,19X	028
00162	61*	4,1HT,18X,3HTWI,/,)	029
00163	62*	120 FORMAT(1H0,5X,25HINLET PRESSURE P0 = ,F10.3,11H LBF/SQ.FT,/,	034
00163	63*	1 6X,25HREF. VELOCITY W0 = ,F10.5,11H FT/SEC ,/,	035
00163	64*	2 6X,25HREF. TEMPERATURE T00 = ,F10.3,11H R ,/,	036
00163	65*	3 6X,25H REYNOLDS NO = ,E20.5,/,	
00163	66*	4 6X,25H PRANDTL NO = ,F20.6,/,	038
00163	67*	5 6X,25H DELTA = ,F20.6,/,	
00163	68*	6 6X,25H REL.PRESSURE IS ,E20.6,/,)	039
00163	69*	C	040
00164	70*	END	042

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* FMINV \*\*\*\*\*

\*\*\*\*\* FMINV \*\*\*\*\*

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QFOR S ME NASAS FMINV ME NASAS FMINV  
FOR S9A-07/13/72-20:56:31 (0.)

SUBROUTINE FMINV ENTRY POINT 000254

STORAGE USED: CODE(1) 000306; DATA(0) 001260; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000046	1056	0001	000151	111L	0001	000051	1116	0001	000071	1176	0001	000104	1236
0001	000125	1336	0001	000130	1376	0001	000172	1506	0001	000175	1546	0001	000227	1646
0001	000212	20L	0000	R	001214	AA	0000	R	001217	B	0000	I	001212	I
0000	I	001215	I1	0000	I	001220	I2	0000	I	001213	J	0000	I	001216
												0000	R	000000
														XMAT

00101	1*	SUBROUTINE FMINV (A,X,N,M)	1
00103	2*	DIMENSION A(N,N),X(N),XMAT(25,26)	2
00104	3*	DO 1 I=1,N	3
00107	4*	XMAT(I,M) = X(I)	4
00110	5*	DO 1 J=1,M	5
00113	6*	1 XMAT(I,J) = A(I,J)	6
00116	7*	DO 20 I=1,N	7
00121	8*	AA = XMAT(I,I)	8
00122	9*	DO 5 J=1,M	9
00125	10*	5 XMAT(I,J) = XMAT(I,J)/AA	10
00127	11*	IF (I.EQ.1) GO TO 11	11
00131	12*	I1 = I-1	12
00132	13*	DO 10 K=1,I1	13
00135	14*	B = XMAT(K,I)	14
00136	15*	DO 10 J=1,M	15
00141	16*	10 XMAT(K,J) = XMAT(K,J) - XMAT(I,J) * B	16
00144	17*	IF (I.EQ.N) GO TO 20	17
00146	18*	11 I2 = I+1	18
00147	19*	DO 15 K=I2,N	19
00152	20*	B = XMAT(K,I)	20
00153	21*	DO 15 J=1,M	21
00156	22*	15 XMAT(K,J) = XMAT(K,J) - XMAT(I,J) * B	22
00161	23*	20 CONTINUE	23
00163	24*	DO 25 I=1,N	24
00166	25*	25 X(I) = XMAT(I,M)	25
00170	26*	RETURN	26
00171	27*	END	27

END OF COMPILATION: NO DIAGNOSTICS.

\*\*\*\*\* FMINV \*\*\*\*\*

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QHDG.P \*\*\*\*\* HFL/CFFC43 \*\*\*\*\*

\*\*\*\*\* HFL/CFFC43 \*\*\*\*\*

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1

QFOR,S ME\*NASA5.HFL/CFFC43,ME\*NASA5.HFL/CFFC43  
FOR S9A-07/13/72-20:56:34 (0.)

FUNCTION HFL

ENTRY POINT 000022

STORAGE USED: CODE(1) 000024; DATA(0) 000014; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 HFL      0000    000007 INJP5      0000 R 000004 THETA      0000 R 000003 TO      0000 R 000001 X1  
0000 R 000002 X2

00101	1*		FUNCTION HFL(RHO,T)
00101	2*	C	ENTHALPY AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND
00101	3*	C	TEMPERATURE (R) OF FC-43
00101	4*	C	UNITS BTU/SLUG
00103	5*		DATA X1,X2 /-0.020092,5.4054E-04/,T0 /401.67/
00107	6*		THETA = T-T0
00110	7*		HFL = (X1+X2*(T+T0)/2.0)*THETA*32.174
00111	8*		RETURN
00112	9*		END

END OF COMPILATION:      NO    DIAGNOSTICS.

QHDG,P \*\*\*\*\* HFL/CFFC75 \*\*\*\*\*

\*\*\*\*\* HFL/CFFC75 \*\*\*\*\*

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QFOR: S ME\*NASA5.HFL/CFFC75, ME\*NASA5.HFL/CFFC75  
FOR S9A-07/13/72-20:56:36 (0:)

FUNCTION HFL ENTRY POINT 000022

STORAGE USED: CODE(1) 000024; DATA(0) 000014; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000000 HFL      0000 000007 INJPS      0000 R 000004 THETA      0000 R 000003 TO      0000 R 000001 X1  
0000 R 000002 X2

00101 1\* FUNCTION HFL(RHO,T)  
00101 2\* C  
00101 3\* C THIS FUNCTION SUBPROGRAM COMPUTES :  
00101 4\* C ENTHALPY AS A FUNCTION OF DENSITY (SLUG/CU,FT) AND  
00101 5\* C TEMPERATURE (R) OF FC-75  
00101 6\* C UNITS BTU/SLUG  
00101 7\* C  
00103 8\* DATA X1,X2 /0.115082,2.4333E-04/,T0 /324.67/  
00107 9\* THETA = T-T0  
00110 10\* HFL = (X1+X2\*(T+T0)/2.0)\*THETA\*32.174  
00111 11\* RETURN  
00112 12\* END

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* HFL/CFHE \*\*\*\*\*

\*\*\*\*\* HFL/CFHE \*\*\*\*\*

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QFOR: S ME\*NASAS.HFL/CFHE, ME\*NASAS.HFL/CFHE  
FOR S9A-07/13/72-20:56:37 (0.)

FUNCTION HFL ENTRY POINT 000110

STORAGE USED: CODE(1) 000124; DATA(0) 000047; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 PF  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000005 A	0000 R 000002 A1	0000 R 000003 B1	0000 R 000004 C	0000 R 000016 DRH0
0000 R 000017 DRH02	0000 R 000013 DT	0000 R 000015 OTM2	0000 R 000000 HFL	0000 000034 INJPS
0000 R 000022 P	0003 R 000000 PF	0000 R 000001 R	0000 R 000011 RHOX	0000 R 000010 RH00
0000 R 000014 TV2	0000 R 000012 TX	0000 R 000007 T0	0000 R 000021 UFL	0000 R 000006 U0
0000 R 000020 Y				

```
00101 1* FUNCTION HFL(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C ENTHALPY AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND
00101 5* C TEMPERATURE (R) OF HELIUM
00101 6* C UNITS BTU/SLUG
00101 7* C
00103 8* DATA R,A1,B1,C,A /2077.02,136.9595,3.5002295E-03,10.000658,1.49610
00103 9* 13E-02/,U0,T0,RH00 /39922.0,10.938889,4.669193/
00114 10* RHOX = RH0*515.4275
00115 11* TX = T/1.8
00116 12* DT = TX-T0
00117 13* TM2 = 1.0/(TX*TX)
00120 14* DTM2 = 1.0/(T0*T0)-TM2
00121 15* ORH0 = RH00-RHOX
00122 16* DRH02= RHOX*RHOX-RH00*RH00
00123 17* Y = 3.0*R*C*TM2
00124 18* UFL = U0+R*(1.5*DT+141.22973*DTM2)+(Y*A1)*ORH0-(Y*B1-A1*A)*DRH02/
00124 19* 12.0
00125 20* P = PF(RHO,T)
00126 21* HFL = UFL*0.013844+P/(RHO*778.26)
00127 22* RETURN
00130 23* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* HFL/CFNAK, \*\*\*\*\*

\*\*\*\*\* HFL/CFNAK \*\*\*\*\*

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QFOR:5 ME\*NASAS.HFL/CFNAK,ME\*NASAS.HFL/CFNAK  
FOR S9A-07/13/72-20:56:40 (0.)

FUNCTION HFL ENTRY POINT 000100

STORAGE USED: CODE(1) 000112; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 CAPP  
0004 PF  
0005 ALOG  
0006 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000017 C	0003 R 000000 CAPP	0000 R 000010 C1	0000 R 000011 C2	0000 R 000007 DT
0000 R 000000 HFL	0000 R 000014 H1	0000 R 000020 H2	0000 000030 INJP5	0004 R 000000 PF
0000 R 000012 P0	0000 R 000016 P1	0000 R 000013 THETA	0000 R 000015 TK	0000 R 000006 T0
0000 R 000001 X1	0000 R 000002 X2	0000 R 000003 X3	0000 R 000004 X4	0000 R 000005 X5

```

00101      1*      FUNCTION HFL(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      ENTHALPY AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND
00101      5*      C      TEMPERATURE (R) OF NAK 78.6
00101      6*      C      UNITS BTU/SLUG
00101      7*      C
00103      8*      DATA X1,X2,X3,X4,X5 /0.2255,-0.016292,0.005396,-0.000758,0.000054/
00103      9*      1,T0,DT /469.67,300.0/,C1,C2 /58.773064,-0.008433/,P0 /2116.224/
00116     10*      THETA = (T-T0)/DT
00117     11*      H1 = (((X5*THETA/5.0+X4/4.0)*THETA+X3/3.0)*THETA+X2/2.0)*THETA
00117     12*      1+X1)*(T-T0)
00120     13*      TK = CAPP(RHO,T)
00121     14*      P1 = PF(RHO,T)-P0
00122     15*      C = C1+C2*T
00123     16*      H2 = ((1.0+T*C2/C)*ALOG(1.0+TK*P1))/(C*TK*778.26)
00124     17*      HFL = (H1+H2)*32.174
00125     18*      RETURN
00126     19*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* HFL/CFSIL \*\*\*\*\*



\*\*\*\*\* HFL/CFSIL \*\*\*\*\*

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QFOR S ME\*NASA5.HFL/CFSIL ME\*NASA5.HFL/CFSIL  
FOR 59A-07/13/72-20:56:42 (0.)

FUNCTION HFL ENTRY POINT 000300

STORAGE USED: CODE(1) 000326; DATA(0) 000132; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 PF  
0004 POLY  
0005 NWDJ5  
0006 NIO25  
0007 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000051	IF	0000	R	000040	A	0000	R	000007	A1	0000	R	000041	A11	0000	R	000010	A2	
0000	R	000011	A3	0000	R	000012	A4	0000	R	000013	A5	0000	R	000042	B	0000	R	000014	B1
0000	R	000043	B11	0000	R	000015	B2	0000	R	000016	B3	0000	R	000017	B4	0000	R	000020	B5
0000	R	000044	C	0000	R	000021	C1	0000	R	000045	C11	0000	R	000022	C2	0000	R	000025	DT
0000	R	000033	DT2	0000	R	000000	HFL	0000	R	000037	H1	0000	R	000050	H2	0000	R	000114	INJP5
0000	R	000046	P	0003	R	000000	PF	0004	R	000000	POLY	0000	R	000047	P1	0000	R	000034	THETA
0000	R	000035	THETA1	0000	R	000036	THETA2	0000	R	000023	T0	0000	R	000024	T01	0000	R	000032	T02
0000	R	000026	X1	0000	R	000027	X2	0000	R	000030	X3	0000	R	000031	X4	0000	R	000001	Z

```

00101 1*      FUNCTION HFL(RHO,T)
00101 2*      C
00101 3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4*      C      ENTHALPY AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND
00101 5*      C      TEMPERATURE (R) OF DOW CORNING 200 SILICON OIL (1 C5)
00101 6*      C      UNITS BTU/SLUG
00101 7*      C      TEMPERATURE .GE. 359.67 AND .LE. 859.67
00101 8*      C
00103 9*      DIMENSION Z(6)
00104 10*     DATA A1,A2,A3,A4,A5 /12.35,2.98333,1.1,-0.48333,0.1/,B1,B2,B3,B4,B
00104 11*     15 /-1.5,-0.01333,-1.18,0.57333,-0.1/,C1,C2 /0.7767,-0.0288/,T0,T01
00104 12*     2,DT /559.67,609.67,50.0/,X1,X2,X3,X4 /0.46,0.00471,0.00141,0.00004
00104 13*     33/,T02,DT2 /539.67,80.0/
00132 14*     THETA = (T-T0)/DT
00133 15*     THETA1 = (T-T01)/DT
00134 16*     THETA2 = (T-T02)/DT2
00135 17*     H1 = (((X4*THETA2/4.0+X3/3.0)*THETA2+X2/2.0)*THETA2+X1)*(T-T02)
00136 18*     A = (((A5*THETA+A4)*THETA+A3)*THETA+A2)*THETA+A1)*1.0E-06
00137 19*     A11 = (((4.0*A5*THETA+3.0*A4)*THETA+2.0*A3)*THETA+A2)*(1.0E-06)/DT
00140 20*     B = (((B5*THETA+B4)*THETA+B3)*THETA+B2)*THETA+B1)*1.0E-09
00141 21*     B11 = (((4.0*B5*THETA+3.0*B4)*THETA+2.0*B3)*THETA+B2)*(1.0E-09)/DT
00142 22*     C = C1+C2*THETA1
00143 23*     C11 = C2/DT
00144 24*     P = PF(RHO,T)

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\*\*\*\*\* HFL/CFSIL \*\*\*\*\*

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00145 25* P1 = P/144.0-14.696
00146 26* Z(1)= 0.0
00147 27* Z(2)= 1+T*C11/C
00150 28* Z(3)= (A11*T-A*Z(2))/2.0
00151 29* Z(4)= (T*(B11-A*A11)-B*Z(2))/6.0
00152 30* Z(5)= -(A*B11+A11*B)*T/8.0
00153 31* Z(6)= -B*B11*T/20.0
00154 32* H2 = POLY(6,Z,P1)
00155 33* HFL = (H1+H2/(C*337.37))*32.174
00156 34* IF (T.LT.360.67.OR.T.GT.860.67.OR.P.GT.146116.224.OR.P.GT.110116.2
00156 35* 124.AND.T.LT.460.67) WRITE(6,1) T,P
00163 36* 1 FORMAT (1H0,43HENTHALPY OF SILICON OIL, OUT OT RANGE, T = ,F10.5,6
00163 37* 1H, P = ,F15.5,/)
00164 38* RETURN
00165 39* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QH06.P \*\*\*\*\* INTERP \*\*\*\*\*

\*\*\*\*\* INTERP \*\*\*\*\*

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QFOR:5 ME\*NASA5.INTERP,ME\*NASA5.INTERP  
FOR S9A-07/13/72-20:56:45 (0.)

SUBROUTINE INTERP ENTRY POINT 000171

STORAGE USED: CODE(1) 000226; DATA(0) 000063; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 YINT  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000037	1053	0001	000054	1106	0001	000062	1156	0001	000106	1246	0001	000122	1276			
0001	000130	1343	0000	I	000012	I	0000	000022	INJPS	0000	I	000014	J	0000	I	000013	K
0000	R	000015	P	0003	R	000000	YINT	0000	R	000000	YY						

```
00101 1* SUBROUTINE INTERP(NX1,MZ1,XX1,ZZ1,YY1,NX2,MZ2,XX2,ZZ2,YY2)
00101 2* C
00101 3* C THIS SUBROUTINE MAPS A TWO-DIMENSIONAL FUNCTION YY1 FROM ONE GRID
00101 4* C (XX1,ZZ1) ONTO ANOTHER TWO-DIMENSIONAL FUNCTION YY2 WITH ANOTHER GRID
00101 5* C (XX2,ZZ2)
00101 6* C NX1,MZ1 NUMBER OF NODAL POINTS OF THE ORIGINAL GRID (EACH LESS
00101 7* C THAN OR EQUAL TO 10)
00101 8* C NX2,MZ2 NUMBER OF NODAL POINTS OF THE NEW GRID (EACH LESS THAN
00101 9* C OR EQUAL TO 10)
00101 10* C
00103 11* DIMENSION XX1(NX1), ZZ1(MZ1), YY1(10,10), YY(10), XX2(NX2),
00103 12* 1 ZZ2(MZ2), YY2(10,10)
00104 13* DO 2 I = 1,NX1
00107 14* DO 1 K = 1,MZ1
00112 15* 1 YY(K) = YY1(K,I)
00114 16* DO 2 J = 1,MZ2
00117 17* P = ZZ2(J)
00120 18* 2 YY2(J,I) = YINT(ZZ1,YY,MZ1,3,P)
00123 19* DO 4 J = 1,MZ2
00126 20* DO 3 K = 1,NX1
00131 21* 3 YY(K) = YY2(J,K)
00133 22* DO 4 I = 1,NX2
00136 23* P = XX2(I)
00137 24* 4 YY2(J,I) = YINT(XX1,YY,NX1,3,P)
00142 25* RETURN
00143 26* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* MAIN \*\*\*\*\*

\*\*\*\*\* MAIN \*\*\*\*\*

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QFOR,S ME\*NASA5.MAIN,ME\*NASA5.MAIN  
FOR S9A-07/13/72-20:56:47 (0.)

MAIN PROGRAM

STORAGE USED: CODE(1) 003774; DATA(0) 006145; BLANK COMMON(2) 001115

COMMON BLOCKS:

0003	ADBH	000352
0004	DVM	000002
0005	VELALT	001132
0006	CNV	000010
0007	QRD	003721
0010	QIN	001610
0011	TC	000042
0012	ABSRST	000601
0013	SSF	000010
0014	GEOM	000020
0015	FLDINL	000457
0016	DVCMFL	000002

EXTERNAL REFERENCES (BLOCK, NAME)

0017	DERIVM
0020	CNTLM
0021	TCALC
0022	CLOSE
0023	ADIABH
0024	QINCID
0025	TK
0026	RHOF
0027	VISC
0030	CPF
0031	THCF
0032	HFL
0033	FLSTRT
0034	THCTB
0035	THCMP
0036	DEFINT
0037	SHAPEF
0040	RKS
0041	NINTRS
0042	NRNLS
0043	NWDUS
0044	NI01S
0045	NI02S
0046	NRDUS
0047	NRBUS
0050	NWBUS
0051	NWNLS
0052	SQRT
0053	ASIN
0054	NSTOPS

## STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	005361	1055F	0001	002064	1076G	0001	001320	1100L	0001	002075	1104G	0001	002104	1112G
0001	002113	1120G	0001	002203	1140G	0001	002245	1151G	0000	005372	1155F	0001	002275	1163G
0001	000732	12L	0001	002730	1246G	0001	002731	1251G	0000	005403	1255F	0001	002735	1256G
0001	002742	1264G	0001	003113	1324G	0000	005406	1355F	0001	003360	1410G	0001	003363	1413G
0001	003431	1424G	0001	003434	1427G	0001	003477	1440G	0001	003506	1443G	0001	003530	1452G
0000	005425	1455F	0001	003537	1461G	0001	003662	1500G	0000	005167	155F	0000	005436	1555F
0000	005454	16F	0000	005447	2F	0001	000136	200L	0001	000032	2000L	0001	001360	2100L
0000	005455	22F	0001	000120	227G	0001	000160	237G	0001	001110	24L	0001	000161	242G
0001	000221	250G	0000	005207	255F	0001	000235	263G	0001	001167	28L	0000	005450	3F
0001	001217	30L	0001	000141	300L	0001	002761	312L	0000	005457	313F	0001	003107	314L
0001	000336	310G	0000	005476	320F	0000	005722	321F	0001	000364	325G	0001	000410	337G
0001	001472	34L	0001	003461	341L	0001	000426	346G	0000	005243	355F	0001	000444	355G
0001	000456	360G	0001	003656	364L	0001	000474	372G	0000	005452	4F	0001	000506	400G
0001	000524	407G	0001	000552	416G	0001	000564	424G	0001	000576	432G	0001	000610	440G
0000	005257	450F	0001	000644	455G	0001	000656	463G	0001	000670	472G	0001	000702	500G
0001	000171	500L	0001	000714	506G	0001	000726	514G	0001	000764	535G	0001	001012	543G
0001	001022	550G	0001	001030	554G	0000	005274	555F	0001	001036	560G	0001	001053	571G
0001	001070	600G	0000	005310	655F	0001	001211	665G	0001	001247	676G	0001	001255	703G
0001	001315	710G	0001	001336	722G	0001	001345	727G	0001	001416	740G	0000	005320	755F
0001	000514	8L	0001	000240	800L	0000	005332	855F	0000	005344	955F	0000	006017	9998F
0001	003770	9909L	0000	R 000621	A	0011	R 000003	ABEMF	0000	R 004355	AITB	0003	R 000001	AL
0000	R 004270	ALIMIT	0013	000002	ALPHN	0000	R 004256	ALPHA	0012	000000	ALPHFN	0012	000226	ALPHMP
0005	R 000310	ALTA	0005	R 000454	ALTR	0000	R 004265	AMAN	0000	R 004264	AN	0000	R 004300	ANGINC
0000	R 004263	AT	0000	R 004371	ATOT	0002	000562	AUX2	0000	R 004361	BIOTNO	0000	R 004350	BOL
0000	R 004257	BT	0020	R 000000	CNTLM	0002	R 000752	COB	0006	R 000007	COEFF	0002	R 001107	COH
0003	000161	COFL	0002	R 000574	CONFN	0003	000173	CONFNA	0002	R 000740	CONMP	0003	000147	CP
0030	R 000000	CPI	0002	R 001070	CPD	0007	003465	CSF	0013	R 000002	CTA	0003	R 000000	D
0036	R 000000	DEFINT	0002	R 001072	DELTA	0000	R 001441	DELY	0017	R 000000	DERIVM	0000	R 004325	DITB
0000	R 004241	DITBI	0002	R 000764	DOB	0000	R 004347	DTA	0003	000326	DTL	0000	R 004401	DTME
0000	R 004266	DTIRTE	0006	R 000003	DX	0002	R 001110	DXI	0000	R 004344	DXIFN	0002	R 001053	DXIMP
0002	R 001052	DXITB	0007	R 003464	DXX2	0007	R 003472	DXX21	0000	R 000311	DY	0000	R 003101	DYST
0002	R 001051	DZ	0007	R 003463	DZMFN	0007	003410	EBFN	0007	003441	EBMP	0003	R 000244	EFF
0003	000205	EMIS	0012	000461	EMITFN	0012	000512	EMITMP	0007	003467	EXTIFN	0007	003471	EXTIMP
0007	003466	EXTSPN	0007	003470	EXTSMP	0012	000517	EXTVCT	0002	R 001103	FENTH	0002	R 001045	FFO
0000	004500	FIN	0000	004444	FINA	0015	000456	FLDINT	0000	R 004364	FLMASI	0014	R 000016	FLMASS
0000	004565	FLOW	0000	R 004316	FLU	0000	R 004317	FLUID	0000	004457	FLUIDA	0002	R 001101	FLUX
0016	R 000001	FLUXI	0002	001041	FMACH	0000	R 004314	FM	0000	R 004365	FNMASI	0000	R 004362	FNMASS
0000	R 004315	FNMTL	0002	R 001040	FNU	0002	R 001046	FOF	0002	R 000012	FP	0002	001042	FPR
0002	001037	FR	0002	R 001050	FRAD	0002	R 001111	FRD	0002	R 001036	FRE	0002	R 001104	FREJ
0002	R 001043	FRL	0002	R 001044	FRM	0002	R 000036	FT	0002	R 000024	FW	0002	R 001113	FXHW
0002	R 001112	FXOH	0002	001047	FZ	0000	R 004260	GAMMA	0000	004750	GINPT	0003	R 000013	H
0003	R 000231	HAB	0000	R 004231	HOG	0032	R 000000	HFL	0000	R 004356	HFLD	0002	R 001105	HFN
0000	R 004246	HFI	0000	I 004301	I	0000	I 004404	IBKP	0011	I 000001	ICASE	0000	I 004405	IERR
0002	I 001003	IFLOW	0000	I 004403	IFVD	0000	I 004304	IHI	0000	I 004305	IHZ	0006	I 000002	IOPTN
0000	I 004237	ISYM	0000	I 004323	IT	0011	I 000000	ITAPE	0000	I 004276	IU	0000	I 004322	IUL
0000	I 004277	IW	0000	I 004324	II	0000	I 004302	J	0000	I 004311	K	0000	I 004307	K1
0000	I 004310	K2	0002	I 001024	LC	0007	I 003461	LCT	0015	I 000001	LFLD	0002	I 001032	LIM
0002	I 001031	LIMWRT	0002	I 001011	LL1	0002	I 001012	LL2	0002	I 001013	LL3	0002	I 001014	LL4
0002	I 001015	LL5	0002	I 001016	LL6	0002	I 001017	LL7	0002	I 001006	LMP	0002	I 001004	LT
0002	I 001005	LTB	0000	I 004376	LTBJ	0002	I 001007	LTBMZ	0002	I 001010	LTBMZ	0000	I 004271	LTS
0007	I 003462	LTT	0003	R 000064	M	0000	004700	MANIFD	0007	I 003460	MCVRD	0002	R 001101	MDOT
0000	R 000000	MDOTI	0000	I 004345	MMEP	0002	I 001030	MMI	0002	I 001033	MODA	0000	004474	MRI
0000	I 004272	MRSTRT	0002	I 001025	MSTOTR	0002	I 000777	MZ	0005	I 001130	NA	0002	I 001034	NCCZ
0002	I 001035	NCONV	0002	001002	NCTL	0002	I 001026	NCTM	0002	I 001023	NEGUS	0015	I 000000	NFLDTA
0002	I 001027	NM1	0005	I 001131	NR	0002	I 001001	NRMP	0002	I 001020	NRMP1	0002	I 001022	NRMP2

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0002 I 001000 NRTB	0000 I 004244 NRTBI	0002 I 001021 NRTBI	0000 I 004306 NS	0006 I 000004 NSRAD
0010 I 001605 NSRD	0003 I 000076 NT	0014 I 000012 NTBS	0000 I 004274 NTF	0010 I 001604 NTM
0000 I 004402 NTRY	0000 I 004275 NT1	0002 I 000776 NX	0000 R 001751 PD	0000 R 004372 PFLD
0000 R 004261 PHI	0002 R 001063 PHIF	0002 R 001062 PHIM	0011 R 000004 PHIN	0000 R 004240 PHN
0002 R 001071 PI	0003 R 000052 PIN	0000 R 004367 PLMASI	0014 R 000015 PLMASS	0000 R 004354 PRNLTL
0000 R 004320 PRO	0000 R 004255 PROB	0000 R 004321 PROMTL	0000 R 004627 PROTFLR	0002 R 001065 PO
0010 R 000454 QIFN	0010 R 001274 QITB	0000 R 004515 QNML	0003 R 000314 QOUT	0002 R 001102 QREF
0002 R 000360 QRFN	0002 R 000524 QRMP	0010 R 000144 QSFN	0010 R 000764 QSTB	0010 R 001606 QTO
0000 R 001131 R	0002 R 001074 RDTWRT	0000 R 004353 REY	0026 R 000000 RHOF	0000 R 004374 RHOFLO
0002 R 001114 RHOFN	0000 R 004250 RHOFNI	0000 R 004352 RHOI	0000 R 004252 RHOMET	0000 R 004334 RHOMP
0000 R 004251 RHOMPI	0000 R 004333 RHOTB	0000 R 004243 RHOTBI	0002 R 001067 RHOO	0000 R 004331 RITB
0002 R 001073 RLIMIT	0000 R 004332 ROTB	0002 R 001100 RTEND	0000 R 004705 RUNOPT	0013 R 000003 R1
0013 R 000000 R2	0000 R 002261 SD	0014 R 000000 SFN	0000 R 004273 SIDE	0000 R 004351 SLOPE
0000 R 004336 SMP	0006 R 000005 SMP	0000 R 004327 SROOT	0000 R 004245 SROOTI	0000 R 004340 SRO2
0007 R 003473 SS	0007 R 003670 SST	0006 R 000000 STAGX	0000 R 004326 STB	0000 R 004242 STBI
0000 R 004330 STIP	0000 R 004247 STIPI	0014 R 000017 STR	0006 R 000006 ST4	0000 R 004410 SYSTEM
0013 R 000001 T	0005 R 000000 TA	0000 R 004254 TAU	0000 R 004357 TAVG	0003 R 000217 TB
0000 R 004312 TBM	0000 R 004366 TBMASI	0014 R 000014 TBMAS	0000 R 004313 TBMTL	0002 R 000132 TEMP
0000 R 004267 TEND	0000 R 004373 TFLD	0003 R 000026 TH	0031 R 000000 THCF	0035 R 000000 THCMP
0034 R 000000 THCTB	0000 R 004262 THETA	0002 R 001077 TI	0015 R 000146 TIFLD	0000 R 004400 TIME
0003 R 000040 TIN	0002 R 001076 TINTL	0025 R 000000 TK	0003 R 000270 TL	0010 R 000000 TM
0015 R 000002 TMEFLD	0002 R 000276 TMP	0000 R 004335 TNN	0014 R 000013 TNXL	0011 R 000002 TO
0000 R 004346 TOL	0000 R 004370 TOTMSI	0000 R 004363 TOTMSS	0003 R 000340 TOUT	0005 R 000144 TR
0002 R 001075 TREF	0007 R 000000 TRMTX	0003 R 000077 TSTAR	0000 R 004303 TSTARM	0000 R 004536 TUBE
0000 R 004426 TUBEA	0002 R 000050 TW	0010 R 001607 TX	0002 R 001064 TO	0005 R 000620 VELA
0000 R 004253 VELM	0005 R 000764 VELR	0006 R 000001 VERTX	0027 R 000000 VISC	0003 R 000135 VSC
0000 R 006020 W	0015 R 000312 WIFLD	0016 R 000000 WRAT	0013 R 000006 WWA	0013 R 000007 WWB
0000 R 004337 WWC	0000 R 004343 WWD	0000 R 004341 WWR1	0000 R 004342 WWR2	0002 R 001066 WO
0000 R 006020 W1	0000 R 006033 W2	0000 R 004375 XFNU	0002 R 000536 XIFN	0002 R 000005 XIMP
0002 R 000000 XITB	0002 R 001106 XL	0000 R 004360 XNUSLT	0002 R 001054 XRE	0013 R 000005 XSMP
0013 R 000004 XSPM	0002 R 001057 XX10	0002 R 001060 XX11	0002 R 001061 XX12	0000 R 004377 XX13
0007 R 003446 XX2	0004 R 000000 XX20	0002 R 001055 XX3	0002 R 001056 XX4	0004 R 000001 XX48
0000 R 000001 Y	0000 R 002571 YS	0000 R 003721 YSIMP	0000 R 003411 YST	0002 R 000550 ZETA
0007 R 003453 ZZ2				

00100 1\* C  
 00100 2\* C THIS PROGRAM PROVIDES :  
 00100 3\* C 1- INPUT AND OUTPUT  
 00100 4\* C 2- UNIT CONVERSION  
 00100 5\* C 3- COMPUTATION OF DESIGN PARAMETERS  
 00100 6\* C 4- PREPARATION FOR PRINCIPLE INTEGRATION  
 00100 7\* C  
 00100 8\* C  
 00100 9\* C INPUT DATA  
 00100 10\* C \*\*\*\*\*  
 00100 11\* C  
 00100 12\* C THE FOLLOWING CARDS DESCRIBE EACH VARIABLE IN THE NAMELISTS  
 00100 13\* C WITH THEIR CORRESPONDING UNITS  
 00100 14\* C  
 00100 15\* C NAMELIST /SYSTEM/  
 00100 16\* C ISYM = 0 SINGLE, FLAT, SYMMETRICAL PANEL WITH STRAIGHT TUBES  
 00100 17\* C = 1 CURVED PANELS, STRAIGHT TUBES  
 00100 18\* C = 2 FLAT PANELS, U-SHAPED TUBES  
 00100 19\* C = 3 CURVED PANELS, U-SHAPED TUBES

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00100 20* C = 4 FLAT PANELS, STRAIGHT TUBES, NON-SYMMETRICAL LOADING
00100 21* C NSRD NUMBER OF RADIATING SIDES (1 OR 2)
00100 22* C NT NUMBER OF FLAT, SYMMETRICAL PANELS, EACH WITH NTBS TUBES
00100 23* C TSTAR ARRAY OF SINK TEMPERATURES FOR CARD INPUT
00100 24* C
00100 25* C NAMELISTS TUBEA, FINA, FLUIDA, AND MRI ARE USED ONLY FOR THE
00100 26* C NON-SYMMETRICAL CASE (ISYM>0)
00100 27* C
00100 28* C NAMELIST /TUBE/
00100 29* C D INNER DIAMETER OF TUBE [IN]
00100 30* C AL TUBE LENGTHS FOR STRAIGHT TUBE CASE [FT]
00100 31* C W1 INLET-OUTLET TUBE SPACING FOR U-SHAPE CASE [IN]
00100 32* C W2 CROSSOVER TUBE SPACING FOR U-SHAPE CASE [IN]
00100 33* C
00100 34* C NAMELIST /FINA/
00100 35* C H HALF-WIDTHS OF FINS FOR STRAIGHT TUBE CASE [IN]
00100 36* C TH FIN THICKNESSES [IN]
00100 37* C ABEMF FIN ABSORBTIVITY/EMISIVITY RATIO (USED WITH MRI INPUT)
00100 38* C
00100 39* C NAMELIST /FLUIDA/
00100 40* C TIN ENTRANCE (AND REFERENCE) TEMPERATURES [R]
00100 41* C PIN ENTRANCE (AND REFERENCE) PRESSURES [LBF/SQ.FT]
00100 42* C M COOLANT REFERENCE MASS FLOW RATE PER PANEL [LBM/HR]
00100 43* C
00100 44* C NAMELIST /MRI/
00100 45* C ITAPE UNIT NUMBER ATTACHED TO MRI TAPE
00100 46* C ICASE DATA CASE NUMBER ON MRI TAPE
00100 47* C NTM NUMBER OF DATA POINTS IN DATA CASE
00100 48* C TO TIME POINT IN ORBIT FROM WHICH TO TAKE DATA [MIN]
00100 49* C PHIN ANGLES OF PANELS RELATIVE TO MRI BASE DATA [DEGREES]
00100 50* C
00100 51* C THE NAMELIST QNML IS USED ONLY IN THE SYMMETRICAL CASE (ISYM=0)
00100 52* C
00100 53* C NAMELIST /QNML/
00100 54* C ITAPE UNIT NUMBER ATTACHED TO MRI TAPE
00100 55* C ICASE DATA CASE NUMBER ON MRI TAPE
00100 56* C NTM NUMBER OF DATA POINTS IN DATA CASE
00100 57* C TO STEADY-STATE TIME POINT AT WHICH TO TAKE DATA [MIN]
00100 58* C PHN ANGLE OF PANEL RELATIVE TO MRI BASE DATA [DEGREES]
00100 59* C
00100 60* C NAMELIST /TUBE/
00100 61* C DITBI TUBE INNER DIAMETER FOR SYMMETRICAL CASE [IN]
00100 62* C STBI TUBE THICKNESS [IN]
00100 63* C XL TUBE LENGTH FOR SYMMETRICAL CASE [FT]
00100 64* C RHOBTBI TUBE DENSITY [LBM/CU.FT]
00100 65* C MZ NUMBER OF NODAL POINTS ALONG THE TUBE LENGTH (LESS THAN OR
00100 66* C EQUAL TO 10)
00100 67* C NRTBI NUMBER OF NODAL POINTS IN THE TUBE WALL RADIAL DIRECTION
00100 68* C (LESS THAN OR EQUAL TO 5)
00100 69* C NTBS NUMBER OF TUBES
00100 70* C
00100 71* C THE NAMELIST FLOW IS USED ONLY IN THE SYMMETRICAL CASE (ISYM=0)
00100 72* C
00100 73* C NAMELIST /FLOW/
00100 74* C MDOFI TOTAL COOLANT REFERENCE MASS FLOW RATE [LBM/HR]
00100 75* C TO ENTRANCE (AND REFERENCE) TEMPERATURE [R]
00100 76* C PO ENTRANCE (AND REFERENCE) PRESSURE [LBF/SQ.FT]

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00100 77* C
00100 78* C  NAMELIST /FIN/
00100 79* C  NX      NUMBER OF NODAL POINTS ALONG THE FIN HEIGHT (LESS THAN
00100 80* C           OR EQUAL TO 10)
00100 81* C  SROOTI  FIN ROOT THICKNESS FOR SYMMETRICAL CASE [IN]
00100 82* C  HFNI    FIN HEIGHT FOR SYMMETRICAL CASE [IN]
00100 83* C  STIPI   FIN TIP THICKNESS FOR SYMMETRICAL CASE [IN]
00100 84* C  RHOFNI  FIN DENSITY [LBM/CU.FT]
00100 85* C  STAGX   DISTANCE FROM THE STAGNATION POINT ON THE SHUTTLE TO THE
00100 86* C           CENTER OF THE RADIATOR PANEL [FT]
00100 87* C  VERTX   OVERALL DIMENSION OF THE RADIATOR PANEL IN THE DIRECTION
00100 88* C           PARALLEL TO THE ACCELERATION OF GRAVITY [FT]
00100 89* C
00100 90* C  NAMELIST /PROTLR/
00100 91* C  NRMP    NUMBER OF NODAL POINTS IN THE PROTECTION LAYER RADIAL
00100 92* C           DIRECTION (LESS THAN OR EQUAL TO 5)
00100 93* C  RHOMPI  PROTECTION LAYER DENSITY [LBM/CU.FT]
00100 94* C  VELM    METEOROID VELOCITY [FT/SEC]
00100 95* C  TAU     TIME THAT VULNERABLE AREA IS EXPOSED TO METEOROID
00100 96* C           ENVIRONMENT [DAYS]
00100 97* C  PROB    PROBABILITY OF NO DAMAGE CAUSED BY METEOROID IMPACTS
00100 98* C           [DIMENSIONLESS]
00100 99* C  ALPHA   EXPERIMENTAL CONSTANT THAT RELATES METEOROID FLUX AND
00100 100* C          MASS [G4/(DAY.SQ.FT)]
00100 101* C  BTA     EXPERIMENTAL CONSTANT THAT RELATES METEOROID FLUX AND MASS
00100 102* C          [DIMENSIONLESS]
00100 103* C  GAMMA   EMPIRICAL CONSTANT USED TO ADJUST PREDICTED PENETRATION
00100 104* C          DEPTH TO ONES OBSERVED EXPERIMENTALLY [DIMENSIONLESS]
00100 105* C  PHI     EMPIRICAL CONSTANT [DIMENSIONLESS]
00100 106* C  THETA   EMPIRICAL CONSTANT [DIMENSIONLESS]
00100 107* C  ATK     EMPIRICAL CONSTANT USED TO ACCOUNT FOR SPALLING ON A
00100 108* C          TARGET OF FINITE THICKNESS [DIMENSIONLESS]
00100 109* C  AN      EXPERIMENTAL CONSTANT THAT DESCRIBES PENETRATION DEPTH AS
00100 110* C          A FUNCTION OF ANGLE OF INCIDENCE [DIMENSIONLESS]
00100 111* C
00100 112* C  NAMELIST /MANIFD/
00100 113* C  AMAN    TOTAL MANIFOLD AREA PROJECTED INTO THE FIN PLANE [SQ.FT]
00100 114* C
00100 115* C  NAMELIST /RUNOPT/
00100 116* C
00100 117* C  MSTOTR  = 1 TO COMPUTE STEADY STATE CONDITIONS
00100 118* C           = 2 TO SIMULATE TRANSIENT SYSTEM PERFORMANCE IN THE
00100 119* C           SYMMETRICAL CASE ONLY
00100 120* C  DTWRT   FIXED TIME INTERVAL BETWEEN DATA PRINTOUT DURING
00100 121* C           INTEGRATION [HR]
00100 122* C  TEND    TERMINATION TIME FOR TRANSIENT PERFORMANCE CALCULATIONS
00100 123* C           [HR]
00100 124* C  ALIMIT  ABSOLUTE ERROR LIMIT PER INTEGRATION STEP [DIMENSIONLESS]
00100 125* C  RLIMIT  RELATIVE ERROR LIMIT PER INTEGRATION STEP [DIMENSIONLESS]
00100 126* C  TI      INITIAL TEMPERATURE [R]
00100 127* C  LIMWRT  MAXIMUM NUMBER OF DATA RECORDING DURING INTEGRATION TOWARD
00100 128* C           STEADY STATE, EXCLUSIVE OF INITIAL CONDITIONS RECORD AND
00100 129* C           STEADY STATE RECORD
00100 130* C  NCONV   = 0 NO AERODYNAMIC HEATING
00100 131* C           = 1 ASCENT
00100 132* C           = 2 REENTRY
00100 133* C  LTT     = 0 OPTICAL PROPERTIES INDEPENDENT OF TEMPERATURE

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00100	134*	C	= 1 OPTICAL PROPERTIES DEPENDENT ON TEMPERATURE
00100	135*	C	= 2 QRFN AND GRMP ARE BOTH EQUAL TO ZERO BUT QINCID IS
00100	136*	C	CALLED
00100	137*	C	LFLD = 1 CONSTANT FLUID INLET CONDITIONS
00100	138*	C	= 2 VARIABLE FLUID INLET CONDITIONS
00100	139*	C	LTS = 0 UNIFORM INITIAL TEMPERATURE EQUAL TO TI
00100	140*	C	= 1 INITIAL TEMPERATURE AS OBTAINED BY PREVIOUS
00100	141*	C	COMPUTATIONS
00100	142*	C	
00100	143*	C	NAMelist/GINPT/
00100	144*	C	IT CONTAINS THE VARIABLES IN MOST OF THE PREVIOUS NAMELISTS, AND
00100	145*	C	IN ADDITION
00100	146*	C	MRSTRT = 1 NEW VELOCITY AND ALTITUDE PROFILES
00100	147*	C	= 2 NEW IRRADIATION HISTORY
00100	148*	C	= 3 BOTH 1 AND 2
00100	149*	C	= 4 NEW COOLANT INLET CONDITIONS
00100	150*	C	= 5 BOTH 1 AND 4
00100	151*	C	= 6 BOTH 2 AND 4
00100	152*	C	= 7 ALL (1, 2, AND 4)
00100	153*	C	= 8 ONLY NEW PARAMETERS FROM GINPT
00100	154*	C	IF THIS THIS NAMELIST IS OMITTED, THE PROGRAM WILL TERMINATE
00100	155*	C	
00100	156*	C	
00100	157*	C	THE FOLLOWING CARDS DESCRIBE THE INPUT VARIABLES, OTHER THAN THOSE
00100	158*	C	IN THE NAMELISTS, WITH THEIR CORRESPONDING UNITS
00100	159*	C	
00100	160*	C	NA NUMBER OF ORDERED PAIRS OF ELEMENTS IN THE VELOCITY-TIME
00100	161*	C	ARRAY FOR ASCENT PROFILES
00100	162*	C	NR NUMBER OF ORDERED PAIRS OF ELEMENTS IN THE ALTITUDE-TIME
00100	163*	C	ARRAY FOR REENTRY PROFILES
00100	164*	C	
00100	165*	C	TA ELEMENT OF TIME ARRAY (NA VALUES), SELECTED FOR ASCENT
00100	166*	C	VELOCITY AND ELEVATION PROFILES [SEC]
00100	167*	C	TR ELEMENT OF TIME ARRAY (NR VALUES), SELECTED FOR REENTRY
00100	168*	C	VELOCITY AND ELEVATION PROFILES [SEC]
00100	169*	C	
00100	170*	C	VELA ELEMENTS OF VELOCITY ARRAY (NA VALUES), SELECTED FOR
00100	171*	C	ASCENT VELOCITY PROFILE OF ORBITER [FT/SEC]
00100	172*	C	VELR ELEMENTS OF VELOCITY ARRAY (NR VALUES), SELECTED FOR
00100	173*	C	REENTRY VELOCITY PROFILE OF ORBITER [FT/SEC]
00100	174*	C	ALTA ELEMENT OF ALTITUDE ARRAY (NA VALUES), SELECTED FOR
00100	175*	C	ASCENT ELEVATION PROFILE OF ORBITER [FT]
00100	176*	C	ALTR ELEMENT OF ALTITUDE ARRAY (NR VALUES), SELECTED FOR
00100	177*	C	REENTRY ELEVATION PROFILE OF ORBITER [FT]
00100	178*	C	
00100	179*	C	TBM,TBMTL TUBE MATERIAL SPECIFICATION
00100	180*	C	(UP TO 12 ALPHANUMERIC CHARACTERS)
00100	181*	C	FNM,FNMTL FIN MATERIAL SPECIFICATION
00100	182*	C	(UP TO 12 ALPHANUMERIC CHARACTERS)
00100	183*	C	FLU,FLUID FLUID SPECIFICATION
00100	184*	C	(UP TO 12 ALPHANUMERIC CHARACTERS)
00100	185*	C	PRO,PROMTL PROTECTION LAYER MATERIAL SPECIFICATION
00100	186*	C	(UP TO 12 ALPHANUMERIC CHARACTERS)
00100	187*	C	
00100	188*	C	TMEFLD TIME AT WHICH FLUID INLET CONDITIONS ARE SPECIFIED [HR]
00100	189*	C	TIFLD INSTANTANEOUS FLUID INLET TEMPERATURE [R]
00100	190*	C	WIFLD INSTANTANEOUS FLUID MASS FLOW RATE [LBM/HR]

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00100 191* C NFLDTA NUMBER OF DATA POINTS DEFINING INLET CONDITIONS
00100 192* C (LESS THAN OR EQUAL TO 100)
00100 193* C
00100 194* C *****
00100 195* C
00100 196* C DECLARATION
00100 197* C
00101 198* PARAMETER NTP = 10
00103 199* PARAMETER NTP1 = NTP+1
00104 200* REAL MDOT,MDOTI,M
00105 201* EXTERNAL OERIVM,CNTLM
00106 202* DIMENSION Y(200),DY(200),A(200),R(200),DELY(200),PD(200),SD(200),
00106 203* 1 YS(200),DYST(200),YST(200),YSIMP(200),HDG(6),W1(NTP1),W2(NTP1),
00106 204* 2 W(NTP1,2)
00107 205* COMMON XITB(5),XIMP(5),FP(10),FW(10),FT(10),TW(10,5),TEMP(10,10),
00107 206* 1 TMP(10,5),GRFN(10,10),GRMP(10),XIFN(10),ZETA(10),AUX2(10),
00107 207* 2 CONFN(10,10),CONMP(10),COB(10),DOB(10)
00110 208* COMMON NX,MZ,NRTB,NRMP,NCTL,IFLOW,LT,LTB,LMP,LTBMZ,LTB2MZ,LL1,
00110 209* 1 LL2,LL3,LL4,LL5,LL6,LL7,NRMP1,NRTB1,NRMP2,NEQUS,LC,MSTOTR,
00110 210* 2 NCTM,NM1,MN1,LIMWRT,LIM,MODA,NCCZ,NCONV
00111 211* COMMON FRE,FR,FNU,FMACH,FPR,FRL,FRM,FFO,FOF,FZ,FRAD,DZ,DXITB,DXIMP
00111 212* 1 ,XRE,XX3,XX4,XX10,XX11,XX12,PHIM,PHIF,TO,PO,W0,RH00,CP0,PI
00111 213* 2 ,DELTA,RLIMIT,ROTWRT,TREF,TINTL,TI,RTEND,FLUX,GREF,FENTH,
00111 214* 3 FREJ,HFN,XL,COH,DXI,FRO,FXOH,FXHW,RHOFN
00112 215* COMMON /ADBH/ D,AL(NTP),H(NTP1),TH(NTP),TIN(NTP),PIN(NTP),
00112 216* 1 M(NTP),NT,TSTAR(NTP,3),VSC(NTP),CP(NTP),CONFL(NTP),
00112 217* 2 CONFNA(NTP),EMIS(NTP),TB(NTP),HAB(NTP1),EFF(NTP,2),
00112 218* 3 TL(NTP,2),QOUT(NTP),DTL(NTP),TOUT(NTP)
00113 219* COMMON/DVM/XX20,XX48
00114 220* COMMON/VELALT/TA(100),TR(100),ALTA(100),ALTR(100),VELA(100),
00114 221* 1 VELR(100),NA,NH
00115 222* COMMON/CNV/STAGX,VERTX,IOPTN,DX,NSRAD,SMPC,ST4,COEFF
00116 223* COMMON /QRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZ2(5),
00116 224* 1 MCVRD,LCT,LTT,DZMFN,
00116 225* 2 DX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DX21,SS(5,5,5)
00116 226* 3 ,SSTT(5,5)
00117 227* COMMON /QIN/TM(100),QSFN(100,2),QIFN(100,2),QSTB(100,2),
00117 228* 1 QITB(100,2),NTM,NSRD,QTO,TX
00120 229* COMMON /TC/ ITAPE,ICASE,TO,ABEMF,PHIN(NTP,3)
00121 230* COMMON /ABSRST/ALPHFN(5,5,6),ALPHMP(5,31),EMITFN(5,5),EMITMP(5),
00121 231* 1 EXTVC(50)
00122 232* COMMON /SSF/R2,T,ALPFN,R1,XSPM,XSMP,WWA,WWB
00123 233* COMMON /GEOM/ SFN(10),NTBS,TNXL,TBMASS,PLMASS,FLMASS,STR
00124 234* COMMON /FLOINL/ NFLDTA,LFLD,TMEFLD(100),TIFLD(100),WIFLD(100),
00124 235* 1 FLDINT
00125 236* COMMON /DVCML/ WRAT,FLUXI
00126 237* EQUIVALENCE (MDOT,FLUX),(ALPFN,CTA),(W(1,1),W1(1)),(W(1,2),W2(1))
00126 238* C
00127 239* NAMELIST /SYSTEM/ ISYM,NSRD,NT,TSTAR
00130 240* NAMELIST /TUBEA/ D,AL,W1,W2
00131 241* NAMELIST /FINA/ H,TH,ABEMF
00132 242* NAMELIST /FLUIDA/ TIN,PIN,M
00133 243* NAMELIST /MRI/ ITAPE,ICASE,NTM,TO,PHIN
00134 244* NAMELIST /QNML/ ITAPE,ICASE,NTM,TO,PHN
00135 245* NAMELIST/TUBE/ DITB,STBI,XL,RHOTBI,MZ,NRTBI,NTBS
00136 246* NAMELIST/FLOW/MDOTI,TO,PO
00137 247* NAMELIST/FIN/NX,SROOTI,HFNI,STIPI,RHOFNI,STAGX,VERTX

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00140 248*      NAMELIST/PROTLR/NRMP,RHOMPI,RHOMET,VELM,TAU,PROB,ALPHA,BTA,GAMMA,
00140 249*      1      PHI,THETA,ATK,AN
00141 250*      NAMELIST/MANIFD/ AMAN
00142 251*      NAMELIST/RUNOPT/MSTOTR,DTWRTE,TEND,ALIMIT,RLIMIT,TI,LIMWRT,NCONV,
00142 252*      1      LTT,LFLD,LTS
00143 253*      NAMELIST/GINPT/DITBI,STBI,XL,RHOTBI,MZ,NRTBI,NTBS,MDOTI,TO,PO,NX,
00143 254*      1      SROOTI,HFNI,STIPI,RHOFNI,NRMP,RHOMPI,RHOMET,VELM,TAU,
00143 255*      2      PROB,ALPHA,BTA,GAMMA,PHI,THETA,ATK,AN,AMAN,MSTOTR,
00143 256*      3      DTWRTE,TEND,ALIMIT,RLIMIT,TI,LIMWRT,NCONV,STAGX,VERTX,
00143 257*      4      LTT,MRSTRT,NSRD,ITAPE,ICASE,NTM,TO,PHN
00143 258*
00143 259*      C
00143 260*      C
00144 261*      155 FORMAT(1H1,/,59X,A6,' SURFACE',/,54X,I2,1X,A6,A2,'-TUBE PANELS',
00144 262*      1      /,58X,I1,' RADIATING ',A5,/)
00145 263*      255 FORMAT(//,' MRI TAPE PARAMETERS:',7X,'UNIT:',I3,4X,' CASE',
00145 264*      1      I3,3X,I4,' DATA POINTS:',5X,F8.3,' MINUTES INTO ORBIT',/,
00145 265*      2      ' FIN ABSORBTIVITY/EMISSIVITY RATIO:',F8.4)
00146 266*      355 FORMAT(//,' FIN ANGLES RELATIVE TO MRI TAPE DATA (DEGREES)',
00146 267*      1      /,10(2X,1PE10.4))
00147 268*      455 FORMAT(//,' FIN ANGLES AT OUTLET TUBES (MRI RELATIVE DEGREES)',
00147 269*      1      /,10(2X,1PE10.4))
00150 270*      555 FORMAT(//,' TUBE SPECIFICATIONS',6X,1PE10.4,
00150 271*      1      ' INCHES INSIDE DIAMETER')
00151 272*      655 FORMAT(//,' TUBE LENGTH (FEET)',/,10(2X,1PE10.4))
00152 273*      755 FORMAT(//,' INLET-OUTLET TUBE SPACING (INCHES)',/,11(2X,1PE10.4))
00153 274*      855 FORMAT(//,' CROSSOVER TUBE SPACING (INCHES)',/,11(2X,1PE10.4))
00154 275*      955 FORMAT(//,' FIN SPECIFICATIONS',/,11(2X,1PE10.4))
00154 276*      1055 FORMAT(//,' FIN HALF-WIDTH (INCHES)',/,11(2X,1PE10.4))
00155 277*      1155 FORMAT(//,' SINK TEMPERATURE (DEGREES R)',/,10(2X,1PE10.4))
00156 278*      1255 FORMAT(10(2X,1PE10.4))
00157 279*      1355 FORMAT(//,' FLUID SPECIFICATIONS',/,
00157 280*      1      ' INLET TEMPERATURE (DEGREES R)',/,10(2X,1PE10.4))
00160 281*      1455 FORMAT(//,' INLET PRESSURE (LBF/SQ.FT.)',/,10(2X,1PE10.4))
00161 282*      1555 FORMAT(//,' MASS FLOW RATE (LBM/HR)',/,10(2X,1PE10.4))
00162 283*
00162 284*      C
00163 285*      C
00163 286*      DATA PI/3.1415927/,SIDE/'SIDE',HDG/' FLAT','CURVED','STRAIGHT',
00163 287*      1      'U-SHAPED'/
00163 288*
00167 289*      C
00170 290*      MRSTRT = 15
00170 291*      READ(5,SYSTEM)
00173 292*      NSRD = MAX(MIN(NSRD,2-AND(ISYM,1)),1)
00174 293*      NSRAD = NSRD
00175 294*      IF (ISYM.NE.0) GO TO 2000
00177 295*      NT = 1
00200 296*      GO TO 8
00201 297*      2000 READ(5,TUBEA)
00204 298*      READ(5,FINA)
00207 299*      READ(5,FLUIDA)
00212 300*      READ(5,MRI)
00215 301*      NTF = (NT-1)*AND(ISYM,1)+1
00216 302*      NT1 = NT+1
00217 303*      IF (ITAPE.LE.0) GO TO 300
00221 304*      IF (ISYM.NE.3) GO TO 200
00223 305*      IU = (NT+1)/2
00224 306*      IW = NT/2+1

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00225 305*      ANGINC = (PHIN(IU,2)+PHIN(IW,2)-PHIN(IU,1)-PHIN(IW,1))/(NT-1)
00226 306*      DO 100 I = 1,NT
00231 307*      100 PHIN(I,3) = (PHIN(IU,1)+PHIN(IW,1)+ANGINC*(I-1))/2
00233 308*      200 CALL TCALE(NTF)
00234 309*      300 IF (NTF.EQ.NT) GO TO 500
00236 310*      DO 400 J = 1,NSRD
00241 311*      DO 400 I = 2,NT
00244 312*      400 TSTAR(I,J) = TSTAR(1,J)
00247 313*      500 IF (ISYM.NE.3.OR.ITAPE.LE.0) GO TO 800
00251 314*      PHN = (PHIN(IU,3)+PHIN(IW,3))/2
00252 315*      TSTARM = 0
00253 316*      DO 600 I = 1,NT
00256 317*      PHN(I,3) = PHN
00257 318*      600 TSTARM = TSTARM+TSTAR(I,3)
00261 319*      TSTARM = TSTARM/NT
00262 320*      DO 700 I = 1,NT
00265 321*      700 TSTAR(I,3) = TSTARM
00267 322*      800 IF (NSRD.EQ.2) SIDE = 'SIDES'
00271 323*      IH1 = AND(ISYM,1)+1
00272 324*      IH2 = AND(ISYM,2)+3
00273 325*      WRITE(6,155) HDG(IH1),NT,HDG(IH2),HDG(IH2+1),NSRD,SIDE
00303 326*      IF (ITAPE.GT.0) WRITE(6,255) ITAPE,ICASE,NTM,TO,ABEMF
00313 327*      IF (ITAPE.GT.0) WRITE(6,355) (PHIN(I,1),I=1,NTF)
00322 328*      IF (ISYM.EQ.3.AND.ITAPE.GT.0) WRITE(6,455) (PHIN(I,2),I=1,NTF)
00331 329*      WRITE(6,555) 0
00334 330*      IF (AND(ISYM,2).EQ.0) WRITE(6,655) (AL(I),I=1,NT)
00343 331*      IF (AND(ISYM,2).NE.0) WRITE(6,755) (W1(I),I=1,NT1)
00352 332*      IF (AND(ISYM,2).NE.0) WRITE(6,855) (W2(I),I=1,NT1)
00361 333*      WRITE(6,955) (TH(I),I=1,NT)
00367 334*      IF (AND(ISYM,2).EQ.0) WRITE(6,1055) (H(I),I=1,NT1)
00376 335*      WRITE(6,1155) (TSTAR(I,1),I=1,NT)
00404 336*      IF (ISYM.EQ.3) WRITE(6,1255) (TSTAR(I,3),I=1,NT)
00413 337*      IF (NSRD.EQ.2.OR.ISYM.EQ.3) WRITE(6,1255) (TSTAR(I,2),I=1,NT)
00422 338*      WRITE(6,1355) (TIN(I),I=1,NT)
00430 339*      WRITE(6,1455) (PIN(I),I=1,NT)
00436 340*      WRITE(6,1555) (M(I),I=1,NT)
00436 341*
00436 342*      C
00436 343*      C
00444 344*      C
00446 345*      ASCENT AND REENTRY PROFILE SPECIFICATIONS READ IN
00452 346*      8 IF (AND(MRSTRT,1).EQ.0) GO TO 12
00453 347*      READ(5,2) NA,NR
00461 348*      2 FORMAT (2I10)
00467 349*      READ (5,3) (TA(I), I = 1,NA)
00470 350*      READ (5,3) (TR(I), I = 1,NR)
00476 351*      3 FORMAT (10F8.3)
00504 352*      READ (5,4) (VELA(I),I=1,NA)
00512 353*      READ (5,4) (VELR(I),I=1,NR)
00520 354*      READ (5,4) (ALTA(I),I=1,NA)
00520 355*      READ (5,4) (ALTR(I),I=1,NR)
00521 356*      4 FORMAT ( 8E10.1)
00523 357*      C
00527 358*      12 IF (AND(MRSTRT,2).EQ.0) GO TO 24
00531 359*      IF (MRSTRT+ISYM.EQ.15) READ(5,QNML)
00532 360*      IF (ITAPE.GT.0) GO TO 24
00533 361*      PHN = 0.0
00533 361*      ICASE = 0
00533 361*      NS = 6

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00534 362*      DO 20 J = 1,NSRD
00537 363*      IU = J+2
00540 364*      K1 = J+3-2
00541 365*      K2 = K1+2
00542 366*      DO 14 I = 1,NTM
00545 367*      READ(5,3) TM(1),(QSFN(K,1),K=K1,K2),(QIFN(K,1),K=K1,K2),
00545 368*      1 (QSTB(K,1),K=K1,K2)
00564 369*      IF (J.EQ.2) READ(3) NS,TM(1),(QSFN(K,1),QIFN(K,1),QSTB(K,1),K=1,3)
00577 370*      14 WRITE(IU) NS,TM(1),(QSFN(K,1),QIFN(K,1),QSTB(K,1),K=1,6)
00612 371*      20 CALL CLOSE(J+2,1)
00612 372*      C
00612 373*      C MATERIAL SPECIFICATIONS READ IN
00614 374*      24 IF (MRSTRT.NE.15) GO TO 28
00616 375*      READ(5,16) TBM,TBMTL,FNM,FNMTL,FLU,FLUID,PRO,PROMTL
00616 376*      C
00616 377*      C SYSTEM SPECIFICATIONS READ IN
00616 378*      C
00630 379*      15 READ(5,TUBE)
00633 380*      IF (ISYM.EQ.0) READ(5,FLOW)
00637 381*      READ(5,FIN)
00642 382*      READ(5,PROTLR)
00645 383*      READ(5,MANIFD)
00645 384*      C
00645 385*      C PROGRAM CONTROL PARAMETERS READ IN
00645 386*      C
00650 387*      READ(5,RUNOPT)
00653 388*      16 FORMAT(2A6)
00653 389*      C
00653 390*      C FLUID INLET CONDITIONS READ IN
00653 391*      C
00654 392*      IF (LFLD.EQ.1) GO TO 30
00656 393*      28 IF (AND(MRSTRT,4).EQ.0) GO TO 30
00660 394*      READ(5,2) NFLDTA
00663 395*      READ(5,22) (TMEFLD(I),TIFLD(I),WIFLD(I),I=1,NFLDTA)
00673 396*      22 FORMAT(3F20.6)
00673 397*      C
00674 398*      30 IUL = AND(ISYM,2)+1
00675 399*      DO 840 I = 1,NT
00700 400*      840 M(I) = M(I)/(32.174*NTBS)
00702 401*      DO 9997 IU = 1,IUL
00705 402*      IF (AND(ISYM,2).EQ.0) GO TO 2100
00707 403*      IW = AND(IU,1)
00710 404*      H(1) = W(1,2-IW)
00711 405*      H(NT+1) = MIN(W1(NT+1),W2(NT+1))
00712 406*      IF (NT.EQ.1) GO TO 1100
00714 407*      DO 1000 IT = 2,NT
00717 408*      1000 H(IT) = W(1,2-IW)/2
00721 409*      1100 DO 1300 IT = 1,NT
00724 410*      AL(IT) = 0
00725 411*      I1 = IT+1
00726 412*      DO 1200 I = I1,NT1
00731 413*      1200 AL(IT) = W(1,1+IW)+AL(IT)
00733 414*      1300 AL(IT) = (2-AND(IU,1))*AL(IT)/12
00735 415*      2100 IF (ISYM.GT.0) CALL ADIABH(BOOL(AND(ISYM,3).EQ.3)*MOD(4-IU,3)+1)
00737 416*      DO 9997 IT = 1,NT
00742 417*      IF (ISYM.EQ.0) GO TO 34
00744 418*      MSTOTR = 1

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00745 419* IF (ITAPE.GT.0) PHN = PHIN(IT,IU)
00747 420* DITBI = D
00750 421* XL = AL(IT)
00751 422* MDOTI = 4(IT)*32.174*NTBS
00752 423* TO = TIN(IT)
00753 424* TIN(IT) = TOUT(IT)
00754 425* PO = PIN(IT)
00755 426* SROOTI = TH(IT)
00756 427* STIPI = TH(IT)
00757 428* HFNI = (HAB(IT)*EFF(IT,1)+(2*H(IT+1)-HAB(IT+1))*EFF(IT,2))/
00757 429* 1 (EFF(IT,1)+EFF(IT,2))-D/2-STBI
00760 430* 34 IF (AND(MRSTRT,2).NE.0) CALL QINCID (MSTOTR,PHN)
00760 431* C LISTING OF SYSTEM SPECIFICATIONS AND CONTROL PARAMETERS
00760 432* C
00762 433* WRITE(6,9998)
00764 434* WRITE(6,QNML)
00767 435* WRITE(6,TUBE)
00772 436* WRITE(6,FLOW)
00775 437* WRITE(6,FIN)
01000 438* WRITE(6,PROTLR)
01003 439* WRITE(6,MANIFD)
01006 440* WRITE(6,RUNOPT)
01006 441* C
01006 442* C UNIT CONVERSION ON INPUT DATA
01006 443* C
01011 444* DITB = DITBI/12.0
01012 445* STB = STBI /12.0
01013 446* SROOT = SROOTI/12.0
01014 447* STIP = STIPI/12.0
01015 448* HFN = HFNI /12.0
01016 449* RITB = DITB/2.0
01017 450* MDOT = MDOTI/32.174
01020 451* ROTB = (RITB+STB)*12.0
01021 452* RHOTB = RHOTBI/32.174
01022 453* RHOFN = RHOFNI/32.174
01023 454* RHOMP = RHOMPI/32.174
01023 455* C
01024 456* C TNN = FLOAT(NTBS)
01024 457* C
01024 458* C METEOROID PROTECTION LAYER THICKNESS
01024 459* C
01025 460* SYPC = TK(GAMMA,ATK,BTA,RHOMET,THETA,PHI,AN,ALPHA,VELM,PROB,
01025 461* 1 TAU,RHOMPI,XL,NN,AMAN,TO,ROTB)
01026 462* SMP = SMP/12.0
01026 463* C
01026 464* C INTERSECTION BETWEEN OUTER TUBE CIRCLE AND UPPER FIN PLANE
01026 465* C
01027 466* WWA = (STIP-SROOT)/(2.0*HFN)
01030 467* WWC = WWA**2+1.0
01031 468* SRO2 = SROOT/2.0
01032 469* WWR1 = RITB+STB
01033 470* WWR2 = WWR1+SMP
01034 471* WW3 = SRO2-WWA*WWR1
01035 472* WWD = SQRT(WWC*WWR2**2-WWB**2)
01036 473* XS4P = ((-WWA*WWB+WWD)/WWC-WWR1)/HFN
01036 474* C
01036 475* C CONTROL INTEGERS

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01036 476* C
01037 477* NRTB = NRTBI
01040 478* NM1 = NX-1
01041 479* MM1 = MZ-1
01042 480* NRMP1 = NRMP-1
01043 481* NRMP2 = NRMP-2
01044 482* NRTB1 = NRTB-1
01045 483* LT = MZ*NM1
01046 484* LTB = LT+MZ
01047 485* LMP = LTB+MZ*NRTB
01050 486* LTBMZ = LTB+MZ
01051 487* LTB2MZ = LTBMZ+MZ
01052 488* LL1 = LTB*(NRTB-2)*MZ
01053 489* LL2 = LL1-MZ
01054 490* LL3 = LMP+MZ
01055 491* LL4 = LL1+MZ
01056 492* LL5 = LMP+NRMP2*MZ
01057 493* LL6 = LL5-MZ
01060 494* LL7 = LL6-MZ
01061 495* LC = MZ/2
01062 496* IF(NCONV.EQ.1) IOPTN = 0
01064 497* IF(NCONV.EQ.2) IOPTN = 1
01064 498* C
01064 499* C GRID INTERVALS
01064 500* C
01066 501* OZ = 1.0/FLOAT(MM1)
01067 502* DXI = 1.0/FLOAT(NM1)
01070 503* DXITB = STB/(RITB*FLOAT(NRTB1))
01071 504* DXIFN = HFN/RITB*OXI
01072 505* DXIMP = SMP/(RITB*FLOAT(NRMP1))
01073 506* DELTA = OITB/XL
01073 507* C
01073 508* C GRID SPACINGS
01073 509* C
01074 510* XITB(1) = 1.0
01075 511* DO 50 I = 2,NRTB
01100 512* 50 XITB(I) = XITB(I-1)+DXITB
01102 513* XIMP(1) = XITB(NRTB)
01103 514* DO 55 I = 2,NRMP
01106 515* 55 XIMP(I) = XIMP(I-1)+DXIMP
01110 516* ZETA(1) = 0.0
01111 517* DO 60 J = 2,MZ
01114 518* 60 ZETA(J) = ZETA(J-1)+DZ
01116 519* XIFN(1) = 0.0
01117 520* DO 65 J = 2,NX
01122 521* 65 XIFN(J) = XIFN(J-1)+DXI
01122 522* C
01124 523* MMEP = (INT(2.0*XSPM/DXI)+1)/2
01125 524* MCVRD = MMEP
01126 525* IF(XSMP.GE.XIFN(MMEP+1)) MMEP = MMEP+1
01126 526* C
01130 527* OXX2 = (1.0-XSMP)/4.5
01131 528* DXX21 = 2.0
01132 529* XX2(1) = XSMP+DXX2/2.0
01133 530* XSPM = ((0.5+FLOAT(MCVRD))*DXI-XSMP)/DXI
01134 531* IF(MCVRD.EQ.0) XSPM = (DXI-2.0*XSMP)/DXI
01136 532* ZZ2(1) = 0.0

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01137 533*      DO 66 I = 2,5
01142 534*      Z22(I) = Z22(I-1)+0.25
01143 535*      66 XX2(I) = XX2(I-1)+DXX2
01143 536*      C
01143 537*      C      GENERAL CONSTANTS
01143 538*      C
01145 539*      TOL = SROOT/(2.0*HFN)
01146 540*      CTA = TOL-STIP/(2.0*HFN)
01147 541*      DTA = SQRT(1.0+CTA**2)
01150 542*      DO 70 I=2,NM1
01153 543*      BOL = TOL-CTA*XIFN(I)
01154 544*      DOB(I) = DTA/(2.0*BOL)
01155 545*      70 COB(I) = CTA/BOL
01155 546*      C
01157 547*      SLOPE = (SROOT-STIP)/(2.0*HFN)
01160 548*      SFN(1) = SROOT
01161 549*      SFN(NX) = STIP
01162 550*      DO 80 I = 2,NM1
01165 551*      80 SFN(I) = STIP+2.0*HFN*SLOPE*(1.0-XIFN(I))
01165 552*      C
01165 553*      C
01167 554*      RH00 = RHOF(P0,T0)
01170 555*      RH0I = RHOF(P0,TI)
01171 556*      XRE = 2.0*MOOT/(PI*TNN*RITB)
01172 557*      FRL = (DELTA/2.0)**2
01173 558*      FFO = 4.0/(XRE*DELTA)*RH00/RHOTB
01174 559*      FOF = FFO*RHOTB/RHOMP
01175 560*      REY = XRE/VISC(RH0I,TI)
01176 561*      FRE = RH00*DELTA/XRE
01177 562*      FRM = ((RITB+STB)/XL)**2
01200 563*      FRD = 1.714E-09*T0**3
01201 564*      FRAD = FRD*RITB
01202 565*      FRD = FRD*HFN
01203 566*      PRLNTL = (VISC(RH0I,TI)*CPF(RH0I,TI))/THCF(RH0I,TI)
01204 567*      QREF = FRD*T0*TNN**2.0
01205 568*      AITB = TNN*PI*RITB**2
01206 569*      W0 = XRE/(DITB*3600.0*RH00)
01207 570*      CPO = CPF(RH00,T0)
01210 571*      HFLO = HFL(RH00,T0)
01211 572*      FENTH = FLUX*(HFLO+W0**2/1556.36)
01212 573*      FREJ = PI*T0
01213 574*      FXOH = (HFN/XL)**2
01214 575*      FXHW = XL/(W0*3600.0*HFN**2)
01214 576*      C
01215 577*      T = SRO2/HFN
01216 578*      R1 = WNR1/HFN
01217 579*      R2 = WNR2/HFN
01220 580*      DZMFN = XL/(HFN*4.0)
01220 581*      C
01221 582*      TREF = XL*DITB/XRE*RH00
01222 583*      ROTWRT = DTWRT/TREF
01223 584*      RTEND = TEND/TREF
01224 585*      STR = T0*HFN*(SROOT+STIP)*TNN*SQRT(RHOFN/(PI*TREF))
01224 586*      C
01225 587*      XX3 = DXITB/DXIMP
01226 588*      XX4 = DXITB/DXIFN
01227 589*      XX10 = 1.0/(4.0*XIMP(NRMP)-DXIMP)

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01230 590*      XX11 = 4.0*XX10*(2.0*XIMP(NRMP)-DXIMP)/(DXIMP**2)
01231 591*      XX12 = 8.0*XX10*XIMP(NRMP)/DXIMP
01231 592*      C
01232 593*      PHIF = 4.0*ASIN(SROOT/(2.0*WWR1+SMP))
01233 594*      PHIM = 2.0*PI-4.0*ASIN(SROOT/(2.0*WWR2))
01234 595*      PHIF = PHIF*RITB/HFN
01235 596*      COM = PHIM*WWR2/(2.0*HFN)
01235 597*      C
01235 598*      C
01236 599*      COEFF = 3600./STAGX
01237 600*      ST4 = 0.1714E-08*T0**4.0
01240 601*      QTO = ST4
01241 602*      DX = HFN*DXI*12.0
01241 603*      C
01241 604*      C INITIAL TEMPERATURE PROFILES
01241 605*      C
01242 606*      IF (MRSTRT-LTS*8.LE.0) GO TO 312
01244 607*      TINTL = TI/T0
01245 608*      DO 310 J=1,MZ
01250 609*      DO 302 I=1,NRTB
01253 610*      302 TW(J,I) = TINTL
01255 611*      DO 305 I=1,NRMP
01260 612*      305 TMP(J,I) = TINTL
01262 613*      CONMP(J) = 0.0
01263 614*      DO 310 I=1,NX
01266 615*      CONFN(J,I) = 0.0
01267 616*      310 TEMP(J,I) = TINTL
01267 617*      C
01267 618*      C INITIAL FLOW CONDITIONS
01267 619*      C
01272 620*      312 IFLOW = 1
01273 621*      FI(1) = TW(1,1)
01274 622*      IF (LFLO.EQ. 2) FI(1) = TIFLO(1)/T0
01276 623*      WRAT = 1.0
01277 624*      FLUXI = MDOTI
01300 625*      IF (LFLO.EQ. 2) WRAT = WIFLO(1)/FLUXI
01302 626*      CALL FLSTRT(REY,PRNLNTL,DELTA)
01302 627*      C
01302 628*      C CRITERIA FOR LUMPED-PARAMETER TREATMENT OF TUBE PLUS
01302 629*      C PROTECTION LAYER
01302 630*      C
01303 631*      TAVG = 0.5*(TI+T0)
01304 632*      XNUSLT = FNU*DELTA*REY*PRNLNTL/4.0
01305 633*      BIOTNO = XNUSLT/DITB*THCF(RH00,T0)*(STB/THCTB(TAVG)+SMP/THCMP
01305 634*      1 (TAVG))
01306 635*      WRITE(6,313) XNUSLT,BIOTNO
01312 636*      313 FORMAT(1H,5X,25HINIT. NUSSULT NO. NU = ,E20.6//,
01312 637*      1 6X,25HWALL BIOT NO. BI = ,E20.6//)
01313 638*      IF(BIOTNO.GT.0.05) GO TO 314
01315 639*      NRTB = 0
01316 640*      LL4 = MZ*NX
01317 641*      LL5 = LL4
01320 642*      LL6 = MZ-2
01321 643*      LTB = LL4
01322 644*      NEQUS = MZ*(NX+1)
01322 645*      C
01322 646*      C

```

```

01322 647* C MASS CALCULATIONS
01322 648* C
01323 649* 314 DO 315 J=1,MZ
01326 650* 315 Y(J) = 1.0/FW(J)
01330 651* TNXL = TNN*XL
01331 652* FLMASS = TNXL*AITB*RH00*DEFINT(Y,DZ,MZ)
01332 653* FNMASS = TNXL*HFN*RHOFN*(SROOT+STIP)
01333 654* TBMASS = TNXL*PI*STB*(DITB+STB)*RH0TB
01334 655* PLMASS = TNXL*SMP*RH0MP*(DITB+2.0*STB+SMP)*PHIM/2.0
01335 656* TOTMSS = FLMASS+FNMASS+PLMASS+TBMASS
01336 657* FLMASI = FLMASS*32.174
01337 658* FNMASI = FNMASS*32.174
01340 659* TBMASI = TBMASS*32.174
01341 660* PLMASI = PLMASS*32.174
01342 661* TOTMSI = TOTMSS*32.174
01342 662* C
01343 663* ATOT = TNXL*(2.0*(HFN+STB)+DITB)
01344 664* WRITE(6,320) XL,DITBI,STBI,TBMTL,TBMASI,NTBS,
01344 665* 1 HFN,SROOTI,STIPI,FNM,FNMTL,FNMASI,NSRAD,
01344 666* 2 FLU,FLUID,FLMASI
01367 667* WRITE (6,321) SMP,PLMASI,PRO,PROMTL,
01367 668* 1 TOTMSI,ATOT
01377 669* 320 FORMAT(1H0,15X,17HSYSTEM PARAMETERS,/,16X,17H*****//,
01377 670* 2 6X,35HTUBE LENGTH , XL = ,F10.3,11H FT //,
01377 671* 3 6X,35H INTERNAL DIAMETER , DITB = ,F10.3,11H IN //,
01377 672* 4 6X,35H WALL THICKNESS , STB = ,F10.3,11H IN //,
01377 673* 5 6X,35H MATERIAL , ,2A6 //,
01377 674* 6 6X,35H MASS (ALL TUBES) , MTB = ,F10.4,11H LBM //,
01377 675* 1 6X,35H NUMBER OF TUBES, NTBS = ,I10 //,
01377 676* 7 6X,35HFIN HEIGHT , HFN = ,F10.3,11H IN //,
01377 677* 8 6X,35H THICKNESS AT ROOT , SROOT = ,F10.3,11H IN //,
01377 678* 9 6X,35H THICKNESS AT TIP , STIP = ,F10.3,11H IN //,
01377 679* 1 6X,35H MATERIAL , ,2A6 //,
01377 680* 2 6X,35H MASS (ALL FIN) , MFM = ,F10.4,11H LBM //,
01377 681* 3 6X,35H NO. OF FIN SIDES RADIATING = ,I10 //,
01377 682* 3 6X,35HCOOLANT FLUID IS , ,2A6 //,
01377 683* 3 6X,35H MASS (IN ALL TUBES) , MFL = ,F10.4,11H LBM //,
01400 684* 321 FORMAT (1H0,6X,35HPROTECTION LAYER THICKNESS, SMP = ,F10.3,11H IN
01400 685* 1 //,
01400 686* 6 6X,35H MASS, MMP = ,F10.3,11H LBM //,
01400 687* 7 6X,35H MATERIAL IS , ,2A6 //,
01400 688* 8 6X,35HTOTAL MASS (EXCL. MANIFLD.) MTOT = ,F10.4,11H LBM //,
01400 689* 9 6X,39HTOTAL AREA (SINGLE NORMAL PROJECTION) ,//,
01400 690* 1 6X,35H ATOT = ,F10.4,11H SQ FT //)
01400 691* C
01400 692* C INITIALIZATION
01400 693* C
01400 694* C
01400 695* C TUBE
01400 696* C
01401 697* C IF(NRTB.EQ.0) GO TO 341
01401 698* C
01403 699* PFLD = P0*FP(LC)
01404 700* TFLD = T0*FT(LC)
01405 701* RHOFLO = RHOF(PFLD,TFLD)
01406 702* XFNU = THCF(RHOFLO,TFLD)/(4.0*THCTB(TINTL))*FNU*REY*PRLNTL
01406 703* 1 *DELTA*DXITB

```

\*\*\*\*\* MAIN \*\*\*\*\*

```

01407 704*      DO 330 J=1,MZ
01412 705*      DO 325 I=2,NRTB1
01415 706*      K      = LTB+MZ*(I-1)+J
01416 707*      325 Y(K)  = TW(J,I)
01420 708*      LTBJ   = LTBJ+J
01421 709*      330 Y(LTBJ) = (XFNU*FT(J)+4.0*TW(J,2)-TW(J,3))/(XFNU+3.0)
01421 710*      C
01421 711*      C      PROTECTION LAYER
01421 712*      C
01421 713*      C
01423 714*      DO 340 J=1,MZ
01426 715*      DO 335 I=2,NRMP1
01431 716*      K      = LMP+MZ*(I-2)+J
01432 717*      335 Y(K)  = TMP(J,I)
01434 718*      Y(LL4+J) = TMP(J,1)
01435 719*      340 Y(LL5+J) = TMP(J,NRMP)
01435 720*      C
01435 721*      C      FIN
01435 722*      C
01437 723*      341 DO 350 I=2,NX
01437 724*      C
01442 725*      DO 350 J=1,MZ
01445 726*      K      = MZ*(I-2)+J
01446 727*      350 Y(K)  = TEMP(J,I)
01446 728*      C
01446 729*      C      FLUID
01446 730*      C
01451 731*      DO 360 J=1,MZ
01454 732*      360 Y(LT+J) = FT(J)
01454 733*      C
01456 734*      IF(NRTB.NE.0) GO TO 364
01460 735*      DO 361 J=1,MZ
01463 736*      K      = LL4+J
01464 737*      361 Y(K)  = TW(J,1)
01464 738*      C
01466 739*      XX3      = (2.0-PHIM/PI)*(RITB+STB+SMP)
01467 740*      XX4      = 2.0*(SROOT-DXI/2.0*(SROOT-STIP))/(PI*HFN)
01470 741*      XX10     = RHQTB*STB*(DITB+STB)/TREF
01471 742*      XX11     = RHOMP*SMP*(DITB+2.0*STB+SMP)*PHIM/(2.0*PI*TREF)
01472 743*      XX12     = RHOFN*DXI*HFN/PI*(SROOT-DXI/4.0*(SROOT-STIP))/TREF
01473 744*      XX13     = PHIM*(RITB+STB+SMP)
01474 745*      XX20     = DXI*HFN/XX13
01475 746*      XX48     = DXI*12.0
01476 747*      FRAD     = FRD*XX13/(HFN*PI)
01476 748*      C
01477 749*      364 DO 365 I=1,NEQU5
01502 750*      A(I)    = ALIMIT
01503 751*      365 R(I)    = RLIMIT
01505 752*      RLIMIT   = RLIMIT*5.0
01505 753*      C
01505 754*      C
01506 755*      IF (OR(MRSTRY*2-LTY).GT.8) CALL SHAPEF
01506 756*      C
01506 757*      C
01506 758*      C
01510 759*      TIME    = 0.0
01511 760*      DTME    = 0.001

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\*\*\*\*\* MAIN \*\*\*\*\*

```

01511 761* C
01512 762* LCT = 1
01513 763* NCTM = 0
01514 764* LIM = 0
01515 765* MODA = 0
01516 766* NCCZ = 0
01516 767* C
01517 768* NTRY = 1
01520 769* IFVD = 0
01521 770* IBKP = 1
01522 771* IERR = 0
01522 772* C
01522 773* C PRINCIPLE INTEGRATION
01522 774* C
01523 775* CALL RKS(DERIVM,CNTLM,Y,DY,A,R,TIME,DTME,NEGUS,IFVD,IBKP,NTRY,IERR
01523 776* 1 ,DELY,PD,SD,YS,YST,DYST,YSIMP)
01524 777* 9997 RLIMIT = R(1)
01527 778* IF (ISYM.NE.0) GO TO 9999
01531 779* READ (5,GINPT, END = 9999)
01534 780* MRSTRT = AND(MRSTRT,7)
01535 781* GO TO 8
01536 782* 9998 FORMAT(1H1)
01537 783* 9999 STOP
01540 784* END

```

END OF COMPILATION: NO DIAGNOSTICS.

QMDG.P \*\*\*\*\* MTXINV \*\*\*\*\*

\*\*\*\*\* MTXINV \*\*\*\*\*

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PAGE 1

QFOR:5 ME\*NASAS.MTXINV,ME\*NASAS.MTXINV  
FOR 59A-07/13/72-20:57:03 (0,)

SUBROUTINE MTXINV ENTRY POINT 000166

STORAGE USED: CODE(1) 000206; DATA(0) 000037; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 QRD 003670

EXTERNAL REFERENCES (BLOCK, NAME)

0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000016	1056	0001	000076	11L	0001	000031	1116	0001	000052	1216	0001	000055	1256			
0001	000117	1366	0001	000122	1426	0001	000137	20L	0000	R	000001	AA	0000	R	000005	B	
0003	003465	CSF	0003	003464	DX2	0003	003472	DX21	0003	003463	OZMFN	0003	003410	EBFN			
0003	003441	EBMP	0003	003467	EXTIFN	0003	003471	EXTIMP	0003	003466	EXTSFN	0003	003470	EXTSMP			
0000	I	000000	I	000011	INJP5	0000	I	000003	I1	0000	I	000006	I2	0000	I	000002	J
0000	I	000004	K	0003	003461	LCT	0003	003462	LTT	0003	003460	MCVRD	0003	003473	SS		
0003	R	000000	TRMTX	0003	003446	XX2	0003	003453	ZZ2								

```

00101      1*      SUBROUTINE MTXINV(N,M)
00101      2*      C
00101      3*      C THIS SUBROUTINE SOLVES A SYSTEM OF LINEAR ALGEBRIC EQUATIONS BY
00101      4*      C MATRIX INVERSION
00101      5*      C
00103      6*      COMMON /QRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZ2(5),
00103      7*      1 MCVRD,LCT,LTT,OZMFN,
00103      8*      2 DX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DX21,SS(5,5,5)
00104      9*      DO 20 I =1,N
00107     10*      AA =TRMTX(I,I)
00110     11*      DO 9 J =1,M
00113     12*      9 TRMTX(I,J)=TRMTX(I,J)/AA
00115     13*      IF (I.EQ.1) GO TO 11
00117     14*      I1 = I-1
00120     15*      DO 10 K =1,I1
00123     16*      B =TRMTX(K,I)
00124     17*      DO 10 J =1,M
00127     18*      10 TRMTX(K,J)=TRMTX(K,J) -TRMTX(I,J) * B
00132     19*      IF (I.EQ.N) GO TO 20
00134     20*      11 I2 = I+1
00135     21*      DO 15 K =I2,N
00140     22*      B =TRMTX(K,I)
00141     23*      DO 15 J =1,M
00144     24*      15 TRMTX(K,J)=TRMTX(K,J) -TRMTX(I,J) * B
00147     25*      20 CONTINUE

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\*\*\*\*\* MTXINV \*\*\*\*\*

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00151 26\* RETURN  
00152 27\* END

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* NUS \*\*\*\*\*

\*\*\*\*\* NUS \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME NASAS.NUS ME NASAS.NUS  
FOR S9A-07/13/72-20:57:08 (0)

SUBROUTINE NUS ENTRY POINT 000573

STORAGE USED: CODE(1) 000642; DATA(0) 000114; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 YINT  
0004 NEXP6\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000550	100L	0001	000354	150G	0001	000374	161G	0001	000407	166G	0001	000443	175G
0001	000243	20L	0001	000456	202G	0001	000513	211G	0001	000526	216G	0001	000324	30L
0001	000431	40L	0001	000501	50L	0000	R	000015	BETA	0000	R	000021	FORCE	0000
0000	R	000017	GP	0000	R	000016	GRASH	0000	I	000023	I	0000	000066	INJP\$
0000	R	000004	NUS	0000	R	000010	REY	0000	R	000014	REYNO	0000	R	000022
														0003
														R
														000000
														YINT

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00101      1*      SUBROUTINE NUS(MACHNO,TATM,CATM,TIN,PRATM,VISATM,RHOATM,
00101      2*      1      STAGX,VERTX,NSRAD,NUS1)
00101      3*      C
00101      4*      C THIS SUBROUTINE COMPUTES :
00101      5*      C A SINGLE NUSSELT NUMBER FOR THE CALCULATION OF THE AERODYNAMIC
00101      6*      C HEATING OF THE ORBITER RADIATOR SYSTEM AT THE CURRENT TIME
00101      7*      C
00103      8*      REAL MACHNO,NUS1,MACH(4),NUS(4)
00104      9*      DIMENSION REY(4)
00105     10*      REYNO = MACHNO*STAGX*CATM*RHOATM/VISATM
00106     11*      IF (MACHNO .GT. 1.0) GO TO 20
00110     12*      IF (MACHNO .LE. 1.0 .AND. MACHNO .GE. 0.5) GO TO 30
00110     13*      C
00110     14*      C FREE CONVECTION NUSSELT NUMBER
00110     15*      C
00112     16*      BETA = 2.0/(TIN+TATM)
00113     17*      GRASH = 32.174*BETA*(RHOATM/VISATM)**2*VERTX**3.*(TIN-TATM)
00114     18*      GP = GRASH*PRATM
00115     19*      IF (GP .GT. 1.0E-01 .AND. GP .LE. 1.0E+04) FREE = 1.585*(GP)
00115     20*      1      **0.195
00117     21*      IF (GP .GT. 1.0E+04 .AND. GP .LE. 1.0E+09) FREE = 0.59*(GP)**
00117     22*      1      0.25
00121     23*      IF (GP .GT. 1.0E+09) FREE = 0.13*(GP)**0.33333
00121     24*      C
00121     25*      C LOW SPEED NUSSELT NUMBERS FOR LAMINAR, TURBULENT AND TRANSITION FLOWS
00121     26*      C
00123     27*      IF (REYNO .LT. 1.0E+05)
00123     28*      1 FORCE = 0.332*REYNO**0.5*PRATM**0.33333
00125     29*      IF (REYNO .GT. 1.0E+06)

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\*\*\*\*\* NUS \*\*\*\*\*

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00125 30* 1 FORCE = 0.0288*REYN0**0.8*PRATM**0.33333
00127 31* IF (REYN0 .LE. 1.0E+06 .AND. REYN0 .GE. 1.0E+05)
00127 32* 1 FORCE = 6.78E-05*PRATM**0.33333*REYN0**1.238
00131 33* NUS1 = FREE*STAGX/VERTX*FORCE
00132 34* GO TO 100
00132 35* C
00132 36* C HIGH SPEED NUSSELT NUMBERS FOR LAMINAR, TURBULENT AND TRANSITION FLOWS
00132 37* C
00133 38* 20 IF (REYN0 .LT. 1.0E+05)
00133 39* 1 NUS1 = 0.3751*PRATM*REYN0**0.5014
00135 40* IF (REYN0 .GT. 1.0E+06)
00135 41* 1 NUS1 = 0.0346*PRATM*REYN0**0.7746
00137 42* IF (REYN0 .LE. 1.0E+06 .AND. REYN0 .GE. 1.0E+05)
00137 43* 1 NUS1 = 3.339E-04*PRATM*REYN0**1.1111
00141 44* GO TO 100
00141 45* C
00141 46* C LAMINAR, TRANSITION AND TURBULENT NUSSELT NUMBERS FOR FLOWS THAT LIE
00141 47* C BETWEEN LOW AND HIGH SPEED MODELS
00141 48* C
00142 49* 30 MACH(1) = 0.4000
00143 50* MACH(2) = 0.6333
00144 51* MACH(3) = 0.8666
00145 52* MACH(4) = 1.1000
00146 53* X = STAGX*CATTM*RHDATM/VISATM
00147 54* DO 34 I = 1,4
00152 55* 34 REY(I) = MACH(I)*X
00154 56* IF (REYN0 .GT. 1.0E+06) GO TO 40
00156 57* IF (REYN0 .LT. 1.0E+05) GO TO 50
00160 58* DO 36 I = 3,4
00163 59* 36 NUS(I) = 3.339E-04*PRATM*REY(I)**1.1111
00165 60* DO 38 I = 1,2
00170 61* 38 NUS(I) = 6.78E-05*PRATM**0.33333*REY(I)**1.238
00172 62* NUS1 = YINT(MACH,NUS,4,4,MACHNO)
00173 63* GO TO 100
00174 64* 40 DO 44 I = 3,4
00177 65* 44 NUS(I) = 0.0346*PRATM*REY(I)**0.7746
00201 66* DO 48 I = 1,2
00204 67* 48 NUS(I) = 0.0288*REY(I)**0.8*PRATM**0.33333
00206 68* NUS1 = YINT(MACH,NUS,4,4,MACHNO)
00207 69* GO TO 100
00210 70* 50 DO 54 I = 3,4
00213 71* 54 NUS(I) = 0.3751*PRATM*REY(I)**0.5014
00215 72* DO 58 I = 1,2
00220 73* 58 NUS(I) = 0.322*REY(I)**0.5*PRATM**0.33333
00222 74* NUS1 = YINT(MACH,NUS,4,4,MACHNO)
00223 75* 100 IF (NSRAD .EQ. 2) NUS1 = 2.0*NUS1
00225 76* RETURN
00226 77* END

```

END OF COMPILATION:

NO DIAGNOSTICS.

OHOG:P \*\*\*\*\* NUSA \*\*\*\*\*



\*\*\*\*\* NUSA \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME+NUSA5,NUSA,ME+NUSA5,NUSA  
FOR S9A-07/13/72-20:57:12 (0,)

FUNCTION NUSA ENTRY POINT 000067

STORAGE USED: CODE(1) 000103; DATA(0) 000024; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NEXP6\$  
0004 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000024 2L 0001 000043 4L 0001 000057 5L 0000 000016 INJP\$ 0000 R 000000 NUSA  
0000 R 000001 RPD

00101 1\* FUNCTION NUSA(REY,PR,DOL)  
00103 2\* REAL NUSA  
00104 3\* IF(REY-2300)1,1,2  
00107 4\* 1 RPD=REY\*PR\*DOL  
00110 5\* NUSA=3.65+0.0668\*RPD/(1+.045\*RPD\*\*.6667)  
00111 6\* GO TO 5  
00112 7\* 2 IF(PR-.1)3,3,4  
00115 8\* 3 NUSA=5+.025\*(REY\*PR)\*\*.8  
00116 9\* GO TO 5  
00117 10\* 4 NUSA=.023\*(REY\*\*.8)\*(PR\*\*.3)  
00120 11\* 5 RETURN  
00121 12\* END

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* PDERIV \*\*\*\*\*

\*\*\*\*\* PDERIV \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASA5,PDERIV,ME\*NASA5,PDERIV  
FOR S9A-07/13/72-20:57:13 (0,)

FUNCTION PDERIV ENTRY POINT 000160

STORAGE USED: CODE(1) 000204; DATA(0) 000020; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 ADBH 000352

EXTERNAL REFERENCES (BLOCK, NAME)

0004 TTIPS  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000036 2L	0001 000114 3L	0003 000001 AL	0003 000161 CONFL	0003 000173 CONFNA
0003 000147 CP	0003 000000 D	0000 R 000001 DM	0003 R 000326 DTL	0003 000244 EFF
0003 000205 EMIS	0003 R 000013 H	0003 R 000231 HAB	0000 000007 INJPS	0003 R 000064 M
0003 000076 NT	0000 R 000000 PDERIV	0003 000052 PIN	0003 000314 GOUT	0003 000217 TB
0003 000026 TH	0003 000040 TIN	0003 R 000270 TL	0003 000340 TOUT	0003 000077 TSTAR
0000 R 000003 T1	0000 R 000002 T2	0003 000135 VSC		

```

00101 1*      FUNCTION PDERIV(J,IT,ITST)
00103 2*      PARAMETER NTP = 10
00104 3*      PARAMETER NTP1 = NTP+1
00105 4*      REAL M
00106 5*      COMMON /ADBH/ D,AL(NTP),H(NTP1),TH(NTP),TIN(NTP),PIN(NTP),
00106 6*      M(NTP),NT,TSTAR(NTP,3),VSC(NTP),CP(NTP),CONFL(NTP),
00106 7*      CONFNA(NTP),EMIS(NTP),TB(NTP),HAB(NTP1),EFF(NTP,2),TL(NTP,2),
00106 8*      GOUT(NTP),DTL(NTP),TOUT(NTP)
00107 9*      IF(J) ,2,3
00112 10*     CALL TTIPS(IT,ITST,1.01*HAB(IT),2*H(IT+1)-HAB(IT+1),DM,T2)
00113 11*     PDERIV=(T2-TL(IT+1,1)-DTL(IT))/(.01*HAB(IT))
00114 12*     RETURN
00115 13*     2   CALL TTIPS(IT,ITST,HAB(IT),2.0*H(IT+1)-1.01*HAB(IT+1),DM,T2)
00116 14*     CALL TTIPS(IT+1,ITST,1.01*HAB(IT+1),2*H(IT+2)-HAB(IT+2),T1,DM)
00117 15*     PDERIV=(T2-T1-DTL(IT))/(.01*HAB(IT+1))
00120 16*     RETURN
00121 17*     3   CALL TTIPS(IT+1,ITST,HAB(IT+1),2*H(IT+2)-1.01*HAB(IT+2),T1,DM)
00122 18*     PDERIV=(TL(IT,2)-T1-DTL(IT))/(.01*HAB(IT+2))
00123 19*     RETURN
00124 20*     END

```

END OF COMPILATION: NO DIAGNOSTICS.

\*\*\*\*\* PDERIV \*\*\*\*\*

DATE 071372

PAGE 2

QHOG:P \*\*\*\*\* PF/CFHE \*\*\*\*\*

\*\*\*\*\* PF/CFHE \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASAS,PF/CFHE,ME\*NASAS,PF/CFHE  
FOR S9A-07/13/72-20:57:16 (0,)

FUNCTION PF ENTRY POINT 000047

STORAGE USED: CODE(1) 000052; DATA(0) 000026; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000005 A	0000 R 000012 ALPHA	0000 R 000002 A1	0000 R 000003 B1	0000 R 000004 C
0000 000017 INJPS	0000 R 000000 PF	0000 R 000001 R	0000 R 000006 RHOX	0000 R 000007 TX
0000 R 000010 V	0000 R 000011 VM2			

```
00101      1*      FUNCTION PF(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      PRESSURE AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND
00101      5*      C      TEMPERATURE (R) OF HELIUM
00101      6*      C      UNITS LBF/SQ.FT
00101      7*      C
00103      8*      DATA R,A1,B1,C,A /2077.02,136.9595,3.5002295E-03,10.000658,1.49610
00103      9*      13E-02/
00111     10*      RHOX = RHO*515.4275
00112     11*      TX   = T/1.8
00113     12*      V    = 1.0/RHOX
00114     13*      VM2  = RHOX*RHOX
00115     14*      ALPHA = C*RHOX/(TX**3)
00116     15*      PF   = (R*TX*(1.0-ALPHA)*(V+B1)-A1*(1.0-A*RHOX))*VM2*2.08846E-02
00117     16*      RETURN
00120     17*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* PF/CFNAK \*\*\*\*\*

\*\*\*\*\* PF/CFNAK \*\*\*\*\*

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QFOR:5 ME\*NASA5.PF/CFNAK,ME\*NASA5.PF/CFNAK  
FOR S9A-07/13/72-20:57:18 (0,)

FUNCTION PF ENTRY POINT 000030

STORAGE USED: CODE(1) 000040; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 CAPP  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000004 C	0003 R 000000 CAPP	0000 000010 INJP5	0000 R 000000 PF	0000 R 000003 P0
0000 R 000005 TK	0000 R 000001 X1	0000 R 000002 X2		

```
00101 1* FUNCTION PF(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C PRESSURE AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND
00101 5* C TEMPERATURE (R) OF NAK 78.6
00101 6* C UNITS LBF/SQ.FT
00101 7* C
00103 8* DATA X1,X2 /58.773064,-0.008433/,P0 /2116.224/
00107 9* C = (X1+X2*T)/32.174
00110 10* TK = CAPP(RHO,T)
00111 11* PF = (RHO/C-1.0)/TK+P0
00112 12* RETURN
00113 13* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* PF/CFSIL \*\*\*\*\*

\*\*\*\*\* PF/CFSIL \*\*\*\*\*

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QFOR S ME NASAS.PF/CFSIL ME NASAS.PF/CFSIL  
FOR S9A-07/13/72-20:57:20 (0,)

FUNCTION PF ENTRY POINT 000155

STORAGE USED: CODE(1) 000162; DATA(0) 000072; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 ALOG  
0004 SQRT  
0005 NWDJ5  
0006 NIO25  
0007 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000030	1F	0000	R	000022	A	0000	R	000001	A1	0000	R	000002	A2	0000	R	000003	A3	
0000	R	000004	A4	0000	R	000005	A5	0000	R	000023	B	0000	R	000006	B1	0000	R	000007	B2
0000	R	000010	B3	0000	R	000011	B4	0000	R	000012	B5	0000	R	000024	C	0000	R	000013	C1
0000	R	000014	C2	0000	R	000017	DT	0000		000061	INJPS	0000	R	000027	P	0000	R	000000	PF
0000	R	000025	RH01	0000	R	000020	THETA	0000	R	000021	THETA1	0000	R	000015	T0	0000	R	000016	T01
0000	R	000026	Z																

```

00101      1*      FUNCTION PF(RH0,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      PRESSURE AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND TEMPERATURE
00101      5*      C      (R) OF DOW CORNING 200 SILICON OIL (1 CS)
00101      6*      C      UNITS LBF/SQ.FT
00101      7*      C      TEMPERATURE .GE. 359.67 AND .LE. 859.67
00101      8*      C      TEMP. GE. 360.67 AND LE. 860.67
00103      9*      DATA A1,A2,A3,A4,A5 /12.35,2.98333,1.1,-0.48333,0.1/,B1,B2,B3,B4,
00103     10*      185 /-1.5,-0.01333,-1.18,0.57333,-0.1/,C1,C2 /0.7767,-0.0288/,
00103     11*      2T0,T01,DT /559.67,609.67,50.0/
00123     12*      THETA = (T-T0)/DT
00124     13*      THETA1 = (T-T01)/DT
00125     14*      A = (((A5*THETA+A4)*THETA+A3)*THETA+A2)*THETA+A1)
00126     15*      B = (((B5*THETA+B4)*THETA+B3)*THETA+B2)*THETA+B1)
00127     16*      C = C1+C2*THETA1
00130     17*      RH01 = RH0*32.174/62.42
00131     18*      Z = A*A+2.0*B*ALOG(RH01/C)*1000.0
00132     19*      P = (-A+SQRT(Z))*1000.0/B
00133     20*      PF = (P+14.696)*144.0
00134     21*      IF (T.LT.360.67.OR.T.GT.860.67.OR.PF.GT.146116.224.OR.PF.GT.110116
00134     22*      1.224.AND.T.LT.460.67) WRITE(6,1) T,PF
00141     23*      1 FORMAT (1H0,43HPRESSURE OF SILICON OIL, OUT OF RANGE, T = ,F10.5,
00141     24*      16H, P = ,F15.5,/)
00142     25*      RETURN
00143     26*      END

```

\*\*\*\*\* PF/CFSIL \*\*\*\*\*

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END OF COMPILATION: NO DIAGNOSTICS.

BHDG.P \*\*\*\*\* POLY \*\*\*\*\*

\*\*\*\*\* POLY \*\*\*\*\*

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1

QFOR S ME\*NASA5.POLY,ME\*NASA5.POLY  
FOR S9A-07/13/72-20:57:23 (0,)

FUNCTION POLY ENTRY POINT 000036

STORAGE USED: CODE(1) 000044; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000012 1076 0000 000003 INJPS 0000 I 000002 K 0000 I 000001 L 0000 R 000000 POLY

```
00101      1*      FUNCTION POLY(N,A,X)
00101      2*      C
00101      3*      C THIS SUBROUTINE EVALUATES POLYNOMIAL
00101      4*      C POLY = A(1)+A(2)*X+A(3)*X**2+.....+A(N)*X**(N-1)
00101      5*      C
00103      6*      DIMENSION A(N).
00104      7*      POLY = 0.
00105      8*      L = N
00106      9*      DO 1 K=1,N
00111     10*      POLY = POLY*X+A(L)
00112     11*      1 L = L-1
00114     12*      RETURN
00115     13*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHD6:P \*\*\*\*\* QINCID \*\*\*\*\*



\*\*\*\*\* QINCID \*\*\*\*\*

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QFOR S ME NASAS QINCID ME NASAS QINCID  
FOR 59A-07/13/72-20:57:27 (0.)

SUBROUTINE QINCID ENTRY POINT 001001

STORAGE USED: CODE(1) 001027; DATA(0) 000665; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 QIN 001610  
0004 TC 000042

EXTERNAL REFERENCES (BLOCK, NAME)

0005 YINT  
0006 SHAJE  
0007 DEFINT  
0010 NRBS  
0011 NIO2\$  
0012 NBS\$  
0013 NIO1\$  
0014 NREWS  
0015 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000036 10L	0001 000036 1236	0001 000064 1426	0001 000124 1606	0001 000132 1656
0001 000151 1736	0001 000057 20L	0001 000166 2036	0001 000073 22L	0001 000273 2256
0001 000337 2403	0001 000354 2476	0001 000404 2636	0001 000230 30L	0001 000547 3016
0001 000255 32L	0001 000730 3246	0001 000741 3306	0001 000313 36L	0001 000325 38L
0001 000507 58L	0001 000633 70L	0001 000643 80L	0004 000003 ABEMF	0000 R 000000 ANG
0007 R 000000 DEFINT	0000 I 000605 I	0004 I 000001 ICASE	0000 I 000603 IFILE	0000 000633 INJP\$
0000 I 000604 IP	0004 I 000000 ITAPE	0000 I 000617 J	0000 I 000610 K	0000 I 000614 K0
0000 I 000615 K1	0000 I 000612 M1	0000 I 000613 M2	0000 I 000611 NP	0000 I 000606 NS
0003 I 001605 NSRD	0003 I 001604 NTM	0000 I 000607 N1	0000 R 000616 PANG	0004 000004 PHIN
0000 R 000343 QA	0000 R 000077 QALB	0003 R 000454 QIFN	0003 R 001274 QITB	0000 R 000463 QP
0000 R 000151 QPLN	0000 R 000223 QS	0003 R 000144 QSFN	0003 R 000764 QSTB	0000 R 000025 QSUN
0003 001606 QTO	0003 R 000000 TM	0004 R 000002 TO	0003 001607 TX	0005 R 000000 YINT

00101 1\* SUBROUTINE QINCID(MSTOTR,PHN)  
00101 2\* C  
00101 3\* C THIS SUBROUTINE PROVIDES :  
00101 4\* C 1- DATA TRANSFER FROM THE SRI INCIDENT RADIATION COMPUTER PROGRAM  
00101 5\* C 2- AVERAGING OF RADIANT FLUX OVER CIRCUMFERENCE OF TUBE CROSS-  
00101 6\* C SECTION  
00101 7\* C  
00103 8\* C PARAMETER NTP = 10  
00104 9\* C PARAMETER MAXSID = 20  
00105 10\* C PARAMETER MAXSD1 = MAXSID+1  
00106 11\* C COMMON /QIN/ TM(100),QSFN(100,2),QIFN(100,2),QSTB(100,2),

\*\*\*\*\* GINCID \*\*\*\*\*

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00106 12* 1 QITB(100,2),NTM,NSRD,QTO,TX
00107 13* COMMON /TC/ ITAPE,ICASE,TO,ABEMF,PHIN(NTP,3)
00110 14* DIMENSION ANG(MAXSD1),QSUN(MAXSD1,2),QALB(MAXSD1,2),
00110 15* 1 QPLN(MAXSD1,2),QS(4,MAXSID),QA(4,MAXSID),QP(4,MAXSID)
00111 16* IFILE = ITAPE
00112 17* IF (ITAPE.LE.0) IFILE = NSRD+2
00114 18* IP = 1
00115 19* IF (ABS(PHN).GT.0.0) IP = 2
00117 20* IF (ICASE.LE.1) GO TO 20
00121 21* TM(1) = -(10.0**30)
00122 22* DQ 16 I = 2,ICASE
00125 23* 10 TM(2) = TM(1)
00126 24* READ (IFILE) NS,TM(1)
00132 25* IF (TM(1).GT.TM(2)) GO TO 10
00134 26* 16 CONTINUE
00136 27* BACKSPACE IFILE
00137 28* 20 IF (MSTOTR.EQ.2) GO TO 30
00141 29* DO 21 I = 1,2
00144 30* 21 READ (IFILE)
00147 31* N1 = 2
00150 32* 22 READ (IFILE) NS,TM(1)
00154 33* N1 = N1+1
00155 34* IF (TM(1).LT.TO.AND.N1.LT.NTM-1) GO TO 22
00157 35* DO 24 I = 1,3
00162 36* 24 BACKSPACE IFILE
00164 37* DO 26 I = 1,4
00167 38* 26 READ (IFILE) NS,TM(I),(QS(I,K),QA(I,K),QP(I,K),K=1,NS)
00202 39* DO 28 K = 1,NS
00205 40* QSUN(K,1) = YINT(TM,QS(1,K),4,4,TO)
00206 41* QALB(K,1) = YINT(TM,QA(1,K),4,4,TO)
00207 42* 28 QPLN(K,1) = YINT(TM,QP(1,K),4,4,TO)
00211 43* TM(1) = 0.0
00212 44* NTM = 1
00213 45* 30 IF (ICASE.GT.1.OR.MSTOTR.EQ.1) GO TO 32
00215 46* READ (IFILE) NS
00220 47* BACKSPACE IFILE
00221 48* 32 NP = NS/2
00222 49* M1 = (NP-1)/2
00223 50* M2 = NP/2
00224 51* DO 34 I = 0,NS
00227 52* 34 ANG(I+1) = FLOAT(I*360+180)/NS
00231 53* IF (IP.EQ.1) GO TO 38
00233 54* K0 = 0
00234 55* 36 K0 = K0+1
00235 56* IF (ANG(K0)+ANG(1).LE.PHN) GO TO 36
00237 57* 38 DO 90 I = 1,NTM
00242 58* IF (MSTOTR.EQ.2) READ (IFILE) NS,TM(I),(QSUN(K,1),QALB(K,1),
00242 59* 1 QPLN(K,1),K=1,NS)
00255 60* IF (IP.EQ.1) GO TO 58
00257 61* QSUN(NS+1,1) = QSUN(1,1)
00260 62* QALB(NS+1,1) = QALB(1,1)
00261 63* QPLN(NS+1,1) = QPLN(1,1)
00262 64* DO 54 K = 1,NS
00265 65* K1 = MIN(MAX(MOD(K+K0-2,NS),1),NS-2)
00266 66* PANG = ANG(K)+PHN
00267 67* IF (PANG.GE.ANG(NS+1)) PANG = PANG-360.0
00271 68* QSUN(K,2) = YINT(ANG(K1),QSUN(K1,1),4,4,PANG)

```

\*\*\*\*\* QINCIO \*\*\*\*\*

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00272 69*      QALB(K,2) = YINT(ANG(K1),QALB(K1,1),4,4,PANG)
00273 70*      QPLN(K,2) = YINT(ANG(K1),QPLN(K1,1),4,4,PANG)
00275 71*      54      CALL SHADE(QSUN(1,IP),NS)
00276 72*      58      CALL SHADE(QALB(1,IP),NS)
00277 73*      CALL SHADE(QPLN(1,IP),NS)
00300 74*      DO 90 J = 1,NSRD
00303 75*      N1 = NP*(J-1)+1
00304 76*      IF (M1.EQ.M2) GO TO 70
00306 77*      QSFN(I,J) = YINT(ANG(M1),QSUN(N1+M1-1,IP),4,4,90.0)
00307 78*      QIFN(I,J) = YINT(ANG(M1),QALB(N1+M1-1,IP),4,4,90.0)+
00307 79*      1      YINT(ANG(M1),QPLN(N1+M1-1,IP),4,4,90.0)
00310 80*      GO TO 80
00311 81*      70      QSFN(I,J) = QSUN(N1+M1,IP)
00312 82*      QIFN(I,J) = QALB(N1+M1,IP)+QPLN(N1+M1,IP)
00313 83*      80      QSTB(I,J) = DEFIN(T(QSUN(N1,IP),0.5,NP)
00314 84*      90      QITB(I,J) = DEFIN(T(QALB(N1,IP),0.5,NP)+
00314 85*      1      DEFIN(T(QPLN(N1,IP),0.5,NP)
00317 86*      REWIND IFILE
00320 87*      IF (MSTOTR.EQ.2) RETURN
00322 88*      NTM = 4
00323 89*      DO 100 I = 2,4
00326 90*      TM(I) = 20*(I-1)
00327 91*      DO 100 J = 1,NSRD
00332 92*      QSFN(I,J) = QSFN(1,J)
00333 93*      QIFN(I,J) = QIFN(1,J)
00334 94*      QSTB(I,J) = QSTB(1,J)
00335 95*      100     QITB(I,J) = QITB(1,J)
00340 96*      RETURN
00341 97*      END

```

END OF COMPILATION:

NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* GRAD \*\*\*\*\*

\*\*\*\*\* GRAD \*\*\*\*\*

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QFOR:5 ME\*NASA5.GRAD:ME\*NASA5.GRAD  
FOR 59A-07/13/72-20:57:31 (0.)

SUBROUTINE GRAD ENTRY POINT 000772

STORAGE USED: CODE(1) 001013; DATA(0) 001077; BLANK COMMON(2) 001115

COMMON BLOCKS:

0003 GRD 003721  
0004 QIN 001610  
0005 ABSRST 000601  
0006 SSF 000010  
0007 AVGABS 000251

EXTERNAL REFERENCES (BLOCK, NAME)

0010 INTERP  
0011 YINT  
0012 EMIT  
0013 AVGEAT  
0014 ABSORB  
0015 TRMATX  
0016 MTXINV  
0017 EXITAV  
0020 DEFNT  
0021 NERR2\$  
0022 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000245 100L	0001 000010 122G	0001 000012 126G	0001 000046 135G	0001 000055 142G
0001 000073 147G	0001 000123 156G	0001 000112 2L	0001 000304 206G	0001 000306 211G
0001 000346 225G	0001 000362 235G	0001 000366 241G	0001 000422 252G	0001 000434 256G
0001 000443 261G	0001 000524 273G	0001 000536 277G	0001 000632 313G	0001 000633 316G
0001 000645 324G	0001 000702 337G	0001 000733 346G	0001 000735 352G	0001 000332 500L
0001 000344 550L	0001 000360 600L	0001 000407 700L	0001 000641 810L	0001 000673 820L
0001 000717 910L	0006 000002 ALPFN	0005 000000 ALPHFN	0005 000226 ALPHMP	0002 000562 AUX2
0000 R 000163 AX1	0000 R 000327 AX2	0000 R 000473 AX3	0000 R 000637 AX4	0002 000752 COB
0002 001107 COH	0002 000574 CONFN	0002 000740 CONMP	0002 001070 CP0	0003 R 003465 CSF
0020 R 000000 DEFNT	0002 001072 DELTA	0002 000764 DOB	0002 001110 DXI	0002 001053 DXIMP
0002 001052 DXITB	0003 R 003464 DXX2	0003 003472 DXX21	0002 001051 DZ	0003 003463 DZMFN
0003 R 003410 EBFN	0003 R 003441 EB4P	0012 R 000000 EMIT	0005 000461 EMITFN	0005 000512 EMITMP
0003 R 003467 EXTIFN	0003 R 003471 EXTIMP	0003 R 003466 EXT5FN	0003 R 003470 EXT5MP	0005 R 000517 EXTVCY
0002 001103 FENTH	0002 001045 FFO	0002 001101 FLUX	0002 001041 FMACH	0002 001040 FNU
0002 001046 FOF	0002 000012 FP	0002 001042 FPR	0002 001037 FR	0002 001050 FRAD
0002 001111 FRD	0002 001036 FRE	0002 001104 FREJ	0002 001043 FRL	0002 001044 FRM
0002 000036 FT	0002 000024 FW	0002 001113 FXHW	0002 001112 FXOH	0002 001047 FZ
0002 001105 HFN	0000 I 001006 I	0002 001003 IFLOW	0000 I 001007 III	0000 I 001024 IJK
0000 I 001022 IL	0000 001046 INJPS	0000 I 001005 J	0000 I 001023 JL	0000 I 001016 K
0000 I 001025 KJL	0000 I 001021 KL	0002 001017 L	0002 001024 LC	0003 I 003461 LCT
0002 001032 LIM	0002 001031 LIMWRT	0002 001011 LL1	0002 001012 LL2	0002 001013 LL3
0002 001014 LL4	0002 001015 LL5	0002 001016 LL6	0002 001017 LL7	0002 001006 LMP

\*\*\*\*\* GRAD \*\*\*\*\*

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0002	001004	LT	0002	001005	LTB	0002	001007	LTMZ	0002	001010	LTB2MZ	0003	I	003462	LTT													
0003	I	003460	MCVRD	0002	001030	MM1	0002	001033	MOD	0002	001025	MSTOTR	0002	I	000777	MZ												
0002	001034	NCCZ	0002	001035	NCONV	0002	001002	NCTL	0002	001026	NCTM	0002		001023	NEQUS													
0002	001027	NM1	0002	I	001001	NRMP	0002	001020	NRMP1	0002	001022	NRMP2	0002		001000	NRTB												
0002	001021	NRTB1	0004	I	001605	NSRD	0004	I	001604	NTM	0002	I	000776	NX	0002	001063	PHIF											
0002	001062	PHIM	0002	001071	PI	0002	001065	PO	0000	R	000017	QFN	0004	R	000454	QIFN												
0004	R	001274	QITB	0000	R	000012	QMP	0002	001102	QREF	0002	R	000360	GRFN	0002	R	000524	GRMP										
0004	R	000144	QSFN	0004	R	000764	QSTB	0004	R	001606	QTO	0002		001074	ROTWRT	0002		001114	RHOFN									
0002	001067	RH00	0002	001073	RLIMIT	0002	001100	RTEND	0006		000003	R1	0006		000000	R2	0006		000000	R2								
0003	R	003473	SS	0003	R	003670	SSTT	0006		000001	T	0000	R	001004	TALB	0002	R	000132	TEMP	0002	R	000276	TMP					
0007	R	000100	TFIN	0002	001077	TI	0002	001076	TINTL	0003	R	000000	TRMTX	0006		000006	WWA	0002	R	000536	XIFN	0006	R	000004	XSPM			
0000	R	000000	TMPP	0007	R	000244	TPR	0002	001075	TREF	0007		000062	XXFN	0002	001061	XX12	0003	R	003446	XX2	0002	R	000550	ZETA			
0002	000050	TW	0004	R	001607	TX	0002	R	001064	TO	0000	R	001013	XCMP	0006		000005	XSMF	0007		000000	XXMP	0002		001061	XX12		
0002	001066	WO	0000	R	001012	XCFN	0002	R	001054	XRE	0007	R	000062	XXFN	0002	001061	XX12	0000	R	001020	Z	0003	R	003446	XX2			
0002	000000	XITB	0002	001106	XL	0000	R	001015	XXCMP	0002	001057	XX10	0002	001060	XX11	0011	R	000000	YINT	0003	R	003446	XX2	0002	R	000550	ZETA	
0000	R	001014	XXCFN	0000	R	001015	XXCMP	0007	R	000062	XXFN	0002	001060	XX11	0011	R	000000	YINT	0003	R	003446	XX2	0002	R	000550	ZETA		
0007	000031	XXMP	0002	001057	XX10	0002	001057	XX10	0002	001060	XX11	0011	R	000000	YINT	0003	R	003446	XX2	0002	R	000550	ZETA	0002	R	000550	ZETA	
0002	001055	XX3	0002	001056	XX4	0002	001056	XX4	0011	R	000000	YINT	0003	R	003446	XX2	0002	R	000550	ZETA	0002	R	000550	ZETA	0002	R	000550	ZETA
0000	R	001010	ZFN	0000	R	001011	ZMP	0003	R	003453	ZZ2																	

```

00101 1*      SUBROUTINE GRAD
00101 2*      C
00101 3*      C THIS SUBROUTINE COMPUTES :
00101 4*      C 1- INVERSION OF TRANSFER MATRIX
00101 5*      C 2- SOLUTION OF RADIOSITY EQUATION
00101 6*      C 3- GRID MAPPING
00101 7*      C
00101 8*      C
00101 9*      C NET RADIANT HEAT LOSS, AVERAGED OVER REPRESENTATIVE AREA ELEMENT
00101 10*     C (OVER BOTH SIDES OF THE FIN, BOTH HALVES OF THE TUBE)
00101 11*     C
00103 12*     COMMON XITB(5),XIMP(5),FP(10),FW(10),FT(10),TW(10,5),TEMP(10,10),
00103 13*     1 TMP(10,5),ORFN(10,10),GRMP(10),XIFN(10),ZETA(10),AUX2(10),
00103 14*     2 CONFN(10,10),CONMP(10),COB(10),DOB(10)
00104 15*     COMMON NX,MZ,NRTB,NRMP,NCTL,IFLOW,LT,LTB,LMP,LBMZ,LTB2MZ,LL1,
00104 16*     1 LL2,LL3,LL4,LL5,LL6,LL7,NRMP1,NRTB1,NRMP2,NEQUS,LC,MSTOTR,
00104 17*     2 NCTM,NM1,M1,LIMWRT,LIM,MOD,NCCZ,NCONV
00105 18*     COMMON FRE,FR,FNU,FMACH,FPR,FR,FRM,FFO,FOF,FZ,FRAD,DZ,DXITB,DXIMP
00105 19*     1 ,XRE,XX3,XX4,XX10,XX11,XX12,PHIM,PHIF,T0,P0,W0,RH00,CP0,PI
00105 20*     2 ,DELTA,RLIMIT,ROTWRT,TREF,TINTL,TI,RTEND,FLUX,QREF,FENTH,
00105 21*     3 FREJ,HFN,XL,COH,DXI,FRD,FXOH,FXHW,RHOFN
00106 22*     COMMON /GRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZ2(5),
00106 23*     1 MCVRD,LCT,LT,DTZFN,
00106 24*     2 DXX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DXX21,SS(5,5,5)
00106 25*     3 ,SSTT(5,5)
00107 26*     COMMON /QIN/TM(100), QSFN(100,2), QIFN(100,2), QSTB(100,2),
00107 27*     1 QITB(100,2), NTM, NSRD, QTO, TX
00110 28*     COMMON /ABSRST/ALPHFN(5,5,6),ALPHMP(5,31),EMITFN(5,5),EMITMP(5),
00110 29*     1 EXTVCT(50)
00111 30*     COMMON /SSF/R2, T, ALPHN,R1,XSPM,XSMP,WWA,WWB
00112 31*     COMMON/AVGABS/XXMP(5,5),XXXMP(5,5),XXFN(7),XXXFN(7),
00112 32*     1 TFIN(10,10),TPR(5)
00113 33*     DIMENSION TMPP(10),GMP(5),QFN(10,10),
00113 34*     1 AX1(100),AX2(100),AX3(100),AX4(100)

```

```

00114 35* DATA TSOL,TALB/10400.0,480.0/
00114 36* C
00117 37* IF(LTT.GT.1) GO TO 910
00121 38* DO 1 J = 1,5
00124 39* QMP(J) = 0.0
00125 40* DO 1 I = 1,5
00130 41* 1 QFN(J,I) = 0.0
00130 42* C
00133 43* CALL INTERP(NX,MZ,XIFN,ZETA,TEMP,5,5,XX2,ZZ2,TFIN)
00133 44* C
00134 45* DO 5 J = 1,MZ
00137 46* 5 TMPP(J) = TMP(J,NRMP)
00141 47* DO 10 J = 1,5
00144 48* TPR(J) = YINT(ZETA,TMPP,MZ,3,ZZ2(J))
00145 49* EBMP(J) = TPR(J)**4
00146 50* DO 10 I = 1,5
00151 51* 10 EBFN(J,I) = TFIN(J,I)**4
00151 52* C
00154 53* III = 1
00155 54* 2 DO 3 J = 1,NTM
00160 55* AX1(J) = QIFN(J,III)
00161 56* AX2(J) = QSFN(J,III)
00162 57* AX3(J) = QITB(J,III)
00163 58* 3 AX4(J) = QSTB(J,III)
00165 59* EXTIFN = YINT(TM,AX1,NTM,3,TX)/QTO
00166 60* EXTFSN = YINT(TM,AX2,NTM,3,TX)/QTO
00167 61* EXTIMP = YINT(TM,AX3,NTM,3,TX)/QTO
00170 62* EXTSMF = YINT(TM,AX4,NTM,3,TX)/QTO
00171 63* ZFN = EXTFSN+EXTIFN
00172 64* ZMP = EXTSMF+EXTIMP
00173 65* XCFN = (EXTFSN*EMIT(TSOL)+EXTIFN*EMIT(TALB))/ZFN
00174 66* XCMP = (EXTSMF*EMIT(TSOL)+EXTIMP*EMIT(TALB))/ZMP
00174 67* C
00175 68* GO TO (100,500),LCT
00175 69* C
00176 70* 100 CALL AVGEMT(TO)
00177 71* XXCFN = (EXTFSN*XXFN(7)+EXTIFN*XXFN(6))/ZFN
00200 72* XXCMP = (EXTSMF*XXFN(7)+EXTIMP*XXFN(6))/ZMP
00201 73* CALL ABSORB(TO,XCFN,XCMP,XXCFN,XXCMP)
00202 74* CALL TRMATX
00203 75* IF (LTT.NE.0) GO TO 500
00205 76* DO 115 I = 1,30
00210 77* DO 110 J = 31,60
00213 78* 110 TRMTX(I,J) = 0.0
00215 79* K = I+30
00216 80* 115 TRMTX(I,K) = 1.0
00220 81* CALL MTXINV(30,60)
00221 82* LCT = 2
00221 83* C
00222 84* 500 CALL EXITAV
00223 85* GO TO (550,600),LCT
00223 86* C
00224 87* 550 DO 555 I = 1,30
00227 88* 555 TRMTX(I,31) = EXTVCT(I)
00231 89* CALL MTXINV(30,31)
00232 90* L = 31
00233 91* GO TO 700

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00233  92*  C
00234  93*  600 DO 610  I = 1,30
00237  94*      Z      = 0.0
00240  95*      DO 605  J = 1,30
00243  96*      K      = J+30
00244  97*      605 Z      = Z+TRMTX(I,K)*EXTVCT(J)
00246  98*      610 TRMTX(I,1) = Z
00250  99*      L      = 1
00250 100*  C
00250 101*  C
00251 102*  700 DO 730  J=1,5
00254 103*      KL      = J+25
00255 104*      DO 720  IL=1,5
00260 105*      DO 715  JL=1,5
00263 106*      K      = (IL-1)*5+JL
00264 107*      715 AX1(JL) = CSF*SS(IL,JL,J)*TRMTX(K,L)
00266 108*      AX3(IL) = SST(J,IL)*TRMTX(IL+25,L)
00267 109*      720 AX2(IL) = DEFNT(AX1,0.25,5)
00271 110*      QMP(J)  = -(ZMP+DEFNT(AX2,DX2,5)+AX2(1)*DX2/2.0+DEFNT(AX3,
00271 111*      1      0.25,5)-TRMTX(KL,L))/2.0+QMP(J)
00272 112*      DO 730  I=1,5
00275 113*      IJK      = (I-1)*5+J
00276 114*      DO 725  JL=1,5
00301 115*      KJL      = 25+JL
00302 116*      725 AX1(JL) = SS(I,J,JL)*TRMTX(KJL,L)
00304 117*      730 QFN(J,I) = -(ZFN+DEFNT(AX1,0.25,5)-TRMTX(IJK,L))+QFN(J,I)
00304 118*  C
00304 119*  C
00307 120*      CALL INTERP(5,5,XX2,ZZ2,QFN,NX,MZ,XIFN,ZETA,QRFN)
00310 121*      IF(MCVRD.LE.0) GO TO 810
00310 122*  C
00312 123*      DO 800  I = 1,MCVRD
00315 124*      DO 800  J = 1,MZ
00320 125*      800 QRFN(J,I) = 0.0
00320 126*  C
00323 127*      810 DO 815  J = 1,MZ
00326 128*      815 GRMP(J) = YINT(ZZ2,QMP,5,3,ZETA(J))
00330 129*      IF (III.EQ.2) GO TO 820
00332 130*      IF (NSRD.EQ.1) GO TO 820
00334 131*      III      = 2
00335 132*      GO TO 2
00335 133*  C
00336 134*      820 DO 825  J = 1,MZ
00341 135*      K      = MCVRD+1
00342 136*      825 QRFN(J,K) = QRFN(J,K)*XSPM
00344 137*      RETURN
00345 138*      910 DO 915  J=1,MZ
00350 139*      GRMP(J)  = 0.0
00351 140*      DO 915  I=1,NX
00354 141*      915 QRFN(J,I) = 0.0
00357 142*      RETURN
00360 143*      END

```

END OF COMPILATION:

NO DIAGNOSTICS.

\*\*\*\*\* GRAD \*\*\*\*\*

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QHDG:P \*\*\*\*\* REFP \*\*\*\*\*



\*\*\*\*\* REFP \*\*\*\*\*

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QFOR,S ME\*NASAS,REFP,ME\*NASAS,REFP  
FOR S9A-07/13/72-20:57:38 (0.)

SUBROUTINE REFP ENTRY POINT 000316

STORAGE USED: CODE(1) 000367; DATA(0) 000163; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 CPAIR  
0004 POLY  
0005 NEXP65  
0006 EXP  
0007 NERRJ5

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000076 2L	0001 000214 20L	0001 000065 2236	0001 000154 3L	0001 000172 4L
0000 R 000066 A0	0000 R 000067 A1	0000 R 000100 A10	0000 R 000101 A11	0000 R 000070 A2
0000 R 000071 A3	0000 R 000072 A4	0000 R 000073 A5	0000 R 000074 A6	0000 R 000075 A7
0000 R 000076 A8	0000 R 000077 A9	0000 R 000064 CF1	0000 R 000065 CF2	0003 R 000000 CPAIR
0000 R 000056 CP1	0000 R 000057 CP2	0000 R 000060 CP3	0000 R 000061 CP4	0000 R 000110 DH
0000 R 000063 DT	0000 R 000104 ELEVM	0000 R 000106 GEOM	0000 R 000105 GEOHM	0000 R 000007 GH
0000 R 000031 GT	0000 R 000052 G0	0000 R 000000 H	0000 I 000107 I	0000 000144 INJP5
0000 I 000111 J	0000 R 000102 MWT	0000 R 000041 P	0000 R 000113 PATM	0004 R 000000 POLY
0000 R 000116 PSL	0000 R 000115 PZ	0000 R 000054 RU	0000 R 000055 RU1	0000 R 000020 T
0000 R 000112 TMB	0000 R 000103 TMK	0000 R 000062 T0	0000 R 000053 WMO	0000 R 000114 Z

```

00101 1* SUBROUTINE REFP (ELEV,REFTP,REFPR,REFVIS,REFRHO,REFK,REFCP,
00101 2* 1 REFGAM)
00101 3* C
00101 4* C THIS SUBROUTINE COMPUTES :
00101 5* C PROPERTIES OF ATMOSPHERIC AIR EVALUATED AT THE HIGH SPEED
00101 6* C REFERENCE TEMPERATURE
00101 7* C
00101 8* C ELEVATION IN FEET
00101 9* C REFERENCE TEMP. IN DEGREES R.
00101 10* C PRANDTL NUMBER DIMENSIONLESS
00101 11* C DYN. VISCOSITY IN LBM/ FT.SEC.
00101 12* C DENSITY IN LBM/CUBIC FOOT
00101 13* C THERMAL COND. IN BTU/SEC FT DEG R
00101 14* C SPECIFIC HEAT IN BTU/LBM DEG R
00103 15* DIMENSION H(7),GH(9),T(9),GT(8),P(9)
00104 16* DATA H(1),H(2),H(3),H(4),H(5),H(6),H(7) /0.0,1.0,-1.5731262E-07,2.
00104 17* 14656553E-14,-3.8667054E-21,6.0621354E-28,-9.5013649E-35/,GH(1),GH(
00104 18* 22),GH(3),GH(4),GH(5),GH(6),GH(7),GH(8),GH(9) /0.0,11.0,20.0,32.0,4
00104 19* 37.0,52.0,61.0,79.0,90.0/,T(1),T(2),T(3),T(4),T(5),T(6),T(7),T(8),T
00104 20* 4(9) /288.15,216.65,216.65,228.65,270.65,270.65,252.65,180.65,180.6
00104 21* 55/,GT(1),GT(2),GT(3),GT(4),GT(5),GT(6),GT(7),GT(8) /-6.5,0.0,1.0,2
00104 22* 6.8,0.0,-2.0,-4.0,0.0/,P(1),P(2),P(3),P(4),P(5),P(6),P(7),P(8),P(9)

```

\*\*\*\*\* REFP \*\*\*\*\*

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00104 23* 7 /2116.22,472.681,114.345,18.1298,2.31631,1.23225,0.380323,0.02167
00104 24* 82,0.0027138/,60,WMO,RU,RU1 /9.80665,28.9644,8314.32,1545.31/,CP1,C
00104 25* 9P2,CP3,CP4 /0.239573,-0.000127,0.000051,0.000014/,TO,OT /270.0,90.
00104 26* 10/,CF1,CF2 /1000.0,0.671969/
00173 27* DATA A0,A1,A2,A3,A4,A5,A6,A7,A8,A9,A10,A11/
00173 28* 1 0.10E+01, 0.3533367370E-01, -0.7474788290E-03,
00173 29* 2 -0.2121572232E-03, -0.1325255219E-04, 0.5344159692E-06,
00173 30* 3 -0.1322745646E-07, 0.1965359762E-09, -0.1723714966E-11,
00173 31* 4 0.8707590786E-14, -0.2341816445E-16, 0.2597772972E-19/,
00210 32* REAL MWT
00211 33* TMK = REFTP/1.8
00212 34* REFFVIS = ((1.458E-06*TMK**1.5)/(TMK+110.4))*CF2
00213 35* REFK = ((6.325E-07*TMK**1.5)/(TMK+245.4*10.0**(-12.0/TMK)))*CF2
00214 36* REFCP = CPAIR(REFTP)
00215 37* IF (ELEV.GT. 301000.) GO TO 20
00217 38* ELEVH = ELEV*0.3048
00220 39* GEOHM = POLY(7,H,ELEVH)
00221 40* GEOH = GEOHM/0.3048
00222 41* DO 1 I=1,9
00225 42* DH = GH(I)-GEOHM/CF1
00226 43* IF (DH.GT.0.0) GO TO 2
00230 44* 1 CONTINUE
00232 45* 2 J = I-1
00233 46* DH = GH(J)-GEOHM/CF1
00234 47* TMB = T(J)-GT(J)*DH
00235 48* IF (J.EQ.2.OR.J.EQ.5.OR.J.EQ.8) GO TO 3
00237 49* PATM = P(J)*(T(J)/TMB)**(60*WMO*CF1/(RU*GT(J)))
00240 50* GO TO 4
00241 51* 3 PATM = P(J)*EXP(60*WMO*DH*CF1/(RU*T(J)))
00242 52* 4 REFRHO = WMO*PATM/(RU1*REFTP)
00243 53* REFRP = REFFVIS*REFCP/REFK
00244 54* REFGAM = REFCP/(REFCP-0.0686)
00245 55* RETURN
00246 56* 20 Z = ELEV/(3280.8399)
00247 57* MWT = 28.9644-0.0309491*(Z-90.0)
00250 58* PZ = 1.0/((A0+Z*(A1+Z*(A2+Z*(A3+Z*(A4+Z*(A5+Z*(A6+Z*(A7+Z
00250 59* 1 *(A8+Z*(A9+Z*(A10+Z*A11))))))))**4)
00251 60* PSL = 2116.22657
00252 61* PATM = PSL*PZ
00253 62* REFRHO = PATM*MWT/(1545.31*REFTP)
00254 63* REFRP = REFFVIS*REFCP/REFK
00255 64* REFGAM = REFCP/(REFCP-(1.98585/MWT))
00256 65* RETURN
00257 66* END

```

END OF COMPILATION:

NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* RHOF/CFFC43 \*\*\*\*\*

\*\*\*\*\* RHOF/CFFC43 \*\*\*\*\*

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QFOR,S ME\*NASAS.RHOF/CFFC43,ME\*NASAS.RHOF/CFFC43  
FOR S9A-07/13/72-20:57:42 (0.)

FUNCTION RHOF ENTRY POINT 000014

STORAGE USED: CODE(1) 000016; DATA(0) 000011; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000004 INJP% 0000 R 000000 RHOF 0000 R 000001 X1 0000 R 000002 X2

```
00101      1*      FUNCTION RHOF(P,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      DENSITY AS A FUNCTION OF PRESSURE (LBF/SQ.FT) AND
00101      5*      C      TEMPERATURE (R) OF FC-43
00101      6*      C      UNITS SLUG/QU.FT
00101      7*      C
00103      8*      DATA X1,X2 /157.0883,-0.076167/
00106      9*      RHOF = (X1+X2*T)/32.174
00107     10*      RETURN
00110     11*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* RHOF/CFFC75 \*\*\*\*\*

\*\*\*\*\* RHOF/CFFC75 \*\*\*\*\*

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QFOR>S ME\*NASA5.RHOF/CFFC75,ME\*NASA5.RHOF/CFFC75  
FOR S9A-07/13/72-20:57:44 (0.)

FUNCTION RHOF ENTRY POINT 000014

STORAGE USED: CODE(1) 000016; DATA(0) 000011; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000004 INJP\$ 0000 R 000000 RHOF 0000 R 000001 X1 0000 R 000002 X2

```
00101      1*      FUNCTION RHOF(P,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      DENSITY AS A FUNCTION OF PRESSURE (LBF/SQ.FT) AND
00101      5*      C      TEMPERATURE (R) OF FC-75
00101      6*      C      UNITS SLUG/QU.FT
00101      7*      C
00103      8*      DATA X1,X2 /155.522,-0.085/
00106      9*      RHOF = (X1+X2*T)/32.174
00107     10*      RETURN
00110     11*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG>P \*\*\*\*\* RHOF/CFHE \*\*\*\*\*

\*\*\*\*\* RHOF/CFHE \*\*\*\*\*

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QFOR S ME\*NASA5.RHOF/CFHE,ME\*NASA5.RHOF/CFHE  
FOR S9A-07/13/72-20:57:47 (0,)

FUNCTION RHOF ENTRY POINT 000067

STORAGE USED: CODE(1) 000077; DATA(0) 000023; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 PF  
0004 CAPP  
0005 NERR3

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000015 2L	0001	000055 4L	0004 R 000000 CAPP	0000 R 000005 DIFF	0000 R 000006 DPDRHO
0000 R	000007 DRHO	0000	000014 INJP	0000 I 000001 N	0003 R 000000 PF	0000 R 000004 P1
0000 R	000002 R	0000 R	000000 RHOF	0000 R 000003 RHO1		

```

00101      1*      FUNCTION RHOF(P,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      DENSITY AS A FUNCTION OF PRESSURE (LBF/SQ.FT) AND
00101      5*      C      TEMPERATURE (R) OF HELIUM
00101      6*      C      UNITS SLUG/QU.FT
00101      7*      C
00103      8*      N      = 1
00104      9*      R      = 12419.164
00105     10*      RHO1 = P/(R*T)
00106     11*      IF (0.388-RHO1)1,1,2
00111     12*      1 RHO1 = 0.388
00112     13*      2 P1 = PF(RHO1,T)
00113     14*      DIFF = ABS((P-P1)/P)
00114     15*      IF (DIFF-1.0E-12)4,3,3
00117     16*      3 DPDRHO= 1.0/(RHO1*CAPP(RHO1,T))
00120     17*      DRHO = (P-P1)/DPDRHO
00121     18*      RHO1 = RHO1+DRHO
00122     19*      N      = N+1
00123     20*      IF (N-10)2,2,4
00126     21*      4 RHOF = RHO1
00127     22*      RETURN
00130     23*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* RHOF/CFNAK \*\*\*\*\*

\*\*\*\*\* RHOF/CFNAK \*\*\*\*\*

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PAGE 1

QFOR>S ME\*NASAS.RHOF/CFNAK,ME\*NASAS.RHOF/CFNAK  
FOR S9A-07/13/72-20:57:49 (0.)

FUNCTION RHOF ENTRY POINT 000031

STORAGE USED: CODE(1) 000040; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 CAPP  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0003 R 000000 CAPP	0000 000011 INJP	0000 R 000003 P0	0000 R 000006 P1	0000 R 000004 RHO
0000 R 000000 RHOF	0000 R 000005 TK	0000 R 000001 X1	0000 R 000002 X2	

```
00101 1* FUNCTION RHOF(P,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C DENSITY AS A FUNCTION OF PRESSURE (LBF/SQ.FT) AND
00101 5* C TEMPERATURE (R) OF NAK 78.6
00101 6* C UNITS SLUG/QU.FT
00101 7* C
00103 8* DATA X1,X2 /58.773064,-0.008433/,P0 /2116.224/
00107 9* RHO = (X1+X2*T)/32.174
00110 10* TK = CAPP(RHO,T)
00111 11* P1 = P-P0
00112 12* RHOF = RHO*(1.0+TK*P1)
00113 13* RETURN
00114 14* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* RHOF/CFSIL \*\*\*\*\*

\*\*\*\*\* RHOF/CFSIL \*\*\*\*\*

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PAGE 1

2FOR S ME\*NASAS.RHOF/CFSIL,ME\*NASAS.RHOF/CFSIL  
FOR S9A-07/13/72-20:57:51 (0.)

FUNCTION RHOF ENTRY POINT 000122

STORAGE USED: CODE(1) 000127; DATA(0) 000065; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 EXP  
0004 NWDUS  
0005 NIO2S  
0006 NERR3S

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000027	1F	0000 R 000022 A	0000 R 000001 A1	0000 R 000002 A2	0000 R 000003 A3
0000 R	000004	A4	0000 R 000005 A5	0000 R 000023 B	0000 R 000006 B1	0000 R 000007 B2
0000 R	000010	B3	0000 R 000011 B4	0000 R 000012 B5	0000 R 000024 C	0000 R 000013 C1
0000 R	000014	C2	0000 R 000017 DT	0000 000054 INJPS	0000 R 000025 P1	0000 R 000026 RHO
0000 R	000000	RHOF	0000 R 000020 THETA	0000 R 000021 THETA1	0000 R 000015 T0	0000 R 000016 T01

```

00101      1*      FUNCTION RHOF(P,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      DENSITY AS A FUNCTION OF PRESSURE (LBF/SQ.FT) AND
00101      5*      C      TEMPERATURE (R) OF DOW CORNING 200 SILICON OIL (1 CS)
00101      6*      C      UNITS SLUG/CU.FT
00101      7*      C      TEMPERATURE .GE. 359.67 AND .LE. 859.67
00101      8*      C
00103      9*      DATA A1,A2,A3,A4,A5 /12.35,2.98333,1.1,-0.48333,0.1/,B1,B2,B3,B4,
00103     10*      1B5 /-1.5,-0.01333,-1.18,0.57333,-0.1/,C1,C2 /0.7767,-0.0288/,
00103     11*      2T0,T01,DT /559.67,609.67,50.0/
00123     12*      THETA = (T-T0)/DT
00124     13*      THETA1 = (T-T01)/DT
00125     14*      A = (((A5+THETA+A4)*THETA+A3)*THETA+A2)*THETA+A1)*1.0E-06
00126     15*      B = (((B5+THETA+B4)*THETA+B3)*THETA+B2)*THETA+B1)*1.0E-09
00127     16*      C = C1+C2*THETA1
00130     17*      P1 = P/144.0-14.696
00131     18*      RHO = C*EXP(A*P1+0.5*B*P1*P1)
00132     19*      RHOF = RHO*62.42/32.174
00133     20*      IF (T.GE.360.67.AND.T.LE.860.67) RETURN
00135     21*      WRITE (6,1) T
00140     22*      1 FORMAT (1H0,4BHDENSITY OF SILICON OIL, TEMP. OUT OF RANGE, T = ,F8
00140     23*      1,3,/)
00141     24*      RETURN
00142     25*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

\*\*\*\*\* RHOF/CFSIL \*\*\*\*\*

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QMDG.P \*\*\*\*\* RKS \*\*\*\*\*



\*\*\*\*\* RKS \*\*\*\*\*

DATE 071372

PAGE

1

QFOR S ME\*NASA5.RKS,ME\*NASA5.RKS  
FOR S9A-07/13/72-20:57:54 (0.)

SUBROUTINE RKS ENTRY POINT 000643

STORAGE USED: CODE(1) 001040; DATA(0) 000064; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR2\$  
0004 NEXP5\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000010	10L	0001	000313	110L	0001	000333	120L	0001	000045	126G	0001	000343	130L
0001	000071	140G	0001	000355	140L	0001	000105	146G	0001	000130	156G	0001	000417	160L
0001	000150	164G	0001	000177	174G	0001	000500	185L	0001	000510	190L	0001	000013	20L
0001	000232	205G	0001	000270	217G	0001	000524	220L	0001	000530	230L	0001	000543	240L
0001	000374	243G	0001	000032	25L	0001	000552	250L	0001	000554	251L	0001	000572	257L
0001	000604	259L	0001	000623	270L	0001	000456	300L	0001	000615	336G	0001	000054	40L
0001	000060	45L	0001	000006	5L	0001	000076	50L	0001	000123	70L	0001	000135	80L
0000 R	000014	AM	0000 R	000007	AMAX	0000 R	000011	C	0000 R	000010	D	0000 R	000001	DDT
0000 R	000003	DELT	0000 R	000012	E	0000 R	000000	FR10	0000 I	000004	I	0000 I	000005	IFLAG
0000	000030	INJP\$	0000 I	000002	ISYMP	0000 I	000013	J	0000 R	000006	S			

00101	1*		SUBROUTINE RKS(DERIV,CNTRL,Y,DY,A,R,T,DEL,N,IFVD,IBKP,NTRY,	D6000100
00101	2*		1IERR,DELY,PD,SD,YS,YST,DYST,YSIMP)	D6000200
00103	3*		DIMENSION Y(N),DY(N),A(N),R(N),DELY(N),	D6000300
00103	4*		1PD(N),SD(N),YS(N),DYST(N),YST(N),YSIMP(N)	D6000400
00104	5*		EXTERNAL DERIV, CNTRL	D6000500
00104	6*	C	FR10 IS FIFTH ROOT OF TEN	D6000600
00105	7*		FR10=1.5848932	D6000700
00106	8*		IERR=0	D6000800
00106	9*	C		
00106	10*	C	THIS SUBROUTINE COMPUTES :	
00106	11*	C	RUNGE-KUTTA INTEGRATION (PRIMARY)	
00106	12*	C		
00106	13*	C	YS CONTAINS Y VALUES AT LEFT END POINT OF INTEGRATION INTERVAL	D6000900
00106	14*	C		D6001000
00106	15*	C	YSIMP CONTAINS Y FOR SIMPSONS RULE CHECK CHECK NOT MADE FOR	D6001100
00106	16*	C	FIXED STEP MODE ISYMP IS CONTROL PARAMETER =1, FIXED, 2 VARD	D6001200
00106	17*	C		D6001300
00106	18*	C	IF FIXED STEP SIZE GO ONE INTERVAL OF LENGTH DELT AND RETURN TO	D6001400
00106	19*	C	CNTRL, IF VAR GO TWO INTERVALS BEFORE RETURN TO CNTRL	D6001500
00106	20*	C		D6001600
00106	21*	C	IFVD = 0 VARIABLE INTERVAL	D6001700
00106	22*	C	= 1 FIXED	D6001800
00106	23*	C	IBKP = 0 CUT INTERVAL ONCE BEFORE REPEAT (UNDER IFVD=0 )	D6001900
00106	24*	C	= 1 CUT AS REQUIRED	D6002000

\*\*\*\*\* RKS \*\*\*\*\*

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00106 25* C      NTRY = 1  CONTINUE INTEGRATING
00106 26* C      2      RETURN FROM RKS
00106 27* C      3      STEP REPEATED WITH NEW DELT
00106 28* C      4      RESTART
00106 29* C      IERR = 0  NORMAL
00106 30* C      -1     DELT=0, RETURN FROM RKS
00106 31* C      1     A(I)+R(I)*ABS(Y(I)) = 0. , RETURN FROM RKS
00107 32*      5 IF (DEL) 20,10,20
00112 33*      10 IERR=-1
00113 34*      GO TO 270
00114 35*      20 CALL DERIV(Y,DY,T)
00115 36*      NTRY=1
00116 37*      CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)
00117 38*      25 DDT=DEL
00120 39*      IF (IFVD) 40,30,40
00123 40*      30 ISYMP=2
00124 41*      DELT=DEL/2.
00125 42*      DO 31 I=1,N
00130 43*      31 SD(I)=0.0
00132 44*      IFLAG=1
00133 45*      S=1.
00134 46*      GO TO 45
00135 47*      40 ISYMP=1
00136 48*      DELT=DEL
00137 49*      45 DO 46 I=1,N
00142 50*      YST(I)=Y(I)
00143 51*      46 DYST(I)=DY(I)
00145 52*      50 DO 60 I=1,N
00150 53*      DELY(I)=DELT*DY(I)
00151 54*      PD(I)=DELY(I)
00152 55*      60 CONTINUE
00154 56*      GO TO (80,70),ISYMP
00155 57*      70 DO 71 I=1,N
00160 58*      71 SD(I)=SD(I)+S*DY(I)
00162 59*      80 T=T+DELT/2.
00163 60*      DO 85 I=1,N
00166 61*      YS(I)=Y(I)
00167 62*      Y(I)=YS(I)+DELY(I)/2.
00170 63*      85 CONTINUE
00172 64*      CALL DERIV(Y,DY,T)
00173 65*      DO 90 I=1,N
00176 66*      DELY(I)=DELT*DY(I)
00177 67*      PD(I)=PD(I)+2.*DELY(I)
00200 68*      Y(I)=YS(I)+DELY(I)/2.
00201 69*      90 CONTINUE
00203 70*      CALL DERIV(Y,DY,T)
00204 71*      DO 95 I=1,N
00207 72*      DELY(I)=DELT*DY(I)
00210 73*      PD(I)=PD(I)+2.*DELY(I)
00211 74*      Y(I)=YS(I)+DELY(I)
00212 75*      95 CONTINUE
00214 76*      T=T+DELT/2.
00215 77*      CALL DERIV(Y,DY,T)
00216 78*      DO 100 I=1,N
00221 79*      DELY(I)=DELT*DY(I)
00222 80*      PD(I)=PD(I)+DELY(I)
00223 81*      Y(I)=YS(I)+PD(I)/6.

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D6002100
D6002200
D6002300
D6002400
D6002500
D6002600
D6002700
D6002800
D6002900
D6003000
D6003100
D6003200
D6003300
D6003400
D6003500
D6003600
D6003700
D6003800
D6003900
D6004000
D6004100
D6004200
D6004300
D6004400
D6004500
D6004600
D6004700
D6004800
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D6005000
D6005100
D6005200
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D6006100
D6006200
D6006300
D6006400
D6006500
D6006600
D6006700
D6006800
D6006900
D6007000
D6007100
D6007200
D6007300
D6007400
D6007500
D6007600
D6007700

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00224 82* 100 CONTINUE
00226 83* GO TO (110,120),ISYMP
00227 84* 110 NTRY=1
00230 85* CALL DERIV(Y,DY,T)
00231 86* CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)
00232 87* GO TO 300
00233 88* 120 GO TO (130,140),IFLAG
00234 89* 130 S=4.
00235 90* IFLAG=2
00236 91* CALL DERIV(Y,DY,T)
00237 92* GO TO 50
00240 93* 140 CALL DERIV(Y,DY,T)
00241 94* AMAX =0.0
00242 95* DO 180 I=1,N
00245 96* SD(I)=SD(I)+DY(I)
00246 97* YSIMP(I)=YST(I)+DELT*SD(I)/3.
00247 98* D =ABS(Y(I)-YSIMP(I))
00250 99* C =A(I)+R(I)*ABS(Y(I))
00251 100* IF(C .GT. 160,150,160)
00254 101* 150 IERR=1
00255 102* GO TO 270
00256 103* 160 E =ABS(D /C )
00257 104* AMAX=AMAX1(AMAX,E)
00260 105* 160 CONTINUE
00262 106* IF(AMAX-.1.) 215,215,230
00265 107* 215 NTRY= 1
00266 108* CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)
00267 109* 300 IF(NTRY-1) 185,185,310
00272 110* 310 IF(NTRY-2) 270,270,330
00275 111* 330 IF(NTRY-3) 340,340,5
00300 112* 340 T=T-DDT
00301 113* IF(DEL) 259,10,259
00304 114* 185 GO TO (40,190),ISYMP
00305 115* 190 IF(AMAX-.75) 200,25,220
00310 116* 200 IF(AMAX-.075) 210,25,25
00313 117* 210 DEL=DEL*FR10
00314 118* GO TO 25
00315 119* 220 DEL=DEL/FR10
00316 120* GO TO 25
00317 121* 230 I =1+ I8KP
00320 122* GO TO (240,250),I
00321 123* 240 T=T-DEL
00322 124* DEL=DEL/FR10
00323 125* GO TO 259
00324 126* 250 J=1
00325 127* 251 AM=AMAX/10.**J
00326 128* IF(1.-AM) 255,257,257
00331 129* 255 J=J+1
00332 130* GO TO 251
00333 131* 257 T=T-DEL
00334 132* DEL=DEL/(FR10**J)
00335 133* 259 DO 245 I=1,N
00340 134* DY(I)=DYST(I)
00341 135* 245 Y(I)=YST(I)
00343 136* GO TO 25
00344 137* 270 RETURN
00345 138* END

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D6007800
D6007900
D6008000
D6008100
D6008200
D6008300
D6008400
D6008500
D6008600
D6008700
D6008800
D6008900
D6009000
D6009100
D6009200
D6009300
D6009400
D6009500
D6009600
D6009700
D6009800
D6009900
D6010000
D6010100
D6010200
D6010300
D6010400
D6010500
D6010600
D6010700
D6010800
D6010900
D6011000
D6011100
D6011200
D6011300
D6011400
D6011500
D6011600
D6011700
D6011800
D6011900
D6012000
D6012100
D6012200
D6012300
D6012400
D6012500
D6012600
D6012700
D6012800
D6012900
D6013000
D6013100
D6013200
D6013300
D6013400

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END OF COMPILATION: NO DIAGNOSTICS.

BHD6P \*\*\*\*\* RKSF \*\*\*\*\*

\*\*\*\*\* RKSF \*\*\*\*\*

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PAGE

1

QFOR,S ME\*NASA5,RKSF,ME\*NASA5,RKSF  
FOR S9A-07/13/72-20:57:59 (0,)

SUBROUTINE RKSF ENTRY POINT 000643

STORAGE USED: CODE(1) 001040; DATA(0) 000064; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR2\$  
0004 NEXP5\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000010	10L	0001	000313	110L	0001	000333	120L	0001	000045	126G	0001	000343	130L
0001	000071	140G	0001	000355	140L	0001	000105	146G	0001	000130	156G	0001	000417	160L
0001	000150	164G	0001	000177	174G	0001	000500	185L	0001	000510	190L	0001	000013	20L
0001	000232	205G	0001	000270	217G	0001	000524	220L	0001	000530	230L	0001	000543	240L
0001	000374	243G	0001	000032	25L	0001	000552	250L	0001	000554	251L	0001	000572	257L
0001	000604	259L	0001	000623	270L	0001	000456	300L	0001	000615	336G	0001	000054	40L
0001	000060	45L	0001	000006	5L	0001	000076	50L	0001	000123	70L	0001	000135	80L
0000 R	000014	AM	0000 R	000007	AMAX	0000 R	000011	C	0000 R	000010	D	0000 R	000001	DDT
0000 R	000003	DELT	0000 R	000012	E	0000 R	000000	FR10	0000 I	000004	I	0000 I	000005	IFLAG
0000	000030	INJP\$	0000 I	000002	ISYMP	0000 I	000013	J	0000 R	000006	S			

00101	1*	SUBROUTINE RKSF(deriv,cntrl,y,dy,a,r,t,del,n,ifvd,ibkp,ntry,	D6000100
00101	2*	1terr,deley,po,so,ys,yst,dyst,ysimp)	D6000200
00103	3*	DIMENSION Y(N),DY(N),A(N),R(N),DELY(N),	D6000300
00103	4*	1PD(N),SD(N),YS(N),DYST(N),YST(N),YSIMP(N)	D6000400
00104	5*	EXTERNAL deriv, cntrl	D6000500
00104	6*	C FR10 IS FIFTH ROOT OF TEN	D6000600
00105	7*	FR10=1.5848932	D6000700
00106	8*	1ERR=0	D6000800
00106	9*	C	
00106	10*	C THIS SUBROUTINE COMPUTES :	
00106	11*	C RUNGE-KUTTA INTEGRATION (SECONDARY)	
00106	12*	C YS CONTAINS Y VALUES AT LEFT END POINT OF INTEGRATION INTERVAL	D6000900
00106	13*	C	D6001000
00106	14*	C YSIMP CONTAINS Y FOR SIMPSONS RULE CHECK CHECK NOT MADE FOR	D6001100
00106	15*	C FIXED STEP MODE ISYMP IS CONTROL PARAMETER =1.FIXED,2.VAR	D6001200
00106	16*	C	D6001300
00106	17*	C IF FIXED STEP SIZE GO ONE INTERVAL OF LENGTH DELT AND RETURN TO	D6001400
00106	18*	C CNTRL, IF VAR GO TWO INTERVALS BEFORE RETURN TO CNTRL	D6001500
00106	19*	C	D6001600
00106	20*	C IFVD = 0 VARIABLE INTERVAL	D6001700
00106	21*	C = 1 FIXED	D6001800
00106	22*	C IBKP = 0 CUT INTERVAL ONCE BEFORE REPEAT (UNDER IFVD=0 )	D6001900
00106	23*	C = 1 CUT AS REQUIRED	D6002000
00106	24*	C NTRY = 1 CONTINUE INTEGRATING	D6002100

\*\*\*\*\* RKSF \*\*\*\*\*

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00106 25* C 2 RETURN FROM RKS
00106 26* C 3 STEP REPEATED WITH NEW DELT
00106 27* C 4 RESTART
00106 28* C IERR = 0 NORMAL
00106 29* C -1 DELT=0, RETURN FROM RKS
00106 30* C 1 A(I)+ R(I)*ABS(Y(I)) = 0. , RETURN FROM RKS
00107 31* 5 IF(DEL) 20,10,20
00112 32* 10 IERR=-1
00113 33* 60 TO 270
00114 34* 20 CALL DERIV(Y,DY,T)
00115 35* NTRY=1
00116 36* CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)
00117 37* 25 DDT=DEL
00120 38* IF(IFVD) 40,30,40
00123 39* 30 ISYMP=2
00124 40* DELT=DEL/2.
00125 41* DO 31 I=1,N
00130 42* 31 SD(I)=0.0
00132 43* IFLAG=1
00133 44* S=1.
00134 45* GO TO 45
00135 46* 40 ISYMP=1
00136 47* DELT=DEL
00137 48* 45 DO 46 I=1,N
00142 49* YST(I)=Y(I)
00143 50* 46 DYST(I)=DY(I)
00145 51* 50 DO 60 I=1,N
00150 52* DELY(I)=DELT*DY(I)
00151 53* PD(I)=DELY(I)
00152 54* 60 CONTINUE
00154 55* GO TO (80,70),ISYMP
00155 56* 70 DO 71 I=1,N
00160 57* 71 SD(I)=SD(I)+S*DY(I)
00162 58* 80 T=T+DELT/2.
00163 59* DO 85 I=1,N
00166 60* YS(I)=Y(I)
00167 61* Y(I)=YS(I)+DELY(I)/2.
00170 62* 85 CONTINUE
00172 63* CALL DERIV(Y,DY,T)
00173 64* DO 90 I=1,N
00176 65* DELY(I)=DELT*DY(I)
00177 66* PD(I)=PD(I)+2.*DELY(I)
00200 67* Y(I)=YS(I)+DELY(I)/2.
00201 68* 90 CONTINUE
00203 69* CALL DERIV(Y,DY,T)
00204 70* DO 95 I=1,N
00207 71* DELY(I)=DELT*DY(I)
00210 72* PD(I)=PD(I)+2.*DELY(I)
00211 73* Y(I)=YS(I)+DELY(I)
00212 74* 95 CONTINUE
00214 75* T=T+DELT/2.
00215 76* CALL DERIV(Y,DY,T)
00216 77* DO 100 I=1,N
00221 78* DELY(I)=DELT*DY(I)
00222 79* PD(I)=PD(I)+DELY(I)
00223 80* Y(I)=YS(I)+PD(I)/6.
00224 81* 100 CONTINUE

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D6002200
D6002300
D6002400
D6002500
D6002600
D6002700
D6002800
D6002900
D6003000
D6003100
D6003200
D6003300
D6003400
D6003500
D6003600
D6003700
D6003800
D6003900
D6004000
D6004100
D6004200
D6004300
D6004400
D6004500
D6004600
D6004700
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D6006500
D6006600
D6006700
D6006800
D6006900
D6007000
D6007100
D6007200
D6007300
D6007400
D6007500
D6007600
D6007700
D6007800

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00226 82*      GO TO (110,120),ISYMP
00227 83*      110 NTRY=1
00230 84*      CALL DERIV(Y,DY,T)
00231 85*      CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)
00232 86*      GO TO 300
00233 87*      120 GO TO (130,140),IFLAG
00234 88*      130 S=4.
00235 89*      IFLAG=2
00236 90*      CALL DERIV(Y,DY,T)
00237 91*      GO TO 50
00240 92*      140 CALL DERIV(Y,DY,T)
00241 93*      AMAX =0.0
00242 94*      DO 180 I=1,N
00245 95*      SO(I)=SO(I)+DY(I)
00246 96*      YSIMP(I)=YST(I)+DEL*SO(I)/3.
00247 97*      D =ABS(Y(I)-YSIMP(I))
00250 98*      C =A(I)+R(I)*ABS(Y(I))
00251 99*      IF(C .LT. 160,150,160)
00254 100*     15) IERR=1
00255 101*     GO TO 270
00256 102*     16) E =ABS(D /C )
00257 103*     AMAX=AMAX1(AMAX,E)
00260 104*     18) CONTINUE
00262 105*     IF(AMAX-1.) 215,215,230
00265 106*     21) NTRY= 1
00266 107*     CALL CNTRL(Y,DY,DEL,T,NTRY,IFVD)
00267 108*     30) IF(NTRY-1) 185,185,310
00272 109*     31) IF(NTRY-2) 270,270,330
00275 110*     33) IF(NTRY-3) 340,340,5
00300 111*     34) T=T-DDT
00301 112*     IF(DEL) 259,10,259
00304 113*     18) GO TO (40,190),ISYMP
00305 114*     19) IF(AMAX-.75) 200,25,220
00310 115*     20) IF(AMAX-.075) 210,25,25
00313 116*     21) DEL=DEL*FR10
00314 117*     GO TO 25
00315 118*     22) DEL=DEL/FR10
00316 119*     GO TO 25
00317 120*     23) I =1+ 18KP
00320 121*     GO TO (240,250),I
00321 122*     24) T=T-DEL
00322 123*     DEL=DEL/FR10
00323 124*     GO TO 259
00324 125*     25) J=1
00325 126*     251 AM=AMAX/10.**J
00326 127*     IF(I.-AM) 255,257,257
00331 128*     255 J=J+1
00332 129*     GO TO 251
00333 130*     257 T=T-DEL
00334 131*     DEL=DEL/(FR10**J)
00335 132*     259 DO 245 I=1,N
00340 133*     DY(I)=DYST(I)
00341 134*     245 Y(I)=YST(I)
00343 135*     GO TO 25
00344 136*     270 RETURN
00345 137*     END

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06007900
06008000
06008100
06008200
06008300
06008400
06008500
06008600
06008700
06008800
06008900
06009000
06009100
06009200
06009300
06009400
06009500
06009600
06009700
06009800
06009900
06010000
06010100
06010200
06010300
06010400
06010500
06010600
06010700
06010800
06010900
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06011300
06011400
06011500
06011600
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06011800
06011900
06012000
06012100
06012200
06012300
06012400
06012500
06012600
06012700
06012800
06012900
06013000
06013100
06013200
06013300
06013400

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END OF COMPILATION: NO DIAGNOSTICS.

QHDG.P \*\*\*\*\* SHADE \*\*\*\*\*



\*\*\*\*\* SHADE \*\*\*\*\*

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PAGE 1

QFOR,S ME\*NASA5.SHADE,ME\*NASA5.SHADE  
FOR S9A-07/13/72-20:58:04 (0,)

SUBROUTINE SHADE ENTRY POINT 000073

STORAGE USED: CODE(1) 000104; DATA(0) 000023; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000027	10L	0001	000015	106G	0001	000057	123G	0000	000005	INJP\$	0000	I	000001	J
0000	I	000003	J1	0000	I	000002	J2	0000	R	000000	QM				

00101	1*		SUBROUTINE SHADE(Q,NS)
00103	2*		DIMENSION Q(1)
00104	3*		QM = 0.0
00105	4*		DO 10 J = 1,NS
00110	5*		IF (QM.GT.Q(J)) GO TO 10
00112	6*		QM = Q(J)
00113	7*		J2 = J
00114	8*	10	CONTINUE
00116	9*		J1 = 1
00117	10*		IF (J2.LE.NS/2) J1 = NS/2+1
00121	11*		J2 = J1+NS/2-1
00122	12*		DO 20 J = J1,J2
00125	13*	20	Q(J) = 0.0
00127	14*		RETURN
00130	15*		END

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* SHAPEF \*\*\*\*\*

\*\*\*\*\* SHAPEF \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASA5.SHAPEF,ME\*NASA5.SHAPEF  
FOR S9A-07/13/72-20:58:06 (0.)

SUBROUTINE SHAPEF ENTRY POINT 001020

STORAGE USED: CODE(1) 001035; DATA(0) 000152; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 GRD 003721  
0004 SSF 000010

EXTERNAL REFERENCES (BLOCK, NAME)

0005 ACOS  
0006 ATAN  
0007 SQRT  
0010 COS  
0011 SIN  
0012 NEXP6\$  
0013 ASIN  
0014 ALOG  
0015 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000036 1176	0001 000037 1226	0001 000040 1256	0001 000066 1356	0001 000307 1746
0001 000053 20L	0001 000447 2166	0001 000452 2216	0001 000457 2256	0001 000671 2616
0001 000752 2746	0001 000107 30L	0001 000757 3006	0001 000115 35L	0001 000603 62L
0001 000620 64L	0001 000634 65L	0001 000442 70L	0000 R 000046 A	0004 R 000002 ALPFN
0000 R 000047 AR	0000 R 000050 ARC	0000 R 000051 AR2	0000 R 000035 BA	0000 R 000027 BETA
0000 R 000066 BETA1	0000 R 000067 BETA2	0000 R 000014 CF	0000 R 000043 CPB	0003 R 003465 CSF
0000 R 000015 DCL	0003 003464 DXX2	0003 003472 DXX21	0000 R 000064 DZ	0003 R 003463 DZMFN
0000 R 000045 DZ2	0003 003410 E8FN	0003 003441 EBMP	0003 003467 EXTIFN	0003 003471 EXTIMP
0003 003466 EXT5FN	0003 003470 EXT5MP	0000 R 000062 GAMA	0000 R 000036 GAMMA	0000 I 000021 I
0000 000112 INJPS	0000 I 000022 J	0000 I 000023 K	0000 I 000024 L	0003 003461 LCT
0003 003462 LTT	0000 I 000052 M	0003 003460 MCVRD	0000 R 000032 OB	0000 R 000034 OP
0000 R 000033 P8	0000 R 000017 PHI	0000 R 000037 PHI0	0000 R 000055 PHI1	0000 R 000012 PI
0000 R 000013 PI2	0000 R 000040 PR4	0000 R 000044 RC	0000 R 000031 RH0	0000 R 000030 RH02
0000 R 000053 RL	0000 R 000057 RL2	0000 R 000016 RR	0000 R 000041 RR0	0000 R 000042 RR02
0000 R 000054 RT	0000 R 000061 RTL	0000 R 000056 RT2	0004 R 000003 R1	0004 R 000000 R2
0003 R 003473 SS	0000 R 000063 SST	0003 R 003670 SSTT	0004 R 000001 T	0000 R 000020 THETA
0000 R 000060 THETA1	0003 000000 TRMTX	0000 R 000070 WF	0004 R 000006 WWA	0004 R 000007 WWB
0000 R 000000 X	0000 R 000065 XB	0004 R 000005 XSMP	0004 000004 XSPM	0000 R 000005 XX
0003 R 003446 XX2	0000 R 000025 Y	0000 R 000026 YY	0003 R 003453 Z22	

00101 1\* SUBROUTINE SHAPEF  
00101 2\* C  
00101 3\* C THIS SUBROUTINE COMPUTES THE SHAPE FACTOR BETWEEN :  
00101 4\* C 1- TUBE AND NEAR FIN

\*\*\*\*\* SHAPEF \*\*\*\*\*

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00101      5*      C      1- TUBE AND FAR FIN
00101      6*      C      3- ADJACENT TUBES
00101      7*      C
00103      8*      COMMON /GRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZ2(5),
00103      9*      1      MCVRD,LCT,LTT,DZMFN,
00103     10*      2      DX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DX21,SS(5,5,5)
00103     11*      3      ,SSTT(5,5)
00104     12*      COMMON /SSF/R2, T, ALPFN,R1,XSPM,XSMP,WWA,WWB
00105     13*      DIMENSION X(5), XX(5)
00106     14*      PI      = 3.14159
00107     15*      PI2     = PI/2.0
00110     16*      CF      = DZMFN*4.0
00111     17*      DCL     = 2.0+2.0*R1
00112     18*      RR      = R2*R2
00113     19*      PHI     = ACOS((XSMP+R1)/R2)
00114     20*      THETA    = 1.570796-PHI
00115     21*      CSF     = 1.0/(R2*THETA)
00116     22*      DO 10 I=1,5
00121     23*      DO 10 J=1,5
00124     24*      DO 10 K=1,5
00127     25*      10 SS(I,J,K) = 0.0
00133     26*      L      = 0
00134     27*      20 DO 50 I=1,5
00137     28*      IF (L.EQ.1) GO TO 30
00141     29*      X(I)    = XX2(I)+R1
00142     30*      XX(I)    = XX2(I)
00143     31*      Y      = WWA*X(I)+WWB
00144     32*      YY     = Y*Y
00145     33*      IF (L.EQ.0) GO TO 35
00147     34*      30 X(I)  = DCL-X(I)
00150     35*      XX(I)  = 2.0-XX2(I)
00151     36*      35 BETA  = ATAN(Y/X(I))
00152     37*      RH02    = X(I)*X(I)+YY
00153     38*      RH0     = SQRT(RH02)
00154     39*      OB      = ACOS(R2/RH0)
00155     40*      PB      = PHI-BETA
00156     41*      OP      = OB-PB
00157     42*      BA      = BETA+ALPFN
00160     43*      GAMMA  = OB+BA
00161     44*      IF (L.EQ.1) BA = BETA-ALPFN
00163     45*      PHIO    = ATAN(DZMFN/(2.0*XX(I)))
00164     46*      SS(I,1,1) = ((1.0-SIN(GAMMA))/PI)*(PHIO+SIN(2.0*PHIO)/2.0)*4.0+
00164     47*      1      SS(I,1,1)
00165     48*      PR4     = 4.*PI*RH0
00166     49*      KR0     = 2.*R2*RH0
00167     50*      KR02    = RRO*RRO
00170     51*      CPB    = COS(PB)
00171     52*      RC      = RRO*CPB
00172     53*      J      = 1
00173     54*      DO 50 K=2,5
00176     55*      DZ2    = ((ZZ2(K)-ZZ2(J))*CF)**2
00177     56*      A      = RR+RH02+DZ2
00200     57*      AR      = A-A*RR02
00201     58*      ARC     = A-RC
00202     59*      AR2     = A-2.0*RR
00203     60*      SS(I,J,K) = (COS(BA)/PR4*(ALOG(ARC/AR2)-1.0+AR2/ARC)+SIN(BA)/PR4*
00203     61*      1      (OP+(2.0*RRO2+DZ2-A*AR)/(AR**1.5)*(ASIN((RRO-A*COS(OB)

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00203 62*      2      )/AR2)-ASIN((RRO-A*CPB)/ARC))+((4.0*A*DZ2/AR-2.0)*((R2
00203 63*      3      *RHO*SIN(OB))/AR2-(R2*RHO*SIN(PB))/ARC)))*CF+SS(I,J,K)
00204 64*      IF (SS(I,J,K) .LT. 0.0) SS(I,J,K) = 0.0
00206 65*      50 CONTINUE
00211 66*      IF (L.EQ.1) GO TO 70
00213 67*      L      = 1
00214 68*      GO TO 20
00215 69*      70 DO 60 I=1,5
00220 70*      DO 60 K=2,5
00223 71*      SS(I,K,1) = SS(I,1,K)
00224 72*      DO 60 J=2,5
00227 73*      M      = ABS(J-K)+1
00230 74*      60 SS(I,J,K) = SS(I,1,M)
00234 75*      RL      = DCL/R2-COS(PHI)
00235 76*      RT      = T/R2
00236 77*      PHI1     = ATAN(RT/RL)
00237 78*      RT2     = RT*RT
00240 79*      RL2     = RL*RL
00241 80*      THETA1   = ACOS(1.0/SQRT(RT2+RL2))
00242 81*      RTL     = (1.0-RT2)/RL2
00243 82*      GAMA     = PI2-PHI1-THETA1
00244 83*      IF (RTL-1.0E-03) 62,62,61
00247 84*      61 SST   = (GAMA+RL*(SQRT(1.0-RTL)-1.0))/(PI2-PHI)
00250 85*      GO TO 65
00251 86*      62 IF(RTL-1.0E-04) 63,64,64
00254 87*      63 SST   = (GAMA-RL*RTL/2.0)/(PI2-PHI)
00255 88*      GO TO 65
00256 89*      64 SST   = (GAMA-RL*(RTL/2.0+RTL*RTL/8.0))/(PI2-PHI)
00257 90*      65 J      = 1
00260 91*      DO 80 K=1,5
00263 92*      DZ      = (ZZ2(K)-ZZ2(J))*CF
00264 93*      XB      = 2.0*(1.0+R1-R2*COS((PHI1+THETA1)/2.0))
00265 94*      BETA1    = ATAN((DZ-DZMFN/2.0)/XB)
00266 95*      BETA2    = ATAN((DZ+DZMFN/2.0)/XB)
00267 96*      WF      = (BETA2-BETA1)/2.0+(SIN(2.0*BETA2)-SIN(2.0*BETA1))/4.0
00270 97*      SSTT(J,K) = 8.0*SST*WF/PI
00271 98*      80 CONTINUE
00273 99*      DO 90 K=2,5
00276 100*     SSTT(K,1) = SSTT(1,K)
00277 101*     DO 90 J=2,5
00302 102*     M      = ABS(J-K)+1
00303 103*     90 SSTT(J,K) = SSTT(1,M)
00306 104*     RETURN
00307 105*     END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* TCALC \*\*\*\*\*

\*\*\*\*\* TICALC \*\*\*\*\*

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QFOR:5 ME\*NASAS.TICALC,ME\*NASAS.TICALC  
FOR S9A-07/13/72-20:58:10 (0,)

SUBROUTINE TICALC ENTRY POINT 000635

STORAGE USED: CODE(1) 000652; DATA(0) 000670; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 ADBH 000352  
0004 TC 000042  
0005 QIN 001610

EXTERNAL REFERENCES (BLOCK, NAME)

0006 YINT  
0007 SHAJE  
0010 NRBJS  
0011 NIO2\$  
0012 NBSP\$  
0013 NIO1\$  
0014 NEXP6\$  
0015 NREW\$  
0016 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000024	10L	0001	000426	110L	0001	000024	125G	0001	000047	141G	0001	000121	160G
0001	000127	165G	0001	000146	173G	0001	000164	203G	0001	000247	220G	0001	000252	223G
0001	000276	232G	0001	000504	261G	0001	000042	30L	0001	000070	50L	0001	000254	90L
0004 R	000003	ABEMF	0003	000001	AL	0000 R	000562	ANG	0003	000161	CONFL	0003	000173	CONFNA
0003	000147	CP	0003	000000	D	0003	000326	DTL	0003	000244	EFF	0003	000205	EMIS
0003	000013	H	0000 R	000623	HA	0003	000231	HAB	0000 R	000624	HP	0000 R	000622	HS
0000 I	000611	I	0004 I	000001	ICASE	0000	000637	INJPS	0000 I	000614	ISYM	0004 I	000000	ITAPE
0000 I	000615	IU	0000 I	000613	IUL	0000 I	000612	J	0000 I	000616	J1	0000 I	000617	J2
0000 I	000621	K	0003 R	000064	M	0000 I	000610	NS	0005 I	001605	NSRD	0003	000076	NT
0005 I	001604	NTV	0000 R	000620	PANG	0004 R	000004	PHIN	0003	000052	PIN	0000 R	000436	GA
0000 R	000124	QALB	0005	000454	QIFN	0005	001274	QITB	0003	000314	QOUT	0000 R	000510	QP
0000 R	000244	QPLN	0000 R	000364	QS	0005	000144	QSFN	0005	000764	QSTB	0000 R	000004	OSUN
0005	001606	QTO	0000 R	000607	STFBOL	0003	000217	TB	0003	000026	TH	0000 R	000000	TIM
0003	000040	TIN	0003	000270	TL	0005	000000	TM	0004 R	000002	TO	0003	000340	TOUT
0003 R	000077	TSTAR	0005	001607	TX	0003	000135	VSC	0006 R	000000	YINT			

00101 1\* SUBROUTINE TICALC(NTP)  
00103 2\* PARAMETER NTP = 10  
00104 3\* PARAMETER MAXSID = 20  
00105 4\* PARAMETER NTP1 = NTP+1  
00106 5\* PARAMETER MAXSD1 = MAXSID+1  
00107 6\* REAL M  
00110 7\* COMMON /ADBH/ D,AL(NTP),H(NTP1),TH(NTP),TIN(NTP),PIN(NTP),

```

00110      8*      1      M(NTP),NT,TSTAR(NTP,3),VSC(4TP),CP(NTP),CONFL(NTP),
00110      9*      2      CONFNA(NTP),E415(NTP),TB(NTP),HAB(NTP1),EFF(NTP,2),TL(NTP,2),
00110     10*      3      GOUT(NTP),DTL(NTP),TOUT(NTP)
00111     11*      COMMON /TC/ ITAPE,ICASE,TO,ABEMF,PHIN(NTP,3)
00112     12*      COMMON /GIN/ TM(100),QSFN(100,2),QIFN(100,2),QSTB(100,2),
00112     13*      1      QITB(100,2),NTM,NSRD,QTO,TX
00113     14*      DIMENSION TIM(4),QSUN(4,MAXSID),GALB(4,MAXSID),QPLN(4,MAXSID),
00113     15*      1      QS(MAXSD1,2),QA(MAXSD1,2),QP(MAXSD1,2),ANG(MAXSD1)
00114     16*      DATA STFBOL/0.1714E-8/
00116     17*      READ (ITAPE) NS,TIM(1)
00122     18*      IF (ICASE.LE.1) GO TO 30
00124     19*      DO 20 I = 2,ICASE
00127     20*      10      TIM(2) = TIM(1)
00130     21*      READ (ITAPE) NS,TIM(1)
00134     22*      IF (TIM(1).GT.TIM(2)) GO TO 10
00136     23*      20      CONTINUE
00140     24*      30      DO 40 J = 0,NS
00143     25*      40      ANG(J+1) = FLOAT(J*360+180)/NS
00145     26*      READ (ITAPE)
00147     27*      I = 2
00150     28*      50      READ (ITAPE) NS,TIM(1)
00154     29*      I = I+1
00155     30*      IF (TIM(1).LT.TO.AND.I.LT.NTM-1) GO TO 50
00157     31*      DO 60 I = 1,3
00162     32*      60      BACKSPACE ITAPE
00164     33*      DO 70 I = 1,4
00167     34*      70      READ (ITAPE) NS,TIM(I),(QSUN(I,J),GALB(I,J),QPLN(I,J),J=1,NS)
00202     35*      DO 80 J = 1,NS
00205     36*      QS(J,1) = YINT(TIM,QSUN(1,J),4,4,TO)
00206     37*      QA(J,1) = YINT(TIM,GALB(1,J),4,4,TO)
00207     38*      80      QP(J,1) = YINT(TIM,QPLN(1,J),4,4,TO)
00211     39*      QS(NS+1,1) = QS(1,1)
00212     40*      QA(NS+1,1) = QA(1,1)
00213     41*      QP(NS+1,1) = QP(1,1)
00214     42*      IUL = 1
00215     43*      IF (ISYM.EQ.3) IUL = 3
00217     44*      DO 120 IU = 1,IUL
00222     45*      DO 120 I = 1,NTF
00225     46*      J1 = 0
00226     47*      90      J1 = J1+1
00227     48*      IF (ANG(J1)+ANG(1).LE.PHIN(I,IU)) GO TO 90
00231     49*      DO 100 J = 1,NS
00234     50*      J2 = MIN(MAX(MOD(J+J1-2,NS),1),NS-2)
00235     51*      PANG = ANG(J)+PHIN(I,IU)
00236     52*      IF (PANG.GE.ANG(NS+1)) PANG = PANG-360.0
00240     53*      QS(J,2) = YINT(ANG(J2),QS(J2,1),4,4,PANG)
00241     54*      QA(J,2) = YINT(ANG(J2),QA(J2,1),4,4,PANG)
00242     55*      100      QP(J,2) = YINT(ANG(J2),QP(J2,1),4,4,PANG)
00244     56*      CALL SHADE(QS(1,2),NS)
00245     57*      CALL SHADE(QA(1,2),NS)
00246     58*      CALL SHADE(QP(1,2),NS)
00247     59*      PANG = AMOD(PHIN(I,IU)+90.0*360.0)
00250     60*      J1 = 0
00251     61*      110      J1 = J1+1
00252     62*      IF (ANG(J1).LE.PANG) GO TO 110
00254     63*      J2 = 0
00255     64*      IF (PANG.GE.180.0) J2 = NS/2

```

\*\*\*\*\* TCALC \*\*\*\*\*

```
00257 65*      K = J2+MIN(MAX(J1-J2-2,1),NS/2-3)
00260 66*      DO 120 J = IU,NSRD
00263 67*      IF (J.NE.IU) K = MOD(K+NS/2,NS)
00265 68*      HS = YINT(ANG(K),QS(K,2),4,4,PANG)
00266 69*      HA = YINT(ANG(K),QA(K,2),4,4,PANG)
00267 70*      HP = YINT(ANG(K),QP(K,2),4,4,PANG)
00270 71*      120 TSTAR(I,J) = ((HP+(HA+HS)*ABEMF)/STFBOL)**0.25
00274 72*      REWIND ITAPE
00275 73*      RETURN
00276 74*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDS,P \*\*\*\*\* THCF/CFFC43 \*\*\*\*\*

\*\*\*\*\* THCF/CFFC43 \*\*\*\*\*

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QFOR,S ME\*NASA5.THCF/CFFC43,ME\*NASA5.THCF/CFFC43  
FOR S9A-07/13/72-20:58:14 (0,)

FUNCTION THCF ENTRY PQINT 000013

STORAGE USED: CODE(1) 000015; DATA(0) 000010; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000003 INJP5      0000 R 000000 THCF      0000 R 000001 X1      0000 R 000002 X2

```
00101      1*      FUNCTION THCF(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF DENSITY (SLUG/CU.FT)
00101      5*      C      AND TEMPERATURE (R) OF FC-43
00101      6*      C      UNITS BTU/(HR.FT.R)
00101      7*      C
00103      8*      DATA X1,X2 /0.061442,-2.5E-05/
00106      9*      THCF = X1+X2*T
00107     10*      RETURN
00110     11*      END
```

END OF COMPILATION:      NO DIAGNOSTICS.

QHD6,P \*\*\*\*\* THCF/CFFC75 \*\*\*\*\*



\*\*\*\*\* THCF/CFFC75 \*\*\*\*\*

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QFOR: S ME\*NASA5.THCF/CFFC75,ME\*NASA5.THCF/CFFC75  
FOR 59A-07/13/72-20:58:16 (0.)

FUNCTION THCF ENTRY POINT 000013

STORAGE USED: CODE(1) 000015; DATA(0) 000010; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000003 INJP\$ 0000 R 000000 THCF 0000 R 000001 X1 0000 R 000002 X2

```
00101 1* FUNCTION THCF(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C THERMAL CONDUCTIVITY AS A FUNCTION OF DENSITY (SLUG/CU.FT)
00101 5* C AND TEMPERATURE (R) OF FC-75
00101 6* C UNITS BTU/(HR.FT.R)
00101 7* C
00103 8* DATA X1,X2 /0.114181,-6.53E-05/
00106 9* THCF = X1+X2*T
00107 10* RETURN
00110 11* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHD6,P \*\*\*\*\* THCF/CFHE \*\*\*\*\*

\*\*\*\*\* THCF/CFHE \*\*\*\*\*

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QFOR: S ME\*NASA5.THCF/CFHE, ME\*NASA5.THCF/CFHE  
FOR 59A-07/13/72-20:58:18 (0.)

FUNCTION THCF ENTRY POINT 000022

STORAGE USED: CODE(1) 000024; DATA(0) 000015; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 000010 INJP5	0000 R 000000 THCF	0000 R 000007 THETA	0000 R 000005 T0
0000 R 000001 X1	0000 R 000002 X2	0000 R 000003 X3	0000 R 000004 X4	

00101	1*		FUNCTION THCF(RHO,T)
00101	2*	C	
00101	3*	C	THIS FUNCTION SUBPROGRAM COMPUTES :
00101	4*	C	THERMAL CONDUCTIVITY AS A FUNCTION OF DENSITY (SLUG/CU.FT)
00101	5*	C	AND TEMPERATURE (R) OF HELIUM
00101	6*	C	UNITS BTU/(HR.FT.R)
00101	7*	C	
00103	8*		DATA X1,X2,X3,X4 /0.0404,0.0302,-0.0033,0.0003/,T0,DT /160.0,20
00103	9*		10.0/
00112	10*		THETA = (T-T0)/DT
00113	11*		THCF = ((X4*THETA+X3)*THETA+X2)*THETA+X1
00114	12*		RETURN
00115	13*		END

END OF COMPILATION: NO DIAGNOSTICS.

QHD6:P \*\*\*\*\* THCF/CFNAK \*\*\*\*\*

\*\*\*\*\* THCF/CFNAK \*\*\*\*\*

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PAGE 1

QFOR S ME\*NASA5.THCF/CFNAK,ME\*NASA5.THCF/CFNAK  
FOR S9A-07/13/72-20:58:20 (0.)

FUNCTION THCF ENTRY POINT 000024

STORAGE USED: CODE(1) 000026; DATA(0) 000016; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000007 DT	0000 000011 INJP5	0000 R 000000 THCF	0000 R 000010 THETA	0000 R 000006 T0
0000 R 000001 X1	0000 R 000002 X2	0000 R 000003 X3	0000 R 000004 X4	0000 R 000005 X5

```
00101 1* FUNCTION THCF(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C THERMAL CONDUCTIVITY AS A FUNCTION OF DENSITY (SLUG/CU.FT)
00101 5* C AND TEMPERATURE (R) OF NAK 78.6
00101 6* C UNITS BTU/(HR.FT.R)
00101 7* C
00103 8* DATA X1,X2,X3,X4,X5 /13.36,1.414167,-0.142083,-0.069167,0.007083/,
00103 9* 1T0,DT /659.67,300.0/
00113 10* THETA = (T-T0)/DT
00114 11* THCF = (((X5*THETA*X4)*THETA*X3)*THETA*X2)*THETA*X1
00115 12* RETURN
00116 13* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* THCF/CFSIL \*\*\*\*\*

\*\*\*\*\* THCF/CFSIL \*\*\*\*\*

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QFOR,S ME\*NASA5.THCF/CFSIL,ME\*NASA5.THCF/CFSIL  
FOR S9A-07/13/72-20:58:22 (0.)

FUNCTION THCF ENTRY POINT 000043

STORAGE USED: CODE(1) 000047; DATA(0) 000031; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NI025  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000003 1F 0000 000023 INJP5 0000 R 000000 THCF 0000 R 000001 X1 0000 R 000002 X2

```
00101 1* FUNCTION THCF(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C THERMAL CONDUCTIVITY AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND
00101 5* C TEMPERATURE (R) OF DOW CORNING 200 SILICON OIL (1 CS)
00101 6* C UNITS BTU/(HR.FT.R)
00101 7* C TEMPERATURE .GE. 359.67 AND .LE. 859.67
00101 8* C
00103 9* DATA X1,X2 /0.070052,-2.2105E-05/
00106 10* THCF = X1*X2*T
00107 11* IF (T.GE.359.67.AND.T.LE.859.67) RETURN
00111 12* WRITE (6,1) T
00114 13* 1 FORMAT (1H0,61HTHERMAL CONDUCTIVITY OF SILICON OIL, TEMP. OUT OF R
00114 14* 1ANGE, T = ,F8.3,/)
00115 15* RETURN
00116 16* END
```

END OF COMPILATION: NO DIAGNOSTICS.

BHDG,P \*\*\*\*\* THCFN/FNAL \*\*\*\*\*

\*\*\*\*\* THCFN/FNAL \*\*\*\*\*

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QFOR,S ME\*NASAS,THCFN/FNAL,ME\*NASAS,THCFN/FNAL  
FOR S9A-07/13/72-20:58:25 (0,)

FUNCTION THCFN ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NIO2S  
0005 NERR3S

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011	1F	0000 R 000007	DT	0000	000031	INJPS	0000 R 000000	THCFN	0000 R 000010	THETA
0000 R	000006	T0	0000 R 000001	X1	0000 R 000002	X2	0000 R 000003	X3	0000 R 000004	X4	
0000 R	000005	X5									

```
00101      1*      FUNCTION THCFN(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      C      ALUMINUM 7075
00101      6*      C      UNITS BTU/(HR.FT.R)
00101      7*      C      TEMP. GE. 300.0.AND.LE. 1200.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4,X5 /88.5,13.0665,0.33275,-1.758,0.25375/,T0,DT /
00103     10*      1400.0,200.0/
00113     11*      THETA = (T-T0)/DT
00114     12*      THCFN = (((X5*THETA*X4)*THETA*X3)*THETA*X2)*THETA*X1
00115     13*      IF (T.GE.300.0.AND.T.LE.1200.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,58HTHERMAL CONDUCTIVITY OF ALUMINUM, TEMP. OUT OF RANG
00122     16*      1E, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* THCFN/FNBR \*\*\*\*\*

\*\*\*\*\* THCFN/FNBR \*\*\*\*\*

DATE 071372

PAGE 1

QFOR: S ME\*NASA5.THCFN/FNBR, ME\*NASA5.THCFN/FNBR  
FOR S9A-07/13/72-20:58:27 (0,)

FUNCTION THCFN ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NI025  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011 IF	0000 R 000007 DT	0000	000031 INJPS	0000 R 000000 THCFN	0000 R 000010 THETA
0000 R	000006 T0	0000 R 000001 X1	0000 R	000002 X2	0000 R 000003 X3	0000 R 000004 X4
0000 R	000005 X5					

```

00101      1*      FUNCTION THCFN(T)
00101      2*      C
00101      3*      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      BERYLIUM WITH 0.84-1.68 DEO
00101      6*      UNITS BTU/(HR.FT.R)
00101      7*      TEMP. GE. 400.0 AND LE. 1700.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4,X5 /108.863,-10.5643,0.82683,-0.20167,0.020167/,
00103     10*      1T0,DT /400.0,200.0/
00113     11*      THETA = (T-T0)/DT
00114     12*      THCFN = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00115     13*      IF (T.GE.400.0.AND.T.LE.1700.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,58HTHERMAL CONDUCTIVITY OF BERYLIUM, TEMP. OUT OF RANG
00122     16*      1E, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* THCFN/FNBU \*\*\*\*\*

\*\*\*\*\* THCFN/FNCU \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASAS,THCFN/FNCU,ME\*NASAS,THCFN/FNCU  
FOR S9A-07/13/72-20:58:29 (0,)

FUNCTION THCFN ENTRY POINT 000051

STORAGE USED: CODE(1) 000055; DATA(0) 000034; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ\$  
0004 NI02\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000007	1F	0000	R	000005	DT	0000	000026	INJPS	0000	R	000000	THCFN	0000	R	000006	THETA
0000	R	000004	T0	0000	R	000001	X1	0000	R	000002	X2	0000	R	000003	X3		

```
00101      1*      FUNCTION THCFN(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF COPPER
00101      5*      C                        UNITS BTU/(HR.FT.R)
00101      6*      C      TEMP. GE. 500.0 AND LE. 1800.0 R
00101      7*      C
00103      8*      DATA X1,X2,X3 /228.369,-2.62067,-0.04033/,T0,DT /600.0,200.0/
00111      9*      THETA = (T-T0)/DT
00112     10*      THCFN = (X3*THETA*THETA+X2)*THETA+X1
00113     11*      IF (T.GE.500.0.AND.T.LE.1800.0) RETURN
00115     12*      WRITE (6,1) T
00120     13*      1 FORMAT (1H0,56HTHERMAL CONDUCTIVITY OF COPPER, TEMP. OUT OF RANGE,
00120     14*      1 T = ,F8.3,/)
00121     15*      RETURN
00122     16*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* THCMP/MPAL \*\*\*\*\*

\*\*\*\*\* THCMP/MPAL \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASA5.THCMP/MPAL,ME\*NASA5.THCMP/MPAL  
FOR S9A-07/13/72-20:58:31 (0,)

FUNCTION THCMP ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NIO2\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011	1F	0000	R	000007	DT	0000	000031	INJPS	0000	R	000000	THCMP	0000	R	000010	THETA	
0000	R	000006	T0	0000	R	000001	X1	0000	R	000002	X2	0000	R	000003	X3	0000	R	000004.X4
0000	R	000005	X5															

```

00101      1*      FUNCTION THCMP(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      C      ALUMINUM 7075
00101      6*      C      UNITS BTU/(HR.FT.R)
00101      7*      C      TEMP. GE. 300.0.AND.LE. 1200.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4,X5 /88.5,13.0665,0.33275,-1.758,0.25375/,T0,DT /
00103     10*      1400.0,200.0/
00113     11*      THETA = (T-T0)/DT
00114     12*      THCMP = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00115     13*      IF (T.GE.300.0.AND.T.LE.1200.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,58HTHERMAL CONDUCTIVITY OF ALUMINUM, TEMP. OUT OF RANG
00122     16*      1E, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG,P \*\*\*\*\* THCMP/MPBR \*\*\*\*\*



\*\*\*\*\* THCMP/MPBR \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASA5.THCMP/MPBR,ME\*NASA5.THCMP/MPBR  
FOR S9A-07/13/72-20:58:34 (0.)

FUNCTION THCMP ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJS  
0004 NIO25  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011	1F	0000 R 000007	DT	0000	000031	INJPS	0000 R 000000	THCMP	0000 R 000010	THETA
0000 R	000006	T0	0000 R 000001	X1	0000 R 000002	X2	0000 R 000003	X3	0000 R 000004	X4	
0000 R	000005	X5									

```

00101      1*      FUNCTION THCMP(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      C      BERYLIUM WITH 0.84-1.68 BEO
00101      6*      C      UNITS BTU/(HR.FT.R)
00101      7*      C      TEMP. GE. 400.0 AND LE. 1700.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4,X5 /108.863,-10.5643,0.82683,-0.20167,0.020167/,
00103     10*      1T0,DT /400.0,200.0/
00113     11*      THETA = (T-T0)/DT
00114     12*      THCMP = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00115     13*      IF (T.GE.400.0.AND.T.LE.1700.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,58HTHERMAL CONDUCTIVITY OF BERYLIUM, TEMP. OUT OF RANG
00122     16*      1E, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

BHDG:P \*\*\*\*\* THCMP/MPCU \*\*\*\*\*

\*\*\*\*\* THCMP/MPCU \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASAS.THCMP/MPCU ME\*NASAS.THCMP/MPCU  
FOR S9A-07/13/72-20:58:36 (0,)

FUNCTION THCMP ENTRY POINT 000051

STORAGE USED: CODE(1) 000055; DATA(0) 000034; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NI025  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000007 1F	0000 R 000005 DT	0000	000026 INJP5	0000 R 000000 THCMP	0000 R 000006 THETA
0000 R	000004 T0	0000 R 000001 X1	0000 R	000002 X2	0000 R 000003 X3	

```
00101      1*      FUNCTION THCMP(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF COPPER
00101      5*      C                      UNITS BTU/(HR.FT.R)
00101      6*      C      TEMP. GE. 500.0 AND LE. 1800.0 R
00101      7*      C
00103      8*      DATA X1,X2,X3 /228.369,-2.62067,-0.04033/,T0,DT /600.0,200.0/
00111      9*      THETA = (T-T0)/DT
00112     10*      THCMP = (X3*THETA+THETA+X2)*THETA+X1
00113     11*      IF (T.GE.500.0.AND.T.LE.1800.0) RETURN
00115     12*      WRITE (6,1) T
00120     13*      1 FORMAT (1H0.56HTHERMAL CONDUCTIVITY OF COPPER, TEMP. OUT OF RANGE,
00120     14*      1 T = ,F8.3,/)
00121     15*      RETURN
00122     16*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG P \*\*\*\*\* THCTB/TBAL \*\*\*\*\*

\*\*\*\*\* THCTB/TBAL \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASA5.THCTB/TBAL,ME\*NASA5.THCTB/TBAL  
FOR S9A-07/13/72-20:58:40 (0.)

FUNCTION THCTB ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDJ\$  
0004 NIO2\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011	1F	0000 R 000007 DT	0000	000031	INJP\$	0000 R 000000 THCTB	0000 R 000010 THETA
0000	R 000006	T0	0000 R 000001 X1	0000	R 000002	X2	0000 R 000003 X3	0000 R 000004 X4
0000	R 000005	X5						

```

00101      1*      FUNCTION THCTB(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      C      ALUMINUM 7075
00101      6*      C      UNITS BTU/(HR.FT.R)
00101      7*      C      TEMP. GE. 300.0.AND.LE. 1200.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4,X5 /88.5,13.0665,0.33275,-1.758,0.25375/,T0,DT /
00103     10*      1400.0,200.0/
00113     11*      THETA = (T-T0)/DT
00114     12*      THCTB = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00115     13*      IF (T.GE.300.0.AND.T.LE.1200.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,58HTHERMAL CONDUCTIVITY OF ALUMINUM, TEMP. OUT OF RANG
00122     16*      1E, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDS,P \*\*\*\*\* THCTB/TBBR \*\*\*\*\*

\*\*\*\*\* THCTB/TBBR \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASA5.THCTB/TBBR ME\*NASA5.THCTB/TBBR  
FOR 59A-07/13/72-20:58:42 (0,)

FUNCTION THCTB ENTRY POINT 000054

STORAGE USED: CODE(1) 000060; DATA(0) 000037; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NI02\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011 1F	0000 R 000007 DT	0000	000031 INJPS	0000 R 000000 THCTB	0000 R 000010 THETA
0000	R 000006 T0	0000 R 000001 X1	0000	R 000002 X2	0000 R 000003 X3	0000 R 000004 X4
0000	R 000005 X5					

```

00101      1*      FUNCTION THCTB(T)
00101      2*      C
00101      3*      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF
00101      5*      C      BERYLIUM WITH 0.84-1.68 BE0
00101      6*      C      UNITS BTU/(HR.FT.R)
00101      7*      C      TEMP. GE. 400.0 AND LE. 1700.0 R
00101      8*      C
00103      9*      DATA X1,X2,X3,X4,X5 /108.863,-10.5643,0.82683,-0.20167,0.020167/,
00103     10*      1T0*DT /400.0,200.0/
00113     11*      THETA = (T-T0)/DT
00114     12*      THCTB = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1
00115     13*      IF (T.GE.400.0.AND.T.LE.1700.0) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0.58HTHERMAL CONDUCTIVITY OF BERYLIUM, TEMP. OUT OF RANG
00122     16*      1E, T = ,F8.3,/)
00123     17*      RETURN
00124     18*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG.P \*\*\*\*\* THCTB/TBCU \*\*\*\*\*

\*\*\*\*\* THCTB/TBCU \*\*\*\*\*

DATE 071372

PAGE

1

QFOR,S ME\*NASA5.THCTB/TBCU,ME\*NASA5.THCTB/TBCU  
FOR S9A-07/13/72-20:58:44 (0,)

FUNCTION THCTB ENTRY POINT 000051

STORAGE USED: CODE(1) 000055; DATA(0) 000034; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDU\$  
0004 NIO2\$  
0005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000007 1F	0000 R 000005 DT	0000	000026 INJP\$	0000 R 000000 THCTB	0000 R 000006 THETA
0000 R	000004 T0	0000 R 000001 X1	0000 R	000002 X2	0000 R	000003 X3

```
00101      1*      FUNCTION THCTB(T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      THERMAL CONDUCTIVITY AS A FUNCTION OF TEMPERATURE (R) OF COPPER
00101      5*      C                        UNITS BTU/(HR.FT.R)
00101      6*      C      TEMP. GE. 500.0 AND LE. 1800.0 R
00101      7*      C
00103      8*      DATA X1,X2,X3 /228.369,-2.62067,-0.04033/,T0,DT /600.0,200.0/
00111      9*      THETA = (T-T0)/DT
00112     10*      THCTB = (X3*THETA+THETA*X2)*THETA*X1
00113     11*      IF (T.GE.500.0.AND.T.LE.1800.0) RETURN
00115     12*      WRITE (6,1) T
00120     13*      1 FORMAT (1H0,56HTHERMAL CONDUCTIVITY OF COPPER, TEMP. OUT OF RANGE,
00120     14*      1 T = ,F8.3,/)
00121     15*      RETURN
00122     16*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

DHDG:P \*\*\*\*\* TK \*\*\*\*\*

\*\*\*\*\* TK \*\*\*\*\*

DATE 071372

PAGE 1

QFOR S ME\*NASAS.TK\*ME\*NASAS.TK  
FOR S9A-07/13/72-20:58:46 (0.)

FUNCTION TK ENTRY POINT 000211

STORAGE USED: CODE(1) 000245; DATA(0) 000062; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 ELAS  
0004 ALOG  
0005 NEXP6\$  
0006 SQRT  
0007 NWDU\$  
0010 NI02\$  
0011 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000112	13L	0001	000127	14L	0001	000174	15L	0000	000020	16F	0001	000201	17L	
0000	R	000003	B	0000	R	000001	C	0000	R	000017	CC	0000	R	000004	D
0000	R	000002	EE	0003	R	000000	ELAS	0000	R	000005	F	0000	R	000006	G
0000	I	000012	KG	0000	R	000007	P	0000	R	000010	Q	0000	R	000016	R
0000	R	000000	TK	0000	R	000013	TN	0000	R	000014	V	0000	R	000011	S

```

00101      1*      FUNCTION TK(GAMMA,A,BETA,DENSM,THETA,PHI,AN,ALPHA,VELM,
00101      2*      1      PO,TAU,DENST,W,TNN,AMAN,TIN,ROUT)
00101      3*      C
00101      4*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      5*      C      THE THICKNESS OF THE PROTECTION LAYER TK ,IN INCHES
00101      6*      C
00103      7*      DATA C/1.2407013/
00105      8*      EE = 1./(3.*BETA)
00106      9*      B = GAMMA*A
00107     10*      D = 0.0436332*TNN*W
00110     11*      F = (ALPHA*TAU/(-ALOG(P0)))*EE
00111     12*      G = (1./DENSM)**.33333
00112     13*      P = (DENSM/DENST)**PHI
00113     14*      Q = (VEL4/(12.0*SQRT(ELAS(TIN)*32.174/DENST)))*THETA
00114     15*      S = (2./(3.*AN*THETA*BETA+2.))*EE
00115     16*      KG = 0
00116     17*      TN = 0.5
00117     18*      13 KG = KG+1
00120     19*      IF (KG.EQ.1) GO TO 14
00122     20*      IF (KG.GT.10) GO TO 15
00124     21*      TN = TK
00125     22*      14 V = (D*(ROUT+TN)+AMAN)
00126     23*      E = V*EE
00127     24*      R = B*C*E*F*G*P*Q*S
00130     25*      TK = TN-((TN-R)/(1.-(EE*R*D/V)))

```

\*\*\*\*\* TK \*\*\*\*\*

DATE 071372

PAGE 2

```
00131 26*      CC = TK/TN-1.  
00132 27*      IF (ABS(CC).LT..0001) GO TO 17  
00134 28*      GO TO 13  
00135 29*      15 WRITE (6,16)  
00137 30*      16 FORMAT (29HEQN FOR THICK DOESNT CONVERGE)  
00140 31*      17 RETURN  
00141 32*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

DHOGP \*\*\*\*\* TNH \*\*\*\*\*

\*\*\*\*\* TNH \*\*\*\*\*

DATE 071372

PAGE 1

QFOR: S ME\*NASA5.TNH, ME\*NASA5.TNH  
FOR S9A-07/13/72-20:58:48 (0,)

FUNCTION TNH ENTRY POINT 000056

STORAGE USED: CODE(1) 000064; DATA(0) 000030; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 CPAIR  
0004 ENTAIR  
0005 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000037	10L	0001	000004	1066	0001	000041	30L	0003 R	000000	CPAIR	0000 R	000014	DFTN
0004 R	000000	ENTAIR	0000 R	000015	FTN	0000	000020	INJPS	0000 I	000013	N	0000 R	000001	TN
0000 R	000000	TNH												

```
00101 1*      FUNCTION TNH(ENT)
00101 2*      C
00101 3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4*      C      TEMPERATURE OF THE ATMOSPHERE AS A FUNCTION OF ENTHALPY (BTU/LBM)
00101 5*      C      UNITS R
00101 6*      C
00103 7*      DIMENSION TN(10)
00104 8*      TN(1)  = 2500.0
00105 9*      DO 10 N = 1,10
00110 10*     DFTN   = CPAIR(TN(N))
00111 11*     FTN    = ENTAIR(TN(N))-ENT
00112 12*     TN(N+1) = TN(N)-FTN/DFTN
00113 13*     IF (N.EQ. 1) GO TO 10
00115 14*     IF (ABS(TN(N+1)-TN(N)) .LE. 2.0) GO TO 30
00117 15*     10 CONTINUE
00121 16*     30 TNH  = TN(N+1)
00122 17*     RETURN
00123 18*     END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG.P \*\*\*\*\* TRMATX \*\*\*\*\*



\*\*\*\*\* TRMATX \*\*\*\*\*

DATE 071372

PAGE

1

QFOR,S ME\*NASA5,TRMATX,ME\*NASA5,TRMATX  
FOR S9A-07/13/72-20:58:50 (0,)

SUBROUTINE TRMATX ENTRY POINT 000262

STORAGE USED: CODE(1) 000275; DATA(0) 000051; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 GRD 003721  
0004 ABSRST 000601

EXTERNAL REFERENCES (BLOCK, NAME)

0005 NERR3%

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000003	106G	0001	000004	111G	0001	000021	120G	0001	000022	123G	0001	000053	134G
0001	000056	137G	0001	000066	142G	0001	000121	153G	0001	000137	162G	0001	000145	166G
0001	000154	171G	0001	000211	201G	0001	000212	204G	0001	000217	211G	0001	000224	216G
0004	R	000000	ALPHFN	0004	R	000226	ALPHMP	0003	R	003465	CSF	0000	R	000005
0003		003472	DXX21	0003		003463	DZMFN	0003		003410	EBFN	0003		003441
0004		000512	EMITMP	0003		003467	EXTIFN	0003		003471	EXTIMP	0003		003466
0004		000517	EXTVCT	0000	I	000000	I	0000		000027	INJPS	0000	I	000001
0000	I	000002	K	0000	I	000004	L	0003		003461	LCT	0003		003462
0000	I	000003	M1	0000	I	000006	M2	0003	R	003473	SS	0003	R	003670
0003		003446	XX2	0003		003453	ZZ2							

```

00101      1*      SUBROUTINE TRMATX
00101      2*      C
00101      3*      C THIS SUBROUTINE COMPUTES :
00101      4*      C THE TRANSFER MATRIX NECESSARY FOR THE CALCULATION OF THE INCIDENT
00101      5*      C NET RADIANT FLUX
00101      6*      C
00103      7*      COMMON /GRD/ TRMTX(30,60),EBFN(5,5),EBMP(5),XX2(5),ZZ2(5),
00103      8*      1 MCVRD,LCT,LTT,DZMFN,
00103      9*      2 DXX2,CSF,EXTSFN,EXTIFN,EXTSMP,EXTIMP,DXX21,SS(5,5,5)
00103     10*      3 ,SSTT(5,5)
00104     11*      COMMON /ABSRST/ALPHFN(5,5,6),ALPHMP(5,31),EMITFN(5,5),EMITMP(5),
00104     12*      1 EXTVCT(50)
00105     13*      DO 2 I=1,25
00110     14*      DO 1 J=1,25
00113     15*      1 TRMTX(I,J)=0.0
00115     16*      2 TRMTX(I,I)=1.0
00117     17*      DO 4 I=26,30
00122     18*      DO 3 J=26,30
00125     19*      3 TRMTX(I,J)=-(1.0-ALPHMP(I-25,J))*SSTT(I-25,J-25)*0.25
00127     20*      TRMTX(I,26)=TRMTX(I,26)/2.0
00130     21*      TRMTX(I,30)=TRMTX(I,30)/2.0

```

\*\*\*\*\* TRMATX \*\*\*\*\*

```

00131 22*      4 TRMTX(I,I)=1.0+TRMTX(I,I)
00133 23*      DO 5 I=1,5
00136 24*      DO 5 J=1,5
00141 25*      DO 5 K=1,5
00144 26*      M1 =25+K
00145 27*      L  =(I-1)*5 +J
00146 28*      5 TRMTX(L,M1) =-(1.0-ALPHFN(I,J,K))*SS(I,J,K)/4.0
00152 29*      DO 6 I=1,25
00155 30*      TRMTX(I,26) =TRMTX(I,26)/2.0
00156 31*      6 TRMTX(I,30) =TRMTX(I,30)/2.0
00160 32*      CSFXZ = CSF*OXX2/4.0
00161 33*      DO 7 I=1,5
00164 34*      M2 =25+I
00165 35*      DO 7 J=1,5
00170 36*      DO 7 K=1,5
00173 37*      JK  =(J-1)*5+K
00174 38*      7 TRMTX(M2,JK)=-{(1.0-ALPHMP(I,JK))*SS(J,K,I)*CSFXZ
00200 39*      DO 10 I=26,30
00203 40*      DO 8 K=21,25
00206 41*      8 TRMTX(I,K) =TRMTX(I,K)/2.0
00210 42*      DO 9 J=1,21,5
00213 43*      9 TRMTX(I,J) =TRMTX(I,J)/2.0
00215 44*      DO 10 J=5,25,5
00220 45*      10 TRMTX(I,J) =TRMTX(I,J)/2.0
00223 46*      RETURN
00224 47*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHD6.P \*\*\*\*\* TRNSPT \*\*\*\*\*

\*\*\*\*\* TRNSPT \*\*\*\*\*

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QFOR: S ME\*NASA5. TRNSPT: ME\*NASA5. TRNSPT  
FOR S9A-07/13/72-20:58:53 (0.)

SUBROUTINE TRNSPT ENTRY POINT 000143

STORAGE USED: CODE(1) 000171; DATA(0) 000040; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NEXP6\$  
0004 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000062 10L 0001 000106 15L 0001 000050 5L 0000 000030 INJP\$ 0000 R 000000 NUL  
0000 R 000001 XX

00101	1*		SUBROUTINE TRNSPT (RE,PRL,DELTA,FR,FNU)	01
00101	2*	C		
00101	3*	C	THIS SUBROUTINE COMPUTES :	
00101	4*	C	THE NON-DIMENSIONAL FRICTION FACTOR FR ,AND THE NON-DIMENSIONAL	
00101	5*	C	CONVECTIVE FILM COEFFICIENT FNU	
00101	6*	C		
00103	7*		REAL NUL	02
00103	8*	C		03
00104	9*		IF (RE.LT.2300.0) GO TO 10	04
00106	10*		NUL = 0.116*(RE**0.667-125.0)*PRL**0.3333*(1.0+DELTA**0.667)	05
00107	11*		IF (RE.GT.1.0E+06) GO TO 5	06
00111	12*		FR = (0.0054+0.396/RE**0.3)/DELTA	07
00112	13*		GO TO 15	08
00113	14*	5	FR = (0.0032+0.222/RE**0.237)/DELTA	09
00114	15*		GO TO 15	010
00115	16*	10	FR = 64.0/(RE*DELTA)	011
00116	17*		XX = RE*PRL*DELTA	012
00117	18*		NUL = 3.65+0.0668*XX/(1.0+0.045*XX**0.667)	
00120	19*	15	IF (PRL.LT.0.1) NUL = 6.5+0.025*(RE*PRL)**0.8	014
00122	20*		FNU = 4.0*NUL/(DELTA*RE*PRL)	015
00123	21*		RETURN	016
00124	22*		END	017

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* TTIP \*\*\*\*\*

\*\*\*\*\* TTIP \*\*\*\*\*

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QFOR.S ME\*NASAS.TTIP,ME\*NASAS.TTIP  
FOR S9A-07/13/72-20:58:55 (0.)

SUBROUTINE TTIP ENTRY POINT 000043

STORAGE USED: CODE(1) 000060; DATA(0) 000101; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 POLY  
0004 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000005	1166	0001	000007	1216	0000 R 000043 AA	0000 R 000000 CC	0000 R 000052 COEF
0000	I	000057	I	0000	000067	INJP\$	0000 I 000060 J	0003 R 000000 POLY

```
00101 1* SUBROUTINE TTIP (NC,TSOTB,TLOTB)
00103 2* DIMENSION CC(5,7),AA(7),COEF(5)
00104 3* REAL NC
00105 4* DATA(CC(1,2),I=1,5)/-.4815,-.8920814,4.894438,-14.89583,10.39930/,
00105 5* 1 (CC(1,3),I=1,5)/.5365006,.7351966,-12.35190,40.88471,-30.65782/,
00105 6* 2 (CC(1,4),I=1,5)/-.440667,.2304286,11.64157,-44.56229,35.26028/,
00105 7* 3 (CC(1,5),I=1,5)/.212667,-.4978796,-5.648548,24.37024,-19.93047/,
00105 8* 4 (CC(1,6),I=1,5)/-.05333347,.198336,1.40554,-6.58330,5.48609/,
00105 9* 5 (CC(1,7),I=1,5)/.00533335,-.02525962,-.139998,.6925878,-.5833303/
00114 10* AA(1) = 1.0
00115 11* DO 20 I=2,7
00120 12* DO 10 J=1,5
00123 13* 10 COEF(J) = CC(J,I)
00125 14* 20 AA(I) = POLY (5,COEF,TSOTB)
00127 15* TLOTB = POLY (7,AA,NC)
00130 16* RETURN
00131 17* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG.P \*\*\*\*\* TTIPS \*\*\*\*\*

\*\*\*\*\* TTIPS \*\*\*\*\*

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PAGE 1

QFOR S ME+NASA5.TTIPS+ME+NASA5.TTIPS  
FOR S9A-07/13/72-20:58:57 (0.)

SUBROUTINE TTIPS ENTRY POINT 000345

STORAGE USED: CODE(1) 000366; DATA(0) 000110; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 ADBH 000352  
0004 QIN 001610

EXTERNAL REFERENCES (BLOCK, NAME)

0005 NUSA  
0006 EFFICY  
0007 TTIP  
0010 EXP  
0011 NEXP65  
0012 NPRT5  
0013 NI025  
0014 NSTOP5  
0015 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000041	105F	0001	000303	110L	0001	000136	136G	0001	000120	20L	0001	000126	30L		
0000	R	000033	AA	0003	R	000001	AL		0000	R	000021	BB1	0000	R	000006	COEF
0003	R	000161	CONFL	0003	R	000173	CONFNA		0003	R	000147	CP	0000	R	000013	C1
0000	R	000030	C4	0000	R	000020	C5		0003	R	000000	D	0000	R	000035	OFNTB
0000	R	000036	DTB	0003	R	000326	DTL		0003	R	000244	EFF	0003	R	000205	EMIS
0000	R	000027	ETAPR2	0000	R	000024	ETA1		0000	R	000026	ETA2	0000	R	000034	FNTB
0003	R	000231	HAB	0000	R	000064	INJPS		0000	I	000017	L	0003	R	000064	M
0000	R	000003	NCPR2	0000	R	000000	NC1		0000	R	000001	NC2	0004	I	001605	NSRD
0004	R	001604	NTM	0005	R	000000	NUSA		0000	R	000004	PI	0003	R	000052	PIN
0004	R	000454	QIFN	0004	R	001274	QITB		0003	R	000314	QOUT	0004	R	000144	QSFN
0004	R	001606	QTO	0000	R	000010	REY		0000	R	000005	SIGMA	0003	R	000217	TB
0000	R	000032	TB4	0003	R	000026	TH		0003	R	000040	TIN	0003	R	000270	TL
0000	R	000040	TL0TB2	0004	R	000000	TM		0003	R	000340	TOUT	0000	R	000023	TS0TB
0000	R	000016	TSTR	0000	R	000015	TS4		0004	R	001607	TX	0000	R	000012	U
													0001	R	000014	C2
													0000	R	000011	DOL
													0000	R	000025	ETAPRI
													0003	R	000013	H
													0000	R	000002	NCPR1
													0003	R	000076	NT
													0000	R	000007	PR
													0004	R	000764	QSTB
													0000	R	000031	TB3
													0000	R	000037	TL0TB1
													0003	R	000077	TSTAR
													0003	R	000135	VSC

00101	1*		SUBROUTINE TTIPS(IT,ITST,H1,H2,TL1,TL2)
00103	2*		PARAMETER NTP = 10
00104	3*		PARAMETER NTP1 = NTP+1
00105	4*		REAL M
00106	5*		COMMON /AOBH/ D,AL(NTP),H(NTP1),TH(NTP),TIN(NTP),PIN(NTP),
00106	6*	1	M(NTP),NT,TSTAR(NTP,3),VSC(NTP),CP(NTP),CONFL(NTP),
00106	7*	2	CONFNA(NTP),EMIS(NTP),TB(NTP),HAB(NTP1),EFF(NTP,2),TL(NTP,2),
00106	8*	3	QOUT(NTP),DTL(NTP),TOUT(NTP)
00107	9*		COMMON /QIN/ TM(100),QSFN(100,2),QIFN(100,2),QSTB(100,2),

\*\*\*\*\* TTIPS \*\*\*\*\*

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```

00107 10* 1 QITB(100,2),NTM,NSRD,QTO,TX
00110 11* REAL NUSA,NC1,NC2,NCPR1,NCPR2
00111 12* DATA PI/3.1415927/
00113 13* DATA SIGMA/1.714E-9/
00115 14* COEF=EMIS(IT)*SIGMA/(CONFNA(IT)*TH(IT))
00116 15* PR=VSC(IT)*CP(IT)/CONFL(IT)
00117 16* REY=48*M(IT)/(PI*D*VSC(IT))
00120 17* DOL=D/(12*AL(IT))
00121 18* U=PI*CONFL(IT)*AL(IT)*NUSA(REY,PR,DOL)/(M(IT)*CP(IT))
00122 19* C1=M(IT)*CP(IT)*(1-EXP(-U))
00123 20* C2=NSRD*EMIS(IT)*SIGMA*AL(IT)/12
00124 21* TB(IT)=.9*TIN(IT)
00125 22* IF(NSRD.NE.2)GO TO 20
00127 23* TS4=(TSTAR(IT,1)**4+TSTAR(IT,2)**4)/2
00130 24* TSTR=TS4**.25
00131 25* GO TO 30
00132 26* 20 TS4=TSTAR(IT,ITST)**4
00133 27* TSTR=TSTAR(IT,ITST)
00134 28* 30 CONTINUE
00135 29* DO 100 L=2,10
00140 30* C5=TB(IT)**2*COEF/12
00141 31* BB1=C5*H1**2
00142 32* BB2=C5*H2**2
00143 33* NC1=TB(IT)*BB1
00144 34* NC2=TB(IT)*BB2
00145 35* NCPR1=3*BB1
00146 36* NCPR2=3*BB2
00147 37* TSOTB=TSTR/TB(IT)
00150 38* CALL EFFICY(NC1,NCPR1,TSOTB,TB(IT),ETA1,ETAPR1)
00151 39* EFF(IT,1) = ETA1*100.0
00152 40* CALL EFFICY(NC2,NCPR2,TSOTB,TB(IT),ETA2,ETAPR2)
00153 41* EFF(IT,2) = ETA2*100.0
00154 42* C4=ETA1*H1+ETA2*H2
00155 43* TB3=TB(IT)**3
00156 44* TB4=TB3*TB(IT)
00157 45* AA=TB4-TS4
00160 46* FNTB=C1*(TIN(IT)-TB(IT))-C2*C4*AA
00161 47* DFNTB=-C1-C2*(4*TB3*C4+AA*(ETAPR1*H1+ETAPR2*H2))
00162 48* DTB=FNTB/DFNTB
00163 49* TB(IT)=TB(IT)-DTB
00164 50* IF(ABS(DTB).LE.0.005)GO TO 110
00166 51* 100 CONTINUE
00170 52* PRINT 105
00172 53* 105 FORMAT(' BASE TEMPERATURE DOES NOT CONVERGE')
00173 54* STOP
00174 55* 110 CONTINUE
00175 56* CALL TTIP(NC1,TSOTB,TLOTB1)
00176 57* CALL TTIP(NC2,TSOTB,TLOTB2)
00177 58* TL1=TLOTB1*(TB(IT)-TSTR)+TSTR
00200 59* TL2=TLOTB2*(TB(IT)-TSTR)+TSTR
00201 60* RETURN
00202 61* END

```

END OF COMPILATION: NO DIAGNOSTICS.

\*\*\*\*\* TTIPS \*\*\*\*\*

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QNDG:P \*\*\*\*\* VISC/CFFC43 \*\*\*\*\*

\*\*\*\*\* VISC/CFFC43 \*\*\*\*\*

DATE 071372

PAGE

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QFOR:5 ME\*NASAS.VISC/CFFC43,ME\*NASAS.VISC/CFFC43  
FOR S9A-07/13/72-20:59:00 (0.)

FUNCTION VISC ENTRY POINT 000032

STORAGE USED: CODE(1) 000036; DATA(0) 000021; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 EXP  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 000012 INJPS	0000 R 000007 THETA	0000 R 000005 T0	0000 R 000000 VISC
0000 R 000010 VNU	0000 R 000001 X1	0000 R 000002 X2	0000 R 000003 X3	0000 R 000004 X4

```
00101      1*      FUNCTION VISC(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      VISCOSITY AS A FUNCTION OF DENSITY (SLUG/CU.FT)
00101      5*      C      AND TEMPERATURE (R) OF FC-43
00101      6*      C      UNITS SLUG/(FT.HR)
00101      7*      C
00103      8*      DATA X1,X2,X3,X4 /2.76,-2.043483,0.362,-0.034717/,T0,DT /439.67,90
00103      9*      1.0/
00112     10*      THETA = (T-T0)/DT
00113     11*      VNU = EXP(((X4*THETA+X3)*THETA+X2)*THETA+X1)
00114     12*      VISC = VNU*RHO*0.03875
00115     13*      RETURN
00116     14*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHD6,P \*\*\*\*\* VISC/CFFC75 \*\*\*\*\*



\*\*\*\*\* VISC/CFFC75 \*\*\*\*\*

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QFOR: S ME\*NASA5.VISC/CFFC75,ME\*NASA5.VISC/CFFC75  
FOR S9A-07/13/72-20:59:03 (0.)

FUNCTION VISC ENTRY POINT 000032

STORAGE USED: CODE(1) 000036; DATA(0) 000021; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 EXP  
0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000006 DT	0000 000012 INJPS	0000 R 000007 THETA	0000 R 000005 T0	0000 R 000000 VISC
0000 R 000010 VNU	0000 R 000001 X1	0000 R 000002 X2	0000 R 000003 X3	0000 R 000004 X4

```
00101      1*      FUNCTION VISC(RHO,T)
00101      2*      C
00101      3*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      4*      C      VISCOSITY AS A FUNCTION OF DENSITY (SLUG/CU.FT)
00101      5*      C      AND TEMPERATURE (R) OF FC-75
00101      6*      C      UNITS SLUG/(FT.HR)
00101      7*      C
00103      8*      DATA X1,X2,X3,X4 /1.639,-1.312933,0.25265,-0.02471667/,T0,DT / 409
00103      9*      1.67,60.0/
00112     10*      THETA = (T-T0)/DT
00113     11*      VNU = EXP(((X4*THETA+X3)*THETA+X2)*THETA+X1)
00114     12*      VISC = VNU*RHO*0.03875
00115     13*      RETURN
00116     14*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* VISC/CFHE \*\*\*\*\*

\*\*\*\*\* VISC/CFHE \*\*\*\*\*

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DFOR S ME\*NASAS.VISC/CFHE ME\*NASAS.VISC/CFHE  
FOR S9A-07/13/72-20:59:04 (0.)

FUNCTION VISC ENTRY POINT 000016

STORAGE USED: CODE(1) 000022; DATA(0) 000011; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NEXP6\$  
0004 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000003 INJPS 0000 R 000000 VISC

00101 1\* FUNCTION VISC(RHO,T)  
00101 2\* C  
00101 3\* C THIS FUNCTION SUBPROGRAM COMPUTES :  
00101 4\* C VISCOSITY AS A FUNCTION OF DENSITY (SLUG/CU.FT)  
00101 5\* C AND TEMPERATURE (R) OF HELIUM  
00101 6\* C UNITS SLUG/(FT.HR)  
00101 7\* C  
00103 8\* VISC = 2.58394E-05\*T\*\*0.647  
00104 9\* RETURN  
00105 10\* END

END OF COMPILATION: NO DIAGNOSTICS.

QMDG,P \*\*\*\*\* VISC/CFNAK \*\*\*\*\*

\*\*\*\*\* VISC/CFNAK \*\*\*\*\*

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PAGE 1

QFOR S ME\*NASA5.VISC/CFNAK ME\*NASA5.VISC/CFNAK  
FOR S9A-07/13/72-20:59:07 (0.)

FUNCTION VISC ENTRY POINT 000025

STORAGE USED: CODE(1) 000027; DATA(0) 000017; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 R 000007 DT	0000 000012 INJP\$	0000 R 000010 THETA	0000 R 000006 TO	0000 R 000000 VISC
0000 R 000001 X1	0000 R 000002 X2	0000 R 000003 X3	0000 R 000004 X4	0000 R 000005 X5

```
00101 1* FUNCTION VISC(RHO,T)
00101 2* C
00101 3* C THIS FUNCTION SUBPROGRAM COMPUTES :
00101 4* C VISCOSITY AS A FUNCTION OF DENSITY (SLUG/CU.FT)
00101 5* C AND TEMPERATURE (R) OF NAK 78.6
00101 6* C UNITS SLUG/(FT.HR)
00101 7* C
00103 8* DATA X1,X2,X3,X4,X5 /1.316,-0.896667,0.419833,-0.102833,0.009667/,
00103 9* 1T0-DT /659.67,300.0/
00113 10* THETA = (T-T0)/DT
00114 11* VISC = (((X5*THETA+X4)*THETA+X3)*THETA+X2)*THETA+X1)/32.174
00115 12* RETURN
00116 13* END
```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG P \*\*\*\*\* VISC/CFSIL \*\*\*\*\*

\*\*\*\*\* VISC/CFSIL \*\*\*\*\*

DATE 071372

PAGE 1

QFOR:5 ME\*NASAS.VISC/CFSIL,ME\*NASAS.VISC/CFSIL  
FOR S9A-07/13/72-20:59:09 (0,)

FUNCTION VISC ENTRY POINT 000061

STORAGE USED: CODE(1) 000067; DATA(0) 000040; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 EXP  
0004 NWDJ5  
0005 NIO25  
0006 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000011 IF	0000 R 000006 DT	0000	000030 INJP5	0000 R 000007 THETA	0000 R 000005 TO
0000 R	000000 VISC	0000 R 000010 VNU	0000 R	000001 X1	0000 R 000002 X2	0000 R 000003 X3
0000 R	000004 X4					

```

00100      1*      C
00100      2*      C      THIS FUNCTION SUBPROGRAM COMPUTES :
00101      3*      C      FUNCTION VISC(RHO,T)
00101      4*      C      VISCOSITY AS A FUNCTION OF DENSITY (SLUG/CU.FT) AND
00101      5*      C      TEMPERATURE (R) OF DOW CORNING 200 SILICON OIL (1 CS)
00101      6*      C      UNITS SLUG/(FT.HR)
00101      7*      C      TEMPERATURE .6E. 359.67 AND .LE. 859.67
00101      8*      C
00103      9*      DATA X1,X2,X3,X4 /0.683,-1.0845,0.3065,-0.04/,T0,DT /459.67,100.0/
00112     10*      THETA = (T-T0)/DT
00113     11*      VNU = EXP(((X4*THETA+X3)*THETA+X2)*THETA+X1)
00114     12*      VISC = VNU*RHO*0.03875
00115     13*      IF (T.GE.360.67.AND.T.LE.860.67) RETURN
00117     14*      WRITE (6,1) T
00122     15*      1 FORMAT (1H0,50H VISCOSITY OF SILICON OIL, TEMP. OUT OF RANGE, T = ,
00122     16*      1F8.3,/)
00123     17*      RETURN
00124     18*      END

```

END OF COMPILATION: NO DIAGNOSTICS.

QHDG:P \*\*\*\*\* YINT \*\*\*\*\*

\*\*\*\*\* YINT \*\*\*\*\*

DATE 071372

PAGE 1

QFOR,S ME\*NASAS.YINT,ME\*NASAS.YINT  
FOR S9A-07/13/72-20:59:11 (0.)

FUNCTION YINT ENTRY POINT 000265

STORAGE USED: CODE(1) 000315; DATA(0) 000136; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDUS  
0004 NIO2S  
0005 NERR3S

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000075	1136	0001	000146	1306	0001	000172	1406	0001	000201	1436	0001	000215	1506
0001	000106	2L	0001	000230	6L	0000	000060	7F	0001	000240	8L	0000	000072	9F
0000	I	000053	I	0000	000111	INJPS	0000	I	000057	J	0000	I	000056	L
0000	I	000051	MM	0000	I	000054	M0	0000	I	000052	M1	0000	R	000013
0000	R	000001	XX	0000	R	000000	YINT	0000	R	000037	YY	0000	R	000025
														X0

```

00101      1*      FUNCTION YINT(X,Y,N,M,P)
00101      2*      C
00101      3*      C THIS SUBROUTINE COMPUTES :
00101      4*      C AITKEN INTERPOLATION
00101      5*      C X ARRAY OF ABSCISSAE
00101      6*      C Y ARRAY OF ORDINATES
00101      7*      C N NUMBER OF ELEMENTS IN ARRAYS X AND Y
00101      8*      C P X-VALUE OF INTERPOLATION
00101      9*      C M DEGREE OF POLYNOMIAL PASSING THROUGH Y(X) IN THE VICINITY OF P
00101     10*      C
00103     11*      DIMENSION X(N),Y(N),XX(10),XP(10),X0(10),YY(10)
00104     12*      IF(P.LT.(2.*X(1)-X(2)).OR.P.GT.(2.*X(N)-X(N-1))) GO TO 8
00106     13*      MM = MAX(MIN(M,10),2)
00107     14*      IF (N.LT.MM.OR.N.LE.2) GO TO 6
00111     15*      M1 = MM/2
00112     16*      DO 1 I = 1,N
00115     17*      IF (P .LE. X(I)) GO TO 2
00117     18*      1 CONTINUE
00121     19*      2 M0 = I-M1
00122     20*      ME = M0+MM-1
00123     21*      IF (M0 .LT. 1) M0 = 1
00125     22*      IF (ME .GT. N) ME = N-MM+1
00127     23*      DO 3 I = 1,MM
00132     24*      L = M0+I-1
00133     25*      XP(I) = X(L)-P
00134     26*      X0(I) = X(L)
00135     27*      3 YY(I) = Y(L)
00137     28*      DO 5 I = 2,MM
00142     29*      DO 4 J = I,MM

```

\*\*\*\*\* YINT \*\*\*\*\*

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```
00145      30*      4 XX(J) = (YY(I-1)*XP(J)-YY(J)*XP(I-1))/(XQ(J)-XQ(I-1))
00147      31*      DO 5 J = 1,MM
00152      32*      5 YY(J) = XX(J)
00155      33*      YINT = YY(MM)
00156      34*      RETURN
00157      35*      6 WRITE (6,7)
00161      36*      7 FORMAT (/,'1X,45HINTERPOL. IS IMPOSSIBLE, DATA ARRAY TOO SMALL)
00162      37*      RETURN
00163      38*      8 WRITE (6,9) P
00166      39*      9 FORMAT (/,'1X,29HINTERPOL. IS IMPOSSIBLE, P = ,E12.4,12HOUT OF RANG
00166      40*      1E)
00167      41*      RETURN
00170      42*      END
```

END OF COMPILATION: NO DIAGNOSTICS.

QMDG:P \*\*\*\*\* CFFC43 \*\*\*\*\*

\*\*\*\*\* CFFC43 \*\*\*\*\*

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QELT,L ME\*NASA5.CFFC43

ELT 005-07/13-20:59

000001 000 CLASS

CFFC43

000002 000 IN

BETA,CAPPA,CPF,HFL,RHOF,THCF,VISC

000003 000 DEF

BETA,CAPPA,CPF,HFL,RHOF,THCF,VISC

000004 000 END

QHD6,P \*\*\*\*\* CFFC75 \*\*\*\*\*

\*\*\*\*\* CFFC75 \*\*\*\*\*

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1

QELT,L ME\*NASA5.CFFC75

ELT 005-07/13-20:59

000001 000 CLASS

CFFC75

000002 000 IN

BETA,CAPPA,CPF,HFL,RHOF,THCF,VISC

000003 000 DEF

BETA,CAPPA,CPF,HFL,RHOF,THCF,VISC

000004 000 END

QHDG,P \*\*\*\*\* CFHE \*\*\*\*\*



\*\*\*\*\* CFHE \*\*\*\*\*

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QELT,L ME\*NASA5.CFHE

ELT 005-07/13-20:59

000001 000 CLASS

CFHE

000002 000 IN

BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC

000003 000 DEF

BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC

000004 000 END

QHDG,P \*\*\*\*\* CFNAK \*\*\*\*\*

\*\*\*\*\* CFNAK \*\*\*\*\*

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QELT.L ME\*NASAS.CFNAK

ELT 005-07/13-20:59

000001 000 CLASS

CFNAK

000002 000 IN

BETA.CAPPA.CPF.HFL.PF.RHOF.THCF.VISC

000003 000 DEF

BETA.CAPPA.CPF.HFL.PF.RHOF.THCF.VISC

000004 000 END

QHDG.P \*\*\*\*\* CFSIL \*\*\*\*\*

\*\*\*\*\* CFSIL \*\*\*\*\*

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1

QELT,L ME\*NASAS.CFSIL

ELT 005-07/13-20:59

000001 000 CLASS

CFSIL

000002 000 IN

BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC

000003 000 DEF

BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC

000004 000 END

QHDG,P \*\*\*\*\* FNAL \*\*\*\*\*

\*\*\*\*\* FNAL \*\*\*\*\*

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QELT,L ME\*NASAS,FNAL

ELT 005-07/13-20:59

000001	000	CLASS	FNAL
000002	000	IN	CPFN,DTHCFN,THCFN
000003	000	DEF	CPFN,DTHCFN,THCFN
000004	000	END	

QHDG,P \*\*\*\*\* FNBR \*\*\*\*\*

\*\*\*\*\* FNBR \*\*\*\*\*

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1

QELT,L ME\*NASA5,FNBR

ELT 005-07/13-20:59

000001 000 CLASS

FNBR

000002 000 IN

CPFN,DTHCFN,THCFN

000003 000 DEF

CPFN,DTHCFN,THCFN

000004 000 END

QHDG,P \*\*\*\*\* FNCU \*\*\*\*\*

\*\*\*\*\* FNCU \*\*\*\*\*

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PAGE 1

DELTA L ME\*NASA5.FNCU

ELT 005-07/13-20:59

000001 000 CLASS

000002 000 IN

000003 000 DEF

000004 000 END

FNCU

CPFN,DTHCFN,THCFN

CPFN,DTHCFN,THCFN

QHD6,P \*\*\*\*\* MPAL \*\*\*\*\*

\*\*\*\*\* MPAL \*\*\*\*\*

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PAGE 1

QELT:L ME:NASA5.MPAL

ELT 005-07/13-20:59

000001 000 CLASS

MPAL

000002 000 IN

CPMP,OTHCMP,ELAS,THCMP

000003 000 DEF

CPMP,OTHCMP,ELAS,THCMP

000004 000 END

QHDG:P \*\*\*\*\* MPBR \*\*\*\*\*

\*\*\*\*\* MPBR \*\*\*\*\*

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1

DELTA ME\*NASAS.MPBR

ELT 005-07/13-20:59

000001 000 CLASS

000002 000 IN

000003 000 DEF

000004 000 END

MPBR

CPMP,DTHCMP,ELAS,THCMP

CPMP,DTHCMP,ELAS,THCMP

QHDG:P \*\*\*\*\* MPCU \*\*\*\*\*



\*\*\*\*\* MPCU \*\*\*\*\*

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QELT:L ME\*NASAS.MPCU  
ELT 005-07/13-20:59

000001	000	CLASS	MPCU
000002	000	IN	CPMP,DTHCMP,ELAS,THCMP
000003	000	DEF	CPMP,DTHCMP,ELAS,THCMP
000004	000	END	

QHDG:P \*\*\*\*\* SC293 \*\*\*\*\*

\*\*\*\*\* SCZ93 \*\*\*\*\*

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DELTA ME+NASA5.SCZ93

ELT 005-07/13-20:59

000001	000	CLASS	SCZ93
000002	000	IN	AVGEMT+EMIT
000003	000	DEF	AVGEMT+EMIT
000004	000	END	

QHDG,P \*\*\*\*\* TBAL \*\*\*\*\*

\*\*\*\*\* TBAL \*\*\*\*\*

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1

DELT,L ME\*NASAS,TBAL  
ELT 005-07/13-20:59

000001	000	CLASS	TBAL
000002	000	IN	CPTB,OTHCTB,THCTB
000003	000	DEF	CPTB,OTHCTB,THCTB
000004	000	END	

QHDG,P \*\*\*\*\* TBBR \*\*\*\*\*

\*\*\*\*\* TBBR \*\*\*\*\*

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QELT,L ME\*NASA5.TBBR

ELT 005-07/13-20:59

000001 000 CLASS

TBBR

000002 000 IN

CPTB,DTHCTB,THCTB

000003 000 DEF

CPTB,DTHCTB,THCTB

000004 000 END

QNDG,P \*\*\*\*\* TBCU \*\*\*\*\*

\*\*\*\*\* TBCU \*\*\*\*\*

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1

QELT,L ME\*NASA5.TBCU

ELT 005-07/13-20:59

000001 000 CLASS

TBCU

000002 000 IN

CPTB,DTHCTB,THCTB

000003 000 DEF

CPTB,DTHCTB,THCTB

000004 000 END

QHDG,P \*\*\*\*\* VER1 \*\*\*\*\*

\*\*\*\*\* VER1 \*\*\*\*\*

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DELT L ME NASAS VER1

ELT 005-07/13-20:59

000001 000 IN

000002 000 IN

000003 000 END

CFSIL FNAL MPBR TBAL SCZ93

MAIN

QHDG N

QHDG P \*\*\*\*\* CFFC43 \*\*\*\*\*

\*\*\*\*\* CFFC43 \*\*\*\*\*

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PAGE 1

QMAP:RS ME\*NASAS.CFFC43  
MAP 0023-07/13-20:59 -(0,)

1. NEW LIB ME\*NASAS.  
2. CLASS CFFC43  
3. IN BETA,CAPPA,CPF,HFL,RHOF,THCF,VISC  
4. DEF BETA,CAPPA,CPF,HFL,RHOF,THCF,VISC  
5. END

# R-OPTION COLLECTION

## LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1) COMBINED ODD COUNTERS	000164	OCTAL	116	DECIMAL
S(2) COMBINED EVEN COUNTERS	000105	OCTAL	69	DECIMAL
3 COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

## COMBINED ELEMENTS

BETA/CFFC43	1	000000	000016	0	000000	000007
				2	BLANK\$COMMON	
CAPPA/CFFC43	1	000017	000030	0	000010	000015
				2	BLANK\$COMMON	
CPF/CFFC43	1	000031	000046	0	000016	000026
				2	BLANK\$COMMON	
HFL/CFFC43	1	000047	000072	0	000027	000042
				2	BLANK\$COMMON	
RHOF/CFFC43	1	000073	000110	0	000043	000053
				2	BLANK\$COMMON	
THCF/CFFC43	1	000111	000125	0	000054	000063
				2	BLANK\$COMMON	
VISC/CFFC43	1	000126	000163	0	000064	000104
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.303 SECONDS

\*\*\*\*\* CFFC43 \*\*\*\*\*

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QHDG:P \*\*\*\*\* CFFC75 \*\*\*\*\*



\*\*\*\*\* CFFC75 \*\*\*\*\*

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PAGE 1

QMAP,RS ME\*NASA5.CFFC75  
MAP 0023-07/13-20:59 -(0.)

1. NEW LIB ME\*NASAS.  
2. CLASS CFFC75  
3. IN BETA,CAPPA,CPF,HFL,RHOF,THCF,VISC  
4. DEF BETA,CAPPA,CPF,HFL,RHOF,THCF,VISC  
5. END

# R-OPTION COLLECTION

## LENGTHS OF LOCATION COUNTERS ASSIGNED

\$(1)	COMBINED ODD COUNTERS	000164	OCTAL	116	DECIMAL
\$(2)	COMBINED EVEN COUNTERS	000105	OCTAL	69	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

## COMBINED ELEMENTS

BETA/CFFC75	1	000000 000016	0	000000 000007
			2	BLANK\$COMMON
CAPPA/CFFC75	1	000017 000030	0	000010 000015
			2	BLANK\$COMMON
CPF/CFFC75	1	000031 000046	0	000016 000026
			2	BLANK\$COMMON
HFL/CFFC75	1	000047 000072	0	000027 000042
			2	BLANK\$COMMON
RHOF/CFFC75	1	000073 000110	0	000043 000053
			2	BLANK\$COMMON
THCF/CFFC75	1	000111 000125	0	000054 000063
			2	BLANK\$COMMON
VISC/CFFC75	1	000126 000163	0	000064 000104
			2	BLANK\$COMMON

END OF COLLECTION - TIME 0.203 SECONDS

\*\*\*\*\* CFFC75 \*\*\*\*\*

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QHDG:P \*\*\*\*\* CFHE \*\*\*\*\*

\*\*\*\*\* CFHE \*\*\*\*\*

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QMAP/RS ME\*NASA5.CFHE  
MAP 0023-07/13-20:59 -(0,)

```

1.NEW LIB ME*NASA5.
2. CLASS CFHE
3. IN BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC
4. DEF BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC
5. END

```

#### R-OPTION COLLECTION

```

LENGTHS OF LOCATION COUNTERS ASSIGNED
$(1) COMBINED ODD COUNTERS 000635 OCTAL 413 DECIMAL
$(2) COMBINED EVEN COUNTERS 000267 OCTAL 183 DECIMAL
3 COMMONBLOCK BLANK$COMMON 000000 OCTAL 0 DECIMAL

```

#### COMBINED ELEMENTS

BETA/CFHE	1	000000 000111	0	000000 000036
			2	BLANK\$COMMON
CAPPA/CFHE	1	000112 000200	0	000037 000067
			2	BLANK\$COMMON
CPF/CFHE	1	000201 000271	0	000070 000120
			2	BLANK\$COMMON
HFL/CFHE	1	000272 000415	0	000121 000167
			2	BLANK\$COMMON
PF/CFHE	1	000416 000467	0	000170 000215
			2	BLANK\$COMMON
RHOF/CFHE	1	000470 000566	0	000216 000240
			2	BLANK\$COMMON
THCF/CFHE	1	000567 000612	0	000241 000255
			2	BLANK\$COMMON
VISC/CFHE	1	000613 000634	0	000256 000266
			2	BLANK\$COMMON

END OF COLLECTION - TIME 0.315 SECONDS

\*\*\*\*\* CFHE \*\*\*\*\*

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QHDG:P \*\*\*\*\* CFNAK \*\*\*\*\*

\*\*\*\*\* CFNAK \*\*\*\*\*

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PAGE 1

QMAP,RS ME\*NASAS.CFNAK  
MAP 0023-07/13-20:59 -(0.)

1.NEW LIB ME\*NASAS.  
2. CLASS CFNAK  
3. IN BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC  
4. DEF BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC  
5. END

#### R-OPTION COLLECTION

#### LENGTHS OF LOCATION COUNTERS ASSIGNED

\$(1)	COMBINED ODD COUNTERS	000467	OCTAL	311	DECIMAL
\$(2)	COMBINED EVEN COUNTERS	000224	OCTAL	148	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

#### COMBINED ELEMENTS

BETA/CFNAK	1	000000	000016	0	000000	000007
				2	BLANK\$COMMON	
CAPPA/CFNAK	1	000017	000073	0	000010	000041
				2	BLANK\$COMMON	
CPF/CFNAK	1	000074	000177	0	000042	000073
				2	BLANK\$COMMON	
HFL/CFNAK	1	000200	000311	0	000074	000132
				2	BLANK\$COMMON	
PF/CFNAK	1	000312	000351	0	000133	000147
				2	BLANK\$COMMON	
RHOF/CFNAK	1	000352	000411	0	000150	000166
				2	BLANK\$COMMON	
THCF/CFNAK	1	000412	000437	0	000167	000204
				2	BLANK\$COMMON	
VISC/CFNAK	1	000440	000466	0	000205	000223
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.273 SECONDS

\*\*\*\*\* CFNAK \*\*\*\*\*

DATE 071372

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DHDG/P \*\*\*\*\* CFSIL \*\*\*\*\*

\*\*\*\*\* CFSIL \*\*\*\*\*

DATE 071372

PAGE 1

QMAP,RS ME\*NASAS.CFSIL  
MAP 0023-07/13-20:59 -(0,)

1.NEW LIB ME\*NASAS.  
2. CLASS CFSIL  
3. IN BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC  
4. DEF BETA,CAPPA,CPF,HFL,PF,RHOF,THCF,VISC  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1)	COMBINED ODD COUNTERS	001734	OCTAL	988	DECIMAL
S(2)	COMBINED EVEN COUNTERS	000717	OCTAL	463	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

#### COMBINED ELEMENTS

BETA/CFSIL	1	000000	000176	0	000000	000075
				2	BLANK\$COMMON	
CAPPA/CFSIL	1	000177	000344	0	000076	000161
				2	BLANK\$COMMON	
CPF/CFSIL	1	000345	000736	0	000162	000314
				2	BLANK\$COMMON	
HFL/CFSIL	1	000737	001264	0	000315	000446
				2	BLANK\$COMMON	
PF/CFSIL	1	001265	001446	0	000447	000540
				2	BLANK\$COMMON	
RHOF/CFSIL	1	001447	001575	0	000541	000625
				2	BLANK\$COMMON	
THCF/CFSIL	1	001576	001644	0	000626	000656
				2	BLANK\$COMMON	
VISC/CFSIL	1	001645	001733	0	000657	000716
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.469 SECONDS

\*\*\*\*\* CFSIL \*\*\*\*\*

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QHDG:P \*\*\*\*\* FNAL \*\*\*\*\*



\*\*\*\*\* FNAL \*\*\*\*\*

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PAGE 1

QMAP,RS ME\*NASA5.FNAL  
MAP 0023-07/13-20:59 -(0.)

1.NEW LIB ME\*NASA5.  
2. CLASS FNAL  
3. IN CPFN,DTHCFN,THCFN  
4. DEF CPFN,DTHCFN,THCFN  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1)	COMBINED ODD COUNTERS	000202	OCTAL	130	DECIMAL
S(2)	COMBINED EVEN COUNTERS	000116	OCTAL	78	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

##### COMBINED ELEMENTS

CPFN/FNAL	1	000000	000061	0	000000	000037
				2	BLANK\$COMMON	
DTHCFN/FNAL	1	000062	000121	0	000040	000056
				2	BLANK\$COMMON	
THCFN/FNAL	1	000122	000201	0	000057	000115
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.171 SECONDS

QMDG,P \*\*\*\*\* FNBR \*\*\*\*\*

\*\*\*\*\* FNBR \*\*\*\*\*

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PAGE 1

QMAP,RS ME\*NASA5.FNBR  
MAP 0023-07/13-20:59 -(0,)

1. NEW LIB ME\*NASA5.  
2. CLASS FNBR  
3. IN CPFN,DTHCFN,THCFN  
4. DEF CPFN,DTHCFN,THCFN  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1)	COMBINED ODD COUNTERS	000200	OCTAL	128	DECIMAL
S(2)	COMBINED EVEN COUNTERS	000115	OCTAL	77	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

#### COMBINED ELEMENTS

CPFN/FNBR	1	000000	000057	0	000000	000036
				2	BLANK\$COMMON	
DTHCFN/FNBR	1	000060	000117	0	000037	000055
				2	BLANK\$COMMON	
THCFN/FNBR	1	000120	000177	0	000056	000114
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.169 SECONDS

QHDG,P \*\*\*\*\* FNCU \*\*\*\*\*

\*\*\*\*\* FNCU \*\*\*\*\*

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PAGE 1

QMAP\*RS ME\*NASA5.FNCU  
MAP 0023-07/13-20:59 -(0,)

1. NEW LIB ME\*NASA5.  
2. CLASS FNCU  
3. IN CPFN\*DTTCFN\*THCFN  
4. DEF CPFN\*DTTCFN\*THCFN  
5. END

# R-OPTION COLLECTION

## LENGTHS OF LOCATION COUNTERS ASSIGNED

\$(1)	COMBINED ODD COUNTERS	000163	OCTAL	115	DECIMAL
\$(2)	COMBINED EVEN COUNTERS	000102	OCTAL	66	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

## COMBINED ELEMENTS

CPFN/FNCU	1	000000 000050	0	000000 000030
			2	BLANK\$COMMON
DTTCFN/FNCU	1	000051 000105	0	000031 000045
			2	BLANK\$COMMON
THCFN/FNCU	1	000106 000162	0	000046 000101
			2	BLANK\$COMMON

END OF COLLECTION - TIME 0.165 SECONDS

QHDG\*P \*\*\*\*\* MPAL \*\*\*\*\*

\*\*\*\*\* MPAL \*\*\*\*\*

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PAGE 1

DMAP\*RS ME\*NASA5.MPAL  
MAP 0023-07/13-20:59 -(0.)

1.NEW LIB ME\*NASA5.  
2. CLASS MPAL  
3. IN CPMP.DTHCMP.ELAS.THCM  
4. DEF CPMP.DTHCMP.ELAS.THCM  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1)	COMBINED ODD COUNTERS	000263	OCTAL	179	DECIMAL
S(2)	COMBINED EVEN COUNTERS	000156	OCTAL	110	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

#### COMBINED ELEMENTS

CPMP/MPAL	1	000000	000061	0	000000	000037
				2	BLANK\$COMMON	
DTHCMP/MPAL	1	000062	000121	0	000040	000056
				2	BLANK\$COMMON	
ELAS/MPAL	1	000122	000202	0	000057	000116
				2	BLANK\$COMMON	
THCMP/MPAL	1	000203	000262	0	000117	000155
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.206 SECONDS

DHOG\*P \*\*\*\*\* MPBR \*\*\*\*\*

\*\*\*\*\* MPBR \*\*\*\*\*

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QMAP,RS ME\*NASA5.MPBR  
MAP 0023-07/13-20:59 -(0,)

1.NEW LIB ME\*NASA5.  
2. CLASS MPBR  
3. IN CPMP,OTHCMP,ELAS,THCMP  
4. DEF CPMP,DTHCMP,ELAS,THCMP  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1)	COMBINED ODD COUNTERS	000261	OCTAL	177	DECIMAL
S(2)	COMBINED EVEN COUNTERS	000155	OCTAL	109	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

##### COMBINED ELEMENTS

CPMP/MPBR	1	000000	000057	0	000000	000036
				2	BLANK\$COMMON	
DTHCMP/MPBR	1	000060	000117	0	000037	000055
				2	BLANK\$COMMON	
ELAS/MPBR	1	000120	000200	0	000056	000115
				2	BLANK\$COMMON	
THCMP/MPBR	1	000201	000260	0	000116	000154
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.192 SECONDS

QHDG,P \*\*\*\*\* MPCU \*\*\*\*\*

\*\*\*\*\* MPCU \*\*\*\*\*

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QMAP,RS ME\*NASA5,MPCU  
MAP 0023-07/13-20:59 -(0,)

1. NEW LIB ME\*NASA5.  
2. CLASS MPCU  
3. IN CPMP,DTHCMP,ELAS,THCMP  
4. DEF CPMP,DTHCMP,ELAS,THCMP  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1)	COMBINED ODD COUNTERS	000244	OCTAL	164	DECIMAL
S(2)	COMBINED EVEN COUNTERS	000142	OCTAL	98	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

#### COMBINED ELEMENTS

CPMP/MPCU	1	000000	000050	0	000000	000030
				2	BLANK\$COMMON	
DTHCMP/MPCU	1	000051	000105	0	000031	000045
				2	BLANK\$COMMON	
ELAS/MPCU	1	000106	000166	0	000046	000105
				2	BLANK\$COMMON	
THCMP/MPCU	1	000167	000243	0	000106	000141
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.187 SECONDS

QHDG,P \*\*\*\*\* SC293 \*\*\*\*\*

\*\*\*\*\* SCZ93 \*\*\*\*\*

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QMAP:RS ME\*NASAS.SCZ93  
MAP 0023-07/13-20:59 -(0,)

1. NEW LIB ME\*NASAS.  
2. CLASS SCZ93  
3. IN AVGEMT.EMIT  
4. DEF AVGEMT.EMIT  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1)	COMBINED ODD COUNTERS	000156	OCTAL	110	DECIMAL
S(2)	COMBINED EVEN COUNTERS	000056	OCTAL	46	DECIMAL
3	COMMONBLOCK AVGABS	000251	OCTAL	169	DECIMAL
4	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

#### COMBINED ELEMENTS

AVGEMT/SCZ93	1	000000	000120	0	000000	000037
	3	AVGABS		2	BLANK\$COMMON	
EMIT/SCZ93	1	000121	000155	0	000040	000055
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.148 SECONDS

RHDG:P \*\*\*\*\* TBAL \*\*\*\*\*

\*\*\*\*\* TBAL \*\*\*\*\*

DATE 071372

PAGE 1

QMAP:RS ME\*NASA5.TBAL  
MAP 0023-07/13-21:00 -(0,)

1. NEW LIB ME\*NASA5.  
2. CLASS TBAL  
3. IN CPTB.DTHCTB.THCTB  
4. DEF CPTB.DTHCTB.THCTB  
5. END

# R-OPTION COLLECTION

## LENGTHS OF LOCATION COUNTERS ASSIGNED

S(1)	COMBINED ODD COUNTERS	000202	OCTAL	130	DECIMAL
S(2)	COMBINED EVEN COUNTERS	000116	OCTAL	78	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

## COMBINED ELEMENTS

CPTB/TBAL	1	000000	000061	0	000000	000037
				2	BLANK\$COMMON	
DTHCTB/TBAL	1	000062	000121	0	000040	000056
				2	BLANK\$COMMON	
THCTB/TBAL	1	000122	000201	0	000057	000115
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.170 SECONDS

QHDG:P \*\*\*\*\* TBBR \*\*\*\*\*



\*\*\*\*\* TBBR \*\*\*\*\*

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QMAP:RS ME\*NASA5.TBBR  
MAP 0023-07/13-21:00 -(0.)

1.NEW LIB ME\*NASA5.  
2. CLASS TBBR  
3. IN CPTB.DTHCTB.THCTB  
4. DEF CPTB.DTHCTB.THCTB  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

\$(1)	COMBINED ODD COUNTERS	000200	OCTAL	128	DECIMAL
\$(2)	COMBINED EVEN COUNTERS	000115	OCTAL	77	DECIMAL
3	COMMONBLOCK BLANKSCOMMON	000000	OCTAL	0	DECIMAL

##### COMBINED ELEMENTS

CPTB/TBBR	1	000000	000057	0	000000	000036
				2	BLANKSCOMMON	
DTHCTB/TBBR	1	000060	000117	0	000037	000055
				2	BLANKSCOMMON	
THCTB/TBBR	1	000120	000177	0	000056	000114
				2	BLANKSCOMMON	

END OF COLLECTION - TIME 0.167 SECONDS

QMDG:P \*\*\*\*\* TBCU \*\*\*\*\*

\*\*\*\*\* TBCU \*\*\*\*\*

DATE 071372

PAGE 1

QMAP,RS ME\*NASAS.TBCU  
MAP 0023-07/13-21:00 -(0.)

1. NEW LIB ME\*NASAS.  
2. CLASS TBCU  
3. IN CPTB.DTHCTB,THCTB  
4. DEF CPTB.DTHCTB,THCTB  
5. END

#### R-OPTION COLLECTION

##### LENGTHS OF LOCATION COUNTERS ASSIGNED

\$(1)	COMBINED ODD COUNTERS	000163	OCTAL	115	DECIMAL
\$(2)	COMBINED EVEN COUNTERS	000102	OCTAL	66	DECIMAL
3	COMMONBLOCK BLANK\$COMMON	000000	OCTAL	0	DECIMAL

##### COMBINED ELEMENTS

CPTB/TBCU	1	000000	000050	0	000000	000030
				2	BLANK\$COMMON	
DTHCTB/TBCU	1	000051	000105	0	000031	000045
				2	BLANK\$COMMON	
THCTB/TBCU	1	000106	000162	0	000046	000101
				2	BLANK\$COMMON	

END OF COLLECTION - TIME 0.165 SECONDS

SPREP ME\*NASAS.  
FURPUR 023A-07/13-21:00

QHDG,P \*\*\*\*\* VER1 \*\*\*\*\*

\*\*\*\*\* VER1 \*\*\*\*\*

DATE 071372 PAGE 1

QMAP,S ME\*NASA5.VER1  
MAP 0023-07/13-21:00 -(0,)

1.NEW LIB ME\*NASA5.  
2. IN CFSIL,FNAL,MPBR,TBAL,SC293  
3. IN MAIN  
4. END

ADDRESS LIMITS 001000 052007 053000 116661  
STARTING ADDRESS 046014  
WORDS DECIMAL 21000 IBANK 18354 DBANK

SEGMENT	MAIN	001000	052007	053000	116661
NFTVS/FOR	1	001000	001022		
NBF00\$/FOR				2	053000 055201
NRBLK\$/FOR	1	001023	001045		
N30CV\$/FOR	1	001046	001200	2	055202 055237
NSWTC\$/FOR	1	001201	001222		
N3SBL\$/FOR	1	001223	001257		
NUPDAS\$/FOR	1	001260	001312		
NWBLK\$/FOR	1	001313	001434		
NFTCH\$/FOR	1	001435	001725	2	055240 055275
NINPT\$/FOR	1	001726	002570	2	055276 055315
NOTINS\$/FOR	1	002571	003104	2	055316 055321
NOUT\$/FOR	1	003105	004073	2	055322 055346
NFMT\$/FOR	1	004074	004752	2	055347 055423
NCNVT\$/FOR	1	004753	005177	2	055424 055513
NIOER\$/FOR	1	005200	005352	2	055514 055620
NININ\$/FOR	1	005353	005563	2	055621 055632
NFCHK\$/FOR	1	005564	006445	2	055633 055771
				4	055772 056043
NWEF\$/FOR	1	006446	006647	2	056044 056063
ERUS/MISC					
N1AB\$/FOR				2	056064 056122
NEXP5\$/FOR	1	006650	006735	2	056123 056132
SINCO5\$/FOR	1	006736	007070	2	056133 056154
ATANS\$/FOR	1	007071	007274	2	056155 056206
NOSYMS\$/FOR	1	007275	007534	2	056207 056213
NRWNO5\$/FOR	1	007535	007614	2	056214 056225
NEXP6\$/FOR	1	007615	010011	2	056226 056277
NBKS5\$/FOR	1	010012	010510	2	056300 056326

\*\*\*\*\* VER1 \*\*\*\*\*

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EXPS/FOR	1	010511 010600	2	056327 056347
ALOGS/FOR	1	010601 010720	2	056350 056410
NERRS/FOR	1	010721 011245	2	056411 056554
ASINCOSS/FOR	1	011246 011462	0	056555 056602
SGRTS/FOR	1	011463 011523	2	056603 056614
NLOUTS/FOR	1	011524 012606	2	056615 056652
NFOUTS/FOR	1	012607 013117	2	056653 056674
NFINPS/FOR	1	013120 013433	2	056675 056761
NIBUFS/FOR	1	013434 013476	2	056762 056762
NIERS/FOR	1	013477 013560	2	056763 057116
NOSUFS/FOR	1	013561 013625		
NLINPS/FOR	1	013626 015341	2	057117 057300
TIRS/TECH	1	015342 016026	0	057301 057331
			2	057332 057611
NCLOSS/FOR	1	016027 016175	2	057612 057643
TTIP	1	016176 016255	0	057644 057744
			2	BLANK\$COMMON
EFFICY	1	016256 016436	0	057745 060075
			2	BLANK\$COMMON
DEFNT	1	016437 016516	0	060076 060120
			2	BLANK\$COMMON
EXITAV	1	016517 016626	0	060121 060150
	3	GRD	2	BLANK\$COMMON
	5	ABSRST	4	QIN
MTXINV	1	016627 017034	0	060151 060207
	3	GRD	2	BLANK\$COMMON
TRMATX	1	017035 017331	0	060210 060260
	3	GRD	2	BLANK\$COMMON
			4	ABSRST
ABSORB	1	017332 020131	0	060261 060375
	3	GRD	2	BLANK\$COMMON
	5	AVGABS	4	ABSRST
INTERP	1	020132 020357	0	060376 060460
			2	BLANK\$COMMON
NUS	1	020360 021221	0	060461 060574
			2	BLANK\$COMMON
REFP	1	021222 021610	0	060575 060757
			2	BLANK\$COMMON
CPAIR	1	021611 021720	0	060760 061014
			2	BLANK\$COMMON
TNH	1	021721 022004	0	061015 061044
			2	BLANK\$COMMON
ENTAIR	1	022005 022231	0	061045 061135
			2	BLANK\$COMMON
ATMOS	1	022232 022512	0	061136 061346
			2	BLANK\$COMMON
ALTVEL	1	022513 022721	0	061347 061364
			2	BLANK\$COMMON
RKSF	1	022722 023761	0	061365 061450
			2	BLANK\$COMMON
TRNSPT	1	023762 024152	0	061451 061510
			2	BLANK\$COMMON
DRVLCN (COMMON BLOCK)				061511 061511
DRVLCM (COMMON BLOCK)				061512 061535
CNTLN	1	024153 024263	0	061536 061560
	3	DRVLCM	2	BLANK\$COMMON
			4	DRVLCN

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DERIVL	1	024264 024542	0	061561 061633
	3	FLCMP	2	BLANK\$COMMON
FMINV	1	024543 025050	0	061634 063113
			2	BLANK\$COMMON
PDERIV	1	025051 025254	0	063114 063133
	3	ADBH	2	BLANK\$COMMON
TTIPS	1	025255 025642	0	063134 063243
	3	ADBH	2	BLANK\$COMMON
			4	QIN
NUSA	1	025643 025745	0	063244 063267
			2	BLANK\$COMMON
SHADE	1	025746 026051	0	063270 063312
			2	BLANK\$COMMON
FINT	1	026052 026162	0	063313 063341
			2	BLANK\$COMMON
YINT	1	026163 026477	0	063342 063477
			2	BLANK\$COMMON
D2DX2	1	026500 026662	0	063500 063545
			2	BLANK\$COMMON
DDX	1	026663 027035	0	063546 063613
			2	BLANK\$COMMON
QRAD	1	027036 030050	0	063614 064712
	3	QRD	2	BLANK\$COMMON
	5	ABSRST	4	QIN
	7	AVGABS	6	SSF
CONVEC	1	030051 030631	0	064713 065056
	3	VELALT	2	BLANK\$COMMON
	5	SRTCNV	4	CNV
	7	SSF	6	QRD
POLY	1	030632 030675	0	065057 065073
			2	BLANK\$COMMON
RKS	1	030676 031735	0	065074 065157
			2	BLANK\$COMMON
SHAPEF	1	031736 032772	0	065160 065331
	3	QRD	2	BLANK\$COMMON
			4	SSF
DEFINT	1	032773 033217	0	065332 065377
			2	BLANK\$COMMON
FLCMP (COMMON BLOCK)				065400 065423
FLSTRT	1	033220 033464	0	065424 065656
	3	DVCMFL	2	BLANK\$COMMON
			4	FLCMP
TK	1	033465 033731	0	065657 065740
			2	BLANK\$COMMON
QINCID	1	033732 034760	0	065741 066625
	3	QIN	2	BLANK\$COMMON
			4	TC
ADIABH	1	034761 035502	0	066626 067276
	3	ADBH	2	BLANK\$COMMON
TCALC	1	035503 036354	0	067277 070166
	3	ADBH	2	BLANK\$COMMON
	5	QIN	4	TC
SRTCNV (COMMON BLOCK)				070167 070171
CNTLM	1	036355 040434	0	070172 073614
	3	SRTCNV	2	BLANK\$COMMON
	5	QIN	4	QRD
	7	FLDINL	6	GEOM

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DERIVM	1	040435 043014	8	DVCMFL
	3	DVM	0	073615 074675
	5	FLDINL	2	BLANK\$COMMON
			4	QIN
			6	DVCMFL
AVGABS (COMMON BLOCK)				074676 075146
DVCMFL (COMMON BLOCK)				075147 075150
FLDINL (COMMON BLOCK)				075151 075627
GEOM (COMMON BLOCK)				075630 075647
SSF (COMMON BLOCK)				075650 075657
ABSRST (COMMON BLOCK)				075660 076460
TC (COMMON BLOCK)				076461 076522
QIN (COMMON BLOCK)				076523 100332
QRD (COMMON BLOCK)				100333 104253
CNV (COMMON BLOCK)				104254 104263
VELALT (COMMON BLOCK)				104264 105415
DVM (COMMON BLOCK)				105416 105417
ADBH (COMMON BLOCK)				105420 105771
BLANK\$COMMON (COMMON BLOCK)				105772 107106
CFSIL	1	043015 044750	2	107107 110025
	3	BLANK\$COMMON		
FNAL	1	044751 045152	2	110026 110143
	3	BLANK\$COMMON		
MPBR	1	045153 045433	2	110144 110320
	3	BLANK\$COMMON		
TBAL	1	045434 045635	2	110321 110436
	3	BLANK\$COMMON		
SCZ93	1	045636 046013	2	110437 110514
	3	AVGABS	4	BLANK\$COMMON
MAIN	1	046014 052007	0	110515 116661
	3	ADBH	2	BLANK\$COMMON
	5	VELALT	4	DVM
	7	QRD	6	CNV
	9	TC	8	QIN
	11	SSF	10	ABSRST
	13	FLDINL	12	GEOM
			14	DVCMFL

SYSS\*RLIBS. LEVEL 63

END OF COLLECTION - TIME 4.525 SECONDS

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3PRT:T ME\*NASAS.  
FURPUR 023A-07/13-21:00

ME\*NASAS ELEMENT TABLE

D	NAME	VERSION	TYPE	DATE	TIME	SEQ #	SIZE-PRE,TEXT (CYCLE WORD)	PSRMODE	LOCATION
	ABSORB		FOR SYMB	09 MAY 72	16:50:13	1	21 5 0 1		1792
	ADIABH		FOR SYMB	02 MAY 72	15:31:15	2	24 5 0 1		1813
	ALTVEL		FOR SYMB	09 MAY 72	16:50:24	3	7 5 0 1		1837
	ATMOS		FOR SYMB	09 MAY 72	16:50:26	4	24 5 0 1		1844
	AVGENT	SCZ93	FOR SYMB	09 MAY 72	15:48:01	5	8 5 0 1		1868
	BETA	CFFC43	FOR SYMB	09 MAY 72	16:43:19	6	3 5 0 1		1876
	BETA	CFFC75	FOR SYMB	09 MAY 72	16:44:12	7	3 5 0 1		1879
	BETA	CFHE	FOR SYMB	09 MAY 72	16:40:41	8	6 5 0 1		1882
	BETA	CFNAK	FOR SYMB	09 MAY 72	16:42:30	9	3 5 0 1		1888
	BETA	CFSIL	FOR SYMB	30 JUN 72	15:44:49	10	8 5 0 1		1891
	CAPPA	CFFC43	FOR SYMB	09 MAY 72	16:43:25	11	3 5 0 1		1899
	CAPPA	CFFC75	FOR SYMB	09 MAY 72	16:44:34	12	3 5 0 1		1902
	CAPPA	CFHE	FOR SYMB	09 MAY 72	16:40:48	13	5 5 0 1		1905
	CAPPA	CFNAK	FOR SYMB	09 MAY 72	16:42:37	14	5 5 0 1		1910
	CAPPA	CFSIL	FOR SYMB	30 JUN 72	15:44:52	15	7 5 0 1		1915
	CFFC43		MAP SYMB	10 MAY 72	08:56:52	16	1 5 0 1		1922
	CFFC75		MAP SYMB	10 MAY 72	08:57:18	17	1 5 0 1		1923
	CFHE		MAP SYMB	10 MAY 72	08:58:15	18	1 5 0 1		1924
	CFNAK		MAP SYMB	10 MAY 72	08:58:40	19	1 5 0 1		1925
	CFSIL		MAP SYMB	10 MAY 72	08:59:02	20	1 5 0 1		1926
	CNTLM		FOR SYMB	12 MAY 72	09:06:54	21	77 5 0 1		1927
	CNTLN		FOR SYMB	09 MAY 72	16:49:43	22	12 5 0 1		2004
	CONVEC		FOR SYMB	09 MAY 72	16:51:04	23	28 5 0 1		2016
	CPAIR		FOR SYMB	09 MAY 72	16:50:38	24	6 5 0 1		2044
	CPF	CFFC43	FOR SYMB	09 MAY 72	16:43:31	25	3 5 0 1		2050
	CPF	CFFC75	FOR SYMB	09 MAY 72	16:44:44	26	3 5 0 1		2053
	CPF	CFHE	FOR SYMB	09 MAY 72	16:40:55	27	5 5 0 1		2056
	CPF	CFNAK	FOR SYMB	09 MAY 72	16:42:43	28	5 5 0 1		2061
	CPF	CFSIL	FOR SYMB	30 JUN 72	15:44:54	29	13 5 0 1		2066
	CPFN	FNAL	FOR SYMB	09 MAY 72	16:47:50	30	5 5 0 1		2079
	CPFN	FNBR	FOR SYMB	09 MAY 72	16:48:28	31	5 5 0 1		2084
	CPFN	FNCU	FOR SYMB	09 MAY 72	16:48:13	32	4 5 0 1		2089
	CPMP	MPAL	FOR SYMB	09 MAY 72	16:48:38	33	5 5 0 1		2093
	CPMP	MPBR	FOR SYMB	09 MAY 72	16:49:21	34	5 5 0 1		2098
	CPMP	MPCU	FOR SYMB	09 MAY 72	16:48:52	35	4 5 0 1		2103
	CPTB	TBAL	FOR SYMB	09 MAY 72	16:46:59	36	5 5 0 1		2107
	CPTB	TBBR	FOR SYMB	09 MAY 72	16:47:32	37	5 5 0 1		2112
	CPTB	TBCU	FOR SYMB	09 MAY 72	16:47:18	38	4 5 0 1		2117
	DATA		ELT SYMB	09 MAY 72	14:02:46	39	24 5 0 1		2121
	DDX		FOR SYMB	09 MAY 72	16:51:51	40	8 5 0 1		2145
	DEFINT		FOR SYMB	09 MAY 72	16:52:40	41	6 5 0 1		2153
	DEFNT		FOR SYMB	09 MAY 72	16:52:04	42	3 5 0 1		2159
	DERIVL		FOR SYMB	11 MAY 72	15:27:24	43	18 5 0 1		2162
	DERIVM		FOR SYMB	09 MAY 72	16:52:10	44	61 5 0 1		2180
	DTHCFN	FNAL	FOR SYMB	09 MAY 72	16:47:46	45	4 5 0 1		2241
	DTHCFN	FNBR	FOR SYMB	09 MAY 72	16:48:23	46	4 5 0 1		2245
	DTHCFN	FNCU	FOR SYMB	09 MAY 72	16:48:10	47	4 5 0 1		2249
	DTHCMP	MPAL	FOR SYMB	09 MAY 72	16:48:35	48	4 5 0 1		2253
	DTHCMP	MPBR	FOR SYMB	09 MAY 72	16:49:13	49	4 5 0 1		2257
	DTHCMP	MPCU	FOR SYMB	09 MAY 72	16:48:49	50	4 5 0 1		2261
	DTHCTB	TBAL	FOR SYMB	09 MAY 72	16:46:53	51	4 5 0 1		2265

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OTHCTB	TBBR	FOR SYMB	09 MAY 72	16:47:28	52	4	5	0	1	2269
OTHCTB	TBCU	FOR SYMB	09 MAY 72	16:47:15	53	4	5	0	1	2273
D2OX2		FOR SYMB	09 MAY 72	16:51:54	54	9	5	0	1	2277
EFFICY		FOR SYMB	23 MAR 72	16:32:42	55	8	5	0	1	2286
ELAS	MPAL	FOR SYMB	09 MAY 72	16:48:42	56	5	5	0	1	2294
ELAS	MPBR	FOR SYMB	09 MAY 72	16:49:27	57	5	5	0	1	2299
ELAS	MPCU	FOR SYMB	09 MAY 72	16:48:54	58	5	5	0	1	2304
EMIT	SCZ93	FOR SYMB	09 MAY 72	15:47:54	59	4	5	0	1	2309
ENTAIR		FOR SYMB	09 MAY 72	16:50:46	60	9	5	0	1	2313
EXITAV		FOR SYMB	09 MAY 72	16:50:10	61	7	5	0	1	2322
FINT		FOR SYMB	09 MAY 72	16:52:43	62	4	5	0	1	2329
FLSTRT		FOR SYMB	11 MAY 72	15:48:02	63	27	5	0	1	2333
FMINV		FOR SYMB	23 MAR 72	16:34:18	64	15	5	0	1	2360
FNAL		MAP SYMB	10 MAY 72	08:59:59	65	1	5	0	1	2375
FNBR		MAP SYMB	10 MAY 72	09:00:26	66	1	5	0	1	2376
FNCU		MAP SYMB	10 MAY 72	09:00:51	67	1	5	0	1	2377
HFL	CFFC43	FOR SYMB	09 MAY 72	16:40:25	68	3	5	0	1	2378
HFL	CFFC75	FOR SYMB	09 MAY 72	16:45:21	69	3	5	0	1	2381
HFL	CFHE	FOR SYMB	09 MAY 72	16:40:58	70	6	5	0	1	2384
HFL	CFNAK	FOR SYMB	09 MAY 72	16:42:49	71	5	5	0	1	2390
HFL	CFSIL	FOR SYMB	30 JUN 72	15:44:56	72	12	5	0	1	2395
INTERP		FOR SYMB	09 MAY 72	16:52:45	73	7	5	0	1	2407
MAIN		FOR SYMB	09 MAY 72	15:58:27	74	197	5	0	1	2414
MPAL		MAP SYMB	10 MAY 72	09:02:03	75	1	5	0	1	2611
MPBR		MAP SYMB	10 MAY 72	09:02:27	76	1	5	0	1	2612
MPCU		MAP SYMB	10 MAY 72	09:03:11	77	1	5	0	1	2613
MTXINV		FOR SYMB	09 MAY 72	16:51:59	78	7	5	0	1	2614
NUS		FOR SYMB	09 MAY 72	16:52:39	79	20	5	0	1	2621
NUSA		FOR SYMB	20 APR 72	13:23:20	80	3	5	0	1	2641
PDERIV		FOR SYMB	02 MAY 72	15:35:16	81	7	5	0	1	2644
PI	CFHE	FOR SYMB	09 MAY 72	16:41:10	82	4	5	0	1	2651
PF	CFNAK	FOR SYMB	09 MAY 72	16:42:58	83	3	5	0	1	2655
PF	CFSIL	FOR SYMB	30 JUN 72	15:45:03	84	8	5	0	1	2658
POLY		FOR SYMB	09 MAY 72	16:51:39	85	3	5	0	1	2666
QINCID		FOR SYMB	09 MAY 72	16:26:12	86	25	5	0	1	2669
GRAD		FOR SYMB	09 MAY 72	16:50:18	87	33	5	0	1	2694
REFP		FOR SYMB	09 MAY 72	16:50:58	88	21	5	0	1	2727
RHOF	CFFC43	FOR SYMB	09 MAY 72	16:43:09	89	3	5	0	1	2748
RHOF	CFFC75	FOR SYMB	09 MAY 72	16:44:04	90	3	5	0	1	2751
RHOF	CFHE	FOR SYMB	09 MAY 72	16:40:33	91	5	5	0	1	2754
RHOF	CFNAK	FOR SYMB	09 MAY 72	16:42:23	92	4	5	0	1	2759
RHOF	CFSIL	FOR SYMB	30 JUN 72	15:44:47	93	7	5	0	1	2763
RKS		FOR SYMB	09 MAY 72	16:51:28	94	73	5	0	1	2770
RKSF		FOR SYMB	09 MAY 72	16:51:18	95	73	5	0	1	2843
SCZ93		MAP SYMB	10 MAY 72	09:06:45	96	1	5	0	1	2916
SHADE		FOR SYMB	20 APR 72	14:28:06	97	3	5	0	1	2917
SHAPEF		FOR SYMB	09 MAY 72	16:49:51	98	24	5	0	1	2920
TBAL		MAP SYMB	10 MAY 72	09:41:22	99	1	5	0	1	2944
TBBR		MAP SYMB	10 MAY 72	09:41:33	100	1	5	0	1	2945
TBCU		MAP SYMB	10 MAY 72	09:05:22	101	1	5	0	1	2946
TCALC		FOR SYMB	02 MAY 72	15:36:13	102	20	5	0	1	2947
THCF	CFFC43	FOR SYMB	09 MAY 72	16:43:49	103	3	5	0	1	2967
THCF	CFFC75	FOR SYMB	09 MAY 72	16:46:38	104	3	5	0	1	2970
THCF	CFHE	FOR SYMB	09 MAY 72	16:41:06	105	4	5	0	1	2973
THCF	CFNAK	FOR SYMB	09 MAY 72	16:42:55	106	4	5	0	1	2977
THCF	CFSIL	FOR SYMB	30 JUN 72	15:45:00	107	5	5	0	1	2981
THCFN	FNAL	FOR SYMB	09 MAY 72	16:47:40	108	5	5	0	1	2986



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THCFN	FNBR	FOR SYMB	09 MAY 72	16:48:17	109	5	5	0	1	2991
THCFN	FNCU	FOR SYMB	09 MAY 72	16:47:57	110	5	5	0	1	2996
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THCTB	TBBR	FOR SYMB	09 MAY 72	16:47:24	115	5	5	0	1	3021
THCTB	TBCU	FOR SYMB	09 MAY 72	16:47:04	116	5	5	0	1	3026
TK		FOR SYMB	09 MAY 72	16:51:09	117	7	5	0	1	3031
TNH		FOR SYMB	09 MAY 72	16:50:50	118	4	5	0	1	3038
TRMATX		FOR SYMB	09 MAY 72	16:50:04	119	11	5	0	1	3042
TRNSPT		FOR SYMB	09 MAY 72	16:52:30	120	11	5	0	1	3053
TTIP		FOR SYMB	23 MAR 72	16:33:46	121	5	5	0	1	3064
TTIPS		FOR SYMB	02 MAY 72	15:37:06	122	15	5	0	1	3069
VER1		MAP SYMB	10 MAY 72	16:52:30	123	1	5	0	1	3084
VISC	CFFC43	FOR SYMB	09 MAY 72	16:43:41	124	4	5	0	1	3085
VISC	CFFC75	FOR SYMB	09 MAY 72	16:45:47	125	4	5	0	1	3089
VISC	CFHE	FOR SYMB	09 MAY 72	16:41:03	126	3	5	0	1	3093
VISC	CFNAK	FOR SYMB	09 MAY 72	16:42:52	127	4	5	0	1	3096
VISC	CFSIL	FOR SYMB	30 JUN 72	15:44:58	128	5	5	0	1	3100
YINT		FOR SYMB	09 MAY 72	16:28:50	129	10	5	0	1	3105
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ALTVEL		RELOCATABLE	13 JUL 72	20:53:02	132	1	7			3168
ATMOS		RELOCATABLE	13 JUL 72	20:53:04	133	1	16			3176
AVGEMT	SCZ9?	RELOCATABLE	13 JUL 72	20:53:06	134	1	6			3193
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CNTLN		RELOCATABLE	13 JUL 72	20:54:31	146	2	5			3330
CONVEC		RELOCATABLE	13 JUL 72	20:54:37	147	3	22			3337
CPAIR		RELOCATABLE	13 JUL 72	20:54:40	148	1	6			3362
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DOX		RELOCATABLE	13 JUL 72	20:55:16	163	1	7			3456
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DEFNT		RELOCATABLE	13 JUL 72	20:55:24	165	1	3			3474

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DERIVL		RELOCATABLE	13 JUL 72	20:55:27	166	2	11	3478
DERIVM		RELOCATABLE	13 JUL 72	20:55:36	167	3	71	3491
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FINT		RELOCATABLE	13 JUL 72	20:56:27	185	1	5	3658
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VISC	CFNAK	RELOCATABLE	13 JUL 72	20:59:09	239	1	2	4354
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TBBR		RELOCATABLE	13 JUL 72	21:00:03	255	2	11	4639
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VER1		ABSOLUTE	13 JUL 72	21:00:45	257		948	4664
								5612

NEXT AVAILABLE LOCATION-

ASSEMBLER PROCEDURE TABLE EMPTY

COBOL PROCEDURE TABLE EMPTY

FORTRAN PROCEDURE TABLE EMPTY

ENTRY POINT TABLE

D NAME	LINK	D NAME	LINK	D NAME	LINK	D NAME	LINK	D NAME	LINK
ABSORB	130	ADIABH	131	ALTVEL	132	ATMOS	133	AVGEMT	253
AVGEMT	134	BETA	135	BETA	136	BETA	137	BETA	138
BETA	139	BETA	246	BETA	245	BETA	244	BETA	243
BETA	242	CAPPA	140	CAPPA	246	CAPPA	141	CAPPA	245
CAPPA	142	CAPPA	244	CAPPA	143	CAPPA	243	CAPPA	144
CAPPA	242	CNTLM	145	CNTLN	146	CONVEC	147	CPAIR	148
CPF	246	CPF	245	CPF	149	CPF	150	CPF	244
CPF	151	CPF	152	CPF	243	CPF	153	CPF	242
CPFN	249	CPFN	248	CPFN	247	CPFN	154	CPFN	155
CPFN	156	CPMP	157	CPMP	252	CPMP	251	CPMP	250
CPMP	158	CPMP	159	CPTB	256	CPTB	255	CPTB	254

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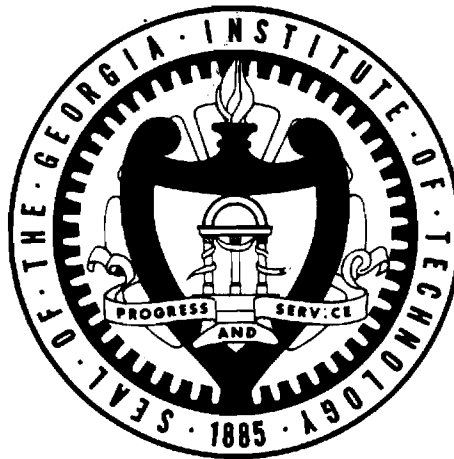
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DEFNT	165	DERIVL	166	DERIVM	167	DTHCFN	249	DTHCFN	170
DTHCFN	248	DTHCFN	247	DTHCFN	168	DTHCFN	169	DTHCMP	252
DTHCMP	251	DTHCMP	250	DTHCMP	171	DTHCMP	172	DTHCMP	173
DTHCTB	256	DTHCTB	255	DTHCTB	254	DTHCTB	176	DTHCTB	174
DTHCTB	175	D2DX2	177	EFFICY	178	ELAS	252	ELAS	251
ELAS	250	ELAS	179	ELAS	180	ELAS	181	EMIT	182
EMIT	253	ENTAIR	183	EXITAV	184	FINT	185	FLSTRT	186
FMINV	187	FORMAINS	194	HFL	246	HFL	245	HFL	188
HFL	244	HFL	189	HFL	190	HFL	191	HFL	243
HFL	192	HFL	242	INTERP	193	MTXINV	195	NUS	196
NUSA	197	PDERIV	198	PF	199	PF	200	PF	201
PF	246	PF	245	PF	244	POLY	202	QINCIO	203
GRAD	204	REFP	205	RHOF	206	RHOF	207	RHOF	243
RHOF	208	RHOF	210	RHOF	209	RHOF	246	RHOF	245
RHOF	242	RHOF	244	RKS	211	RKSF	212	SHADE	213
SHAPEF	214	TCALC	215	THCF	219	THCF	243	THCF	242
THCF	220	THCF	245	THCF	244	THCF	218	THCF	217
THCF	216	THCF	246	THCFN	221	THCFN	222	THCFN	223
THCFN	249	THCFN	248	THCFN	247	THCMP	252	THCMP	251
THCMP	225	THCMP	250	THCMP	226	THCMP	224	THCTB	256
THCTB	227	THCTB	228	THCTB	255	THCTB	254	THCTB	229
TK	230	TNH	231	TRMATX	232	TRNSPT	233	TTIP	234
TTIPS	235	VISC	242	VISC	243	VISC	245	VISC	236
VISC	239	VISC	244	VISC	238	VISC	246	VISC	237
VISC	240	YINT	241						

DBRKPT PRINTS

GEORGIA INSTITUTE OF TECHNOLOGY  
School of Mechanical Engineering  
Atlanta, Georgia

Final Report  
Part 3

SPACE RADIATOR SIMULATION  
SYSTEM ANALYSIS



Contract No. NAS 9-10415  
by  
William Z. Black and Wolfgang Wulff

Sponsored by the  
Power Generation Branch  
Manned Spacecraft Center  
NATIONAL AERONAUTICS AND SPACE ADMINISTRATION  
Houston, Texas

April 1972

GEORGIA INSTITUTE OF TECHNOLOGY  
SCHOOL OF MECHANICAL ENGINEERING  
ATLANTA, GEORGIA

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SCHOOL OF MECHANICAL ENGINEERING  
GEORGIA INSTITUTE OF TECHNOLOGY  
ATLANTA, GEORGIA 30332

April 1972

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NATIONAL AERONAUTICS AND SPACE ADMINISTRATION  
Houston, Texas

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William Z. Black, Ph.D.  
Associate Professor

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Wolfgang Wulff, Ph.D.  
Associate Professor

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Stothe P. Kezios, Ph.D.  
Director, School of Mechanical Engineering

## FOREWORD

This report covers the results of one portion of a two year research project carried out by the School of Mechanical Engineering at the Georgia Institute of Technology, Atlanta, Georgia for the NASA Manned Spacecraft Center, Houston, Texas, under Contract No. NAS 9-10415. This report summarizes the results of the radiator simulation analysis. The users manual for the execution of the computer program resulting from the analysis in this report is contained in a separate report. A third report covers the development of a simplified system simulation and the initial phases of a system optimization procedure. The project title is "Study of Design Parameters of Space Base and Space Shuttle Heat Rejection Systems." The work reported here was monitored by Dr. W. E. Simon of the Power Generation Branch of NASA MSC, Houston, Texas, and was carried out by Dr. W. Z. Black and Dr. W. Wulff as Co-Investigators and Mr. S. M. Morcos, Mr. S. L. Yao, and Mr. R. M. Hinson graduate students in the School of Mechanical Engineering under the direction of Dr. S. P. Kezios.

The work carried out by Dr. W. Z. Black is reflected in the analysis of Part II, Chapters 2, 7, 8, 11 and 14. Dr. Wulff is responsible for the analysis of Part II, Chapters 1 and 3 through 6, Chapters 9, 10, 12 and 13 as well as the numerical analysis in Part III. Mr. Morcos completed the property analysis reported in the Appendices A and B. Mr. Yao completed the surface coating property and the shape factor analysis reported in Appendices C and D, respectively.



## SUMMARY

A transient heat transfer analysis was carried out on a space radiator heat rejection system exposed to an arbitrarily prescribed combination of aerodynamic heating, solar, albedo and planetary irradiation. A rigorous analysis was carried out for the radiation panel and tubes lying in one plane and an approximate analysis was used to extend the rigorous analysis to the case of a curved panel. For the rigorous analysis the radiator system consists of equally spaced parallel coolant flow channels, all in one plane and connected by plane fin panels of trapezoidal cross-section, symmetric with respect to two normal planes, one passing through the tube axis, the other through the center between adjacent tubes. Investigated was one typical tube-fin element and the result extended over the entire system, on the basis of the above symmetry.

The rigorous analysis was extended, by approximate methods, to include radiator system which do not conform to the above symmetry restrictions. As a result, radiator systems can be treated whose coolant flow channels do not lie in one plane, provided that the radiative interaction between neighboring tube-fin elements is small when compared with the radiant flux densities at the panel. Moreover, radiator systems with non-uniform irradiation, non-uniform coolant inlet conditions and U-shaped coolant channels can be treated, provided that the spacing between channels is small when compared with the channel length.

The analysis permits the consideration of both gaseous and liquid coolant fluids, including liquid metals, under prescribed, time-dependent inlet conditions. The flow channels are covered with the same passive thermal control coating with optically diffuse but wavelength and temperature dependent optical properties.

The major results of the analysis are the prediction of both transient and steady-state, two-dimensional temperature profiles, the local and total heat rejection rates, the coolant flow pressure drop in the flow channel, and the total system weight and the protection layer thickness.

A computer program consisting of 62 program units was coded to execute the numerical solution of the system of differential equations

occurring in the analysis and to predict principal design parameters. The modular program structure readily permits later modifications. A separate final report entitled "Space Radiator Simulation, Manual for Computer Program" has been prepared which describes the computer programs [29].

A simplified analysis was carried out to aid the detailed analysis and to serve as the basis of systematic optimization. This analysis is covered in a companion report, entitled "Simplified Analysis and Optimization of Space Base and Space Shuttle Heat Rejection Systems" [30]. Regarding the heat rejection rates, its results are, for the test cases carried out so far, within 4% in agreement with the results of the detailed analysis. However, this rigorous analysis has greater applicability and detail.

This engineering analysis report is one of three final reports. The other two reports are a user's manual describing the computer code of this extensive, rigorous analysis [29] and the final report covering the simplified radiator system analyses and system optimization which describe both analysis and computer codes [30].

This report is written in two principal parts. The analysis and the governing equations are contained in the first part, Chapter II, titled Analysis. The numerical techniques are covered in Chapter III. Details which the reader may need to expand the program are placed in the appendices.

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## NOMENCLATURE

a	empirical constant used to account for spalling of protection layer (Eq. 8.8)	
a	function of temperature, used to express the isothermal compressibility and the zero-pressure isobaric thermal expansion coefficient, see also b and c (Eq. 12.9)	
a <sub>z</sub>	axial fluid flow acceleration	ft/sec <sup>2</sup>
A	area	ft <sup>2</sup>
A <sub>n</sub>	coefficients in polynomial for atmospheric pressure at high altitudes (Eq. 14.8)	-
A <sub>s</sub>	surface area	ft <sup>2</sup>
A <sub>x</sub>	cross-sectional area perpendicular to x coordinate	ft <sup>2</sup>
A <sub>z</sub>	cross-sectional area perpendicular to z coordinate	ft <sup>2</sup>
b	defined in Eq. 12.9 , see also a	
B	fin geometrical parameter (Eq. 2.14)	-
B <sub>N</sub>	coefficients in polynomial for atmospheric density at high altitudes (Eq. 14.8 )	-
c	atmospheric speed of sound	ft/sec
c	negative slope of fin sides	-
c	defined in Eq. 12.9 , see also a	
c	speed of light	ft/sec
$\bar{c}$	speed of sound in meteoroid protection material	ft/sec
c <sub>p</sub>	specific heat at constant pressure	Btu/(slug R)
c <sub>v</sub>	specific heat at constant volume	Btu/(slug R)

$c_v$	zero-pressure specific heat at constant volume	Btu/(slug R)
$d$	diameter of meteoroid particle	in
$d$	tube diameter	ft
$D$	fin geometrical parameter (Eq. 2.15)	-
$E$	emissive power	Btu/(hr ft <sup>2</sup> )
$E$	modulus of elasticity	lbf/in <sup>2</sup>
$E$	relative error	-
$f$	Fanning friction factor	-
$F$	cumulative meteoroid flux (eq. 8.4)	1/(ft <sup>2</sup> day)
$F$	dimensionless parameter, Eqs. 3.31-34	
$g_c$	32.174 ft lbf/(lbf sec <sup>2</sup> )	
$h$	Plank's constant, $h = 6.625 \times 10^{-34}$ W s <sup>2</sup>	
$h$	specific enthalpy	Btu/slug
$H$	geopotential altitude	ft
$H$	fin height, from root to tip	ft
$h_c$	convective film coefficient	Btu/(hr ft <sup>2</sup> R)
$h_i$	convective heat transfer coefficient used in reference enthalpy method	lbm/(hr ft <sup>2</sup> )
$k$	Boltzmann constant, $k = 1.380 \times 10^{-23}$ W s/K	
$k$	thermal conductivity	Btu/(hr ft R)
$L$	temperature gradient in atmosphere (Table 4 )	(K/km)
$L$	tube (and fin) length	ft
$M$	Mach number of orbiter	-
$M$	mass	slug
$M$	molecular weight	-
$M_{ij}$	transfer matrix Eq. 6.19	



$M_m$	inverted M	-
M	defined by Eq. 10.22	
n	constant that describes depth of penetration as a function of angle of incidence (Eq. 8.6)	-
n	number of tubes	
n	refractive index	-
N	cumulative number of meteoroid impacts	-
$N_{Bi}$	Biot number	-
$N_{Fo}$	Fourier number	-
$N_{Gr}$	Grashof number	-
$N_{Nc}$	Conductance parameter	-
$N_{Nu}$	Nusselt number	-
$N_{Re}$	Reynolds number	-
$N_{Pr}$	Prandtl number	-
p	absolute pressure	lbf/ft <sup>2</sup>
$P_j$	nondimensional excitation vector, Eq. 6.21	-
$P_j$	excitation vector, Eq. 6.18	
$P_o$	probability of no damage due to meteoroid impact	-
$P_\infty$	depth of penetration of a meteoroid particle into an infinite target.	in
$q''$	heat flux	Btu/(hr ft <sup>2</sup> )
$\tilde{q}$	nondimensional heat flux	-
$\bar{Q}_o$	nondimensional inlet power flux	-
$\bar{Q}_{rad}$	defined by Eq. 10.21	-
r, z	polar coordinates, see Fig. 1	ft
r	recovery factor	-

$R^*$	universal gas constant	ft lbf/(lb mole R)
$R$	outer meteoroid protection layer radius	ft
$s$	thickness	ft
$s_f = \frac{s_r}{2}$	fin half thickness at root	ft
$\bar{s}_f$	normalized fin thickness (Eq. 2.13)	-
$\overline{s_i s_j}$	direct exchange area (Eq. 6.1)	ft <sup>2</sup>
$S$	constant in atmospheric viscosity equation (Eq. 3.7)	K
$SS$	direct exchange "area", partially evaluated	ft
$t$	time	hr
$T$	absolute temperature	R
$u$	specific interval energy	Btu/slug
$V$	velocity of orbiter	ft/sec
$\bar{V}$	velocity of meteoroid relative to radiator	ft/sec
$w$	coolant flow velocity	ft/sec
$w_j$	nondimensional radiosity, Eq. 6.21	-
$W_j$	radiosity	Btu/(hr ft <sup>2</sup> )
$x$	distance from orbiter stagnation point to radiator panel	ft
$x, z$	rect. Cart. coordinates, see Fig. 1	ft
$x_{ij}, x_{ijk}$	auxiliary parameters, defined by Eqs. 6.10 and 6.11	-
$y$	overall length of radiator in direction parallel to gravity	ft
$y$	transverse fin coordinate	ft
$z$	axial distance	ft
$Z$	geometric altitude	ft

## Greek Symbols

$\alpha$	experimental meteoroid flux parameter, (Eq. 8.4)	$1/(\text{ft}^2 \text{day gm}^8)$
$\alpha$	thermal diffusivity	$\text{ft}^2/\text{hr}$
$\alpha_{ij}$	total hemispherical absorptance	-
$\beta$	constant in atmospheric viscosity equation, (Eq. 3.7)	$\text{kg}/(\text{sec mK}^{1/2})$
$\beta$	experimental meteoroid flux parameter, (Eq. 8.4)	-
$\beta$	isobaric thermal expansion coefficient, see Eq. 3.5	$1/R$
$\gamma$	ratio of specific heats	-
$\delta$	diameter-over-length ratio	-
$\delta_{ij}$	Kronecker delta	-
$\epsilon$	total hemisphere emittance	-
$\zeta$	dimensionless axial distance, Eq. 3.19	-
$\eta$	dimensionless radius, Eq. 3.25	-
$\theta$	dimensionless temperature, Eq. 3.21	-
$\theta$	empirical constant (Eq. 8.1)	-
$\kappa$	isothermal compressibility, Eq. 3.4	$\frac{2}{\text{ft}^2/\text{lbf}}$
$\lambda$	angle between path of meteoroid and normal to protected surface (Eq. 8.6)	deg.
$\lambda$	wave length	ft
$\lambda_1$	parameter, defined by Eq. 4.11	-
$\lambda_2$	parameter, defined by Eq. 4.12	-
$\mu$	dynamic viscosity	$\text{slug}/(\text{ft sec})$
$\nu$	dimensionless density	-
$\nu$	kinematic viscosity	$\text{ft}^2/\text{sec}$
$\xi$	dimensionless x coordinate	-

$\pi$	dimensionless pressure, Eq. 3.22	
$\rho$	density	slug/ft <sup>3</sup>
$\sigma$	Stefan-Boltzmann constant	Btu/(hr ft <sup>2</sup> R <sup>4</sup> )
$\tau$	dimensionless time, Eq. 3.20	-
$\tau$	time radiator is exposed to meteoroid environment	day
$\phi$	angle between surface normal and radiation beam	-
$\phi$	dimensionless property Eqs. 3.27-3.30, Eq. 4.4	-
$\phi, \phi^*$	polar angle in Eq. 6.15	-
$\phi_{Nc}$	modified conduction parameter, Eq. 5.6	
$\phi$	empirical constant (Eq. 8.1)	-
$\chi$	quadrature coefficient	-
$\psi$	circumferential fraction, defined by Eq. 4.3	-
$\psi$	residual, see Eqs. 12.1 and 12.2	-
$\omega$	dimensionless velocity Eq. 3.23	-

#### Subscripts

$aw$	adiabatic wall
$aero$	aerodynamic
$b$	values at the endpoints of straight line segments of atmospheric temperature profile, (Table 4)
$c$	coolant fluid
$c$	critical value
$c$	enclosure
$c_p$	referring to specific heat at constant pressure

e	outer surface of meteoroid protection layer, (environment)
f	referring to friction factor
f	fin
F	Fourier number
i, j	position index
m	meteoroid protection layer
M	Mach number
net,rad	net radiant
p	meteoroid particle
p	referring to pressure
r	fin root
t	fin tip
t	target material
t	tubes
w	channel wall
w	surface or wall condition
z	axial distance
z	thermal equation of state
o	entrance (reference) conditions
o	sea level values
1	upper fin side
2	lower fin side
$\infty$	free stream condition
$\zeta$	partial differentiation with resp. to $\zeta$
$\eta$	partial differentiation with resp. to $\eta$
$\kappa$	referring to compressibility

$\lambda$             monochromatic

$\tau$             partial differentiation with respect to  $\tau$

#### Superscripts

$*$             evaluated at reference temperature  $T^*$

## I. OBJECTIVE

The purpose of the analysis presented here is to develop a radiator system simulation which serves (i) to provide the design parameters necessary for the development of the radiator system, (ii) to predict the transient radiator performance under prescribed environmental and operation conditions, (iii) to predict the system response to conditions which lead to coolant fluid temperatures outside their operational temperature range, and (iv) to use the system performance data to suggest design options for shuttle radiator panel.

The class of system analyzed here is described in the following section. The system parameters produced for design purposes are those which describe coolant fluid flow field and the thermal state of the radiator structures as well as the geometry and the weight of the system components.

The arbitrarily prescribed environmental conditions consist of the specification of ascent and reentry profiles and of solar, planetary and albedo irradiation as a function of time.

The prescribed operating conditions are the coolant fluid inlet properties.

The analysis is designed to accommodate spectral characteristics of surface coatings, specified as functions of temperature. The analysis is, however, restricted to optically diffuse coatings.

Both coolant and structural material properties are accepted as prescribed functions of temperature. Coolant properties are specified as functions of pressure or density as well as temperature.

The analysis serves as the basis for a large-scale computer code in modular form. Material properties, complex mathematical operations and readily identifiable tasks in the computer code are written as subprograms.

The computer code is designed to simulate space radiator heat rejection systems during ascent, reentry and mission phases of the spacecraft and to optimize the radiator system configuration via enumeration of parameter sets.

The Analysis is covered in the following chapter. The numerical techniques employed are discussed in Chapter III, while the preparation for the program units describing material properties is deferred to the Appendices.

The computer code is described in a separate manual [29].

When the contract began over two years ago, the emphasis on radiator design was primarily on the space base heat rejection system. Since that time, the emphasis has shifted so that it is now heavily placed on the shuttle vehicle. Furthermore, the responsibility of developing the heat rejection system has been shifted from the Power Generation Branch and the design philosophy has changed to an integrated system which includes waste heat from sources other than the power generation system which was anticipated as the sole source of waste heat when the contract began. Due to the shift in design philosophy, the report will not recommend detailed design considerations, although once heat loads from other sources are known, the analysis presented here can be used to aid in the design of an integrated radiator system.



## II. ANALYSIS

The analysis is presented in three major parts. In the first part are developed the governing equations of transient heat transfer within and from the radiator system; these equations are the basis of the numerical simulation. The second part of the analysis is devoted to the computation of design parameters dictated by operational conditions, while third part covers the development of thermodynamic and transport properties of structural materials, coolant fluids, and the atmosphere.

### A. System Description

Two radiator systems are considered in this analysis. The first system considers a flat radiator panel divided by regularly spaced coolant channels all having identical inlet coolant fluid properties. This system is treated rigorously. The second system considers a non-symmetrical radiator panel. The non-symmetrical conditions can be caused by a curvature in the radiator panel or by coolant channels that are parallel but not equally spaced or formed in the shape of a U. This system is treated in an approximate manner. The details of the non-symmetrical analysis are given in Section II D.

For the purposes of system simulation, the radiator system consists of four components:

- (i) the fin
- (ii) the coolant fluid
- (iii) the coolant flow channel
- (iv) the meteoroid protection layer

Two coordinate systems were introduced (see Figure 1), one for the fin and the other for the flow channel and the meteoroid protection layer. The rectangular Cartesian system  $(x,z)$  for the fin has its  $z$ -axis parallel to the tube axis, starting at the inlet plane, and its  $x$ -axis passing through the line of profile symmetry, with  $x = 0$  designating the root of the fin. Cylindrical coordinates  $(r,z)$  are used for both the tube and the meteoroid protection layer, with the  $z$ -axis along the tube.

The radiator system is describable in terms of the following six dependent state variables:

(i) for the fin:

the fin temperature  $T_f(t;x,z)$

(ii) for the coolant fluid:

the fluid temperature  $T_c(t;z)$

the fluid pressure  $p(t;z)$

the fluid velocity  $w(t;z)$

(iii) for the fluid flow channel:

the tube wall temperature  $T_w(t;r,z)$

(iv) for the meteoroid protection layer

the protection layer temperature  $T_m(t;r,z)$

These six dependent variables must satisfy four equations of energy conservation, one for each component of the system, and further, the equations of mass conservation and of momentum balance for the fluid. These conservation equations take on the form of partial differential equations subject to initial and boundary conditions. Finally, the energy conservation equation for the fin involves the net radiant and convective power fluxes leaving the fin.

In the following Sections, II B.2 through II B.5 are discussed, in that order, the six principal governing equations associated with the four system components listed above, the subsidiary equations governing the radiative heat exchange and, lastly, the convective heat transfer between coolant fluid and tube wall and between the fin and the atmosphere during ascent and reentry.



## B. HEAT TRANSFER

### 1. Introduction

The objective of the radiator simulation is to predict both the transient and the steady state heat transfer characteristics under pre-determined operating conditions, stationary in the latter case and dynamic in the former. Both cases were treated as initial value problems, and the principal governing equations are partial differential equations which are linear in the time-derivatives of first order at most.

Under stationary boundary conditions steady state will be reached, regardless of initial conditions\*, as all partial derivatives with respect to time vanish on account of dissipative effects within the system. For the computer simulation of steady state conditions this means that the process of advancing in time can be discontinued as soon as all variables  $y_i$ ,  $i = 1, 2, \dots N$ , have reached their expected asymptotic values  $(y_i)_\infty$  with sufficient accuracy, that is when for some chosen  $\epsilon_i$

$$\delta_i = |y_i - (y_i)_\infty| \leq \epsilon_i \quad (1.1)$$

has been reached. The expected asymptotic values  $(y_i)_\infty$  can be estimated on the basis of the recognition that for large enough values of the time,  $t > t_M$

$$y_i \rightarrow (y_i)_\infty \pm ae^{-bt} \quad (1.2)$$

with,  $a > 0$ ,  $b > 0$ , to be determined;  $t > t_M$ . The evaluation  $\delta_i$  during the numerical integration is covered in Chapter III.

---

\*Subject to certain continuity requirements which are discussed in Chapter III.

## 2. The Fin

The objective of this section is to develop the energy equation for the fin. The derivation is based on the assumptions that the thermal conductivity and specific heat of the fin material are functions of temperature while the fin density is constant. The energy balance for the fin accounts for both radiative and convective fluxes from the fin surfaces. The development of a method to predict the net radiant flux from the fin surfaces can be found in Section II-6. The procedure used for the evaluation of the convective flux from the fin surface is outlined in Section II-7.

The energy balance on a differential volume of the fin can be expressed as

$$\frac{\partial}{\partial x} \left[ k_f A_x \frac{\partial T_f}{\partial x} \right] dx + \frac{\partial}{\partial z} \left[ k_f A_z \frac{\partial T_f}{\partial z} \right] dz + q''_{\text{net,rad}} A_s + q''_{\text{aero}} A_s = \rho_f V c_{pf} \frac{\partial T_f}{\partial t} \quad (2.1)$$

where  $T$  is the fin temperature and the coordinate system is shown in Fig. 1. The areas  $A_x$  and  $A_z$  represent the cross-sectional areas of the fin perpendicular to the heat flow in the  $x$  and  $z$  directions, and  $A_s$  represents the total non-adiabatic surface area of the fin element. The symbol  $V$  is the volume of the fin element. The properties of the fin material are represented by the symbols  $k_f$ ,  $\rho_f$  and  $c_{pf}$  which stand for the thermal conductivity, density and specific heat, respectively. The terms  $q''_{\text{net,rad}}$  and  $q''_{\text{aero}}$  appearing in Eq. 2.1 denote the radiative and convective heat gain from the surroundings to the fin surfaces.

The first two terms in Eq. 2.1 constitute the net conduction of energy into the fin element. The third and fourth terms on the left hand side of the equation stand for the net radiation and convection gain, respectively, from the surfaces of the fin element. The term of the right

hand side of the equation represents the storage of internal energy within the fin element.

The appropriate areas and the volume to be substituted into Eq. 2.1 are

$$dA_x = 2[s_f - cx] dz \quad (2.2)$$

$$dA_z = 2[s_f - cx] dx \quad (2.3)$$

$$dA_s = dx dz / [c^2 + 1]^{1/2} \quad (2.4)$$

$$dV = 2[s_f - cx] dx dz \quad (2.5)$$

where  $s_f$  is the fin half thickness at its root and  $c$  is the negative slope of the fin side surfaces.

Substituting Eqs. 2.2 through 2.5 into Eq. 2.1 and simplifying the resulting equation yields

$$\rho_f c_{pf} \frac{\partial T_f}{\partial t} = k_f \left[ \frac{\partial^2 T_f}{\partial x^2} + \frac{\partial^2 T_f}{\partial z^2} \right] + \frac{dk_f}{dT} \left[ \left( \frac{\partial T_f}{\partial x} \right)^2 + \left( \frac{\partial T_f}{\partial z} \right)^2 \right] \quad (2.6)$$

$$- \left( \frac{ck_f}{t-cx} \right) \frac{\partial T}{\partial x} + \frac{\sqrt{c^2+1}}{2(t-cx)} [q''_{\text{net,rad}} + q''_{\text{aero}}]$$

The Normalization of Eq. 2.6 is achieved by defining nine dimensionless quantities:

Let

$$\xi = x/H \quad (2.7)$$

$$\zeta = z/L \quad (2.8)$$

represent the nondimensional fin coordinate perpendicular and parallel to the tube, respectively. Then

$$\tau = tw_o/L \quad (2.9)$$

is the dimensionless time. The symbol  $w_o$  stands for the velocity of the coolant fluid entering the tube.

The nondimensional fin temperature is defined as

$$\theta_f(\tau, \xi, \zeta) = \frac{T_f(\tau, \xi, \zeta)}{T_o} \quad (2.10)$$

where  $T_o$  is the temperature of the coolant fluid entering the tube.

The dimensionless conduction parameter  $N_{Nc}$  and the Fourier number are defined as



$$N_{Nc} = H\sigma T_o^3/k_f \quad (2.11)$$

$$N_{Fo} = \frac{\alpha_f L}{w_o H^2} = \frac{k_f L}{\rho_f c_{pf} w_o H^2} \quad (2.12)$$

and the nondimensional geometrical quantities are defined as

$$\bar{s}_f = s_f/H \quad (2.13)$$

$$B(\xi) = 2(\bar{s}_f - c\xi) \quad (2.14)$$

$$D = (c^2 + 1)^{1/2} \quad (2.15)$$

where  $c$  is the negative slope of the fin side surfaces. For a non-tapered fin  $c$  is zero.

Both the net radiative and convective flux terms appearing in Eq. 2.6 are normalized by dividing each term by  $\sigma T_o^4$ , or

$$\tilde{q}_{net,rad} = \frac{q''_{net,rad}}{\sigma T_o^4} \quad (2.16)$$

$$\tilde{q}_{aero} = \frac{q''_{aero}}{\sigma T_o^4} \quad (2.17)$$

The energy equation, Eq. 2.6, may now be written in terms of the nondimensional quantities given in Eqs. 2.7 through 2.17. The resulting nondimensional equation is

$$\begin{aligned} \dot{\theta}_f = N_{Fo} \left\{ (\theta_f)_{\xi\xi} + \left(\frac{H}{L}\right)^2 (\theta_f)_{\zeta\zeta} + \frac{T_o}{k_f} \frac{dk_f}{dt} \left[ (\theta_f)_{\xi}^2 + \left(\frac{H}{L}\right)^2 (\theta_f)_{\zeta}^2 \right] \right. \\ \left. - 2 \frac{c}{B} (\theta_f)_{\xi} + \frac{D}{B} N_{Nc} [\tilde{q}_{net,rad} + \tilde{q}_{net,conv}] \right\} \end{aligned} \quad (2.18)$$

The dot superscript appearing in Eq. 2.18 denotes differentiation with respect to nondimensional time and the subscripts  $\xi$  and  $\zeta$  represent partial differentiation with respect to the dimensionless coordinates indicated.

The normalized energy equation, Eq. 2.18, defines the rate of change of the dimensionless fin temperature  $\theta_f$ .

The Boundary Conditions for Eq. 2.18 are taken as follows:

- (i) The fin root is at the temperature of the outside of the tube.
- (ii) The fin tip is insulated.
- (iii) The portion of the fin in contact with the inlet manifold is at the outside temperature of the manifold.
- (iv) The portion of the fin in contact with the exit manifold is at the temperature of the outlet manifold.

Written in mathematical terms these four boundary conditions are

$$\theta_f(\tau, 0, \zeta) = \theta_w(\tau, \xi_o, \zeta) \quad (2.19)$$

$$\frac{\partial \theta_f}{\partial \xi}(\tau, 1, \zeta) = 0 \quad (2.20)$$

$$\theta_f(\tau, \xi, 0) = \theta_w(\tau, \xi_o, 0) \quad (2.21)$$

$$\theta_f(\tau, \xi, 1) = \theta_w(\tau, \xi_o, 1) \quad (2.22)$$

Implied in Eqs. 2.21 and 22 is, firstly, that the fluid temperature in the manifolds be equal to the fluid temperature at the tube inlet (inlet manifold) and at the tube exit (exit manifold) and that it remain unaltered along the manifold and vary only in time; secondly, that the temperature drop through the manifold wall be equal to that through the tube wall. The latter assumption is well justified because the temperature drop is exceedingly small.

It may be noted in connection with the boundary conditions that the radiative interactions between manifold and fin as well as between manifold and tube are not taken into consideration at this time.

The Initial Condition for Eq. 2.18 may be any arbitrary relation representing a continuous temperature distribution over the fin, including the boundaries. The selection of the initial fin temperature distribution is left to the user.

### 3. The Coolant Fluid

The objective is to develop a unified treatment of all possible coolant fluids, that is gases, dielectric fluids and liquid metals. Three principal governing equations are sought which, together with the necessary thermodynamic and transport properties specified for each fluid of interest, define the fluid temperature  $T_c$ , the fluid pressure  $p$  (and thus the thermodynamic state of the fluid) and the fluid velocity  $w$ , all as functions of time  $t$  and axial distance  $z$ .

The Continuity Equation for one dimensional flow through channels of constant cross-sections reads

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial z} (\rho w) = 0 \quad (3.1)$$

where  $\rho$  represents the fluid density. Replacing the density through the thermal equation of state

$$p = p(\rho, T_c) \quad (3.2)$$

renders the continuity equation in terms of derivatives of the primary variables  $T_c$ ,  $p$  and  $w$ :

$$\kappa \frac{\partial p}{\partial t} - \beta \frac{\partial T_c}{\partial t} = - \frac{\partial w}{\partial z} + w \left( \beta \frac{\partial T_c}{\partial z} - \kappa \frac{\partial p}{\partial z} \right) \quad (3.3)$$

where  $\kappa$  and  $\beta$  stand for the isothermal compressibility and the isobaric expansion coefficient, respectively

$$\kappa = \frac{1}{\rho} \left( \frac{\partial \rho}{\partial p} \right)_{T_c} \quad (3.4)$$

$$\beta = - \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T_c} \right)_{\rho} \quad (3.5)$$

### The Momentum Equation

$$\rho \left( \frac{\partial w}{\partial t} + w \frac{\partial w}{\partial z} \right) = - \frac{\partial p}{\partial z} - \frac{4f}{d} \rho \frac{w^2}{2} + \rho a_z \quad (3.6)$$

constitutes the balance between inertia forces on the left-hand side, pressure forces, wall friction and external field forces (gravity) in axial direction, on the right-hand side of Eq. 3.6. The Fanning friction factor  $f$  is a function of the Reynolds-number  $N_{Re} = d w / \nu$ , where  $d$  and  $\nu$  stand for the tube diameter and the kinematic viscosity, respectively and subscript  $o$  designates the fluid inlet conditions. The following relations are used to compute the Fanning friction factor:

for  $N_{Re} < 2300$  (laminar flow)

$$4f = \frac{64}{N_{Re}} \quad (3.7)$$

for  $2300 \leq N_{Re} \leq 10^6$  (Ref. 1)

$$4f = 0.0054 + 0.396 N_{Re}^{-0.3} \quad (3.8)$$

for  $N_{Re} > 10^6$  (Ref. 2)

$$4f = 0.0032 + 0.221 N_{Re}^{-0.237} \quad (3.9)$$

Equations 3.8 and 3.9 could be replaced by a single equation (Ref.3)

$$4f = [0.86859 \ln \frac{N_{Re}}{1.964 \ln N_{Re}} - 3.8215]^{-2} \quad (3.10)$$

which, however, requires more computational effort.

The Energy Equation. Let  $u$ ,  $h$ ,  $\bar{h}_c$  and  $T_w = T_w(t; r_1, k)$  represent, respectively the internal energy and the enthalpy of the fluid, the convective film coefficient and the tube wall temperature at the fluid-wall interface. For one-dimensional flow through channels of constant cross-section and with heating or cooling from the wall, conservation of energy requires that, with respect to a stationary reference frame

$$\begin{aligned} \frac{\partial}{\partial t} [\rho(u + \frac{w^2}{2})] + \frac{\partial}{\partial z} [\rho w (h + \frac{w^2}{2} - a_z z)] &= \frac{4}{d} \bar{h}_c (T_w - T_c) \\ + \frac{\partial}{\partial z} (k_c \frac{\partial T_c}{\partial z}) \end{aligned} \quad (3.11)$$

The first term constitutes the storage of thermal and kinetic energies; the storage of potential energy, being negligibly small for expected accelerations, is ignored. The second term on the left-hand side stands for the convection of thermal, kinetic and potential energies as well as the power associated with the pressure. The right-hand side contains, firstly, the convective heat transfer from the channel wall to the fluid and, secondly, the axial heat conduction term which will later be shown to be negligibly small for all fluids. The factor  $4/d$  in front of the convective heat transfer term results from the ratio of the channel circumference to the cross-sectional area, evaluated for a circular tube. The symbol  $k_c$  represents the thermal conductivity of the coolant.

Given a caloric equation of state

$$u = u(\rho, T_c) \quad (3.12)$$

or an equivalent expression for the specific heat,  $c_p$ , one can write, with the aid of Eqs. 3.2, 4 and 5

$$dh = c_p dT_c + \frac{1}{\rho} (1 - \beta T_c) dp \quad (3.13)$$

$$du = (c_p - \frac{p\beta}{\rho}) dT_c + \frac{1}{\rho} (\kappa p - \beta T) dp \quad (3.14)$$

and then recast Eq. 3.11 in terms of the derivatives of the principal variables  $T_c$ ,  $p$  and  $w$ :

$$\begin{aligned} \rho c_p \frac{\partial T_c}{\partial t} - \beta T_c \frac{\partial p}{\partial t} + \rho w \frac{\partial w}{\partial t} &= \frac{4}{d} \bar{h}_c (T_w - T_c) \\ &+ \frac{\partial}{\partial z} (k_c \frac{\partial T_c}{\partial z}) - \rho w [c_p \frac{\partial T_c}{\partial z} + \\ &\frac{1}{\rho} (1 - \beta T) \frac{\partial p}{\partial z} + w \frac{\partial w}{\partial z} - a_z] \end{aligned} \quad (3.15)$$

Thus, three governing equations, Eqs. 3.3, 3.6 and 3.15, have been established which define the three principal variables  $T_c$ ,  $p$  and  $w$ , provided the initial and boundary conditions are properly specified and the convective film coefficient can be predicted. Since the normalization of these equations, followed by an order-of-magnitude comparison will indicate that the conductive term in Eq. 3.15 is negligible so as to simplify the boundary conditions, the discussion of the initial and boundary conditions is deferred until after the normalization.

The convective film coefficient is computed from the following relationships between the Nusselt number  $N_{Nu} = \bar{h}_c d/k_c$ , the Reynolds number

$N_{Re}$  and the Prandtl number  $N_{Pr} = \mu c_p / k_c$  where  $\mu$  represents the dynamic viscosity:

For  $N_{Pr} < 0.1$  (liquid metals)

$$N_{Nu} = 6.5 + 0.025 (N_{Re} N_{Pr})^{0.8} \quad (3.16)$$

which produces Nusselt numbers between those appropriate for uniform heat flux (Martinelli) and for uniform wall temperature (Seban and Shimazaki) (Ref. 4).

For  $N_{Pr} > 0.1$  and

for  $N_{Re} < 2300$  (laminar flow, Ref. 5)

$$N_{Nu} = 3.65 + \frac{0.0668 (N_{Re} N_{Pr}^\delta)}{1 + 0.045 (N_{Re} N_{Pr}^\delta)} \quad (3.17)$$

for  $N_{Re} \geq 2300$  (turbulent flow, Ref. 5)

$$N_{Nu} = 0.116 (N_{Re}^{0.667} - 125) N_{Pr}^{0.333} (1 + \delta) \quad (3.18)$$

where  $\delta = d/L$  stands for the tube diameter-over-length ratio.

The Normalization of Eqs. 3.3, 6 and 15 is carried out for the purpose of scaling and performing an order-of-magnitude comparison. The computational effort is also reduced in the process.

Let

$$\zeta = z/L \quad (3.19)$$

$$\tau = tw_o/L \quad (3.20)$$



represent the nondimensional axial distance and the nondimensional time, respectively, and let the dimensionless state variables be defined as

$$\theta_c(\tau, \zeta) = \frac{T_c(t, z)}{T_o} \quad (3.21)$$

$$\pi(\tau, \zeta) = \frac{p(t, z)}{p_o} \quad (3.22)$$

$$\omega(\tau, \zeta) = \frac{w(t, z)}{w_o} \quad (3.23)$$

and to represent the nondimensional temperature, pressure and velocity of the fluid, that is, the principal dependent fluid flow variables. The subscript "o" designates the constant reference state of the fluid at the tube entrance. Introduce next the nondimensional density

$$v(\tau, \zeta) = \frac{\rho(t, z)}{\rho_o}, \quad (3.24)$$

the nondimensional radial distance from the channel axis

$$\eta = \frac{2r}{d}, \quad (3.25)$$

and the nondimensional tube wall temperature

$$\theta_w(\tau; \eta, \zeta) = \frac{T_w(t; r, z)}{T_o} \quad (3.26)$$

Notice, that all dependent variables lie between zero and unity, except  $v$  and  $\omega$  whose product ( $v\omega$ ) remains essentially equal to unity with neither  $v$  nor  $\omega$  departing far from unity. The reference temperature  $T_o$  is, under normal operating conditions, the highest temperature in the system.

Furthermore, consider the following  $\phi$ -values to vary along the channel axis:

$$\phi_{\beta} = T_o^{\beta} \quad (3.27)$$

$$\phi_{\kappa} = p_o^{\kappa} \quad (3.28)$$

$$\phi_{cp} = \frac{c_p}{c_p (\rho_o, T_o)} = \frac{c_p}{c_{p,o}} \quad (3.29)$$

$$\phi_k = \frac{k}{k (\rho_o, T_o)} = \frac{k}{k_o} \quad (3.30)$$

and finally, the following constant F-parameters:

$$F_p = \frac{p_o}{\rho_o w_o^2} \quad (3.31)$$

$$F_f = 4f \frac{L}{d} = \frac{4f}{\delta} \quad (3.32)$$

$$F_z = \frac{p_o}{\rho_o c_{p,o} T_o} \quad (3.33)$$

$$F_M = F_z / F_p = \frac{w_o^2}{C_{p,o} T_o} \quad (3.34)$$

Let the dot above a variable designate partial differential with respect to the nondimensional time  $\tau$  and the subscript  $\zeta$  partial differentiation with respect to the dimensionless axial coordinate  $\zeta$ . Then the principal conservation equations, Eqs. 3.3, 3.6 and 3.15 read, respectively and in nondimensional form:

$$\phi_\beta [\dot{\theta}_c + \omega(\theta_c)_\zeta] = \omega_\zeta + \phi_\kappa (\dot{\pi} + \omega\pi_\zeta) \quad (3.35)$$

$$\dot{\omega} + \omega \omega_\zeta = -\frac{F_p}{v} \pi_\zeta - F_f \frac{\omega^2}{2} \quad (3.36)$$

$$\begin{aligned} \dot{\theta}_c - F_z \frac{\phi_\beta}{\phi_{cp}} \frac{\theta_c}{v} + F_M \frac{1}{\phi_{cp}} \omega \omega_\zeta &= \delta \frac{4N_{Nu}}{N_{Re} N_{Pr}} \cdot \frac{1}{v\phi_{cp}} (\theta_\omega - \theta_c) + \frac{\delta^2}{4N_{Nu}} \\ &\times [\phi_\kappa (\theta_c)_\zeta]_\zeta - \omega(\theta_c)_\zeta + \\ &F_z \frac{1-\theta_c \phi_\beta}{\phi_{cp}} \cdot \frac{1}{v} \pi_\zeta + F_M \frac{1}{\phi_{cp}} \omega \omega_\zeta \end{aligned} \quad (3.37)$$

These are the three equations which define the three time rates of change,  $\dot{\theta}_c$ ,  $\pi$  and  $\omega$ . It can be seen from Eq. 3.37 that the axial conduction is always small, of an order less than  $\delta^2$  (since  $N_{Nu} > 3$ ), when compared with the convective term, unless the fluid should reach the wall temperature within the very first small fraction of the tube length. Axial conduction is hence ignored as  $\delta^2 \approx 10^{-6}$ , and the order of differentiation of Eq. 3.37 is reduced to one.

$$\dot{\theta}_c - F_z \frac{\phi_\beta}{\phi_{cp}} \frac{\theta_c}{v} \dot{\pi} + F_M \frac{1}{\phi_{cp}} \dot{\omega} = \frac{4N_{Nu}}{\delta N_{Rc} N_{Pr}} \frac{\theta_w - \theta_c}{v \phi_{cp}} - \left\{ \omega (\theta_c)_\zeta + F_z x \right. \\ \left. \frac{1 - \theta_c \phi_\beta}{v \phi_{cp}} \pi_\zeta + \frac{F_M}{\phi_{cp}} \omega_\zeta \right\} \quad (3.38)$$

The Boundary Conditions to be imposed on Eqs. 3.35, 3.36 and 3.38 are chosen, at the channel entrance, to be

- (i) mass flow rate  $\dot{m}$ , prescribed as a function of time
- (ii) constant inlet pressure  $P_o$ .
- (iii) continuous transition, of the inlet fluid temperature, from an initial temperature  $\theta = \theta_i$  to the constant operational temperature  $\theta(t,0) = 1$ , or an arbitrarily prescribed inlet fluid temperature that is a continuous function of time

These boundary conditions accommodate the calculation of the steady state conditions as well as of the most likely start-up operation toward stationary operating conditions. Notice that there are no step changes implied in any of the dependent variables which is essential for the numerical integration. Writing these boundary conditions more specifically, one gets at  $\zeta = 0$

$$\theta(\tau, 0) = 1 - (1 - \theta_i) e^{-7\tau} \quad (3.39)$$

$$\pi(\tau, 0) = 1 \quad (3.40)$$

$$\omega(\tau, 0) = \frac{1}{v(\pi, \theta)} \quad (3.41)$$

Here, the time constant was chosen arbitrarily so as to have the fluid reach, within 0.1%, its steady inlet temperature at the inlet during the

time interval that it takes a fluid particle to pass through the channel. Equation 3.41 is given through the thermal equation of state, Eq. 3.2.

The Initial Conditions appropriate to the system of Eqs. 3.35, 3.36 and 3.38 are derived from the requirement that the flow should initially be at steady state with the fluid inlet temperature equal to the uniform channel wall temperature. Setting the time derivatives equal to zero in Eqs. 3.35, 3.36 and 3.38 results in a system of three ordinary differential equations which are linear in  $d\pi/d\zeta$ ,  $d\theta_c/d\zeta$  and  $d\omega/d\zeta$ :

$$F_p \frac{d\pi}{d\zeta} + \frac{d\omega}{d\zeta} = -F_f v \frac{\omega^2}{2} \quad (3.42)$$

$$F_z \frac{\phi_\beta}{v} \frac{d\pi}{d\zeta} + \frac{d\theta_c}{d\zeta} + \frac{F_M}{\phi_{cp}} \omega \frac{d\omega}{d\zeta} = \frac{4N_{Nu}}{\delta N_{Rc} N_{Pr}} (\theta_w - \theta_c) \quad (3.43)$$

$$\phi_K \frac{d\pi}{d\zeta} - \phi_\beta \frac{d\theta_c}{d\zeta} + \frac{1}{\omega} \frac{d\omega}{d\zeta} = 0 \quad (3.44)$$

These equations can be solved subject to the boundary conditions at  $\zeta = 0$

$$\left. \begin{aligned} \pi &= 1 \\ \theta_c &= \theta_i \\ \omega &= \frac{1}{v(\pi, \theta_c)} \end{aligned} \right\} \quad (3.45)$$

provided the function  $\theta_w = \theta_w(0; 1, \zeta)$  prescribing the initial channel wall temperature is specified. For the present analysis  $\theta_w$  was set equal to  $\theta_i$ . Equations 3.42 through 3.45 define the initial flow field.

Quasi-Steady Flow. It may be recognized that the momentum transport takes place at a much smaller time scale than the transport of thermal energy in that the pressure and the velocity fields adjust virtually

instantaneously to a change in flow inlet conditions while the response of the temperature field to a change in channel wall temperature takes considerably longer, the reason being that the pressure perturbations propagate along the channel with the speed of sound. Unless one is specifically interested in the motion of sound waves one may consider the dynamics of the flow field, that is the pressure and velocity distributions, as part of the boundary conditions and imposed instantly and adiabatically by the flow inlet conditions.

The fluid temperature remains stationary during the dynamic adjustment and the time rate of change of both pressure and temperature remain small since ordinarily the pressure gradient remains balanced by the wall shear (and by the convective acceleration in the case of a gaseous coolant medium). Consequently, Eqs. 3.42 and 3.44 may serve to establish the pressure and velocity fields at all times, subject to boundary conditions given by Eqs. 3.40 and 3.41 while the temperature field remains defined by Eqs. 3.35, 36, 38 and the initial conditions discussed above. Particularly, solving Eqs. 3.35, 36 and 38 for  $\dot{\theta}_c$  gives the differential equation which governs the temperature field:

$$\dot{\theta}_c = \frac{1}{1 - \frac{F_z}{\phi_{cp}} \frac{\phi_\beta^2}{\phi_\kappa} \frac{\theta}{v}} \left\{ \frac{4N_{Nu}}{\delta N_{Re} N_{Pr}} (\theta_w - \theta_c) \right. \\ \left. + F_z \frac{\phi_\beta}{\phi_{cp} \phi_\kappa} \times \left[ \phi_\beta \omega(\theta_c)_\zeta - \frac{\theta_c}{v} \omega_\zeta \right] \right. \\ \left. - \omega[(\theta_c)_\zeta + F_M F_f \frac{1}{\phi_{cp}} \frac{\omega^2}{2}] \right\} \quad (3.46)$$

This completes the discussion of the development of the governing differential equations for the coolant fluid. All equations are solved numerically as discussed in Chapter III. The thermodynamic properties  $c_p$ ,  $\beta$ , and  $\kappa$  are derived from the thermal equation of state, Eq. 3.2 and from the zero-pressure specific heat or other available properties. All thermodynamic functions as well as the transport properties  $\kappa$  and  $\mu$  are considered, in general, as functions of two state variables, that is, of  $\rho$  and  $T$  or of  $p$  and  $T$ , as discussed in Section E of Chapter II.

#### 4. The Flow Channel

The flow channel is treated as a circular tube with inner radius  $r_i = d/2$  and outer radius  $r_o = r_i + s_t$ . The tube wall temperature  $T_w(t; r, z)$  is defined through the familiar equation of energy conservation, written for the case of circular symmetry:

$$\frac{\partial T_w}{\partial t} = \alpha_w \left[ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T_w}{\partial r} \right) + \frac{\partial^2 T_w}{\partial z^2} \right] + \frac{1}{(\rho c)_w} \frac{dk_w}{dT} \left[ \left( \frac{\partial T_w}{\partial r} \right)^2 + \left( \frac{\partial T_w}{\partial z} \right)^2 \right], \quad (4.1)$$

where  $\alpha_w$  represents the thermal diffusivity of the wall material and  $(\rho c)_w$  its volumetric heat capacity. The boundary conditions are

at  $r = r_i$

$$\bar{h}_c (T_w - T_c) = k_w \frac{\partial T_w}{\partial r} \quad (4.2)$$

at  $r = r_o$

$$k_w \frac{\partial T_w}{\partial r} - \psi k_f \frac{\partial T_f}{\partial x} - (2\pi - \psi) k_m \frac{\partial T_m}{\partial r} = 0 \quad (4.3)$$

Here

$\bar{h}_c$  is the convective film coefficient,

$T_c$  the fluid temperature,

$k$  the thermal conductivity,

$\psi$  the portion of outer tube circumference in contact with fins, expressed in radians,

and the subscripts  $w$ ,  $f$  and  $m$  designate, respectively, tube wall, fin and meteoroid protection layer. Equations 4.2 and 3 constitute the continuity of the heat flux at the fluid-wall interface and at the wall-fin and wall-protection layer interfaces. Circumferential temperature variations are ignored.



After introducing

$$\eta = \frac{r}{r_i}, \quad \zeta = \frac{x}{H}; \quad \tau = \frac{tw_o}{L}; \quad \theta_w = \frac{T_w}{T_o},$$

$$\phi_c = \frac{k_c}{k_w}, \quad \phi_f = \frac{k_f}{k_w} \frac{r_i}{H},$$

$$\phi_m = \frac{k_m}{k_w} \quad N_{Bi} = \frac{\bar{h}_c d}{k_w}, \quad (4.4)$$

$$\delta = \frac{d}{L}, \quad \phi_{\kappa w} = \frac{T_o}{k_w} \frac{dk_w}{dT}, \quad \phi_{Fo,w} = \frac{\alpha_w L}{w_o r_i^2}$$

with  $H$  representing the fin height, that is, the distance between the fin root and fin tip, one may recast Eqs. 4.1, 2 and 3 to read

$$\begin{aligned} \dot{\theta}_w = \phi_{Fo,w} \left\{ \frac{1}{\eta} \frac{\partial}{\partial \eta} \left( \eta \frac{\partial \theta_w}{\partial \eta} \right) + \left( \frac{\delta}{2} \right)^2 \frac{\partial^2 \theta_w}{\partial \zeta^2} \right. \\ \left. + \phi_{\kappa w} \left[ \left( \frac{\partial \theta_w}{\partial \eta} \right)^2 + \left( \frac{\delta}{2} \right)^2 \left( \frac{\partial \theta_w}{\partial \zeta} \right)^2 \right] \right\} \end{aligned} \quad (4.5)$$

at  $\eta = 1$

$$\frac{1}{2} N_{Bi} (\theta_w - \theta_c) = \frac{\partial \theta_w}{\partial \eta} \quad (4.6)$$

at  $\eta = \eta_0$

$$\frac{\partial \theta_w}{\partial \eta} - \psi \phi_f \frac{\partial \theta_f}{\partial \xi} - (2\pi - \psi) \phi_m \frac{\partial \theta_m}{\partial \eta} = 0 \quad (4.7)$$

The superscript dot represents partial differentiation with respect to nondimensional time  $\tau$ . As before in the treatment of the coolant fluid one recognizes that, with  $\delta^2 \approx 10^{-6}$ , axial conduction remains insignificant. Thus, Eq. 4.5 takes on this final form

$$\dot{\theta}_w = \phi_{Fo,w} \left\{ \frac{1}{\eta} \frac{\partial}{\partial \eta} \left( \eta \frac{\partial \theta_w}{\partial \eta} \right) + \phi_{kw} \left( \frac{\partial \theta_w}{\partial \eta} \right)^2 \right\} \quad (4.8)$$

Equations 4.6, 7 and 8 define the tube wall temperature, provided Eqs. 4.6 and 7 hold at some initial time and the initial temperature  $\theta_w(0; \eta, \zeta)$  is prescribed as a sufficiently smooth function of  $\eta$  and  $\zeta$ .

Low Biot Number. When one compares possible heat fluxes at the fluid-wall interface with the possible radiant fluxes from the outer surface of the meteoroid protection layer covering the tube, one concludes that the maximum fluxes occur at the inner tube wall and that the Biot number  $N_{Bi}$  in Eq. 4.6 is the largest ratio to be expected of external to internal thermal conductances. Thus, if  $N_{Bi}$  is small, say less than 0.05 (Ref. 6), then the temperature variation inside the tube wall is too small for experimental detection and a computation of the detailed temperature distribution on the basis of Eq. 4.8 cannot be justified as the associated computational effort is considerable.

In cases where the equivalent Biot number, representing the total thermal resistance within the channel wall and the protection layer, is small, the tube, the protection layer and a representative portion of the fin root are combined into a single control volume as depicted in Fig. 2. The equivalent Biot number  $\bar{N}_{Bi}$  and the chosen limit are

$$\bar{N}_{Bi} = N_{Nu} \frac{k_c}{d} \left( \frac{s_w}{k_w} + \frac{s_m}{k_m} \right) < 0.05 \quad (4.9)$$

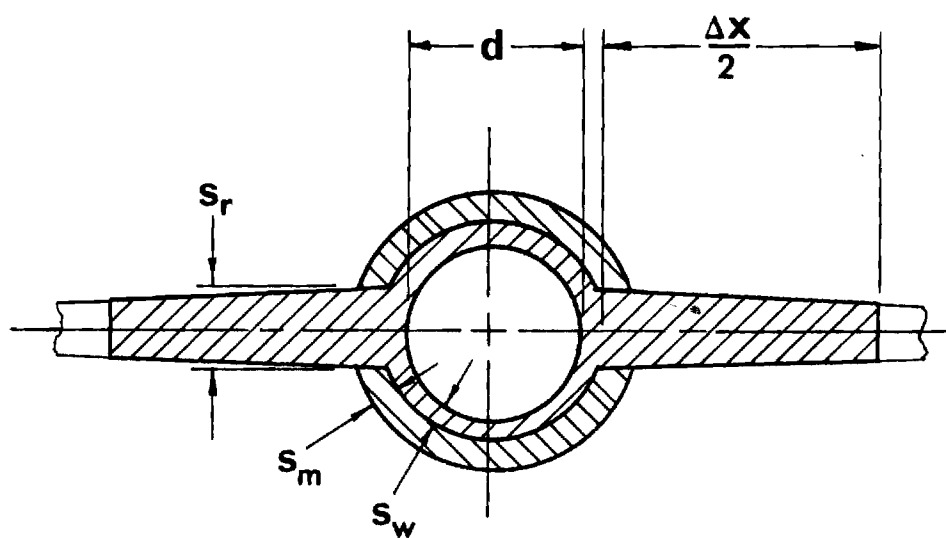


Figure 2. Control Volume for Low-Biot Number Cases

where  $s_w$  and  $s_m$  stand for, respectively, the tube wall and the protection layer thicknesses. The combined volumetric heat capacity per unit of axial distance for the control volume consists of three parts, the first one for the tube wall, the second for the protection layer and the third for the first half fin element:

$$\pi d (\rho c s)_w \left\{ \left(1 + \frac{s_w}{d}\right) + \left(1 + 2 \frac{s_w}{d} + \frac{s_m}{d}\right) \frac{(\rho c s)_m}{(\rho c s)_w} + \frac{\Delta x}{\pi d} \frac{s_r}{s_w} \times \right. \\ \left. \left[1 - \frac{\Delta \xi}{4} \left(1 - \frac{s_t}{s_r}\right)\right] \frac{(\rho c)_f}{(\rho c)_w} \right\} = \lambda_1 \pi d (\rho c s)_w \quad (4.10)$$

where the subscripts  $w$ ,  $m$ ,  $f$ ,  $r$  and  $t$  designate, respectively, parameters of the tube wall, the meteoroid protection layer, the fin, the fin root and the fin tip and where

- $d$  is the tube diameter,
- $\rho$  the density,
- $s$  the thickness,
- $\Delta x$  the node spacing on the fin, and
- $\Delta \xi = \Delta x/H$ , the nondimensional node spacing

Heat enters the control volume, per units of time and axial distance, by convection from the fluid

$$\pi d \bar{h}_c (T_w - T_c),$$

and by convection and/or radiation from outside

$$(2\pi - \psi) \left(\frac{d}{2} + s_w + s_m\right) q''_m + \Delta x (q''_1 + q''_2)_f$$

where the subscripts 1 and 2 designate upper and lower fin sides, respectively, and where  $q''$  represents the sum of the convective and of the net radiant fluxes entering the outer surfaces. In the case of  $q''_m$ , the average over upper and lower portions of the outer circumference is to be taken.

Heat leaves the control volume through the fin by conduction, again per units of time and axial distance

$$2(s \frac{\partial T}{\partial x})_f$$

evaluated at  $x = \Delta x/2$ .

Combining the last four expressions into the energy balance leads to this expression

$$\lambda_1 \pi d (\rho c s)_w \frac{\partial T_w}{\partial t} = \pi d \bar{h}_c (T_w - T_c) + \pi d \lambda_2 q''_m + \Delta x (q''_1 + q''_2)_f + 2 (s \frac{dT_f}{dx}) \Big|_{\frac{\Delta x}{2}} \quad (4.11)$$

where  $\lambda_1$  is defined by Eq. 4.10 and

$$\lambda_2 = (2 - \frac{\psi}{\pi}) (\frac{1}{2} + \frac{s_w + s_m}{d}) \quad (4.12)$$

Equation 4.11 replaces Eqs. 4.6, 7 and 8 as well as Eqs. 5.3, and 5 governing the temperature distribution in the meteoroid protection layer and, lastly, Eq. 2.19 which constitutes one of the boundary conditions for the differential equation governing the fin temperature distribution. The only condition under which all the above equations may be replaced by the single equation, Eq. 4.11, is given by Eq. 4.9. Finally, it may be noted that Eq. 4.11 could also be normalized but since no new dimensionless groups result from such normalization it is omitted here.

### 5. The Meteoroid Protection Layer

The differential equation governing the temperature distribution within the meteoroid protection layer which covers the flow channel, is identical to that for the channel wall, Eq. 4.8, except for the two dimensionless parameters, the Fourier coefficient  $\phi_{Fo}$  and the conductivity temperature coefficient  $\phi_k$  which now must be evaluated for the protection layer material:

$$\phi_{Fo,m} = \frac{\alpha_m}{\alpha_w} \phi_{Fo,w} \quad (5.1)$$

$$\phi_{k,m} = \frac{T_o}{k_m} \frac{dk_m}{dT} \quad (5.2)$$

After ignoring the axial conduction for the reasons stated in Section 4 one obtains as the nondimensional energy conservation equation

$$\dot{\theta}_m = \phi_{Fo,m} \left\{ \frac{1}{\eta} \frac{\partial}{\partial \eta} \left( \eta \frac{\partial \theta_m}{\partial \eta} \right) + \phi_{k,m} \left( \frac{\partial \theta_m}{\partial \eta} \right)^2 \right\} \quad (5.3)$$

Two boundary conditions are required, one of which is given by Eq. 4.7 while the other one is dictated by the heat flux continuity at the outer boundary:

at  $r = r_e$

$$k_m \frac{\partial T_m}{\partial r} = q''_m \quad (5.4)$$

where  $q''$  is defined as the net flux entering both by radiation and/or aerodynamic heating. Equation 5.4 reads in nondimensional form at  $\eta = \eta_e$

$$\frac{\partial \theta_m}{\partial \eta} = \phi_{Nc} (\tilde{q}_{net,rad} + \tilde{q}_{aero}) \quad (5.5)$$

with  $\phi_{Nc}$  representing a local conductance parameter

$$\phi_{Nc} = \frac{\sigma d T_o^3}{2k_m} \quad (5.6)$$

and

$$\tilde{q}_{net,rad} = \frac{q''_{net,rad}}{\sigma T_o^4} \quad (5.7)$$

$$\tilde{q}_{aero} = \frac{q''_{aero}}{\sigma T_o^4} \quad (5.8)$$

Here,  $\sigma$  represents the Stefan-Boltzmann constant, and  $q''_{net,rad}$  and  $q''_{aero}$  the net radiant incident heat flux and the convective heat flux, respectively.

If the initial temperature distribution is given, and if Eqs. 4.7 and 5.5 hold initially, then the protection layer temperature  $\theta_m$  is completely defined as a function of time and location by Eqs. 5.3, 5 and 4.7, provided the incident radiation and the aerodynamic heating are prescribed as functions of time. The prediction of these incident heat fluxes is discussed in Section 6 and 7 of this Chapter.

In the case of low Biot numbers at the coolant fluid-tube wall interface the meteoroid protection layer is lumped together with the tube wall as discussed in Section 4 of Chapter II.

## 6. Radiation

The objective of this section is to develop a procedure to predict the net radiant heat flux incident on fin and tube surfaces which are exposed to any combination so solar, albedo and planetary irradiation. Included into the assessment of radiative transfer is the radiative energy exchange between fin panel and flow channel or its protective coating as well as the effect of structural panels in the vicinity of the fin system; however, not included is any possible gas radiation as could conceivably be encountered during reentry.

In seeking the proper mathematical model it is recognized, firstly, that the prevailing thermal radiant energy lies in either the visible (solar irradiation) or the infrared portion of the spectrum, and, secondly, that the fin system is coated, for the purpose of optical optimization, with a dielectric paint. Consequently, spectrally dependent optical properties must be dealt with, but, for the latter reason, the transfer matrix of radiative exchange can be expected to remain temperature insensitive over some range of operational conditions, a fact which is very much appreciated from the computational view point.

For the analysis, the fin surface  $A_f$ , the outer surface on the flow channel  $A_n$ , and the structural surface(s)  $A_n$ , are all considered as parts of an enclosure  $C$  which is completed by a set of arbitrarily concave, non-reflecting imaginary surfaces  $A_e$  which connect  $A_m$ ,  $A_f$  and  $A_n$  and along which is specified the emerging net radiant heat flux representing solar, albedo and planetary irradiation. The sum  $A_m + A_f + A_n + A_e$  is the inner surface  $A_c$  of the enclosure.

Three steps are necessary for the prediction of the incident net radiant heat flux  $q''_{\text{net,rad}}$ . Firstly, the elemental exchange areas\*

$$\frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} = \frac{\cos \phi_i \cos \phi_j}{\pi r^2} \quad (6.1)$$

---

\*For terminology and notations consult Ref. 7, Chapter 2.



need to be computed on the basis of the geometric relation between fin panels and flow channels. Here, the symbol  $r$  designates the distance between two area elements  $dA_i$  and  $dA_j$  which are visible from each other, and  $\phi_i$  and  $\phi_j$  represent the respective angles between  $r$  and the surface normals on  $dA_i$  and  $dA_j$ . The next step is to compute, from its definition, the radiosity or leaving radiant flux density,  $W_j$ :

$$W_j = \int_0^\infty W_{j,\lambda} d\lambda = \int_0^\infty \epsilon_{j,\lambda} E_{j,\lambda} d\lambda + \quad (6.2)$$

$$\int_{A_c} \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} \int_0^\infty (1 - \epsilon_{j,\lambda}) W_{i,\lambda} d\lambda dA_i$$

where  $W_{j,\lambda}$ ,  $E_{j,\lambda}$  and  $\epsilon_{j,\lambda}$  stand for the monochromatic radiosity, the monochromatic black body emissive power

$$E_{j,\lambda} = \frac{2 hc^2 n^2}{\lambda^5} \frac{1}{e^{(hc)/(\lambda k T_j)} - 1} \quad (6.3)$$

and the monochromatic hemispherical emittance, respectively; the subscripts  $i$  and  $j$  designate two discrete points on  $A_c$ , and  $\lambda$  represents the wavelength. The Eq. 6.3 constitutes Planck's law of monochromatic emissive power intensity;  $h$ ,  $c$  and  $k$  stand for, respectively, Planck's constant, the speed of light in vacuo and the Boltzmann constant. The third and last step is to calculate, on the basis of local energy balance, the net incident heat flux

$$(q''_{\text{net,rad}})_j = \int_0^\infty q''_{j,\lambda} d\lambda = \int_{A_c} (W_i - W_j) \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} dA_i \quad (6.4)$$

It should be obvious that Eqs. 6.1, 2 and 4 contain all the necessary fundamental principles but their evaluation will introduce a number of simplifications and modifications, each selected for the particular system of interest. Specifically, Eq. 6.4 will have to supplement Eq. 6.2 for portions of  $A_c$  where the heat flux is specified. More importantly, however, there is a choice to be made in view of the computational process regarding particularly Eq. 6.2. One may either solve the monochromatic version of Eq. 6.2  $n$  times for the  $n$  significant spectral intervals encountered and thus face the ultimate task of solving  $n \times N$  simultaneous linear algebraic equations when  $N$  discrete points on  $A_c$  need to be considered (possibly at several time steps during the calculation process) and then integrate the resulting total interchange areas over the spectrum (see Ch. 5.6 of Ref. 7). Or, one may force the non-gray surface analysis into a gray surface analysis by placing the burden of complexity on the evaluation of appropriate optical properties. Both techniques afford any desirable accuracy of allowing for the spectral differences in surface properties, limited only by available calculation time; but the latter technique was chosen because, as a result of this choice, the complexity remains at peripheral parts of the computer code which are more accessible for later modifications toward greater sophistication, also the complexity may turn out, in almost all cases, to reduce partly to simple hand calculations.

After introducing

$$\epsilon_i = \frac{\int_0^\infty \epsilon_{i,\lambda} E_{i,\lambda} d\lambda}{\int_0^\infty E_{i,\lambda} d\lambda} = \frac{1}{E_i} \int_0^\infty \epsilon_{i,\lambda} E_{i,\lambda} d\lambda \quad (6.5)$$

$$\alpha_{ij} = \frac{\int_0^\infty \epsilon_{i,\lambda} W_{j,\lambda} d\lambda}{\int_0^\infty W_{j,\lambda} d\lambda} \quad (6.6)$$

Equation 6.2 simplifies to

$$W_j = \epsilon_j E_j + \int_{A_c} \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} (1 - \alpha_{ij}) W_i dA_i \quad (6.7)$$

which reduces to the gray-surface radiosity equation whenever Eq. 6.6 reduces to  $\alpha_{ij} = \epsilon_i$ . The difficulty now lies in evaluating Eq. 6.6 even though the radiosity  $W_{j,\lambda}$  is as yet unknown.

By successively substituting the right-hand side of Eq. 6.2, in its monochromatic form, for  $W_{i,\lambda}$  on the right-hand side of that equation, one obtains first (Ref. 8)

$$W_{j,\lambda} = \epsilon_{j,\lambda} E_{j,\lambda} + (1 - \epsilon_{j,\lambda}) \left\{ \int_{A_c} \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} \left[ \epsilon_{i,\lambda} E_{i,\lambda} + (1 - \epsilon_{i,\lambda}) \int_{A_c} \frac{\partial^2 \overline{s_i s_m}}{\partial A_i \partial A_m} \epsilon_{m,\lambda} E_{m,\lambda} + \dots \right] dA_i \right\} \quad (6.8)$$

and subsequently  $\alpha_{ij}$  as the quotient of two infinite series. Since the enclosure radiation is dominated by the fin-sun and fin-sky interaction and since Eq. 6.8 contributes significantly only to the fin-flow channel interaction, the infinite series in Eq. 6.8 may be terminated after two terms (two reflections; the resulting error is less than the uncertainty in  $\epsilon_\lambda$ ), and Eq. 6.6 becomes:

$$\alpha_{ij} = \frac{X_{ij} E_j + \int_{A_c} \frac{\partial^2 \overline{s_j s_k}}{\partial A_j \partial A_k} E_k (X_{ik} - X_{ijk}) dA_k}{\epsilon_j E_j + \int_{A_c} \frac{\partial^2 \overline{s_j s_k}}{\partial A_j \partial A_k} E_k (\epsilon_k - X_{jk}) dA_k} \quad (6.9)$$

where

$$X_{ij} = \int_0^\infty E_{j,\lambda} \epsilon_{j,\lambda} \epsilon_{i,\lambda} d\lambda / E_j \quad (6.10)$$

$$X_{ijk} = \int_0^\infty E_{k,\lambda} \epsilon_{k,\lambda} \epsilon_{j,\lambda} \epsilon_{i,\lambda} d\lambda / E_k \quad (6.11)$$

In cases where the net radiant flux is specified over portions of  $A$ , the emissive power  $E$  is to be replaced by the net radiant flux  $q''$  in Eqs. 6.9, 10 and 12, which results in one additional term each in the numerator and the denominator of Eq. 6.9.

In summary, the incident net radiant heat flux for the diffuse, non-gray enclosure is calculated on the basis of an approximate gray-surface analysis in accordance with Eqs. 6.4, 7 and 9 through 11. The spectral differences of the surfaces are accounted for in Eqs. 9 through 11. The remainder of this section is devoted to the solution of the radiosity equation, Eq. 6.7.

Recalling that  $A_c$  is the sum  $A_m + A_n + A_f + A_e$  of the outer channel surface  $A_m$ , the possibly present, nearby structural surfaces  $A_n$ , the fin surface  $A_f$ , and the remainder of the enclosure  $A_e$ , one recognizes that the integrals over  $A_c$  in Eqs. 6.4, 7 and 9 need to be evaluated twice for each of the four parts, namely once with  $j = 1$  representing the fin area and then with  $j = 2$  for the exposed channel area. Since the incident solar, albedo and planetary radiant flux intensities are uniform over the fin area and averaged over the circumference of the channel area

$$\int_{A_e} \frac{\partial^2 \overline{s_i s_1}}{\partial A_i \partial A_1} (1 - \alpha_{i1}) W_i dA_i = (1 - \alpha_{e1}) q''_{e1} \quad (6.12)$$

$$\int_{A_e} \frac{\partial^2 \overline{s_1 s_2}}{\partial A_1 \partial A_2} (1 - \alpha_{12}) W_i dA_i = (1 - \alpha_{e2}) q''_{e2} \quad (6.13)$$

where  $q''$  designates incident solar, albedo and planetary heat fluxes, appropriately averaged over a chosen area element. Should any structural surfaces obstruct the incident radiant fluxes ( $A_n \neq 0$ ) then the right-hand sides of Eqs. 6.12 and 13 would have to be modified and reduced in the shaded portions of  $A_1$  and  $A_2$ ; and, if there are  $m$  such surfaces,

$$\int_{A_n} \frac{\partial^2 \overline{s_i s_j}}{\partial A_i \partial A_j} (1 - \alpha_{ij}) W_i dA_i = \sum_{k=1}^m \frac{\partial \overline{s_k s_j}}{\partial A_j} (1 - \alpha_{kj}) W_k, \quad (6.14)$$

$$j = 1, 2, \dots, m+2.$$

Obviously, the radiosity and the temperature are assumed to be uniform over each structural component. No such structural components were included in the analysis reported here, and Eq. 6.14 is taken to be zero.

This leaves only the integrals over  $A_m$  and  $A_f$  to be discussed. Moreover, since Eq. 6.1 is symmetric with respect to its subscripts  $i$  and  $j$ , the elemental exchange area is to be evaluated only once.

Considering first the fin, that is  $j = 1$  and  $i = 2$ , and the fact that over the channel surface the radiosity and the temperature are considered to be functions of axial distance only

$$\int_{A_2} \frac{\partial^2 \overline{s_1 s_i}}{\partial A_1 \partial A_i} (1 - \alpha_{1i}) W_i dA_i =$$

$$\int_{z=0}^{z=L} [1 - \alpha_{1i}(z)] W_i(z) \int_{\phi=0}^{\phi^*} \frac{\partial}{\partial A_1} \left( \frac{\overline{s_1 s_i}}{\Delta A_1} \right) R d\phi dz = \quad (6.15)$$

$$\int_{z=0}^{z=L} [1 - \alpha_{1i}(z)] W_i(z) SS(z; x_f, z_f) dz$$

where  $L$  designates the tube length;  $R$  and  $\phi$  are the polar coordinates of  $A_2$ , with origin on the tube axis, with  $\phi = 0$  and  $\phi = \phi^*$  representing, respectively, the root of the fin and the contact line between the tube and its tangent plane through the center of  $\Delta A_1$  on the fin. The first step in Eq. 6.15 was obtained by integrating over  $\Delta A_1$  and subsequently applying the mean-value theorem of integral calculus, while the second step simply defines the exchange function for every point  $(x_f, z_f)$  on the fin which was integrated in closed form for the right-circular flow channel. The result is shown in Appendix D.

The exchange function of the tube with respect to the fin is obtained by dividing  $SS$  in Eq. 6.15 by  $(R_\phi^*)$ . Thus

$$\int_{A_1} \frac{\partial^2 \overline{s_2 s_1}}{\partial A_2 \partial A_1} (1 - \alpha_{2i}) W_i dA_1 = \frac{1}{R_\phi^*} \int_{z=0}^L \int_{x=0}^H [1 - \alpha(z, x_f, z_f)] W(x_f, z_f) SS(z, x_f, z_f) dx_f dz_f \quad (6.16)$$

For the numerical evaluation of the integrals a suitable quadrature such as the trapezoidal rule is chosen so as to render Eq. 6.7 in this form

$$P_j = \sum_i M_{ji} W_i \quad (6.17)$$

$i, j = 1, 2, \dots, N$

which is a system of  $N$  linear algebraic equations for the  $N = (n_x + 1)(n_z + 2)$  unknown values of the radiosity  $W_i$ . Here,  $n_x$  and  $n_z$  are the numbers of subdivisions chosen in the  $x$ - and the  $z$ -directions, respectively. The vector  $P$  on the left-hand side of Eq. 6.17 is called the excitation vector

$$P_j = -\epsilon_j E_j - (1 - \alpha_{ej}) q''_{ej} \quad (6.18)$$

on the right-hand side, the transfer matrix  $M_{ji}$  is given by

$$\delta_{ji} - \chi_i [1 - \alpha_{ji}] SS_{ji} = M_{ji} \quad (6.19)$$

where

$$\delta_{ji} = \begin{matrix} 0 & i \neq j \\ 1 & \text{for } i=j \end{matrix}$$

is the Kronecker delta,  $\chi_i$  is a suitable quadrature coefficient and  $SS_{ji}$  is given either by Eq. 6.15 or by Eq. 6.16 depending on whether  $j$  refers to the fin or to the channel, respectively. There is no matrix multiplication implied in Eq. 6.19, hence the underscores.

In the present program phase, Eq. 6.17 is solved at every time step only when the transfer matrix is sufficiently temperature sensitive, otherwise the transfer matrix is completely inverted only once to yield the unknown

radiosity at any time.

$$W_i = \sum_j (M_{ij})^{-1} P_j \quad (6.20)$$

through a simple matrix multiplication. It may be noted that the most significant temperature dependence of optical properties is contained in the excitation vector  $P_j$ .

All radiant heat fluxes are normalized with respect to  $\sigma T_o^4$  where  $T_o$  designates the reference temperature, that is the fluid inlet temperature. Exchange factors are nondimensional and need not be normalized.

$$w_j = \frac{W_j}{\sigma T_o^4}, \quad \tilde{q}_j = \frac{q_j}{\sigma T_o^4}, \quad p_j = \frac{P_j}{\sigma T_o^4} \quad (6.21)$$

The nondimensional forms of Eqs. 6.4 and 6.20 are used to compute  $\tilde{q}_{\text{net,rad}}$  in Eqs. 2.17, 4.11 and 5.5.



## 7. Aerodynamic Heating

The aerodynamic heating model used to evaluate the convective flux from the radiator surface on the orbiter vehicle is subdivided into three major regimes. The first regime encompasses low speed flow for which the heat transfer coefficients are determined from expressions appearing in standard heat transfer texts for flow over a flat plate. The second regime consists of a model for high speed flow in which the convective heat transfer coefficient is evaluated from an experimental correlation for flow over the upper surface of the shuttle orbiter vehicle (Ref. 9). The third section of the model encompasses a low to high speed transitional flow regime. Within this regime the heat transfer coefficient is an interpolated value that lies between the values obtained in the low and high speed regimes. Calculations for the convective heat flux for all three regimes are based on Eckert's reference enthalpy method (Ref. 10).

Within each of the three regimes the heat transfer coefficient is calculated for cases where the flow is laminar, transitional or fully turbulent. In addition to the evaluation of the heat flux when the flow is forced, the procedure accounts for heat transfer by free convection at times when the shuttle vehicle is stationary or moving with a relatively low velocity.

The program for the evaluation of the aerodynamic heating rate is divided into six sub-tasks each of which is written as a separate subprogram. This procedure allows for changes in the periphery of the program without affecting the program foundation. The basic calculations are carried out in and controlled from the SUBROUTINE CONVEC. Atmospheric temperature and speed of sound are evaluated within the SUBROUTINE ATMOS. Atmospheric properties evaluated at the reference temperature are calculated within the SUBROUTINE REFP. The orbiter velocity and altitude are evaluated in the FUNCTION subprogram ALTVEL. The SUBROUTINE NUS evaluates the Nusselt number for the radiator system.

It should be noted that the analysis does not account for the effects of shock wave interaction or interference heating caused by flow interference between the orbiter, booster or any supporting structure.

The analysis for the determination of the aerodynamic heating first requires the evaluation of a reference temperature which is used for the determination of all air properties. The reference temperature is a function of the Prandtl number and recovery factor of the air, as well as the vehicle Mach number.

The Mach number in turn is a function of the altitude and velocity of the orbiter at any instant time. Altitude and velocity profiles for the orbiter are contained in data arrays supplied by the user (see Part A-3c of the U Manual. If the value of the Integer I is used to specify either ascent or reentry phase, then the N paired data points which define the velocity V and elevation Z as a function of time t may be expressed functionally as

$$V_i = V_i (I, t_i) \quad i = 1, 2, \dots, N \quad (7.1)$$

$$Z_i = Z_i (I, t_i) \quad i = 1, 2, \dots, N \quad (7.2)$$

Once the orbiter velocity and altitude are known as a function of time, the vehicle Mach number M may be calculated from the equation

$$M = \frac{V}{c} \quad (7.3)$$

The speed of sound c is an atmospheric property that is a function only of the altitude.

The reference temperature is a function of the recovery factor r which for laminar flow is

$$r = \sqrt{N_{Pr}}$$

and for turbulent flow (Ref. 11) is

$$r = N_{Pr} \frac{(8 + 0.528M^2/(22 + M^2))}{(7.4)}$$

where the Prandtl number  $N_{Pr}$  is an atmospheric property. To avoid a discontinuity in the value of the recovery factor between the laminar and turbulent flow models, Eq. 7.4 was used as the expression for the recovery factor for both flow models. The resulting error in the reference temperature was found to be approximately 5 R in an extreme case.

All of the properties for the atmosphere used in the evaluation of the heat transfer coefficient are evaluated at the high speed reference temperature. Eckert (Ref. 10) recommends the expression

$$i^* = 0.5 (i_\infty + i_w) + 0.11 r (\gamma - 1) M^2 i_\infty \quad (7.5)$$

for the reference enthalpy  $i^*$  which can be converted to the reference temperature  $T^*$  once the relationship

$$T^* = T^* (i^*) \quad (7.6)$$

between the atmospheric enthalpy and temperature is known. The subscripts " $\infty$ " and "w" in Eq. 7.5 refer to the enthalpy of the air evaluated at the free stream and surface temperature, respectively, and  $\gamma$  is the ratio of the specific heats for air.

It should be mentioned that when velocities are low ( $M \rightarrow 0$ ) Eqs. 7.5 and 6 yield a reference temperature that approaches the film temperature  $(T_\infty + T_w)/2$ . As a result Eqs. 7.5 and 6 were used to evaluate the reference temperature for all three flow regimes, i.e., low speed, high speed and the transitional regimes.

The convective flux is the product of the convective heat transfer coefficient and the difference between the air enthalpy evaluated at the surface temperature and at the adiabatic wall temperature. The adiabatic wall

enthalpy  $i_{aw}$  is related to the free stream enthalpy, recovery factor and vehicle velocity by the relationship

$$i_{aw} = i_{\infty} + \frac{rV^2}{2g_c} \quad (7.7)$$

The convective heat transfer coefficient  $h_i$  used in the reference enthalpy method may be expressed in terms of the Nusselt number  $N_{Nu}$

$$N_{Nu} = \frac{h_i x c_p^*}{k^*} \quad (7.8)$$

where  $c_p^*$  and  $k^*$  denote the atmospheric specific heat and thermal conductivity evaluated at the reference temperature  $T^*$ . The symbol  $x$  denotes a characteristic length of the radiator system which for forced convection is the distance from the stagnation point on the shuttle to the center of the radiator panel.

The expressions for the orbiter Nusselt number selected for the low speed and high speed regime and for laminar, transitional and turbulent flow are summarized in Table 1. Values for the Nusselt number for conditions lying between the low and high speed regimes were obtained by interpolation so that the convective heat flux from the shuttle varies continually from one regime to the other. The symbols  $N_{Re}$  and  $N_{Nu}$  appearing in the table denote the Reynolds number and Prandtl number, respectively, where

$$N_{Re} = \frac{\rho^* V x}{\mu^*}$$

$$N_{Pr} = \frac{\mu^* c_p^*}{k^*}$$

The "\*" superscript on each property indicates that the property is evaluated at the temperature  $T^*$ .

TABLE 1  
FORCED CONVECTION NUSSELT NUMBER FOR ORBITER

	LOW SPEED REGIME $M \leq 0.5$	HIGH SPEED REGIME $M \geq 1.0$
LAMINAR FLOW $Re < 1.0 \times 10^5$	$N_{Nu} = 0.332 N_{Re}^{1/2} N_{Pr}^{1/3}$	$N_{Nu} = 0.375 N_{Re}^{0.5014} N_{Pr}$
TRANSITIONAL FLOW $1 \times 10^5 \leq Re \leq 1 \times 10^6$	$N_{Nu} = 6.78 \times 10^{-5} N_{Re}^{1.238} N_{Pr}^{1/3}$	$N_{Nu} = 3.339 \times 10^{-4} N_{Re}^{1.111} N_{Pr}$
TURBULENT FLOW $Re > 1 \times 10^6$	$N_{Nu} = 0.0288 N_{Re}^{0.8} N_{Pr}^{1/3}$	$N_{Nu} = 0.0346 N_{Re}^{0.7746} N_{Pr}$

For the low speed regime the expressions for the Nusselt number are those for laminar transitional and turbulent flow over a flat plate (Ref. 6<sup>\*</sup>). Nusselt number relationships for high speed flow regime are taken from Ref. 9 where experimental wind tunnel data are presented for a delta space shuttle orbiter. The data are for leeward surface heat transfer at angles of attack between 10° and 30° and Mach numbers of 8 and 16. The Nusselt number is shown to be relatively independent of angle of attack so that the high speed correlation may be applied to both the ascent phase for which the angle of attack is approximately zero and the reentry phase when the angle of attack may approach 60°. The scatter in the data of Ref. 9 from the selected Nusselt relationships is on the order of 100%.

The leeward surface correlation was selected because the aerodynamic heating rates in this region of the shuttle are relatively low when compared to heating rates for the lower body or windward surface. Estimates place the peak reentry temperature of the lower surface stagnation line around 2100 F while the peak leeward temperature is estimated to be about 600 F (Ref. 12). Therefore, placing the radiator on the upper surface of the orbiter not only will result in a more efficient operation upon reentry, but also will minimize the need for reservicing of the surface coating on the radiator panels.

The aerodynamic heating analysis includes free convection from the radiator surface during pre-launch operation and when the shuttle vehicle is moving with a relatively low velocity. The expression for the free convection Nusselt number is a function of the Grashof Prandtl product where the Grashof number  $N_{Gr}$  is given by

$$N_{Gr} = g\beta \left(\frac{\rho}{\mu}\right)^2 y^3 (T_w - T_\infty)$$

where  $g$  represents the acceleration of gravity,  $\rho$ ,  $\mu$  and  $T_\infty$  are the atmospheric density, dynamic viscosity and temperature, respectively and  $T_w$  denotes the radiator surface temperature. The symbol  $y$  denotes the overall dimension of the radiator panel in the direction parallel to the acceleration of gravity. Since the atmosphere is assumed to be an ideal gas, the

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\* pp. 296 and 313

coefficient of thermal expansion  $\beta$  is simply the reciprocal of the average absolute temperature of the air

$$\beta = \frac{2}{T_w + T_\infty} \quad .$$

The free convection Nusselt numbers  $N_{Nu_f}$  and their applicable ranges of Grashof Prandtl product used for this analysis are

$$N_{Nu_f} = 1.585 (N_{Gr} N_{Pr})^{0.195} \quad (10^{-1} < N_{Gr} N_{Pr} < 10^4) \quad (7.9a)$$

$$N_{Nu_f} = 0.590 (N_{Gr} N_{Pr})^{0.250} \quad (10^4 \leq N_{Gr} N_{Pr} \leq 10^9) \quad (7.9b)$$

$$N_{Nu_f} = 0.130 (N_{Gr} N_{Pr})^{0.333} \quad (N_{Gr} N_{Pr} > 10^9) \quad (7.9c)$$

The values for the free convection Nusselt number given in Eq. 7.9 were multiplied by the ratio  $(x/y)$  and then added to the forced convection Nusselt number to obtain a value that accounts for combined free and forced convection in the low speed regime.

Equations 7.1 through 7.8 combined with the appropriate Nusselt number relationship from Table 1 for forced convection and Eq. 7.9 for free convection are sufficient to determine the convective heat flux into the radiator surface which is given by

$$q''_{aero} = h_i (i_\infty - i_{aw}) \quad (7.10)$$

The convective flux may be normalized by dividing by the heat flux  $\sigma T_o^4$  or

$$q_{\text{aero}} = \frac{h_i (i_{\infty} - i_{\text{aw}})}{\sigma T_o^4} \quad (7.11)$$

where  $T_o$  is the fluid temperature at the inlet plane to the flow channel. The normalized convective heat flux given by Eq. 7.11 is used in the energy equation for both the fin (Eq. 2.18) and the meteoroid protection layer (Eq. 5.5).



### C. DESIGN PARAMETERS

Certain design parameters are necessary for the design specification, for the selection of the optimum radiator system and even the system definition as required for the heat transfer analysis. In the following are discussed, in that order, the prediction of the meteoroid protection layer thickness, the system weight and a collection of nondimensional groups which define the radiator system, the operational conditions, and the performance characteristics.

## 8. Meteoroid Protection Thickness

In this section an engineering equation is developed to predict the thickness of a meteoroid protection layer required to cover all radiator surfaces that might be damaged by the impact of a meteoroid. Several assumptions have been made during the derivation. They are:

1. The meteoroid particle is spherical.
2. The meteoroid flux is isotropic.
3. Poissons distribution law describes the probability of an impact of a meteoroid.

It should be mentioned that any equation used to predict meteoroid protection thickness is only as accurate as the experimental data used in that equation. Even though much information has been published in recent years concerning protection theories, there is still considerable question as to the density, velocity and mass distribution of meteoroid particles in outer space. In addition to these uncertainties, two basic models for penetration theory have been proposed within the last decade and there appears to be no close agreement between the two. Experimental verification of either model has been hampered by the fact that particle velocities used in experimental tests have only recently approached the meteoroid velocity range. In short, an extremely reliable theory for the prediction of protection layer thickness does not presently exist.

Structural materials that can be used in this study as a protection layer are copper, aluminum and beryllium. While both copper and aluminum were selected primarily as fin and tube materials due to their superior heat transfer characteristics, beryllium was chosen for its protection capabilities. The penetration theory predicts a protection layer thickness that decreases as the modulus of elasticity increases. Therefore, beryllium becomes an attractive protection material due to its high modulus of elasticity and its relatively low density. In fact studies (Refs. 13 and 14) have shown that beryllium can significantly reduce protection layer weight.

The basic equation (Ref. 15) for the depth of penetration of a meteoroid particle into a target of infinite depth is

$$P_{\infty} = \gamma d \left( \frac{\rho_p}{\rho_t} \right)^{\phi} \left( \frac{\bar{v}}{\bar{c}} \right)^{\theta} \quad (8.1)$$

where

$\gamma$  empirical constant generally accepted to be in the range of 1.5 to 2.5

$\rho_p$  density of the meteoroid particle

$\rho_t$  density of the target material

$\bar{v}$  velocity of the meteoroid particle

$\bar{c}$  velocity of sound in the target material

$d$  diameter of the meteoroid particle

$\phi$  constant between 1/3 and 2/3

$\theta$  constant between 1/3 and 2/3

The ratio of the meteoroid velocity and the velocity of sound in the target represents a target Mach number. The velocity of sound in the target can be expressed in terms of the modulus of elasticity

$$\bar{c} = \sqrt{E_t g_c / \rho_t} \quad (8.2)$$

where

$E_t$  modulus of elasticity of the target material

$g_c$  proportionality constant relating mass units to force units

$\rho_t$  target density

If the meteoroid particle is assumed to be spherical, the diameter may be written in terms of its mass

$$d = \left( \frac{6M_p}{\pi \rho_p} \right)^{1/3} \quad (8.3)$$

where

$M_p$  meteoroid mass

$\rho_p$  meteoroid density

The probability that an exposed surface will be struck by a meteoroid during a period of time can only be determined after the distribution of meteoroids of a given mass is known. This information is usually given in the form of an equation such as

$$F = \alpha M_p^{-\beta} \quad (8.4)$$

where

$F$  cumulative number of impacts of particles with mass  $M$  or larger per unit area per unit time

$M_p$  mass of the meteoroid particle.

The symbols  $\alpha$  and  $\beta$  represent experimentally determined constants. Published (Ref. 16) values of  $\alpha$  and  $\beta$  vary over a considerable range, but they lie within the limiting values,

$$\begin{aligned} 1.3 \times 10^{-15} < \alpha < 2.54 \times 10^{-9} & \quad [\text{ft}^2 \text{ day gm}^{-\beta}]^{-1} \\ 1.11 < \beta < 1.34 & \quad [\text{Dimensionless}] \end{aligned}$$

The cumulative number of impacts on a surface with a vulnerable area of  $A$  during a mission time of  $\tau$  by a meteoroid of mass  $M_p$  or larger is then

$$N = FA\tau = A\tau \alpha M_p^{-\beta} \quad .$$

It is generally assumed (Ref. 17) that meteoroids are randomly distributed in outer space and that each collision can be described by Poisson's probability law. From the Poisson distribution function, the probability of zero events occurring  $P_0$  when the average number of events is  $N$  is

$$P_0 = e^{-N}$$

or

$$\ln P_0 = -N$$

Substituting the value for  $N$  gives the probability that no meteoroid of mass  $M$  or larger will impact on the surface of area  $A$  during time  $\tau$  of

$$\ln P_0 = A\tau\alpha M_p^{-\beta}$$

or

$$M_p = \frac{\alpha A\tau}{-\ln P_0}^{1/\beta} \quad (8.5)$$

To account for the fact that all meteoroids do not strike the protection layer normally, the meteoroid velocity  $\bar{V}$  may be replaced by a critical velocity  $\bar{V}_c$  where

$$\bar{V}_c = \bar{V}(\cos \lambda)^n \quad (8.6)$$

$\lambda$  angle between the direction of  $\bar{V}$  and the normal to the protection surface

$n$  an experimentally determined constant.

If  $n$  is selected to be unity the damage to the protection layer caused by an oblique collision is based on the meteoroid's normal component of velocity. A more conservative approach would be to set  $n = 0$  in which case all particles are considered to impact normally.

If the meteoroid flux is assumed to be isotropic the angle dependence may be replaced by

$$(\cos \lambda)^n = \left( \frac{2}{3n\theta\beta+2} \right)^{1/3\beta} \quad (8.7)$$

Finally account must be taken for the fact that the meteoroid will not impact on an infinite target, but one with a finite thickness. As a result even though the meteoroid may not penetrate the protection layer, spalling may damage the radiator panel. To account for spalling the thickness of meteoroid protection  $t$  used should be larger than the predicted penetration into an infinite target or

$$t = aP_\infty \quad (8.8)$$

Accepted values of  $a$  lie between 1.5 and 2.0.

Equations 8.1 through 8.7 may now be substituted into Eq. 8.8 to give an expression for the meteoroid protection layer thickness. The result is

$$t = a\gamma\left(\frac{6}{\pi}\right)^{1/3} \left(\frac{\alpha A_T}{-\ln P_Q}\right)^{1/3\beta} \rho_P^{-1/3} \left(\frac{\rho_P}{\rho_t}\right)^\phi \left(\frac{V}{12(E_{tc}/\rho_t)^{1/2}}\right)^\theta \left(\frac{2}{3n\theta\beta+2}\right)^{1/3\beta} \quad (8.9)$$

where

- $t$  - thickness of protection layer (inches)
- $a$  - experimental constant (dimensionless)
- $\gamma$  - experimental constant (dimensionless)
- $\alpha$  - experimental constant that relates meteoroid flux to mass (gm/(day ft<sup>2</sup>))
- $\beta$  - experimental constant that relates meteoroid flux to mass (dimensionless)

- $A$  - vulnerable area requiring protection ( $\text{ft}^2$ )  
 $\tau$  - mission time (days)  
 $P_o$  - probability of no damage caused by impact of meteoroid (dimensionless)  
 $\rho_p$  - density of meteoroid particle ( $\text{gm}/\text{cm}^3$ )  
 $\rho_t$  - density of protection layer ( $\text{lb}_m/\text{ft}^3$ )  
 $\bar{V}$  - velocity of meteoroid ( $\text{ft}/\text{sec}$ )  
 $E_t$  - modulus of elasticity of protection material ( $\text{lb}_f/\text{in}^2$ )  
 $g$  -  $32.174 \text{ lb}_m \text{ ft}/\text{lb}_f \text{ sec}^2$   
 $\theta$  - experimental constant (dimensionless)  
 $\phi$  - experimental constant (dimensionless)  
 $n$  - experimental constant that describes penetration depth as a function of angle of incident (dimensionless).

#### Selection of Values for Experimental Constants

Values for the experimental constant  $\rho$ ,  $\beta$ ,  $\rho_p$  and  $\bar{V}$  used in Eq. 8.9 were selected from the Manned Spacecraft Center publication for meteoroid environment criterion (Ref. 18).

The values for  $\alpha$  and  $\beta$  for meteoroids having a mass between  $1 \text{ gm}$   $10^{-6}$  gm used in Eq. 8.4 are

$$\alpha = 1.888 \times 10^{-10} \quad \text{gm}^\beta/(\text{ft}^2 \text{ day}) \quad (8.10)$$

$$\beta = 1.213 \quad (8.11)$$

The average meteoroid density is

$$\rho_p = 0.5 \text{ gm}/\text{cm}^3 \quad (8.12)$$

and the average meteoroid velocity is

$$\bar{V} = 20 \text{ km/sec.}$$

(8.13)

Values chosen for the remaining constants appearing in Eq. 8.9 are summarized in Table 2. Values of these constants are also listed in the table that will yield optimistic (minimum) and pessimistic (maximum) thicknesses for the meteoroid protection layer.

	Recommended Value	Pessimistic Value	Optimistic Value
a	1.75	2.0	1.5
$\gamma$	1.50	2.5	1.5
$\phi$	1/2	1/3	2/3
$\theta$	2/3	2/3	1/3
n	1.0	0	1.0

TABLE 2. Empirical Constants for Meteoroid Protection Layer Thickness

The following is an analysis of the sensitivity that the meteoroid protection layer thickness has to the uncertainty in the values of the five parameters listed in Table 2. This information will enable the user to judge his selection of these constants from within the recommended ranges.

An expression for the error in the meteoroid protection thickness may be obtained by taking the logarithm of Eq. 8.9 followed by differentiating the resulting equation. This process yields the equation

$$\frac{dt}{t} = \frac{da}{a} + \frac{d\gamma}{\gamma} + \phi \left( \frac{\rho_p}{\rho_t} \right) \frac{d\phi}{\phi} + \left[ \theta \ln \left( \frac{V}{c} \right) - \frac{n\theta}{3n\theta\beta+2} \right] \frac{d\theta}{\theta} - \left[ \frac{n\theta}{3n\theta\beta+2} \right] \frac{dn}{n} \quad (8.14)$$

If the symbol  $E_a$  is selected to represent the relative error in the meteoroid



protection thickness resulting from a relative uncertainty in the value for the parameter  $a$ , then it is evident from Eq. 8.14 that

$$E_a = \frac{dt/t}{da/a} = 1.0$$

when all other parameters are held constant. Similarly the error caused by an uncertainty in the value of  $\gamma$  will be

$$E_\gamma = \frac{dt/t}{d\gamma/\gamma} = 1.0$$

When the value for  $\theta$  is taken to be the recommended value of  $2/3$  and  $\beta$  is set equal to the value fixed by MSC's environmental model (Eq. 8.11), the resulting error in the meteoroid protection layer thickness due to an uncertainty in the value for  $n$  is

$$E_n = \frac{dt/t}{dn/n} = - \left( \frac{n\theta}{3n\theta\beta+2} \right) = - 0.15$$

The magnitude of the error for the final two parameters  $\phi$  and  $\theta$  are a function of the material selected for the protection layer. In order to give an indication of the range of errors that can be expected for various protection materials, the errors were calculated for the three structural materials that were selected in the program: aluminum, beryllium and copper. If the meteoroid particle density is assumed to be fixed at the value recommended by the MSC environmental model (Eq. 8.12), then an uncertainty in the value of  $\phi$  from the recommended value of  $1/2$  would cause an error in the protection thickness of

$$E_\phi = \frac{dt/t}{d\phi/\phi} = \phi \ln \left( \frac{\rho_p}{\rho_t} \right)$$

which for each of the three structural materials results in the following errors

$$\begin{aligned}
 E_{\phi} &= -0.85 && \text{aluminum} \\
 E_{\phi} &= -0.65 && \text{beryllium} \\
 E_{\phi} &= -1.44 && \text{copper.}
 \end{aligned}$$

The high density and low modulus of elasticity of copper makes its protection characteristics rather undesirable. For this reason the error of 1.44 indicated for copper probably will never be experienced in practice, and this value should be considered as a limiting case.

An uncertainty in the value of  $\theta$  from the recommended value of  $2/3$  would cause an error in the protection layer thickness equal to

$$E_{\theta} = \frac{dt/t}{d\theta/\theta} = \left[ \theta \ln \left( \frac{\bar{V}}{c} \right) - \frac{n\theta}{3n\theta\beta+2} \right]$$

which for each of the three structural materials is:

$$\begin{aligned}
 E_{\theta} &= 0.77 && \text{aluminum} \\
 E_{\theta} &= 0.20 && \text{beryllium} \\
 E_{\theta} &= 1.01 && \text{copper.}
 \end{aligned}$$

The errors calculated in the analysis are summarized in Table 3.

TABLE 3. Ratio of Relative Error in Thickness to Relative Uncertainty in Empirical Constants

Protection Material	Aluminum	Beryllium	Copper
Parameter			
a	1.0	1.0	1.0
$\gamma$	1.0	1.0	1.0
$\phi$	-0.85	-0.65	-1.44
$\theta$	0.77	0.20	1.01
n	-0.15	-0.15	-0.15

If the error values for copper are excluded, the protection thickness is most sensitive to variations in the parameters  $a$  and  $\gamma$  and least sensitive to variations in the parameter  $n$ . Even though the protection thickness is least sensitive to the selection of  $n$ , it should be noted that the 100% variation between the optimistic and pessimistic value of  $n$  is the largest of all of the parameters. Also it should be noted that the signs on the values in Table 3 indicate that an increase in the parameters  $a$ ,  $\gamma$  and  $\theta$  result in an increase in the protection layer thickness, while an increase in the values for  $\phi$  and  $n$  result in a decrease in the protection layer thickness. This fact can be verified by the choice of the values of each of the parameters listed in Table 2. The values labeled as those which will produce a pessimistic value for the protection thickness are maximum values for  $a$ ,  $\gamma$  and  $\theta$  and minimum values for  $\phi$  and  $n$ .

To further evaluate the effect the meteoroid protection thickness has on the performance of the fin system, the temperature of the coolant fluid at the exit plane of the flow channel was evaluated first under the "pessimistic" conditions for the meteoroid protection layer, second for the "recommended" conditions and finally for the "optimistic" conditions. The results of these computer runs are shown below.

Case	Protection Layer Thickness-Inches	Normalized Outlet Fluid Temperature $T/T_o$
Pessimistic	0.377	0.8855
Recommended	0.063	0.8922
Optimistic	0.020	0.8932

Even though the thickness of the meteoroid protection layer varies by nearly a factor of 20, the resulting error in the enthalpy drop is only

$$\frac{0.8932 - 0.8855}{1 - 0.8922} \cdot 100\% = 7.15\%$$

### 9. The Mass of the System

The system mass is computed, firstly as a convenience for the user and secondly for the purpose of the planned system optimization. The system mass includes

- (i) the mass of the fluid in all tubes

$$M_c = n_t \frac{d^2 \pi}{4} \int_0^L \rho_c dz = n_t \rho_{c,o} \frac{d^2 \pi}{4} L \int_0^1 v d\zeta \quad (9.1)$$

- (ii) the mass of the fins

$$M_f = n_t H L \rho_f (s_r + s_t) \quad (9.2)$$

- (iii) the mass of all tubes

$$M_w = n_t s_w L \cdot \pi (d + s_w) \rho_w \quad (9.3)$$

and

- (iv) the mass of the protection layer

$$M_m = n_t s_m L \rho_m [\pi(d + s_w + s_m) - s_r] \quad (9.4)$$

but it does not include the thermal coating nor the mass of the manifold and the fluid in the manifold. In Eqs. 9.1 through 4 represent

- $n_t$  the number of tubes
- $d$  the tube diameter
- $\rho$  the density
- $L$  the tube length
- $H$  the fin height, distance between fin root and fin tip
- $s$  the thickness

while the subscripts designate  $\rho$  and  $s$  as follows

- $c$  coolant fluid
- $f$  fin

m meteoroid protection layer  
r fin root  
t fin tip  
w tube wall  
o inlet condition.

The integral in Eq. 9.1 is time-dependent and evaluated at the initial conditions.

## 10. Nondimensional System Parameters

The governing equations in the preceding radiator system analysis are developed in non-dimensional form for the purpose of (i) reducing the number of parameters, (ii) evolving the set of relevant parameters, and (iii) presenting the results in a general form which is applicable to groups of systems rather than an individual system. A summary of parameters is presented here for the detailed analysis discussed in the preceding chapters.

The transient flow field in the coolant channel and the temperature field over the fin can be represented as functions of:

### a) the independent variables

$$\text{time} \quad \tau = \frac{t w_o}{L} \quad (10.1)$$

$$\text{axial coordinate} \quad \zeta = \frac{z}{L} \quad (10.2)$$

$$\text{radial coordinate} \quad \eta = \frac{2r}{d} \quad (10.3)$$

$$\begin{array}{l} \text{transverse coordinate} \\ \text{normal to the channel} \\ \text{axis} \end{array} \quad \xi = \frac{x}{H}$$

### b) the dependent system variables

$$\text{fin temperature} \quad \theta_f (\xi, \zeta; \tau) = \frac{T_f}{T_o} \quad (10.4)$$

$$\text{channel temperature} \quad \theta_w (\eta, \zeta; \tau) = \frac{T_w}{T_o} \quad (10.5)$$

$$\begin{array}{l} \text{meteroid protection} \\ \text{layer temperature} \end{array} \quad \theta_m (\eta, \zeta; \tau) = \frac{T_m}{T_o} \quad (10.6)$$

$$\text{coolant fluid temperature} \quad \theta_c (\zeta; \tau) = \frac{T_c}{T_o} \quad (10.7)$$

$$\text{coolant fluid pressure} \quad \pi(\zeta, \tau) = \frac{p}{p_0} \quad (10.8)$$

$$\text{coolant fluid velocity} \quad \omega(\zeta; \tau) = \frac{w}{w_0} \quad (10.9)$$

The solution to the problem will depend on the geometry of the system, the material properties and the definition of operational conditions. There was no attempt made to establish similitude with respect to the material properties because the scaling laws would either be too restrictive to allow for general property variations or too complex (for instance, the concept of corresponding states for gases). Consequently, the  $\phi$ -parameters defined in Eqs. 3.27 through 3.30, in Eq. 4.4 and in Eqs. 5.1, 2 and 6 are omitted from this summary; they constitute temperature and pressure variation of properties. This leaves the following list of parameters, in addition to  $n$ , the number of tubes:

c) the geometric parameters

$$\text{fin height-to-length ratio} \quad \bar{H} = \frac{H}{L} \quad (10.10)$$

$$\text{fin profile slope} \quad c = \frac{s_r - s_t}{2H} \quad (10.11)$$

$$\text{fin root thickness} \quad \bar{s}_r = \frac{s_r}{H} \quad (10.12)$$

$$\text{tube diameter-to-length ratio} \quad \delta = \frac{d}{L} \quad (10.13)$$

$$\text{channel wall thickness} \quad \bar{s}_w = \frac{2s_w}{d} \quad (10.14)$$

$$\text{protection layer thickness} \quad \bar{s}_m = \frac{2s_m}{d} \quad (10.15)$$

d) the operational parameters

coolant flow Reynolds number  $N_{Re} = \frac{dw_o}{v_o}$  (10.1)

Prandtl number  
(representing coolant selection)  $N_{Pr} = \left(\frac{\mu c_p}{k_c}\right)_o$  (10.1)

inlet pressure heat  $F_p = \left(\frac{p}{\rho w}\right)_o$  (10.1)

compressibility  $\bar{Q}_o = \frac{\pi n \rho d^2 c_p}{4 \sigma T_o^3}$  (10.1)

inlet coolant power flux  $F_z = \left(\frac{p}{\rho c_p T}\right)_o$  (10.20)

where n is the number of tubes

incident radiant heat flux  $\bar{Q}_{rad} = \frac{\alpha_o q''}{\epsilon_o \sigma T_o^4}$  (10.2)

meteoroid velocity  $M_m = \frac{v_m}{c_m}$  (10.2)

protection layer density  
(representing selection of  
protection layer material)  $\phi_\rho = \frac{\rho_{mt}}{\rho_m}$  (10.21)

where  $\rho_{mt}$  is the density of the meteoroids.



Similarity for ascent and reentry operations is difficult to establish unless one restricts oneself to similar velocity-altitude profiles which can be represented by the

$$\text{max. Mach number} \qquad M_{\text{max}} = (v/c)_{\text{max}} \qquad (10.24)$$

and its corresponding (through same altitude)

$$\text{Reynolds number} \qquad (N_{\text{Re}})_{\infty} = \frac{\rho_{\infty} v L}{\mu_{\infty}} \qquad (10.25)$$

$$\text{Prandtl number} \qquad (N_{\text{Pr}})_{\infty} = \frac{\mu_{\infty} c_{p\infty}}{k_{\infty}} \qquad (10.26)$$

$$\begin{array}{l} \text{Grashof number} \\ \text{(before launch)} \end{array} \qquad N_{\text{Gr}} = \frac{g L^3 \beta \Delta T}{\nu^2}$$

This completes the list of non-dimensional groups resulting from the detailed analysis.

#### D. Extension of the Analysis to Related Systems

##### 11. Non-Symmetrical Heating

The purpose of this section is to describe a segment of the program which is intended to extend the application of the radiator simulation to non-symmetrical flow conditions in adjacent tubes. This program will also account for, in an approximate manner, a radiator panel which does not lie in one plane, or a panel which consists of U-shaped tubes.

In the original simulation of the radiator system, it was assumed that the coolant fluid entering all tubes has identical properties. As a result, the operating conditions and heat loss is determined for a single fin segment between two adjacent tubes and these conditions are assumed to exist for all other fin segments. In reality, this situation will exist only when all tubes are fed by a relatively large manifold whose flow rate and temperature entering each tube is unaffected by the removal of coolant fluid at each succeeding channel entrance.

The actual flow situation could possibly be far from the idealized case which was assumed to exist because it leads to a simplified mathematical model. A manifold of realistic size will lose heat by radiation from its surface and by conduction into the fin elements so that the coolant fluid entering individual flow channels will experience a difference in temperature. Furthermore, unless the manifold is carefully reduced in size after passing each tube inlet, neighboring tubes will not receive identical mass flow rates. Also, two adjacent tubes may be fed by the outlet flow from separate fuel cells which may be operating under different load conditions causing non-symmetrical conditions in adjacent fin panels. In short, a situation where two tubes receive fluid at different inlet conditions is to be expected.

For the purpose of extending the application of the main program to situations discussed above, a series of program units were written and integrated into the main program unit. These programs calculate the location of the adiabatic plane on a fin which separates two tubes having different inlet flow rates and temperatures. The fin height to the adiabatic plane is then used as input to the main program. The calculations with the main program remain unaffected since the mathematical model requires only that the input value given for the fin height is that distance from the tube to the location of the adiabatic plane.

The analysis considered here may also account for, in an approximate fashion, fin elements that do not lie in one plane. If the radiator panel is wrapped around a cylindrical structural component (Fig. 3a) adjacent fin panels will be under the influence of different effective sink temperatures. In this analysis the assumption is made that the regular breaks in the radiator panel occur at the adiabatic plane between the tubes rather than at the tubes themselves (Fig. 3b) which is probably the most reasonable location. If the cylindrical panel is composed of numerous tubes and the curvature is gradual, the difference between the two cases should be negligible.

The program accepts two sink temperatures supplied by the user or it calculates the correct sink temperature from the MRI incident flux data. The analysis of this section assumes steady state, one-dimensional conduction in an untapered fin. The fin may radiate from one or both sides when the fin panel is in one plane. When the panel is curved the fin is assumed to radiate only from the convexed surface. Fluid and fin properties are assumed constant at the inlet conditions. No convection from the fin surface and no radiant blocking of the tubes is considered. The only incident fluxes considered in this analysis are those accounted for in the MRI program; i.e. solar, earth and earth albedo.

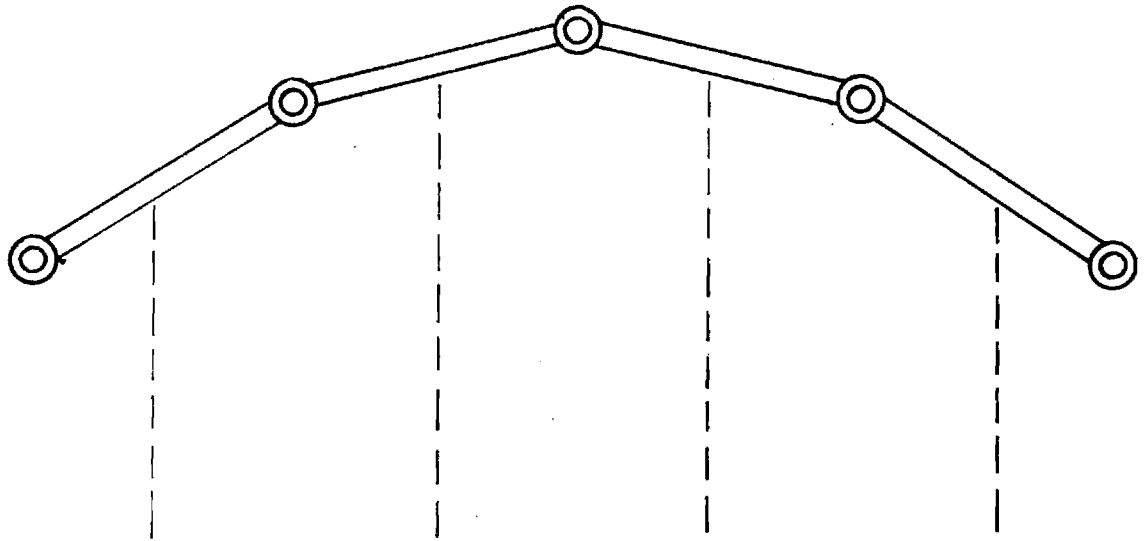
The user is cautioned that the analysis considered here assumes a one dimensional model and as such represents an approximation to the actual operating conditions of the non-symmetrical radiator. This caution is particularly applicable to the panel that contains U-shaped tubes, because this situation can lead to a highly two dimensional condition.

The analysis considers the  $i$ th tube of a radiator panel of thickness  $t$ , inside tube diameter  $d$ , tube length  $L_i$  and distance between tubes of  $2H$  (Fig. 4). The energy transferred from the  $i$ th tube carrying fluid with an inlet temperature of  $T_{oi}$ , specific heat of  $c_p$  and mass flow rate of  $\dot{m}_i$  is

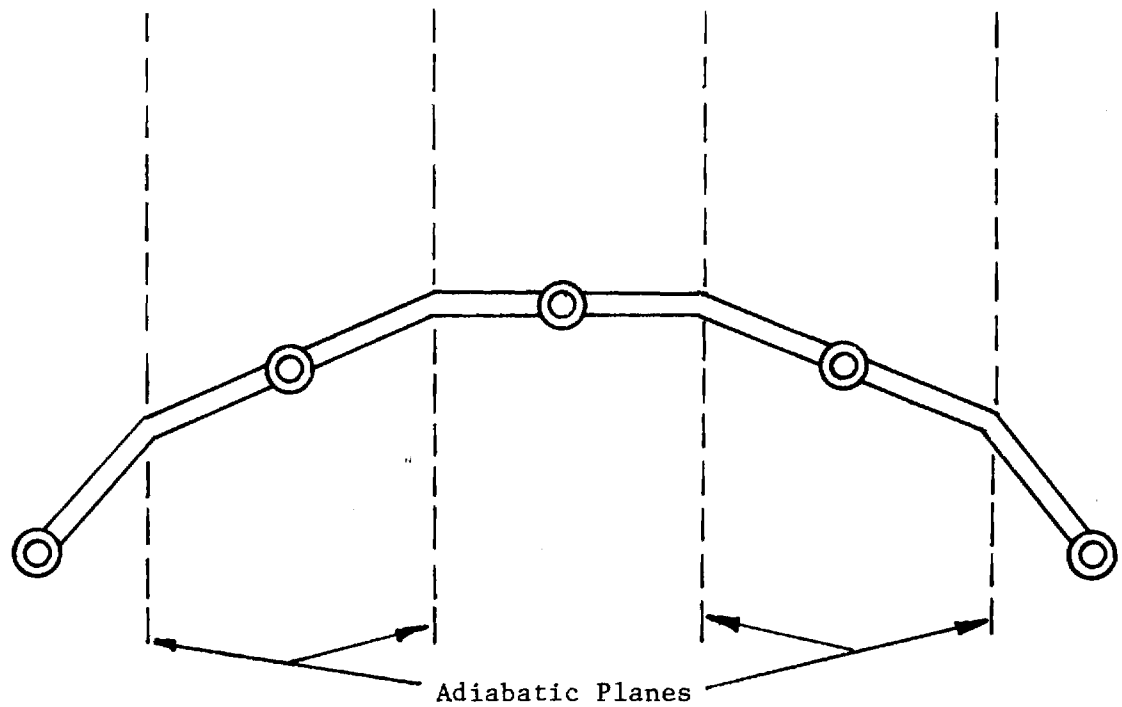
$$q_i = \dot{m}_i c_p (T_{oi} - T_{bi}) (1 - e^{-U_i}) \quad (11.1)$$

The quantity  $U_i$  is given by

$$U_i = \frac{\pi d h_i L_i}{\dot{m}_i c_p} \quad (11.2)$$



(a) Fin Segments Inclined At Tube Locations-  
Geometry For Physical Model.



(b) Fin Segments Inclined At Location Of Adiabatic  
Planes-Geometry For Mathematical Model

Fig. 3

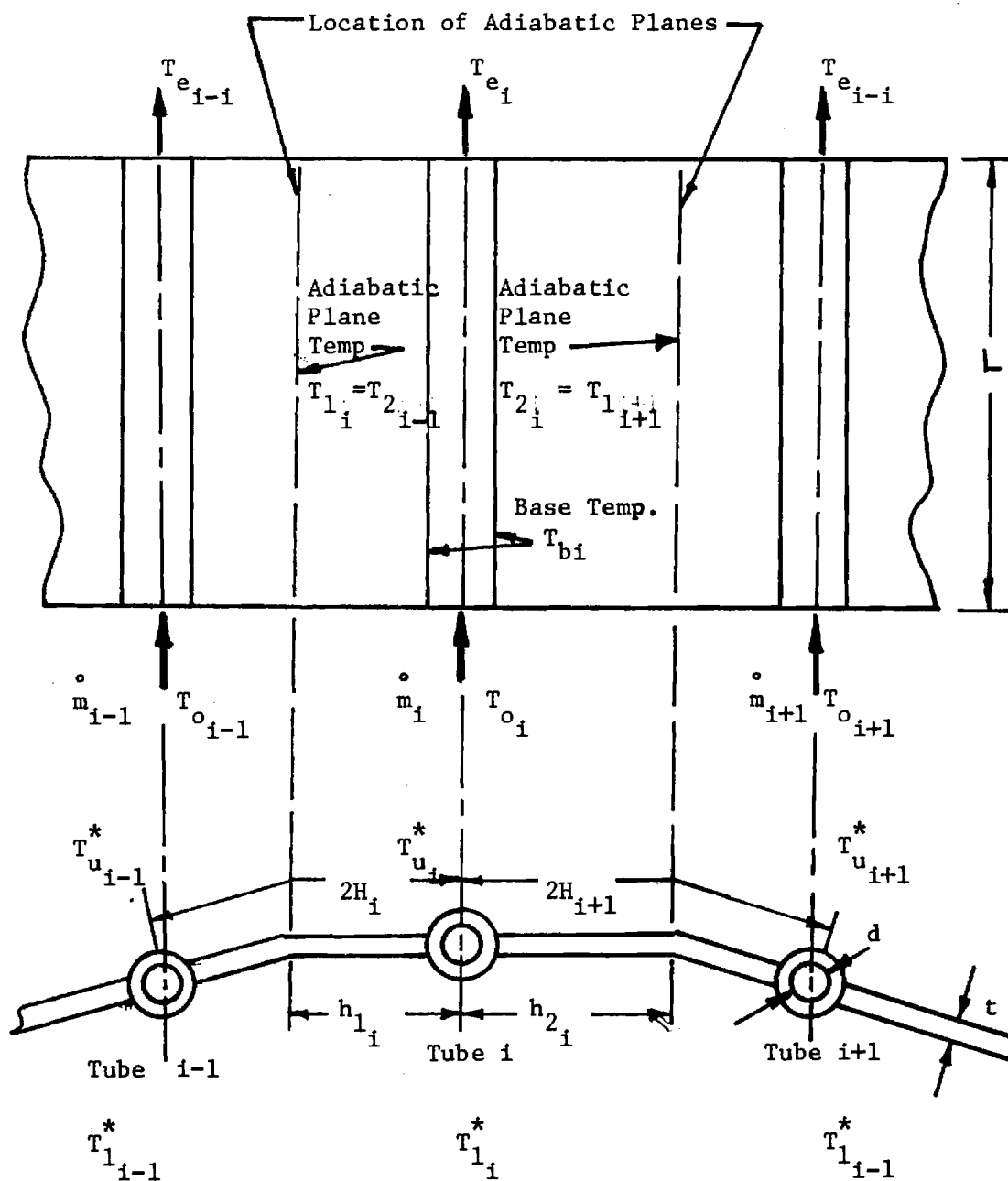


Figure 4. Fin Geometry for Non-Symmetrical Panel

where  $h_i$  is the convective heat transfer coefficient of the fluid in the tube and  $T_{bi}$  is the average fluid bulk temperature (also assumed to be the fin root temperature.)

The convective heat transfer coefficient  $h_i$  in Eq. 11.1 is evaluated from the Nusselt number which is related to the Reynold and Prandtl numbers by the expressions

$$N_{Nu} = 5 + 0.025 (N_{Re} N_{Pr})^{0.8} \quad \text{for} \quad \begin{cases} N_{Pr} < 0.1 \\ N_{Re} > 2300 \end{cases} \quad (11.3 a)$$

$$N_{Nu} = 0.023 (N_{Re}^{0.8} N_{Pr}^{0.3}) \quad \text{for} \quad \begin{cases} N_{Pr} > 0.1 \\ N_{Re} > 2300 \end{cases} \quad (11.3 b)$$

$$N_{Nu} = 3.65 + \frac{0.0668 [N_{Re} N_{Pr} (d/L)]}{1 + 0.045 [N_{Re} N_{Pr} (d/L)]^{2/3}} \quad N_{Re} \leq 2300 \quad (11.3 c)$$

The heat rejected from the coolant fluid given in Eq. 11.1 under the assumption of steady state and one-dimensional heat transfer can be determined by calculating the net radiant loss from the fin surfaces. If one side of the  $i$ th fin surface is radiating to an environment with an effective sink temperature of  $T_i^*$  then

$$q_i = \eta_{1i} \epsilon h_{1i} L_i \sigma (T_{bi}^4 - T_i^{*4}) + \eta_{2i} \epsilon h_{2i} L_i \sigma (T_{bi}^4 - T_i^{*4}) \quad (11.4)$$

where  $\eta_{1i}$  and  $\eta_{2i}$  are the fin effectivenesses for the fin attached to the left and right of the  $i$ th tube, respectively, and  $h_{1i}$  and  $h_{2i}$  are the distances between the tube centerline and the adiabatic plane for the fin attached to the left and right of the  $i$ th tube.

When the fin radiates from both sides, the effective sink temperature will be different for both sides and the net radiative flux will be

$$q_i = 2\eta_{1i} \epsilon h_{1i} L_i \sigma (T_{bi}^4 - T_i^{*4}) + 2\eta_{2i} \epsilon h_{2i} L_i \sigma (T_{bi}^4 - T_i^{*4}) \quad (11.5)$$

where the sink temperature  $T_i^{*4}$  becomes

$$T_i^* = \left( \frac{T_{ui}^{*4} + T_{li}^{*4}}{2} \right)^{1/4} \quad (11.6)$$

The subscripts u and l in Eq. 11.6 denote the sink temperatures for the upper and lower surfaces of the fin respectively.

The fin effectiveness in Eqs. 11.4 and 11.5 is a function of the conductance parameter  $N_c$  and the ratio of the sink temperature to base temperature  $T^*/T_b$  or for the ith tube

$$\eta_{1i} = \eta(N_{cli}, T_i^*/T_{bi}) \quad (11.7)$$

$$\eta_{2i} = \eta(N_{c2i}, T_i^*/T_{bi}) \quad (11.8)$$

The conductance parameter is defined as

$$N_{cli} = \frac{\epsilon \sigma T_{bi}^3 h_{1i}^2}{Kt} \quad \text{and} \quad (11.9)$$

$$N_{c2i} = \frac{\epsilon \sigma T_{bi}^3 h_{2i}^2}{Kt} \quad (11.10)$$

where K is the thermal conductivity of the fin material.

In addition to having a continuous slope in the fin temperature profile at the adiabatic plane, the fin temperature must also be continuous at this plane. For a one dimensional fin the temperature at the adiabatic plane T is a function of the conductance parameter and the ratio of sink temperature to base or

$$T = T(N_c, T^*/T_b) \quad (11.11)$$

As the program searches for the position of the adiabatic plane, the distances between the tube centerline and adiabatic plane is systematically varied until the fin tip temperatures given by Eq. 11.11 for two adjacent tubes are equal. As an example, consider the  $i$ th and  $i+1$  tube. Using the subscript 1 to denote a fin segment attached to the left of a tube and the subscript 2 to denote the fin segment attached to the right of a tube, the tip temperature profile will be continuous when

$$T_{2i} - T_{1i+1} \longrightarrow 0$$

If the difference in these two tip temperatures is denoted by  $\delta_i$ , then  $\delta_i$  becomes a function of  $T_{2i}$  and  $T_{1i+1}$ . By combining Eq. 11.11 and the definition of  $N_c$  these two tip temperatures may be specified by the functional relationship

$$T_{2i} = f(T_{bi}, h_{2i}) \quad \text{and} \quad (11.12)$$

$$T_{1i+1} = f(T_{bi+1}, h_{1i+1}) \quad (11.13)$$

The fin base temperatures of the  $i$ th and  $i+1$  tube can be determined by Eqs. 11.7 through 11.9 or written functionally

$$T_{bi} = f(h_{1i}, h_{2i}) \quad \text{and} \quad (11.14)$$

$$T_{bi+1} = f(h_{1i+1}, h_{2i+1}) \quad (11.15)$$

If Eqs. 11.12 through 11.15 are combined, the difference in the two temperatures  $\delta_i$  may be expressed as

$$\delta_i = \delta_i(h_{1i}, h_{2i}, h_{1i+1}, h_{2i+1}) \quad (11.16)$$

But from the geometry of the radiator system



$$h_{1i} = 2H_i - h_{2i-1} \quad \text{and} \quad (11.17)$$

$$h_{1i+1} = 2H_{i+1} - h_{2i} \quad (11.18)$$

where  $2H_i$  and  $2H_{i+1}$  is the distance between the centerlines of tubes  $i-1$  and  $i$  and tubes  $i$  and  $i+1$ , respectively.

By combining Eqs. 11.17 and 11.18 with Eq. 11.16, the final functional relationship for the difference in the fin temperatures at the adiabatic plane is given by

$$\delta_i = \delta_i(h_{2i-1}, h_{2i}, h_{2i+1}) \quad (11.19)$$

In other words  $\delta_i$  can be expressed entirely in terms of the distance from tube centerline to the adiabatic plane for the tube under consideration plus the same distance for tubes on either side.

The approach to the solution for the location for all adiabatic planes is one which gives  $\delta_i = 0$  for all  $i$  tubes. The program utilizes a Newton iteration to determine an appropriate change in  $i$ th adiabatic plane  $\Delta h_i$ . Written in matrix notation, the relationship between  $\Delta h_i$  and the difference in tip temperatures  $\delta_i$  is

$$[\delta_i] = [P_{ij}][\Delta h_i] \quad (11.20)$$

where the elements of the matrix  $p_{ij}$  are given by

$$[P_{ij}] = \frac{\partial \delta_i}{\partial h_j} \quad (11.21)$$

and the elements of the vector  $\Delta h_i$  are given by

$$\Delta h_i = h_i^{k+1} - h_i^k \quad (11.22)$$

where  $k$  denotes the  $k$  th iteration used to find a set of adiabatic plane locations for all fin elements which result in  $\delta_i = 0$  for all  $i$  tubes. A mathematical subroutine is used to invert the matrix  $[p_{ij}]$  so that the values for all adiabatic planes may be calculated from

$$[\Delta h_i] = [p_{ij}]^{-1} [\delta_i] \quad (11.23)$$

Once the distance to all adiabatic planes have been determined for all tubes which lead to continuous tip temperatures and continuous, zero slopes at the fin tips, the net heat transfer  $q_i$  from each fin segment may be determined from Eqs. 11.5 through 11.10. The exit fluid temperature from each tube  $T_{e_i}$  may then be determined from

$$q_i = \dot{m} \text{ cp}(T_{e_i} - T_{o_i}) \quad (11.24)$$

Since the main program unit requires only a single distance between the tube centerline and the adiabatic plane, the two distances  $h_{1i}$  and  $h_{2i}$  must be averaged to yield a single value. The averaging process produces a value  $\bar{h}_i$  which results in the same heat transfer for the symmetrical fin. This single value for each tube is calculated by using equation (11.5) resulting in

$$\bar{h}_i = \frac{\eta_{1i} h_{1i} + \eta_{2i} h_{2i}}{\eta_{1i} + \eta_{2i}} \quad (11.25)$$

As previously defined, the subscripts 1 and 2 denote the two halves of the fin attached to both sides of the  $i$ th tube. Values for the fin effectiveness are evaluated from Eq. 11.7.

The program considers the case of the radiator panel with U-shaped tubes to be approximated by three sequential parallel tube sections.

The program first calculates the fin base temperature and the distance to the adiabatic plane for the first section of the three tube segments. The heat transferred from the fluid is then calculated from

Eq. 11.1. The outlet fluid temperature from the first tube  $T_e$  segment is then calculated from the expression

$$q = \dot{m} c_p (T_e - T_o)$$

The exit fluid temperature simply becomes the inlet fluid temperature to the second tube segment (the base of the U). The same process is repeated for the remaining two tube segments resulting in two more values for the distance to the adiabatic plane. The single value for the distance to the adiabatic plane necessary for entry to the main program unit is determined from an average weighted with respect to the length of each tube segment.

### E. Property Fundamentals

The fundamental principles used to prepare the required thermophysical properties for inclusion into the computer code are exhibited in this chapter while the specific details concerning the materials treated in this program phase are placed into the appendices.

The principles involved are those of macroscopic thermodynamics treated in most elementary texts. The approach of deriving analytic expressions for the required properties is not unique because the starting point is dictated by the availability of experimental data. The result, however, must be of the same form regardless of whether, for instance, the coolant fluid is gaseous or liquid.

The properties of the structural materials are the least problematic ones since they depend on the temperature at most; and the standard polynomial collocation methods are entirely sufficient. Care must be exercised, however, that the collocation imply continuous fourth derivatives for highest integration efficiency or, less desirable, at least continuous representation of the property itself which may exclude piecewise allocation of degrees higher than one.

What is said about the properties of structural materials holds in principle for the description of the atmosphere whose properties depend only on altitude. Even though the optical properties of the thermal coating depend, in general, on wave-length and temperature, the spectral dependence is integrated into the averaged ("gray") properties (see Eqs. 6.5 and 6), and the results are functions of one variable, the temperature. Consequently, there remains but the discussion of the coolant fluid properties which depend strictly on two state variables.

In macroscopic thermodynamics there are required two equations of state for the description of a substance, namely the thermal equation of state  $f(\rho, p, T) = 0$  which relates any one of density  $\rho$ , pressure  $p$  and temperature  $T$  to the remaining two, and the caloric equation of state, perhaps in the form of  $c_v^0 = f(T)$  where  $c_v^0$  is the zero-pressure specific heat at constant volume. These two equations are sufficient to develop all

of the required thermodynamic functions, namely:

- (i) specific heat at constant pressure  $c_p(\rho, T)$
- (ii) isobaric thermal expansion coefficient  $\beta(\rho, T)$
- (iii) isothermal bulk modulus  $\kappa(\rho, T)$
- (iv) enthalpy  $h(\rho, T)$

These functions are discussed in Section 12.

The transport properties, namely the thermal conductivity  $k(\rho, T)$  and the dynamic viscosity  $\mu(\rho, T)$  are correlated on the basis of residuals as explained in Section 13. The properties of the atmosphere are dealt with in Section 14.

## 12. Thermodynamic Properties

The first step in developing thermodynamic properties is to secure a thermal equation of state

$$f(p, \rho, T) = 0 \quad . \quad (12.1)$$

For almost all pure gases and air, this equation can be found in the literature, either in the form suggested by Benedict-Webb-Rubin (virial expansion) or in that suggested by Beattie-Bridgeman. Both equations are explicit in  $p$ ,

$$p = p(\rho, T) \quad (12.2)$$

so that  $\kappa$  is immediately obtained from Eq. 11.2 by implicit differentiation

$$\kappa(\rho, T) = \frac{1}{\rho} \left( \frac{\partial \rho}{\partial p} \right)_T \quad (12.3)$$

which can be evaluated as  $\kappa(p, T)$  after inversion of Eq. 11.2 into

$$\rho = \rho(p, T) \quad . \quad (12.4)$$

The inversion of Eq. 12.2 is facilitated by computing  $(\partial p / \partial \rho)_T$  from Eq. 11.2 and subsequently applying the Newton-Raphson method along the specified isotherm with temperature  $T$  in Eq. 12.4

The isobaric thermal expansion coefficient

$$\beta = - \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p$$

is also obtained by implicit differentiation of Eq. 12.2 while keeping the left-hand side constant.

After having collocated the zero-pressure specific heat at constant volume; that is,  $c_v^0(T)$  by a power polynomial in  $T$ , one obtains first the specific heat at constant volume

$$c_v(\rho, T) = c_v^0(T) - T \int_0^p \left( \frac{\partial^2 p}{\partial T^2} \right)_\rho \frac{d\rho'}{(\rho')^2} \quad (12.5)$$

and then the specific heat at constant pressure

$$c_p(\rho, T) = c_v(\rho, T) + \frac{T\beta^2}{\rho\kappa} \quad (12.6)$$

The derivative in Eq. 12.5 is obtained from Eq. 12.2; and  $\beta$  and  $\kappa$  are both function of  $\rho$  or  $p$  and  $T$ .

Finally the enthalpy  $h$  is calculated from its definition

$$h(\rho, T) = \frac{p}{\rho} + u(\rho, T) \quad (12.7)$$

where the internal energy  $u$  may be obtained by two successive integrations, the first one along an isotherm (over  $\rho$ ), and the second one along an isochore (over  $T$ ):

$$u = u(\rho_0, T_0) + \int_{T_0}^T \int_{\rho_0}^p c_v(\rho', T') d\rho' dT' + \int_{\rho_0}^p \left[ p - \frac{T\beta}{\kappa} \right] \frac{d\rho'}{(\rho')^2} \quad (12.8)$$

Liquids can be treated, in principle, as gases; except that the equation of state, Eq. 12.1, is rarely available. One may find, with little difficulty the zero-pressure isobaric expansion coefficient  $\beta_0 = \beta(T)$ , and then represent adequately the isothermal bulk modulus by

$$\kappa(p, T) = a(T) + b(T) p \quad (12.9)$$

Under any circumstances, one must satisfy

$$\left( \frac{\partial \beta}{\partial p} \right)_T + \left( \frac{\partial \kappa}{\partial T} \right)_p = 0 \quad (12.10)$$

which yields, from Eq. 12.9

$$\beta(p, T) = -a'p - \frac{b'}{2} p^2 + c \quad (12.11)$$

where primes indicate differentiation with respect to  $T$ , of the polynomials  $a(T)$  and  $b(T)$  in Eq. 12.9. Equations 12.9 and 11 yield for the density

$$\rho(p, T) = \rho(0, T_0) e^{[a(T)p + \frac{1}{2} b(T)p^2 - \int_{T_0}^T c(T') dT']} \quad (12.12)$$

The specific heats and the enthalpy are to be derived as for gases (see Eqs. 12.6 and 7). Other possibilities are to develop  $\kappa$  from the speed of sound and the ratio of specific heats; but the reader is warned not to imply  $\kappa = 0$  or  $c_p = c_v$ , unless there is sufficient evidence to support these assumptions.



### 13. Transport Properties

While the thermal conductivity  $k$  and the dynamic viscosity  $\mu$  of liquids may often times be adequately represented by functions of temperature  $T$  alone (facilitated by polynomial collocation), these same properties for gases depend on density as well. It is recognized that the difference, or residue

$$\psi_1(\rho) = k(\rho, T) - k^*(T) \quad (13.1)$$

between the thermal conductivity  $k(\rho, T)$  and the low-pressure thermal conductivity  $k^*(T)$  depends only on the density. Similarly, for the dynamic viscosity

$$\psi_2(\rho) = \mu(\rho, T) - \mu^*(T) . \quad (13.2)$$

Hence  $k$  and  $\mu$  can be represented by the sum of two polynomials, one in  $\rho$  and the other in  $T$ . The residuals  $\psi_1$  and  $\psi_2$  are published for a number of gases or may be developed from property data (Ref. 19 for  $N_2$  and He).

#### 14. Atmospheric Properties

For the prediction of aerodynamic heat fluxes incident on the radiator system during ascent and reentry (see Section 7) the evaluation of the following atmospheric properties are required:

- Temperature
- Pressure
- Density
- Molecular Weight
- Speed of Sound
- Viscosity
- Thermal Conductivity
- Specific Heat at Constant Pressure
- Enthalpy

The model for these atmospheric properties is presented in two sections. The first covers altitudes from sea-level to 301,000 feet. Within this range the molecular weight is assumed to be constant and the temperature variation with altitude is a sequence of connected line segments. The second section of the model covers altitudes above 301,000 feet where the molecular weight decreases linearly with altitude. For this altitude range the approximate polynomial expressions for density and pressure suggested in Part 4 of Ref. 20 were used. Errors between the values given by the approximate expression and the 1962 Standard Model are less than 5% over the entire range of altitudes.

Atmospheric air is assumed to be an ideal gas for all altitudes. Therefore compressibility effects at low altitudes are neglected. The error in computed densities resulting from the ideal gas assumption may be as high as 0.05% for altitudes below 6 miles, but becomes less than 0.01% above 12 miles (Ref. 20). The air is also assumed to be in hydrostatic equilibrium.

All properties except geopotential altitude, specific heat and enthalpy are evaluated from expressions presented in Refs. 20 and 21. The expression for geopotential altitude was taken from Ref. 22, while specific heat and enthalpy data were taken from Ref. 23.

The model developed for the atmospheric properties is considered to be applied to altitudes up to 100 miles and to latitudes between 30 and 60°N. It is anticipated that atmospheric properties are not needed for altitudes exceeding 100 miles, because the convective heat flux from the fin system will be negligible at this altitude and above.

The properties for the earth's atmosphere are known with increased uncertainty as the altitude increases. In fact, the 1962 Standard Atmosphere (Ref. 21) consists of four regions as follows:

0 - 20 km	Standard
20 - 32 km	Proposed Standard
32 - 90 km	Tentative
90 - 700 km	Speculative

Any uncertainty in the atmospheric properties will naturally be reflected as an error in the convective heat flux on the shuttle vehicle. Fortunately during the ascent phase of the shuttle operation the convective flux from the radiator system is fairly small compared to the radiative flux by the time the shuttle has approached altitudes for which the atmospheric properties are considered to be "speculative"; on the other hand during re-entry, significant convective fluxes are known to exist at altitudes above 90 km. As a result every effort should be made to revise the existing atmospheric property model at high altitudes as new data become available.

The atmospheric model is based on several primary constants. The sea-level pressure, temperature, molecular weight, density and acceleration of gravity and the universal gas constant were assigned the fixed values of

$$\begin{aligned}
 P_o &= 2116.22 \text{ lbf/ft}^2 \\
 T_o &= 518.67 \text{ R} \\
 M_o &= 28.9644 \\
 \rho_o &= 0.07647 \text{ lbm/ft}^3 \\
 g_o &= 32.1741 \text{ ft/sec}^2 \\
 R^* &= 1545.31 \text{ ft lbf/lb mole R}
 \end{aligned}$$

Properties for Altitudes Less than 301,000 feet.

a. Geopotential Altitude - H

The state variables for air are expressed in terms of the single variable, the geopotential altitude

$$H = \int_0^Z \frac{g(z)}{g_0} dz =$$

$$Z - 1.573126 \times 10^{-7} Z^2 + 2.4656553 \times 10^{-14} Z^3$$

$$- 3.8667054 \times 10^{-21} Z^4 + 6.0621354 \times 10^{-28} Z^5$$

$$- 9.5013649 \times 10^{-35} Z^6 \quad (14.1)$$

where Z is the geometric altitude in meters,  $g_0$  is the acceleration of gravity at sea-level and  $g(Z)$  denotes the local acceleration of gravity. See Ref. 22 for details.

b. Temperature - T

The general expression of the temperature as a function of geopotential altitude is

$$T = T_b + L(H - H_b) \quad (14.2)$$

$T_b$  and  $H_b$  are the endpoints of straight-line segments representing  $T(H)$  and are listed, together with  $L(H)$  in the following table.

c. Molecular Weight - M

The molecular weight is constant at a value of 28.9644.

d. Pressure - P

Within a region where the temperature varies linearly, the ideal equation of state and the hydrostatic yield the following expressions for

$H_b$ (km)	L (K/km)	$T_b$ (K)
0		288.15
	- 6.5	
11		216.65
	0.0	
20		216.65
	1.0	
32		228.65
	2.8	
47		270.65
	0.0	
52		270.65
	-2.0	
61		252.65
	-4.0	
79		180.65
	0.0	
90		180.65

TABLE 4. Lapse Rate and Base Temperatures for  
Atmospheric Model

pressure:

$$\frac{P}{P_b} = \left[ \frac{T_b}{T_b + L(H - H_b)} \right]^{\frac{g_o M_o}{R^* L}} \quad (L \neq 0) \quad (14.3)$$

$$\frac{P}{P_b} = \exp \left[ - \frac{g_o M_o (H - H_b)}{R^* T_b} \right] \quad (L = 0) \quad (14.4)$$

$$\begin{aligned} \text{i.e. } 11 &\leq H \leq 20 \text{ km} \\ 47 &\leq H \leq 52 \text{ km} \\ 79 &\leq H \leq 90 \text{ km} \end{aligned}$$

The subscript "o" denotes a quantity evaluated at sea-level and the subscript "b" denotes a quantity evaluated at the base of one of the straight line segments of the atmospheric model.

c. Density -  $\rho$

The density may be calculated from the ideal equation of state once the temperature and pressure have been evaluated.

$$\rho = \frac{MP}{R^* T} \quad (14.5)$$

f. Speed of Sound -  $c$

The speed of sound was evaluated from the expression

$$c = \left[ \gamma \frac{R^* T}{M} \right]^{1/2} \quad (14.6)$$

For altitudes less than 301,000 feet the ratio of specific heats is taken to have a fixed value of 1.40.

g. Viscosity -  $\mu$

The dynamic viscosity was evaluated from the expression

$$\mu = \frac{\beta T^{3/2}}{T + S} \quad (14.7)$$

where

$$\beta = 1.458 \times 10^{-6} \frac{\text{kg}}{\text{sec m(K)}^{1/2}}$$

and

$$S = 110.4 \text{ K}$$

h. Specific Heat at Constant Pressure  $c_p$  and Enthalpy  $i$

Values for  $c_p$  and  $i$  between the temperatures of 100 R and 6400 R were taken from the standard Gas Tables (Ref. 23) and placed in the program in tabular form. A value of  $c_p$  and  $i$  at any temperature intermediate to a pair of tabular values was determined by an interpolation routine (see Section III 4).

Properties for Altitudes Greater than 301,000 Feet

a. Pressure - P

The pressure for altitudes between 301,000 and 528,000 feet is based on the polynomial approximation given in Part IV of Ref. 20. The pressure is written in terms of the sea-level pressure  $P_o$  in the form

$$P = P_o \left\{ \sum_{n=0}^{11} [A_n Z^n]^{-4} \right\} \quad (14.8)$$

where Z is the geometric altitude and values for  $A_n$  appear in Table 4.1 of Ref. 20.

b. Density -  $\rho$

The density is written in terms of a similar polynomial

$$\rho = \rho_o \left\{ \sum_{n=0}^{11} [B_n Z^n]^{-4} \right\} \quad (14.9)$$

where values of  $B_n$  appear in Table 4.1 of Ref. 20.

c. Molecular Weight - M

The molecular weight is assumed to vary linearly with altitude Z (see Fig. I.2.7 in Ref. 21). The resulting expression for M is

$$M = 28.9644 - 0.030949 (Z - 90)$$

where Z is in km.

d. Temperature - T

The temperature is calculated from the values for pressure, density and molecular weight indicated above from the ideal equation of state



$$T = \frac{PM}{\rho R^*}$$

e. Speed of Sound - c

For altitudes greater than 301,000 feet the equation for the speed of sound is the same one as used for the lower altitudes, but the ratio of the specific heats is no longer assumed to be equal to 1.40. The ratio of the specific heats varies with the molecular weight according to the expression

$$\gamma = \frac{c_p}{c_p - R^*/M}$$

The remaining properties are calculated using identical expressions to those outlined for altitudes less than 301,000 feet.

### III. NUMERICAL TECHNIQUES

#### 1. Introduction

The analysis carried out in Chapter II lead, as far as the mathematical problem formulation is concerned, to three initial value problems and one matrix manipulation. The three initial-value problems are to establish

- (1) the initial conditions for the coolant fluid, defined by Eqs. 3.42 through 3.45,
- (ii) the dynamics of the coolant flow, defined by Eqs. 3.40 through 42 and 44,
- (iii) the temperature field throughout the system, defined by Eqs. 2.18 through 2.22 for the fin, Eqs. 3.39 and 46 for the coolant, Eqs. 4.6, 7 and 8 for the channel wall, Eqs. 4.7, 5.3 and 5 for the protection layer, or Eq. 4.11 replacing Eqs. 2.19, 4.6, 7 and 8 and 5.3 and 5 in the case where Eq. 4.9 is satisfied. These equations must be supplemented by the specification of the initial, non-dimensional temperature everywhere in the system.

Each initial-value problem is solved by a fourth-order Runge-Kutta-Simpson integration discussed in Section III-2.

The radiosity equation, Eq. 6.17 requires the matrix manipulation, namely either the solution of a system of linear algebraic equations, or a matrix inversion whenever the optical properties of the thermal control coating are considered temperature independent. Either task is accomplished by elementary row operations which transform, in a single process, the augmented coefficient matrix into a row-reduced echelon matrix. The reader is referred for this transformation to standard texts on linear algebra (Ref. 9).

Additional mathematical operations are programmed as subprograms which may be generally applied and which are discussed in Sections III-3 through III-7 in this order: an evaluation of polynomials in one variable, an Aitken interpolation, first and second differentiation, definite integration and integration with variable upper integration limit for functions of equally spaced arguments, and solution to system of linear algebraic equations.

## 2. The Runge-Kutta-Simpson Integration

Two types of initial-value problems are to be solved in this program. The first type includes Item (i) and (ii) mentioned in the Introduction, namely the fluid dynamics exclusive of the transient fluid temperature field, and involves ordinary, first-order differential equations, linear in the derivatives with respect to the axial distance  $\zeta$ , that is Eqs. 3.42, 43 and 44. The equations are solved explicitly for these derivatives so as to take on the general form of Eq. 2.1:

$$\frac{dy_i}{dx} = f_i(x, y_1, y_2, \dots, y_n) \quad (2.1)$$

$$y_i(0) = a_i, \quad i = 1, 2, \dots, N \quad (2.2)$$

Equation 2.2 constitutes the appropriate initial conditions. The other type of initial-values problem, mentioned as item (iii) in the Introduction, involves partial differential equations which are linear and of the first order in the time-derivatives; moreover, all equations, Eqs. 2.18, 3.46, 4.8 and 5.3, are explicit in the time derivatives. Having subdivided the radiator system into intervals, equally spaced in each appropriate domain (fluid, wall, fin, etc.), and then written the different equations corresponding to each one of the resulting  $N$  interior nodal points, one may discretize the spatial derivatives occurring on the right-hand sides of the partial differential equations. The result is a set of ordinary differential equations, with time as the independent variable but of a form which is identical to Eq. 2.1. Equation 2.2 is given by the initial temperature distribution; a uniform temperature was chosen for the first start of the integration ( $a_i = a$ ;  $i = 1, 2, \dots, N$ ), subsequent integrations during optimization runs are expected to start from the previously computed steady-state temperature distributions. The boundary conditions may be satisfied in three different ways. Either, the temperatures are computed directly from the finite-difference equation representing the boundary conditions at the end of every time step, or secondly, the boundary conditions may be included into the system of Eqs. 2.1 after differentiation

with respect to time, or lastly, an equation of the form of Eqs. 2.1 may be derived directly from a control volume bounded at one side by the boundary of interest. All three possibilities have been utilized.

Discretization introduces obviously a truncation error; all spatial derivatives are represented consistently with a truncation error proportional to the square of the local spatial interval (see Sect. III 5) but higher order terms may be included anytime by modifying a single program unit each for the first and second derivatives.

The system of Eqs. 2.1 and 2.2 is solved by a fourth-order Runge-Kutta integration, that is if the  $f_i$  in Eqs. 2.1 have continuous fourth-order derivatives, the time-related accuracy of the integration is of order four (Refs. 26 and 27). Under much weaker conditions, namely uniform Lipschitz continuity, Ref. 26, the accuracy is still first-order and stability is secured. It may be noted that the Lipschitz continuity is also the prerequisite for uniqueness of the solution to Eqs. 2.1.

An existing single-precision, floating-point Runge-Kutta-Simpson subprogram, SUBROUTINE RKS, written by R. Schubert at the Aerospace Corporation was used. Its fixed-step integration mode was employed for the integration of the fluid flow variables along the channel axis, while the transient temperature field was integrated with variable time steps, chosen automatically so as to keep the "truncation error" per time step below a specified limit. The absolute and relative errors  $A_i$  and  $R_i$  are specified, by the user, for each variable  $y_i$ , and after every Runge-Kutta integration step a Simpson integration is carried out over the same interval and with the intermediate derivatives as used in the former integration. From the difference  $D_i$  between the two integrations is calculated the "truncation error" measure

$$E_m = \max E_i = \left| \frac{D_i}{A_i + R_i |y_i|} \right|, \quad i = 1, \dots, N \quad (2.3)$$

and if  $0.75 < E_m$  then the time step DEL is divided by  $\sqrt[5]{10}$  and the step is repeated, if  $0.075 < E_m \leq 0.75$  then DEL is multiplied by  $\sqrt[5]{10}$  for the

subsequent step.

All variables  $y_i$  are set equal to their initial values in the program which calls RKS. During the integration RKS interacts directly with two other subroutines, namely DERIV and CNTRL, whose names are the first elements of the argument list in the call statement. The first subroutine, DERIV, serves to compute all  $N$  derivatives  $dy_i/dx$  in accordance with Eqs. 2.1. The second subroutine, CNTRL, controls the output during integration and the termination of integration. Output of current values of all variables along with important system parameters is provided under two different integration modes: general transient system simulation,  $MSTOR = 0$  in NAMELIST/RUNOPT/, produces output in arbitrarily chosen, fixed time steps, DTWRTE, up to the final time TEND, both specified in NAMELIST/RUNOPT/ and in hours; the second mode serves to compute the steady-state conditions and is invoked by setting  $MSTOR = 1$  and by specifying the number LIMWRT of time intervals DTWRTE at which output is desired.

The integration under the second mode ( $MSTOR = 1$ ) is terminated as soon as the expected truncation error due to program termination is less than five times the specified relative error per time step, RLIMIT, that is  $R_i$  in Eq. 2.3. The largest truncation error associated with the  $j$ -th time step is anticipated on the basis of Eqs. 1.1 and 2 in Section II as follows

$$\delta_j = \max_i \delta_{i,j} = \max_i \left\{ \Delta_j \tau \frac{\dot{y}_{i,j}}{\ln \frac{\dot{y}_{i,j} - 1}{\dot{y}_{i,j}}} \right\}, \quad i = 1, 2, \dots, N \quad (2.4)$$

The maximum is taken from all  $N$  modal points,  $\Delta_j \tau$  is the current integration step size with index  $j$ , and  $\dot{y}_i$  stands for the  $dy_i/dx$  in Eqs. 2.1.

The argument list of RKS (and RKSF) is as follows:

- |      |                                      |  |
|------|--------------------------------------|--|
| (i)  | DERIV, name of derivative subroutine | } declared as EXTERNAL<br>in calling program |
| (ii) | CNTRL, name of control subroutine    |  |

- (iii) Y , array name\*, containing the  $y_i$ 's in Eqs. 2.2 \*\*
- (iv) DY, array name\*, containing the  $dy_i/dx$  in Eqs. 2.1
- (v) A , array name\*, containing the  $A_i$ 's in Eq. 2.3 \*\*
- (vi) R , array name\*, containing the  $R_i$ 's in Eq. 2.3 \*\*
- (vii) T , the independent variable X in Eqs. 2.1 \*\*
- (viii) DEL, the integration step\*\*, DEL  $\neq$  0
- (ix) N , (integer) the number of equations\*\*
- (x) IFVD = 0: variable step size\*\*, see Eq. 2.3  
= 1: fixed step size equal to DEL
- (xi) IBKP = 0: adjust step size at most once before repeat,\*\*  
= 1: adjust in accordance to Eq. 2.3
- (xii) NTRY = 1: continue integration\*\*, normal start  
= 2: return from RKS  
= 3: repeat last step with new DEL  
= 4: restart
- (xiii) IERR = 0, normal integration  
= -1 indicates singularity when IFVD = 0  
= +1 indicates denominator vanishes in Eq. 2.3 at some time during integration.
- (xiv) through (xx) are array names\* with which the user need not be concerned except YS that contains the y's in 2.1 at the previous times step: DELY, PD, SD, YS, YST, DYST, YSIMP.

} to be changed  
in  
CNTRL

The SUBROUTINE DERIV communicates with RKS only via its argument list which contains, in this order, Y, DY, and T, as specified above under iii, iv, and vii. Here, the current values of Y and T are supplied to DERIV, and the corresponding values of DY returned by DERIV to RKS.

The SUBROUTINE CNTRL communicates with RKS also via its argument list. It contains Y, DY, DEL, T, NTRY, IFVD as specified above under iii, iv, viii, vii, xii and x, respectively. From the array Y are available for output all the results of integration. The time step may be modified to reach a specific time value; and by specifying NTRY one controls the integration process from

---

\* Declared in calling program as array with dimension size equal to the number of differential equations.

\*\* To be specified prior to the calling statement.

variable to fixed step size during the integration by resetting IFVD.

This completes the discussion of the integration of both ordinary and partial differential equations as they occur in the analysis developed in Chapter II. The discussion is deemed sufficient to enable the user to apply the RKS routine to other problems as well.

### 3. The Evaluation of Polynomials

All polynomials

$$z = a_0 + a_1x + a_2x^2 + \dots + a_Nx^N \quad (3.1)$$

are carried out in a function subprogram based on the simple, efficient recurrence relation

$$\begin{aligned} z_0 &= a_N \\ z_{i+1} &= x \cdot z_i + a_{i+1} \\ i &= 0, 1, \dots, N-1 \end{aligned} \quad (3.2)$$

$$z = z_N$$

The coefficients  $a$ ,  $i = 0, 1, \dots, N$  must be placed, in the calling program, into an array of dimension  $(N + 1)$ ,  $N$  is an arbitrary positive integer.

The procedure is coded as a function subprogram called  $POLY(N,A,X)$ , where  $X$  is the argument  $x$  in Eq. 3.1,  $A$  is the array containing  $M = N + 1$  elements starting with  $A(1) = a_0$ .



#### 4. Aitken Interpolation

Experimental data and supporting computer results which are not represented by analytic expressions are interpolated by Aitkens interpolation technique (Ref. 27). An  $(n + 1)$ -point Lagrangian interpolation is reduced to a sequence of  $1/2 n (n + 1)$  linear interpolations. The interval spacing is arbitrary; and any number  $M > n$  of ordered pairs  $(x_i, y_i)$  can be supplied in the calling program. The  $n$  points of interpolation are spaced equally about the point  $x$  of interpolation. It should be noted, however that unless  $n = N$  or  $n = 2$  the result  $y(x)$  is not continuous in general. Care must also be taken that all nodes  $x_1, x_2, \dots$  are distinct.

The procedure is coded as a function subprogram called  $YINT(X, Y, M, N, P)$ , where  $X$  and  $Y$  are the names of arrays that have the same dimension  $M$  and contain the ordered pairs  $(x_i, y_i)$ ,  $i = 1, 2, \dots, M$  such that  $x_1 < x_2 < \dots < x_M$ . The number  $n$  of points used for the interpolation is specified as  $N$ , and the value of  $x$  at which to interpolate is supplied as  $P$ . Note that  $2 \leq N \leq M$  must be satisfied.

## 5. Numerical Differentiation

The first and second derivative of tabulated functions of equally spaced arguments is carried out in SUBROUTINE DDX(Y,DY,DX,N) and in SUBROUTINE D2DX2(Y,D2Y,DX,N), respectively. Each subroutine requires that two arrays be declared in a DIMENSION statement in the calling program, to have at least N elements; one array for the set of ordinates  $Y \rightarrow y_i$  supplied by the calling program, the other array for the return of the results, that is  $DY \rightarrow dy_i/dx$  or  $D2Y \rightarrow d^2y_i/dx^2$ . The argument interval  $\Delta x$  and the number of ordinates  $y_i$  are to be specified as DX and N, respectively. However, in order that terms of order  $\Delta x$  be retained including at the endpoints of the domain, N must be no less than 3 for DDX and 4 for D2DX2. The truncation error is of order  $y'''(\Delta x)^2$  and  $y^{IV}(\Delta x)^2$ , respectively, for DDX and D2DX2.

## 6. Numerical Integration

The definite integral

$$F = \int_{x_1}^{x_N} y(x_i) dx, \quad 1 \leq i \leq N; \quad N \geq 2$$

and the indefinite integral

$$G_j = \int_{x_1}^{x_j} y(x_i) dx + G(x_1)$$

$$1 \leq i \leq N; \quad 1 < j \leq N; \quad N > 3$$

of a tabulated function  $y_i$  of an equally spaced argument,  $x_1$ ,  $x_1 + \Delta x$ ,  $x_1 + 2 \Delta x, \dots, x_1 + (N - 1)\Delta x$  is carried out by a modified Simpson integration in the FUNCTION DEFINT(Y,DX,N) Subprogram and in the SUBROUTINE FINT(Y,YO,DX,N,F), respectively.

For DEFINT the ordinates  $y_i$  are to be placed in the array Y whose dimension of no less than N elements must be declared in the calling program. The argument interval and the number of ordinates are specified as DX and N, respectively.

For FINT there are two array declarations necessary in the calling program, both for at least N elements; one for the integrand  $Y \rightarrow y_i$  and the other for the integral  $F \rightarrow G_i$ . The integration constant  $G(x_1)$ , the argument interval and the number of ordinates are to be supplied as YO, DX and N, respectively.

The truncation error of composite Simpson integration is

$$\frac{x_1 - x_N}{180} (\Delta x)^4 y^{(iv)}(\xi) \quad \text{with} \quad x_1 \leq (\xi) \leq x_N.$$

## 7. Solution to Systems of Linear Algebraic Equations

Systems of linear algebraic equations are solved by the Gauss-Jordan elimination process. The same technique is used to invert matrices.

Consider the system of  $n$  linear algebraic equations

$$\sum_{j=1}^n A_{ij} X_j = y_i .$$

The solution is obtained by performing that sequence of elementary row operations on the augmented coefficient matrix

$$\left( \begin{array}{cccccc} A_{11} & A_{12} & . & . & . & A_{1n} & y_1 \\ A_{21} & & & & & . & y_2 \\ . & & & & & . & . \\ . & & & & & . & . \\ . & & & & & . & . \\ . & & & & & . & . \\ A_{n1} & & & & & A_{nn} & y_n \end{array} \right)$$

which leads to the row-reduced echelon matrix

$$\left( \begin{array}{cccccc} 1 & 0 & . & . & . & 0 & x_1 \\ 0 & 1 & & & & . & x_2 \\ . & . & . & 0 & & . & . \\ . & & . & . & & . & . \\ . & & & . & . & . & . \\ . & & 0 & . & . & 0 & . \\ 0 & & & & 0 & 1 & x_n \end{array} \right)$$

Elementary row operations are defined by

- (i) multiplication of a row by the scalar  $c \neq 0$
- (ii) replacement of the  $r$ -th row by the  $r$ -th row plus  $c$  times the  $s$ -th row;  $c \neq 0$ ,  $r \neq s$ ;  $r, s \leq n$
- (iii) interchange of any two rows.

The augmented coefficient matrix consists of the coefficient matrix  $A_{ij}$  in the first  $n$  columns and the known column vector  $y_i$  in its last,  $(n+1)$ -st column. The row-reduced echelon matrix has the properties that

- (i) the first non-zero element in each non-zero row is 1,
- (ii) every zero-row occurs below every non-zero row,
- (iii) if the first  $r$  rows,  $i = 1, 2, \dots, r$  have their non-zero entry in column  $k_i$  then the  $k_i$ 's satisfy  $k_1 < k_2 < \dots < k_r$ .

In our specific case, the row-reduced echelon matrix has the identity matrix in place of the coefficient matrix.

When a matrix is inverted then the augmented coefficient matrix consists of the  $n \times n$  coefficient matrix in its first  $n$  columns and the  $n \times n$  identity matrix in the second  $n$  columns,  $j = n+1, n+2, \dots, 2n$ . The process indicated above leads  $n \times n$  identity matrix in the first  $n$  columns and the inverted coefficient matrix in the second  $n$  columns, from  $j = n+1$  through  $j = 2n$ .

The particular elementary row operations required are

- (i) division of the  $i$ -th row by  $A_{ii}$ ,
- (ii) subsequent multiplication of the resulting  $i$ -th row by the element  $A_{ki}$  of the  $k$ -th row,
- (iii) and subsequent replacement of the  $k$ -th row by the difference between the  $k$ -th row and the  $i$ -th row.

This process has to be repeated, essentially, for each row  $i = 1, \dots, n$ .

## IV RECOMMENDATIONS AND CONCLUSIONS

The analysis described in this report has been successful in simulating the transient heat transfer characteristics of a radiator system under operational conditions expected in flight. The analysis serves as a basis for a rigorous computer program that has been systematically sub-divided into modular subprograms. The modular concept facilitates repeated simulation runs with different structural materials, meteoroid protection material, coolant fluids, and thermal control coatings.

The program predicts the net system heat rejection, the fin, tube and fluid temperature profiles, fluid pressure field as well as the meteoroid protection layer thickness and mass of the entire system. Optimization of the system performance may be achieved through enumeration of the parameter sets.

The program has been thoroughly checked and run for a large number of different simulation cases. A sample representation of these cases may be found in reference [29]. These sample runs have aided in the design of the radiator system during ascent, reentry, transient orbital and steady state orbital conditions. As a result of these computer runs it is recommended that an unprotected radiator not be used during reentry phases of the shuttle operation, because during reentry the aerodynamic heating has been found to exceed the ability of the radiator to reject heat. Also experience gained from the sample runs has shown that the rigorous analysis should not be used for parameter studies of the heat rejection system until a satisfactory optimum domain has been identified by the use of the Simplified Analysis [30]. Displays of typical results are presented in that report.

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APPENDIX A  
STRUCTURAL MATERIAL PROPERTIES

The thermodynamic and mechanical properties of all materials which make up the radiator system are summarized in Appendices A through C.

Appendix A contains properties for the three structural materials: copper, aluminum and beryllium. Copper and aluminum are intended to be used primarily as fin and tube material, while beryllium was selected for a meteoroid protection material. Four properties are evaluated for each material: specific heat at constant pressure, thermal conductivity, modulus of elasticity, while  $(1/k) (dk/dT)$  is computed by differentiating the thermal conductivity relationship with respect to temperature.

Appendix B contains properties for four coolant fluids: helium, Dow Corning 200 Silicon oil, the liquid metal NaK, and two 3-M Company fluorochemical liquids FC-43 and FC-75. Six properties are evaluated for each coolant fluid: isobaric thermal expansion coefficient, isothermal compressibility, specific heat at constant pressure, enthalpy, thermal conductivity and dynamic viscosity. In addition, two equations of state are included for each fluid, one explicitly in density and one explicit in pressure, while the property  $(1/k) (dk/dT)$  is computed by differentiating the thermal conductivity relationship with respect to temperature.

Appendix C contains the total hemispherical emittance and two auxiliary radiative properties used in the program for the surface coating Z-93.

All property relationships listed in these Appendices are presented in analytical form obtained by fitting a power polynomial through the data points. The data points listed in the tables are taken from the reference entered before each table. Numerical techniques used for the curve fitting process are explained in Section III.

The polynomial expression for each property has been compared with the referenced data and within the listed temperature range has been found to deviate by no more than the percentage error indicated.



## I. COPPER

1. Specific Heat

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 1, 1967, pp. 456-7.

Data Points:

T	C <sub>p</sub>
600 R	0.0920 Btu/(lbm R)
1000	0.0975
2000	0.1112

Polynomial Fit:

Temperature Range: 400 to 2000 R

Equation:

$$C_p = (0.08375 + 1.375 \times 10^{-5} \text{ TR}^{-1}) \times 32.174 \text{ Btu/(slug R)} \quad (\text{A.1})$$

Maximum Error: There was no difference between the computed value and the input data within the accuracy of the computer.

2. Thermal Conductivity

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 1, 1967, pp. 458-9.

Data Points:

T	k
600 R	228.369 Btu/(hr ft R)
800	225.708
1000	222.805
1200	219.418
1400	215.306

Polynomial Fit:

Temperature Range: 500 to 1800 R

Equation:

$$k = (228.369 - 2.62067 \theta - 0.04033 \theta^3) \text{ Btu/(hr ft R)} \quad (\text{A.2})$$

where

$$\theta = \frac{T - 600.0\text{R}}{200.0\text{R}} \quad (\text{A.3})$$

Maximum Error: 0.87%

### 3. Temperature Variation of Thermal Conductivity

Eq. A.2 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{1}{200} \frac{(-2.62067 - 0.121 \theta^2)}{(228.369 - 2.62067 \theta - 0.04033 \theta^3)} \text{ R}^{-1} \quad (\text{A.4})$$

### 4. Modulus Elasticity:

Reference: "Material Manual," TRW Equipment Laboratories, February 1966, Report No. ER-6756, Contract No. NAS 9-4884, Fig. 50.

Data Points:

t	Y
0 F	$16.55 \times 10^6 \text{ lbf/in}^2$
400	14.35
800	9.65
1200	3.82

Polynomial Fit:

Temperature Range: 500 to 1600 R

Equation:

$$Y = (16.55 - 0.4933 \theta - 1.935 \theta^2 + 0.2283 \theta^3) \times 1.44 \times 10^8 \text{ (lb f/ft}^2\text{)} \quad (\text{A.5})$$

where

$$\theta = \frac{T - 459.67 \text{ R}}{200 \text{ R}} \quad (\text{A.6})$$

Maximum Error: 0.44%

## II. ALUMINUM 7075

1. Specific Heat

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 2-11, 1967, pp. 810-11.

Data Points:

T	$c_p$
400 R	0.182 Btu/(lbm R)
600	0.209
800	0.226
1000	0.244
1200	0.270

Polynomial Fit:

Temperature Range: 300 to 1200 R

Equation:

$$c_p = (0.182 + 0.03616 \theta - 0.011417 \theta^2 + 0.00233 \theta^3 - 0.000083 \theta^4) \quad (A.7)$$

X 32.174 Btu/(slug R)

where

$$\theta = \frac{T - 400 \text{ R}}{200 \text{ R}} \quad (A.8)$$

Maximum Error: 0.34%

## 2. Thermal Conductivity

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 2-11, pp. 812-13.

Data Points:

T	k
400 R	88.50 Btu/(hr ft R)
600	100.395
800	105.96
1000	104.024
1200	99.18

Polynomial Fit:

Temperature Range: 300 to 1200 R

Equation:

$$k = (88.5 + 13.0665 \theta + 0.33275 \theta^2 - 1.758 \theta^3 + 0.25375 \theta^4)$$

$$\text{Btu/(hr ft R)} \quad (\text{A.9})$$

where

$$\theta = \frac{T - 400 \text{ R}}{200 \text{ R}} \quad (\text{A.10})$$

Maximum Error: 0.96%

### 3. Temperature Variation of Thermal Conductivity

Equation A.9 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \quad (A.11)$$

$$\frac{1}{200} \frac{(13.0665 + 0.6655 \theta - 5.25 \theta^2 + 1.015 \theta^3)}{(88.5 + 13.0665 \theta + 0.33275 \theta^2 - 1.758 \theta^3 + 0.25375 \theta^4)} R^{-1}$$

### 4. Modulus of Elasticity

Reference: "Material Manual," TRW Equipment Laboratories, February 1966, Report ER-6756, NAS 9-4884, Fig. 50.

Data Points:

t	Y
0 F	$10.71 \times 10^6$ lbf/in <sup>2</sup>
200	9.90
400	8.50
600	6.15

Polynomial Fit:

Temperature Range: 500 to 1200 R

Equation:

$$Y = (10.71 - 0.63 \theta - 0.115 \theta^2 - 0.06 \theta^3) \times 1.44 \times 10^8 \text{ lbf/ft}^2 \quad (A.12)$$

where

$$\theta = \frac{T - 459.67}{200} \frac{R}{R} \quad (\text{A.13})$$

Maximum Error: 0.38%

## III. BERYLLIUM (1/2 - 3% Be O)

1. Specific Heat

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 6-11, 1967, pp. 753-4.

Data Points:

T	$c_p$
800 R	0.536 Btu/(lbm R)
1000	0.585
1200	0.622
1400	0.652
1600	0.680

Polynomial Fit:

Temperature Range: 400 to 1700 R

Equation:

$$c_p = (0.536 + 0.05667 \theta - 0.0085 \theta^2 + 0.00083 \theta^3) \quad (\text{A.14})$$

X 32.174 Btu/(slug R)

where

$$\theta = \frac{T - 800 \text{ R}}{200 \text{ R}} \quad (\text{A.15})$$

Maximum Error: 0.88%



## 2. Thermal Conductivity

Reference: Touloukian, Y. S., "Thermophysical Properties of High Temperature Solid Materials," Thermophysical Properties Research Center, Purdue University, Vol. 6-11, 1967, pp. 757-9.

Data Points:

T	k
400 R	108.863 Btu/(hr ft R)
600	98.944
800	89.751
1000	80.80
1200	72.091

Polynomial Fit:

Temperature Range: 400 to 1700 R

Equation:

$$k = (108.863 - 10.5643 \theta + 0.82683 \theta^2 - 0.20167 \theta^3 + 0.020167 \theta^4) \text{ Btu/(hr ft R)} \quad (\text{A.16})$$

where

$$\theta = \frac{T - 400 \text{ R}}{200 \text{ R}} \quad (\text{A.17})$$

Maximum Error: 0.90%

## 3. Temperature Variation of Thermal Conductivity

Equation A.15 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} =$$

(A.18)

$$\frac{1}{200} \frac{(-10.5643 + 1.65367 \theta - 0.60501 \theta^2 + 0.080668 \theta^3)}{(108.863 - 10.5643 \theta + 0.826830 \theta^2 - 0.20167 \theta^3 + 0.020167 \theta^4)} R^{-1}$$

#### 4. Modulus of Elasticity

Reference: "Material Manual," TRW Equipment Laboratories, February 1966, Report ER-6756, Contract No. NAS 9-4884, Fig. 51.

Data Points:

t	Y
0 F	44.36 x 10 <sup>6</sup> lbf/in <sup>2</sup>
400	40.41
800	33.95
1200	21.80

Polynomial Fit:

Temperature Range: 500 to 1700 R

Equation:

$$Y = (44.36 - 3.755 \theta + 0.335 \theta^2 - 0.53 \theta^3) \times 1.44 \times 10^8 \text{ lbf/ft}^2 \quad (\text{A.19})$$

where

$$\theta = \frac{T - 459.67 \text{ R}}{400 \text{ R}} \quad (\text{A.20})$$

Maximum Error: 0.28%

APPENDIX B  
COOLANT FLUID PROPERTIES

## I. HELIUM

1. Equation of State Explicit in Pressure

Reference: Akin, S. W., Trans. ASME, Vol. 72, p. 751, 1950.

This reference was used for all Helium properties and for brevity it is not repeated as the reference for the properties listed below.

Equation: The National Bureau of Standards has published a Benedict-Webb-Rubin equation for helium; this equation was found to be valid only up to the specified pressure limit of 3000 lbf/in<sup>2</sup>. Preference was therefore given to the following Beattie-Bridgeman equation:

$$p = \rho^2 [RT(1 - \alpha) \left( \frac{1}{\rho} + B_1 \right) - A] \quad (\text{B.1})$$

where 
$$\alpha = \frac{C}{T^3} \rho$$

$$A = A_1(1 - a \rho)$$

The values of the constants in Eq. B.1, in MKSA units, are:

$$\begin{aligned} R &= 2.07702 \times 10^3 \text{ Nm/kgK} \\ A_1 &= 1.369595 \times 10^2 \text{ Nm}^4/\text{kg}^2 \\ B_1 &= 3.5002295 \times 10^{-3} \text{ m}^3/\text{kg} \\ C &= 1.0000658 \times 10^1 \text{ km}^3/\text{kg} \\ a &= 1.496103 \times 10^{-2} \text{ m}^3/\text{kg} \end{aligned}$$

Temperature Range: 160 to 860 R

Pressure Range: 2116 to 360000 lbf/ft<sup>2</sup>

Maximum Error: 0.095%

2. Equation of State Explicit in Density

Since the equation of state is needed explicit in density, Eq. B.1 was solved using Newton-Raphson iteration method along an isotherm to give

$$\rho_{i+1} = \rho_i - \frac{p - p(\rho_i)}{(\partial p / \partial \rho)_T} \quad (\text{B.2})$$

Using Eq. B.1, one obtains

$$\left(\frac{\partial p}{\partial \rho}\right)_T = RT + 2(RB_1T - A_1 - \frac{CR}{T^2})\rho + 3(A_1a - \frac{CRB_1}{T^2})\rho^2 \quad (\text{B.3})$$

### 3. Isobaric Thermal Expansion Coefficient

The isobaric thermal expansion coefficient is defined by the equation

$$\beta = - \frac{1}{\rho} \left(\frac{\partial \rho}{\partial T}\right)_p \quad (\text{B.4})$$

Since the equation of state (Eq. B.1) is explicit in the pressure, one can write  $\beta$  as:

$$\beta = \frac{1}{\rho} \frac{(\partial p / \partial T)_\rho}{(\partial p / \partial \rho)_T} \quad (\text{B.5})$$

or

$$\beta = \frac{R[\rho + (B_1 + \frac{2C}{T^3})\rho^2 + \frac{2CB_1}{T^3}\rho^3]}{\rho[RT + 2\rho(RB_1T - A_1 - \frac{CR}{T^2}) + 3\rho^2(A_1a - \frac{CRB_1}{T^2})]} \quad (\text{B.6})$$

### 4. Isothermal Compressibility

The isothermal compressibility is defined by the equation

$$\kappa = \frac{1}{\rho} \left( \frac{\partial \rho}{\partial p} \right)_T \quad (\text{B.7})$$

Making use of Eq. B.3, the isothermal compressibility can be written as

$$\kappa = 1/\rho \left[ RT + 2\rho(RB_1T - A_1 - \frac{CR}{T^2}) + 3\rho^2(A_1a - \frac{CRB_1}{T^2}) \right] \quad (\text{B.8})$$

##### 5. Specific Heat at Constant Pressure:

Equation: Experimental and quantum statistical data for helium show that at zero-pressure, the specific heat at constant volume is independent of temperature

$$c_v^0 = \frac{3}{2} R \quad (\text{B.9})$$

Using Maxwell's equations, the following expression is obtained

$$c_v = c_v^0 - T \int_{\rho_0}^{\rho} \left( \frac{\partial^2 p}{\partial T^2} \right)_{\rho} \frac{d\rho'}{(\rho')^2} \quad (\text{B.10})$$

From the equation of state, (Eq. B.1) the integration is carried out in closed form to give

$$c_v = R \left[ \frac{3}{2} + 6a \left( 1 + \frac{\rho}{2} B_1 \right) \right] \quad (\text{B.11})$$

The relation between specific heat at constant pressure and that at constant volume is given by:

$$c_p = c_v + \frac{TB^2}{\rho\kappa} \quad (\text{B.12})$$

Temperature Range: 180 to 900 R

Pressure Range: 2116 to 216000 lbf/ft

Maximum Error: 0.58% for cp

## 6. Enthalpy

The variation of internal energy with both temperature and density is

$$du = c_v dT + [p - T \left( \frac{\partial p}{\partial T} \right)_\rho] \frac{d\rho}{\rho^2} .$$

Substitution of  $c_v$  from Eq. B.11 and equation of state data from Eq. B.1 followed by integration along an isochore and an isotherm, one obtains

$$u = u_o + R \left[ \frac{3}{2} (T - T_o) + 3 \rho C \left( 1 + \frac{B_1}{2} \right) \left( \frac{1}{T_o^2} - \frac{1}{T^2} \right) \right] + \left( \frac{3 R C}{T^2} + A_1 \right)$$

$$(\rho_o - \rho) + \frac{1}{2} \left( \frac{3 C R B_1}{T^2} - A_1 \right) (\rho_o^2 - \rho^2)$$

where  $u_o = 3.992 \times 10^4$  j/kg

$T_o = 10.938889$  K

$\rho_o = 4.669193$  kg/m<sup>3</sup>

with T in K and u in j/kg.

The enthalpy may then be determined from the equation

$$h = u + \frac{p}{\rho}$$

## 7. Thermal Conductivity

Data Points:

T	k
160 R	0.0404 Btu/(hr ft R)
360	0.0676
560	0.090
760	0.1094

Polynomial Fit:

Temperature Range: 160 to 860 R

Equation:

$$k = (0.0404 + 0.0302 \theta - 0.0033 \theta^2 + 0.0003 \theta^3) \text{ Btu/(hr ft R)} \quad (\text{B.13})$$

where

$$\theta = \frac{T - 160 \text{ R}}{200 \text{ R}}$$

Maximum Error: 0.54%

## 8. Temperature Variation of Thermal Conductivity

Eq. B. 13 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{1}{200} \frac{(0.0302 - 0.0066 \theta + 0.0009 \theta^2)}{(0.0404 + 0.0302 \theta - 0.0033 \theta^2 + 0.0003 \theta^3)} \frac{1}{R} \quad (\text{B.14})$$

## 9. Dynamic Viscosity

Equation: Viscosity correlations are usually based on the concept of residual viscosity:

$$\psi_1(\rho) = \mu(\rho, T) - \mu^*(T) \quad (\text{B.15})$$



where

$\psi_1$  is the residual viscosity (function of density alone).

$\mu^*$  is the dynamic viscosity at atmospheric pressure.

For helium, the dynamic viscosity is given by:

$$\mu = \mu^* = (2.58394 \times 10^{-5} T/^{\circ}\text{R})^{0.647} \text{ slug/(ft hr)} \quad (\text{B.16})$$

Temperature Range: 160 to 660 R

Maximum Error: 0.29%.

## II. SILICON OIL

The following properties are for Dow Corning 200 Silicon Oil (1 Centistoke at 77 F).

1. Isothermal Compressibility

Reference: Gunst, S. B., "Density-Pressure Relationships for Two Low-Viscosity Dimethyl Siloxanes," Trans. ASME 72, May 1950, pp. 401-7.

Data Points: Variation of  $\kappa$  with temperature at 0 psig and 500 psig are given below:

t	$\kappa_0$	$\kappa^0_{500}$
100 F	$12.35 \times 10^{-6} \text{ in}^2/\text{lb f}$	$11.60 \times 10^{-6} \text{ in}^2/\text{lb f}$
150	16.05	14.94
200	20.45	18.82
250	26.25	23.86
300	36.55	31.88

Polynomial Fit:

Temperature Range: 560 to 760 R

Pressure Range: 2116 to 74116 lb f/ft<sup>2</sup>

Equation: The variation of  $\kappa_0$  with temperature at 0 psig is given by

$$\kappa_0 = (12.35 + 2.9833 \theta + 1.1 \theta^2 - 0.48333 \theta^3 + 0.1 \theta^4) \times 10^{-6} \quad (\text{B.17})$$

$\text{in}^2/\text{lb f}$

where  $\theta = \frac{T - 559.67 \text{ R}}{50 \text{ R}} \quad (\text{B.17a})$

$\kappa$  is assumed to vary linearly on the above range of pressure, hence

$$\kappa = a + b p \quad (\text{B.18})$$

where

$$a = \kappa_o \quad (\text{B.18a})$$

$$b = \left. \frac{\partial \kappa}{\partial p} \right|_T \approx \frac{\kappa_{500} - \kappa_o}{500 \text{ psi}}$$

and  $p$  is the pressure in psig and  $\kappa$  is in  $\text{in}^2/\text{lbf}$

Fitting a power polynomial through  $\left( \frac{\partial \kappa}{\partial p} \right)_T$  with the same values for temperature as indicated in the table results in the equation

$$b = (-1.5 - 0.0133 \theta - 1.18 \theta^2 + 0.57333 \theta^3 - 0.1 \theta^4) \times 10^{-9} \quad (\text{B.19})$$

$\text{in}^4/\text{lbf}^2$

where  $\theta$  is given in Eq. B.17a.

**Maximum Error:** There was no difference between the computed and the input data within the accuracy of the computation.

## 2. Equation of State Explicit in Density

**Reference:** Gunst, S. B., "Density-Pressure Relationships for Two Low-Viscosity Dimethyl Siloxanes, "Trans. ASME, May 1950, pp. 401-7.

**Data Points:** Values for the variation of density with temperature at 0 psig are given below:

$t$	$\rho_o$
150 F	0.7767 gm/cm <sup>3</sup>
200	0.7479
250	0.7188
300	0.6900

Polynomial Fit:

Temperature Range: 540 to 760 R

Pressure Range: 2116 to 146116 lbf/ft<sup>2</sup>

Equation:

$$\rho_o = (0.7767 - 0.0288 \theta) \times 1.94 \text{ slug/ft}^3 \quad (\text{B.20})$$

where

$$\theta = \frac{T - 609.67 \text{ R}}{50 \text{ R}} \quad (\text{B.20a})$$

The variation of density with pressure is given by

$$dz = \frac{d\rho}{\rho} = -\beta dT + \kappa dp \quad (\text{B.21})$$

Integration along the isotherm  $T_o = 609.67 \text{ R}$  and from  $p' = 0 \text{ psig}$  to  $p' = p$ , using Eq. B.18 results in

$$z(p, T_o) = a(T_o)p + b(T_o) \frac{p^2}{2}$$

Integration along the isobar  $p$ , from  $T' = T_o$ , to  $T' = T$ , using Eq. B.25, yields

$$z(p, T) - z(p, T_o) = p \int_{T_o}^T a'(T') dT' + \frac{p^2}{2} \int_{T_o}^T b'(T') dT' - \int_{T_o}^T c(T') dT'$$

which after simplification reduces to

$$\rho = \rho_o e^{ap + \frac{1}{2} b p^2} \quad (\text{B.22})$$

where values for  $a$  and  $b$  are given in Eq. B.18a.

Maximum Error: 0.128%

### 3. Equation of State Explicit in Pressure

Since the equation of state is needed explicit in pressure, Eq. B.22 was rearranged to yield

$$p = \frac{1}{b} \left[ -a + \sqrt{a^2 + 2b \ln \frac{\rho}{\rho_o}} \right] \quad (\text{B.23})$$

#### 4. Isobaric Thermal Expansion Coefficient

The zero-pressure isobaric thermal expansion coefficient can be written as:

$$\beta_o = \frac{1}{\rho_o} \left( \frac{\partial \rho_o}{\partial T} \right)_p$$

where  $\rho_o$  is given by Eq. B.20.

or

$$\beta_o = \frac{0.0288}{50(0.7767 - 0.0288 \theta)} \quad \frac{1}{R} \quad (\text{B.24})$$

where  $\theta$  is given by Eq. B.20a.

Making use of Eq. B.21, together with the principle of an exact differential, one may write

$$\left( \frac{\partial \beta}{\partial p} \right)_T = - \left( \frac{\partial \kappa}{\partial T} \right)_p$$

From Eq. B.18, the variation of  $\beta$  with both pressure and temperature is given by

$$\beta = \beta_o - a' p - \frac{b'^2}{2} p^2 \quad (\text{B.25})$$

where the prime superscript indicates differentiation with respect to temperature. The expressions for  $a$  and  $b$  as a function of temperature are given in Eqs. B.17 and 19, respectively.

### 5. Specific Heat at Constant Pressure

Reference: Dow Corning, Bulletin 05-145, February 1966.

Data Points: The available data for the variation of zero pressure specific heat at constant pressure for 2 centistokes are:

t	$c_p^o$	
80 F	0.448	Btu/(lbm F)
160	0.454	
240	0.463	
320	0.476	
400	0.491	

Polynomial Fit:

Temperature Range: 540 to 860 R

Pressure Range: 2116 to 74116 lbf/ft<sup>2</sup>

Equation: The above data for 2 centistokes silicon oil were multiplied by the ratio of  $c_p^o$  for 1 centistoke to 2 centistokes at 77°F, to give the following expression for the zero-pressure specific heat at constant pressure for 1 centistoke silicon oil

$$c_p^o = (0.46 + 0.00471 \theta + 0.00141 \theta^2 + 0.000043 \theta^3) \times 32.174 \text{ Btu/(slug R)} \quad (\text{B.26})$$

where

$$\theta = \frac{T - 539.67 \text{ R}}{80 \text{ R}} \quad (\text{B.26a})$$

The variation of  $c_p$  with pressure is given by

$$c_p = c_p^o - T \int_{p_o}^p \left( \frac{\partial^2}{\partial T^2} \right) \left( \frac{1}{\rho} \right) dp' \quad (\text{B.27})$$

Since the exponent in Eq. B.22 is small, the equation for the density, when expanded in a power series, may be truncated after the second

term in the expansion. Applying Eq. B.27 to the two-term expression for density as a function of pressure and temperature by integrating along an isotherm  $T$  from  $p' = 0$  psig to  $p' = p$ , results in the expression for

$$c_p = c_p^o - \frac{TI}{\rho_o} \quad (B.28)$$

where

$$I = z_1 p + z_2 p^2 + z_3 p^3 + z_4 p^4 + z_5 p^5$$

and

$$z_1 = 2 \left( \frac{\rho_o'}{\rho_o} \right)^2$$

$$z_2 = \frac{1}{2} \left[ \left( 2 \frac{\rho_o'}{\rho_o} a' - a'' \right) - a z_1 \right]$$

$$z_3 = \frac{1}{3} \left[ a'^2 - \frac{b''}{2} + \frac{\rho_o'}{\rho_o} b' \right] - a \left( 2 \frac{\rho_o'}{\rho_o} a' - a'' \right) + \frac{1}{2} (a^2 - b) z_1$$

$$z_4 = \frac{1}{4} \left[ a' b' - a \left( a'^2 - \frac{b''}{2} + \frac{\rho_o'}{\rho_o} a' \right) + \frac{1}{2} (a^2 - b) \left( 2 \frac{\rho_o'}{\rho_o} a' - a'' \right) + \frac{1}{2} a b z_1 \right]$$

$$z_5 = \frac{1}{10} \left[ \frac{b'^2}{2} - 2a a' b' + (a^2 - b) \left( a'^2 - \frac{b''}{2} + \frac{\rho_o' b'}{\rho_o} \right) + a b \left( 2 \frac{\rho_o'}{\rho_o} a' - a'' \right) + \frac{1}{2} b^2 z_1 \right]$$

where the prime superscript indicates differentiation with respect to temperature. The symbol  $a$  represents the zero pressure isothermal compressibility defined in equation B.17 and  $b$  is defined by equation B.19.

Maximum Error: For zero pressure specific heat at constant pressure the maximum error was 0.065%. For higher pressures no experimental data were available for comparison. However, the

expression for enthalpy was numerically differentiated with respect to temperature at constant pressure and compared with the computed values of specific heat at constant pressure. The comparison showed no difference within the accuracy of the computation.

## 6. Enthalpy

The variation of enthalpy with both pressure and temperature is given by:

$$dh = c_p^0 dT + \frac{1}{\rho} [1 - TB] dp$$

This expression was integrated along the isobar  $p = 0$  psig from  $T' = 539.65$  R to  $T' = T$ , and then along an isotherm  $T$  from  $p' = 0$  psig to  $p = p$ , to give

$$\begin{aligned} h = & \{ 80(0.46 + 0.00471 \theta + 0.00141 \theta^2 + 0.000043 \theta^3) \\ & + \frac{1}{\rho_o \cdot 337.37} \left[ -\frac{1}{20} b b' T p^5 - \frac{1}{8} (a b' + a' b) T p^4 \right. \\ & + \frac{1}{6} (T(b' - a a') - b(1 + T \frac{\rho_o'}{\rho_o})) p^3 + \frac{1}{2} (a' T - a (1 \\ & \left. + T \frac{\rho_o'}{\rho_o})) p^2 + (1 + T \frac{\rho_o'}{\rho_o}) p \right] \times 32.174 \} \text{ Btu/slug} \quad (B.29) \end{aligned}$$

where  $\theta$  is given by Eq. B.26a  
 $T$  is temperature in R  
 $a$  is given by Eq. B.17 in  $\text{in}^2/\text{lbf}$   
 $\rho_o$  is given by Eq. B.20 in  $\text{slug}/\text{ft}^3$   
 $b$  is given by Eq. B.19 in  $\text{in}^4/\text{lbf}^2$   
 $p$  is pressure in  $\text{lbf}/\text{in}^2$   
 and primes denote differentiation with temperature.



## 7. Dynamic Viscosity

Reference: Dow Corning, Bulletin 05-153, July 1966.

Data Points:

t	v
0 F	1.98 centistokes
100	0.874
200	0.56
300	0.41

Polynomial Fit:

Temperature range: 460 to 760 R

Equation:

$$\ln v = (0.683 - 1.0845 \theta + 0.3065 \theta^2 - 0.04 \theta^3) \quad (\text{B.30})$$

$$\text{where } \theta = \frac{T - 459.67 \text{ R}}{100.0 \text{ R}}$$

where  $v$  is in centistokes and the dynamic viscosity is given by

$$\mu = v \rho \quad (\text{B.31})$$

Maximum Error: 0.77%

## 8. Thermal Conductivity

Reference: Dow Corning, Bulletin 05-145, February 1966.

Data Points: The available data for the variation of thermal conductivity with temperature for 2 centistokes are:

t	k	
- 100 F	0.0674	Btu/(hr ft F)
100	0.0626	
300	0.0578	

Polynomial Fit:

Temperature Range: 360 to 860 R

Equation: The procedure that was used for specific heat at constant pressure was followed to get an expression for the variation of thermal conductivity with temperature for 1 centistoke silicon oil. The resulting expression for the thermal conductivity is given by

$$k = (7.0052 - 2.2105 \times 10^{-3} T/R) \times 10^{-2} \text{ Btu/(hr ft R)} \quad (\text{B.32})$$

Maximum Error: There was no difference between the computed and the input data within the accuracy of the computation.

#### 9. Temperature Variation of Thermal Conductivity

Eq. B.32 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{- 2.2105 \times 10^{-3}}{7.0052 - 2.2105 \times 10^{-3} T/^{\circ}\text{R}} \quad 1/R \quad (\text{B.33})$$

## III NAK - (78.6% K)

The following physical properties of NaK(78.6 wt% K) were extracted from the latest version of the "Liquid Metals Handbook, Sodium and NaK Supplement" (to be published). Some typical properties of NaK are:

Melting Point: 92 F

Boiling Point: 1445 F

Surface Tension: 0.00739 lbf/ft at Melting point

Since all property values were taken from this single reference, the reference is omitted in each section below.

1. Equation of State Explicit in Density

Data Points: Values for the variation of density with temperature at zero pressure are given by

t	$\rho_o$
200 F	53.21 lbm/ft <sup>3</sup>
500	50.68
800	48.15
1100	45.62
1400	43.09

Polynomial Fit:

Temperature Range: 660 to 1860 R

Equation:

$$\rho_o = (58.773064 - 0.008433 T/R)/32.174 \text{ slug/ft}^3 \quad (\text{B.34})$$

Since the isothermal compressibility for NaK is assumed to be independent of pressure, Eq. B22 reduces to

$$\rho = \rho_o e^{Kp} \quad (\text{B.35})$$

where  $p$  is the gage pressure. Since the exponent is small, the power series expansion for Eq. B.35 may be truncated after the second term and the variation of density with both pressure and temperature is given by

$$\rho = \rho_o (1 + \kappa p) \quad (\text{B.36})$$

An expression for  $\kappa$  is given later in this section.

Maximum Error: At zero pressure, there was no difference between the computed and the input data, within the accuracy of the computation. For higher pressures there were no experimental data available for comparison.

## 2. Equation of State Explicit in Pressure

Since the equation of state is needed explicit in pressure, Eq. B.36 was rearranged to yield

$$p = \frac{1}{\kappa} \left( \frac{\rho}{\rho_o} - 1 \right) \quad (\text{B.37})$$

## 3. Isobaric Thermal Expansion Coefficient

The isobaric thermal expansion coefficient is defined in Eq. B.4. From Eq. B.34, the zero pressure isobaric thermal expansion coefficient is given by:

$$\beta_o = \frac{0.008433}{(58.773064 - 0.008433\text{TR}^{-1})} \frac{1}{R} \quad (\text{B.38})$$

Due to the lack of experimental data, the isobaric thermal expansion coefficient was assumed to be independent of pressure.

#### 4. Isothermal Compressibility

In view of the experimental difficulties associated with the measurement of isothermal compressibility at elevated temperatures, such data are not generally available for liquid metals. However, the well-known relationship between velocity of sound  $c$ , density  $\rho$ , and isentropic compressibility  $\kappa_s$  is

$$\kappa_s = \frac{1}{\rho c^2} \quad (\text{B.39})$$

which makes an alternative approach to the problem possible, if velocities of sound can be measured. Under these circumstances the isothermal compressibility may be obtained from the relation

$$\kappa = \gamma \kappa_s \quad (\text{B.40})$$

where

$$\gamma = \frac{c_p}{c_v}$$

The relation between  $c_p$  and  $c_v$  is given by

$$c_p - c_v = \frac{T\beta^2}{\rho\kappa} \quad (\text{B.41})$$

From Eqs. B.40 and B.41, one gets

$$\kappa = \kappa_s + \frac{T\beta^2}{\rho c_p^2}$$

or

$$\kappa_o = \left\{ \frac{1}{\rho} \left[ \frac{1}{c^2} + \frac{T\beta^2}{c_p^2} \right] \right\}_{T_o, P_o} \quad (\text{B.42})$$

Due to the lack of experimental data Eq. B.42 was evaluated at the absolute pressure  $p_o$  of one atmosphere and the temperature of  $T_o = 1260$  R.

$$\text{at } T_o = 1260 \text{ R}$$

$$\rho_o = 48.15 \text{ lbm/ft}^3$$

$$c_o = 7544 \text{ ft/sec}$$

$$\beta_o = 1.75149 \times 10^{-4} \text{ 1/R}$$

$$C_{p,o} = 0.2091 \text{ Btu/(lbm R)}$$

### 5. Specific Heat at Constant Pressure

Data Points: For zero-pressure specific heat at constant pressure are given by:

t	$c_p^o$
200 F	0.2255 Btu/(lbm F)
500	0.1239
800	0.2093
1100	0.2091
1400	0.2120

Polynomial Fit:

Temperature Range: 660 to 1860 R

Equation:

$$c_p^o = (0.2255 - 0.016292 \theta + 0.00539 \theta^2 - 0.000758 \theta^3 +$$

$$0.000054 \theta^4) \times 32.174 \text{ Btu/(slug R)}$$

(B.44)

where

$$\theta = \frac{T - 659.67 \text{ R}}{300 \text{ R}} \quad (\text{B.44a})$$

The variation of  $c_p$  with pressure is given by

$$c_p = c_p^o - T \int_{p_o}^p \frac{\partial^2}{\partial T^2} \left( \frac{1}{\rho} \right)_p dp'$$

Integrating along the isotherm  $T$  from  $p = 0$  psig to  $p' = p$ , using Eqs. B.36 and B.38 results in

$$c_p = c_p^o - \frac{2T\beta_o^2}{\rho_o \kappa} \ln (1 + \kappa p) \quad (\text{B.45})$$

Maximum Error: At zero pressure, the maximum error was 0.075%.

## 6. Enthalpy

The variation of enthalpy with both pressure and temperature is given by

$$dh = c_p dT + \frac{1}{\rho} [1 - T\beta] dp$$

The enthalpy was arbitrarily chosen to be zero near the melting point, or  $T = 469.67^\circ \text{R}$ . The above expression was integrated along an isobar  $p = 0$  psig from  $T' = 469.67^\circ \text{R}$  to  $T' = T$ , and then along an isotherm  $T$  from  $p' = 0$  psig to  $p' = p$ , to give

$$h = \left[ 300 \left( 0.2255 \theta - 0.016292 \frac{\theta^2}{2} + 0.00539 \frac{\theta^3}{3} - 0.000758 \frac{\theta^4}{4} + 0.000054 \frac{\theta^5}{5} \right) + \frac{1 - T\beta}{\rho_o \kappa} \ln (1 + \kappa p) \right] \times \quad (\text{B.46})$$

$$\frac{1}{778.26} \text{ ] } 32.174 \text{ Btu/slug}$$

where  $\rho$  and  $\rho_o$  are in  $\text{lbm/ft}^3$ ,  $\kappa$  is in  $\text{ft}^2/\text{lb f}$   $T_o$  in R,  $\theta$  is given in Eq. B.44a,  $\beta$  is in  $\text{R}^{-1}$  and  $p$  is gage pressure in  $\text{lb f/ft}^2$ .

### 7. Thermal Conductivity

Data Points:

t	k
200 F	13.36 Btu/(hr ft F)
500	14.57
800	15.18
1100	15.03
1400	14.13

Polynomial Fit:

Temperature Range: 660 to 1860 R

Equation:

$$k = (13.36 + 1.414167 \theta - 0.142083 \theta^2 - 0.069167 \theta^3 + 0.007083 \theta^4) \text{ Btu/(hr ft R)} \quad (\text{B.47})$$

where

$$\theta = \frac{T - 659.67 \text{ R}}{300 \text{ R}}$$

Maximum Error: 0.2%

### 8. Temperature Variation of Thermal Conductivity:

Eq. B.47 was differentiated with respect to temperature to yield

$$\frac{1}{k} \frac{dk}{dT} = \frac{1}{300} \frac{(1.414167 - 0.284166\theta - 0.207501\theta^2 + 0.028332\theta^3)}{(13.36 + 1.414167\theta - 0.142083\theta^2 - 0.069167\theta^3 + 0.007083\theta^4)} \frac{1}{R} \quad (\text{B.48})$$



## 9. Dynamic Viscosity

Data Points:

t	$\mu$
200 F	1.1316 lbm/(ft hr)
500	0.746
800	0.534
1100	0.411
1400	0.340

Polynomial Fit:

Temperature Range: 660 to 1860 R

Equation:

$$\mu = (1.316 - 0.896667 \theta + 0.419833 \theta^2 - 0.102833 \theta^3 + 0.009667 \theta^4)/32.174 \quad \text{slug/(ft hr)} \quad (\text{B.49})$$

where

$$\theta = \frac{T - 659.67R}{300R}$$

Maximum Error: 1.2%

## IV. FC-75 INERT FLUOROCHEMICAL LIQUID

The thermodynamic and transport properties of FC-75 fluid are extracted from "3M Brand Inert Fluorochemical Liquids, 3M Company, Chemical Division, 1965."

Due to the lack of experimental data, all properties were evaluated at atmospheric pressure and were assumed to be pressure independent.

At one atmosphere some typical properties are:

Nominal Boiling Point: 216 F  
 Pour Point: - 135 F  
 Surface Tension, at 77F: 15 dynes/cm

### 1. Equation of State

Data Points:

t	$\rho$	
- 50 F	120.7	lbm/ft <sup>3</sup>
70	110.5	
190	100.3	

Polynomial Fit:

Temperature Range: - 80 to 216 F

Equation:

$$\rho = (155.522 - 0.085 T R^{-1}) \times 32.174 \text{ slug/ft}^3 \quad (\text{B.50})$$

Maximum Error: There was no difference between the computed and input data, within the accuracy of computation.

## 2. Isobaric Thermal Expansion Coefficient

Using the definition of the isobaric thermal expansion coefficient, Eq. B.4, and Eq. B.50, one obtains

$$\beta = \frac{0.085}{155.522 - 0.085 T R^{-1}} \frac{1}{R} \quad (\text{B.51})$$

## 3. Isothermal Compressibility

Since the equation of state (Eq. B.50) was assumed to be pressure independent, the isothermal compressibility defined by Eq. B.7 was assigned the value of zero

$$\kappa \equiv 0$$

## 4. Specific Heat at Constant Pressure

**Data Points:** The variation of the zero pressure specific heat at constant pressure is given by:

t	$c_p^0$
80° F	0.2464 Btu/(lbm F)
140	0.2610
200	0.2756

**Polynomial Fit:**

Temperature Range: 70 to 210 F

Equation:

$$c_p^0 = (0.115082 + 2.4333 \times 10^{-4} T R^{-1}) \times 32.174$$

Btu/(slug R)

(B.52)

The variation of  $c_p$  with pressure is given by Eq. B.27. With  $\rho$  independent of pressure, Eq. B.27 was integrated along an isotherm  $T$  from  $p' = 0$  psig to  $p' = p$  resulting in:

$$c_p = c_p^o - \frac{2\beta^2 T}{\rho} p \quad (B.53)$$

A check on the magnitude of the terms in Eq. B.53, using typical running conditions, showed that the term  $2\beta^2 TP/\rho$  was only 0.00064% of  $c_p$ . Therefore,  $c_p$  was taken to be a function of temperature alone, namely

$$c_p = c_p^o \quad (B.54)$$

Maximum Error: 0.025%

## 5. Enthalpy

The variation of enthalpy with both pressure and temperature is given by

$$dh = c_p^o dT + \frac{1}{\rho} (1 - T\beta) dp \quad (B.55)$$

This expression was integrated along the isobar  $p = 0$  psig from  $T' = T = 324.67$  R to  $T' = T$ , and then along an isotherm  $T$  from  $p' = 0$  psig to  $p' = p$  to give

$$h = [0.115082 (T - T_o) + \frac{1}{2} \times 2.4333 \times 10^{-4} \quad (B.56)$$

$$(T^2 - T_o^2)] \times 32.174 + \frac{p}{778.26 \rho} (1 - T\beta) \text{ Btu/slug}$$

where  $\rho$  is in slug/ft<sup>3</sup>,  $p$  is gage pressure in lbf/ft<sup>2</sup>,  $T$  is in R and  $\beta$  is in R<sup>-1</sup>.

A check of the magnitude of the terms in Eq. B.56, using typical running conditions, showed that the last term which accounts for the pressure variation is only 0.00087% of  $h$ . Hence the enthalpy was taken to be a function of temperature alone, or

$$h = [0.115082 (T - T_0) + \frac{1}{2} \times 2.4333 \times 10^{-4} (T^2 - T_0^2)] \times 32.174 \text{ Btu/slug} \quad (\text{B.57})$$

**Maximum Error:** No data for enthalpy at atmospheric pressure were available for comparison. However, when the value of  $c_p$  (Eq. B.52) was compared with the result of differentiation of  $h$  with respect to temperature, there was no difference within the accuracy of the computation.

## 6. Thermal Conductivity

**Data Points:**

t	k
- 50 F	0.08745 Btu/(hr ft F)
50	0.0809
150	0.0744

**Polynomial Fit:**

**Temperature Range:** - 100 to 216 F

**Equation:**

$$k = 0.114181 - 6.53 \times 10^{-3} T R^{-1} \text{ Btu/(hr ft F)} \quad (\text{B.58})$$

**Maximum Error:** 0.044%

### 7. Dynamic Viscosity

Data Points: The variation of kinematic viscosity with temperature is given by:

t	v	
- 50 F	5.15	centistokes
10	1.74	
70	0.84	
130	0.50	

Polynomial Fit:

Temperature Range: - 80 to 190 F

Equation:

$$v = e (1.639 - 1.312933 \theta + 0.25265 \theta^2 - 0.02471667 \theta^3) \text{ centistokes} \quad (\text{B.59})$$

where

$$\theta = \frac{T - 409.67 \text{ R}}{60 \text{ R}}$$

The dynamic viscosity is given by

$$\mu = \nu \rho \quad (\text{B.60})$$

Maximum Error: 0.82%

## V. FC-43 INERT FLUROCHEMICAL LIQUID

The thermodynamic and transport properties of the FC-43 are extracted from "3M Brand Inert Fluorochemical Liquids, 3M Company, Chemical Division, 1965."

Due to the lack of experimental data, all properties were evaluated at atmospheric pressure and were assumed to be pressure independent.

At one atmosphere some typical properties are:

Nominal Boiling Point: 345 F  
 Pour Point: - 58 F  
 Surface Tension at 77F: 16 dynes/cm

### 1. Equation of State

Data Points:

t	$\rho$	
- 20 F	123.6	lbm/ft <sup>3</sup>
130	112.15	
280	100.75	

Polynomial Fit:

Temperature Range: - 50 to 340 F

Equation:

$$\rho = (157.0883 - 0.076167 T R^{-1}) \times 32.174 \text{ slug/ft}^3 \quad (\text{B.61})$$

Maximum Error: There was no difference between the computed and input data, within the accuracy of computation.

## 2. Isobaric Thermal Expansion Coefficient

Using the definition of the isobaric thermal expansion coefficient, Eq. B.4, and Eq. B.61, one obtains

$$\beta = \frac{0.076167}{157.0883 - 0.076167 T R^{-1}} \frac{1}{R} \quad (\text{B.62})$$

## 3. Isothermal Compressibility

Since the equation of state (Eq. B.61) was assumed to be pressure independent, the isothermal compressibility defined by Eq. B.7 was assigned the value of zero.

$$\kappa \equiv 0$$

## 4. Specific Heat at Constant Pressure

Data Points: The variation of zero pressure specific heat at constant pressure is given by:

t	$c_p^0$
40 F	0.25 Btu/(lbm F)
77	0.27

Polynomial Fit:

Temperature Range: 40 to 77 F

Equation:

$$c_p^0 = (-0.020092 + 5.4054 \times 10^{-4} T R^{-1}) \times 32.174$$

Btu/(slug R)

(B.63)



A similar procedure to that used in the case of FC-75 (See Section B IV 4) has shown that to a close approximation  $c_p$  is pressure independent, or

$$c_p = c_p^0 \quad (\text{B.64})$$

Maximum Error: There was no difference between the computed and input data within the accuracy of computation.

### 5. Enthalpy

The variation of enthalpy with both pressure and temperature is given by Eq. B.55. Integration of this equation in a procedure similar to that followed for the coolant fluid FC-75 yields

$$h = [-0.020092 (T - T_0) + \frac{1}{2} \times 5.4054 \times 10^{-4} (T^2 - T_0^2)] \times$$

$$32.174 + \frac{p}{778.26\rho} (1 - T\beta) \text{ Btu/slug} \quad (\text{B.65})$$

where  $T_0 = 401.67 \text{ R}$ ,  $\rho$  is in slug/ft<sup>3</sup>,  $T$  is in R,  $\beta$  is in R<sup>-1</sup>, and  $p$  is gage pressure in lbf/ft<sup>2</sup>.

A check of the magnitude of the terms in Eq. B.64, using typical running conditions, showed that the last term which accounts for the pressure variation is only 0.00084% of  $h$ . Therefore the enthalpy was taken to be a function of temperature alone or

$$h = [-0.020092 (T - T_0) + \frac{1}{2} \times 5.4054 \times 10^{-4} (T^2 - T_0^2)] \times$$

$$32.174 \text{ Btu/slug} \quad (\text{B.66})$$

Maximum Error: No data for enthalpy at atmospheric pressure were available for comparison. However, when the value of  $c_p$  (Eq. B.63) was compared with the result of differentiation of  $h$  with respect to temperature, there was no difference within the accuracy of the computations.

## 6. Thermal Conductivity

Data Points:

t	k
- 50 F	0.0512 Btu/(hr ft F)
50	0.0487
150	0.0462

Polynomial Fit:

Temperature Range: - 58 to 250 F

Equation:

$$k = 0.061442 - 2.5 \times 10^{-5} T R^{-1} \text{ Btu/(hr ft F)} \quad (\text{B.67})$$

Maximum Error: 0.11%

## 7. Dynamic Viscosity

Data Points: The variation of kinematic viscosity with temperature is given by:

t	$\nu$
- 20 F	15.80 centistokes
70	2.84
160	0.855
250	0.35

Polynomial Fit:

Temperature Range: - 80 to 320 F

Equation:

$$\nu = e (2.76 - 2.043483 \theta + 0.362 \theta^2 - 0.034717 \theta^3) \text{ centistokes} \quad (\text{B.68})$$

where

$$\theta = \frac{T - 439.67 \text{ R}}{90 \text{ R}}$$

The dynamic viscosity is given by

$$\mu = \nu \rho \quad (\text{B.69})$$

Maximum Error: 1.0%

## APPENDIX C

Optical Properties

Three optical properties are required in the radiative analysis discussed in Chapter 6, namely the total hemispherical emittance

$$\epsilon(T) = \frac{1}{E_b(T)} \int_0^{\infty} \epsilon_{\lambda}(T) E_{b,\lambda}(T) d\lambda \quad (C.1)$$

and the two auxiliary functions (see Eqs. 6.10 and 6.11)

$$XX(T_1, T_2) = \frac{1}{E_b(T_2)} \int_0^{\infty} \epsilon_{\lambda}(T_1) \epsilon_{\lambda}(T_2) E_{b,\lambda}(T_2) d\lambda \quad (C.2)$$

$$XXX(T_1, T_2, T_3) = \frac{1}{E_b(T_3)} \int_0^{\infty} \epsilon_{\lambda}(T_1) \epsilon_{\lambda}(T_2) \epsilon_{\lambda}(T_3) E_{b,\lambda}(T_3) d\lambda \quad (C.3)$$

In view of the temperature independence of the spectral emittance for dielectrics, the two functions XX and XXX are functions of a single temperature, the temperature of the surface element represented by the last subscript on the left-hand sides of Eqs. 6.10 and 6.11:

$$XX(T) = \frac{1}{E_b(T)} \int_0^{\infty} \epsilon_{\lambda}^2 E_{b,\lambda}(T) d\lambda \quad (C.4)$$

$$XXX(T) = \frac{1}{E_b(T)} \int_0^{\infty} \epsilon_{\lambda}^3 E_{b,\lambda}(T) d\lambda \quad (C.5)$$

## I. SURFACE COATING Z-93

The functions C.1, C.4 and C.5 of the previous section are evaluated for the zinc oxide/potassium silicate coating Z-93 on the basis of spectral reflectance data measured by IITRI and published in the NASA Contractor Report No. 1420, titled Emissivity Coatings for Low-Temperature Space Radiators, by G. R. Cunningham, J. R. Grammer, and F. J. Smith, Lockheed Aircraft Corp., Sunnyvale, Calif., Sept. 1969, pp. 66 through 81.

The evaluated functions defined through Eqs. C.1, 4 and 5 are collocated by power polynomials of this form

$$f(T) = \sum_{i=0}^N a_i T^i \quad (C.6)$$

For the total hemispherical emittance, a fourth degree power polynomial was found to be satisfactory with

$$\begin{aligned} a_0 &= 0.8990103 \\ a_1 &= -0.1400633 \times 10^{-3} \\ a_2 &= 0.387900 \times 10^{-6} \\ a_3 &= -0.3937509 \times 10^{-9} \\ a_4 &= 0.1015627 \times 10^{-12} \end{aligned}$$

For the auxiliary functions XX and XXX the coefficients are

XX	XXX
$a_0 = 0.7804112$	$0.6538383$
$a_1 = -0.5527205 \times 10^{-4}$	$0.1144374 \times 10^{-3}$
$a_2 = 0.2530228 \times 10^{-6}$	$-0.2432286 \times 10^{-7}$
$a_3 = -0.3229181 \times 10^{-9}$	$-0.1437500 \times 10^{-9}$
$a_4 = 0.8854202 \times 10^{-13}$	$0.4947915 \times 10^{-13}$

## APPENDIX D

## I. The Fin-To-Tube Shape Factor

A closed-form integration for the view factor of the fin with respect to the tube was carried out by Mr. Yao. This view factor occurs in Eqs. 6.15 and 6.16. Only the final results are given here.

The reader should recognize that some of the symbols defined below (Eqs. D.1 through 6) apply only here.

Let  $(x_f, y_f, z_f)$  designate the position of the center of an area element  $A_f$  on the fin and  $z_m$  the same on the tube. Let  $r_e$ ,  $s_t$  and  $s_r$  represent respectively, the outer tube radius, the fin tip and the fin root thickness, and let the fin height be given as  $H$ .

Then, with

$$\rho = x_f^2 + y_f^2 \quad (D.1)$$

$$\beta = \arctan \frac{y_f}{x_f} \quad (D.2)$$

$$\alpha = \arctan \frac{s_r - s_t}{2r_c} \quad (D.3)$$

$$\phi = \arctan \frac{s_r}{2r_e} \quad (D.4)$$

$$a = \rho^2 + r_e^2 + (z_f - z_m)^2 \quad (D.5)$$

$$\phi^* = \arcsin \frac{r_e}{\rho} \quad (D.6)$$

one obtains first

$$Z_1 = \ln \frac{a-2r_e \rho \cos(\phi^*-\beta)}{a-2r_e \rho \cos(\phi-\beta)} + \frac{a-2r_e^2 \cos(\alpha+\beta)}{a-2r_e \rho \cos(\phi^*-\beta)} - \frac{a-2r_e^2 \cos(\alpha+\beta)}{a-2r_e \rho \cos(\phi-\beta)} \quad (D.7)$$

$$Z_2 = \frac{8r_e^2 \rho^2 (z_t - z_m)^2 + 4\rho^2 r_e^2 a - a^3}{2(a^2 - 4r_e^2 \rho^2)^{3/2}} \quad (D.8)$$

$$Z_3 = \arcsin \frac{2r_e \rho - a \cos(\phi^*-\beta)}{a-2r_e \rho \cos(\phi^*-\beta)} - \arcsin \frac{2r_e \rho - a \cos(\phi-\beta)}{a-2r_e \rho \cos(\phi-\beta)} \quad (D.9)$$

$$Z_4 = \frac{2a(z_f - z_m)^2 - a^2 + 4r_e^2 \rho^2}{a^2 - 4r_e^2 \rho^2} \quad (D.10)$$

$$Z_5 = \frac{r_e \rho \sin(\phi^*-\beta)}{a-2r_e \rho \cos(\phi^*-\beta)} - \frac{r_e \rho \sin(\phi-\beta)}{a-2r_e \rho \cos(\phi-\beta)} \quad (D.11)$$

The final result is

$$SS = \Delta A_1 \Delta z \frac{-Z_1}{4\pi\rho} + \frac{\sin(\alpha+\beta)}{2\pi\rho} \frac{\phi^*-\phi}{2} + Z_2 Z_3 Z_4 Z_5 \quad (D.12)$$

This expression contains all the geometric relations that are required for fin-channel radiative interaction. It needs to be evaluated only once for every fin element.

The values of SS given by Eq. D-12 vary greatly depending on the fin and tube elements under consideration. Typically the shape factor between tube element and adjacent fin element is three to five orders of magnitudes larger than the shape factor between tube element and the next closest fin

element. If this large variation in shape factor is allowed to remain, unrealistic oscillations in the fin radiosity will occur and sizeable truncation errors will result when the radiant fluxes are integrated across the fin surface.

To eliminate these truncation errors the local shape factor for the root fin element is replaced by a mean value between two adjacent elements which have the same Z location. The mean value for the shape factor between a tube element and its adjacent fin element is calculated from

$$SS = \left[ \frac{1 - \sin \gamma}{\pi} \right] \left[ \phi_o + \frac{\sin 2\phi_o}{2} \right] \quad (D.13)$$

rather than Eq. D.12.

Expressions for  $\gamma$  and  $\phi_o$  are:

$$\gamma = \tan^{-1} \left[ \frac{Z_m - Z_f}{X_f} \right]$$

$$\frac{r_e [\cos \phi_o + \tan \phi_o \sin \phi_o] - y_f}{r_e + X_f} - \tan \phi_o = 0$$



## II. Tube to Tube Shape Factor

The simplified analysis has indicated that optimum dimensions of the radiator system will result in close tube spacing. Therefore for the case of an optiminually designed system, the radiant interaction between adjacent tubes must be taken into account. This section summarizes the results of the shape factor between two fin elements. The procedure used is Hottel's crossed string method which is valid for infinitely long elements that are generated by a straight line moving parallel to itself. The finite length of the tube elements to accounted for by multiplying Hottel's result by a weighting factor.

From Hottel's crossed string method (see Fig. 5 ) the shape factor between two infinitely long tubes is:

$$F_{1-2} = \left[ \frac{1}{\pi/2 - \phi} \right] \left\{ \gamma + \left[ \left( \frac{t}{R_2} \right)^2 + \left( \frac{2L}{R_2} - \cos \phi \right)^2 - 1 \right]^{1/2} - \left( \frac{2L}{R_2} - \cos \phi \right) \right\} \quad (D.14)$$

where

$$\phi = \sin^{-1} \left( \frac{t}{R_2} \right)$$

$t$  = one half the fin thickness at its root.

$R_2$  = outside radius of the tube

$L$  = one-half the distance between tube centers

$$\gamma = \frac{\pi}{2} - (\theta_1 + \phi_1)$$

$$\phi_1 = \tan^{-1} \left( \frac{t}{2L - R_2 \cos \phi} \right)$$

$$\theta_1 = \cos^{-1} \left( \frac{R_2 \sin \phi_1}{t} \right)$$

The weighting factor for two tube elements of width  $\Delta Z$  located at  $Z_1, Z_2$  (see Fig. 5 ) is

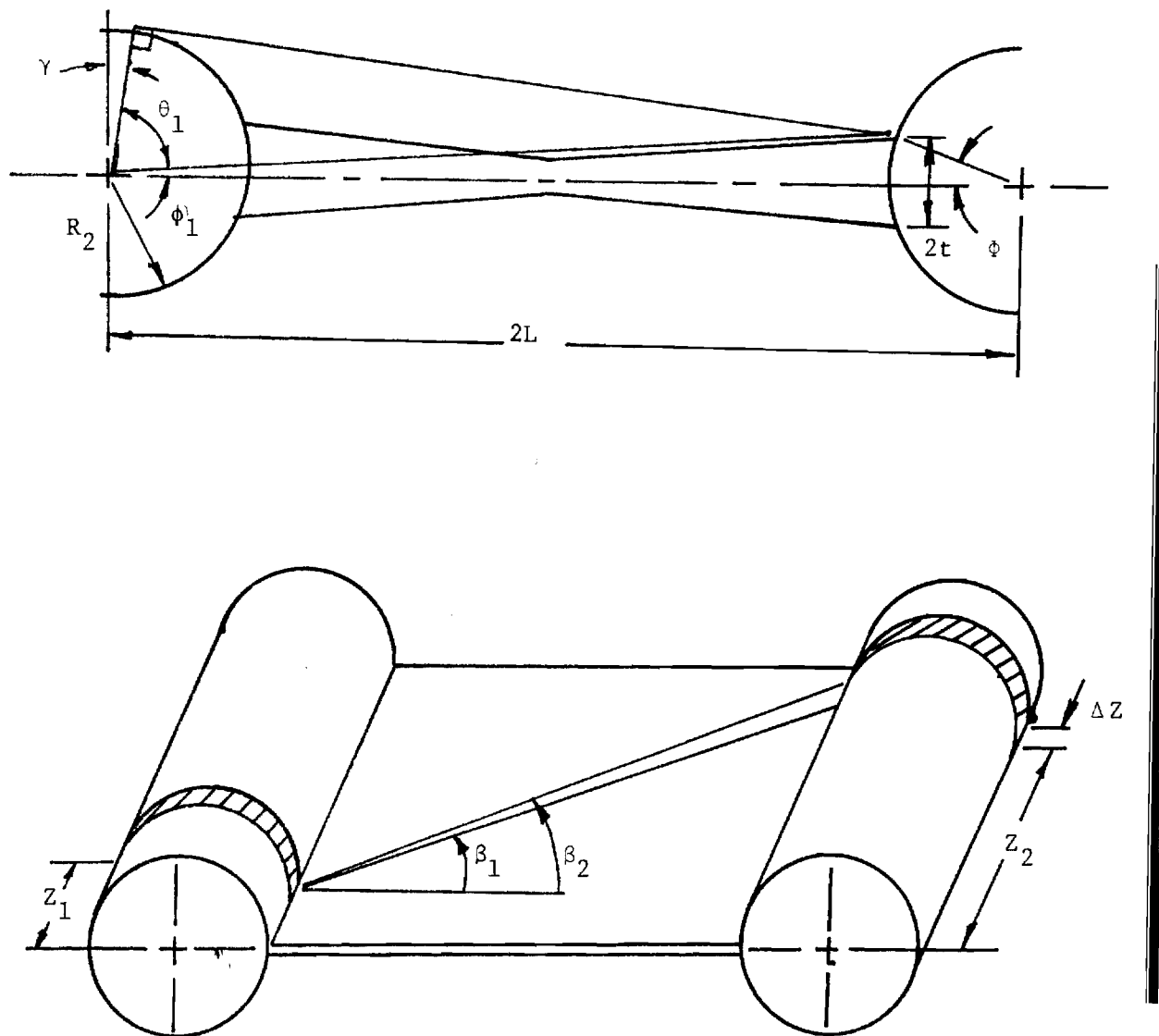


Fig. 5 Shape Factor Between Adjacent Tubes

$$WF = \int_{\beta_1}^{\beta_2} \cos^2 \beta d\beta = \frac{1}{2} (\beta_2 - \beta_1) + \frac{1}{4} (\sin 2\beta_2 - \sin 2\beta_1) \quad (D.15)$$

where

$$\beta_1 = \tan^{-1} \left[ \frac{z_2 - \Delta z/2 - z_1}{2(L-R_2)} \right]$$

$$\beta_2 = \tan^{-1} \left[ \frac{z_2 + \Delta z/2 - z_1}{2(L-R_2)} \right]$$

The shape factor between tube element is now

$$SS_{1-2} = \frac{WF}{\pi/2} F_{1-2} \quad (D.16)$$

where the value for  $F_{1-2}$  is given by Eq. D.14.

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